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RESEARCH REPORTS - 1980 NASA / ASEE SUMMER FACULTY FELLOWSHIP PROGRAM

University of Alabama
University, Alabama

and

University of Alabama in Huntsville
Huntsville, Alabama

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16. ABSTRACT <p>This document is a collection of reports on the research conducted by the participants in the 1980 NASA/ASEE Summer Faculty Fellowship Program which was held at the George C. Marshall Space Flight Center (MSFC) for the sixteenth consecutive year. The 1980 program ran from May 26, 1980 through August 1, 1980 and was conducted by the University of Alabama in cooperation with MSFC and the University of Alabama in Huntsville. As in the past, the program was operated under the auspices of the American Society for Engineering Education (ASEE) with sponsorship and funding by the University Affairs Office, NASA Headquarters, Washington, D.C. The University of Alabama and Marshall Space Flight Center conducted one of the seven Aeronautical and Space Research Programs which were funded nationally during the summer of 1980. The other programs were conducted at six other NASA centers throughout the country in cooperation with other universities.</p> <p>The fellows spent ten weeks working with MSFC personnel on an individual research problem of mutual interest to the fellow and MSFC.</p>					
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RESEARCH REPORTS

1980 NASA/ASEE SUMMER FACULTY FELLOWSHIP PROGRAM

George C. Marshall Space Flight Center
The University of Alabama
and
The University of Alabama in Huntsville

EDITORS:

Dr. B. F. Barfield, Professor of Mechanical Engineering
The University of Alabama

Mr. Marion I. Kent, Office of University Affairs
Marshall Space Flight Center

Dr. James Dozier, Director, Research and Technology Office
Marshall Space Flight Center

Dr. Gerald Karr, Professor of Mechanical Engineering
University of Alabama in Huntsville

NASA CR - 161511

PREFACE

For the sixteenth consecutive year, a NASA/ASEE Summer Faculty Fellowship Research Program was conducted at the Marshall Space Flight Center (MSFC). The program was conducted by The University of Alabama and MSFC during the period May 26, 1980 through August 1, 1980. The program was operated under the auspices of the American Society for Engineering Education (ASEE). The program at MSFC, as well as those at five other NASA Centers, was sponsored and funded by the Office of University Affairs, NASA Headquarters, Washington, D.C. The basic objectives of the programs, which are in the seventeenth year of operation nationally, are:

- a. To further the professional knowledge of qualified engineering and science faculty members;
- b. To stimulate an exchange of ideas between participants and NASA;
- c. To enrich and refresh the research and teaching activities of participants; institutions; and,
- d. To contribute to the research objectives of the NASA Centers.

The Faculty Fellows spent ten weeks at MSFC engaged in a research project commensurate with their interests and background and worked in collaboration with a NASA/MSFC Colleague. This document is a compilation of Fellow's reports on their research during the Summer of 1980. University of Alabama Report No. BER-259-94 presents the Co-Directors' report on the administrative operations of the Research Fellowship Program. Further information can be obtained by contacting any of the editors.

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1980

NASA/ASEE SUMMER FACULTY RESEARCH FELLOWSHIP PROGRAM

MARSHALL SPACE FLIGHT CENTER
THE UNIVERSITY OF ALABAMA

DEVELOPMENT OF A FACILITY
FOR LARGE-SCALE TESTING
OF MULTISTORY BUILDINGS

Prepared By:	Daniel P. Abrams, Ph. D.
Academic Rank:	Assistant Professor
University and Department:	University of Colorado at Boulder Civil, Environmental and Architectural Engineering
NASA/MSFC: (Laboratory) (Division)	Space Sciences Atmospheric Sciences
MSFC Counterpart:	Dr. Nicholas C. Costes
Date:	August 1, 1980
Contract No. :	NGT-01-002-099 (University of Alabama)



Development of a Facility for Large-Scale Testing of Multistory Buildings

By

Daniel P. Abrams, Ph. D.
Assistant Professor of Civil Engineering
University of Colorado
Boulder, Colorado

Abstract

Current experimental research pertaining to response of structures subjected to lateral loads such as strong winds or earthquake motions is limited to either tests of single structural components or small-scale (approximately one-tenth) multistory structures. The feasibility of developing a facility where large-scale multistory structures could be loaded to failure is discussed.

The test facility would consist of a series of hydraulic actuators mounted on reaction frames currently in use at the Marshall Space Flight Center for structural testing of spacecraft components. The actuators would be controlled from signals computed by an on-line analysis of measured data. This method of loading could be used to simulate inertial forces resisted by a structure behaving in the non-linear range of response subjected to motion at the base or to impulses along the height as would occur during a strong earthquake or wind.

Development and utilization of the test facility would occur in two phases. In the first phase, the testing system would be developed and verified for loading a structure within a single plane. The second phase would utilize the loading system by testing a half-scale ten-story reinforced concrete building 45 feet tall. Further development of the test facility may include the capability of loading individual floor levels to simulate translational and rotational inertia in three directions.

ACKNOWLEDGEMENTS

The writer expresses his appreciation to the National Aeronautics and Space Administration, and the University of Alabama and the American Society of Engineering Education for funding and conducting the Summer Faculty Fellowship Program at the Marshall Space Flight Center under the direction of Marion Kent of NASA and Dr. Robert Barfield of the University of Alabama.

Special gratitude is extended to Dr. Nicholas Costes who provided many creative ideas and stimulating conversations, and was responsible for introducing the writer to numerous helpful individuals at Marshall.

The writer wishes to thank Dr. George McDonough for his support of the project and George Shofner, Charles Watson, Frank Vinz and Gerry Waggoner for their technical advice and constructive comments.

Credit is given to Willa Eslick for her patience in typing the paper.

INTRODUCTION

"Earthquakes have caused, and can cause in the future, enormous loss of life, injury, destruction of property, and economic and social disruption."

This statement quoted from the Earthquake Hazards Reduction Act of 1977 [1] demonstrates the importance for safer structures in regions of high seismicity. More specifically, the primary objective of the Act passed by Congress is

"The development of technologically and economically feasible design and construction methods and procedures to make new and existing structures, in areas of seismic risk, earthquake resistant..."

In accordance with this objective, several investigators have examined both analytically and experimentally the response of structures subjected to strong ground motions. Of the experimental studies, investigations have been limited to response of one of two general specimen types: large-scale single components of more complex structures, or small-scale (on the order of one-twelfth) multistory structures. Large-scale multistory structures have not been tested to destruction because costs of developing a loading system and constructing a test structure were thought previously to be prohibitive.

This paper presents the feasibility of developing a facility where large-scale multistory structures could be loaded to failure. The proposed test facility would consist of a series of hydraulic actuators mounted on reaction frames currently in use at the Marshall Space Flight Center (MSFC) for structural testing of spacecraft components. Development and utilization of the test facility is discussed in terms of the cost-value relationship of using large-scale test structures for earthquake engineering research.

OBJECTIVES

Results of an experimental study using a large-scale test structure could be the necessary evidence required to transfer existing research findings to standards for building earthquake-resistant structures. Moreover, tests of large-scale structures may help discover and develop new methods of designing and constructing safer structures. Measured response of a large-scale multistory structure would:

- (1) verify the capability of the testing system to simulate inertial loads resulting from a programmed impulse;

(2) for the first time, provide comprehensive data for an in-depth study of both the overall and local response of a multistory structure subjected to a strong ground motion;

(3) verify currently used small-scale modeling procedures thus extend experimental research of multistory structures using shaking tables;

(4) provide a data base for testing newly developed analytical models used for calculating nonlinear dynamic response of multistory structures;

(5) serve as a "proof test" of recent improvements in earthquake-resistant design procedures developed with mathematical and small-scale physical models;

(6) provide data to evaluate methods of rehabilitation of existing and damaged structures such as epoxy grout injections or masonry in-fill panels.

The objective of the study presented in this paper is to examine the feasibility of developing a large-scale testing facility and to outline a test program utilizing the facility to illustrate these attributes of large-scale testing.

A hypothetical structure (Fig. 1) has been chosen as an example of a typical test article. The structure is a one-half scale reinforced concrete building consisting of two three-bay frames coupled to a slender shear wall at ten levels with floor slabs poured monolithically. The configuration was chosen to replicate a similar configuration previously used with small-scale models [2, 3, 4]. The scale was chosen so that sizes of members and reinforcement would be representative of actual construction. It is planned to develop a testing system to load this structure laterally at all ten levels from a single direction.

OUTLINE OF TEST PROGRAM

An outline of the test program is presented in Fig. 2. The program would consist of three stages. In the first stage the testing system would be developed using existing loading equipment at MSFC and a computer analysis that would be developed. The second stage would consist of verifying operation of the testing system by constructing and loading a replica of a small-scale (approximately 8-ft. tall) ten-story structure tested previously on a shaking table [2]. The loading procedure which would consist of a series of slowly applied loading reversals simulating inertial forces could be validated

by comparing displacement response of the structure tested on a shaking table with that of the structure loaded laterally with actuators. If agreement can be made, then the loading system would be developed further to test the 45-ft tall structure shown in Fig. 1 using the same load control system and on-line computer analysis with larger size actuators and reaction structure.

It is planned to test the 45-ft tall half-scale structure using the same input base - acceleration history as used for the 8-ft tall structure. In this way, scaling relationships may be examined for modeling reinforced concrete at small scales. After the structure has been tested, it can be repaired and loaded once again to investigate methods of rehabilitation. The test structure then can be used as a loading system for investigating the behavior of masonry sub-panels by repairing the structure once again, in filling each frame panel with masonry and retesting. In this manner, each of sixty frame panels, or masonry elements, will be subjected to a different ratio of shear to normal stress thus providing a larger population of data than currently exists in all of the past masonry research.

LOADING SYSTEM

The proposed loading system consists of a series of hydraulic actuators mounted to an existing reactor frame used presently for testing portions of the liquid oxygen tanks of the Space Shuttle. The reactor structure (Fig. 3) which is located within the structural test tower (Fig. 4) of the load test annex of building 4619 has sufficient lateral-load capacity to support the 20 actuators (two per level, Table 1) required to load the 45-ft tall test structure. Minor modifications of the reaction structure would be necessary to support individual actuators. In addition, the reaction structure could be used to support actuators mounted in transverse and vertical directions for further development of the loading facility.

A contingency plan could also be developed to use the existing test tower (Fig. 4) if removal of the reaction frame were necessary for other reasons. The test area measures 51 x 49 x 47 meters high. A double trolley crane with a total capacity of 54 metric tons is accessible for erection of the test structure. Further detailed information about the facility is given in Reference 5.

The existing load control system located in building 4619 has been cited [6] by the Japanese as being a unique system for controlling several actuators simultaneously. A closed-loop control system using a PDP-11 computer monitors response of a specimen at intervals of 1/60th of a second and adjusts the load to attain a preselected specimen response. This method of control is particularly useful if it is desired to displace a structure through

a specified deflected shape. Actuators loading a structure at several levels can be programmed to displace prescribed amounts despite interaction among actuators which has troubled investigators previously. The capability of the load-control system to coordinate simultaneous actuator movement is proven by numerous experiments including a recent one where several actuators were used to load in three directions an ortho-grid assembly used for part of the floor system in Space lab. The proposed loading facility would utilize ten of fifty-eight channels of the existing load-control system.

The actuators would be controlled from signals generated by an on-line analysis of measured data to simulate inertial loads resulting from a prescribed impulse. For a standard time increment of the impulse duration, each actuator would displace an amount calculated from previous force and displacement measurements and the prescribed forcing function. The differential equation to be satisfied for each floor level is:

$$m\ddot{x} + F_r = p(t) \quad (1)$$

where m is the mass of the floor and lumped portions of the columns and wall, \ddot{x} is the acceleration of the mass, F_r is the restoring force of the structure, and $p(t)$ is the prescribed forcing function. Using finite differences, the acceleration may be related to the displacement over a time increment by

$$\ddot{x}_i = (x_{i+1} - 2x_i + x_{i-1})/(\Delta t)^2 \quad (2)$$

where Δt denotes the time interval and i refers to the particular time increment. The new displacement, x_{i+1} , that controls the actuator movement, may be calculated from \ddot{x}_i which is determined from equation (1)

$$m\ddot{x}_i = p(t)_i - F_{ri} \quad (3)$$

For an impulse at the base of a multistory structure, $p(t)$ becomes $m\ddot{x}_0$ where \ddot{x}_0 is the input acceleration at base. Any prescribed ground motion including translational, rocking or torsional motions may be input to the on-line analysis to load the fixed-base structure "pseudo dynamically."

The restoring force, F_r , is measured continuously during the loading which is particularly important for a nonlinearly behaving structure because the restoring force or stiffness of the structure is constantly changing and sometimes unpredictable. For a nonlinear dynamics problem the fact that the inertial forces are dependent on the stiffness, and the stiffness is dependent on the loading history, justifies the on-line analysis as an irreplaceable technique for simulating inertial loads.

Because the pseudo-dynamic test method is not velocity dependent, the actuators may be operated as slow as the hydraulic power supply requires. Some investigators have even advocated that slowing the loading process down to a nearly static rate is good for observing propagation of cracks and other types of progressive damage. It should be noted that the loading method does not account for effects of viscous, or velocity dependent damping, and strain-rate effects. However, it is reasonable to assume that most of the energy dissipation in reinforced concrete structures behaving within the nonlinear range of response is attributable to hysteretic damping which is not velocity dependent. Furthermore, strain-rate effects for velocities corresponding to the fundamental mode and possibly the second mode of vibration of prototype structures have been shown to be negligible [7].

EXPERIMENTAL PROCEDURES

To economize on formwork the test structure (Fig. 1) would be erected from planar wall and frame elements cast in the horizontal plane and lifted to the vertical position. Reinforcement at base of columns and wall would be anchored in concrete blocks prestressed to the floor of the test area to prevent welding of reinforcing bars. Floor slabs could then be cast to complete erection of the structure. A minor amount of disassembly of the reaction structure may be necessary for lifting and placement of the structure.

Instrumentation would consist of two displacement and two load transducers at each level of the structure. Additionally, strain gages would be placed on reinforcing bars to detect internal forces in members and to infer bond characteristics between concrete and reinforcing bars. An estimated maximum of 200 channels of data would be a small percent of the 6000 channel Structural Test Data Acquisition System (STDAS) available at MSFC.

The test structure would be subjected to a series of lateral loads simulating response to the ground motion measured at El Centro, California, in the 1940 Imperial Valley earthquake (NS component). The intensity of the input motion would be scaled according to an analysis used for design of the structure so that similar levels of damage would be attained as for the small-scale structures. After the first test run, or loading, the structure would be loaded corresponding to an input motion of twice the intensity to further investigate strength and stiffness deterioration and energy dissipation characteristics. This pattern may be repeated until destruction occurs. At that time, the structure may be repaired using epoxy grout injections in the cracked concrete and tested once again. If the rehabilitation process can regain a sufficient amount of the original stiffness, the structure can be repaired once again and then strengthened with concrete-block masonry

placed in each frame panel. Instrumentation can be installed to measure normal and shear stresses on each masonry panel when the structure is loaded once again.

In addition to the pseudo-dynamic testing, a series of low-amplitude free and forced vibration tests can be run to determine dynamic characteristics such as modal frequencies, shapes, and damping factors.

COST ESTIMATE

Approximate costs of developing a large-scale testing facility to load a 45-ft tall ten-story structure are itemized below. Costs include verification of the pseudo-dynamic loading system, and construction of the reinforced concrete test structure. Cost estimates have been compiled by engineers at MSFC who would be directly involved with the proposed work.

(1) Verify pseudo-dynamic testing method. Costs include testing an 8-ft tall model structure with the newly developed test method.

Reaction structure	\$ 5,000
Loading system	\$20,000
Instrumentation	-
Construction of 8-ft model	\$8,000
	<hr/>
	\$33,000
Man months	30

(2) Develop large-scale testing facility. Costs include modifying existing structural test equipment and purchasing needed actuators, hardware, cables, hydraulic lines and miscellaneous items.

Reaction structure	\$30,000
Loading system	\$65,000
	<hr/>
	\$95,000
Man months	35

(3) Constructing and testing 45-ft tall structure

Construction of structure	\$85,000
Instrumentation	\$7,000
Disassembly	\$10,000
	<hr/>
	\$102,000
Man months	25

(4) Total	\$230,000
Man months	90

The overall cost of the project appears to be justifiable when compared with a similar attempt in Japan [6] and a previous feasibility study [8], each costing well over twenty million dollars.

EXTENDED DEVELOPMENT OF THE TEST FACILITY

Development of the loading system could be extended further in several steps to result ultimately in a loading system that could simulate translational inertia of each floor slab in three directions in addition to rotary inertia about three axes. Base motions input to the on-line analysis could then contain three translational components: two horizontal and one vertical, and three rotational components: torsion and rocking about each horizontal axis.

The existing reaction structure, hydraulic power supply, and load-control system could be utilized. Two additional horizontal actuators, and two additional vertical actuators would be required per floor level.

Operation of the loading system could be verified by constructing several test structures at a small scale in duplicate. One of each pair of structures would be tested on the six-degree-of-freedom shaking table located in building 4663. The other structure of the pair would be tested using the newly developed loading system. Comparing response of each structure would verify the reliability of the pseudo-dynamic test method for multiaxial loading.

Furthermore, the existing structural test tower (Fig. 4) located in the local test annex of building 4619 is sufficiently tall and strong to accommodate a test structure up to 100-ft tall. A twenty-story half-scale, or ten-story full-scale, building could be tested with the additional cost primarily attributable to the purchase and support of additional actuators.

SUMMARY AND CONCLUSIONS

The feasibility of developing a facility for testing large-scale multistory buildings to failure has been presented. The facility would consist of a series of hydraulic actuators mounted to existing structural test equipment at the Marshall Space Flight Center. Displacement of each actuator would be controlled by an on-line analysis of measured data to simulate motion of a multistory structure subjected to a prescribed series of impulses input at the base as would occur during an earthquake. Construction, instrumentation, and test procedures are outlined for testing a 45-ft tall reinforced concrete structure to illustrate the worth of large-scale testing and the approximate costs.

The role of a large-scale testing facility for earthquake engineering research has been shown to be consistent with the objectives of the Earthquake Hazards Reduction Act of 1977. Development of such a facility at the MSFC using existing structural test equipment appears to be feasible technically and justifiable economically.

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Table 1

Actuator Requirements
(2 per level)

<u>Level</u>	<u>Capacity</u> (kips)	<u>Bore</u> (inches)	<u>Stroke</u> (inches)
10	35	5	48
9	35	5	48
8	35	5	38
7	35	5	38
6	25	4	29
5	25	4	29
4	25	4	19
3	25	4	19
2	15	3	10
1	15	3	10

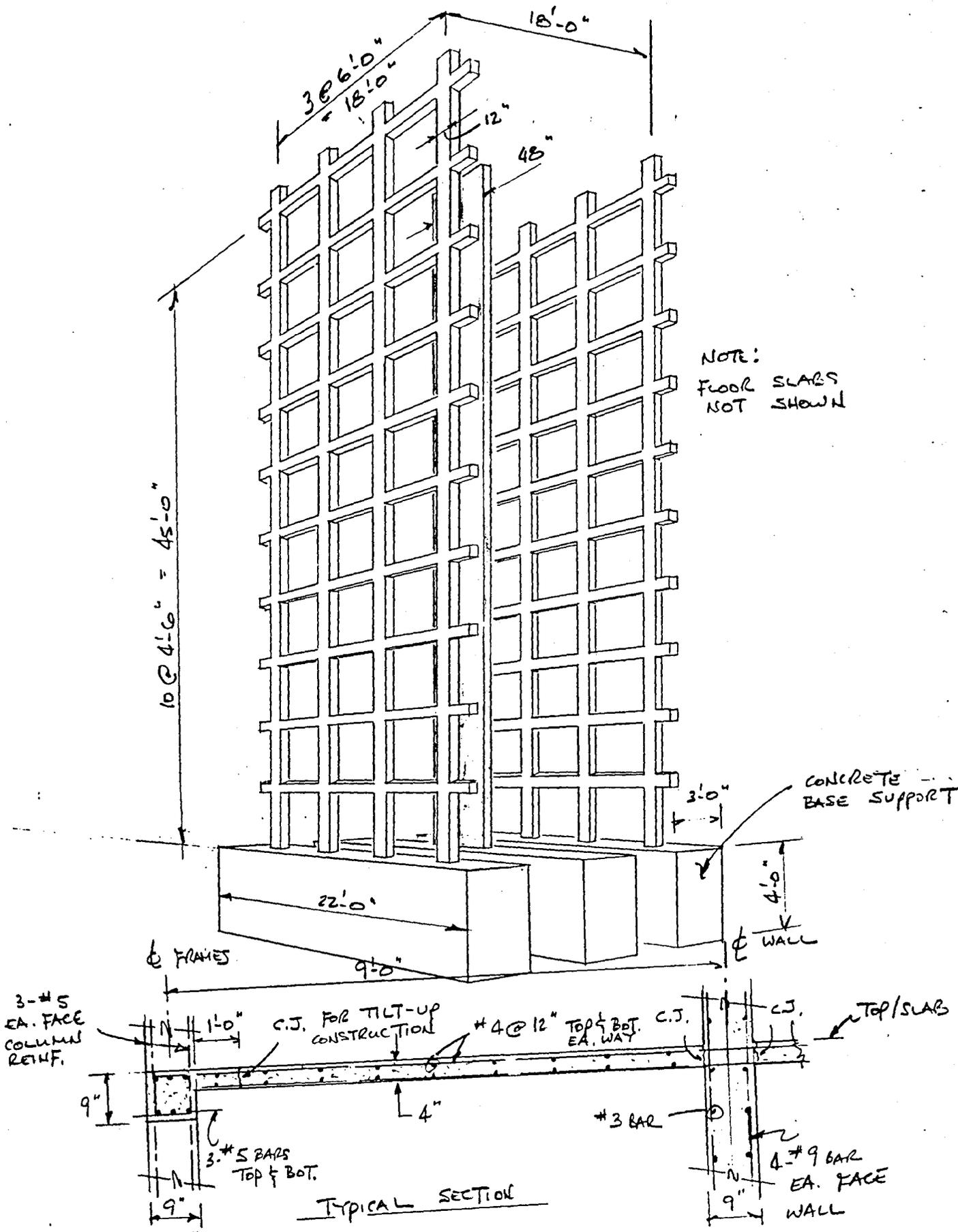


Fig. 1 Test Structure

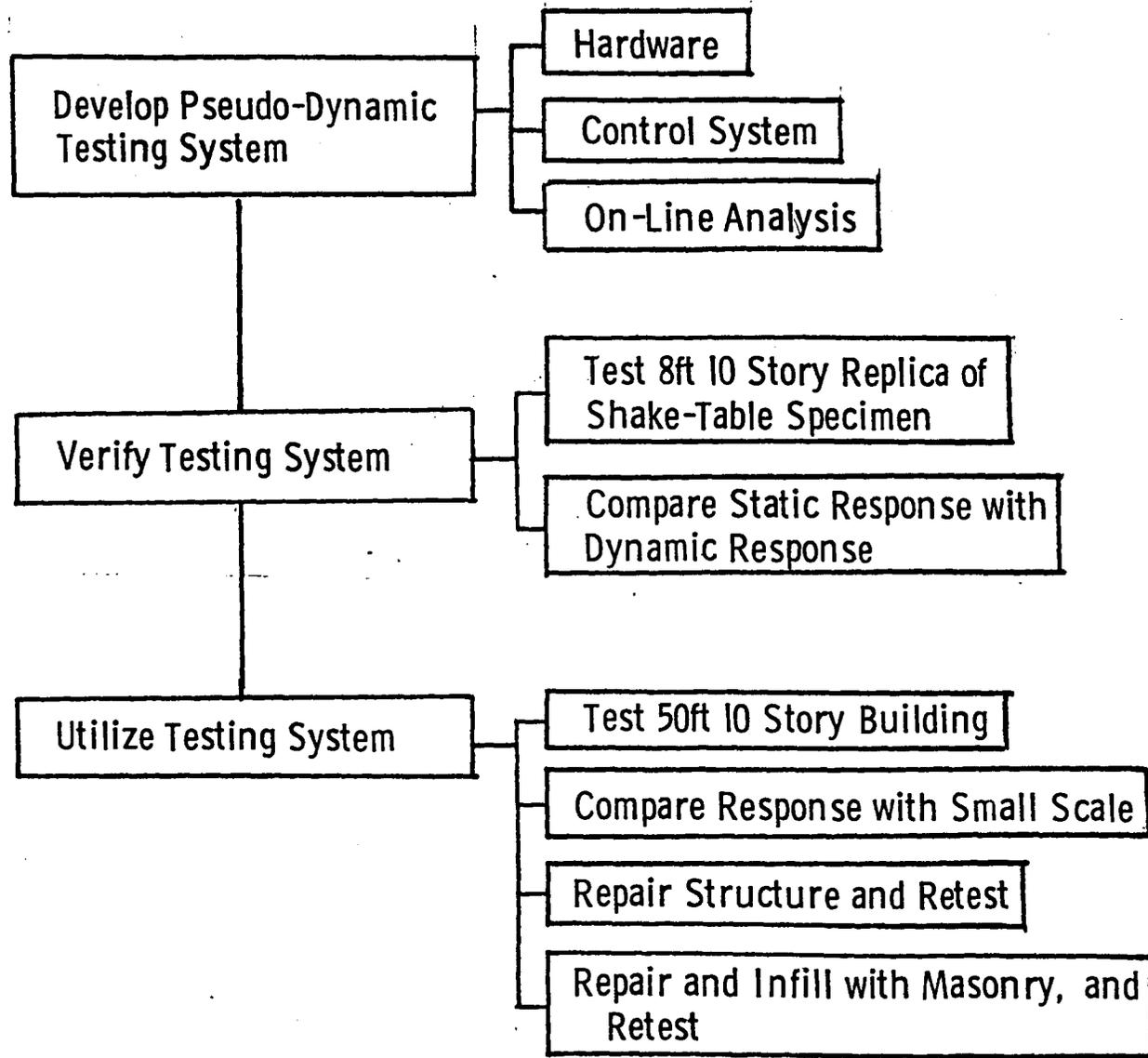


Fig. 2 Outline of Experimental Program

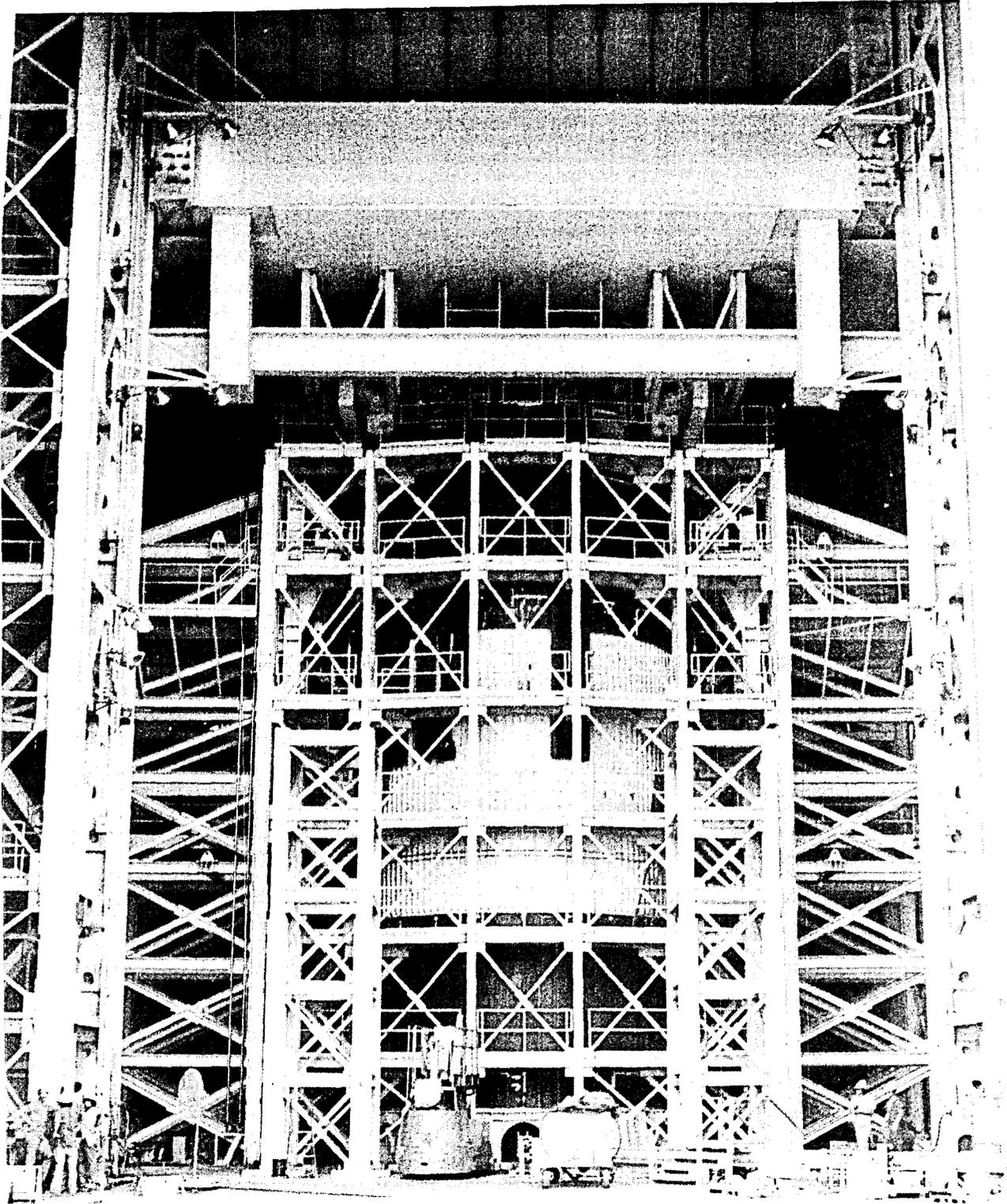


Fig. 3 Reaction Structure

I-14

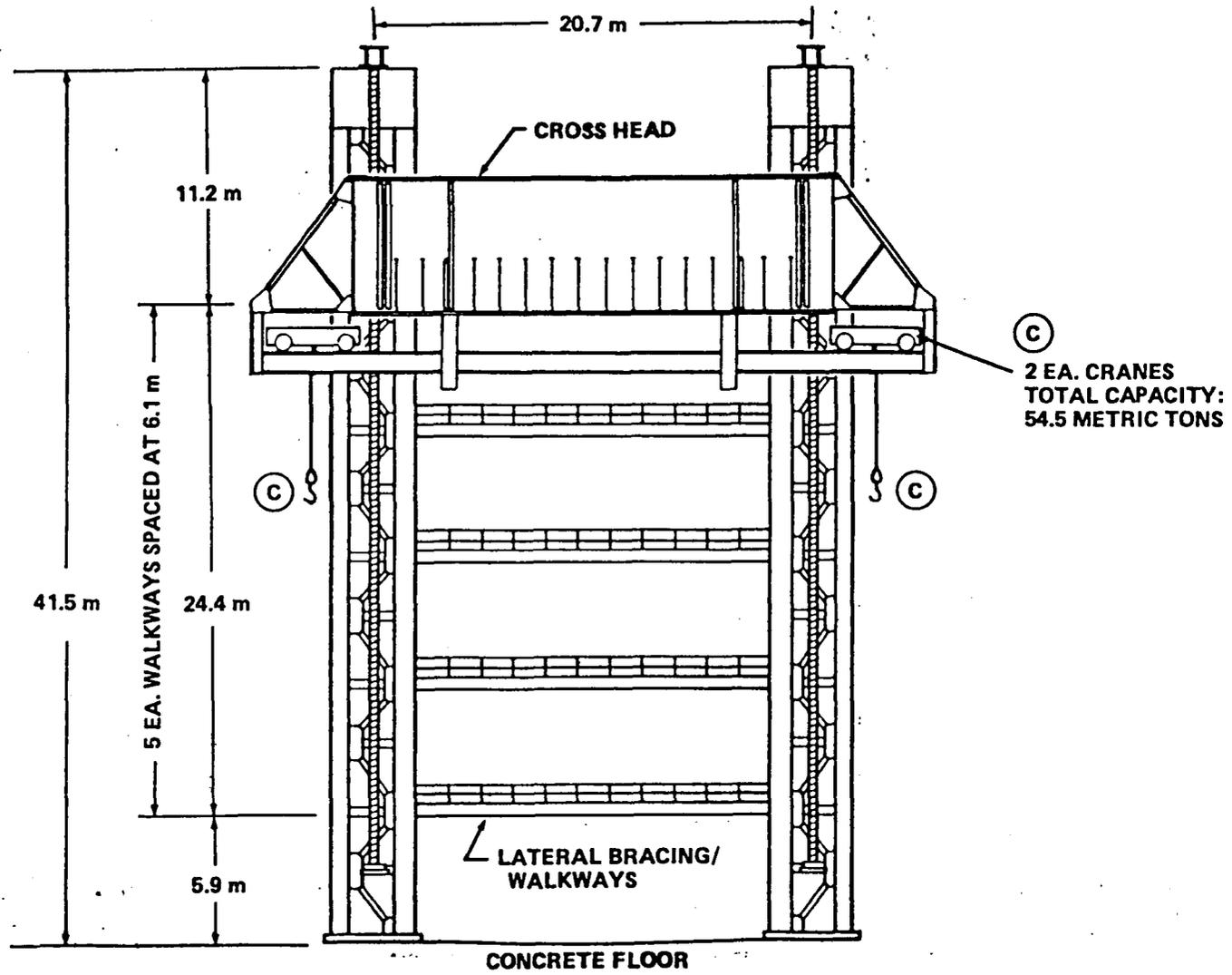


Fig. 4 Structural Test Tower

1980

NASA/ASEE SUMMER FACULTY RESEARCH FELLOWSHIP PROGRAM

MARSHALL SPACE FLIGHT CENTER
THE UNIVERSITY OF ALABAMA

MSFC PERSONNEL MANAGEMENT TASKS:
RECRUITMENT AND ORIENTATION OF NEW EMPLOYEES

Prepared By:	Thomas A. Brindley, Ph. D.
Academic Rank:	Associate Professor
University and Department:	The University of Alabama in Huntsville Department of Education
NASA/MSFC: Office Division	Personnel Office Personnel Development
MSFC Counterpart:	James R. Johnson
Date:	August 1, 1980
Contract No.:	NGT-01-002-099 (University of Alabama)

MSFC PERSONNEL MANAGEMENT TASKS:
RECRUITMENT AND ORIENTATION OF NEW EMPLOYEES

BY

Thomas A. Brindley, Ph. D.
Associate Professor of Education
The University of Alabama in Huntsville
Huntsville, Alabama

ABSTRACT

In order to encourage highly motivated young students to learn about NASA and consider it for a career, a formal program will be initiated whereby selected students can work on a voluntary basis at Marshall Space Flight Center. These students will work under the guidance and supervision of expert NASA employees. The first task has been to develop the working plan and procedures for this program, called Student Volunteer Service Program, in the writing of MSFC official guidelines, the Marshall Management Instruction (the MMI) which is a binding document that defines policy and establishes procedures and guidelines. Particular considerations written into the MMI after numerous consultations, interviews, and discussions about a satisfactory policy, include: arrangements to be made between the student, the school authorities, and concerned MSFC employees; management of the work assignments; and procedures for the student's welfare and safety.

The second task has been the development of a Recruitment Brochure for the attraction of new employees, especially scientists and engineers. Since private industry has certain advantages in hiring, particularly in offering higher initial salaries, it is imperative that NASA stress the valid reasons why working at MSFC would be of great benefit to the prospective employee. Thus, the brochure needs to stress aspects of ongoing NASA projects which show how various disciplines of science and engineering apply.

The third task assigned has been to develop a plan called Orientation of New Employees. The main intention is to establish a procedure whereby new employees can best become acquainted in their new work with a minimum of stress involved in coming to a new organization. This plan attempts to schedule various experiences that will enable the new employee to understand not only the particular work situation for which the employee is assigned, but to show the overall mission and operations of the Marshall Space Flight Center. The plan is to indicate the types of orientation believed necessary to aid the new employee in adjusting to work at MSFC.

ACKNOWLEDGEMENTS

I wish to thank Jim Johnson, Norm Hochberger, and Katy Lyle in particular for their constant and diligent review of my work, for their many suggestions, and for their great encouragement. Many others in the Personnel Office of MSFC have been very helpful and supportive not only in acquainting me with aspects of the specific tasks but in familiarizing me with the overall concerns of personnel management in general. I am also indebted to the many supervisors, scientists, engineers, technicians, and other employees in many different organizations of MSFC for taking their time to introduce me to various missions and to the many accomplishments of the dedicated people here at Marshall Space Flight Center. And, truly, I wish to express grateful appreciation to Helene Zazulak, who typed many manuscripts and this final report for me this summer.

INTRODUCTION

The Personnel Development Division of the Personnel Office requested that three tasks be accomplished by this NASA/ASEE Summer Faculty Fellow. The tasks were to investigate the needs and ramifications of several aspects of recruiting and orientation of employees and to write up a study or report on each of the subjects, taking each project as far as it could be carried in the ten weeks. Specifically, these management tasks were:

- 1) the orientation of new employees;
- 2) a recruiting brochure for the hiring of new scientists and engineers; and,
- 3) a student volunteer service program.

These three tasks have been completed and are reported here in respective sections.

The research undertaken involved the reading of all materials pertinent to past actions on the subjects and numerous interviews, consultations, and discussions with concerned MSFC employees about satisfactory policies related to these tasks. The tasks also demanded that this writer understand as much as possible about NASA and MSFC in a holistic, comprehensive way. Since the author had little previous acquaintance with MSFC in either a technological sense or in an organizational management sense, it was necessary to learn about NASA/MSFC in a short time so that the tasks could be begun. Thus, in effect this writer experienced his own orientation program.

VOLUNTARY STUDENT SERVICE PROGRAM

In 1978, the MSFC Personnel Office in response to various university and school requests in the Huntsville area initiated inquiries into the feasibility of forming a plan to describe means whereby exceptional students could work at MSFC on a voluntary basis. The initial exchange of memorandum and letters regarding the subject were described as the "Worksite Experience Program."

Laboratory supervisors were asked if they had needs in their organizations for visiting students. Several responded that they would be interested in allowing students to work in their laboratories. The Associate Director of the Center and the Director of Science and Engineering were especially interested in pursuing this program.

Coincidentally and unrelated to local efforts, guidelines were being discussed at the national level to provide for voluntary student services. The Civil Service Reform Act of 1978, allowed agencies to devise memoranda or official guidelines which would establish workable programs of this nature. MSFC temporarily delayed pursuing this further until the assignment for this NASA/ASEE Fellow.

Until the Civil Service Reform Act of 1978, there were no provisions for agencies of the Federal government to accept voluntary or gratuitous services from interested persons or parties. Thus, visiting individual researchers and interested students could find no means to offer services to a Federal agency because no employment status existed for them. There were no arrangements or agreements to allow students to take part in government programs gratuitously. Thus, a talented high school student could not work at MSFC in a laboratory where the youngster could gain in knowledge and experience. Such an opportunity would also encourage the youngster to think about future employment at NASA after completing college work.

The first task assigned and completed was to write a proposed Marshall Management Instruction (the MMI) which is a binding document that defines policy and establishes procedures and guidelines. The MMI sets up

a formal mechanism whereby selected, highly motivated young students can work on a voluntary basis at MSFC. The following study plan, following the format of the Marshall Management Instruction, describes the Voluntary Student Service Program.

STUDY PLAN: THE MARSHALL MANAGEMENT INSTRUCTION (DRAFT)

SUBJECT: Student Volunteer Service Program

1. PURPOSE

- a. To provide a means whereby qualified students can volunteer their services to the Center in order to gain experience and knowledge.
- b. To establish Center policy and procedures and to identify responsibilities in the management of the Student Volunteers.
- c. To establish guidelines for the Student Volunteers while they are working at the Center.

2. SCOPE

This instruction is applicable to all elements of MSFC.

3. REFERENCES

- a. FPM Chapter 308, Subchapter 7.
- b. FPM Letter 308-16, November 8, 1979.

4. POLICY

- a. High School students who show outstanding promise in their school work or school activities as determined by their school teachers and administrators in all areas of school study, especially, but not limited to science subjects, may wish to receive additional knowledge and expertise and be provided advanced opportunities to learn through on-the-job experience. These students will be allowed to work in a nonpay status on a voluntary basis at the Marshall Space Flight Center during a specified time and under the direction of a designated NASA employee.
- b. Each student assignment will be established and conducted through written agreement between the Marshall Center and the educational institution or with the organization so

- designated by schools or boards of education.*
- c. The participating student is not to be considered a Federal employee for any purposes other than injury compensation or laws related to the Federal Tort Claims Act. Service is not creditable for leave accrual or any other employee benefits.
 - d. Assignments will be tailored individually at the chief discretion of the project officer (Mentor).

5. DEFINITIONS

- a. The definitions of "volunteer service," "student," "half-time student," and "agreement" are those contained in FPM Chapter 308, Subchapter 7 and FPM Letter 308-16.
- b. Mentor -- the MSFC employee responsible for the student's work, welfare, and learning experiences, who is in charge of the student's assignments, scheduling, reporting, and evaluation, and who supervises the student.

6. BACKGROUND

The authority for the acceptance of unconditional gifts and services donated to the agency is contained in the National Aeronautics and Space Act, Section 203 (c)(4). Based upon this authority the Center has been developing a program to allow participation in Center activities by students who are willing to donate their time and effort, in response to interest expressed by local educational institutions. The Civil Service Reform Act of 1978 authorized the acceptance of volunteer service and the Office of Personnel Management implemented regulations outlining procedures for the development of such programs government-wide.

7. GENERAL PROVISIONS

- a. Responsibilities for the student's work at the Center shall be mutually entrusted to the Mentor. Participation in the program is conditioned upon the student's consent by written agreement to certain specified guidelines. Although students are not considered Federal employees, they will be allowed to work alongside and at the direction of Federal employees at the Center.
- b. This program is limited to students enrolled not less than half time in high school, trade school, technical or vocational institution, or junior college, college or university.

* Agreements and forms, as attachments, are not included here.

- c. Students receive no compensation.
- d. The student offers service (comes to work at NASA) with permission of sponsoring school or institution where student is enrolled.
- e. Nominations for participation will be made by the sponsoring school or institution where the student is enrolled. Selection shall be within the sole discretion of the Center.
- f. The student will not replace or displace any Center employee or staff a vacancy.
- g. The student will not be used to replace or substitute for routine and/or clerical duties while on assignment.
- h. The student will not be allowed to enroll or partake in MSFC training courses.
- i. Students will be at least 16 years old and must be at least in their junior year in High School. Students must be citizens of the United States. Participation shall not be based upon race, color, religion, sex, national origin, age, mental or physical handicap, marital status, or political affiliations, as prohibited by law, rule, or regulation.

8. RESPONSIBILITIES AND PROCEDURES

- a. The academic institution will be responsible for:
 - (1) Requesting in writing to the MSFC Center Director placement of nominated students in designated disciplines.
 - (2) Recommending students who have demonstrated a serious interest in gaining work experience at the Center and who have demonstrated outstanding curricular and/or extra-curricular achievements as deemed appropriate by the academic institution.
 - (3) Executing and complying with the terms of the Memorandum of Understanding with MSFC.
- b. The MSFC Personnel Office will be responsible for:
 - (1) Selecting participants from those nominated by their educational institution.
 - (2) Identifying work locations.
 - (3) Appointing a Mentor to supervise the student during the student's assignment at MSFC.
 - (4) Negotiating appropriate agreements.
 - (5) Documenting volunteer service on Standard Form 50 (SF-50), "Notification of Personnel Action."
 - (6) Establishing an Official Personnel Folder (OPF).
 - (7) Evaluating the program.
 - (8) Submitting any required reports.
 - (9) Managing a one week orientation program to acquaint the student with the overall NASA-MSFC mission.

- c. The MSFC Security Office will be responsible for:
 - (1) Providing proper authorization of the student's special status while assigned or visiting the Center.
 - (2) Acquainting the student with necessary security measures.
 - (3) Limiting the student only to non-security materials and areas.
- d. The MSFC Mentor will be responsible for:
 - (1) Supervising the student's work and learning experiences.
 - (2) Supervising the student's assignments and schedules.
 - (3) Assigning the student a definite worksite and monitoring the student's activities.
 - (4) Evaluating the student's performance and submitting any necessary reports to the Personnel Office.
 - (5) Maintaining informal relationships with responsible parties of the student's academic institution.
- e. The student will be responsible for:
 - (1) Complying with rules and regulations governing the internal operations of the Center. Any infractions of responsibilities may be cause for termination of the program.
 - (2) Being at the work location during the hours specified in the work experience Memorandum of Understanding or in other designated places upon approval of the Mentor.
 - (3) Following the authority of the assigned Mentor.
 - (4) Contacting the Mentor if absence from work is necessary.
 - (5) Caring for and properly maintaining all equipment and facilities used.
 - (6) Providing transportation to and from the MSFC worksite.
- f. Mutual agreements between parties:
 - (1) A written agreement, called the Memorandum of Understanding, will be signed by the Director of the Personnel Office and the representative of the participating institution. The agreement will explain the general assignments, time of assignment, and any special requirement desired by the parties.
 - (2) A "Notification of Personnel Action," Form 50, will be completed and signed by Department Head and Applicant.
 - (3) An "Application For Use of Research Facility, Equipment, or Computation Service" will be completed and signed by designated persons.
 - (4) A "Memorandum Agreement Relative to an Unconditional Donation of Services and Acceptance Thereof," will be completed and signed by designated persons.

THE RECRUITMENT BROCHURE

The second management task has been the development of a recruitment brochure for the attraction of new employees, especially scientists and engineers. In 1968, the first reduction in force (RIF) affected MSFC. From time to time thereafter various reductions of employees occurred through 1979, so that the work force which was once over 7000 in the heyday of the Apollo-Saturn missions fell by Summer, 1980, to 3600 employees. Coupled with this, there are over 1500 employees who have reached twenty years of service and are eligible for retirement. The rate of employee attrition will certainly increase in the next five years unless new people can be brought in. Although there has been some hiring in critical skill areas over the last twelve years, there is abundant need to recruit and hire scientists and engineers at MSFC to maintain a strong and active work force. Even though there is a 3561 end of fiscal year 1980 employment ceiling at MSFC imposed by Congressional budgetary considerations, new hiring is very necessary.

Recruitment teams are sent to selected universities each year. There are numerous kinds of materials, but there is no special handout available for recruiters to utilize which is aimed definitely at attracting employees to this center to work on the kinds of projects in process for which MSFC is renowned, and to praise the advantages of living in the Huntsville, Alabama area. A recruitment brochure is deemed necessary to enhance the search for new employees of critical skills in science and engineering.

There were several important considerations that were discussed in the numerous interviews and inquiries taken by this researcher in order to determine a final plan for the nature of the brochure. It was determined that the particular engineering and scientific skills in which a prospective employee has been educated should be shown in their direct relationship to the kinds of projects and activities being developed at this particular center. A prospective employee should feel that his or her university preparation and past experience will be properly utilized at MSFC. The employee has to feel that there are opportunities to work on significant projects, to acquire a sense of mission and purpose, to see one's work in meaningful relation to the whole, and to work with materials and processes that enhance one's skills and draw upon one's best talents. The new employee has to be shown that MSFC allows for personal and professional growth and development and that MSFC provides abundant opportunities for advancement and recognition. Since private industry has certain advantages in hiring, particularly in offering higher initial salaries, a recruiting brochure must show how the Federal pay and

promotion scale and the MSFC organization allow an employee to gain in pay and status rather rapidly over the years. New employees are concerned about the area in which they live. Thus, the many advantages of the good life in the Tennessee Valley should be described.

The actual development of the brochure itself included a number of steps. The researcher had to ascertain what the needs are at MSFC and what kind of employees are being sought. The researcher had to learn what MSFC offers that would be of concern and interest to possible new employees. What especially attracts recent college graduates and/or experienced hands? Can NASA/MSFC attract candidates of the highest quality in the face of considerable competition from other governmental organizations and from private industry?

Various federal agencies, other NASA centers, and some aerospace companies were contacted about their own recruiting efforts. After this information was gathered, the researcher had to discuss the plan of the brochure with various supervisors, chiefly the head of recruitment.

Then the researcher made several plans, called thumbnail sketches. Priorities of arrangement and order were determined. The final sketch was presented to the graphics department to make a draft mockup. This is the stage reached by the end of the seventh week. Thereafter, each draft must then be developed further and refined. Appropriate pictures must be acquired and adequate text must be written. Then a final mockup is made by the graphics department. This is presented to various directors for approval and then submitted to the printers for the final edition.

The organizational pattern of the draft mockup recruitment brochure as completed follows in this order. First, an attractive cover is planned to show the chief ongoing project for which MSFC is mainly responsible -- the Space Shuttle itself. At the beginning of the booklet the Director of the Center introduces MSFC to prospective employees. The Center itself is shown and the geographic relationship of MSFC to other field centers is represented. Chief accomplishments of NASA in the past, in the present, and planned for the future are pictorially shown. Then the main missions and projects of the Marshall Center are depicted. These include the Space Shuttle, the Spacelab Payloads, various important scientific laboratories, and some of the spin-offs, materials processing in space, and special assigned tasks at MSFC, such as solar heating and cooling. Integrated into the portrayal of these various missions are references indicating how the education and experience of qualified engineers and scientists work on each of the projects in particular. Then some pages describe briefly how future employees will benefit by

working at MSFC. Several pages show how employees have special advantages and opportunities at MSFC in advancement, career development, pay and status, training programs, and professional enhancement. Finally, the attractive surroundings of the Tennessee Valley and the numerous educational, cultural, and social advantages of this area of America are portrayed. The sense of living in a multicultural, pleasant, and attractive community is stressed.

The various thumbnail drafts and the first mockup from graphics are already completed at the time of this writing. However, the bulk of the draft brochure precludes it being inserted as a part of this report. The first draft mockup brochure serves as a springboard for its final development and incorporates and organizes the many conceptual ideas into a meaningful plan for further action. The ongoing continuation of the recruitment brochure lies in the hands of the Personnel Director.

ORIENTATION OF NEW EMPLOYEES

The third task completed has been to develop a study plan called the Orientation of New Employees. The main purpose is to establish a procedure indicating guidelines whereby new employees can best become acquainted in their new work with a minimum of stress involved in working in a new organization.

Presently very little is being done at MSFC in a formal manner to orient new employees about the ways of the Center. There is a one time meeting given by a representative of the Personnel Office, as presented by law in the guidelines of the Office of Personnel Management to describe the necessary work conditions and benefits of employees. This is not a true orientation since it is limited to a description of procedural matters relative to all NASA employees.

The Director and supervisors of the Personnel Office recognize the need for a completely developed orientation plan. But there are differing opinions among them and among various supervisors and employees throughout MSFC about the merits and the type of orientation that would be necessary. Informal inquiries taken among several new employees also show a division of opinion about the nature and need of an orientation program for the recently hired.

Some feel that orientation efforts are a waste of time, others feel they are beneficial. Some believe orientation meetings should be brief and narrowly construed. Others feel that meetings should be very

comprehensive of the entire MSFC operations beyond one's own workplace. Others feel that there is no need to learn about areas unrelated to one's work. Some feel that an orientation schedule should occur within the first few days of a new person's hire or, at most, but a week long. Others feel that a proper orientation cannot be hurried or overloaded in a short time frame, but should be extended over several months. There are many concerns about the nature of an orientation program. Does it encourage better morale in the work force? Does it enhance the sense of mission? Does it build cooperative efforts?

The main argument seems to be whether a new employee needs to become acquainted with the general workings of the new organization, or whether the new employee needs to learn only about one's own particular job and only needs to learn about the wider ramifications of the organization as the person's contacts develop from the job site. Although this might be an individual matter, there seems to be a broad difference in approach between generalists and particularists based upon one's own research training and education. Some scientists and engineers want to work alone and without much to do with others. Some scientists and engineers wish to see how their own discipline relates with others and wish to ascertain how their own particular skills might be relevant to other parts of the whole center.

Management, on the other hand, has to determine what is best for the new employee and what kind of orientation would lead to the best satisfaction of the missions and purposes of the center as a whole. There are conflicting views here, too. Some personnel supervisors feel that the integrated nature of much of the work at NASA demands a general understanding of MSFC as a whole and of different projects and laboratories other than one's own specialty or assigned area. Other supervisors feel that an extensive orientation program would be too time consuming on the parts of both the participants and the presenters and would be too costly.

The orientation plan devised by this researcher incorporates many approaches to meet expected needs of new employees to help them adjust to work at MSFC. This plan attempts to schedule various experiences that will enable the new employee to understand not only the particular work situation in which the employee is assigned, but to show the overall mission and operations of the Marshall Space Flight Center. The study plan which follows here is devised to allow for maximum opportunities, but it may be pared down or condensed if so desired, as the plan moves up the ladder in the approval process.

STUDY PLAN: ORIENTATION OF NEW EMPLOYEES

PROBLEM:

Since 1968, when MSFC's first reduction in force took place, NASA-MSFC has been in a reduced hiring mode. As a result, new employee orientation during this time has been based primarily upon satisfying general information needs. Much of a new person's orientation is conducted informally at the work site by the employee's immediate supervisor. The Personnel Office, on the other hand, gives the new employee basic procedural information concerning OPM regulatory requirements and benefits. This is usually done on a one-to-one basis and later in a short general meeting for those brought on-board that week.

In the last year, NASA-MSFC hiring of new employees has increased. Yet, no plan has been developed to acquaint these new employees with the overall mission and purposes of NASA or with a proper perspective in seeing the relationship of one's own work to the whole. This has generated a need for a more comprehensive, in-depth orientation of new employees.

OBJECTIVES:

The objectives of a plan for the orientation of new employees are:

1. To inform each employee about the basic requirements of adjusting to the new organization. This includes acquiring a knowledge of benefits, hospitalization, security, NASA-MSFC regulations and various other procedural and administrative matters.
2. To provide a method whereby the employee is thoroughly oriented to the work situation. Presently, the employee is acquainted with the work situation in an informal manner.
3. To provide a general understanding of the overall NASA mission and to show how the various organizations work together at MSFC to achieve national goals. Not only will the employee begin to sense the importance of NASA-MSFC as a whole, but also will learn how the new employee's particular work fits into a wider framework.

This study deals with these objectives and outlines the structure and content of a proposed orientation plan that will satisfy the needs

of the new employees and be beneficial to NASA.

PROPOSAL:

The "Orientation Plan for New Employees" will consist of three phases as follows:

1. Administrative Orientation -- procedures, responsibilities and rights, guidelines and benefits.
2. Work Site Orientation -- the particular tasks of the employee at the immediate place of work.
3. NASA-MSFC Orientation -- the overall perspective and acquaintanceship of the mission of MSFC as a whole.

PHASE ONE: ADMINISTRATIVE ORIENTATION

1. Personnel Staffing Specialist Meeting

The very first acquaintanceship of the new employee upon being hired will be a scheduled visit with an assigned Personnel Staffing Specialist (PSS) on an informal, one-to-one, personal basis. The most immediate concerns of the new employee, such as housing, locating main facilities of the Center, work hours, health plans, and pay procedures, will be discussed. The main attempt here is to give the new employee a feeling that he or she is welcome and has a place to find information immediately. A designated number of new employees will be assigned to each of the Personnel Staffing Specialists. Even new employees who have worked with other federal agencies would be expected to participate in this orientation. The PSS will "follow through" with each of the new employees over the course of the entire three phase orientation period. From time to time the PSS will contact the newcomer to inquire about the person's welfare at MSFC. The PSS should give the newcomer a contact phone number on the first visit. Also, the new employee will be given a packet, which includes necessary information about MSFC and Huntsville, employee unions, equal opportunity, and regulations, to keep and study.

2. Personnel Administrative General Meeting

At a designated time, preferably at the beginning of every two week pay-period in order to keep in phase with payroll records, the new employees hired up to that date will meet as a group for a general

orientation conducted, as presently done, by the Personnel Office. The necessary items scheduled for discussion will follow the "New Employee Orientation Checklist," MSFC Form 3688 (April, 1968), which is signed upon completion by both the employee and Staffing Officer and kept in the employee's 201 File. Discussion of any particular points will be encouraged and details will be provided as needed by the new employees.

PHASE TWO: WORK SITE ORIENTATION

It is of utmost importance that the new employee learn to feel at home in his/her own work environment as soon as possible. The adjustment to a new situation will produce some frustrations, but it need not be stressful if proper considerations are given. The new employee will be under the authority of the immediate supervisor. This person should pay great attention to the psychological welfare of the newcomer. Introduction to all fellow employees, an opportunity to see others at work and to see who is responsible to whom, a review of work procedures, and an acquaintance with the location of key facilities and equipment are some of the concerns necessary to the well being of the new employee. This acquaintanceship period takes time, cannot be rushed, and should include some free time to digest the new information. In essence, one is learning some of the informal ways of the organization.

During this time the Personnel Staffing Specialist will be in contact to review progress and handle problems. This is in line with regular personnel procedures which require a ninety day follow-up. Sometimes, several other orientation activities (of Phase Three, below) will be provided during the time the new employee is on the job in this early acquaintanceship period. Such activities might include the bus tour of MSFC or selected lectures and/or interviews by NASA specialists on particular topics. No additional paperwork is required by the supervisor other than the checklist now used routinely by the supervisor in reviewing the completion of special tasks in this part of the orientation. The intent of work site orientation is to keep procedures as natural and informal as possible.

PHASE THREE: NASA-MSFC ORIENTATION

1. The major purpose of this phase is to assure that new employees are given opportunities to see their own particular work in relation to the whole of MSFC and of the entire NASA agency. It is important to the overall mission of NASA that integration and cooperation of efforts be understood by all employees not only better to achieve the goals of NASA, but to enhance esprit de corps. Also, an

understanding of the interrelationships of NASA's many projects often leads to a better feeling about one's own work and aids an individual in one's own personal and professional advancement.

2. Since the new employee will now be at work on the job site, this phase of the orientation must be scheduled so as to cause minimum disruption to the person's work schedule. It is proposed that this general orientation be staggered over a number of weeks to allow the new employee sufficient time to digest the new information and to prevent overburdening the offices of those who present materials by taking time out of their regular workload.
3. Administration of all aspects of Phase Three, including scheduling and monitoring, will be under the management of the Personnel Office.
4. All new employees should be involved together as a group in a one day common orientation. In the morning a bus tour should take the newly hired to various locations of MSFC. They should visit the main projects, test stands, certain laboratory facilities, museum mockups, and special projects, such as the Neutral Buoyancy Facility.

In the afternoon a general meeting should be arranged in the Tenth Floor Conference Room of Building 4200. At this time various selected Center management officials will introduce themselves and conduct a presentation on various important general MSFC topics. Such topics might include a general discussion of the Space Shuttle or of the importance of coordinating payloads.

A representative of the Office of the Director or one of the Executive Staff Offices should give an introductory address called the Center Overview, followed by other senior management speakers, who can lend an aura of importance to the new hiree's appointment into the MSFC workforce. This Conference Room meeting should include as many new hirees as possible at the same time.

5. Up to this time the orientation is the same for all types and categories of employees: scientists and engineers, administrative and professional, technicians, clerical and operational. At this point the orientation is divided into two modules. Each module applies to different categories of new personnel. The difference in orientation lies in the degree of depth and scope. The two modules are:

- 1) Scientists and Engineers, Business and Professional, and

- Technicians (NCC 200/700, 600, 300)
2) Clerical (NCC 500).

6. New employees of both categories (modules) should visit certain basic projects, labs, or offices in common and as a large group. These include:
- 1) selected science and selected engineering laboratories;
 - 2) key projects, such as Shuttle propulsion.
7. The module plan could be modified or compressed if the plan involves too much time of the employees and of the supervisors by cutting down on the staggered sessions and by having the new employees meet together in larger groups.

Module One: Scientists, Engineering, Technicians,
Business and Professionals

1. On a bi-weekly basis a certain number of new hirees will meet as a group to interface with each other, to share experiences, and occasionally to listen to guest speakers about special projects. The purpose of this meeting is to provide for some identifiable group loyalty and cohesion to disparate members of the MSFC work force in order to build up a sense of common solidarity across NASA as a whole, to build a sense of community and shared associations, and to provide for cross-fertilization of ideas. (Precedence for this has already been established by the NASA/ASEE Summer Faculty weekly "fellowship" meetings).
2. An opportunity for new employees to meet key personnel can be developed by inviting such people to these bi-weekly meetings to converse with the group, or to schedule specially arranged times when small numbers of hirees can visit in selected offices.
3. Rather fundamental kinds of programs, projects, and interfaces ought to be included in the itinerary. These might include:
 - a. Advanced Systems, planning layout
Examples: (1) Description of all future projects;
(2) Coal Gasification;
(3) Nuclear Waste Disposal;
(4) Solar Energy Use.
 - b. Space Shuttle operations
Examples: (1) Configuration Change Board of Solid Rocket

Boosters in Building 4610;
(2) Telecon Level II PRCB in Building 4202,
Room 411.

c. Payload and Spacelab Projects

(1) Overview.

d. Science and Engineering Laboratories

Examples: (1) Metallurgical analysis and use of electron
microscopes;

(2) Cell separation in zero gravity

e. MSFC Administrative Support

Examples: (1) Personnel Training;

(2) Procurement;

(3) Plans and Analysis Office;

(4) Overall administration.

Module Two: Clerical

1. Since the new employees of this group will not have had extensive, formal scientific and engineering education, the presentations should be given in less complicated technical detail. However, the general nature of the various projects and tasks indicated in Module One should be of interest to this group. Thus, a schedule should be developed in the Personnel Office for this module following the same pattern.
2. Scheduling for both modules will be done in accordance with new hiring activity and with the concern to distribute presentations among many different people and offices.
3. A special session for Module Two would be a general, overall tour and explanation of a selected number of key installations. This would be different from the usual NASA tour. This would encompass such office functions as procurement, personnel training, computer services, and the public affairs office. Selected scientific laboratories would be included on the itinerary. This tour would be planned to encompass an entire morning.

ORIENTATION PLAN RECOMMENDATIONS:

1. That a comprehensive, in-depth new employee orientation be implemented at MSFC.
2. That the administration and coordination of the Orientation of New Employees be the responsibility of the Personnel Office.
3. That an MMI be published identifying Center policy and assigning duties and responsibilities.

CONCLUSIONS AND RECOMMENDATIONS

The Personnel Office of Marshall Space Flight Center assigned three management tasks to this researcher. All three were completed and the results were presented to the Personnel Director. A meeting of selected supervisors was called so that the researcher could orally present a status report on these topics. They expressed satisfaction with the projects and the work done. Final suggestions were made, although constant surveillance and assistance by these same people had been given continuously during the entire time of investigation and writing.

This report has described the nature of the three tasks and the substance of their results. Since each of the tasks is a topic that is ongoing in nature, the final products will not be ready until all reviewers and authorities give approval. Undoubtedly some changes and revisions will be made. Nevertheless, the basic structure, the main outline and format, and the conceptual plans have been determined at this point.

There was no shortcut in the undertaking of these tasks. First, a thorough orientation and grounding into the nature of the scientific and engineering projects which make up the mission of NASA-MSFC and the nature of the administration and organization of MSFC had to be obtained by this researcher. Then, many different viewpoints and opinions had to be gathered and discussed about the specific nature of the tasks. Materials relevant to the tasks were read and analyzed, but these usually consisted of memos, letters, and similar works previously done. Since most of these materials are on file in various offices at MSFC, they do not form a descriptive bibliography or a list of references, and, consequently, are not stated in this report. Finally, each draft of each task had to be reviewed by an authority on the subject before the next revision could be written. Most often the suggestions of several people were incorporated into the next stage of the writings.

It is recommended that these three projects be continued immediately until their final approval is given by the Director of the Center. A Marshall Management Instruction (MMI) on the Student Volunteer Service Program should be completed in final form following the model presented here. The Recruitment Brochure should follow the general theme as worked upon to this point. Pictures should be selected and minimal text written. A comprehensive Orientation Plan for New Employees should be implemented at MSFC and an MMI should be written to identify Center policy and guidelines. All three of these tasks should be administered by the Personnel Office of MSFC.

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NASA/ASEE SUMMER FACULTY RESEARCH FELLOWSHIP PROGRAM

MARSHALL SPACE FLIGHT CENTER
THE UNIVERSITY OF ALABAMA

THE EFFECT OF TIME DELAY IN THE 6 DOF
MOTION SYSTEM AND ITS COMPENSATION

Prepared By:	Richard Brown, Ph.D.
Academic Rank:	Associate Professor
University and Department	University of Alabama University, AL Department of Mathematics
NASA/MSFC:	Data Systems Laboratory Engineering Computers Division Simulation Systems Branch
MSFC Counterpart:	Frank L. Vinz
Date:	August 1, 1980
Contract No:	NGT-01-002-099 (University of Alabama)

ANALYSIS AND IMPROVEMENT OF CONTACT DYNAMICS SIMULATION
USING THE SIX DEGREE OF FREEDOM MOTION SYSTEM

BY

Richard C. Brown, Ph.D.
Associate Professor of Mathematics
University of Alabama, Tuscaloosa, Alabama

ABSTRACT

The 6DOF motion system has been employed for closed loop simulation of the contact dynamics involved in the docking of two satellites (for example TRS and Skylab). Under most initial conditions however the system gives inaccurate results. Typically in the simulation if docking fails to occur the rebound velocity of the TRS exceeds its approach velocity, thus generating an artificial and unacceptable gain of momentum in the system.

It has long been well known that the presence of a time lag can generate instability in closed loop systems. To see if the time lag (about 200 ms.) in the 6 DOF system can satisfactorily account for its errors, a simple computer mode of the system incorporating this lag and other relevant parameters has been implemented. The model in fact does predict all the observed rebound errors with reasonable precision.

Also, since it is probable that the time delay can only be reduced but not eliminated, various compensation schemes have been devised. Two give adequate correction for delays under 100 ms for a non-preloaded probe. More powerful schemes may be adequate for the present 200 ms. environment and the presence of a preload.

Our study relies heavily on the mathematical theory of second order differential equations with retarded arguments and standard tools of numerical analysis.

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NOMENCLATURE

- τ - Time lag of the 6 DOF system
- Δt - Sample frame time of the AD converter between the sensor and the EAI 8900
- $F(t)$ - Sensor force signal
- ΔF - Change in force signal in time step
- η - Probe compression
- $\bar{\eta}$ - Maximum probe compression
- K - Hooke's constant of probe
- M - Mass of probe
- e - Base of the natural logarithms
- v_0 - Initial velocity of probe
- v_t - Observed rebound velocity of probe
- $v(t)$ - Velocity of probe
- $a(t)$ - Acceleration of probe
- IVP - Initial value problem
- ω_0^2 - K/M
- F_p - Preload of probe
- X - $\eta + F_p/K$
- x_τ - $x(t-\tau)$
- * - Convolution , $f * g := \int_0^+ f(t-s) g(s) ds = \int_0^+ f(s) g(t-s) ds$

- F_t - Force threshold
- $\lambda(t) [a,b]$ - Characteristic function based on the interval $[a,b]$
- h - Step in a digital simulation

- M - Step parameter ; $T = h.M$
- $O(h^k)$ - Error proportional to h^k
- v_n - $v(nh)$
- a_n - $a(nh)$

- v_c - Compensated version of v
- x_c - Compensated version of x

I INTRODUCTION

The LINK Six-Degree-of-Freedom (6 DOF) Motion System has been employed by the Marshall Space Flight Center for the physical simulation of the contact forces and dynamics in the docking of two satellites - for example the Teleoperator Retrieval System (TRS) and Skylab. A similar device was developed at the Johnson Space Center in Houston for contact dynamics simulation prior to the Apollo-Soyuz mission of 1975.

In this section we give an informal description of the system and its performance. Some illustrations, photographs, and values of some key parameters may be found in the Appendix. See in particular the block diagram, Fig.10.

A docking probe assembly is placed on the platform of the LINK 6 DOF Motion System. The platform is free to move with any general motion within its limits of travel. Above the probe is a docking collar or other target which is fixed. The software of the EAI 8900 hybrid computer is capable of simulating a set of initial conditions which might include velocities, accelerations, and rotations preliminary to docking.

Since, as mentioned above, the target is stationary, this information is calculated relative to a coordinate system fixed in the target. The initial conditions are translated by the CAI digital computer into commands to change lengths of the six legs supporting the 6 DOF platform. These commands are executed by the hydraulic actuators in the legs causing the 6 DOF system and the probe to perform motions appropriate to the initial conditions.

When the probe is in contact with the target, sensors, also placed on the table, record the forces and torques encountered by the probe. In the open loop mode this information is recorded on strip charts for later use. If desired software can be developed so that the probe simulates dynamic interaction with the target. This option has so far been rejected since the resulting motion would not be generated by the forces actually encountered in the contact phase. A more useful choice is closed loop simulation where the forces are used in a feedback loop through the EAI 8900 and the CAI computers to control the motion of the probe itself. Basically two integrations of the hybrid subsystem give velocity and position functions of the TRS relative to the target. As in the case of the initial conditions the position function is converted into leg length commands. The result is that the probe is supposed to move as if it was freely responding to its own contact dynamics. This mode when perfected should be a valuable tool in the testing and design of docking hardware.

In preliminary testing of the system, the closed loop simulation has been radically simplified so that probe-target interaction is "head on;" only (vertical) translational motion is accepted. Also no thrusts firings of the chase vehicle are allowed. Table 1 presents results of some of these tests.

Obviously apart from the unacceptable level of noise, there is an artificial and unwarranted gain of momentum in the system. The analysis and attempted resolution of this problem is the central concern of the present study.

We close this section with a brief survey of the rest of the paper. In Section II a simple mathematical model based on the time delay inherent in the system is developed. It is also shown how the model can be modified so that it is "noisy." Section III develops several methods to compensate for the delay. The last two sections contain respectively data and graphs illustrating our results and recommendations for further action.

II THE EFFECT OF TIME LAG ON THE 6 DOF CLOSED LOOP SIMULATION

It has long been known that the presence of time delay can affect the stability of a closed loop feedback system - see for example.

Minorsky [10], [11] or Callendar, Hartree and Porter [4]. It has been verified that a time delay defined as the time it takes for the legs to complete a response to a signal from the force sensor, exists in the 6 DOF system.

This delay (let us call it " τ ") is approximately 200 ms; the various contributions to it by hardware subsystems are shown in the block diagram Fig.10.

A simple physical argument shows how τ might explain the high rebound velocities of table 1. Suppose at time t the probe has reached maximum compression $\bar{\eta}$ and should turn around. The command to do this is executed $t+\tau$ later; in the meantime, the probe has continued to move and overshoot $\bar{\eta}$. Therefore during a time period of length τ additional force has been communicated to the software resulting in a gain in energy. If the probe did not detach, τ sec. after reaching its rest position the overshoot would be repeated and one might expect a sequence of oscillations of increasing amplitude.

2.1 A Mathematical Model - Let us now try to make the above argument mathematically precise. Suppose in the interval $[t, t+\Delta t]$ a change in the force signal ΔF is sampled. When the EAI 8900 computes a change of compression $\Delta \eta$ corresponding to ΔF , it does this by digitally solving the equation (2.1) and computing the solution at $t + \Delta t$

$$(2.1) \quad M \ddot{\eta} = F(t)$$

Now $\eta(t + \Delta t)$ should equal $\eta(t) + \Delta \eta$. However $\Delta \eta$ is not executed until the actuators have finished their response at $t + \tau$. Thus $\eta(t + \tau)$ corresponds to $F(t)$ - $\eta(t)$ equivalently corresponds to $F(t - \tau)$. It follows - (τ includes the sample interval or "frame time" Δt) - that the system behave as if it were solving the equation

$$(2.2) \quad M \ddot{\eta} = F(t - \tau)$$

Thus we could model the effect of delay by using (2.2) instead of (2.1) in a computer simulation.

In case the probe is an ideal spring $F = -k\eta$ and (2.2) becomes

$$(2.3) \quad M \ddot{\eta} + K \eta(t-\tau) = 0$$

(2.2) and (2.3) are examples of retarded differential equations. Because of their importance to control theory applications, such equations have been intensively studied [3], [6], [7] in recent years. The resulting theory has many similarities with the classical theory of differential equations but also important differences. In particular it can be shown that if

$$\tau < 2/\omega_0$$

then "practically all" solutions oscillate with exponentially increasing amplitude (R. D. Driver, private communication, also cf. Driver [6]).

Also unlike differential equations, a correctly formulated initial value problem (IVP) for (2.2) - (2.3) requires initial conditions on an initial interval rather than just at a point. For example, consider the first τ seconds after contact: The probe continues to compress oblivious of contact since the legs have not yet moved in response to $F(t)$. Thus

$$\eta(t) = v_0 t$$

(2.4)

$$\dot{\eta}(t) = v_0, \quad 0 \leq t \leq \tau.$$

These replace the point conditions of a differential equation IVP.

2.2 A Few Refinements - Essentially equations (2.1) and (2.2) will constitute the mathematical model of the 6 DOF system in our report. Before solving it we discuss some possible improvements (only the most significant of which have been implemented).

2.2.1 Probe Nonlinearity - As is shown in Fig. 2, the probe is not an ideal spring. It is both nonlinear and preloaded. The nonlinearity can be modeled by an equation of the form

$$M \ddot{\eta} + K(\eta(t-\tau)) = 0$$

For the graph of $k(u)$ see Fig. . To approximate $K(\eta)$ one might choose a piecewise linear function, a least squares parabolic fit, or a spline. None of these choices is difficult to implement on the computer. However, since $K(\eta)$ is nearly

constant in the neighborhood of 50-60 lbs/ft, its variation is probably not a source of major error, we will omit this feature of the spring from our model.

Of much greater significance is the existence of a preload (F_p) of approximately 69 lbs. By energy considerations a preloaded spring is equivalent to a compressed longer spring having the same Hook's constant; the fictitious compression is F_p/k . This suggests that we define $x(t) = \eta(t) + F_p/k$ and solve

$$\begin{aligned}
 \ddot{x} + \omega_0^2 x - \nu_0 &= 0 \\
 x(t) &= x_0 + \nu_0 t \\
 \dot{x}(t) &= \nu_0 \quad 0 \leq t \leq \tau
 \end{aligned}
 \tag{2.5}$$

$$\begin{aligned}
 x_0 &:= F_p/k \\
 \omega_0^2 &:= k/M \\
 x_\tau &:= x(t-\tau)
 \end{aligned}$$

It will turn out that the preload magnifies the effect of a pure delay.

In this report we will also consider an ideal probe with no preload and $K=1200$ lbs/ft., since this value has been used in a recent analog simulation [1] of the effect of delays on spring motion in the 6-DOF system.

2.2.2 Filters - As the block diagram Fig. 10 shows there are several filters in the system. They contribute to the total delay and probably somewhat distort the signal. In [1] it has been shown that the addition of a low pass filter with transfer function $c/(s+c)$ gives an additional destabilizing effect.

In a digital simulation the presence of such a filter changes (2.3) to the equation

$$M \ddot{\eta} + K e(t) * \eta(t-\tau) = 0$$

(2.6)

$$e(t) := 1 - e^{-ct}$$

In general the presence of several filters with constants c_1, \dots, c_n gives the equation

$$(2.5) \quad M \ddot{\eta} + K \lambda_1 * \lambda_2 * \dots * \lambda_n * \eta(t-\tau)$$

where

$$\lambda_i := 1 - e^{-c_i t}$$

The presence of *convolutions* can be handled by *nested* integration loops in a digital simulation. Since the parameters of the filters actually in the system are unknown we ignore their effect except insofar as they contribute to the delay. Our results compared with tests suggest that this is not a crucial omission.

2.2.3 Force Threshold - In addition to the filters a force threshold (F_t) is used to reduce noise. This means that $F(t)$ is accepted if and only if its absolute value exceeds F_t i.e.,

$$F(t) = \begin{cases} F(t) & \text{if } |F(t)| \geq F_t \\ 0 & \text{otherwise.} \end{cases}$$

Thus the IVP (2.5) becomes

$$(2.6) \quad \begin{aligned} M \ddot{x} + K \lambda(x) [F_t/K, \tau] x &= 0 \\ x(t) &= v_0 t + F_0/K \\ \dot{x}(t) &= v_0, \quad 0 \leq t \leq \tau \end{aligned}$$

Fortunately (2.6) is equivalent to a simpler IVP where F_t plays a role like the preload. The probe must travel F_t/K before force due to compression is "accepted." The elapsed time is $\bar{t} = F_t/Kv_0$: If we regard \bar{t} as time zero, (2.6) becomes

$$\ddot{x} + \omega_0^2 x = 0$$

$$(2.7) \quad x(t) = x_0 + v_0 t$$

$$\dot{x}(t) = v_0$$

$$x_0 = (F_p + F_s) / K$$

$$v(t) = x(t) - F_p / K$$

Note that the system should be very sensitive to F_s . If F_s is large and v_0 is small the only force affecting the probe might be that of the bottoming spring. For example if $K = 60$, $F_s = 50$, $x_0 > 14.4$ in which is greater than the maximum compression of the probe. In the absence of delay, even threshold of 30 lbs. would produce a severe distortion. When delay is present its destabilizing effects are greatly magnified. In the case of a spring with $K = 1200$ and no preload the effects thought not so severe are still noticeable.

The presence of a force threshold also causes some ambiguity in time measurements during the simulation. Suppose t_r is the time during rebound when $Kx = F_s$. Since $F(t)$ is measured directly on the strip chart the simulation period is $[-F_s / K v_0, t_r + F_s / K v_0]$. The period in which the forces are producing a response in the legs is $[\tau, t_r + \tau]$. Currently the exit velocity is measured at $t_r + \tau$, time $t_r + \tau + F_s / K v_0$ from contact. To avoid confusion one needs to keep these various times clearly in mind especially when F_s is high and/or K, v_0 is low.

2.2.4 Variable Delay - The IVP (2.6) is a continuous realization of a discretely sampled system. The AD and DA converters (1), (2) (see Fig. 10) sample the signal at discrete intervals Δt - approximately 31-35 ms. If all the converters were synchronous little would be lost since we in fact solve the IVP in steps on the computer. However, the converters (3), (4) sample at intervals of 25 ms. This means that periodically (3) samples the same signal twice, producing an additional 25 ms delay. Since they do not start in synchrony (they start where they stopped) the effect varies from run to run. We have put a "noisy" delay into the model. The results suggest that the asynchronization explains some of the noise in the data.

2.2.5 Noise from the table - Since the force sensors are in the table, they may feel some of the inertial forces in the table itself. Again forces due to this source are executed seconds later. This suggest that we consider the neutral to second order equation.

$$(2.8) \quad \ddot{x} - \epsilon \frac{M_1}{M} \ddot{x}_T + \omega_0^2 x = 0$$

with the same initial conditions of (2.7) where ϵ is a noise parameter given by a random number generator and M_1 is the mass of the table + probe = 323.5. Preliminary tests (see Appendix I) show that this complication is indeed a potent source of noise.

2.3 Solving the Model

2.3.1 Formulas in closed form - Suppose we linearise (2.7) by taking the first two terms of the Taylor expansion, We obtain

$$\ddot{x} + \omega_0^2 x - \omega_0^2 \tau \dot{x} = 0$$

$$(2.9) \quad x(0) = X_0$$

$$\dot{x}(0) = V_0$$

If $\omega_0^2 \tau \ll 1$ and $X_0 = 0$ (no preload, no threshold) an approximate solution to (2.9) is

$$x(t) = \frac{F_t \tau}{2M} \sin \omega_0 t$$

Hence

$$(2.10) \quad v_F \approx v_0 e^{\frac{1}{2} \omega_0 \tau}$$

Neither of these formulas are accurate for much beyond a few seconds. However, (2.9) gives an error of under 3% for $\tau \leq .2$. If there is a force threshold the formula

$$(2.11) \quad v_F = v_0 e^{\frac{1}{2} \omega_0 \tau} \left(v_0 + F_t \tau / 2M \right)$$

yields an error < 15% provided $F_t < 120$ lbs.

An exact closed form solution to (2.7) or 2.8) can be given by an inductive procedure known as the "Method of Steps." Suppose v and x are known on the interval $[(k-1)\tau, k\tau]$. Then setting

$$t = k\tau + s$$

we have for $k\tau \leq t \leq (k+1)\tau$

$$(2.12) \quad v(t) = v(k\tau) - \omega_0^2 \int_{k\tau}^t x(s) ds$$

$$x(t) = x(k\tau) + \int_{k\tau}^t v(u) du.$$

Since the initial functions are simple the solution may be calculated in closed form, step by step. Thus for $\tau < t \leq 2\tau$

$$v(t) = v_0 + \frac{1}{2} \omega_0^2 v_0 (t-\tau)^2$$

$$x(t) = x_0 + v_0 \tau + \frac{1}{3!} \omega_0^2 v_0 (t-\tau)^3$$

etc.

2.3.2 Numerical Integration- The Method of Steps gives a solution in the form of an infinite series. Since the result tells us nothing about the properties of the solution, nothing is lost if we numerically integrate (2.7). The following algorithm which uses the corrected trapezoid rule ([5] p. 74) has $O(h^4)$ discretization error and has been implemented on the PDP Sigma 5 computer and the writers T159 .

Input $k, v_0, x_0, F_t, F_p, M, k, \tau, N$

$h := \tau/N$

$x_0 \leftarrow x_0 + (F_t + F_p)/k$

For $i = 1$ to $N+1$

$x_i = x_{i-1} + v_0 h$

$v_i = v_0$

Continue

For $i = k+2$ until satisfied:

$$V_{i+1} = V_i - \omega_0^2 \left(\frac{H^2}{2} (X_{i+1-n} + X_{i-n}) + \frac{H^2}{12} (V_{i-n} - V_{i+1-n}) \right)$$

Write V_{i+1}

$$X_{i+1} = X_i + \frac{H^2}{2} (V_i + V_{i+1}) - \frac{\omega_0^2 H^2}{12} (X_{i+1-n} - X_{i-n})$$

$$\eta_{i+1} = X_{i+1} - F_0/k$$

Write η_{i+1}

Continue

2.4 Results - Figures (2)-(7) give the results of several runs with different initial conditions and parameters. Note that the results are consistent with tests of the 6 DOF system and also agree almost perfectly with the results of the analog simulation [1].

III COMPENSATING FOR DELAY

It is hoped that most of the delay can be eliminated by suitable modifications in the hardware. Nevertheless even delays as low as 50 ms can cause noticeable instability. Therefore in this section we develop several algorithms that in some cases can produce near normal performance in spite of delay. Test results for some of the methods described below are given in Section IV.

3.1 Taylor Approximations

Since

$$x_T = x - \tau \dot{x} + \frac{\tau^2}{2} \ddot{x} + \dots + (-1)^n \frac{\tau^n}{n!} x^{(n)}(t), \quad t - \tau < t < t$$

we can truncate the series and solve for an approximation x_c to x - i.e.,

$$x_c \approx x_T + \tau \dot{x} - \frac{\tau^2}{2} \ddot{x} + \dots + (-1)^{n-1} \frac{\tau^{n-1}}{(n-1)!} x^{(n-1)}$$

with error $\mathcal{O}(\tau^n, x^n)$. We then solve.

$$\ddot{x} + \omega_0^2 x_c = 0$$

$$x(0) = x_0$$

(3.1)

$$\eta = x - F_0/K$$

instead of (2.1).

3.1.1 Runge Kutta - In theory (3.1) can be solved for arbitrarily high n by the Runge Kutta algorithm without knowledge of high derivatives. For example if $n = 4$ (3.1) can be converted to the system:

$$(3.2) \quad \begin{pmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \end{pmatrix} = \begin{pmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ F_T & C_1 & C_2 \end{pmatrix} \begin{pmatrix} x_1 \\ x_2 \\ x_3 \end{pmatrix}$$

$$\begin{pmatrix} x_1 \\ x_2 \\ x_3 \end{pmatrix} (0) = \begin{pmatrix} x_0 \\ v_0 \\ 0 \end{pmatrix}$$

where

$$E_T := \text{backwards shift}$$

$$C_1 := -3!/T^2$$

$$C_2 := \frac{3}{T} \left(\frac{2}{T^2} \omega_0^2 + 1 \right)$$

$$X_i := X^{(i-1)}, \quad i = 1, 2, 3$$

Writing (3.2) in the abbreviated form

$$X' = AX$$

$$X(0) = X_0$$

and applying the Runge Kutta algorithm.

we obtain

$$X_{i+1} = X_i + H \left(\frac{K_{1i}}{6} + \frac{K_{2i}}{3} + \frac{K_{3i}}{3} + \frac{K_{4i}}{6} \right)$$

where

$$K_{1i} = X_i + H/2 A X_i$$

$$K_{2i} = X_i + H/2 A K_{1i}$$

$$K_{3i} = X_i + H/2 A K_{2i}$$

$$K_{4i} = X_i + H A K_{3i}$$

The difference between the above version of Runge Kutta and the conventional one for differential equations is that (because of the shift operation) K_{1i} requires Y_{i-1} , K_{2i} requires K_{1i} etc. However, the effect this deviation has on the stability and accuracy of the algorithm is unknown and will require additional investigation. Higher order versions of Runge Kutta are also possible. They can be used alone or combined with equivalent Adams predictor corrector methods. If they work quite high order Taylor approximations can be easily generated.

3.1.2 The Algorithm "FIX*T4" - The following direct approach to (3.1) has been implemented and found to work well. We write.

$$X_T \approx X - T\dot{X} + \frac{T^2}{2}\ddot{X} - \frac{T^3}{6}\ddot{\dot{X}} + \frac{T^4}{24}X^{(4)}$$

$$\dot{X}_T \approx \dot{X} - T\ddot{X} + \frac{T^2}{2}\ddot{\dot{X}} - \frac{T^3}{6}X^{(4)}$$

$$\ddot{X}_T \approx \ddot{X} - T\ddot{\dot{X}} + \frac{T^2}{2}X^{(4)}$$

$$\ddot{x}_\tau \approx \ddot{x} - \tau x^{(iv)}$$

$$x_\tau^{iv} \approx x^{iv}$$

By back substitution the present values of the derivatives can be eliminated. We finally obtain

$$x_c = x_\tau + \tau \dot{x}_\tau + \frac{\tau^2}{2} \ddot{x}_\tau + \frac{\tau^3}{6} \ddot{\dot{x}}_\tau + \frac{\tau^4}{24} x_\tau^{iv} + O(\tau^5, x^{(5)})$$

We can truncate the series obtaining second, third or fourth order corrections.

The 4th order version, however, is as easy to implement as the others and more accurate

"FIX*T4" For $i = 1$ to m

$$v_i = v_0$$

$$x_i = iH v_0 + x_0$$

$$a_i = 0$$

continue

For $i = m+1$ until satisfied

$$(3.3) \quad v_{i+1} = v_i - \omega_0^2 \left\{ \frac{H}{2} (x_{i-m} + x_{i+1-m}) + \frac{H^2}{12} (v_{i-m} - v_{i+1-m}) \right. \\ \left. + (x_{i+1-m} - x_{i-m}) + \frac{\tau^2}{2} (v_{i+1-m} - v_{i-m}) \right. \\ \left. + \frac{\tau^3}{6} (a_{i+1-m} - a_{i-m}) \right\}$$

$$a_{i+1} = -\omega_0^2 x_{i+1-m}$$

$$a_{i+1}^c = -\omega_0^2 (x_{i+1-m} + v_{i+1-m})$$

$$x_{i+1} = x_i + \frac{H}{2} (v_i + v_{i+1}) + \frac{H^2}{12} (a_i^c - a_{i+1}^c)$$

Continue.

The algorithm is $O(h^4; x^{iv})$.

Remark - The corrected trapezoid rule and the two estimates of $\ddot{x}_i, -a_i, a_i^c$ are necessary to maintain accuracy.

Warning - In a 6 DOF simulation

$$q_{i+1} = q_{i+1}^c = F_{i+1} = F(i+1, \Delta t)$$

(The frame time Δt is the step). Also all delayed terms x_r, \dot{x}_r etc. are the present values of x, \dot{x} etc., generated in the hybrid (recall that the backward shift introduced into the algorithm simulates the loop circuit time in the 6 DOF system) With this in mind, we write the algorithm in a form suitable for implementation on the 6 DOF. Let v_{i+1}, x_{i+1} be the velocity and position produced by the hybrid from F by integration. Let v_{i+1}^c, x_{i+1}^c be the compensated versions. Then the main loop in (3.3) looks like

For $n = m + 1$ until satisfied:

v_{i+1} = Integration of F

x_{i+1} = Integration of v

$$v_{i+1}^c = v_i - \omega_0^2 \left\{ H/2 (x_i, x_{i+1}) + H^2/12 (v_i - v_{i+1}) + T (x_{i+1} - x_i) + T^2/2 (v_{i+1} - v_i) + T^3/6 (F_{i+1} - F_i) \right\}$$

(3.4)

$$x_{i+1}^c = x_i + H/2 (v_i + v_{i+1}^c) + H^2/12 (F_i - F_{i+1})$$

$$v_i \leftarrow v_{i+1}^c \quad x_i \leftarrow x_{i+1}^c$$

(i.e., these values replace v_i, x_i in the next iteration of the loop)

Continue

In the remainder of the paper all algorithms will be written the form (3.3) instead of (3.4) and it will be understood that terms x_r and their digital form x_{i-m} etc. refer to present not past values of variables calculated in the hybrid.

3.2 Polynomial Extrapolation - Porter and Stoneman [12] (also see [13]) have proposed the use of Newton-Gregory interpolating polynomials to compute x_c . Specifically they fit a polynomial to $x_{n+1-m}, x_{n+1-(m+1)}, \dots, x_{n+1-(m+6)}$ and extrapolate to compute $x_c \approx x_{n+1}$. Thus by the Newton-Gregory backwards interpolation formula

$$(3.5) \quad X_{n+1}^c \approx X_{n+1-m} + \tau \nabla X_{n+1-m} + \frac{\tau(H+\tau)}{2! H^2} \nabla^2 X_{n+1-m} \\ + \frac{\tau^3}{3! H^3} \nabla^3 X_{n+1-m} + \tau(\tau+H)(2H+\tau) \dots \mathcal{O}(H^{k+1}, X^{k+1})$$

where

$$\nabla X_{n+1-m} = X_{n+1-m} - X_{n+1-(m+1)} \\ \nabla^2 X_{n+1-m} = \nabla X_{n+1-m} - \nabla X_{n+1-(m+1)} \text{ etc}$$

No higher than a cubic should be taken. The procedure might begin with a 4th order Taylor method to compute enough x values beyond $[0, \tau]$ to fit the cubic. For example, the first cubic could be passed through X_{m+1}, \dots, X_{m+4} , or through X_m, \dots, X_{m+3} etc. The best choice would be found experimentally. Since interpolating polynomials are extremely sensitive to noise, we suggest substituting a least squares polynomial of low degree based on as many previous points as possible. If we write the polynomial in the form.

$$P_v(x) = a_0 + a_1 x + \dots + a_v x^v$$

and set $\bar{a} = (a_0, a_1, \dots, a_v)$, \bar{a} is the solution of the system.

$$(3.6) \quad C \bar{a} = \bar{r}$$

where C is the $(v+1) \times (v+1)$ matrix with general component

$$c_{ij} = \sum_{p=m}^{n+1} t_e^{i+j-2}$$

where

$$\bar{r} = \begin{pmatrix} \sum_{p=m}^{n+1} x_p^0 \\ \sum_{p=m}^{n+1} t_e x_p \\ \sum_{p=m}^{n+1} t_e^2 x_p \\ \vdots \\ \sum_{p=m}^{n+1} t_e^v x_p \end{pmatrix}, \quad t_e := \tau H$$

and p is a preassigned integer.

Remark - matrix system (3.6) becomes poorly conditioned for large v ; most machines will give meaningless results at $v = 7$ or 8. We recommend double precision arithmetic and $v \leq 4$.

Because velocities are available at the same time as the position function, another choice would be to use this information and require that the polynomial pass through $x_{k-m}, v_{k-m} \dots$ etc. For formulas for this type of interpolating polynomial see [5] p62ff. The error is $O(h^{m+1})$ where m is the degree of the polynomial (e.g., 5 if three points are used.)

Remarks

- (1) Although error terms for polynomial extrapolation schemes are similar to Taylor methods, their performance may be superior (particularly for the least squares version) since more information is used and sensitivity to noise is reduced.
- (2) Warning - Neither method (or any other known to the author) will make the system stable for more than a short time. The high derivatives present in the error terms ruin the approximation as .

3.3 Transfer Function Approximation - The Laplace transform of x_T is $e^{-Ts} \mathcal{L}\{x\}$. Therefore, it is tempting to find an approximation $P(s)$ to e^{-Ts} and define

$$\begin{aligned} x_c \approx x &= \mathcal{L}^{-1}(P(s) e^{-Ts} \mathcal{L}\{x\}) \\ &= \mathcal{L}^{-1}(P(s)) * x_T \end{aligned}$$

Thus we solve.

$$(3.7) \quad \ddot{x} = -\omega_0^2 \mathcal{L}^{-1}(P(s)) * x_T$$

The advantage of this method is that if P(s) is properly chosen, (3.7) can be solved on an analog computer much more easily than it can be on a digital machine. This technique provided P(s) is "close" to e^{Ts} can be shown to be as good as polynomial extrapolation (see [13] p32ff).

Some Approximations for e^{Ts}

3.3.1 Chebyshev polynomials lead to approximations which are much more accurate than Taylor polynomial approximations of the same degree.

Thus

$$(3.8) \quad e^{Ts} \approx .994792 + .997396 Ts + .541667 (Ts)^2 + .177083 (Ts)^3$$

$$(3.9) \quad \approx .994571 + .997368 Ts + .542991 (Ts)^2 + .177347 (Ts)^3$$

$$(3.10) \quad \approx .994571 + .997368 Ts + .542991 (Ts)^2 + .177347 (Ts)^3$$

See Atkinson [2] Ch 4

The error of each of these approximations is less than 6×10^{-3} if $|Ts| < 1$

Tests based on (3.10) have shown that it is generally superior to a fourth order Taylor approximation when there is no threshold and no preload. For a description of the resulting algorithm "FIX*C" see p below.

3.2.2 Padé Approximations - It is possible to approximate accurately by rational functions of equal degree in the numerator and denominator. The functions are called Pade approximations. Inverting them offers the advantage that numerical or analog differentiation is avoided. Also Padé approximations are much more accurate than Taylor series approximations of equal degree. For example the 2nd degree Pade

$$(3.11) \quad \frac{1 + \frac{rs}{2} + \frac{(rs)^2}{12}}{1 - \frac{rs}{2} + \frac{(rs)^2}{12}}$$

coincides with the first five terms of the Taylor series of e^{rs} the sixth term is $\frac{(rs)^5}{144}$ which is close to $\frac{(rs)^5}{120}$. Hence this approximation is $O(rs)^5$.

The third degree

$$(3.12) \quad \frac{1 + \frac{rs}{2} + \frac{(rs)^2}{10} + \frac{(rs)^3}{120}}{1 - \frac{rs}{2} + \frac{(rs)^2}{10} - \frac{(rs)^3}{120}} \quad \text{cf [9] p 14654}$$

is even better, it coincides with the first seven terms of the Taylor series. Hence, if our previous discussion is correct these approximations should be similar to cubic and six degree polynomial extrapolation. Padé approximation can be further improved using Chebychev polynomials (See Ralson [17] p 296) Thus

$$(3.13) \quad e^{rs} \approx \frac{2353 + 1176rs + 192(rs)^2}{2353 - 1176rs + 192(rs)^2}$$

The error of the approximation for $|rs| < 1$ is $O(10^{-4})$

Inversion of Padé approximations and appeal to (3.7) result in complicated integro-differential equations. For example the second degree Padé yields

$$y'' + \omega_0^2 y + \frac{12}{T} (e^{\frac{3t}{T}} \sin \frac{3t}{T} + e^{\frac{3t}{T}} \cos \frac{3t}{T}) \cdot y = 0$$

Unfortunately all Padé approximations for e^{rs} have poles in the half plane. We conjecture that because of the short contact time this source of instability will not be of major importance.

3.4 Richardson Extrapolation - We finally mention a technique which can "boost" the degree of approximation when used in conjunction with the previous techniques; that is if the error term of a given method is $O(h^p)$ we will obtain by modifying the method an error term $O(h^{p+1})$

Suppose

$$(3.14) \quad X_c(h) = x + C(h^p) + O(h^{p+1})$$

Having the step gives

$$(3.15) \quad X_c\left(\frac{h}{2}\right) = x + \frac{C}{2^p}(h^p) + O(h^{p+1})$$

(The notation " X_c " indicates the dependence of the compensated position on the step h). Multiplying (3.14) by 2^p and subtracting 3.13 gives

$$2^p X_c\left(\frac{h}{2}\right) - X_c(h) \approx (2^p - 1)x + O(h^{p+1})$$

Therefore

$$x = \frac{2^p X_c\left(\frac{h}{2}\right) - X_c(h)}{2^p - 1} + O(h^{p+1})$$

IV GRAPHS AND TABLES

KEY TO GRAPHS

In figures 2 and 3 the vertical scale is 1 cm = .01 ft. while in figures 4 and 5 it is 1 cm = .001 ft. In fig. 7 the vertical scale is 1 cm = .01 ft/sec.

In all position vs time or velocity vs time graphs the horizontal scale is 1 cm = .1 sec.

"1" signifies theoretical performance with the given parameters.

"2" signifies uncompensated performance.

"3" is FIX*C correction.

The dashed line in Figs. 6,7 is FIX*T4 correction.

"4" in Fig. 2 is performance with a thirty pound force threshold.

Fig. 2 shows the distorting effect of the force threshold (at two levels) with attempted correction.

Fig. 3 represents theoretical, uncompensated and compensated performance of an Apollo type probe with typical parameters.

Fig. 4 contrasts the performance and compensation for a preloaded vs non-preloaded probe.

Fig. 5 is similar to Fig. 3 but the probe has no preload.

6 DOF CONTACT DYNAMICS TEST RESULT
(8/27-28/79)

Test involved vertical motion only and a .75" plywood target simulating the docking collar.

I.	V_0	V_f	8/29/79
	.05	.1927	
	.05	.1907	$F_t = 30$ lbs.
	.05	.1971	
	.05	.1730	
		$\bar{V}_f = .1884$	
II.	V_0	V_f	
	.10	.1965	
	.10	.2122	8/17/79
	.10	.2245	
	.10	.2203	$F_t = 30$ lbs
		$\bar{V}_f = .2134$	
III.	V_0	V_f	8/28/79
	.1	.2216	
	.2	.3826	
	.2	.4101	$F_t = 35$

TABLE 1

THE EFFECT OF FORCE THRESHOLD ON REBOUND PARAMETERS

V = .1 ft/sec. K=1200 lbs/ft M=310.56 Slugs τ = .15 sec

F_t	\bar{h} - model	V_f - formula	V_f - Analog Simulation	V_f - Model
0	.0637	.159	.157	.157
20	.0668	.1666	-	.162
30	.0701	.1704	.165	.170
40	.0746	.1742	-	.171
60	.855	.1818	.182	.182
70	.978	.1850	-	.188
80	.984	.1896	-	.1968
90	.102	.1934	-	.1974
100	.112	.1973	-	.198
120	.127	.2048	.22	.223
240	.22	.2510	.312	.332

III-22

TABLE 2

PROBE DEFLECTION PROFILE

η (ft)	F(lbs)
0	69
.125	74.8
.25	81
.417	89.3
.5	93.75
.625	101
.708	106.2
.792	112.5
.854	118
.88	121
.9	125
.911	130

For $\eta > .912$ there is a bottoming spring with $K = 480,000$ lbs/ft

TABLE 3

Probe compression vs. Load

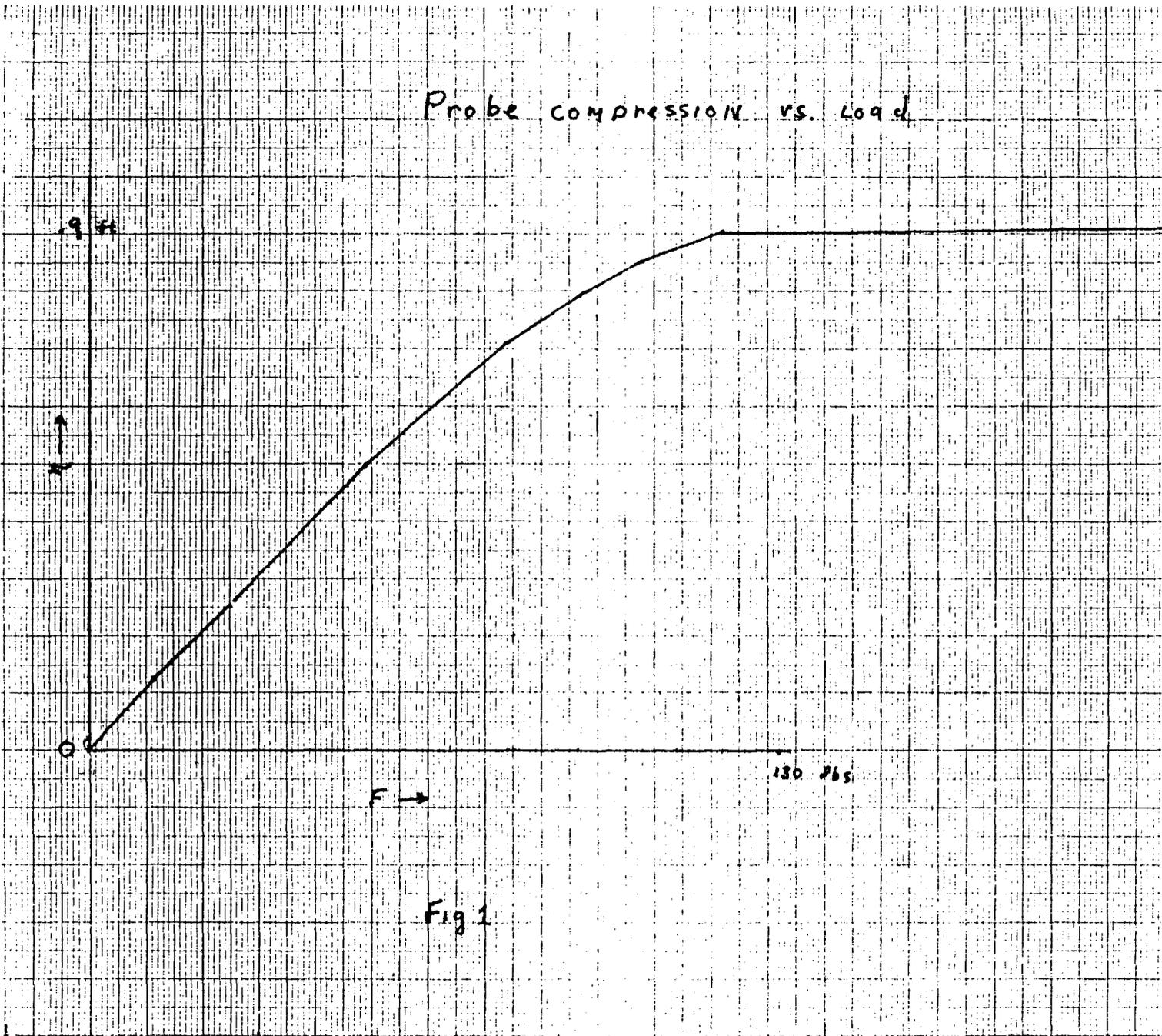


Fig 1

COMPARATIVE CORRECTION BY 4TH ORDER TAYLOR METHOD (FIX*T4) AND
CHEBYCHEV APPROXIMATION (FIX*C)

II F =69 lbs K=60lbs/ft M =311 SLUGS

<u>THEORETICAL</u>			<u>UNCORRECTED</u>			<u>FIX*T4</u>			<u>FIX*C2</u>		
A $\tau = .05$											
V_0	\bar{h}	t	V_0	\bar{h}	t	V_0	\bar{h}	t	V_0	\bar{h}	t
.05	.0055	.45	.0725	.008	.6	.0725	.0083	.6	.0718	.0081	.6
.1	.0223	.9	.125	.027	1.05	.12	.0272	1.025	.135	2.73	1.1
.2	.0869	1.25	.233	.096	1.9	.22	.096	1.85	.23	.0967	1.9
.5	.468	3.6	.536	.491	3.70	.53	.49	3.7	.528	.488	3.7
			1.04	1.44	5.05	1.06	1.45	5.3	1.02	1.43	5.1
III-26											
B $\tau = .1$											
			V_0	\bar{h}	t	V_0	\bar{h}	t	V_0	\bar{h}	t
			.128	.01	1.1	.0949	.0106	.75	.094	.011	1.25
			.170	.0295	1.3	.162	.0225	1.1	.147	.032	1.2
			.269	.106	2.1	.268	.105	2.1	.24	.106	2.0
			.582	.514	3.9	.574	.5	3.9	.55	.506	3.8
			1.13	1.48	3.4	1.06	1.45	5.3	1.04	1.45	5.2
C $\tau = .2$											
			V_0	\bar{h}	t	V_0	\bar{h}	t	V_0	\bar{h}	t
			.129	.0155	1	.128	.0155	1	.129	.015	1
			.184	.0422	1.45	.182	.0419	1.45	.173	.0412	1.4
			.288	.126	.225	.285	.124	2.25	.28	.122	2.25
			.61	.56	4.0	.595	.54	4	.574	.54	3.9
			1.15	1.57	5.3	1.09	1.5	5.35	1.09	1.49	5.35

TABLE 5

SIMULATION RESULTS CORRESPONDING TO TABLE 1

$F_p = 69$ LBS $K = 60$ LBS/FT $M = 311$ SLUGS
 $F_t = 30$ LBS.

I $V_o = .05$ $V_f = .163$

II $V_o = .1$ $V_f = .227$

III $V_o = .2$ $V_f = .33$

IV $F_t = 35$ lbs

$V_o = .2$ $V_f = .343$

TABLE 6

APPROXIMATE FORMULAS FOR A DELAYED SPRING

Assuming $\omega_0 \tau / 2 \ll 1$, the following approximate formulas may be derived for the IVP:

$$\ddot{\eta} + \omega_0^2 \eta_\tau = 0$$

$$\eta(t) = v_0 t \quad 0 \leq t \leq \tau$$

$$\dot{\eta}(t) = v_0$$

$$\eta(t) \approx v_0 / \omega_0 \exp(\omega_0^2 \tau t / 2) \sin \omega_0 t$$

$$\bar{\eta} \approx v_0 / \omega_0 \exp(\pi \tau \omega_0 / 4)$$

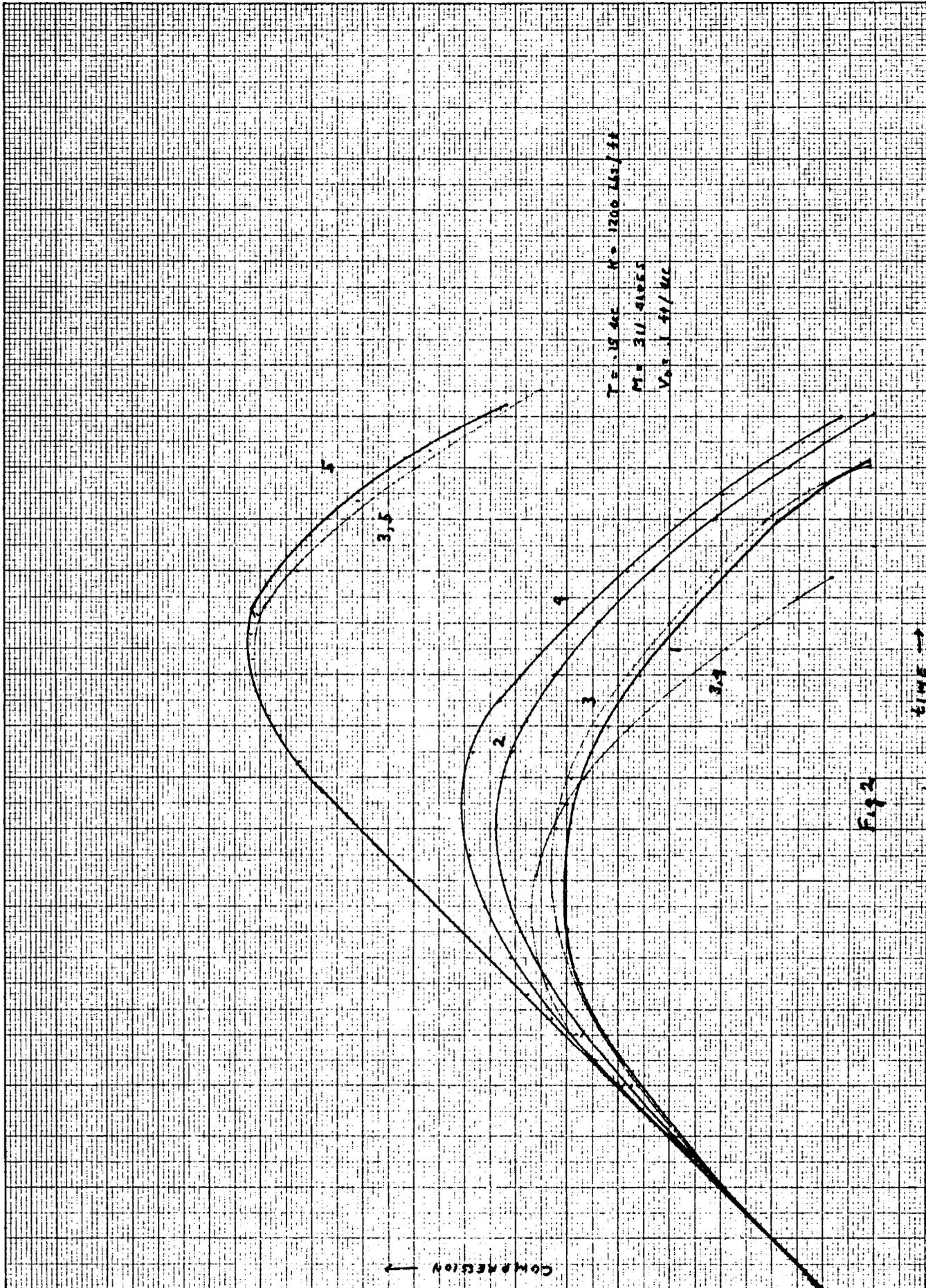
$$v_F \approx v_0 \exp(\pi \tau \omega_0 / 2)$$

These formulas give relative errors of under 3%.

If a force threshold is present then

$$v_F \approx \exp\left(\frac{\pi}{2} \omega_0 \tau\right) \left(v_0 + F_T \tau / 2M \right)$$

TABLE 7



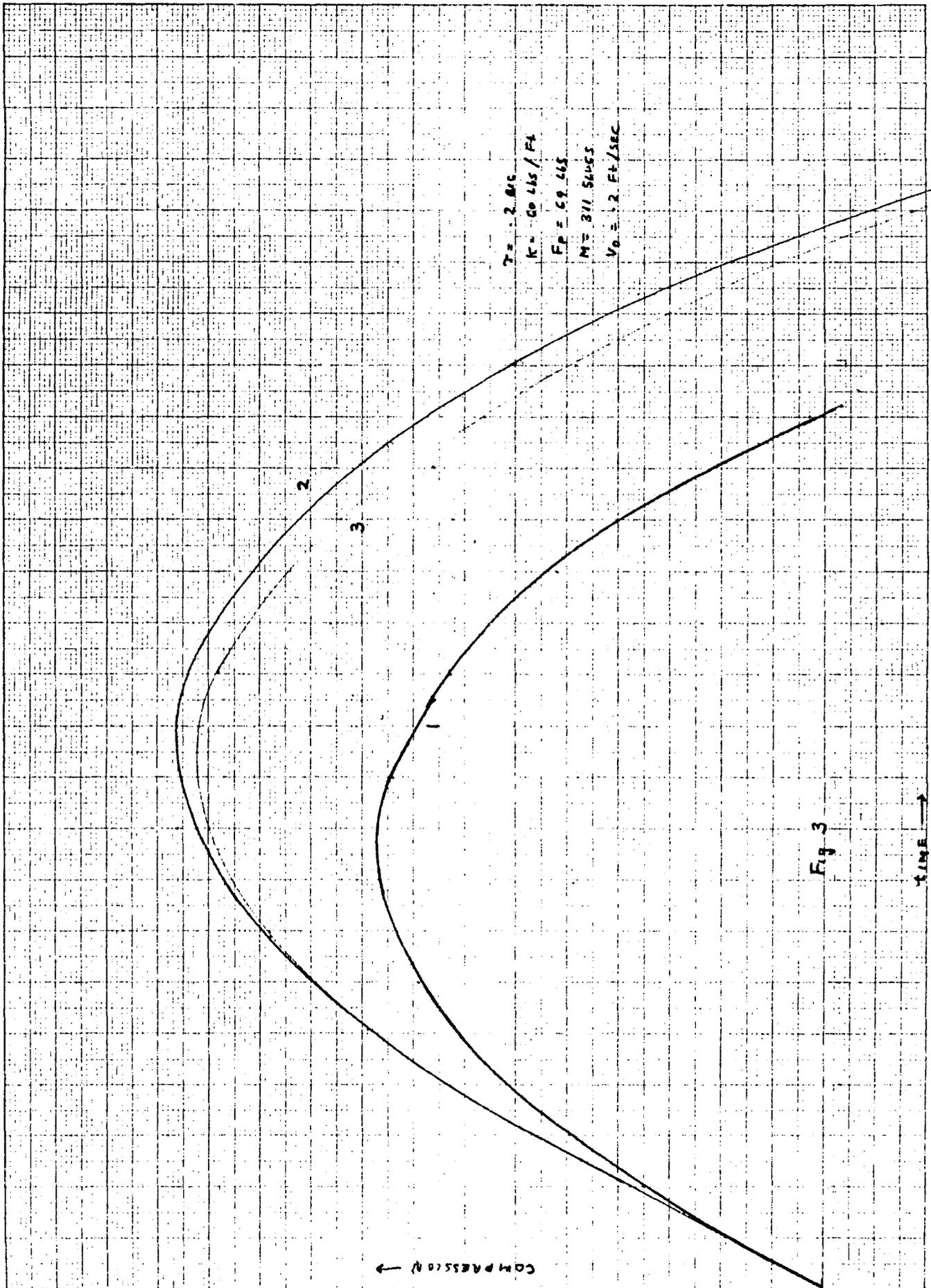
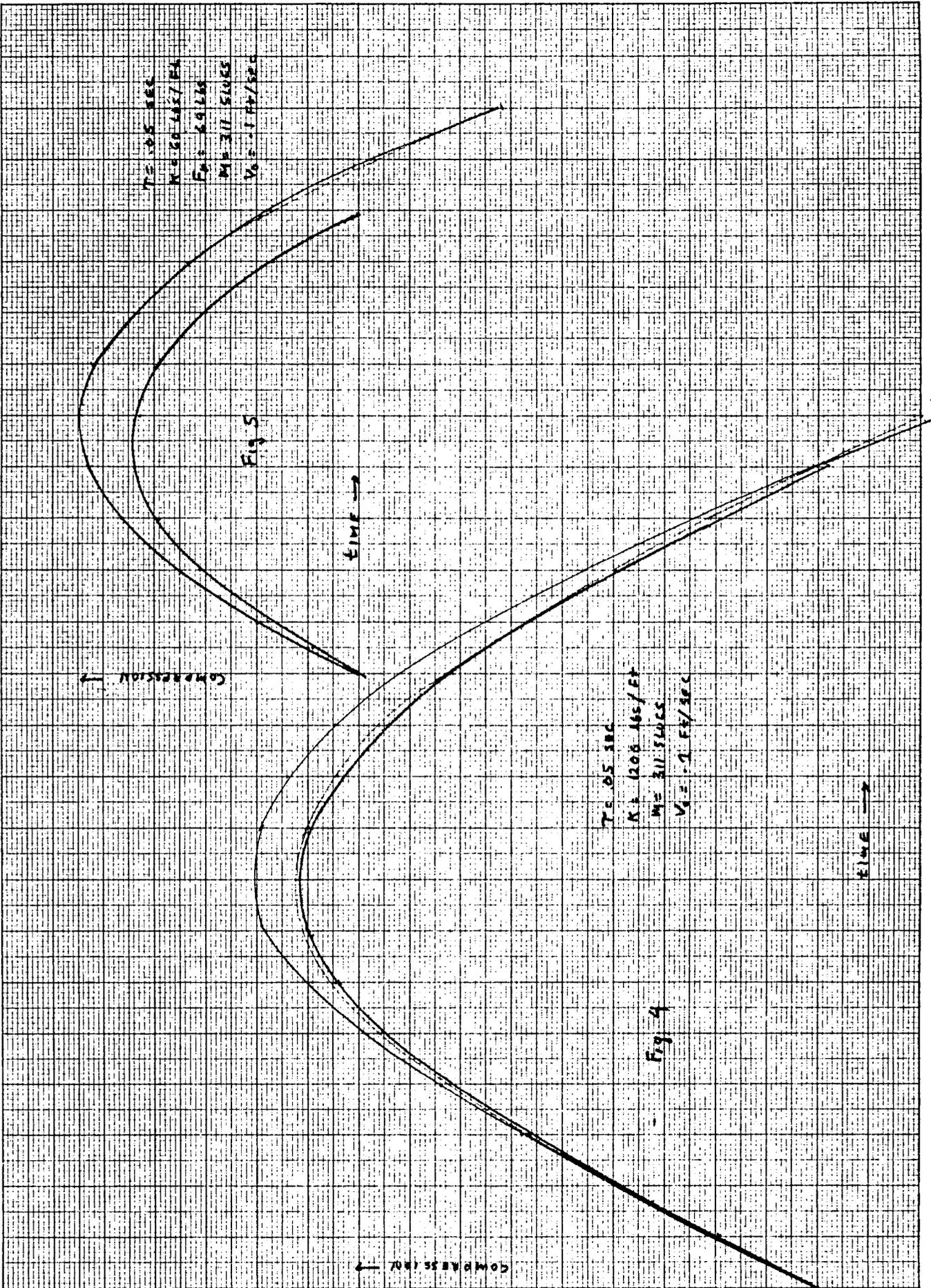
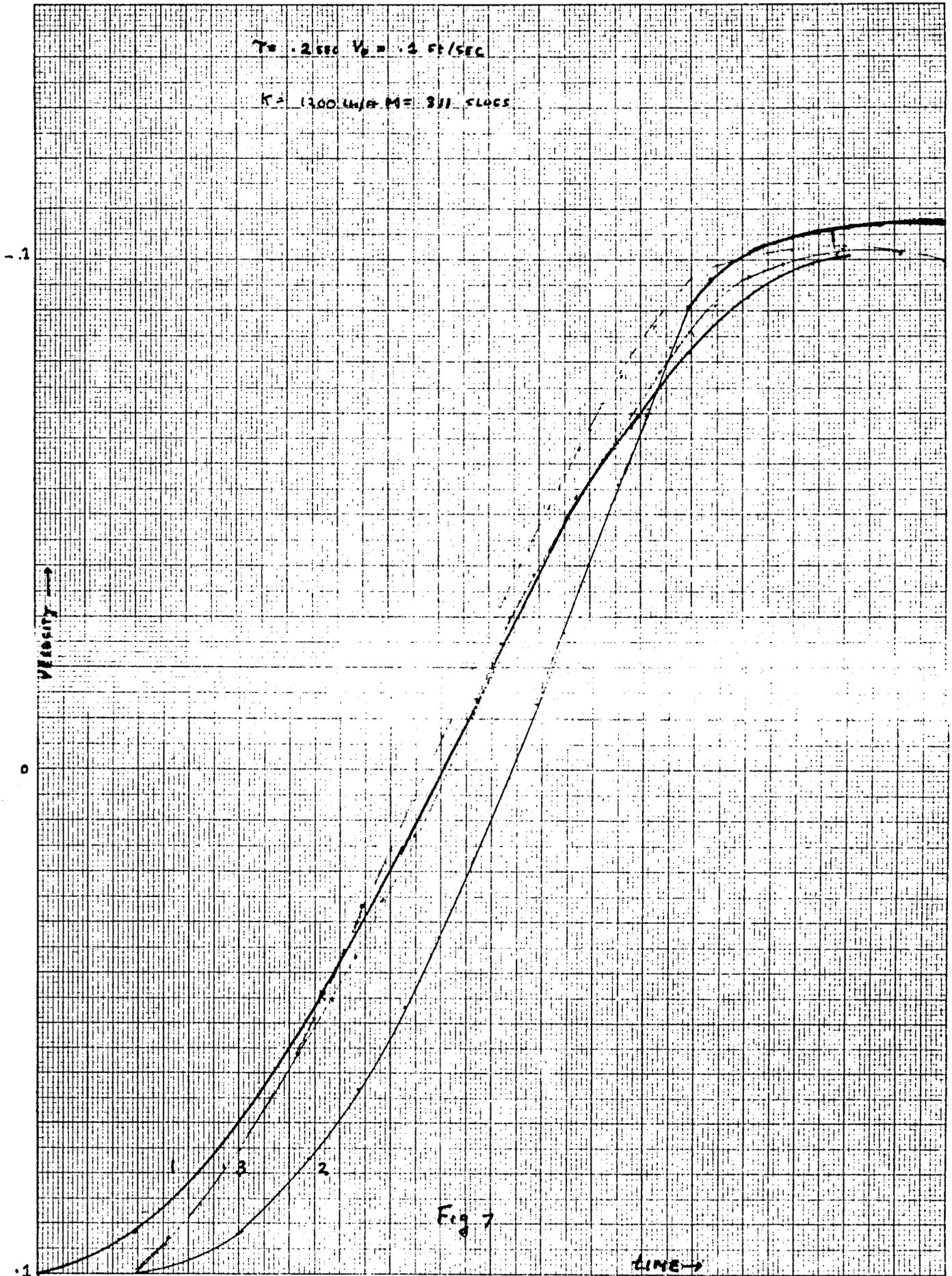


Fig 3





V CONCLUSIONS AND RECOMMENDATIONS

The second order delay equation model developed in this paper agrees with a recent analog simulation [1] of the effect of delay. It is also reasonably consistent (See Table 6 with the observed behavior (up to noise) of the 6 DOF system when $V_0 = .1$ or $.2$. The most disagreement happens at $V_0 = .05$ when our simulation gives a rebound velocity about 13% below the mean of the measurements in Table 1.

The model predicts that for a pure spring with no preload or force threshold V_f / V_0 depends only on τ and γ and not on V_0 . V_0 can also be computed quite accurately by a simple formula without numerically integrating the equation in the model. The use of a threshold increase the effect of delay. The results are relatively most severe at high threshold levels and low initial velocities. For thresholds under 100 lbs and delays less than .2 sec V_f can be predicted with reasonable accuracy by an approximate formula. Verification of these aspects of the model in hardware tests would strengthen the hypothesis that delay is the predominate factor in the observed errors in the system.

These tests and the isolation of other factors - such as phase shifts due to the filters-should be possible when the noise of the system is reduced. Some of this noise is probably caused by the sensitivity of the force sensors on the table to inertial effects, and by variable delay due to the aysynchronization of the AD/DA converters (see 2.2.4-2.2.5 above). There may also be another source of apparent noise: for the parameters chosen to model the probe and a force threshold the rebound velocity changes rapidly towards the end of the simulation. If the rebound velocity is measured at $t_f + \tau$ (cf. 2.2.3) an error of $\pm \Delta t$ in the time of measurement can produce a large relative change in V_f . For example when $V_0 = .2$ V_f is .343 ; 033 sec later it is .354; 066 sec later it is .38, 066 sec before it is .32. Hence, slight differences in time of final measurement in repeated runs can cause variations close to the order of magnitude of those observed.

Every effort ought to be made to reduce noise and delay. Several means to this end are suggested in [1]. We agree with these recommendations. Since a delay as low as 50 ms still causes considerable errors and this may be close to an achievable lower limit, means of compensating for delay need to be implemented.

In this study both a fourth order Taylor method and a Chebychev polynomial approximation to the transfer function e^{1s} gave good results for γ for an unpreloaded spring when no force threshold is used. The velocity profile correction in this case was excellent; penetration errors were reduced to under 10% (most of

the time less than 5%). The Chebychev algorithm "FIX*C" was the most accurate.

We regret however that neither algorithm works in the case of a preloaded spring, when the parameters model the probe tested in Table 1. Indeed it is remarkable that they hardly perturb the uncompensated position time profiles even when $\tau < .05$ sec. Their effects on an unpreloaded spring with a force threshold are more complicated but also unsatisfactory.

If these options are not satisfactory, a deeper mathematical analysis needs to be made on whether compensation of a preloaded system with delay is possible. On this topic the following rough observation seems appropriate. Since the difference between the compensated and the theoretically correct simple harmonic system is $O(\tau^n, x^{(n)})$ where x is the solution of the compensated system and n is the order of the method, the derivative $x^{(n)}$ may grow faster beyond $x = x_c$ than τ^n goes to zero. If this phenomenon occurs for all realistic compensation schemes it is doubtful that the 6 DOF system can simulate contact dynamics unless the delay is reduced far below 50 ms.

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APPENDIX I

INERTIAL NOISE AND THE FORCE SENSOR

The following is an attempted simulation of the effects of noise caused by the inertial forces of the table affecting the sensor. We integrate equation (2.8) (see § 2.2.5), thus representing a "shaking" of the sensor at each step in the movement of the table. Here the "noise parameter" ϵ is a uniformly distributed random variable taking values between 0 and either .5 or 1. The mass of the table is about 12 slugs so that the mass of the table and probe is 323 slugs. Note that the rebound velocities with the model exceed those observed. Parameters: $F = 69$ lbs $K = 60$ lbs/ft $\tau = .2$ sec.

I. $\epsilon = .5$

A. $V_0 = .05$

V_f
 .1778
 .204
 .2001
 .1967

C. $V_0 = .2$

V_f
 .395
 .4012
 .383
 .398

B. $V_0 = .1$

V_f
 .2773
 .2525
 .269
 .2649

II. $\epsilon = 1$

A. $V_0 = .05$

V_f
 .2248
 .2186
 .2356
 .2433

C. $V_0 = .2$

V_f
 .4434
 .46
 .4537
 .4376

B. $V_0 = .1$

V_f
 .3100
 .3429
 .3828
 .3298

APPENDIX II

LIST OF PROGRAMS

All programs are in BASIC completely interactive and implemented on MFC's Xerox Sigma 5 system.

<u>Name</u>	<u>Purpose</u>
LAG*R	Simulates the basic delay model.
PSP*I	Theoretical spring with arbitrary initial conditions
FIX*C	Chebyshev polynomial correction
FIX*T4	Fourth order Taylor method correction
LAG*N	Simulates the delay model with inertial noise

APPENDIX III

THE ALGORITHM "FIX*C"

Let $C_1 := .994526$

$C_2 := .995682$

$C_3 := .543081$

$C_4 := + .179519$

Set

$$X_c := C_1 X_T + C_2 X_T' + C_3 \ddot{X}_T + C_4 \dddot{X}_T$$

The following algorithm solves the IVP

$$X'' + \omega_0^2 X_c = 0$$

$$X(t) = X_0, \quad \dot{X}(t) = V_0, \quad 0 \leq t \leq T$$

For $i = 1$ to m

$$V_i := V_0$$

$$X_i := iH V_0 + X_0$$

Continue

For $i = m+1$ until satisfied

$$\begin{aligned} V_{i+1} = & V_i - \omega_0^2 \left\{ \frac{H}{2} C_1 (X_{i-m} + X_{i+1-m}) + \frac{H}{c} (V_{i-m} - V_{i+1-m}) \right. \\ & + C_2 (X_{i+1-m} - X_{i-m}) + C_3 (V_{i+1-m} - V_{i-m}) \\ & \left. + C_4 (a_{i+1-m} - a_{i-m}) \right\} \end{aligned}$$

$$a_{i+1} = -\omega_0^2 x_{i+1-m}$$

$$a_{i+1}^c = -\omega_0^2 (x_{i+1-m} + v_{i+1-m})$$

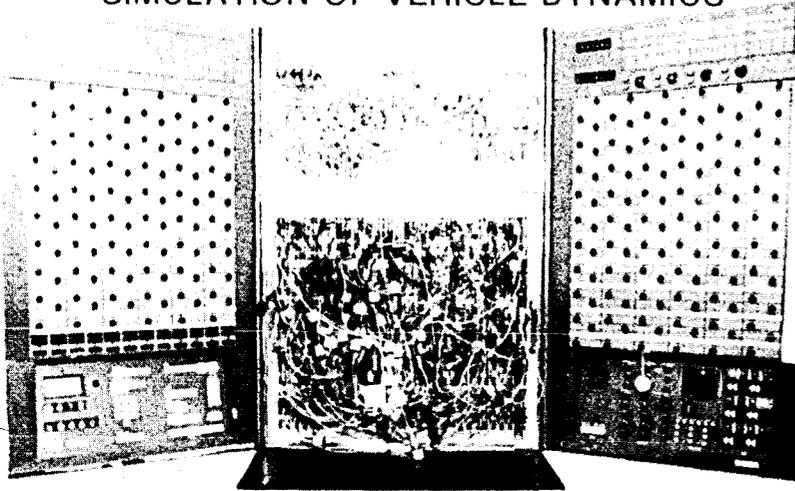
$$x_{i+1} = x_i + H/2 (v_i + v_{i+1}) + H^2/12 (a_i^c - a_{i+1}^c)$$

Continue

APPENDIX IV

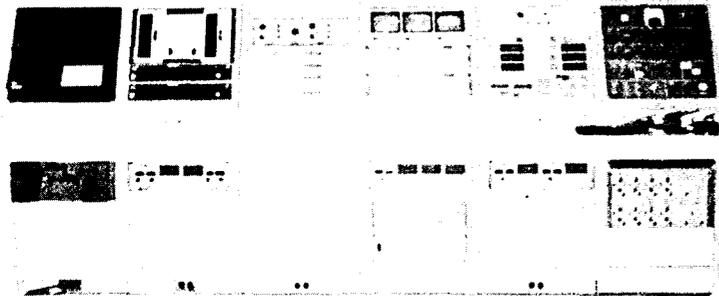
ILLUSTRATIONS OF THE 6 DOF MOTION SYSTEM

SIMULATION OF VEHICLE DYNAMICS

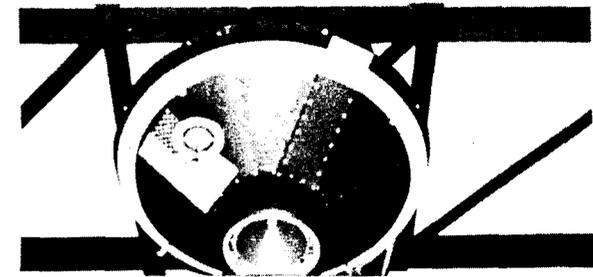


HYBRID COMPUTER

RELATIVE VEHICLE
MOTIONS

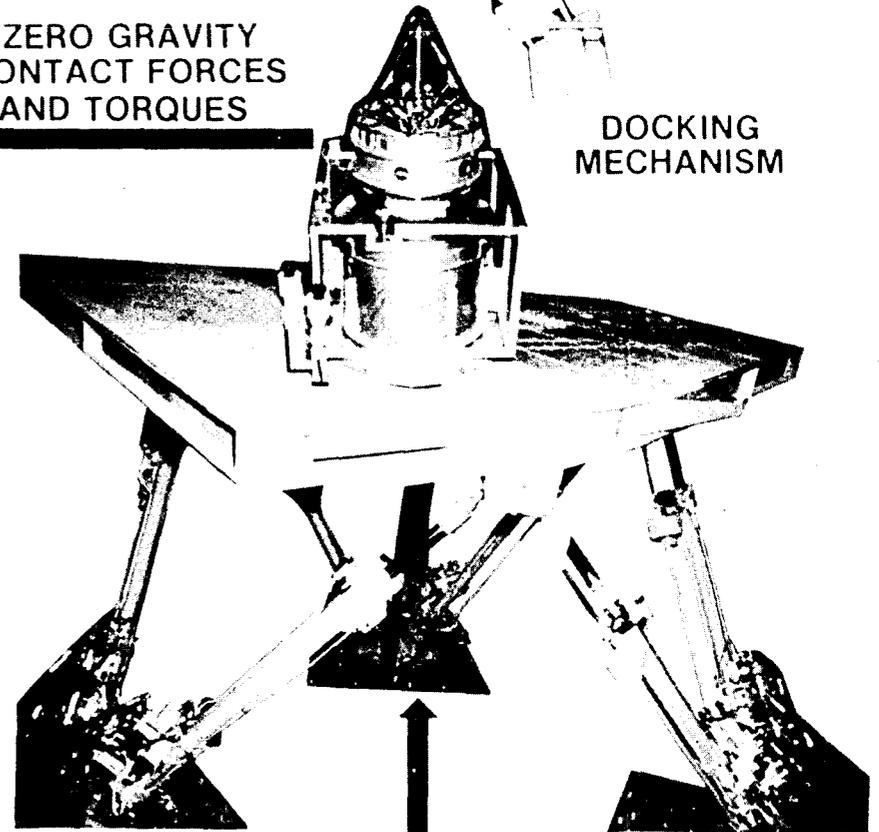


COORDINATE TRANSFORMATIONS



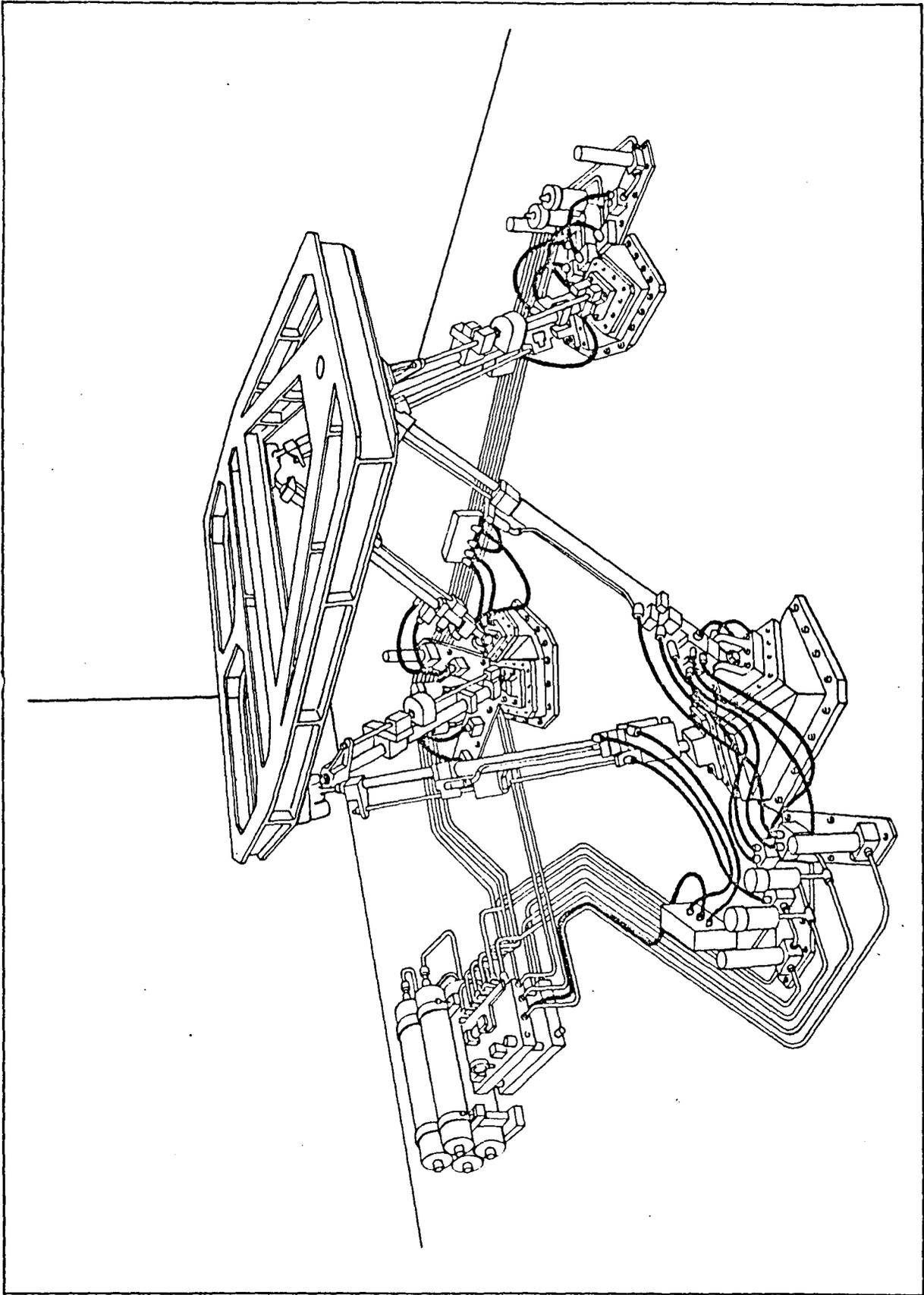
ZERO GRAVITY
CONTACT FORCES
AND TORQUES

DOCKING
MECHANISM



6 D-O-F MOTION SYSTEM

ORBITAL CONTACT DYNAMICS SIMULATION USING SIX DEGREE-OF-FREEDOM MOTION SYSTEM



SIX-DEGREE-OF-FREEDOM MOTION SYSTEM SKETCH

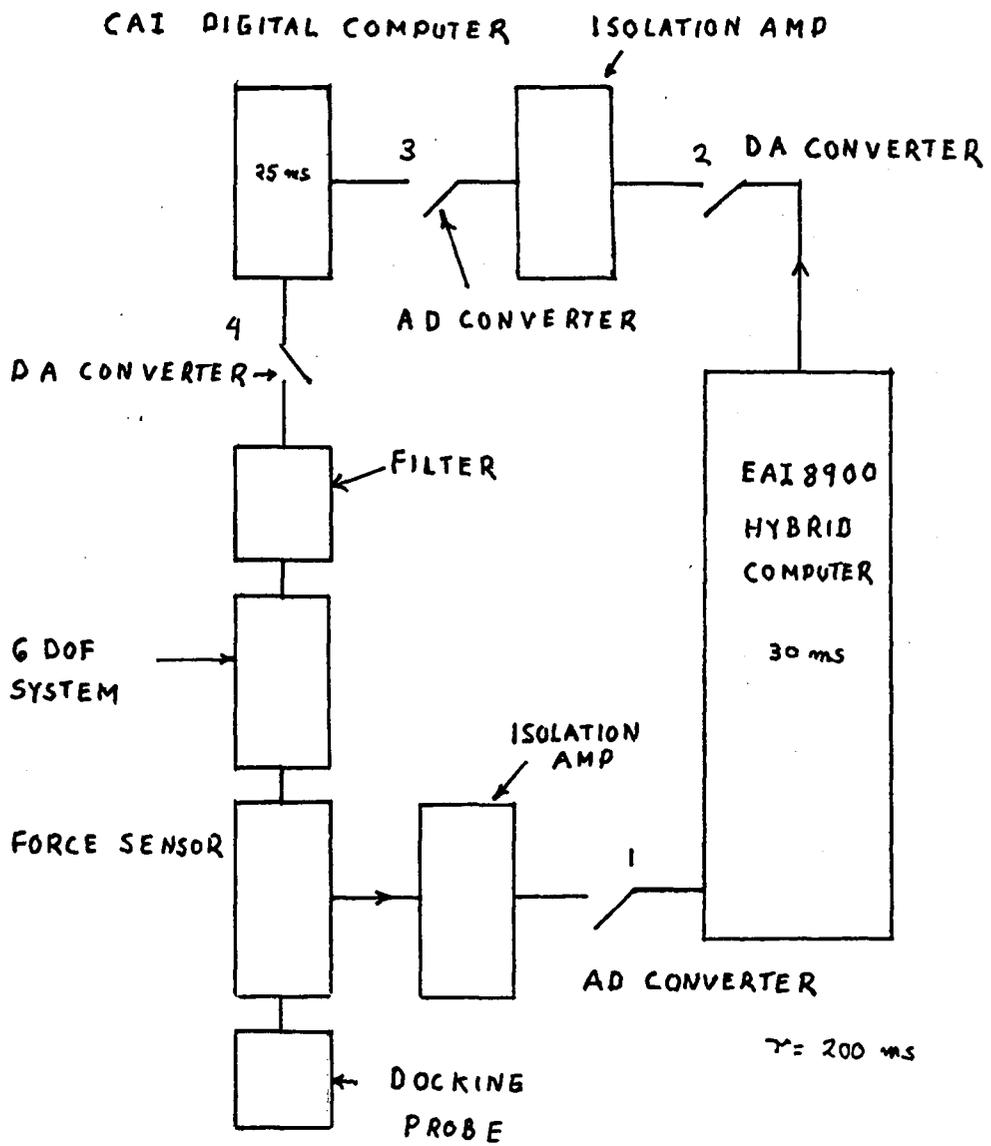


Fig 10

	<u>POSITION</u>	<u>RATE</u>	<u>ACCELERATION</u>	
			NO LOAD	23,000 LB. LOAD
PITCH	+ 30°, -20°	± 15°/SEC.	± 6.5 RAD/SEC ²	± 2 RAD/SEC ²
ROLL	± 22°	± 15°/SEC.	± 7.0 RAD/SEC ²	+ 1.6 - 2.0] RAD/SEC ²
YAW	± 32°	± 15°/SEC	± 6.0 RAD/SEC ²	± 2.0 RAD/SEC ²
VERTICAL	39 IN. UP, 30 IN. DOWN	± 24 IN./SEC.	± 1.6 g	± 1.0 g
LATERAL	± 48 INCHES	± 24 IN./SEC.	± 2.4 g	± 0.6 g
LONGITUDINAL	± 48 INCHES	± 24 IN./SEC.	± 2.0 g	± 0.6 g

PERFORMANCE OF SIX-DEGREE-OF-FREEDOM MOTION SYSTEM

1980

NASA/ASEE SUMMER FACULTY RESEARCH FELLOWSHIP PROGRAM

MARSHALL SPACE FLIGHT CENTER
THE UNIVERSITY OF ALABAMA

PILOT SIGNALS FOR LARGE ACTIVE RETRODIRECTIVE ARRAYS

Prepared By: C. H. Chan, Ph.D.
Academic Rank: Professor of Physics
University and Department: University of Alabama at
Huntsville
Department of Physics
NASA/MSFC
(Laboratory) Preliminary Design Office
(Division) Subsystems Design Division
(Branch) Electrical System Branch
MSFC Counterpart: Woolsey Finnell III
Date: August 1, 1980
Contract No.: NGT-01-002-099
(University of Alabama)

PILOT SIGNALS FOR LARGE ACTIVE RETRODIRECTIVE ARRAYS

BY

C. H. Chan, Ph.D.

Professor of Physics

University of Alabama at Huntsville

Huntsville, Alabama

ABSTRACT

It has been suggested that for large active retrodirective arrays, as in the solar power system, a two-tone uplink pilot signal with frequencies symmetrically situated around the downlink frequency be used in order to reduce ionospheric biases and to lower the cost since a two-tone receiver is economically much cheaper than a single-tone phase-locked receiver. Unfortunately such a system now faces the following well-known difficulties: (i) the π -ambiguity, (ii) a large phase difference between the downlink and uplink signals.

We show in this report how the π -ambiguity can be easily removed by using a two-tone uplink signal with both frequencies situated at one side of the downlink frequency, and the phase difference can be greatly reduced with a three-tone or a four-tone uplink pilot signal.

ACKNOWLEDGMENT

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I. Introduction

It has been suggested¹ that active retrodirective arrays^{2,3} would be particularly suitable as solar power satellite's antennas⁴ because they are inherently failsafe. The active retrodirective array works on the so-called phase-conjugation principle. It electronically points a microwave beam back at the apparent source of an incident pilot signal. Retrodirectivity is achieved by retransmitting from each element of the array a signal whose phase is the conjugation of that received by the element. In the satellite power system, the pilot source on ground may be situated at the center of a large rectenna and the retrodirective array is the space antenna in geosynchronous orbit.

Retrodirectivity can be most easily achieved if the uplink signal and the downlink beam have the same frequency. But due to input-output isolation problems, the uplink frequency is either upshifted or downshifted from the downlink frequency, a phase-locked receiver is used to achieve phase conjugation. When the uplink and downlink frequencies are different and because the ionosphere and transmission lines are dispersive, the conjugated uplink phase is no longer exactly equal to the downlink phase and the beam coherence at the rectenna can be lost. The downlink beam then points to wrong directions and this is known as beam squint.

A two-tone pilot uplink signal with frequencies symmetrically situated around the downlink frequency has subsequently been suggested. The two-tone uplink signal circumvents the beam squint problem. It reduces ionospheric biases and biases due to the dispersion of the transmission line. It also lowers the cost since a two-tone receiver is much

cheaper than a single-tone phase-locked receiver. But, it introduces a new problem, known as the π -ambiguity, and the old problem that the conjugated uplink phase could still differ from the downlink phase by an intolerable amount remains. Raytheon⁵, Boeing⁶, and Rockwell⁷ have all made further suggestions to remedy the problems. Their solutions not only are complicated and require much extra hardware in the already very complicated phase conjugation circuitry, but certain problems still remain.

In this report new designs of two-tone and multi-tone uplink signals with frequencies situated at one side of the downlink frequency are suggested. This method removes the above mentioned difficulties and does not require extra components in the phase conjugation circuit. We shall review in our next section the basic principle of phase conjugation and where the problems are in a symmetrically situated two-tone uplink signal. In section III we show how these problems can be circumvented with our new designs.

II. Difficulties With Symmetrically Situated Two-Tone Uplink Signal

A retrodirective array electronically transmits a microwave beam back to the apparent source of a coherent pilot signal. The beam radiated by self-phasing antenna may or may not be coherent across the aperture but it is coherent when it arrives back at the source. Retrodirectivity is the result of phase conjugation of the pilot signal received by each element of the array. Let the phase of the pilot signal of angular frequency ω received by the k th element of the array at time t be $\phi_k(t) = \omega(t - r_k/c)$ where r_k is the distance from the k th element to the source. We define the conjugate of ϕ_k to be

$$\phi_k^*(t) = \omega(t + r_k/c) + \phi_0$$

where ϕ_0 is an arbitrary phase offset but is constant over the entire array. The phase of the beam received from the k th element by a receiver located at the pilot source ($r = 0$) is, at time t ,

$$\phi_k(t,0) = \omega\left(t + \frac{r_k}{c} - \frac{r_k}{c}\right) + \phi_0 = \omega t + \phi_0$$

Thus the contributions to the field at $r = 0$ from various elements of the array are all in phase at that point.

In the above simple example the uplink frequency was chosen to be the same as the downlink frequency. This restriction is neither necessary nor desirable and is usually avoided because of input-output isolation problems. When these two frequencies are different, a phase-locked receiver is used. Retrodirectivity can still be achieved - provided that the propagating medium is nondispersive.

Due to the fact that single-tone phase-locked receivers are expensive and the ionosphere and transmission lines are dispersive, a two-tone uplink signal with frequencies symmetrically situated around the down-link

frequency was suggested and the average of the phases of the uplink tones be taken as a good estimate of the phase at the downlink frequency. Such a system lowers the cost and removes partially the difference between the uplink and downlink phases but it also introduces a new problem known as the π -ambiguity. We shall review here where these problems are. We shall use the ionosphere as an example to study the effect on the phase conjugation due to the dispersive property of the propagating medium.

The dispersion relation for ionosphere with $\omega \gg \omega_p$ is

$$k = \frac{\omega}{c} \left(1 - \frac{\omega_p^2}{2\omega^2} \right) \quad (1)$$

where c is the speed of light in vacuum, k the wave number, ω the angular frequency and ω_p is the plasma frequency. The plasma frequency ω_p is related to the electron density as

$$\omega_p^2 = \frac{Ne^2}{\epsilon_0 m} \quad (2)$$

where ϵ_0 is the vacuum permittivity, e the charge of the electron, m the mass of the electron and N is the volume electron density.

From Eqs. (1) and (2)

$$k = \frac{2\pi f}{c} - \frac{Ne^2}{4\pi\epsilon_0 fcm}$$

where $f = \omega/2\pi$ is the frequency. For a beam to transverse a path length L , the total phase change is

$$\begin{aligned} \phi &= \int k \, ds \\ &= \frac{2\pi fL}{c} - \frac{e^2}{4\pi\epsilon_0} \frac{1}{fcm} \int N \, ds \\ &= A f - \frac{B}{f} \end{aligned} \quad (3)$$

where $A = \frac{2\pi L}{c}$, $B = \frac{e^2}{4\pi\epsilon_0} \frac{1}{c \text{ m}} \int N \text{ ds}$, and $\int N \text{ ds}$ is the columnal electron density.

With uplink frequencies situated around the downlink frequencies, we have $f_1 = f_D - \Delta f$ and $f_2 = f_D + \Delta f$ where $f_D = 2.45 \text{ GHz}$ is the downlink frequency. Using the notation of Eq. (3)

$$\phi(f_1) = Af_1 - \frac{B}{f_1} = \phi_1$$

and
$$\phi(f_2) = Af_2 - \frac{B}{f_2} = \phi_2$$

then the average of the two phases is

$$\begin{aligned} \bar{\phi} &= \frac{1}{2} (\phi_1 + \phi_2) \\ &= Af_D - \frac{B}{2} \left(\frac{1}{f_D - \Delta f} + \frac{1}{f_D + \Delta f} \right) \\ &= Af_D - \frac{B}{f_D} (1 + \epsilon^2 + \epsilon^4 \dots) \end{aligned} \tag{4}$$

where $\epsilon = \Delta f/f_D$ is a small number. We may compare this phase $\bar{\phi}$ with the downlink phase

$$\phi_D = Af_D - \frac{B}{f_D}$$

e.g. their difference is of the order

$$\Delta\phi = | \phi_D - \bar{\phi} | \approx \frac{B\epsilon^2}{f_D}$$

To estimate this difference, we shall assume

$$\int N \text{ ds} = 5 \times 10^{17} \text{ electrons/m}^2$$

a rather large value for $\int N \text{ ds}$ but taking into account for the possible worst condition. With $\Delta f = 50 \text{ MHz}$, $\Delta\phi$ is estimated to be about 4° , which is not too small.

When we took the average of phases ϕ_1 and ϕ_2 in Eq. (4), there could introduce an ambiguity known as the π -ambiguity. Let

$$\phi_1 + \phi_2 = K(2\pi) + \Delta$$

where $0 \leq \Delta < 2\pi$ and K is a positive, zero or negative integer.

Hence

$$\bar{\phi} = K\pi + \frac{\Delta}{2}$$

In performing the phase average, the $K\pi$ term could get lost. For K even, no damage is done. For K odd a π error is introduced and one would conjugate the wrong phase.

In order to remove these two difficulties, especially the π -ambiguity, Raytheon⁵, Boeing⁶, and Rockwell⁷ have all made suggestions. Their solutions are very complicated and usually require a lot of hardware in the receiving and phase conjugation circuitries with much added cost. Furthermore, their solutions do not solve the problem completely. We shall show in our next section several simple solutions which circumvent the above mentioned difficulties and do not add extra costs.

III. New Designs of Pilot Beam System

In this section, several uplink designs are proposed. The first simple design avoids the π -ambiguity and the rest are improved versions of the first one. They are all free from the π -ambiguity but reduce the phase difference $\Delta \phi$ to various orders.

(i) This is also a two-tone uplink, but the two frequencies are both on one side of the downlink frequency with

$$\begin{aligned} f_1 &= f_D - \Delta f \\ \text{and } f_2 &= f_D - 2\Delta f \end{aligned} \quad (5)$$

where f_D is the downlink frequency. We now let $\bar{\phi} = 2\phi(f_1) - \phi(f_2)$ to be the estimation of the downlink phase. With the notation of Eq. (3)

$$\begin{aligned} \bar{\phi} &= Af_D - B \left(\frac{2}{f_D - \Delta f} - \frac{1}{f_D - 2\Delta f} \right) \\ &= Af_D - \frac{B}{f_D} (1 - 2\epsilon^2 + \dots) \end{aligned}$$

where again $\epsilon = \Delta f/f_D$ is a small number. In this simple design the π -ambiguity is removed since no division of the phase is used anywhere. The combined phase $\bar{\phi}$ is also a good estimate of the downlink phase ϕ_D . Their difference $\Delta \phi = | \phi_D - \bar{\phi} |$ is

$$\Delta \phi = \frac{2B}{f_D} \epsilon^2 \quad (6)$$

which is of the same order as the one with two uplink frequencies situated symmetrically around the downlink frequency and is about 8° when the same values for $\int N ds$ and Δf are used as in the last

section. We also note that in this simple design no extra components are required in the receiving circuitry.

(ii) This is an improved version of the first design. It requires three uplink tones but it also greatly improves the accuracy in estimating the downlink phase ϕ_D . The three frequencies are

$$f_1 = f_D - \Delta f$$

$$f_2 = f_D - 2\Delta f$$

$$\text{and } f_3 = f_D - 3\Delta f$$

We now let

$$\bar{\phi} = 3\phi(f_1) - 3\phi(f_2) + \phi(f_3) \quad (7)$$

to be the estimation of the downlink phase. With the notation of Eq. (3)

$$\begin{aligned} \bar{\phi} &= Af_D - B \left(\frac{3}{f_D - \Delta f} - \frac{3}{f_D - 2\Delta f} + \frac{1}{f_D - 3\Delta f} \right) \\ &= Af_D - \frac{B}{f_D} (1 + 6\epsilon^3 + \dots) \end{aligned}$$

where again $\epsilon = \Delta f/f_D$. In this design, there is no π -ambiguity as before and the difference between ϕ_D and $\bar{\phi}$ is reduced by an extra factor ϵ , e.g. the difference $\Delta\phi$ is now

$$\Delta\phi = \frac{6B}{f_D} \epsilon^3 \quad (8)$$

With $\Delta f = 50$ MHz and $\int N ds = 5 \times 10^{17}$ electrons/m², this difference $\Delta\phi$ is only about 0.5°, which is small.

(iii) This version can be used in the event we would like to have an even smaller $\Delta\phi$ or we would like to use a larger Δf , which

would otherwise result in a too large $\Delta\phi$ even with a three-tone uplink design. These new specifications can be achieved at the expense of adding a fourth tone. The four frequencies are

$$f_1 = f_D - \Delta f$$

$$f_2 = f_D - 2\Delta f$$

$$f_3 = f_D - 3\Delta f$$

$$f_4 = f_D - 4\Delta f$$

and we now let

$$\bar{\phi} = 4 \phi(f_1) - 6 \phi(f_2) + 4 \phi(f_3) - \phi(f_4) \quad (9)$$

$$= Af_D - B \left(\frac{4}{f_D - \Delta f} - \frac{6}{f_D - 2\Delta f} + \frac{4}{f_D - 3\Delta f} - \frac{1}{f_D - 4\Delta f} \right)$$

$$= Af_D - \frac{B}{f_D} (1 - 24 \epsilon^4 + \dots)$$

In this design, the phase difference is reduced further by a factor ϵ , e.g. $\Delta\phi$ is only

$$\Delta\phi = \frac{24B}{f_D} \epsilon^4 \quad (10)$$

With $\Delta f = 50$ MHz and $\int N ds = 5 \times 10^{17}$ electrons/m², this difference is only about 0.04°. Even with $\Delta f = 100$ MHz, this difference is only 0.66° which is still very small.

In all these three designs the uplink frequencies are all on the lower side of the downlink frequency and they are all equally spaced. These are not the only choices. One can use all frequencies on the upper side of the downlink frequency and they need not be equally spaced

either. As a simple example one may very well have a two-tone uplink with

$$\begin{aligned} f' &= f_D + 2\Delta f \\ f'' &= f_D + 3\Delta f \end{aligned} \tag{11}$$

and $\bar{\phi} = 3\phi(f') - 2\phi(f'')$. It will work just as well.

So far we have used the ionosphere as an example to show how the dispersion of the transmission medium could introduce sizable biases between the uplink and downlink phases. For a two-tone uplink signal, this phase difference is about 8° . We suggested one way to suppress it is to use a three-tone uplink. However, it turns out that if the biases were purely due to the ionosphere, it is really not necessary to use a three-tone uplink. This is because even though the phase difference between $\bar{\phi}$ and ϕ_D in our two-tone uplink design is large, its variation from subarray to subarray will be decimal if not infinitesimal. To corroborate more on this statement, from Eq. (6) and the definition of B , the difference between the uplink and downlink phases from the pilot to the k th subarray for a two-tone uplink is

$$\Delta\phi_k = \frac{2(\Delta f)^2}{f_D^3} \frac{\epsilon^2}{4\pi\epsilon_0} \frac{1}{\text{cm}} \int^N ds_k \tag{12}$$

where $\int^N ds_k$ is the columnal electron density along the path from the pilot to the k th subarray. Since the horizontal dimension of the region of the ionosphere that would be transversed by the pilot signal to any subarray is very small, typically of the order less than 100 meters, the transverse variation of the total electron along a path of several hundred kilometers within a tube of diameter less than 100 meters would be very small. Hence even though $\Delta\phi_k$ is estimated to be about 8° , its variation is at least several orders less. Then with

the use of the phase from one of the subarrays as the reference phase, this difference $\Delta\phi_k$ can be subtracted out and the remainder be treated as a constant phase offset which has no effect on the retrodirective beam. In this sense, though the ionosphere is dispersive, it does not cause any problem, and a two-tone uplink is sufficient.

On the other hand, with the use of "central phasing" we cannot avoid the extra path length of transmission lines for some subarrays. Since these lines are not dispersionless, they will introduce a sizable phase difference. This phase difference can also be estimated. If we assume that this transmission line is a wave guide, its dispersion relation is well known

$$k = \frac{2\pi}{c} \sqrt{f^2 - f_\lambda^2} \quad (13)$$

where f_λ is the cut-off frequency. Hence the phase for any link is

$$\phi(f) = \frac{2\pi\ell}{c} \sqrt{f^2 - f_\lambda^2} \quad (14)$$

where ℓ is length of the transmission line. Just as before the phase differences for various uplinks can be calculated. For a two-tone uplink with $\bar{\phi} = 2\phi(f_1) - \phi(f_2)$, $\ell = 500$ meters and $f_\lambda = \frac{1}{2}f_D$, where f_1 and f_2 are given as in Eq. (5), $\Delta\phi$ is

$$\Delta\phi = |\phi_D - \bar{\phi}| = 256^\circ$$

Similarly for a three-tone uplink with $\bar{\phi} = 3\phi(f_1) - 3\phi(f_2) + \phi(f_3)$, one obtains

$$\Delta\phi = 23^\circ$$

For a four-tone uplink with $\bar{\phi} = 4\phi(f_1) - 6\phi(f_2) + 4\phi(f_3) - \phi(f_4)$,

$\Delta\phi$ is further reduced to

$$\Delta\phi = 3^\circ$$

As we see in this example, the phase difference for a two-tone uplink is very large but this number is greatly reduced in a three-tone or four-tone uplink. These multi-tone uplink signals can be used as a useful alternative method to suppress biases due to the dispersion of the transmission line and the medium.

IV. Conclusion

In our last section we illustrated how simple designs can be used to eliminate the π -ambiguity and reduce the ionospheric biases and biases from dispersive transmission lines. We also note that none of our designs require extra components in the receiving circuitry. All one is required to do is to obtain $\bar{\phi}$, which can be achieved rather easily, and simply conjugate it and use it as the phase of the downlink signal leaving the space antenna.

It is also important to remember that we are here to design a pilot beam system as simple as possible with the phase received by the array as close to the downlink phase as possible. We are not asked to and it is not necessary to determine the ionospheric electron density as required in some other designs.

Lastly our designs of pilot beam can be implemented easily in any large retrodirective arrays. Their advantages are (i) avoiding using phase-locked receiver (ii) free from phase ambiguity (iii) greatly reducing biases due to dispersion of the transmission line and medium (iv) very simple to be constructed.

It will be extremely interesting to have such a system built and tested in the very near future.

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ION PLATING STUDIES FOR
HIGH TEMPERATURE APPLICATIONS

Prepared by:	Jack H. Davis, Ph.D.
Academic Rank:	Associate Professor
University and Department:	The University of Alabama in Huntsville Department of Physics
NASA/MSFC: (Laboratory) (Division) (Branch)	Materials and Processes Engineering Physics Physical Sciences
MSFC Counterpart:	Ann F. Whitaker
Date:	August 11, 1980
Contract No.:	NGT-01-002-099 University of Alabama



A. Abstract

An experimental project was undertaken to ion plate by electron beam evaporation Al films onto 4340 steel substrates using (and at the same time troubleshooting) the custom built V.T.A. 7375 electron beam ion plating system. A careful recent literature and commercial vendor survey indicates possible means (reported herein) of improving the trouble plagued V.T.A. system.

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C, Introduction

Holliday¹² gave the following concise introduction to ion plating.

Since first being reported in the literature in 1963 (ref. 1), ion plating has progressed to the point that today it is used in several commercial processes including the aluminum coating of fasteners used in the manufacture of aircraft (ref. 2). Ion plating has several positive qualities. Probably the most important of which are outstanding film adhesion and deposition on all sides of the substrate (including coverage into cavities) whereas normal vacuum deposition gives the usual line-of-sight coverage. Other qualities obtainable, depending on the material deposited, include exceptional corrosion resistance, high film purity, fine grain structure, very low-coefficient of friction, and improved mechanical properties of metals (ref. 3-5).

Several articles have been published describing the ion-plating process (e.g., ref. 6 and 7). Basically the process consists of two phases. In the initial cleaning phase the object to be plated (substrate) is made the cathode of a dc inert gas discharge or plasma. Some of the inert gas atoms are ionized and accelerated towards the substrate. The bombardment of the substrate by these ions having high kinetic energy produces a clean surface preparatory to the actual plating process. In the plating phase the material to be plated is evaporated from the anode while maintaining the inert gas discharge. Some of the coating material atoms are also ionized and accelerated towards the substrate. These ions follow the electric field lines which terminate on all sides of the biased substrate and thereby help coat not only the front but all sides of the substrate. However, since the degree of ionization in many plasmas is very low (0.1-2%), it has been suggested that this mechanism is probably secondary to gas scattering of neutral atoms in contributing to the highthrowing power of ion plating (ref. 8). Thus the energies of the impinging particles range from that obtained from ions accelerated by the potential difference between the anode and cathode to the thermal energy of unionized atoms. These factors, along with the heating of the substrate surface due to its continued bombardment, produce the graded-fused interface, which provides the superior adherence and improved mechanical properties characteristic of ion plating.

D. Objective

A Materials and Processes Laboratory objective is to have the capability to coat a wide variety of mechanical parts with a host of good-bonding protective thick (up to 40 μ) films. Ion plating is well suited for providing good film bonding.

Electron beam evaporation of source materials provides a technique applicable to the widest range of materials, even the refractory metals which are vital to both high temperature rocket engines and to high temperature superconductors. However, most electron beam evaporation ion plating systems were not off the shelf but were custom built and debugged.

The immediate objective was to debug and put into smooth operation the trouble plagued (for its five year life) electron beam evaporation ion plating system Model 7375 custom built by Vacuum Technology Associates (now out of business).

E. Experimental Hardware

The ion plating apparatus used, V.T.A. Model 7375, was custom built by Vacuum Technology Associates and consisted of three main components: (1) The vacuum system, a Varian NRC 3117, attained an ultimate pressure of 10^{-6} torr using a 6" diffusion pump. (2) The electron beam gun system was a Sloan Multihearth 270^o gun powered by a Sloan Model Five/Ten Power Supply. (3) The high voltage supply for the substrate bias was a 10 KV/500 ma unit made by Vacuum Technology Associates.

A typical ion plating apparatus is shown in Fig. 1.

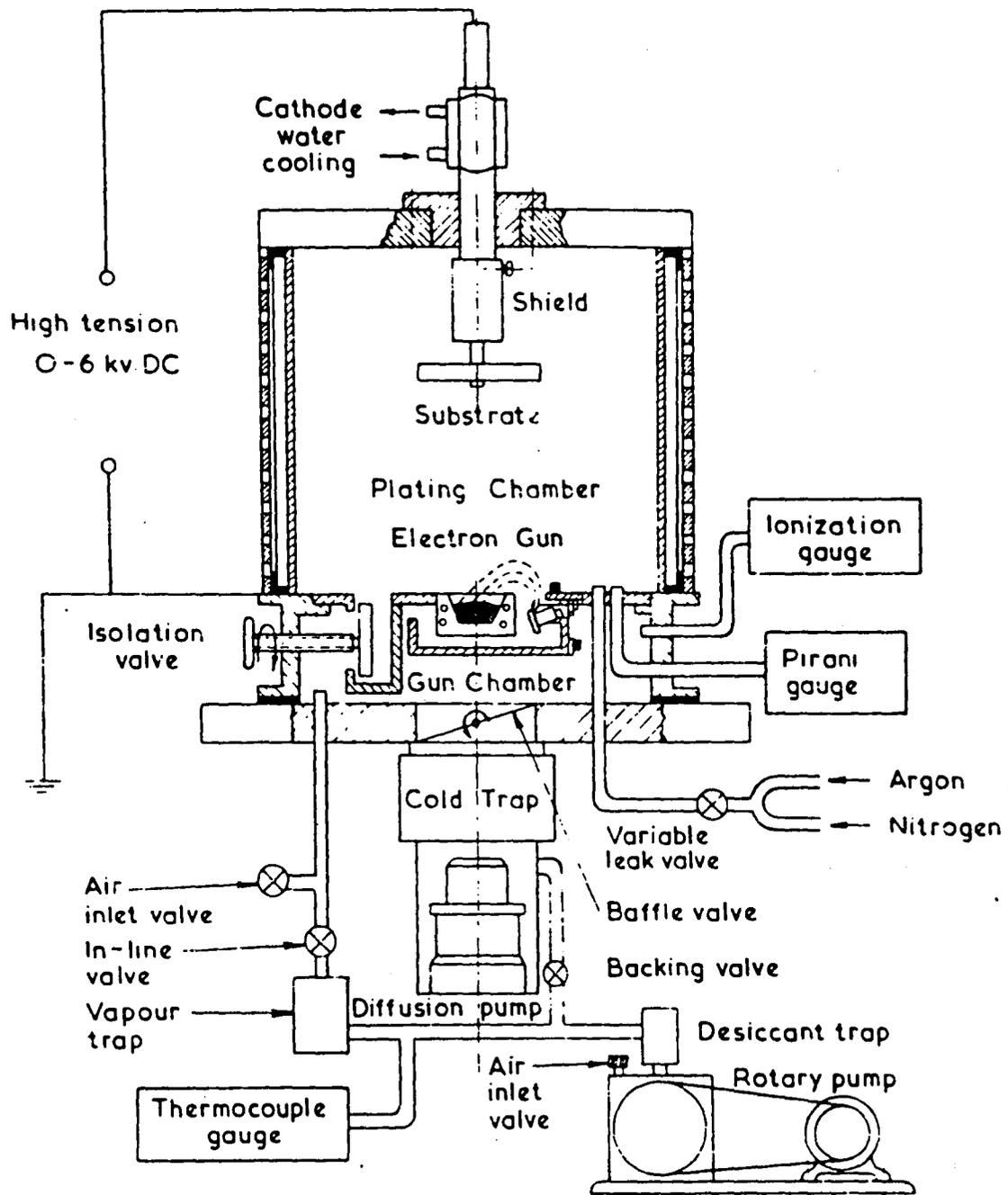


Figure 1. Ion Plating Apparatus with an Electron Beam Gun
(D. G. Teer 1977)

F.1. Experimental Improvements Made

a. A brass cathode sample holder was removed from the system since brass was sputtercoating the substrate during precleaning and during ion plating. The chance of Zn contamination is thus reduced.

b. A cylindrical ($5\frac{1}{2}$ in. diameter by 3 in. long) grounded skirt extension was constructed which provides two inches of extra shielding to the top of the sample's support. This could reduce some of the above brass sputtering and efficiently concentrate the discharge near the sample substrate.

c. Due to modifications, the position of the sample holding rectangle may now be continuously located by a screw adjustment from direct contact with the cooled cathode plate to a position 5" lower.

d. Greater plating rates are anticipated as minimum sample substrate to hearth distances from about 4" to about 2". Some concern existed about reduced throwing power with the sample close; however, Holliday¹¹ found no reduction in throwing power as hearth-substrate distances are reduced to from 5" to 3". The mean-free-path is in the several mm range so the transport is diffusion controlled.

F.2. Discussion of Recent Literature

a. In a recent (Oct. '79) review article on ion plating sources, T. Spalvins¹⁶ mentions the following advantages of Wan's¹⁷ hollow cylinder electron beam source:

- (1) It operates in the plasma.
- (2) It does not have the hot filament or the filament related problems.
- (3) It is easy to design and construct.

The main disadvantage is some contamination due to the cylinder sputtering into the substrate.

b. Ion plating by radio frequency induction evaporation is mentioned by Spalvins¹⁸. The source metal is placed in a ceramic crucible which is centered in the induction coil. Frequency reduction by a factor of 1/6 to 75 MHz prevented coil arcing in the plasma.

c. Rules for ion plating:¹⁶

- (1) Ideal conditions for ion plating include:
 - (a) Edge of cathode dark space is as close as possible to the boat.
 - (b) Dark space is as wide as possible.
- (2) A cathode to anode distance of 15 cm is adequate to establish a glow discharge.
- (3) A metal screen is used to coat plastics and insulators.
- (4) Precleaning (surface oxide removal) is usually complete when the discharge current flattens out with time. It may take one hour for stainless steel.
- (5) Bare wire should not be used to make high voltage connections according to Mattox.⁶

d. References 19 and 20 are listed because they are recent.

e. Discussion

The only reason which comes to mind for having a wide dark space is that the high electric field is only in the dark space and not in the plasma. The larger the dark space volume the greater the region from which to draw ions into the cathode substrate. If the boat to dark volume boundary distance is so reduced (the diffusion transport distance is small), the plating rate will be greater.

The metal shield stack (3" high x 4.5" diameter) is now recognized to have a detrimental effect since it acts as a Faraday (shield) cage around the hearth zeroing the E field needed to pull the positive ions to the substrate.

Even with the above stack shield removed, the top of the crucible is surrounded on 3 sides by a 1½" high by 3" wide magnetized rectangle having 3 sides 1½" above the top of the crucible (about 2" above the crucible bottom). This shielding could reduce the local electric field near the melted source enough to dramatically reduce the coating rate. The Sloan 270° evaporation hearth and gun were designed for vacuum evaporation and not for ion plating; thus, the irregular top

profile is not unexpected. The shielding rectangle arms are magnetized and could not be ground down or removed without a magnetic compensation. Perhaps a large external Helmholtz coil could hopefully compensate and permit removal.

For both ion plating systems shown in ref. 9 and 10 the crucible cup top is flush with the flat baffle plate. Likewise in Teer's²¹ ion plating unit, nothing is shown higher than the hearth. The top of the hearth is flush with the flat baffle plate. See Fig. 1. The Heinz¹³ hearth is only slightly shielded by the horizontal gun.

The sparse $1/2 \mu$ coating on sample #11 could be due to excessive shielding of the source.

F.3. Vendor Search

On June 3rd letters requesting complete brochures with prices on electron beam evaporation ion plating systems and components were sent to all 16 vendors listed under Vacuum Coating Equipment on the Science Instrumentation Index. Not that a new system purchase is expected but that new components might be acquired for a much-needed upgrading of the V.T.A. system.

Denton Vacuum offered a complete unit with a DEG-801 Gun on the DIP-1 ion plating unit.

Hughes uses extensively, but rarely builds, ion plating units for customers. Mr. Cristy at 213/648-2345, Ext. 84369, may however be of help.

Nanotech* literature and prices just arrived listing a wide range of ion plating units with accessories. In fact they offer an option much like NASA's V.T.A. system except the e-beam gun is a Sloan 8 kW or 12 kW (costing about \$31,000) instead of our Sloan 5 kW. Unfortunately no gun mounting drawings were sent. Sloan no longer lists the 5 kW (Five/Ten) power supply. Since the V.T.A. system hearth is rated 12 kW it may be underpowered.

Inflicon Leybold-Heraeus responded indicating no ion plating systems for sale.

F.4. Films Plated

a. e-beam generated films.

All samples were #4340 hot roll steel disk one inch in diameter by $1/4$ inch thick. Samples #3, #4, and #11 were

*Prestwich, Manchester M258WD England
Phone: 061-773 8514

TABLE I (Accuracy \pm 30%)

Sample #	Coating Date	Plasma (μ Hg) P	Back Front (μ m) Al Film Thickness	(in) Elect. Spacing	Shield Used Hearth? Cathode?	(mg) Δ m	Film Material	Front μ " r.m.s. Surf. Smoothness		KV (ma)	Time (min)	Power (kW)	Comments
								Subst.	Film				
3	6 June	5	8 ± 2	3-4	Not Cathode		Al Marz Grade	100	100	.3 (4)	-	-	Numbers listed are estimates
4	12 June		2.5	5-6	Hearth & Cathode	-.1	Brass Al	100	100	1 (200)	-	-	
11	25 June	24	$\frac{0}{.5}$.8	Cathode Top Collar Hearth	.3	Marz Al	5	5	1.5	17 (10- 35)	.5 to 2	
14	31 July	30	8	2	None	20	Marz Al	5	40	2 (20-5)	20		Resistance evap.
12	1 Aug	30	12	2	None	14	Marz Al	5	30	3.5 (10- 20)	60		Resistance evap.

ion coated with Al during trial runs of the V.T.A. 7375 system. Samples #3 and #4 had surface roughness of $\sim 100\mu$ while #11 (12, 13 and 14) were polished to a $\sim 5\mu$ smooth finish on one side. Film thickness on samples 3, 4, and 11 were ~ 8 , 2.5 and $.5\mu$ respectively. Some direct evaporation was possible on #3 where the recorded pressure was in the 5μ range.

Sample #3 was mounted on a bare unshielded 1/2" rod. Sample #4 used the 5" cathode and grounded shield but was spaced away by about 3" by the brass interface. Sample #11 was flat mounted against the cooled cathode (to avoid heat induced blackening) and had an extra 2" of grounding skirt. Samples #4 and #11 also had a grounding cup shield around the hearth. Sample #11 had further shielding in the form of a brim ring around the cup top.

b. While the V.T.A. e-beam evaporation system was being repaired, the Denton resistance heated filaments were used to ion plate 8μ and 12μ Al coatings on samples #14 and #12, respectively. The unshielded sample cathode was screw mounted to a vertical 1/4" steel rod and positioned 2" above the filament.

c. Film surface roughness is several times greater than substrate roughness on samples #12 and #14.

d. The Al was M.R.C. Marz grade while the crucibles were Carbon EB-9 Union 76 (POCO).

F.5. Discussion

a. Samples #3, #4, and #11 were coated under exploratory conditions with the e-beam power being gradually increased. The cathode voltage was adjusted to prevent arcing in the chamber. The above variations plus the fact that only three films were coated suggests that only tentative conclusions should be drawn. The worst conditions (lack of shielding and close substrate source distance) gave the greatest film thickness.

b. Holliday¹¹ & ¹² and this author observed that their ion plated films were rather dull. Jones, Griffith, and Williams²³, ion plating on plastic, also found reflectivities reduced relative to vacuum coating, by about 2.5%, 10%, 18% for Au, Al, and Cu, respectively. Could the trace contaminants always present²² from e-beam evaporation hearth liners in vacuum plating Al account for the decrease reflectivity? Perhaps ion impact induced surface roughness makes the surface less specular (smooth). If one found that plasma cleaning on a 100% reflecting mirror surface causes loss of reflectivity in a lab experiment then the dullness in our films could be explained. The μ sized column film structure is common in ion plating and thus could easily account for the depressed reflectivities.

c. The trouble with the V.T.A. system may be inefficiency due to lack of optimum adjustment and that troubles arise as increased high powers are used to compensate for the inefficiency. The Al was easily melted with .6 kW power (only 12% of the rated full power of 5 kW). The highest power recorded was 2.2 kW which was held for the last 5 minutes before the system blinked out 25 June. The Al melt temperature was considerably above the melting point.

G.1. Suggested improvements which may be incorporated into the V.T.A. Model 7375 electron beam evaporation ion plating system are listed:

a. Baffle plate improvements

During the past two years previous prospective V.T.A. system users¹² designed a complex aluminum baffle plate using a small orifice which does an excellent job of providing the correct macroscope gauge pressure, both for the plasma above and for the filament chamber below the plate. The following further improvements are recommended for this baffle to lower further the filament pressure.

(1) Remachine this baffle totally from stainless steel instead of aluminum to match more closely the refractory materials (tungsten and molybdenum) in the Sloan e-beam gun. The present Al readily melts and evaporates around the Al orifice casting a direct shadow in the lower high vacuum region onto the gun and onto the high voltage electrodes causing shorts and arcs.

(2) Place stainless steel shim stock over the present Al baffle in the heat sensitive areas. This has been done.

(3) Bore a 1" diameter hole in the Al plate centered on the orifice hole. Machine a set of 7 stainless steel seats with orifice sizes 1/8, 3/16, 1/4, 5/16, 3/8, 7/16, 1/2. The smaller sizes would suffice for material which requires no sweep and would provide a higher vacuum around the filament.

(4) Use only a very small quantity of vacuum grease on the O-rings. Use only enough to give a wet look but not enough for the grease to be visible.

(5) A rectangular orifice may enhance the electron beam current while maximizing the gas flow impedance.

(6) The above rectangles could be continuously adjustable slits. If made of a bimetallic strip the adjustment would be automatic.

(7) Add extra fins and shields around the e-beam path to shield the exposed insulators from unintentional e-beam induced evaporation from around the orifice.

(8) The orifice could be placed with its axis horizontal at the top of the beam's arch, thus causing the gas to jet 180° away from the filament and anode, instead of the present 90° jet. A secondary advantage would be that more of the beam's path would be in the higher vacuum. A disadvantage would be increased coating around the orifice.

(9) Again the orifice is only an ~1/2 inch from the filament while the ion gauge is about 10" away. A gauge should be placed within an inch of the orifice with its short tube pointed toward the orifice jet to get the best indication of the filament pressure.

b. Improving other parts of the V.T.A. system other than the baffle plate:

(1) Place low melting point low vapor pressure materials on parts of the vacuum system for overheating indicators for sensitive components such as magnetic coils which must operate under 400 °C.

(2) Purchase at least all the parts listed on the suggested spare parts inventory (cost about \$2000).

(3) Add a shutter to shield initial volatile out-gassing impurities from the substrate during the high vacuum soak of the crucible and source.

(4) In view of the manpower shortage connect a 4-pen recorder to the ion plating system to monitor the cathode voltage and current and the electron beam (emission) current. The fourth pen could monitor such quantities as sample temperature by thermocouple or magnet current.

(5) Dr. Arthur Nunes suggested using a focusing current ring (or an electrostatic ring) around the orifice to contact the beam to a finer thread to minimize orifice material evaporation.

(6) Apply the magnetic field sweep above the orifice so as to keep the orifice size as small as possible.

(7) Use a double hearth liner crucible stack with the inner crucible cut off to provide a lower target which should, because of greater insulation, rise to a higher temperature with less power to provide a higher plating rate.

(8) Entering the Ar leak from the bottom (instead of from the top) would cause some upflow which could enhance the transport of metal to the substrate increasing the deposition rate.

(9) Even with the cathode in its lowest position it is still ~ 6" from the hearth. Extending the substrate to the 2" to 3" range would produce a good enhancement in the deposition rate at no loss of throwing power according to Prof. Holliday's experimental data.¹¹ Suggested optimum distance for best ion plating is about six inches between substrate and source so some loss of adhesion may be expected.

(10) Switch the two baffle O-rings to Viton to assure better high temperature performance.

(11) The V.T.A. 15 KV 500 ma cathode supply was found inoperative due to both a blown fuse and a blown milliampmeter. An 8 amp fuse was found necessary to withstand the current surges during critical arcing while providing some meter protection. This recently added fuse holder should be upgraded to a 15 KV rating.

G.2. Major additions to the V.T.A. Model 7375 ion plating system.

a. Acquire a "Balzers" e-beam gun and hearth which measures the positive ion current which is "proportional to the ion plating rate." Geometry dependent coating rates with ion plating are difficult to determine compared to vacuum evaporation.

b. Purchase an identical Sloan e-beam evaporation unit so that one unit may be refurbished while the other unit is in operation.

c. Use boronnitride-titanium diboride ribbon evaporator boats with resistance heating of Al as done by Walley and Cross¹⁴ in the article "An Aluminum Evaporation Source for Ion Plating." The electrodes must be water cooled and conducting graphite paper provides better electrical contact to the electrodes. This BN-TB₂ is cited in the Handbook of Thin Film Technology, page 1-47, as a crucible material sold under the name (HDA Composite Ceramic) by Union Carbide, New York, NY.

G.3. Conclusions

The V.T.A. system shows promise in that it operated for about a month with no major problems. The V.T.A. low coating rate will probably improve when the excess electrostatic shielding is reduced. However for quick ion plating of Al, Denton resistance evaporation unit is much more reliable.

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1980

NASA/ASEE SUMMER FACULTY RESEARCH FELLOWSHIP PROGRAM

MARSHALL SPACE FLIGHT CENTER
THE UNIVERSITY OF ALABAMA

STATISTICAL WIND PROFILE GUST MODEL

Prepared by:	Deva C. Doss, Ph.D.
Academic Rank:	Professor
University and Department:	The University of Alabama in Huntsville Department of Mathematics
NASA/MSFC: (Laboratory) (Division)	Space Sciences Atmospheric Sciences
MSFC Counterpart:	O. E. Smith
Date:	August 1, 1980
Contract No. :	NGT-01-002-099 (University of Alabama)

STATISTICAL WIND PROFILE GUST MODEL

BY

Deva C. Doss, Ph. D.
Professor of Mathematics
The University of Alabama in Huntsville
Huntsville, Alabama

ABSTRACT

The purpose of this study is to develop a statistical wind profile gust model for the Space Transportation Operations and Trade Studies by using 1800 Jimsphere wind profile data collected at Cape Kennedy during 1965-1972. Wind profiles from the surface to 20 km in component form, i. e., zonal and meridional are processed through the digital filters of different wave length ranges based on the Martin-Graham cosine rolloff model. The residuals obtained from the filtering processes form the data base for the statistical analysis.

For each wind component the gust and gust length at a specified reference altitude in a residual profile are defined. A two-parameter gamma probability marginal distribution seems to fit well the component gust amplitude, and the gust length when redefined. The problem of finding an appropriate bivariate joint distribution of the gust amplitude and length remains to be solved.

The probability distribution of the modulus of the gust amplitudes has been derived under the assumption that they are independently distributed as gamma variates. It seems to fit the observed data.

INTRODUCTION

Accurate wind profile measurements from the surface of the earth to 20 km are made possible by the Jimsphere-radar system. This system consists of a 2 m diameter, constant-volume balloon and an AN/FPS'-16 or equivalent high precision radar which tracks the position of the balloon. The position data collected by the system are smoothed over a 25 m interval. The resulting data are differenced over 50 m intervals to produce wind profile data points.

This study utilizes 1800 wind profile data in component form collected at Cape Kennedy during 1965-1972. They are processed through the digital filters of different wave length ranges based on the Martin-Graham cosine rolloff model. The residuals obtained from the filtering processes form the data base for the statistical analysis (see Figure 1).

1.1 Definitions

Adelfang [1] in a recent report defines gust amplitude and length for a wind component in a residual profile at a specified reference altitude H_0 . For instance in Figure 2 the gust amplitude (gust) for the zonal wind component u' is defined as the maximum value of $|u'|$ in the vicinity of altitude H_0 with like sign to u' at H_0 . The gust length L_u is defined as the altitude difference of the zero crossings on either side of the gust, i. e.,

$$L = H_2 - H_1 \quad (1)$$

where H_1 = altitude of the first zero crossing for the downward scan,

H_2 = altitude of the first zero crossing for the upward scan.

Similarly, the gust amplitude and length are defined for the meridional gust component v' .

1.2 Purpose of the Study

One of the objectives of this study is to develop a theoretical probability gust model for each component as well as a joint model for both the components. Then the probability distribution of the modulus of the gust amplitudes can be derived.

GUST ANALYSIS

In statistical literature it is not uncommon to fit a gamma probability

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DATE 7/28/65 TIME 2259
90-2470M

90-6000M

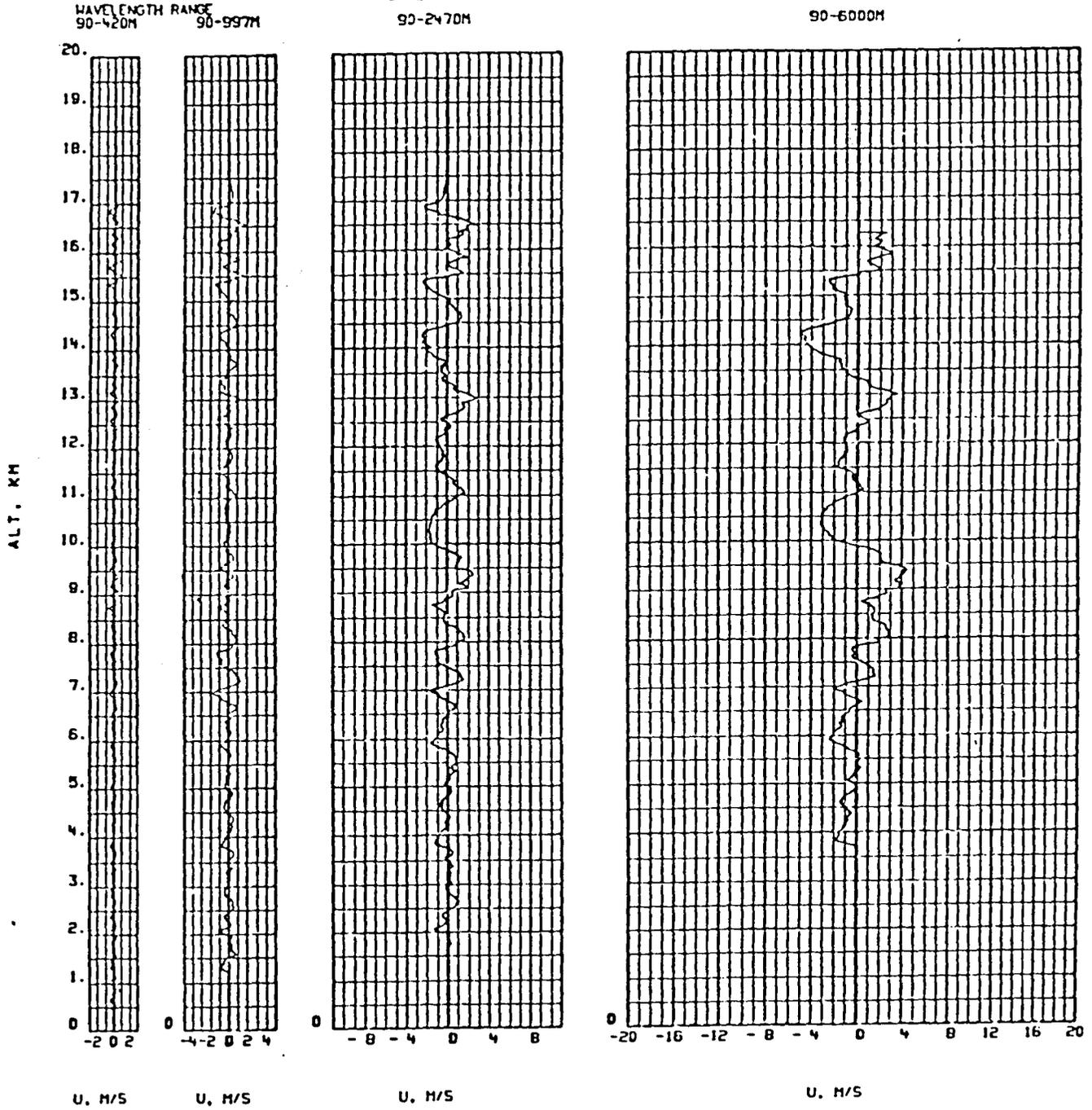
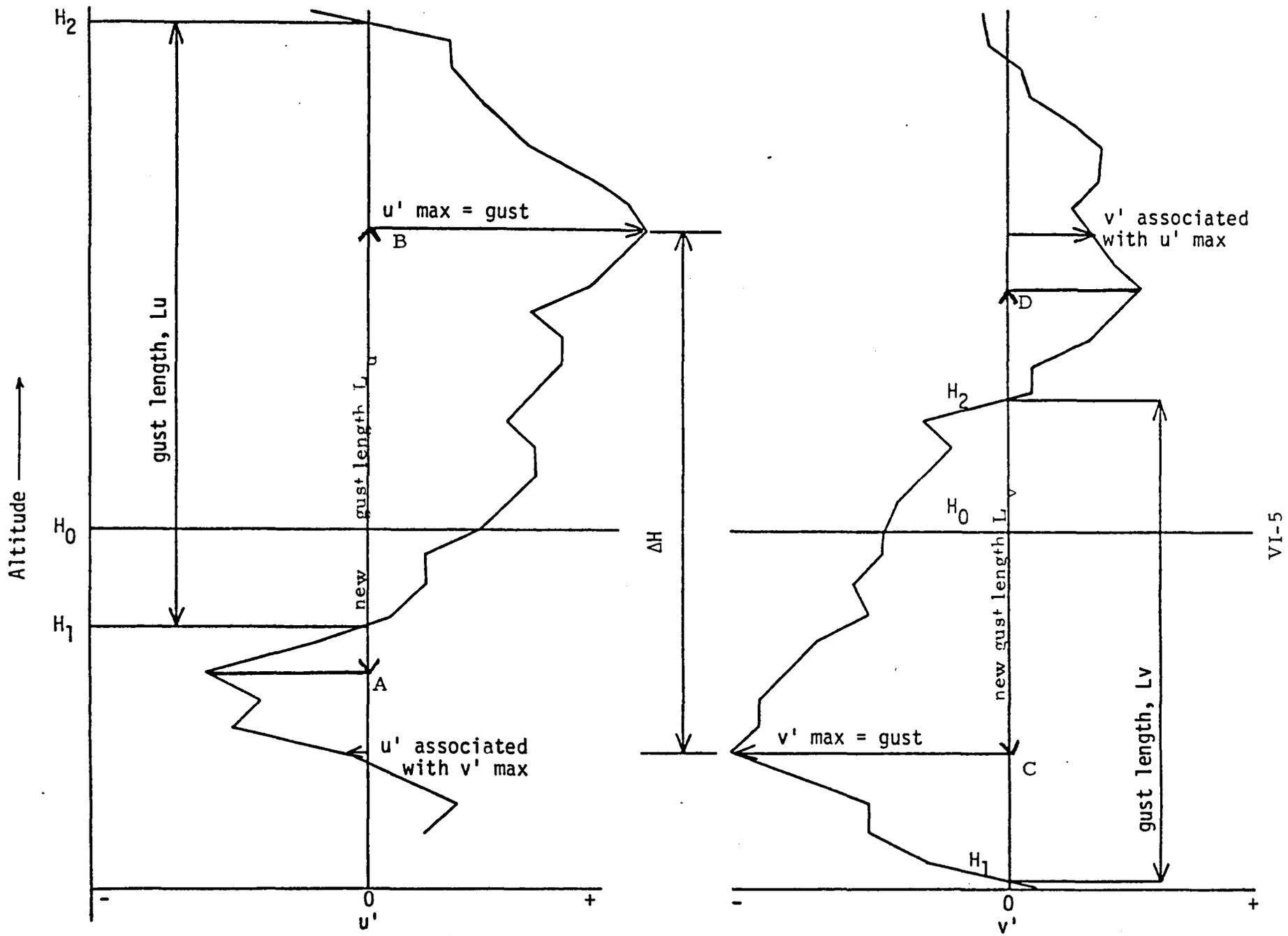


Figure 1. Cape Kennedy Residual Profiles



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Figure 2. Schematic Definition of Gust

model to a climatological measurement. However, when more than one random variable is involved, it is not yet clear whether or not a particular species of several multivariate gamma probability distributions is adequate to describe the behavior of random variables under consideration. This difficulty stems from the fact that any given univariate marginal distributions do not uniquely determine their joint probability distribution. If one marginal of a component of a random vector happens to be different from that of another component, e.g., gamma and beta, then the complexity of reconstructing their joint distribution increases.

2.1 Gust Amplitude

Adelfang [1] has shown that the gust amplitude of any wind component at a given reference altitude for filters of different wave length ranges seems to be distributed as a two-parameter gamma distribution and the fit seems to be good. A gamma probability density of a random variable x is here defined as

$$f(x) = \beta^\gamma x^{\gamma-1} \exp(-\beta x) / \Gamma(\gamma), \quad x \geq 0, \quad (2)$$

$$= 0 \quad \text{otherwise,}$$

where scale parameter β and shape parameter γ are positive and estimated by the maximum likelihood procedure as in Thom [3]

$$\hat{\beta} = \hat{\gamma} / \bar{x} \quad (3)$$

and

$$\hat{\gamma} = \{ 1 + (1 + 4A/3)^{1/2} \} / 4A, \quad (4)$$

$$A = \ln(\bar{x}) - \overline{\ln(x)}. \quad (5)$$

2.2 Gust Length

Although the gamma and truncated normal probability distributions seem to fit fairly the data on gust length, a pronounced bimodality appears to persist in the observed frequency distribution of the gust length. Therefore, there are two options to pursue:

1. To fit a mixture of two distributions
2. To redefine the gust length and hopefully avoid bimodality.

Since the first option involves computation of more parameters and analytical complexity, a redefinition of the gust length is in order. Instead of defining the gust length as the distance between two successive zero crossings, it can be defined as the length of the interval containing the reference altitude H_0 , whose end points form successive extrema of the wind profile. In Figure 2, the distance from A to B is the new gust length for u' and the distance from C to D is the new gust length for v' .

It is encouraging to find that newly defined gust length seems to fit a gamma probability distribution without exhibiting any bimodality. It is premature at present to accept this observation without further investigation.

2.3 Joint Distribution of Gust Amplitude and Length

Once it is assured that the gust length follows a gamma model a search for an appropriate bivariate gamma distribution should be made for the gust amplitude and length. Three bivariate gamma distributions are given in Mardia [2], p. 88. When the shape parameters of gamma marginals are almost equal, the first bivariate distribution involving a modified Bessel function may be considered. Because of the restriction, namely, $y \geq x$ imposed on the third distribution, it cannot be employed in fitting the gust amplitude and length.

2.4 The Modulus of the Gust Amplitude

Since it is found that the correlation between the gust amplitudes of the components u' and v' is less than 0.2, the maximum of modulus

$$R = \{ (u')^2 + (v')^2 \}^{1/2} \quad (6)$$

can be derived from the gamma marginal of the gust amplitudes under the assumption that they are independently distributed. In fact, the probability density of R is:

$$\frac{1}{4} \frac{\beta_1^{\gamma_1} \beta_2^{\gamma_2} R^{\gamma_1 + \gamma_2}}{\Gamma(\gamma_1) \Gamma(\gamma_2)} \sum_{n=0}^{\infty} \frac{(-\beta_1 R)^n}{n!} \sum_{m=0}^{\infty} \frac{(-\beta_2 R)^m}{m!} \frac{\Gamma(\frac{\gamma_1 + n}{2}) \Gamma(\frac{\gamma_2 + m}{2})}{\Gamma(\frac{\gamma_1 + \gamma_2 + n + m + 2}{2})} \quad (7)$$

where all parameters $\beta_1, \beta_2, \gamma_1, \gamma_2$ are positive. This has to be validated by the data.

2.5 Sum of Two Adjacent Amplitudes

Denote the amplitude of the zonal component at the reference altitude H_0 by $|u'|$ and the amplitude just above by $|u'_2|$. Then it has been found for a set of data that the correlation between them is in the vicinity of 0.5. However,

the observed data on their sum $|u'_1| + |u'_2|$ seems to fit a gamma distribution. (Similar observation may be true also for the other component.) It raises a theoretical question: When is the sum of two correlated gamma random variates distributed as a gamma?

CONCLUSIONS AND RECOMMENDATIONS

This report presents several unanswered questions to which answers have to be sought. It is, with reasonable assurance, seen that the gust amplitude follows a two-parameter gamma distribution the probability model of the newly defined gust length seems promisingly to be a gamma distribution.

The joint distribution of the gust amplitude and length, and the distribution of the modulus of gust amplitudes need to be explored. In fact finding the joint distribution of the gust amplitudes and length, of both the components should be the ultimate goal. One realizes that there are many loopholes in this investigation, but it is hoped that by this time next year the statistical model for the gust is resolved.

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1980

NASA/ASEE SUMMER FACULTY RESEARCH FELLOWSHIP PROGRAM

MARSHALL SPACE FLIGHT CENTER
THE UNIVERSITY OF ALABAMA

UPGRADING THE FREE FLYING RENDEZVOUS
AND DOCKING SIMULATOR AND THE ORBITAL
SERVICER SYSTEM

Prepared By:	Robert M. Eastman, Ph.D., P.E
Academic Rank:	Professor
University and Department	University of Missouri - Columbia Department of Industrial Engineering
NASA/MSFC: (Laboratory) (Division) (Branch)	Electronics and Control Guidance, Control & Instrumentation Control Mechanisms
MSFC Counterpart:	Kenneth R. Fernandez
Date:	July 25, 1980
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UPGRADING THE FREE FLYING RENDEZVOUS AND DOCKING SIMULATOR AND THE ORBITAL SERVICER SYSTEM

BY

Robert M. Eastman, PH.D., P.E. Professor of Industrial
Engineering University of Missouri Columbia, Missouri

ABSTRACT

The objective of the research was to make recommendations for upgrading two teleoperator/robotics test and simulation systems based upon a review of latest technology advances in the involved disciplines. The Free Flying Rendezvous and Docking Simulator has two vehicles, the Free Flying Mobility Unit and the Free Flying Target Assembly, operating on air pads on the air bearing epoxy flat floor. The Orbital Servicer System has a six degrees-of-freedom manipulator arm for transferring modules between the Orbital Servicer and a prepared satellite.

Based on the research, a new second-generation Free Flying Mobility Unit is recommended. It will add a sixth degree-of-freedom and incorporate other improvements which will greatly expand the Center's capability to perform evaluation tests and demonstrations of advanced systems concepts for rendezvous and docking in support of the Teleoperator Maneuvering System (TMS) Program. Current plans for the TMS call for a technology readiness in 1985 for placement and retrieval of a satellite/payload in a predetermined orbit with the TMS moving from and returning to the Shuttle Orbiter. The second generation Free Flying Mobility Unit will incorporate TMS design features which will enable it to perform realistic simulations for evaluating rendezvous concepts, ranging sensors, lighting and video systems, and docking mechanisms. At a later time it will incorporate advanced technology features for future TMS missions and will be invaluable in defining the requirements and evaluating promising concepts for automated rendezvous and docking.

The Orbital Servicer System provides the capability for testing and demonstrating concepts for on - orbit servicing of compatibly designed satellites/payloads. The TMS will be the transporting vehicle for the servicer. The manipulator arm of the Orbital Servicer System is presently computer controlled in the trajectory portion of the module transfer operation. The ultimate objective is to fully automate its operation having the manipulator arm move from its stowed position, latch on to a mock-up faulty module, remove the module and stow it in the base of the servicer, latch on to a replacement module, place it in the satellite, and having the manipulator arm return to its stowed position. This will require additional capabilities in several fields basic to robotics research. These include sensors, artificial intelligence, image analysis, communications, computer programming, pattern recognition, kinematics and manipulator design. It is recommended that the Electronics and Control Laboratory move to acquire the basic competencies in robotics necessary to achieve the major objective of full automation.

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Kenneth R. Fernandez as my NASA MSFC counterpart helped with explanation and demonstration of the robotics facilities and test programs. John B. E. Chase was generous with his editorial help, arrangements, and explanation of the factors and issues involved. Ed Guerin and Dave Henderson (Essex Corporation) explained the Free Flying Rendezvous and Docking Simulator and arranged for me to observe the tests. Don Scott and Keith Clark demonstrated the Orbital Servicer System. Charles Cornelius provided logistic support and organizational enlightenment. F. Brooks Moore and Eugene Smith furnished the management support without which no research can prosper. Other members of the Electronics and Control Laboratory helped with information and support.

Mary F. Smith typed and retyped the manuscript and helped in numerous other ways. Eloise Meyer, Bettye Morey and Gayle Dodson were gracious and helpful. Marion Kent's advice was most useful and appreciated.

Dr. Robert Barfield, Project Director, handled the contract substance assisted by Barbara Skaggs. The University of Alabama in Huntsville faculty were gracious hosts socially and professionally.

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INTRODUCTION

The Free Flying Rendezvous and Docking Simulator consists of the Free Flying Mobility Unit and the Free Flying Target Assembly both of which operate on the air bearing epoxy flat floor. The two vehicles are mounted on air pads which provide frictionless motion in one plane for simulating translational movement in space. Simulated propulsion is provided by compressed air jets under the control of the remotely located operator. The simulator is illustrated in Figures 1, 2, and 3 and described in reference (1).

The simulator is an engineering tool used to define, test, evaluate and demonstrate the rendezvous and docking techniques and procedures, and the components and systems designs needed for rendezvous and docking a remotely controlled teleoperator vehicle with a satellite payload. The data recorded include air consumed (measure of fuel consumption), time to dock, docking success, lighting patterns and camera positions. The evaluation and test data influence the selection of the most promising concepts and designs and the refinement of them for potential use in future projects.

The Orbital Servicer System is an engineering model of an on-orbit maintenance system. It contains full-scale typical satellite replacement modules which are exchanged by the six-degree-of-freedom manipulator arm. It also includes the interfacing control system for the manipulator arm. The Servicer System model is used to test, evaluate, and demonstrate the on-orbit maintenance concept and to define design features that would enhance its performance. At present (1980), the operation is partially automated. A computer program controls the teleoperator arm trajectory movement until the module is close to its destination. Then control is transferred to the human operator who completes the module transfer. He operates from a remote location using video cameras and manual controls. The servicer is illustrated in Figure 4 and described in references (2) and (3).

One of the key questions concerning the Orbital Servicer is whether the teleoperator arm can be fully automated to eliminate the human operator (man-in-the-loop). There are advantages and disadvantages to each alternative and strong advocates on each side. A major purpose of the Orbital Servicer System is testing the feasibility of the fully-automated transfer.

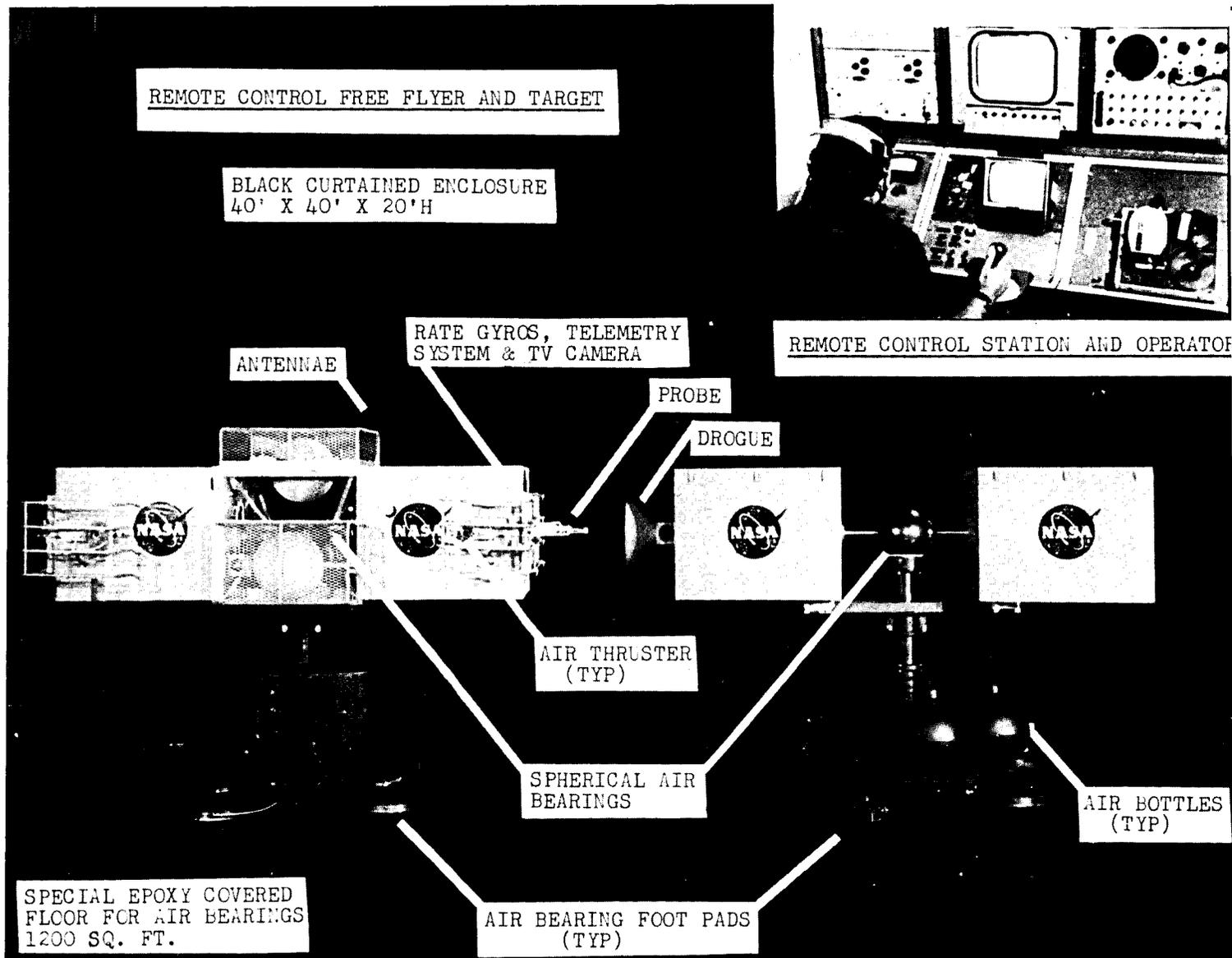


FIGURE 1. FREE FLYING RENDEZVOUS AND DOCKING SIMULATOR

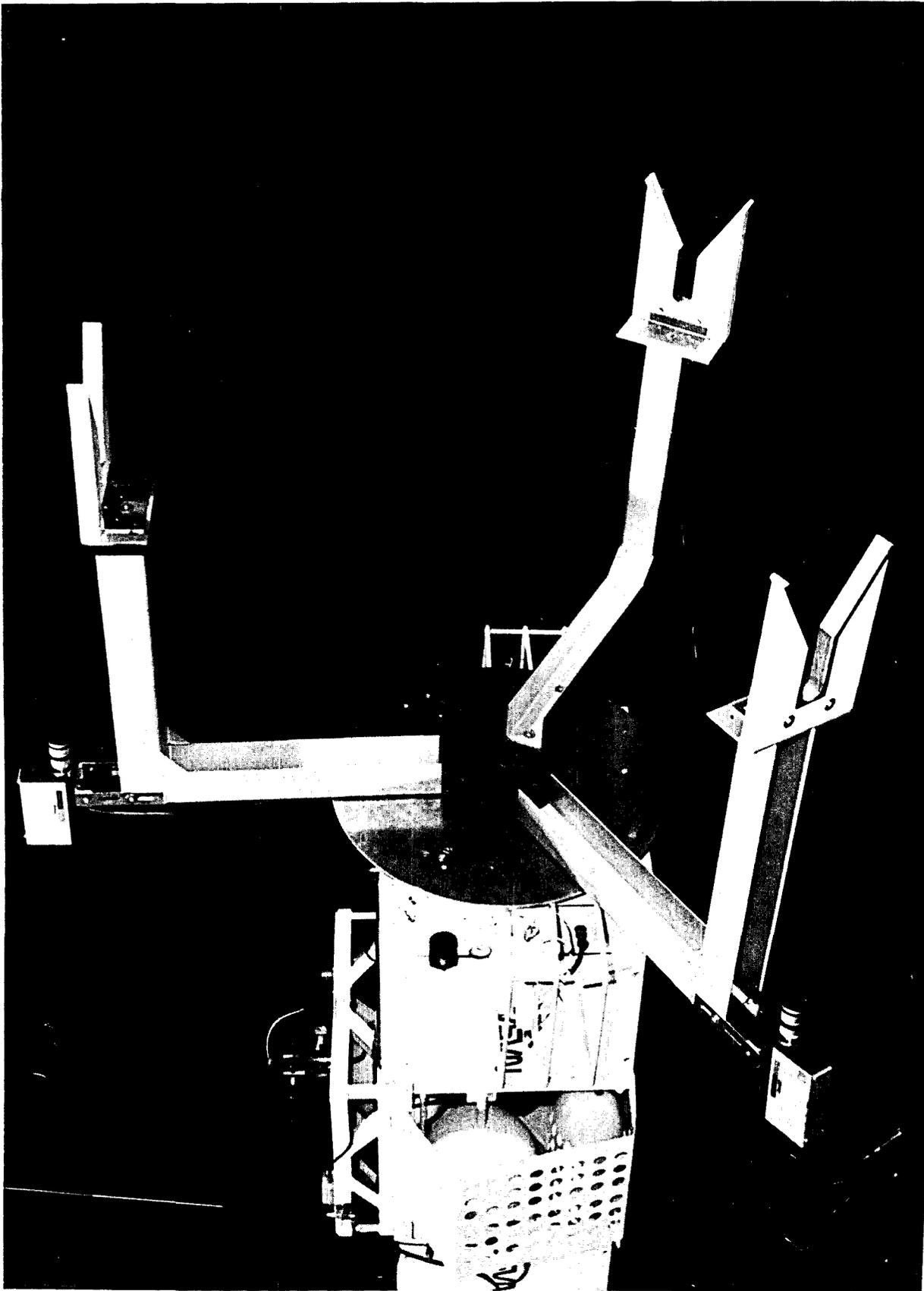


FIGURE 2. FREE FLYING MOBILITY UNIT

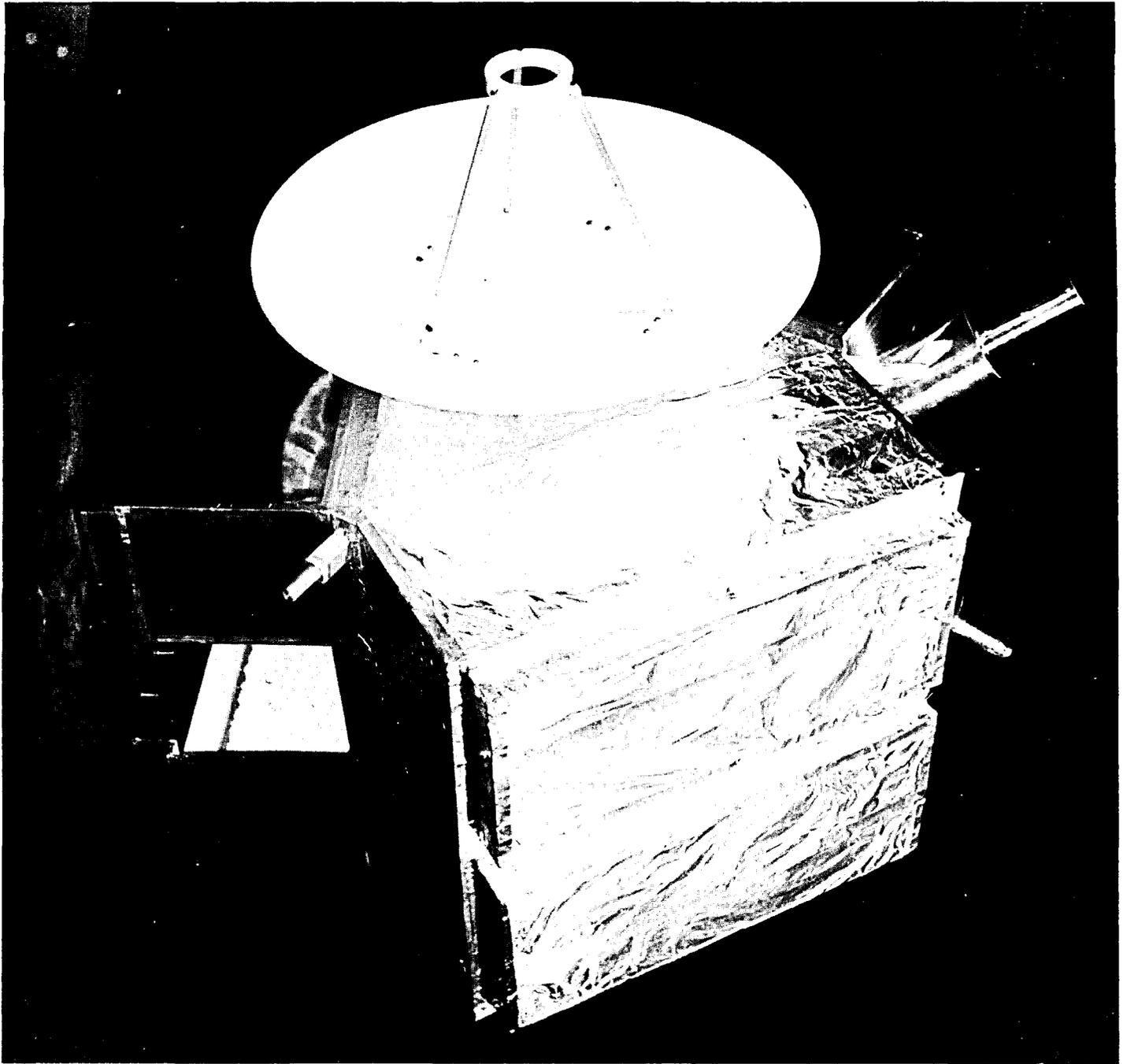


FIGURE 3. FREE FLYING TARGET ASSEMBLY

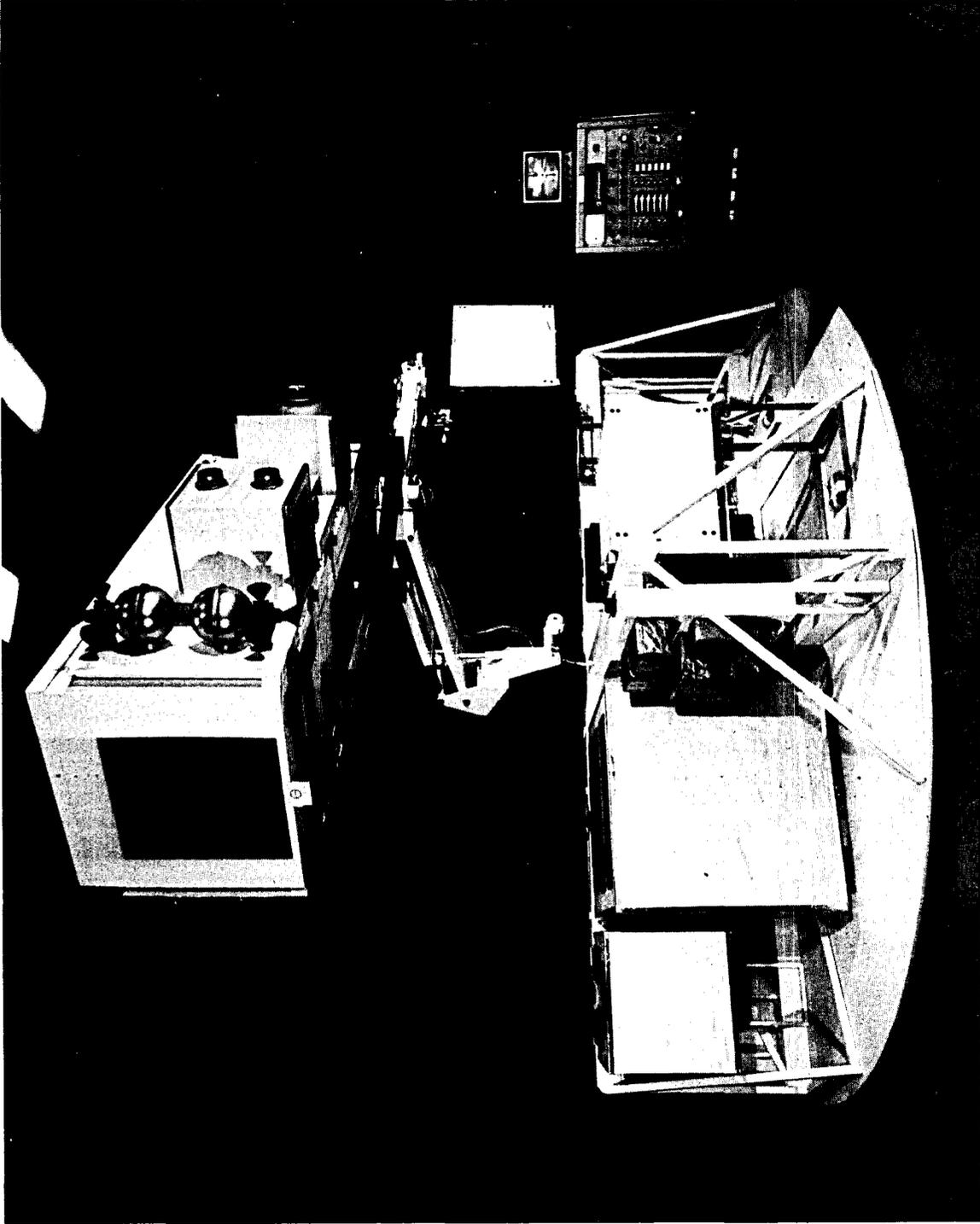


FIGURE 4. ORBITAL SERVICER SYSTEM

OBJECTIVES

The first research objective is to recommend design improvement for the Free Flying Rendezvous and Docking Simulator based upon a review of latest developments in associated disciplines. The subsidiary objectives are:

1. A 6 DOF Free Flying Mobility Unit which could be either a modification of the existing 5 DOF vehicle or a new vehicle.
2. Automatic docking capability.
3. Alleviation or elimination of some current limitations including:
 - (a) Reliability and maintenance problems with existing equipment.
 - (b) Physical balance of the current equipment, i. e., stability in pitch and roll.

The second research objective is to recommend design improvements in Orbital Servicer System based on review of latest developments in appropriate disciplines. Subsidiary objectives are:

1. Automated fast servicing of a class of satellites by robotic arm from the Orbital Servicer vehicle.
2. To review the proposed new electrical self-aligning connector design.
3. To suggest improved methods of rack positioning.

METHOD OF INVESTIGATION

The following methods of investigation were used:

1. Literature search.
2. Material and information from the NSF Robotics Research Workshop held April 15-17, 1980, at the University of Rhode Island and from personnel at the McDonnell Aircraft Company (part of McDonnell Douglas Corporation in St. Louis, MO).
3. Observation of the Free Flying Rendezvous and Docking Simulator and the Orbital Servicer System.

4. Discussions with MSFC personnel.
5. Sketches and calculations.
6. Review of previous and associated work.

SYSTEMS INTEGRATION

NASA Headquarters has assigned MSFC the mission of "Systems Integration" or the co-ordination and interfacing of teleoperator/robotics sub-systems and components into an effective total system to support future NASA programs. The term "System Integration" also connotes optimizing the effectiveness of the system while using scarce resources efficiently.

The two lab facilities, the Free Flying Rendezvous and Docking Simulator and the Orbital Servicer System, are essential for evaluating and demonstrating the integration of the functions and the system performance in a simulated space mission. The full-scale dimension provides the experimental setting necessary for accurate test and evaluation. Scale models can be helpful but there are too many possibilities for discrepancies between actual mission systems and part-scale simulators to permit complete reliance on the part-scale model. Computer modeling and simulation can be efficient and time-saving; however, their results require full-scale physical verification because of the possibilities of conceptual errors and data omissions and inadequacies.

BASIC ROBOTICS RESEARCH OVERVIEW

This section will be devoted to a discussion, necessarily brief, of several fields of basic robotics research pertinent to the main objectives of full automation of the Orbital Servicer Teleoperator System and to upgrading the Free Flying Rendezvous and Docking Simulator.

The following research areas are fundamental to advanced teleoperator and robotics development:

1. Artificial intelligence
2. Sensors
3. Kinematics
4. Control
5. Communications

6. Manipulator design
7. Image analysis
8. Locomotion

ARTIFICIAL INTELLIGENCE

Artificial intelligence is the branch of computer science which studies the way machines can replicate human actions requiring intelligence. It involves receiving and evaluating environmental data and making a decision based upon the data and a set of decision criteria. At an advanced level, the machine can learn from experience and adjust its behavior accordingly. Artificial intelligence is necessary for any teleoperator or robot which has an operation which is not completely and undeviatingly predetermined.

Systems have been developed which will perform at least one but never all of the following: (4)

- (1) Plans can be generated at multiple levels of detail.
- (2) Plans can be viewed as partially ordered sequences of actions with respect to time.
- (3) Each action is expected to produce a single state change characterized by a single primary effect.
- (4) A plan is not generated at all unless the planner determines that it will be totally successful in meeting all specified goals.
- (5) Simple plans requiring information gathering can be generated.
- (6) Unsophisticated techniques for dynamic repair of unsuccessful plans during execution have been developed.
- (7) Plans can be used to control robot devices with simple use of sensory feedback and systematic replanning.

Major research problem areas in intelligent robots are: (4)

1. Planning for parallel execution
2. Planning for information gathering
3. Planning for planning
4. Learning: Using a plan data base
5. Interactive planning
6. Dynamic plan repair
7. Distributed robotics (i. e., disbots)
 - a. Hierarchal planning

- b. Control
- c. Basis
- d. Communication
- e. Plan execution

The central problem of robotic intelligence in the real world is that a robot may have a large number of alternative actions, but the user does not want to be burdened with specifying them and the computer is too busy to compute them anyhow (5). -----"Intelligence and decision making" usually mean the ability of a robot to sense some aspect of its environment and take alternative courses of action.

SENSORS

A sensor is a device which collects data from the environment for transmission to the robotics command system. An important research issue in robotics is sensory control in which the sensor is the first link in the chain which finishes with the completed robot action. Fast and accurate sensing is essential to reaching the goal of fully automated space robotics (6).

Among the research areas to be addressed are:

1. More capable visual processing
2. Faster visual processing
3. Better dynamic models of arms
4. Tactile and force sensors with desirable characteristics
5. Better understanding of the role of compliance

Visual sensing is important in space applications. Some needed improvements are (7):

1. Higher resolution, dimensional density and precision
2. Improved pixel (i. e., picture element) quality
3. Color discrimination
4. Better lenses
5. Better cameras
6. Increased robustness
7. Controlled illumination
8. Software development
9. Range and proximity sensing

Sensor types which have negligible usefulness for space functions are tactile, force, acoustic and temperature.

Radar as a sensor has advantages and disadvantages. The distances involved may be too short to permit radar to function properly. Even more difficult is the problem of interfering reflections from structural components of the system. These reflections will prevent the system from receiving clear and accurate readings on the position of any system component.

However, some current and research in radar may facilitate using radar in fully automated orbital servicer. It has the capability of rapid location of an object and determination of its velocity and direction of movement provided the reflection problem can be solved.

Video cameras produce a good picture for the human operator. The major problem is automated image analysis of the picture. (Image analysis is the conversion of visual pictures such as video to a set of impulses which can be processed and interpreted by a digital computer). The human operator is much better at analysing the video picture and taking appropriate action than is an automated system.

Lasers may be the best sensors for space robotics applications. Much additional research is needed. Light-beam sensors (photoelectric cells) can not produce the detailed image needed. Acoustic sensors lack the medium to transmit sound wave. Contact sensors would require delicate, easily-damaged antennas and would probably be unable to gather enough environmental data to be useful.

Sensors present several robotic control problems such as data filtering and smoothing, multidirectional sensors and data acquisition. Control structures, particularly hierarchical, and control logic need further work. The man-machine interface requires additional development (12).

KINEMATICS

Kinematics and mechanical design refer to the static and dynamic relationships of the physical components of the teleoperator manipulator system. Examples are mass, velocity, momentum, strength and deflection. Correct handling these factors is vital to successful teleoperator and robot operation. Several important research areas remain.

Kinematic performance can be either time-based (velocity and acceleration) or geometric-based (space and orientation). Further work is needed in both areas (8). This effort should get away from position-by-position analysis to a broader knowledge which eliminates the need for the detailed position approach. A general solution to the optimal number of degrees-of-freedom has yet to be resolved. More DOF produce greater flexibility and versatility; fewer DOF produce a system easier to design and control, cheaper to build and generally more reliable.

Some robotic configurations are more efficient than others. However, the knowledge about these configurations is empirical rather than theoretical. Among the research questions to be resolved are:

1. The optimal manipulator configuration for flexibility, dexterity and reliability.
2. Actuator placement and control.
3. Basic design theory for fluid and electrical actuators.
4. Increasing the number of design parameters which can be rationally chosen given the present state of the art.
5. Elimination or minimization of elastic vibration due to system resonance.
6. Coping with large inertial forces at high speeds.

Another practical design problem is that robot and teleoperator performance fall short of theoretical optimum. Some causes are mechanical interference by structural members, limited rotational range and the mass of components particularly the motors (8).

Speed of operation of robots is vitally important to industrial applications, but less so to space applications. High-speed dynamic effects become critical yet little is known definitively about these effects. Present manipulator configuration design and trajectory are not the best for high speed operation. More needs to be done (9). Three difficulties with high-speed operation are:

1. Non-linear dynamic effects
2. Structural flexibility
3. Discrete time increments resulting from micro-computer controls produce undesirable effects.

Some modifications of manipulator design may be necessary to make the design compatible with automated operation. Present plans are based on the assumption that full automation can be achieved by rewriting the control computer program. This may or may not be a valid assumption. At the very least, the manipulator design should be reviewed for compatibility with full automated control.

CONTROL

Control theory is the branch of engineering and science which deals with monitoring and direction of the teleoperator to achieve the desired result. Control theory has two facets:

1. Systems study and modeling and (2) measuring (sensing) the states of the system. System and task modeling are essential to advanced robot control development. Modeling can be defined as quantitative descriptions of the system and tasks in measurable, calculable and controllable terms.

Among the research issues are sensitivity studies of the model, robot control in feedback schemes and multi-unit robotic systems (10).

Four assumptions of robotics control theory are:

1. Robots are complex articulated mechanical devices designed to perform operations normally requiring human skills.
2. Robots are equipped with sensors capable of measuring internal and external states.
3. Robots are equipped with computers.
4. Humans can communicate with robots.

Paul discusses control applied to robotic operations in space. He postulates 6 DOF with either velocity or force but not both specified along each DOF. He also discusses kinematic control and Jacobian control. Each method has advantages (11).

Graupe and Saridis hold that technological advances of the last fifty years depended upon development of system theoretic methodologies (14). Intelligent control uses the digital computer along with advanced techniques of system theory to produce a unified engineering approach. Based on this, control systems for robotic tasks and systems have been developed for prosthesis and orthotic devices and for general purpose robots. They propose a hierarchally intelligent control approach distributed according to "decreasing precision with increasing intelligence." The levels would be:

1. Organization
2. Co-ordination
3. Hardware Control

Present computer-based systems for robot control require the program to direct all movements and actions of the robot. A specially-designed computer language may be used although some programs are in a standard language such as Fortran. The program can include provision for receiving external stimuli and directing action based on the stimuli received. However, there is no provision for stimuli not in the program or for unprogrammed reactions no matter how desirable (15). Libraries of sub-routines can be developed since the programs can be independent of specific locations. The number and usefulness of sub-routines handling contingencies, errors and emergencies has increased.

The major deficiencies of the second-generation programmed robotic systems are (15):

1. The programs are difficult to debug and modify.
2. Achieving the final position with maximum accuracy is inefficient.
3. It is difficult to foresee all possible emergencies and contingencies. This is particularly important in the Space Shuttle - Orbiter Servicer system.

In the next (third) generation of computer robotics control systems, the robot will formulate an appropriate plan of action based on goal directions received from the operator. The robot will have greater artificial intelligence capabilities and much more data than it needs with the present computer program motion direction system. Research on this system is now underway (1980).

The elementary "Pick-and-place" robotic systems are relatively easy to design, program and implement. Many industrial robots are in this category. However, they are unable to handle effectively and reliably events and environments which deviate even slightly from the environment for which the program was designed. However, as the program is modified to handle environmental variations, the complexity of the problem domain and the resulting program increase many-fold (13).

COMMUNICATIONS

Communications will be a major robotics design problem. Data impulses from sensors must be transmitted to the control computer and instructions sent from the control center to the teleoperator components. Another major R&T issue is the computer languages and control algorithms needed for full automation. This is inextricably interconnected to the determination and optimization of the desired operating characteristics.

IMAGE ANALYSIS

The role of image analysis will be the identification and location of the components of the shuttle and the servicer. These components include:

1. Structural components of the Shuttle.
2. Structural components of the Servicer.
3. Alignment of the space vehicle (Servicer and satellite) during docking.
4. The modules at the beginning, during, and of the end of the transfer process.
5. Alignment of the rack which will receive the module.
6. Any component out of its regular position to the extent it might cause damage.
7. Any stray object in a position to cause a collision.

Image analysis is a particularly difficult field in robotics research and application. The processes by which a human operator's mind receives signals from his sensors, and evaluates, interprets and identifies the object seen, its location and its orientation are incredibly complex. Performing these processes automatically has been done only in simple situations with great difficulty.

Basic research in image analysis is going forward. Further progress is essential to the full automation of the Orbital Servicer Teleoperator.

LOCOMOTION

Locomotion is defined as movement of the robot over an irregular terrain surface. This contrasts with space travel in which the robot has no terrain contact. At a cursory glance, locomotion has little to do with the NASA Space Shuttle and Orbital Servicer missions. However, some locomotion research results have application to the simulation of rendezvous and docking operations. Furthermore, locomotion could become important to the MSFC robotics efforts if the mission is changed to include mobile robots for space applications.

THE FREE FLYING RENDEZVOUS AND DOCKING SIMULATOR

The present Free Flying Rendezvous and Docking Simulator uses aging equipment of limited capabilities. The two vehicles were assembled partly from surplus and scrounged components. They require considerable maintenance with increased down time and decreased capability. Further, the data output is limited in comparison to the data potential of a simulator with better equipment and instrumentation.

Second - Generation Free Flying Mobility Unit

A major research and policy question is "Should NASA build a new, second-generation Free Flying Mobility Unit or should it rebuild and improve the present vehicle?" In either case, the vehicle should incorporate a sixth degrees-of-freedom (6 DOF), balance stability in the pitch, roll and Z axes, increased reliability and more data acquisition instrumentation.

The general specifications for a second-generation Free Flying Mobility Unit are:

1. Incorporate six degrees-of-freedom.
2. Maintaining the vehicle balance during docking maneuvers.
3. Minimize weight without interfering with operating characteristics.
4. Be compatible with existing equipment and existing data.
5. Provide for installing alternative docking mechanisms.
6. Have on-board computer capabilities.
7. Provides for lighting and camera alternatives.
8. Have good maintainability and reliability characteristics.
9. Have the necessary communication and control capabilities.
10. Permit modification for automated docking maneuvers.

Advantages of a new second-generation unit are:

1. The desired 6th DOF can be incorporated into the design of the new vehicle more easily than into the existing one.
2. Some faults of the present design can be eliminated in the new vehicle design.
3. The reliability of a new vehicle would be much better.
4. Maintenance costs should be lower at least in the early years of its service life.
5. Vehicle balance can be incorporated into the design and operation.
6. Provision for alternative docking mechanisms can be installed.
7. Additional features such as improved lighting and vision can be incorporated.

Disadvantages of a new second-generation unit are:

1. The cost will be considerable. It may be more difficult to obtain the fund appropriation for a new vehicle than to rebuild the present unit with NASA personnel.
2. The procurement lead-time will probably be a minimum of three years between the date of the approval and the date the new vehicle goes into full service.
3. Unforeseen bugs and normal new model difficulties will arise.

Alternative Design Concepts for a Second-Generation Free Flying Mobility Unit

The following alternative design concepts were considered:

1. A vehicle suspended from a mobile overhead track-mounted vehicle.
2. A new floor mounted vehicle with its mass concentrated towards the center of gravity.
3. A floor-mounted vehicle with automated computer - controlled balancing.
4. A vehicle mounted on top of a two-way track system above the floor level. (Dr. Campbell's concept) This would have some advantages in stability and simpler motion design. However, the mounting might cause interference problems and possible damage (16).

Overhead Track-Mounted Vehicle

The concept of an overhead track vehicle with the docking mechanism suspended underneath is illustrated in Figure 5. The track and carrier will permit locating the mechanism in the X and Y directions. An actuator operated

CONCEPTUAL SKETCH OF OVERHEAD MOUNTED SECOND-GENERATION FREE FLYING MOBILITY UNIT

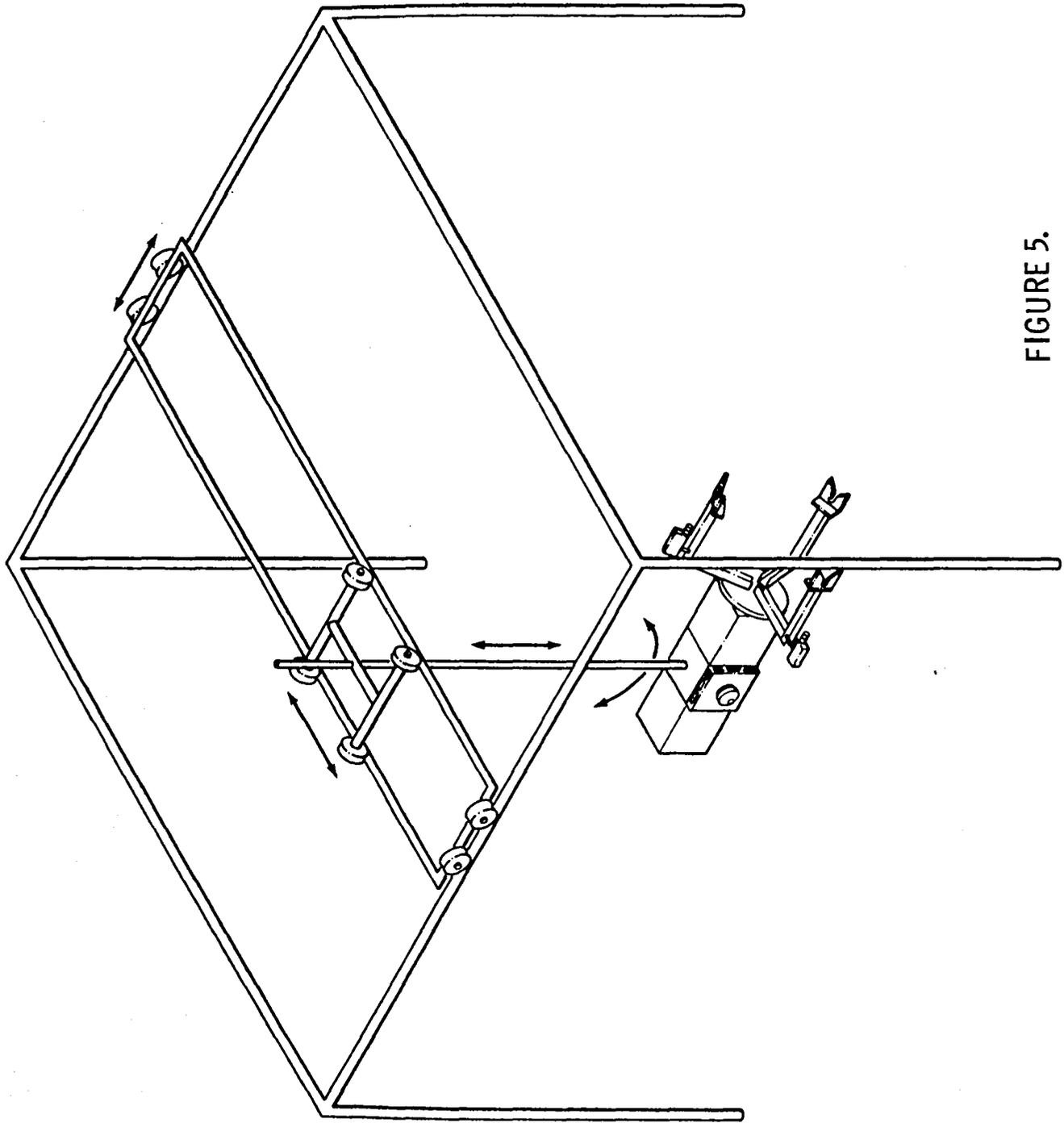


FIGURE 5.

column or beam would raise or lower the mechanism on the Z axis. Pitch yaw and roll would be provided by movement about the lower column end. Communications could be by direct electrical connection rather than by radio.

The advantages of suspending the docking vehicle from an overhead track are:

1. The sixth degree-of-freedom will be easier to incorporate.
2. Some of the technology is known and well-established.
3. Some components may be completely or partially "off-the-shelf." This will reduce the lead-time and possibilities of trouble with new technology.
4. There will be fewer interference problems such as those with the bases of the two present vehicles.

A major disadvantage is:

1. Maintaining vehicle balance through all DOF movements. This will require either a complex mechanical system or a sophisticated computer control program.

The research and design problems include:

1. Adequate building structure which may be difficult to fund.
2. Kinematics and deflection.
3. Mass and momentum.
4. The control system.

Floor-Mounted Free Flying Mobility Unit

A floor-mounted Free Flying Mobility Unit with computer-controlled balancing is a possibility. A sixth degree-of-freedom could be added. This would have the advantages of floor mounting while providing solutions to some major problems with the present model.

The major difficulty with this concept is the design of the control system and the computer programming. Both would be complex and expensive. Further, modifications to the design or latching mechanisms would require extensive redesign, reprogramming and modification.

A second-generation floor-mounted Free Flying Mobility Unit would be similar in some respects to the present model. The design concept is illustrated in Figure 6. A major change is the use of the three actuators to control the plane of the mechanism. This will provide more possibilities of movement and improve the units balance. Some of the other components can be relocated nearer the center of gravity to improve the balance. Other improvements can be incorporated.

CONCEPTUAL SKETCH FLOOR MOUNTED SECOND-GENERATION FREE FLYING MOBILITY UNIT

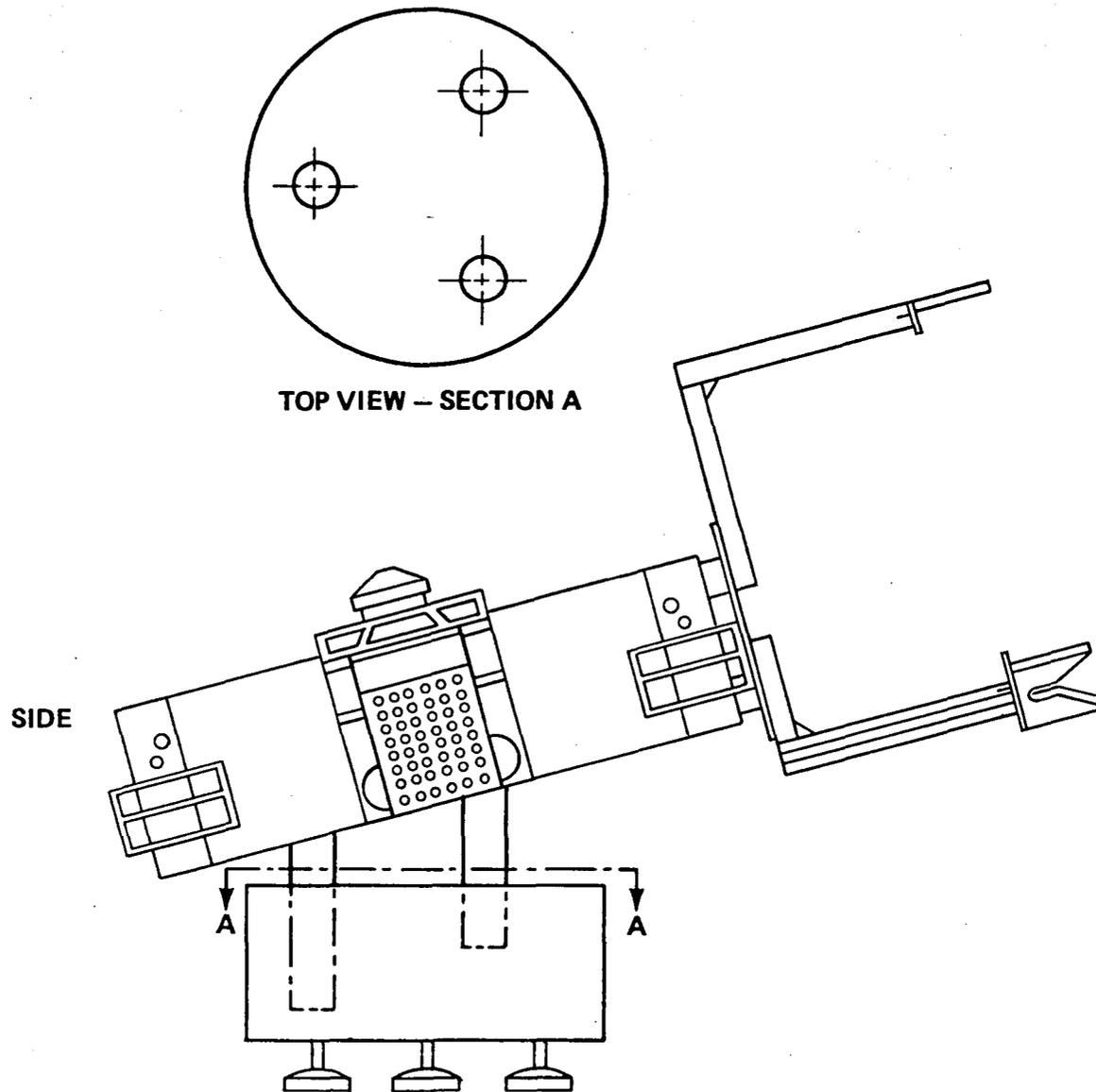


FIGURE 6.

Floor-Mounted Unit with Automated Computer-Controlled Balance

Computer balance control will have the following advantages:

1. Proper design of the balancing system will hold down the mass of the vehicle.
2. The computers can be located remotely from the vehicle. This will hold down the complexity and mass of the vehicle itself.
3. Although trained computer personnel are presently scarce, the number in training who will become available in the near future is increasing substantially.

Major difficulties with Mechanical Balance Control are:

1. Increase in the vehicle mass. This may cause difficulties in operation. The higher mass will increase the impact forces and may also aggravate some imbalance situations.
2. Personnel capable of designing, building and maintaining mechanical devices are scarce and will become almost unavailable within a few years.

Display System Definition

The present display system for the Free Flying Rendezvous and Docking Simulator needs a thorough review to determine whether it meets current requirements and is optimum for the test facility mission.

FULLY-AUTOMATED ORBITAL SERVICER SYSTEM

The objective of full automation of the Orbital Servicer will require a closely coordinated research and test effort involving several advanced robotics disciplines. The result of successful effort will be the elimination of the man-in-the-loop requirement of the present system.

Full automation of the Orbital Servicer Module transfer operation involves complex difficult research issues. It is not enough to extend the present computer controlled trajectory to cover the final movement of the module into the rack and to complete the latching operation. Functions performed by the man-in-loop must be incorporated into the automation. These functions include collision avoidance, path adjustment, coordination of the teleoperator movement and others.

In this discussion, the following assumptions are made:

1. The precision and accuracy, capabilities of the present Orbital Servicer robotic system are inadequate for efficient operation and collision avoidance in the full automation mode.
2. The major configuration features of the present Space Shuttle, Orbital Servicer and TMS will remain substantially unchanged.

Among the manipulator design features which should be reviewed are:

1. Mass of the components and the assembly.
2. Oscillation and deflection.
3. Placement of sensors, controls, motors and similar parts.
4. Collision and damage avoidance features.

Application of Basic Robotics Research to the Orbital Servicer System

In the section "Basic Robotics Research Overview," several basic knowledge areas are discussed. In this section, application to the Orbital Servicer System will be covered.

The first major difference between the present partially automated, partially manual mode and the fully automated mode is the need for substantially increased precision and accuracy in the final approach, contact and latching. Extending the present computer control program will not provide enough accuracy to complete the final mating and latching successfully. In order to attain this accuracy, knowledge from several basic fields must be added.

The first addition will be sensors to determine accurately the position of the module with respect to the rack. The predicted accuracy with the present system is about $\pm 1/2$ inch. This is insufficient to insure proper completion of the maneuver.

A necessary function of the sensors will be to detect deviations from the planned or expected movements and locations. The human operator can recognize situations which deviate from plan and take action to abort or adjust the teleoperator arm action. If there is no man-in-the-loop, the teleoperator must be designed to recognize the situation and to take appropriate action.

The first situation the sensors must recognize is misplacement, deviation or absence of the rack, module or other system component. The other is the presence of an obstruction, accidental or not with which the teleoperator would collide if the programmed paths are continued.

At the time of this report (Summer 1980), it is infeasible to specify the type and technical details of the needed sensors. Several types are mentioned in the previous section.

After the sensors have collected data from the environment, image analysis is necessary to interpret the data for action by the control system. Data impulses received by the sensors must be converted into an information format upon which a decision can be based. This has turned out to be one of the most difficult fields in robotics research and application.

The human's ability to analyze and interpret data collected by his sensory organs is unbelievably complex and difficult to replicate. Yet the teleoperator system must perform this function if it is to carry out its mission.

After the teleoperator system has received and analyzed the environmental data decisions on the teleoperator action must be made by the system without human intervention. This is the domain of artificial intelligence, another basic field in robotics research.

Communications are essential to the teleoperator system operation. Data on the environment and the system must be transmitted to the analysis and control sub-system. Commands resulting from decisions must be transmitted to the functioning sub-systems and components. Accuracy, reliability, speed and other characteristics are important to mission success and efficient operation.

Control function provides the machine direction and monitors of the teleoperator system. Again replicating the process of human control in the fully-automated teleoperator system is an exceedingly complex and difficult task. However, modern control theory research has made considerable progress which will be helpful in this effort.

The kinematics and mechanical design of the teleoperator system and its components will require further research and technology. For example, each additional joint adds degrees of freedom. The number of degrees-of-freedom in the teleoperator exceeds the number which can be analyzed kinematically at the present level of knowledge. Further basic research is needed. Other problems concern the mass, momentum and impact characteristics of the teleoperator system and its components.

End effector design is an field of robotics research important to the Orbital Servicer Teleoperator system. Improvements are needed in the latching mechanism and in the latching and unlatching operations. One such improvement is computer control of the latching and unlatching as the module moves into and out of the rack.

Locomotion, a basic field of robotics research at some institutions, has little relevance to the current mission of neither the Orbital Servicer System nor the Free Flying Rendezvous and Docking Simulator. However, should the missions change, locomotion might become essential to mission success.

Resolver Feedback

The use of resolver feedback for manipulator joint position sensing should be investigated. The current method is subject to inaccuracies due to nonlinearities in the potentiometers, grounding and noise, and variation in the reference voltages. The very accurate resolver feedback is especially needed for the successful implementation of the current dead-reckoning control system.

ADDITIONAL ORBITAL SERVICER SYSTEM PROBLEMS

In addition to full automation of the Orbital Servicer TMS, there are three subsidiary problems in the assignment.

1. Latching mechanism operation.
2. Rack attachment.
3. Electrical self-aligning connector.

Latching Mechanism Operation

The mechanism for latching and unlatching the modules needs improvement in operation and in coordination with the full automation trajectory mode.

The following constraints must be met:

1. Automated operation.
2. Positive latching.
3. No damage to the module or the mechanism of the manipulator arm.
4. No major modification of the present latching mechanism, teleoperator end effectors, components or the existing control program.

One alternative solution is a mechanical device actuated by contact or visual sensors. As the device takes over motion control, the computer control would be released. The mechanism would complete the latching operation. There are a number of difficult problems with this alternative such as the sensing device, powering the latching device and operational difficulties with the mechanism and teleoperator.

A better solution appears to be computer operation and control of the latching operation from start to completion. The coordination problems will be less than with mixed mode control. Full computer control will produce weight savings over mechanical latching devices. In the future, it will be easier to acquire personnel with the necessary computer and electronic skills than personnel with mechanical design, manufacturing and maintenance skills.

Rack Attachment

The present method of attaching the racks to the satellite and to the Orbital Servicer allows some movement and variation in the rack position. This condition creates little difficulty with the manual mode of completing the trajectory. However, it produces major problems for full automation. Variation in the rack position makes completion of the trajectory by dead reckoning infeasible since this operating mode requires that the final rack position be known within close tolerances. If trajectory completion requires complex systems design using image analysis, artificial intelligence, pattern recognition, and other knowledge areas, variable rack position will add

substantially to the difficulty, complexity and expense of full automation.

The constraints on the solution are:

1. No major design change or modification of the module, teleoperator rack or present computer control program.
2. Accuracy of the position must be such that automated teleoperator functioning is feasible.
3. No collision, damage, binding or interference be permitted.

Four Alternate Solutions are:

1. Attach the racks positively and solidly to the Servicer and the Satellite by welding, metal fasteners or similar method.
2. Spring load from one or both sides to center the rack in the desired position.
3. Use artificial vision and feedback with the computer control program to position the module regardless of the location of the rack.
4. Redesign the rack to provide wide entry space tapered to fit the module closely. The best solution appears to be fixed, accurate location of the racks. Full automation will require complex devices and programs to complete the trajectory successfully. Fixed rack positions will hold down the complexity and difficulties of full automation.

Electrical Self-Aligning Connector

Avoiding damage to electrical connectors during transfer of a module between Satellite and Orbiter Servicer is a serious problem. If a pin is bent or the connector is out of alignment, the connection may fail or the connector be bent and damaged.

The proposed solution is an electrical self-aligning connector, NASA Invention Disclosure Case No. MFS - 25211. The plug and receptacle are conical with annular conductive rings spring-loaded for positive contact. This connector does not require precise alignment and is not as liable to damage as the pin-type connector.

It is recommended that the proposed electrical self-aligning connector NASA Invention Disclosure Case No. MFS - 25211 be built and tested. If tests are successful, the design should be incorporated into the TMS and other programs as appropriate.

COMPUTER RESOURCES FOR ROBOTICS

Although MSFC is an acknowledged leader in modern computer facilities, continuous acquisition of new computer technology and upgrading of present computer resources is essential to support the robotics activities.

The robotics and teleoperator efforts should have dedicated computer resources in addition to the general computer resources available at MSFC. The dedicated computers will avoid access delays, provide immediate interface and permit efficient programming and output.

Computers themselves are improving steadily. Storage capacity, operating speed, reliability, flexibility and computing power are steadily increasing while size, weight and cost are going down. There are limits to possible improvements. (The exponential growth curve cannot go on forever.) However, the limits do not appear on the near-time horizon.

Among the dedicated computer facilities that will be needed are:

1. On-board operating and control computers for the Free Flying Mobility Unit in the Free Flying Rendezvous and Docking Simulator. This will provide for maintaining the balance, recording data, and other functions in which the human test subject need not be involved.
2. Graphics software and hardware for both robotics simulators. This may interface with the major computers to gain increased capacity and allow verification through simulation prior to hardware implementation.
3. Full automation of the Orbiter Servicer System will require substantial increase in computer capacity and power in order to perform the necessary command control, image analysis, and other activities.

COMPUTER MODELING

Computer models and simulations can be useful even though they cannot replace full-scale physical mock-ups and simulators. A computer model can be developed, tested and implemented more quickly and less expensively than can a full-scale simulator. The information gained can be useful in design development.

Computer modeling is developing a considerable body of basic knowledge, practical experience, applications and published articles. The topic is included in many current Engineering, Math and Science education programs. The number of professionals with education and training in computer modeling and simulation is steadily increasing. Competence in this area will become increasingly available within NASA and from its contractors.

Computer modeling and simulation does have limits. The validity and reliability of the model output depend upon the extent to which the model represents reality and upon the quality of input data. The right parameters and the correct representations of their behavior must be incorporated in the model. In order to keep the model to manageable size, simplifying assumptions and short cuts are used. These may introduce unacceptable errors. The input data quality is vital to good results. Sometimes there is inadequate communication between the mathematical modeler and the designer-operator of the physical system. The result may be failure to interpret and apply the results properly and to detect anomalies in the model and its output.

A major problem in modeling robotics system is the representation of the physical components and the automated sensory - based interpretation of the data. Another is the present method of representing all trajectories as a set of straight-line elements. This requires a large number of steps and is expensive. A third is incorporation of five or more degrees of freedom; most current modules are limited to three (15).

An important application of robotic programming models is the development of "goal domain" programs. This permits the programmer to specify the goal states of the robot while the robot selects the appropriate actions to achieve the desired goal state (17).

Task specification definition needs to be refined and improved. Too great detail in task definition reduces the scope of adaptive programming to meet variations and contingencies. On the other hand, present state of the art in communications, artificial intelligence, computer languages and artificial vision is inadequate to support goal-oriented task definition in NASA's robotics efforts(13).

CONCLUSIONS AND RECOMMENDATIONS

The major conclusions from the research on robotics in the Free Flying Rendezvous and Docking Simulator and the Orbital Servicer System are:

1. These facilities perform an important service in support of NASA's Space Missions.
2. These facilities should be upgraded and improved.
3. MSFC should move to acquire professional competence in research areas basic to robotics research and technology.

The main recommendations are summarized below. Detailed discussion is the body of this report.

1. A second-generation Free Flying Mobility Unit be acquired. This would provide six degrees of freedom and other improvements infeasible with the present model. The present unit might be kept if its role is complementary to the new unit.
2. The Orbital Servicer System be fully automated.
3. MSFC move to acquire competence in areas necessary to robotics teleoperator development. These include artificial intelligence, sensors, image analysis, pattern recognition, control, communication and computer science applied to robotics.
4. Enhanced computer facilities be acquired for the robotics and teleoperator activities. Some can be additions to Center-wide computer resources; others must be dedicated to the robotics and teleoperator work.
5. Procurement and test of the new electrical self-aligning connector be implemented.
6. Other recommendations are contained in the body of the report. These include rack attachment, latching mechanisms and others.

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THE UNIVERSITY OF ALABAMA

MATERIAL AND PROCESS CONSTRAINTS FOR A FLAT
INTERFACE IN THE BRIDGMAN-STOCKBARGER TECH-
NIQUE

Prepared By:	Larry M. Foster, Ph. D.
Academic Rank:	Assistant Professor
University and Department	University of Alabama in Huntsville Mathematics Department
NASA/MSFC: (Laboratory) (Division) (Branch)	Space Sciences Space Processing Semiconductors and Devices
MSFC Counterpart:	R. J. Naumann
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MATERIAL AND PROCESS CONSTRAINTS
FOR A FLAT INTERFACE IN THE
BRIDGMAN-STOCKBARGER TECHNIQUE

By

Larry M. Foster, Ph. D.
Assistant Professor of Mathematics
University of Alabama in Huntsville
Huntsville, Alabama

ABSTRACT

The functional relationships between various material and process parameters necessary for a flat interface in the Bridgman-Stockbarger technique are given. In particular, two boundary condition cases of

$$(1) \nabla^2 T = P_f \frac{\partial T}{\partial x}, \quad x < 0 \text{ and } 0 < r < 1$$

$$(2) \nabla^2 T = P_s \frac{\partial T}{\partial x}, \quad x > 0 \text{ and } 0 < r < 1$$

$$(3) -K_f \left. \frac{\partial T}{\partial x} \right|_{x=0^-} + K_s \left. \frac{\partial T}{\partial x} \right|_{x=0^+} + L = 0$$

are considered.

$$\text{Case 1: } T(x, 1) = \begin{cases} T_c, & x > 0 \\ T_h, & x < 0 \end{cases}$$

$$\text{Case 2: } \frac{\partial T}{\partial r}(x, 1) = \begin{cases} -B_s T, & x > 0 \\ -B_\lambda (T - T_i), & x < 0 \end{cases}$$

Necessary interval bounds on T_i and T_h are given assuming a flat melt-solid interface at $x = 0$. Approximations of $T(x, r)$ and related error estimates are also given.

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NOMENCLATURE

- P_l - liquid Peclet number
- P_s - solid Peclet number
- T_c - cooler temperature
- T_H - heater temperature
- Q - material melting point
- K_l - liquid thermal conductivity
- K_s - solid thermal conductivity
- \mathcal{L} - crystal growth rate \cdot liquid density \cdot latent heat
- B_s - solid region Biot number
- B_l - liquid region Biot number
- T_i - scaled ambient temperature for liquid region
- T - temperature

INTRODUCTION

The partial differential equations defining crystal growth by the Bridgman - Stockbarger technique [2] are

$$\nabla^2 T = P_l \frac{\partial T}{\partial x}, \quad x < 0$$

and

$$\nabla^2 T = P_s \frac{\partial T}{\partial x}, \quad x > 0$$

where P_l and P_s are the liquid and solid Péclet numbers respectively, T is temperature and x is distance from a flat solid - melt interface. To conserve energy at the interface,

$$-K_l \left(\frac{\partial T}{\partial x} \right)_{x=0^-} + K_s \left(\frac{\partial T}{\partial x} \right)_{x=0^+} + \mathcal{L} = 0$$

where K_l and K_s are the respective liquid and solid thermal conductivities and \mathcal{L} is the product of the crystal growth rate, the melt density and the latent heat of solidification. Two heating mechanisms are investigated; first considered is conductive heating defined by

$$T(x, l) = T_c, \quad x > 0$$

$$T(x, l) = T_h, \quad x < 0$$

followed by Newton heating

$$\frac{\partial T}{\partial r}(x, l) = -B_s T, \quad x > 0$$

$$\frac{\partial T}{\partial r}(x, l) = -B_l (T - T_1), \quad x < 0$$

where T_c and T_h are the respective cooler and heater temperatures, B_s and B_l are the solid and melt Biot numbers respectively, and T_1 is the ambient temperature surrounding melt region. The ambient temperature for the solid region has been scaled to 0.

The assumption of a flat melt-solid interface forces K_s , K_l , T_c , T_h , B_s , B_l , T_1 , P_s , P_l and Q , the material melting temperature, to be functionally dependent. This dependence, the solutions of the above partial differential equations, and various error estimates are given in the next section followed by several numerical experiments. The final section is devoted to conclusions and recommendations.

SOLUTIONS AND APPROXIMATIONS

The solutions of Problems I-II and their computable approximations are given in this section. Also given are the functional relationships between the material and process parameters P_l , P_s , K_l , K_s , T_c , T_H , B_s , B_l , and T , necessary for a solid-melt interface at $x = 0$. We begin with two elementary but fundamental facts.

Fact 1. If (1) $\{a_m\}$ alternates

$$(2) \quad |a_{m+1}| \leq |a_m|$$

$$(3) \quad a_m \rightarrow 0 \text{ as } m \rightarrow \infty$$

then (1) $\sum_1^{\infty} a_m$ converges

$$(2) \quad \left| \sum_1^N a_m - \sum_1^{\infty} a_m \right| \leq |a_{N+1}|$$

Fact 2. For $f(x)$ sufficiently smooth,

$$f'(x) = \frac{f(x+h) - f(x)}{h} + O(h)$$

Problem I.

We wish to solve, in cylindrical coordinates,

$$(1) \quad \nabla^2 T = P_l \frac{\partial T}{\partial x}, \quad x < 0, \quad 0 < r < 1$$

$$(2) \quad \nabla^2 T = P_s \frac{\partial T}{\partial x}, \quad x > 0, \quad 0 < r < 1$$

subject to

$$(3) \quad T(x, 1) = T_c, \quad x > 0$$

$$(4) \quad T(x, 1) = T_H, \quad x < 0$$

$$(5) \quad -K_l \left. \frac{\partial T}{\partial x} \right|_{x=0^-} + K_s \left. \frac{\partial T}{\partial x} \right|_{x=0^+} + \mathcal{L} = 0$$

where $\mathcal{L} =$ crystal growth rate \cdot melt density \cdot latent heat

Condition (5) implies a solid-melt interface at $x = 0$, i. e.,

$$(6) \quad T(0, r) = Q.$$

Unfortunately, conditions (3) - (6) overpose (1) and (2) and generally, overposed problems are impossible to solve. However, in this case, the overposing of (1) and (2) will give constraints on the material and process parameters P_l , P_s , T_c , Q , T_H , K_l , K_s , and \mathcal{L} .

By separation of variables, (1) and (2) with conditions (3), (4) and (6) have solutions [1]

$$(7) \quad T(x, r) =$$

$$T_c + \sum_1^{\infty} \frac{2(Q-T_c)}{\beta_m J_1(\beta_m)} J_0(\beta_m r) \text{Exp} \left[\left(P_s - \sqrt{P_s^2 + 4\beta_m^2} \right) \frac{x}{2} \right],$$

$$x > 0 \quad \text{and} \quad 0 < r < 1$$

$$(8) \quad T(x, r) =$$

$$T_H + \sum_1^{\infty} \frac{2(Q-T_H)}{\beta_m J_1(\beta_m)} J_0(\beta_m r) \text{Exp} \left[\left(P_l + \sqrt{P_l^2 + 4\beta_m^2} \right) \frac{x}{2} \right],$$

$$x < 0 \quad \text{and} \quad 0 < r < 1$$

where $\{\beta_m\}$ is the increasing sequence of all positive roots of J_0 . We next show that Q , K_l , K_s , T_c , T_H , \mathcal{L} , P_s and P_l cannot be arbitrary if (5) is to hold. In fact, (5) will imply that Q , K_s , K_l , T_c , T_H , \mathcal{L} , P_s and P_l satisfy a functional inequality.

Remark 1: An intuitive approach to enlisting (5) is to differentiate (7) and (8) with respect to x and insert the resulting series into (5). But this would require $\sum_1^{\infty} J_1^{-1}(\beta_m)$ to converge, an impossibility. Hence, we will approximate the partial derivatives in (5).

By Fact 2, (5) and (6) imply

$$(9) \quad -K_l \frac{Q - T(-h, 0)}{h} + K_s \frac{T(h, 0) - Q}{h} + \mathcal{L} = O(h)$$

Fact 3. Let $\{\beta_m\}$ be the increasing sequence of all nonnegative roots of $J_0(x)$.
Then

$$\left\{ \frac{1}{\beta_m J_1(\beta_m)} \right\} \text{ alternates in sign, } \left| \frac{1}{\beta_{m+1} J_1(\beta_{m+1})} \right| \leq$$

$$\left| \frac{1}{\beta_m J_1(\beta_m)} \right| \text{ and } \frac{1}{\beta_m J_1(\beta_m)} \rightarrow 0$$

as $m \rightarrow \infty$

Combining Facts 1 and 3 with (7), (8), and (9),

$$(10) \quad \left| 2h - (K_l + K_s)Q + K_s \left\{ T_c + 2(Q - T_c) \sum_1^M \frac{e^{(P_s - \sqrt{P_s^2 + 4\beta_m^2})\frac{h}{2}}}{\beta_m J_1(\beta_m)} \right\} \right.$$

$$\left. + K_l \left\{ T_H + 2(Q - T_H) \sum_1^M \frac{e^{(P_l + \sqrt{P_l^2 + 4\beta_m^2})\frac{-h}{2}}}{\beta_m J_1(\beta_m)} \right\} \right|$$

$$\leq O(h^2) + \left| K_s 2(Q - T_c) \frac{e^{(P_s - \sqrt{P_s^2 + 4\beta_{M+1}^2})\frac{h}{2}}}{\beta_{M+1} J_1(\beta_{M+1})} \right|$$

$$+ \left| K_l 2(Q - T_H) \frac{e^{(P_l + \sqrt{P_l^2 + 4\beta_{M+1}^2})\frac{-h}{2}}}{\beta_{M+1} J_1(\beta_{M+1})} \right|$$

In fact, for computational purposes, if $P_s \ll 1$ and $P_l \ll 1$, then the right hand side of (10) can be replaced by

$$(11) \quad O(h^2) + 2K_s(Q - T_c) \frac{e^{(P_s - 2\beta_{M+1})\frac{h}{2}}}{|\beta_{M+1} J_1(\beta_{M+1})|} \\ + 2K_l(T_H - Q) \frac{e^{(P_l + 2\beta_{M+1})\frac{-h}{2}}}{|\beta_{M+1} J_1(\beta_{M+1})|}$$

Remark 2: (10) forces \mathcal{L} , K_l , K_s , Q , T_c , T_H , P_s and P_l to be functionally dependent. For computational use, we may set

$$(12) \quad \mathcal{L}h - (K_l + K_s)Q + K_s \left\{ T_c + 2(Q - T_c) \sum_1^M \frac{e^{(P_s - \sqrt{P_s^2 + 4\beta_m^2})\frac{h}{2}}}{\beta_m J_1(\beta_m)} \right\} \\ + K_l \left\{ T_H + 2(Q - T_H) \sum_1^M \frac{e^{(P_l + \sqrt{P_l^2 + 4\beta_m^2})\frac{-h}{2}}}{\beta_m J_1(\beta_m)} \right\} \\ = 0$$

for h sufficiently small and M sufficiently large. This gives an equation linear in \mathcal{L} , K_s , K_l , Q , T_c , T_H and nonlinear in P_s and P_l . In particular, (12) gives T_H in terms of \mathcal{L} , K_l , K_s , Q , T_c , P_s and P_l . Furthermore, if (10) is violated, (1) and (2) cannot have a solution satisfying (3) - (6).

Remark 3: Since $\frac{f(x+h) - f(x)}{h} = f'(x + \frac{h}{2}) + O(h^2)$,

it may appear profitable to replace

$$\frac{T(h, 0) - Q}{h} \quad \text{and} \quad \frac{Q - T(-h, 0)}{h} \quad \text{in (9) by}$$

$$\frac{\partial T}{\partial x}(\frac{h}{2}, 0) \quad \text{and} \quad \frac{\partial T}{\partial x}(\frac{-h}{2}, 0) \quad \text{respectively. However,}$$

$$\frac{\partial T}{\partial x}(\frac{h}{2}, 0) = 2(Q - T_c) \sum_1^\infty \frac{P_s - \sqrt{P_s^2 + 4\beta_m^2}}{\beta_m J_1(\beta_m)} e^{(P_s - \sqrt{P_s^2 + 4\beta_m^2})\frac{h}{4}}$$

converges much slower than the estimates in (10).

Remark 4: An interval estimate of linear parameters λ , K_λ , K_s , T_c and T_H can be given. For example, suppose λ , K_λ , K_s , T_c , P_s , P_λ and Q are fixed. Then (10) implies

$$(12) \quad O(h^2) - K_s 2(Q - T_c) \frac{e^{\frac{(P_s - \sqrt{P_s^2 + 4\beta_{M+1}^2})h}{2}}}{|\beta_{M+1} J_1(\beta_{M+1})|}$$

$$- \lambda h + (K_\lambda + K_s)Q + 2QK_\lambda \frac{e^{\frac{(P_\lambda + \sqrt{P_\lambda^2 + 4\beta_{M+1}^2})-h}{2}}}{|\beta_{M+1} J_1(\beta_{M+1})|}$$

$$- K_s \left(T_c + 2(Q - T_c) \sum_1^M \frac{e^{\frac{(P_s - \sqrt{P_s^2 + 4\beta_m^2})h}{2}}}{\beta_m J_1(\beta_m)} \right)$$

$$- K_\lambda 2Q \sum_1^M \frac{e^{\frac{(P_\lambda + \sqrt{P_\lambda^2 + 4\beta_m^2})-h}{2}}}{\beta_m J_1(\beta_m)} \leq$$

$$K_\lambda T_H \left(1 - 2 \sum_1^M \frac{e^{\frac{(P_\lambda + \sqrt{P_\lambda^2 + 4\beta_m^2})-h}{2}}}{\beta_m J_1(\beta_m)} + 2 \frac{e^{\frac{(P_\lambda + \sqrt{P_\lambda^2 + 4\beta_{M+1}^2})-h}{2}}}{|\beta_{M+1} J_1(\beta_{M+1})|} \right)$$

and

$$(13) \quad K_1 T_H \left(1 - 2 \sum_1^M \frac{e^{\frac{(P_1 + \sqrt{P_1^2 + 4\beta_m^2}) - h}{2}}}{\beta_m J_1(\beta_m)} - 2 \frac{e^{\frac{(P_1 + \sqrt{P_1^2 + 4\beta_{M+1}^2}) - h}{2}}}{|\beta_{M+1} J_1(\beta_{M+1})|} \right)$$

$$\pm O(h^2) + 2K_s(Q - T_c) \frac{e^{\frac{(P_s - \sqrt{P_s^2 + 4\beta_{M+1}^2})h}{2}}}{|\beta_{M+1} J_1(\beta_{M+1})|}$$

$$- 2K_1 Q \frac{e^{\frac{(P_1 + \sqrt{P_1^2 + 4\beta_{M+1}^2}) - h}{2}}}{|\beta_{M+1} J_1(\beta_{M+1})|} - 2h + (K_1 + K_s)Q$$

$$- K_s \left(T_c + 2(Q - T_c) \sum_1^M \frac{e^{\frac{(P_s - \sqrt{P_s^2 + 4\beta_m^2})h}{2}}}{\beta_m J_1(\beta_m)} \right)$$

$$- K_1 2Q \sum_1^M \frac{e^{\frac{(P_1 + \sqrt{P_1^2 + 4\beta_m^2}) - h}{2}}}{\beta_m J_1(\beta_m)}$$

Solving (12) and (13) for T_H ,

$$T_1(M, h) + O(h^2) \leq T_H \leq T_2(M, h) + O(h^2)$$

For practical purposes, set

$$(14) \quad T_1(M, h) \leq T_H \leq T_2(M, h)$$

Since we only know bounds on T_H (from (14)), for computational use, approximate

T_H by $T_{H,APP} = \frac{T_2(M, h) + T_1(M, h)}{2}$. If T_H is replaced by $T_{H,APP}$ in (8), then

the resulting error is

$$(15) \quad \left| T_H - T_{H,APP} - 2 \sum_1^{\infty} \frac{T_H - T_{H,APP}}{\beta_m J_1(\beta_m)} J_0(\beta_m r) e^{(P_2 + \sqrt{P_2^2 + 4\beta_m^2}) \frac{x}{2}} \right|$$

which has an upper bound of

$$(16) \quad (T_2(M, h) - T_1(M, h)) \left| \frac{1}{2} - \sum_1^{\infty} \frac{J_0(\beta_m r)}{\beta_m J_1(\beta_m)} e^{(P_2 + \sqrt{P_2^2 + 4\beta_m^2}) \frac{x}{2}} \right|$$

Problem II.

We wish to solve, in cylindrical coordinates,

$$(17) \quad \nabla^2 T = P_1 \frac{\partial T}{\partial x} \quad , \quad x < 0 \quad \text{and} \quad 0 < r < 1$$

$$(18) \quad \nabla^2 T = P_2 \frac{\partial T}{\partial x} \quad , \quad x > 0 \quad \text{and} \quad 0 < r < 1$$

subject to

$$(19) \quad \frac{\partial T}{\partial r}(x, 1) = -B_2 T \quad , \quad x > 0$$

$$(20) \quad \frac{\partial T}{\partial r}(x, 1) = -B_1 (T - T_1) \quad , \quad x < 0$$

$$(21) \quad -K_l \left(\frac{\partial T}{\partial x} \right)_{x=0^-} + K_s \left(\frac{\partial T}{\partial x} \right)_{x=0^+} + \mathcal{L} = 0$$

where \mathcal{L} = crystal rate of growth · liquid density · latent heat, $B_1 \geq 0$ and $B_2 \geq 0$. Condition (21) implies a solid-melt interface at $x = 0$, i. e.,

$$(22) \quad T(0, r) = Q$$

As in Problem I, conditions (19) - (22) overpose (17) and (18). This dilemma can be resolved only if the material and process parameters P_1 , P_2 , Q , T_1 , K_s , K_l and \mathcal{L} are functionally dependent.

Separating variables, (17) and (18) subject to (19), (20) and (22) have solutions [1]

$$(23) \quad T(x, r) = \sum_1^{\infty} \frac{2Q B_s J_0(\beta_m r)}{(\beta_m^2 + B_s^2) J_0(\beta_m)} e^{(P_s - \sqrt{P_s^2 + 4\beta_m^2}) \frac{x}{2}}, \quad x > 0$$

$$(24) \quad T(s, r) = T_1 + \sum_1^{\infty} \frac{2(Q - T_1) B_l J_0(\beta_m r)}{(\alpha_m^2 + B_l^2) J_0(\alpha_m)} e^{(P_l + \sqrt{P_l^2 + 4\beta_m^2}) \frac{x}{2}}, \quad x < 0$$

where $\{\alpha_m\}$ and $\{\beta_m\}$ are the increasing nonnegative sequences of all roots of

$$(25) \quad B_s J_0(\beta_m) = \beta_m J_1(\beta_m)$$

$$(26) \quad B_l J_0(\alpha_m) = \alpha_m J_1(\alpha_m)$$

Fact 4: $\{(\beta_m^2 + B_s^2) J_0(\beta_m)\}$ alternates in sign and $\left| \frac{1}{(\beta_m^2 + B_s^2) J_0(\beta_m)} \right|$

decreases to 0. A similiar result holds for $\left\{ \frac{1}{(\alpha_m^2 + B_l^2) J_0(\alpha_m)} \right\}$

As in Problem I, (21) and Fact 2 imply

$$(27) \quad -K_l \frac{Q - T(-h, 0)}{h} + K_s \frac{T(h, 0) - Q}{h} + \mathcal{L} = O(h)$$

Combining Facts 1 and 4 with (23), (24) and (27),

$$(28) \quad \left| \mathcal{L}h - Q(K_s + K_l) + K_s \sum_1^M \frac{2 B_s Q}{(\beta_m^2 + B_s^2) J_0(\beta_m)} e^{(P_s - \sqrt{P_s^2 + 4\beta_m^2}) \frac{h}{2}} \right. \\ \left. + K_l \left\{ T_1 + \sum_1^M \frac{2(Q - T_1) B_l}{(\alpha_m^2 + B_l^2) J_0(\alpha_m)} e^{(P_l + \sqrt{P_l^2 + 4\alpha_m^2}) \frac{-h}{2}} \right\} \right|$$

$$\leq O(h^2) + \left| \frac{2Q B_s K_s}{(\beta_{M+1}^2 + B_s^2) J_0(\beta_{M+1})} e^{(P_s - \sqrt{P_s^2 + 4\beta_{M+1}^2}) \frac{h}{2}} \right|$$

$$+ \left| \frac{2(Q - T_1) B_l K_l}{(\alpha_{M+1}^2 + B_l^2) J_0(\alpha_{M+1})} e^{(P_l + \sqrt{P_l^2 + 4\alpha_{M+1}^2}) \frac{h}{2}} \right|$$

As in Problem I, (28) implies various interval constraints on Q , K_s , K_l , T_1 , P_s , P_l , B_s and B_l . For illustration, consider T_1 . Because we have transformed (in practice) the Newton cooling condition (19) to imply a 0° ambient temperature around the rod for $x > 0$, $0 < Q < T_1$. (28) then implies

(29)

$$O(h^2) + Q(K_s + K_l) - 2QK_l B_l \sum_1^M \frac{e^{(P_l + \sqrt{P_l^2 + 4\alpha_m^2}) \frac{h}{2}}}{(\alpha_m^2 + B_l^2) J_0(\alpha_m)}$$

$$- 2K_s B_s Q \sum_1^M \frac{e^{(P_s - \sqrt{P_s^2 + 4\beta_m^2}) \frac{h}{2}}}{(\beta_m^2 + B_s^2) J_0(\beta_m)} - \mathcal{L}h$$

$$+ \frac{2B_s Q K_s e^{(P_s - \sqrt{P_s^2 + 4\beta_{M+1}^2}) \frac{h}{2}}}{(\beta_{M+1}^2 + B_s^2) |J_0(\beta_{M+1})|} - \frac{2B_l Q K_l e^{(P_l + \sqrt{P_l^2 + 4\alpha_{M+1}^2}) \frac{h}{2}}}{(\alpha_{M+1}^2 + B_l^2) |J_0(\alpha_{M+1})|}$$

$$\geq T_l K_l \left\{ 1 - 2 B_l \sum_1^M \frac{e^{\frac{(P_l + \sqrt{P_l^2 + 4\alpha_m^2}) - h}{2}}}{(\alpha_m^2 + B_l^2) J_0(\alpha_m)} \right. \\ \left. - 2 B_l \frac{e^{\frac{(P_l + \sqrt{P_l^2 + 4\alpha_{M+1}^2}) - h}{2}}}{(\alpha_{M+1}^2 + B_l^2) |J_0(\beta_{M+1})|} \right\}$$

and

$$(30) \quad O(h^2) - \frac{2 B_s K_s Q e^{\frac{(P_s - \sqrt{P_s^2 + 4\beta_{M+1}^2}) h}{2}}}{(\beta_{M+1}^2 + B_s^2) |J_0(\beta_{M+1})|} +$$

$$\frac{2 Q B_l K_l e^{\frac{(P_l + \sqrt{P_l^2 + 4\alpha_{M+1}^2}) - h}{2}}}{(\alpha_{M+1}^2 + B_l^2) |J_0(\beta_{M+1})|} + Q(K_s + K_l)$$

$$- 2 K_l B_l Q \sum_1^M \frac{e^{\frac{(P_l + \sqrt{P_l^2 + 4\alpha_m^2}) - h}{2}}}{(\alpha_m^2 + B_l^2) J_0(\alpha_m)}$$

$$- 2 K_s B_s Q \sum_1^M \frac{e^{\frac{(P_s - \sqrt{P_s^2 + 4\beta_m^2}) h}{2}}}{(\beta_m^2 + B_s^2) J_0(\beta_m)} - 2h$$

$$\leq T_{l,K_l} \left\{ 1 - 2B_l \sum_1^M \frac{e^{(P_l + \sqrt{P_l^2 + 4\alpha_m^2}) \frac{-h}{2}}}{(\alpha_m^2 + B_l^2) J_0(\alpha_m)} \right. \\ \left. + 2B_l \frac{e^{(P_l + \sqrt{P_l^2 + 4\alpha_{M+1}^2}) \frac{-h}{2}}}{(\alpha_{M+1}^2 + B_l^2) |J_0(\alpha_{M+1})|} \right\}$$

Solving (29) and (30) for T_l ,

$$T_l^{LOW}(M, h) + 0(h^2) \leq T_l \leq T_l^{HIGH}(M, h) + 0(h^2)$$

For computational purposes, let

(31)

$$T_l^{LOW}(M, h) \leq T_l \leq T_l^{HIGH}(M, h)$$

(32)

$$T_{l, APPROX} = (T_l^{LOW}(M, h) + T_l^{HIGH}(M, h)) / 2$$

The error committed by replacing T_l in (24) by $T_{l, APPROX}$ is no more than

(33)

$$\left(T_l^{HIGH}(M, h) - T_l^{LOW}(M, h) \right) \left| \frac{1}{2} - \sum_1^{\infty} \frac{B_l J_0(\alpha_n r)}{(\alpha_n^2 + B_l^2) J_0(\alpha_n)} e^{(P_l + \sqrt{P_l^2 + 4\alpha_n^2}) \frac{x}{2}} \right|$$

Remark 5: It is possible to substitute (23) and (24) into (21) and solve for T_1 . However, this would require computing

(34)

$$\sum_1^{\infty} \frac{P_s - \sqrt{P_s^2 + 4\beta_m^2}}{(\beta_m^2 + B_s^2) J_0(\beta_m)}$$

and

(35)

$$\sum_1^{\infty} \frac{P_l + \sqrt{P_l^2 + 4\alpha_m^2}}{(\alpha_m^2 + B_l^2) J_0(\alpha_m)}$$

Although (34) and (35) conditionally converge, the sums in (28) - (30) and (33) converge faster by several orders of magnitude.

Remark 6: Let $B_s = B_l = P_s = P_l = 0 = \gamma$ and fix $0 < h \ll 1$.

Letting $M = 39$, (10) implies

$$Q = \frac{K_s T_c + K_l T_w}{K_l + K_s}$$

for Problem 1 and (28) gives

$$Q = \frac{K_l T_i}{K_s + K_l}$$

for Problem 2.

NUMERICAL EXAMPLES

The results of several numerical experiments are given in this section.

Example 1: In Figure 1 and Table 1 we illustrate the dependence on M of the interval estimate (14). Let $T_c = 400$, $Q = 700$, $P_S = P_f = 0.001$, $\alpha = 0.01$, $h = 0.1$, $K_S = 0.0032$ and $K_f = 0.002$.

M	Interval Estimate of T_H
9	[1391.34, 1476.13]
14	[1202.19, 1214.26]
19	[1180.56, 1182.74]
24	[1176.34, 1176.54]
29	[1176.34, 1176.54]
34	[1175.92, 1175.93]
39	[1175.90, 1175.90]

Table 1

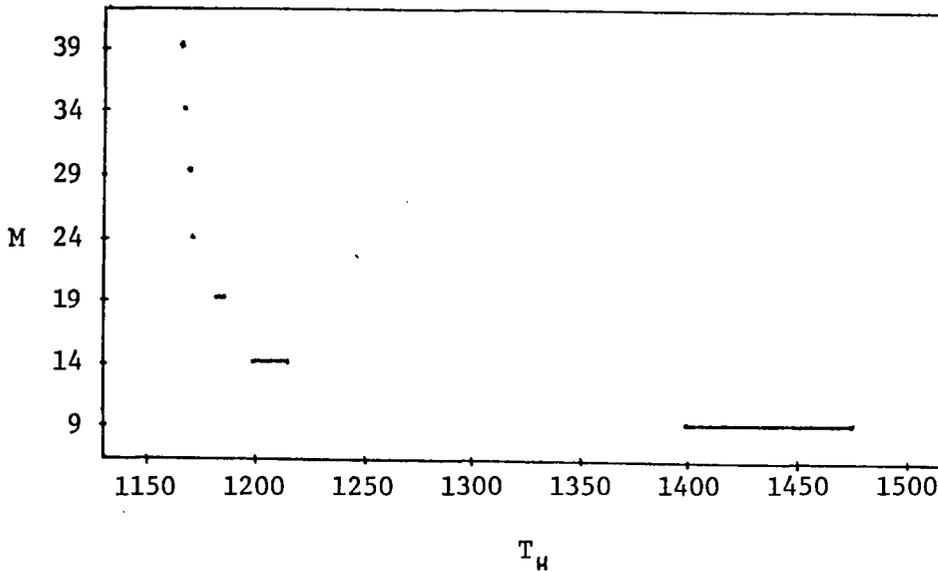


Figure 1

Example 2: In Table 2 and Figure 2 we illustrate the sensitivity of the interval estimate (14) to changes in α . Let $K_f = K_S = P_S = P_f = 0.001$,

$h = 0.1$, $T_c = 200.0$, $Q = 700.0$ and $M = 39$.

λ	Interval Estimate of T_H
.001	[1192.386, 1192.389]
.011	[1116.514, 1116.517]
.021	[1040.642, 1040.645]
.031	[964.770, 964.773]
.041	[888.898, 888.901]
.051	[813.026, 813.029]
.061	[737.154, 737.157]

Table 2

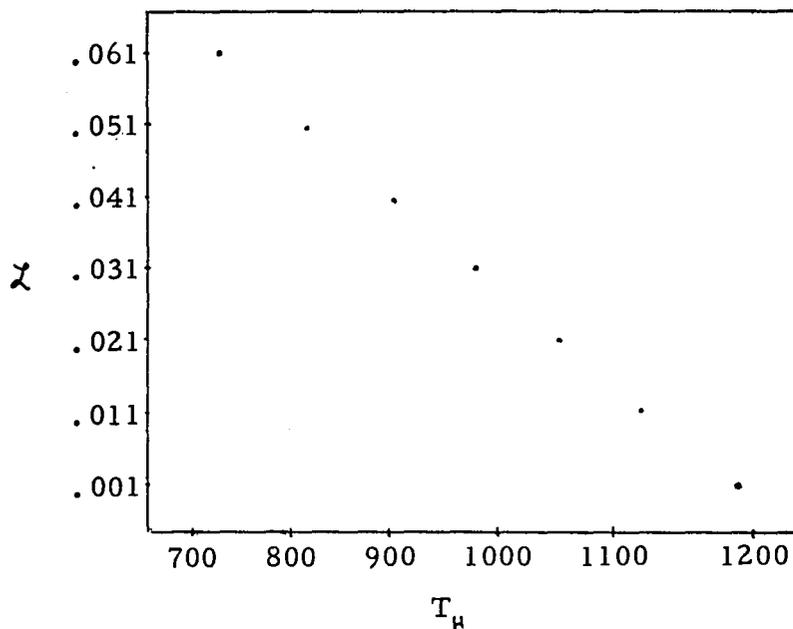


Figure 2

CONCLUSIONS AND RECOMMENDATIONS

Equations (10) and (28) imply that the material and process parameters cannot be arbitrary if a flat melt-solid interface is to be achieved. The sensitivity of one parameter to changes in another parameter (Example 2) suggests great care should be taken in obtaining the various material parameters.

The relationships between M and h that minimize the interval widths in (14) and (31) remain open questions. Further investigations of Problems I and II could incorporate adiabatic boundary zones near the melt-solid interface and the influences of various container configurations.

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NASA/ASEE SUMMER FACULTY RESEARCH FELLOWSHIP PROGRAM

A RANKING ALGORITHM FOR SPACELAB
CREW AND EXPERIMENT SCHEDULING

Prepared by: Robert D. Grone and Frank H. Mathis

Academic Ranks: Assistant Professor

Institution and Department: Auburn University
Department of Mathematics

NASA/MSFC:

Division: Mission Analysis

Branch: Mission Integration

MSFC Counterparts: Jerry D. Weiler and William C. Askew

Contract No: NGT-01-002-099 (University of Alabama)

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CREW AND EXPERIMENT SCHEDULING

by

Robert D. Grone
Assistant Professor
Department of Mathematics
Auburn University
Auburn, Alabama 36849

and

Frank H. Mathis
Assistant Professor
Department of Mathematics
Auburn University
Auburn, Alabama 36849

ABSTRACT

This report is concerned with the problem of obtaining an optimal or near optimal schedule for scientific experiments to be performed on Spacelab missions. The design criteria of a method for solving this problem is two-fold. First, the method must produce a schedule which satisfies all material and resource constraints while maximizing a function assigning a grade to each schedule. Secondly, the method should reduce the amount of pre-flight manpower and computer time currently necessary to achieve this goal and be rapid and flexible enough to allow real-time support of rescheduling problems arising due to launch delays, equipment malfunction, or other factors.

In this paper the current capabilities in this regard are examined and a method of ranking experiments in order of difficulty is developed to support the existing software. Experimental data is obtained from applying this method to the sets of experiments corresponding to Spacelab missions 1, 2, and 3. Finally, suggestions are made concerning desirable modifications and features of second generation software currently being developed for this problem.

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INTRODUCTION

The problem under consideration is that of scheduling a set of experiments to be performed on a given Spacelab mission. Through a preliminary analysis, which is not discussed here, a set of compatible experiments is compiled which are hopefully to be included in a specific mission. The experiments are then converted into model sheets by the principal investigator, who is responsible for the experiment design, and a NASA project engineer familiar with the format and requirements of the model sheets. The model sheet divides each experiment into a sequence of steps, each of which has certain requirements in terms of crew, equipment, and energy usage; and may in addition require that the Spacelab be in a certain position or configuration, or have certain targets available. Examples of targets are planets, communication satellites, and data receiving facilities on the earth's surface. If the experiment is to be performed more than once the number of performances desired is included on the model sheet. A typical Spacelab mission lasts for seven days and involves six crewmembers.

Extremely sophisticated software is available to assist in scheduling of the experiments. The current program is designated by TLP for timeline program. This program takes the model sheets in some order and "front loads" them in this order. By front load, it is meant that the model is scheduled at the earliest possible time which satisfies all the model constraints and which does not conflict with the requirements of previously scheduled models. If it is impossible to schedule a certain model at any time the program then goes to the next model in the sequence. This program is currently being upgraded to a program denoted by ESP for experiment scheduling program, which has the capability of either front loading or back loading any given model, and which is to supersede TLP sometime in 1981.

Even with the extremely advanced and effective software available, the problem of scheduling the experiments in an optimal way currently requires an enormous amount of pre-flight man-hours. By an optimal schedule, we roughly mean one that includes the maximal number of experiments and performances, and which makes maximal use of the available resources. Since each set of models designated for inclusion in a specific mission are only roughly sorted for compatibility, it is frequently impossible to include every performance of every experiment.

The method previously employed has been to feed TLP various random orderings of the model sheets and have the project engineers examine the characteristics of the resulting schedules. From these examinations other orderings are suggested and fed to TLP. After months of such trial and error, the best resulting schedule is then modified and edited by hand by a group of NASA personnel familiar with the characteristics of the given mission. The current schedule for Spacelab mission one has been under development for five years. It is clear that such a time expenditure is not only economically prohibitive, but is unacceptable in light of the fact that the Spacelab program eventually envisions flying several missions a year.

OBJECTIVES

The main object of this paper is to provide a method of ranking the model sheets in order of difficulty. By scheduling the most difficult experiments first it is thought that an optimal schedule will result. In support of this idea a grading function is used which assigns a higher grade to schedules which are in some sense better. The problem is then to find a ranking function of the models which provides an ordering which results in a schedule which maximizes the grading function. Since there are 218 model sheets corresponding to Spacelab mission one, an exhaustive search is clearly out of the question.

The remainder of this paper is devoted to the development and examination of such a ranking function. While the method developed did not result in a schedule superior to the schedule hand produced over a five-year period for Spacelab mission one, it did quickly produce schedule for Spacelab missions two and three, which were superior to the best existing schedules for these missions.

These experimental results are described and suggestions are made on the basis of these for possible modifications or additions to the existing software.

THE RANKING ALGORITHM

The problem is now to produce a ranking algorithm which produces an ordering of the models which produces an optimal schedule in terms of a grading function which is described in the results section. While this has certain aspects of a linear programming problem which might be treated with either the simplex method or the new Khachian method, it is unclear if it is possible to convert this to a linear problem. The method we exhibit is similar to the one described by Garey, Graham, Johnson, and Yao in J. Comb. Theory (A) 21 (1976), 257-298.

We now proceed to describe the method used to rank the models in decreasing order of difficulty. The steps involved are: the construction of a nonnegative real vector corresponding to a single performance of each model, the normalization of these vectors, and the evaluation of the difficulty function on the normalized vectors. The models (vectors) are then arranged in decreasing order of difficulty and this ordering is fed to TLP to produce a schedule.

The vector includes a component corresponding to crew time required. The length of each step is multiplied by the number of crewmen required for the step, and the sum of these terms over all steps in the model is computed to give a figure in crewman-minutes.

The vector includes components for up to 10 resources. On Spacelab mission 1 a total of 4 resources were included: power, high rate multiplex, computer memory, and data transmission. The components for each of these resources were computed by multiplying the length of each step times the level usage of the resource and then summing over all steps.

The vector includes up to 40 components corresponding to various pieces of equipment. As in the case of crew and resources, the number of equipment-minutes is computed by summing the number of minutes a certain piece of equipment is required over all steps of the model. On Spacelab mission 1, a total of 38 pieces of equipment are considered as components of the vector.

The vector includes components corresponding to attitude and maneuvers. If a step requires that the Orbiter be performing a certain maneuver the length of the step is multiplied by 9, and if a step requires that no maneuvers are to be performed during its execution the length of the step is multiplied by 1. The sum over all steps of the model then produces the component corresponding to maneuvers. The attitude (Orbiter orientation) component is computed in an analogous fashion.

The vector includes up to 72 components corresponding to targets; Spacelab mission 1 having a total of 50 such components. Once again, a figure in target-minutes is produced for each target and model by summing the number of minutes each target is required by a particular model over all steps of that model.

The vector includes two components referred to as Δ -start and start-time. For the i th model, let M_i and m_i denote the maximum and minimum respective times at which the performance of the first step of the model may be initiated. The Δ -start component is

$$\Delta_i = L - (M_i - m_i),$$

where L is the total length of the mission. This component is higher for models which are less flexible with respect to possible starting times. The start-time component is

$$S_i = L - M_i,$$

and gives higher values to models which must be started earlier in the mission. The start-time figure is included in view of the front-loading nature of TLP which causes the time and resources at the earlier part of the mission to be used up first.

The next step in the ranking algorithms is the normalization of the vectors corresponding to the models. Let \bar{v}_{ij} denote the j th component of the vector corresponding to the i th model. Hence, if there are m models and n vector components then

$$V = [v_{ij}],$$

is an m -by- n nonnegative real matrix. Spacelab mission 1 corresponds in this fashion to a 218-by-95 matrix. The scaling is obtained by dividing the j th component of each

vector by the sum of the j th components of all the vectors. That is: we normalize the matrix V to a column-stochastic matrix by dividing each column by its column sum. The targets constitute an exception to this rule. The target time component is normalized by dividing by the total length the target is available during the mission rather than the time the target is required by all the models.

The next step in the ranking algorithm is to evaluate the difficulty function at each model (row of V). The programmer has the option of assigning nonnegative weights; $w_j, j=1, \dots, n$, to each of the components of a vector (or alternatively, columns of V). If the i th model has corresponding vector $v_i=(v_{i1}, \dots, v_{in})$, the difficulty of the i th model is defined by

$$d_i = \sqrt{p_i (v_{i1}w_1 + \dots + v_{in}w_n)},$$

where p_i is the number of performances of the i th model requested. The capability of adjusting the weights allows a scheduler to emphasize or de-emphasize certain aspects of the mission deemed more critical or less critical to produce various rankings of the models.

The above described procedure of ranking the models in order is referred to as the model-sort method. A slightly different method referred to as performance-sort was also implemented and tested on Spacelab missions 1, 2, and 3. The idea behind the performance-sort is to treat each performance of an experiment as a separate model. With the model sort program, all possible performances of a given model are scheduled consecutively. With the performance sort program, each performance of a model corresponds to a vector. For example: Spacelab 1 lists 935 performances of the 218 models. Hence, when using model-sort, a 218-by-95 matrix is envisioned; and when using performance-sort a 935-by-95 matrix is envisioned. If v_i is the vector previously described which corresponds to a single performance of the i th model, and the i th model requires p_i performances, the difficulty level of the k th performance of the i th model is

$$\sqrt{k (v_{i1}w_1 + \dots + v_{in}w_n)}; k=1, \dots, p_i.$$

Another feature of the ranking method is the selection of fixed models. This allows the programmer to specify a certain subset of the models or performances which will be scheduled first. For example, on Spacelab mission 1, such items as "crewman sleep" and "crewman lunch" have model sheets corresponding to them. These models are scheduled first and then the remaining models are ranked using either the model-sort or performance-sort method. On Spacelab mission 1 a total of 16 models were fixed.

Another feature of the algorithm is the model weight capability. This is in some sense dual to the vector normalization in that a row is multiplied by a nonnegative scalar in the model weight feature. This allows the programmer to assign weights to any or all of the models which will change the corresponding vector. This option lets the operator increase the difficulty function of a model which is considered particularly important for political or other reasons not reflected in the difficulty function. The authors used this feature to schedule models which were not included in an otherwise excellent schedule. Successive doubling of the weight of a certain model insures that it will appear higher and higher in the ordering until it is the first model scheduled, if necessary.

RESULTS

We proceed to discuss the results obtained using the previously described methods. The schedule produced due to some choice of sorting technique, weights, fixed models, and model weights is evaluated by means of a grading function which is dependent on four parameters: percentage of crew time, percentage of experiment time, percentage of performances, and percentage of models. The average of these four percentages yields the grade of the corresponding schedule. All four parameters are obtained by comparing amount scheduled to amount requested. In this connection, it should be mentioned that the experiment and crew time requested is the minimum experiment time for experiments whose steps have variable length. Hence, it is possible for the percentage score on experiment and crew time, and the grade, to be over 100. Another fact worth noting is that it is possible for a fraction of a model to be performed in some cases. For example, if 3 out of 4 steps in a given model were performed this would be counted as .75 of a model. The percentage of models reflects the extent to which experiment on the mission is performed at least once, and the percentage of performances measures the extent to which all requested experiments (including multiplicities) were scheduled.

For each of Spacelab missions 1, 2, and 3 we include tabulated data comparing the effects of varying sorting technique, weights, fixed models, and model weights. For each mission we include for comparison the best schedule which had been previously produced by the personnel at Marshall Space Flight Center. Their method was to experiment with various orderings, feed them to TLP, obtain some reasonably good schedules, and then to invoke software referred to as TLE (timeline editor) to then shift and fit experiments. The TLE affords the capability of removing and rescheduling a single performance from an existing schedule.

Spacelab mission 1 has undergone the longest development time, and we were unable to exceed the best previous score by our methods. Mission 1 is particularly complicated due to the fact that the set of models chosen for inclusion were selected more for political considerations of being included on the first mission than for compatibility of the various models. It is thought that subsequent missions will be designed around a particular theme such as atmospheric research, materials processing, space physics, biological sciences, or astronomy. On Spacelab mission 1 it was

found, somewhat unexpectedly, that the model sort technique worked best. On Spacelab missions 2 and 3, the performance sort worked best and produced schedules superior to the previous best. The overall best results were obtained by using the weights which had produced the schedule with maximal score and then increasing the model weights of those models which failed to schedule. On these missions every model was scheduled for at least one performance. The following three tables summarize the results obtained on these three missions, and compares these to the previous best schedule.

SPACELAB 1

MODELS - 218 -- PERFORMANCES - 935

MIN. EXP. TIME - 4869.71 HRS. -- MIN. CREW TIME - 800.62 HRS.

WEIGHTING	TYPE OF SORT	MODELS	PERFORMANCES	EXP. TIME	CREW TIME	GRADE
ALL EQUAL	MODEL	183.69	858.61	4939.35	756.25	92.99
ALL EQUAL	PERFORMANCE	184.51	851.22	4847.83	757.56	92.46
TARGETS = 10, Δ-START = 1	MODEL	193.90	884.39	4664.38	762.87	93.64
PREVIOUS BEST		203.03	897.94	4866.52	751.04	95.73

SPACELAB 2

MODELS - 168 -- PERFORMANCES - 1992

MIN. EXP. TIME - 2887.64 HRS. -- MIN. CREW TIME - 647.44 HRS

WEIGHTING	TYPE OF SORT	MODELS	PERFORMANCES	EXP. TIME	CREW TIME	GRADE
ALL EQUAL EXCEPT Δ -START = 0	MODEL	154.54	1931.09	3804.08	682.90	106.54
ALL EQUAL EXCEPT Δ -START = 0	PERFORMANCE	159.50	1971.09	3813.67	693.83	108.28
CREW, RESOURCES, TARGETS, WINDOW SIZE ONLY. (EQUIPMENT AND START TIME = 0)	PERFORMANCE	162.15	1974.09	3820.48	700.44	109.03
ABOVE WITH MODEL WEIGHTS ADJUSTED	PERFORMANCE	164.15	1976.09	3821.98	702.48	109.44
PREVIOUS BEST		161.82	1976.65	3808.10	658.08	107.27

SPACELAB 3

MODELS - 61 -- PERFORMANCES - 592

MIN. EXP. TIME - 2507.48 HRS. -- MIN. CREW TIME - 569.02

WEIGHTING	TYPE OF SORT	MODELS	PERFORMANCES	EXP. TIME	CREW TIME	GRADE
ALL EQUAL	MODEL	57.03	519.00	2476.10	562.52	94.69
ALL EQUAL	PERFORMANCE	57.70	519.00	2480.03	562.55	95.00
TARGET AND Δ-START ONLY	PERFORAMCNE	58.90	520.00	2478.37	565.72	95.66
ABOVE WITH MODEL WEIGHTS ADJUSTED	PERFORMANCE	59.77	521.00	2480.97	568.75	96.22
PREVIOUS BEST		57.03	519.00	2484.29	565.94	94.92

CONCLUSIONS

In this section, we indicate some avenues of approach suggested by our investigations. These suggestions arose largely on an intuitive basis, and are in no particular order.

It is certain that experimentation with weights is necessary to implement this algorithm. It appeared that no one weighting would produce the best results for all Spacelab missions. Perhaps a scheme could be developed to give an approximation to the optimal weighting by considering the resource profiles of the specific mission.

The difficulty function could be modified. The difficulty function discussed in the body of this paper was motivated by the sum norm on nonnegative vectors. That is, if $v = (v_1, \dots, v_n)$ is a nonnegative real vector, the sum norm of v is defined via

$$v_1 + \dots + v_n.$$

We also experimented with a grading function related to the supremum norm

$$\max_i v_i.$$

The results obtained were equal or slightly less than the results described in the previous section. It is rather remarkable that a greater difference was not observed. Further experimentation is possible. Perhaps the Euclidean norm might be investigated.

The grading function could be modified. For example, weights could be assigned to the four parameters involved. Also, the grading function does not take into consideration the evenness of the scheduling distribution. It would be better, for example, if the crew were not completely utilized during the first half of the mission and have a lot of unused time during the second half. Perhaps some mechanism might be devised to give preference to schedules whose resource utilization levels are held fairly constant over the duration of the mission.

An examination of what we shall term a dynamic sort scheme was done. In the method described in this paper no consideration is taken of the fact that a model may become more difficult to schedule if models preceding it deplete the resources it requires. We implemented the following iterative technique to circumvent this phenomena. A re-examination of the normalization procedure employed shows that most components of the vector were normalized by the total amount required by all the models in that component. The target components were normalized by the total available target time rather than the total target time requested. We also experimented with normalizing all components by the amount available corresponding to each component. With this normalization scheme we implemented the following dynamic sort option. In this scheme we would find the most difficult model and then re-normalize the remaining vectors by the amounts remaining of the various resources, etc. This iterative technique has the effect of dynamically increasing the weighting factors on those components which are being depleted fastest. We then assigned various weighting factors and computed the corresponding orderings with and without the dynamic sort option. The ordering produced with the dynamic sort option consistently produced a schedule inferior to the one produced without using the dynamic sort. Even in view of these results, we feel that investigation along these lines might yield superior techniques.

All of the previous discussions have focused on possible ways to improve the method of ordering the models to feed to TLP. Our final suggestion arises from limitations imposed by the front-loading nature of TLP, and involve factors internal to TLP rather than external. With the front-loading scheme, a model is scheduled at the earliest possible time. This results in a schedule which has no flexibility during the first part of the mission. It is fairly certain that this period in the mission is very likely to be plagued with correction and adjustment factors, and perhaps space sickness on the part of the crew. Also, the pieces of equipment are scheduled to begin operation at maximum capacity and then level off. This makes the execution of the mission highly dependent on the Spacelab being able to operate at full capacity immediately. The implementation of a front- or back-loading option on ESP is largely motivated by a desire to avoid this type of scheduling. It would be advantageous to schedule each successive model at the "best" possible time rather than the earliest or latest. "Best" might

be defined by scheduling the model at the time during the mission which keeps the overall resource-available profile the flattest over the duration of the mission. If "best" turns out to be impossible to define, it is possible that even a random choice among possible scheduling times for a model in the sequence might result in a superior scheduling method.

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Marshall Space Flight Center

The University of Alabama

Continuous Flow Electrophoresis:

The Effect of Sample Concentration
on Throughput and Resolution in an
Upward Flowing System

Prepared By:	Thomas S. Jandebaur, Ph.D.
Academic Rank:	Assistant Professor
University and Department:	Athens State College Department of Biology
NASA/MSFC: (Laboratory) (Division) (Branch)	Space Sciences Space Processing Separation Processes
MSFC Counterpart:	R. S. Snyder, Ph.D.
Date:	August 1, 1980
Contract No:	NGT-01-002-099 (University of Alabama)

Continuous Flow Electrophoresis:

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By

Thomas S. Jandebaur, Ph.D.
Assistant Professor of Biology
Athens State College
Athens, Alabama

ABSTRACT

The principle of continuous flow electrophoresis is as follows: a suspension of sample material is introduced as a fine, steady stream into a vertical, flowing "curtain" of electrolyte. A horizontal d.c. electrical field is applied to the curtain. Where a mixture of particles/cells having different electrophoretic mobilities is analyzed, each type particle/cell will assume a different, collectable band position in the curtain electrolyte.

The fundamental behavior of the continuous electrophoretic process in the Beckman Continuous Particle Electrophoresis (CPE) System, with flow in the downward direction, has recently been investigated. The purpose of this research is to determine the effect of sample concentration on throughput and resolution in the CPE System with flow in an upward direction. To maintain upward flow of the curtain buffer and sample stream, and to collect same, certain modifications to the CPE System were made; these also are described.

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Thank you, one and all.

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INTRODUCTION

As developed by Strickler⁵, Hannig² and their coworkers, continuous flow electrophoresis is a technique for separating small amounts of biological material. In this process, a suspension of sample is introduced as a fine, steady stream into a vertical, flowing "curtain" of electrolyte. A horizontal d.c. field is then applied to the curtain. Where a mixture of particles/cells having different electrophoretic mobilities is analyzed, each type particle/cell will assume a different, collectable band position in the curtain electrolyte/buffer.

The fluid containing the particles/cells to be separated moves through a thin chamber. The depth of the chamber is dictated by a need to suppress temperature gradients within the chamber and by a need to reduce chamber wall effects, both of which generate undesirable flow characteristics. As a result of these considerations, so-called "narrow gap" (0.5 mm - 3.0 mm) chambers lack sufficient sample capacity and/or resolving power, particularly with respect to separating cell populations.

OBJECTIVES

Though it is unlikely to improve resolution, increasing particle/cell concentration in the sample should increase sample throughput. However, higher cell concentrations lead to a density mismatch between sample stream and curtain buffer. Small amounts of sedimenting sample, diluted by the curtain buffer subsequently are rapidly displaced within the curtain. This results in broadening of the sample stream and ultimately in poorer resolution at higher sample concentrations.

Tests conducted by McDonnell-Douglas⁶ demonstrated upflow machines to be much more sensitive to the described density mismatch than downflow machines, further limiting sample stream concentration, resolution and throughput of material.

The objective of this study, then, is to determine the limits to throughput and resolution effected by sample concentration in the Beckman Continuous Particle Electrophoresis (CPE) System with flow in the upward direction. The data may then be compared to similar data exacted from the same CPE System, but with flow in the downward direction.

To maintain upward flow of the curtain buffer and sample stream, and to collect the sample, certain modifications to the CPE System were necessary; these will be described.

MATERIALS AND METHODS

To regulate and maintain upward flow of the curtain buffer and sample stream, a 0-140 volt output powerstat was used in tandem with a variable speed fluid pump. Dilute R-1 buffer⁴ cooled to approximately 4°C was pumped through the curtain flow meter at approximately 28cc/min. with the powerstat set at 90 volts.

Sample injection was accomplished using a Sage Instruments Model 355 Syringe Pump. The sample, contained in a 3 cc syringe fitted with a 21 gauge needle and connected to the sample inlet tube by a short piece of PVC tubing, was injected into the electrophoretic cell at an approximate rate of 1 ml/hr (range 1/100 at 24.3% flow). To prevent sedimentation of the sample contained in the syringe, a small stirring bar was placed within the syringe. A magnetic stirrer supported by the syringe pump housing was then inverted over the syringe.

A prototype of the collection system depicted by Figures 5 and 6 facilitated sample collection. The CPE System collecting tubes were connected to the collection bench by 10 inch lengths of 17 gauge PVC tubing fitted with 18 gauge needles at the tubing's distal end. The needle-containing tubes were seated in the 48 hole array of the collection bench. A 48 testtube fraction collector was placed below the collection bench to receive the eluted sample.

The d.c. field was generated by a Hewlett Packard Model 712B power supply; 65 volts/cm constituted the field for each experimental separation.

Concentrated R-1 buffer was used in the electrode rinse and coolant loops. Dilute R-1 buffer (800 ml R-1 concentrate added to 1.2 liters of deionized water) was used as curtain buffer.

To establish the effect of concentration on throughput and resolution in this CPE system, fixed turkey and cow red blood cells were used. These cells are stable, well-characterized standards, having different electrophoretic mobilities; further, they are morphologically distinguishable⁴.

DATA ANALYSIS

Two data sets were collected; the first data set consisting of seven "runs" at sample concentrations ranging from 5×10^7 cells/ml to 2×10^9 cells/ml (Table 1; Figure 3); the second data set consisting of eight runs at the aforementioned seven concentrations plus one run at a sample concentration of 1.5×10^9 cells/ml. A 5-minute stabilizing flow followed by 10 minutes of sample collection without (zero) and with (field) voltage applied constitutes a "run".

Aliquots of the collected sample were examined microscopically; recovered cells were counted using an Improved Neubauer hemocytometer. Percent recoveries for each cell type and for the combined cells were then determined (Table 1). Based on the total number of recovered cells of each type and of the combined cells, percent relative abundances of the recovered cells were determined. This data, plotted for both data sets, is presented in Figures 1 and 2 as percent recovery of fixed cells vs. concentration, and in Figures 3 and 4 as percent relative abundance of recovered fixed cells vs. tube number.

DISCUSSION OF RESULTS

Examination of Table 1 (1st data set) and Figure 1 reveals that, in general, more cells are recovered at lower concentrations than are recovered at higher concentrations, either with (field) or without (zero) voltage applied. This may be explained as a sedimentation effect; i.e., higher cell concentrations engender a density mismatch between sample stream and curtain buffer. Small amounts of sedimenting sample, diluted by the curtain buffer subsequently are rapidly displaced within the curtain. This results in broadening of the sample stream, "spreading" of cells over a broader range of sample collection test tubes (see higher concentrations Tables 2 and 3 and higher concentrations Figures 3 and 4) and ultimately to poorer sample resolution. This interpretation is supported by the lowered recovery of the larger, heavier turkey cells at higher concentrations (Figure 1) and by the enhanced recovery of the smaller, lighter cow cells at higher concentrations (also Figure 1). In part, the apparent increased recovery of cow cells at higher concentrations is, perhaps, due to examining and counting a greater number of sample collection test tubes. In general, the data of Table 1 (2nd data set) and Figure 2 also fit this interpretation, but with notable exception: the turkey cells do not sediment at higher concentrations. It is, perhaps, significant that the electrophoretic cell was coated with 1% bovine serum albumin, the electrode membranes and electrode rinse replaced, and the buffer temperatures lowered to approximately 4°C between collection of the first and second data sets. These measures, taken to improve CPE chamber performance, may have inadvertently altered the previously noted turkey cell sedimentation behavior.

Tables 2 and 3 and Figures 3 and 4 reveal "spreading" of the sample over a widening range of sample collection test tubes with increasing concentration. This is especially evident at the higher concentrations (1.5×10^9 and 2×10^9 cells/ml) at which the sample stream was observed to "fall back" on the inlet tube; periodically, "clouds" of the sample billowed upward in the buffer flow apparently being transported more rapidly than cells remaining in the sample stream.

CONCLUSIONS

With particular reference to Table 3 and Figure 4, data collected after treating the electrophoretic cell as described in Discussion of Results, and with voltage applied, it appears that maximum resolution in the Beckman CPE System, with flow in an upward direction, is achieved at concentrations ranging from 2×10^8 cells/ml to 8×10^8 cells/ml. The widest peak separation is at 2×10^8 cells/ml; however, the sharpest peaks and least overlap between cell populations is at 8×10^8 cells/ml.

Apparently as a result of improved electrophoresis cell performance due to coating the chamber with bovine serum albumin, changing the electrode membranes and rinse, and lowering buffer temperatures, sedimentation effects attending to higher concentrations are diminished.

As measured by recovery of fixed cells, throughput is greatest at lower sample concentrations (ranging from 5×10^7 to 2×10^8 cells/ml in this study), diminished at moderate concentrations (4×10^8 to 1×10^9 cells/ml) and increasing at higher concentrations (1.5×10^9 to 2×10^9 cells/ml) although not increasing to the higher recovery levels noted for lower sample concentrations.

Throughput as measured by recovery of fixed cells is, then, diminished at the concentrations judged most likely to yield satisfactory resolution. The "tradeoff" appears to be as follows: improved recovery/throughput at the expense of resolution.

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TABLE 1. RECOVERY OF FIXED CELLS AT INDICATED CONCENTRATION WITH (FIELD) AND WITHOUT (ZERO) VOLTAGE APPLIED FOR FIRST (1) AND SECOND (2) DATA SETS

Concentration (Cells/ML)	Data Set	Field			Zero		
		% Recovery			% Recovery		
		Comb.	Turkey	Cow	Comb.	Turkey	Cow
5 x 10 ⁷	1	87.22	77.4	98.04	92.88	108.36	77.4
	2	76.38	64.06	89.7	98.56	118.28	78.84
1 x 10 ⁸	1	65.79	69.66	61.92	74.82	98.04	51.6
	2	68.99	81.32	56.67	70.22	86.24	54.2
2 x 10 ⁸	1	74.18	83.84	64.5	81.27	81.28	81.28
	2	69.61	80.08	59.14	53.59	46.82	60.36
4 x 10 ⁸	1	55.15	57.4	52.88	53.86	50.96	56.76
	2	43.12	35.72	50.52	43.12	37.58	48.66
8 x 10 ⁸	1	48.86	45.8	51.92	42.73	42.24	43.22
	2	48.05	53.9	42.2	30.18	32.64	27.72
1 x 10 ⁹	1	50.31	37.66	62.96	39.22	28.9	49.44
	2	29.81	15.28	44.36	48.17	28.08	68.26
1.5 x 10 ⁹	2	54.7	71.95	37.45	27.6	31.7	23.48
2 x 10 ⁹	1	53.66	26.44	80.88	41.34	29.02	53.66
	2	58.27	60.98	55.56	54.45	53.1	55.8

TABLE 2. PERCENT RELATIVE ABUNDANCE OF RECOVERED CELLS BY TUBE NUMBER AT INDICATED CONCENTRATION (CELLS/ML.) WITH (FIELD) AND WITHOUT (ZERO) VOLTAGE APPLIED FIRST DATA SET

Tube #	5 x 10 ⁷				1 x 10 ⁸				2 x 10 ⁸				4 x 10 ⁸			
	Turkey		Cow		Turkey		Cow		Turkey		Cow		Turkey		Cow	
	Zero	Field	Zero	Field	Zero	Field	Zero	Field	Zero	Field	Zero	Field	Zero	Field	Zero	Field
21		0														
22		13.33				0										
23		13.33		0		7.41										
24		46.67		10.53		14.82		0			0					
25		13.33		10.53		40.74		4.17		0		2.00		0		0
26		13.33		31.58		29.63		11.11		7.69		3.08		31.46		1.22
27		0		42.10		7.41		29.63		32.31		0		61.80		4.88
28	4.76			5.26		0		40.74		53.85		2.00		6.74		51.22
29	0		0	0		0		0		4.62		29.23		0		35.37
30	4.76		20.00			0		3.70		1.54		24.62	1.27	0		7.32
31	57.14		33.33		0	5.00				0		15.38	35.44		35.23	0
32	19.05		40.00		26.32	20.00			3.18	1.54			58.23		60.23	
33	14.29		0		57.90	55.00			31.75		0		5.06		4.54	
34	0		6.67		15.79	10.00			49.21		31.75		0		0	
35			0		0	10.00			15.87		56.92					
36						0			0		9.23					
											0					

Table 3. PERCENT RELATIVE ABUNDANCE OF RECOVERED CELLS BY TUBE NUMBER AT INDICATED CONCENTRATION (CELLS/ML.) WITH (FIELD) AND WITHOUT (ZERO) VOLTAGE APPLIED: SECOND DATA SET.

Tube #	5×10^7				1×10^8				2×10^8				4×10^8			
	TURKEY		COW		TURKEY		COW		TURKEY		COW		TURKEY		COW	
	zero	field	zero	field												
13																
14																
15																
16																
17																
18																
19																
20																
21																
22																
23																
24		0								0						
25		7.69				0				3.08						
26		23.08				9.09			0	13.85				0		
27		46.15		0		42.42		8.70		43.08		0		1.72		
28		23.08		61.11		36.36		17.39		21.54		2.08		36.21		0
29		0		27.78		0	9.09		34.78	0		47.92		13.79		39.02
30				11.11		2.86	3.03	0	39.13			16.67		0		46.34
31			0	0		5.71	0	18.18	0	0		0	2.08		0	10.98
32		0		4.17		45.71		50.00		7.89		6.12		0		1.27
33		56.25		50.00		42.86		31.82		36.84		18.37		16.40		16.46
34		37.5		45.83		2.86		0		36.84		53.06		70.49		59.49
35		6.25		0		0				18.42		22.45		13.11		22.78
36		-								0		0		0		0
37																
38																
39																
40																
41																
42																
43																
44																

Tube #	8×10^8				1×10^9				1.5×10^9				2×10^9			
	TURKEY		COW		TURKEY		COW		TURKEY		COW		TURKEY		COW	
	zero	field	zero	field	zero	field	zero	field	zero	field	zero	field	zero	field	zero	field
18									0							
19									0.68							
20									0.46							
21									0				0			
22									0		0		0.20			
23							-		0.91		0.88		0.81			
24							1.12		2.28		0.44		0.40			
25		0					0		5.48		0		0.81			
26		2.29				0			20.78		0	0	1.62			
27		14.86				11.29			26.94		1.75	0.23	4.24	0	0	
28		62.86		0		51.62		0.56	18.95		11.84	0.23	15.35	0.44	0.22	
29		15.43		5.84		24.19		0.25	17.35		23.25	3.02	23.03	2.43	1.33	
30		4.57		65.69		12.90		27.53	0	4.57	0	26.75	10.90	45.66	12.58	12.42
31		0		22.63		0		35.39	10.35	1.37	3.50	29.39	25.99	7.68	25.39	45.23
32	0		0	5.11	0		0	32.58	1.04	0	27.27	3.07	34.34	0.20	30.46	29.49
33	4.72		8.89	0	1.75		2.89	1.12	0.52	0.23	11.89	2.19	19.95	0	22.52	9.76
34	83.02		68.89	0.73	22.81		26.35	0.56	0	0	3.50	0	5.10		6.18	1.55
35	-		-	-	62.28		57.04	0	56.48		37.76	0.44	0.23		0	0
36	12.26		20.00		12.28		12.64		26.42		13.29	0	-			
37	0		2.22		0.88		1.08		1.04		2.10					
38			0		0		0		2.07		0					
39									2.07		0.70					
40									0		0					

Figure 1. Percent Recovery of Fixed Cells vs. Concentration with (Field) and without (Zero) Voltage Applied: First Data Set.

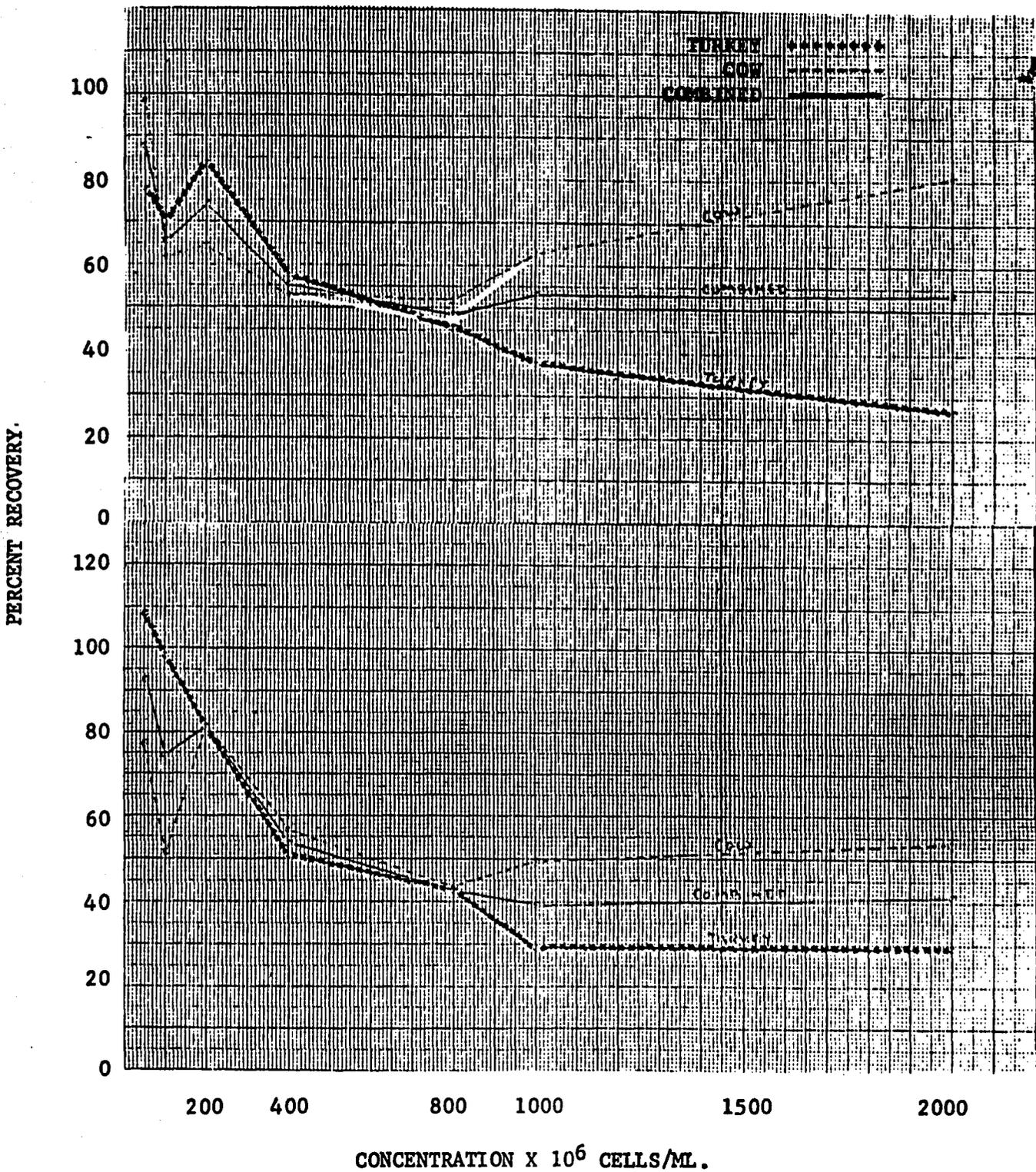


Figure 2. Percent Recovery of Fixed Cells vs. Concentration with (Field) and without (Zero) Voltage Applied: Second Data Set.

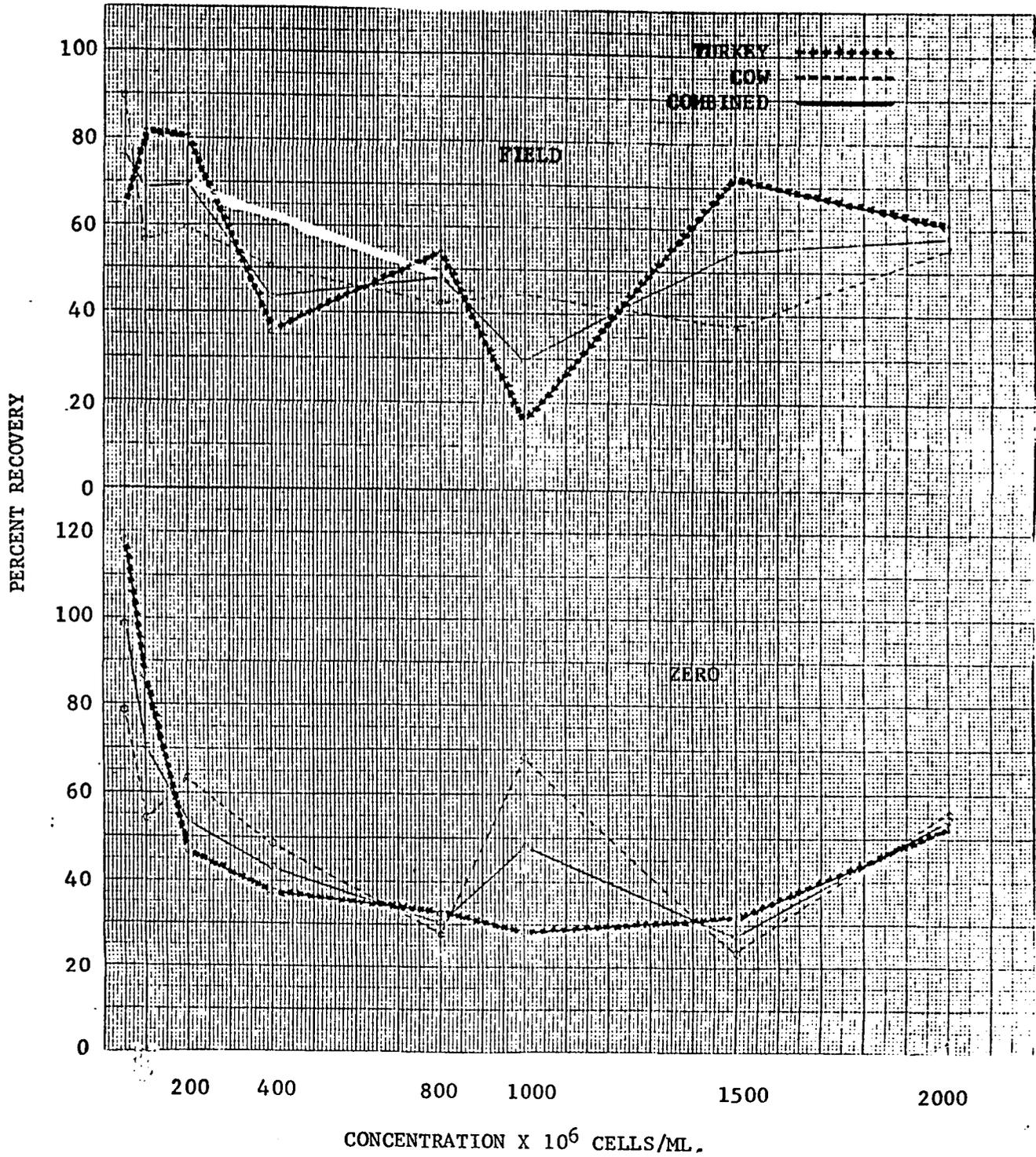


Figure 3. Percent Relative Abundance of Recovered Fixed Cells vs. Tube Number at Indicated Concentration With (Field) and Without (Zero) Voltage Applied: First Data Set.

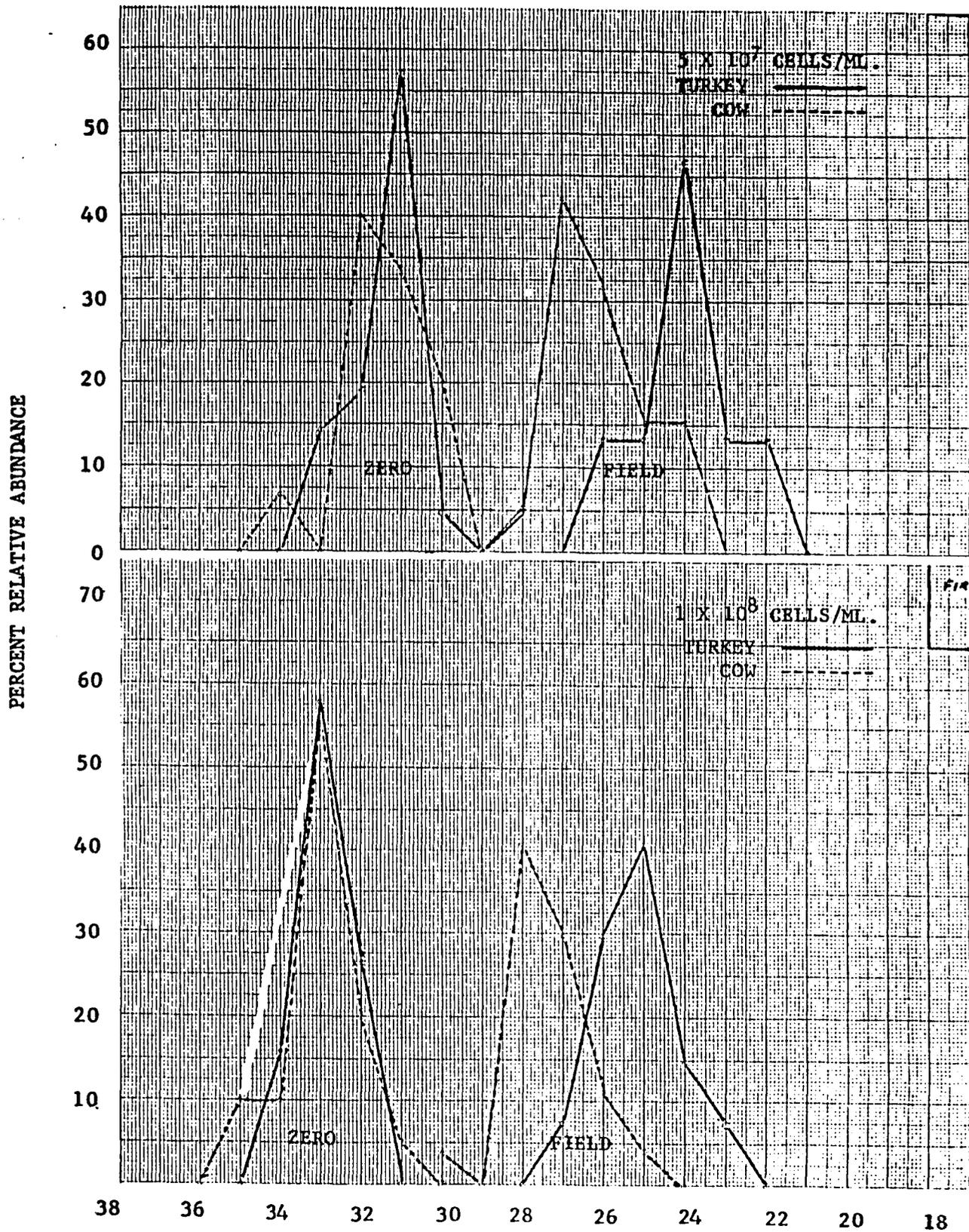
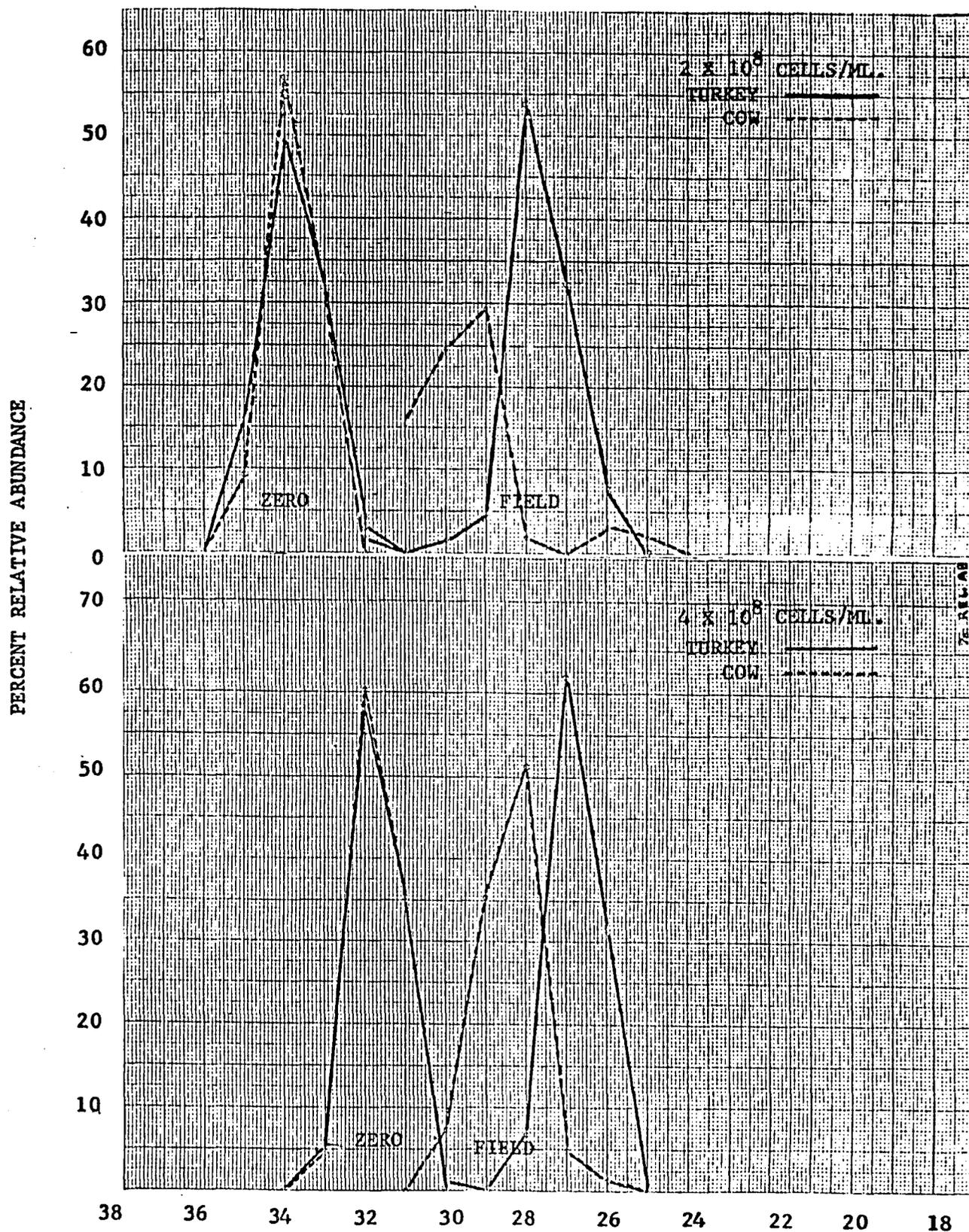


Figure 3. Continued



TUBE NUMBER

X-19

Figure 3. Continued

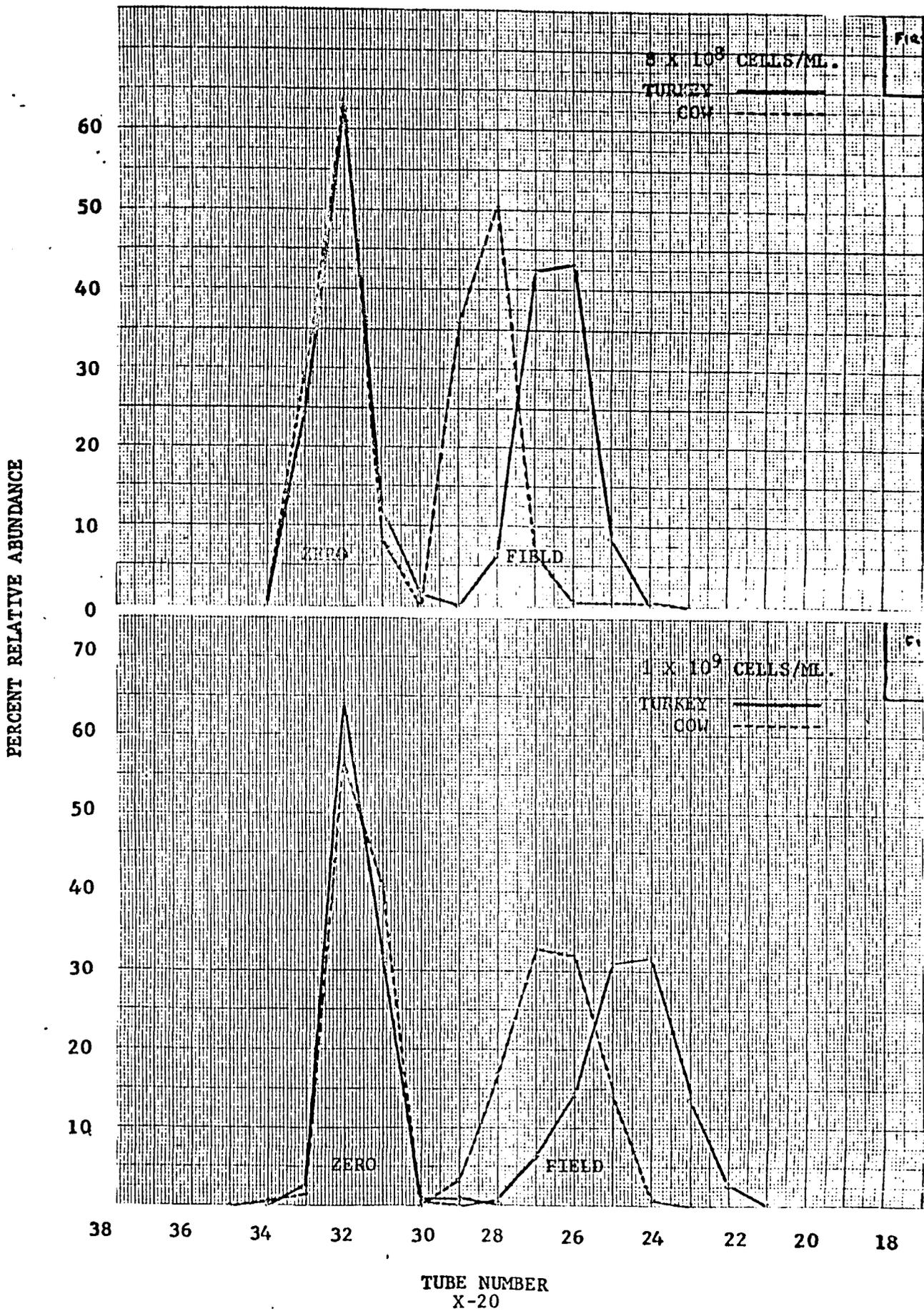


Figure 3. Continued

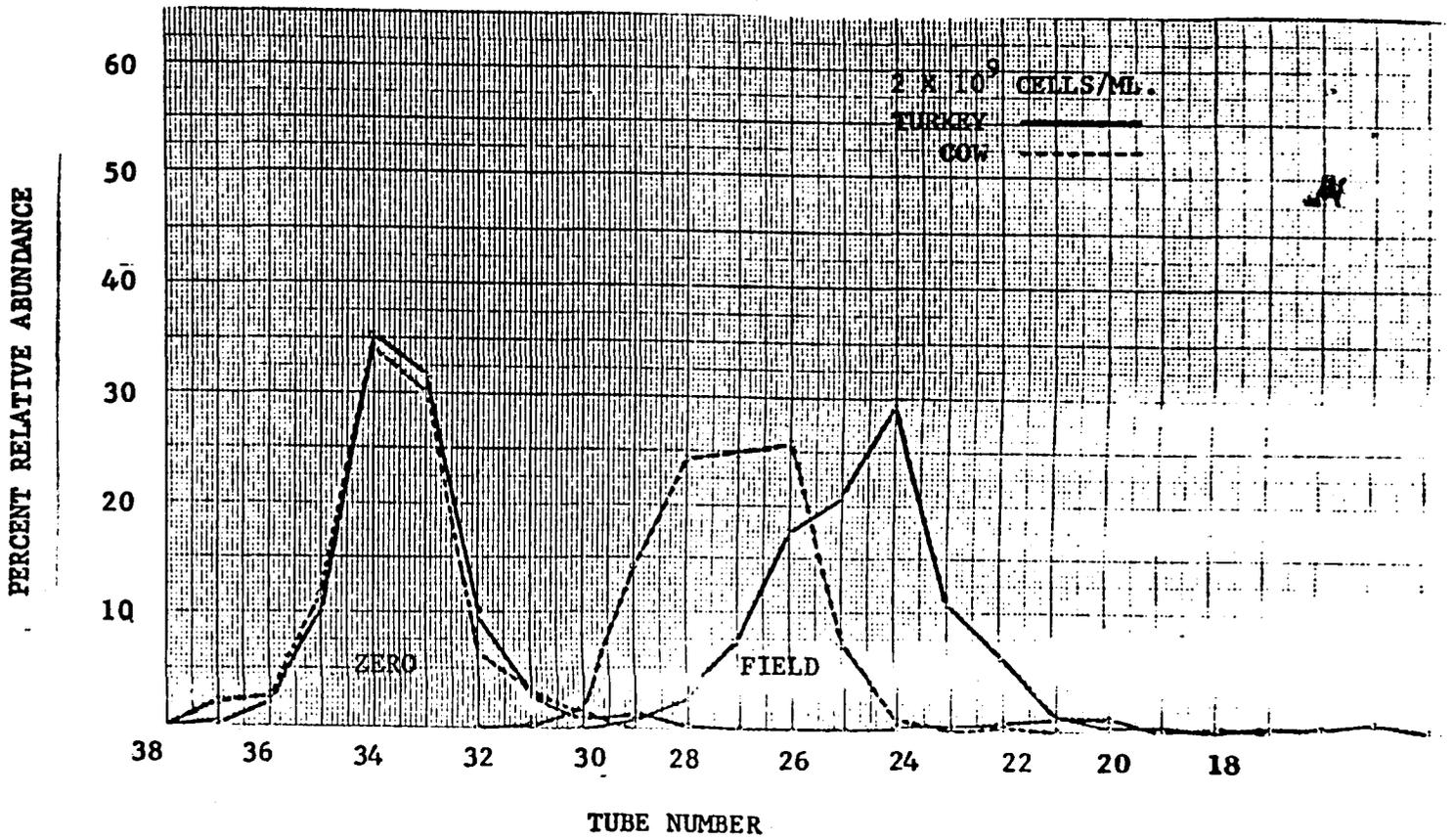


Figure 4. Percent Relative Abundance of Recovered Fixed Cells vs. Tube Number at Indicated Concentration With (Field) and Without (Zero) Voltage Applied: Second Data Set.

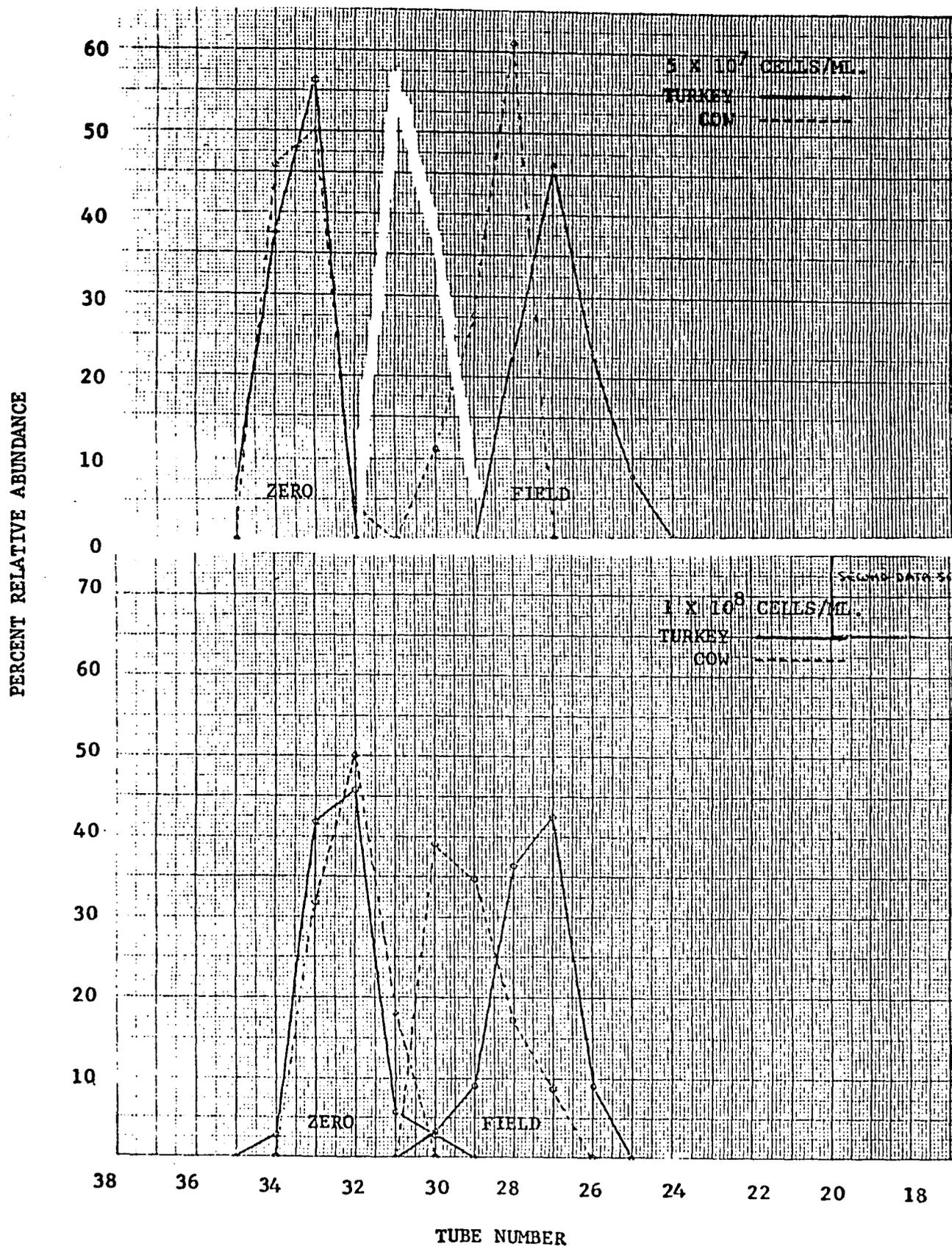


Figure 4. Continued

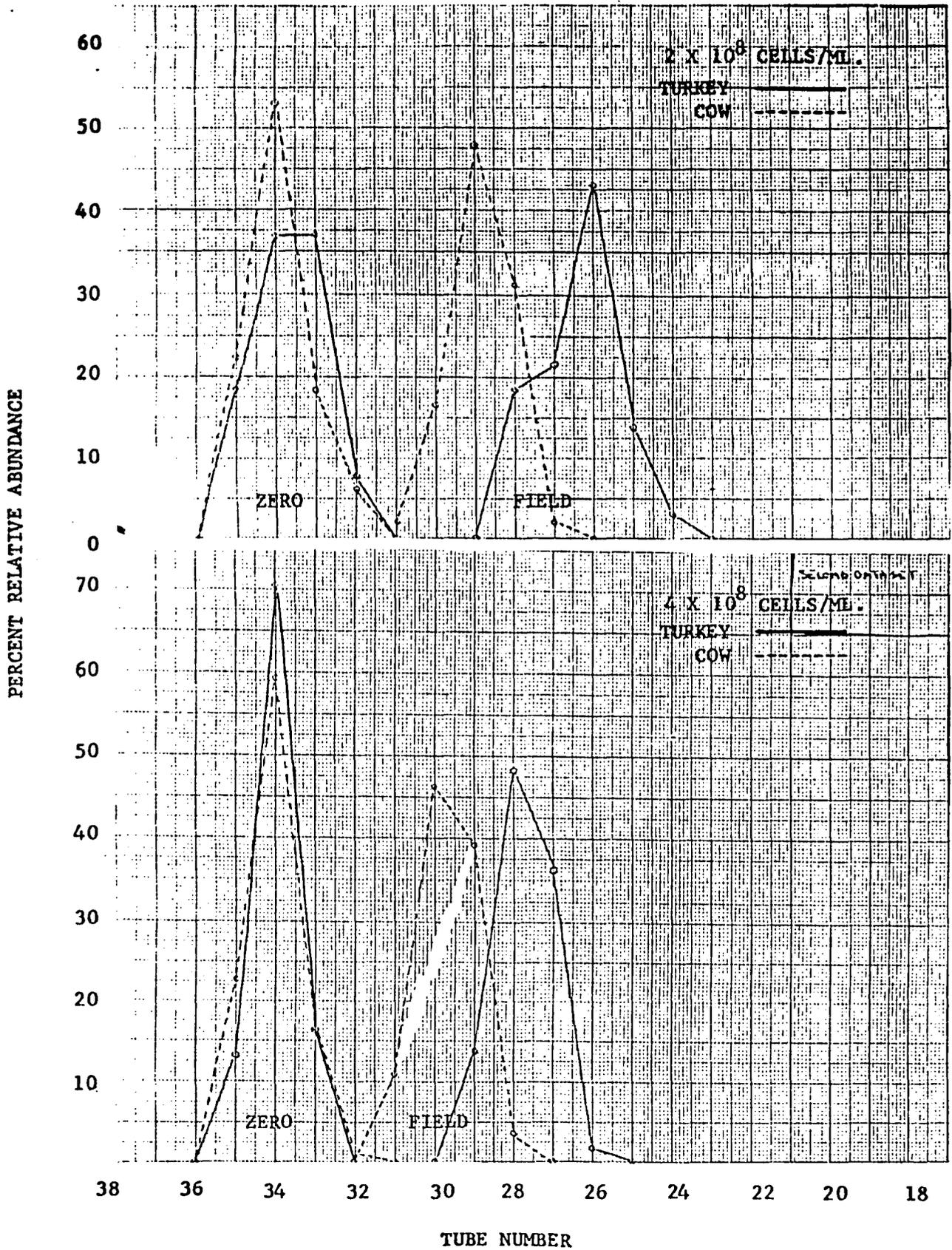


Figure 4. Continued

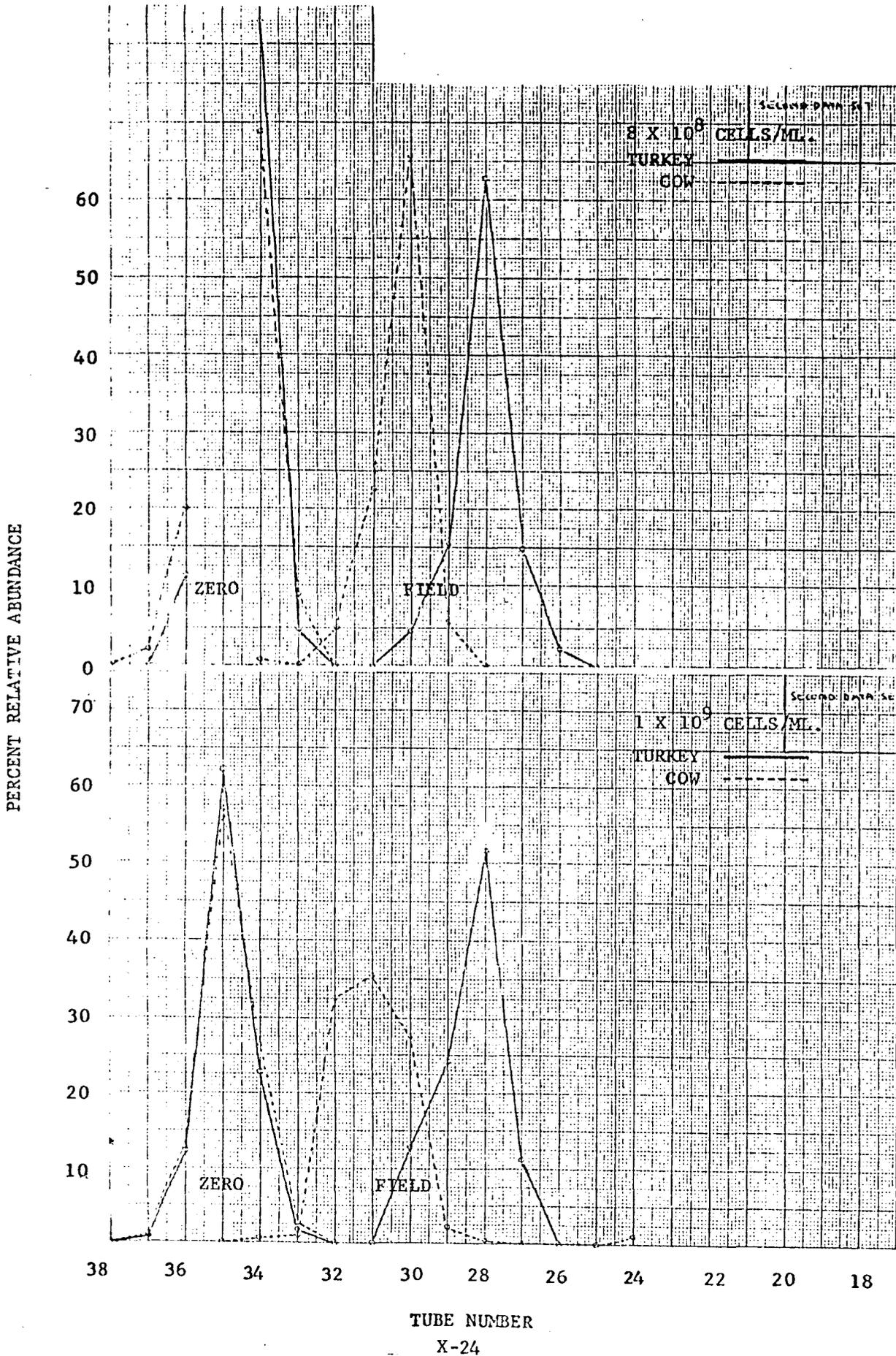


Figure 4. Continued

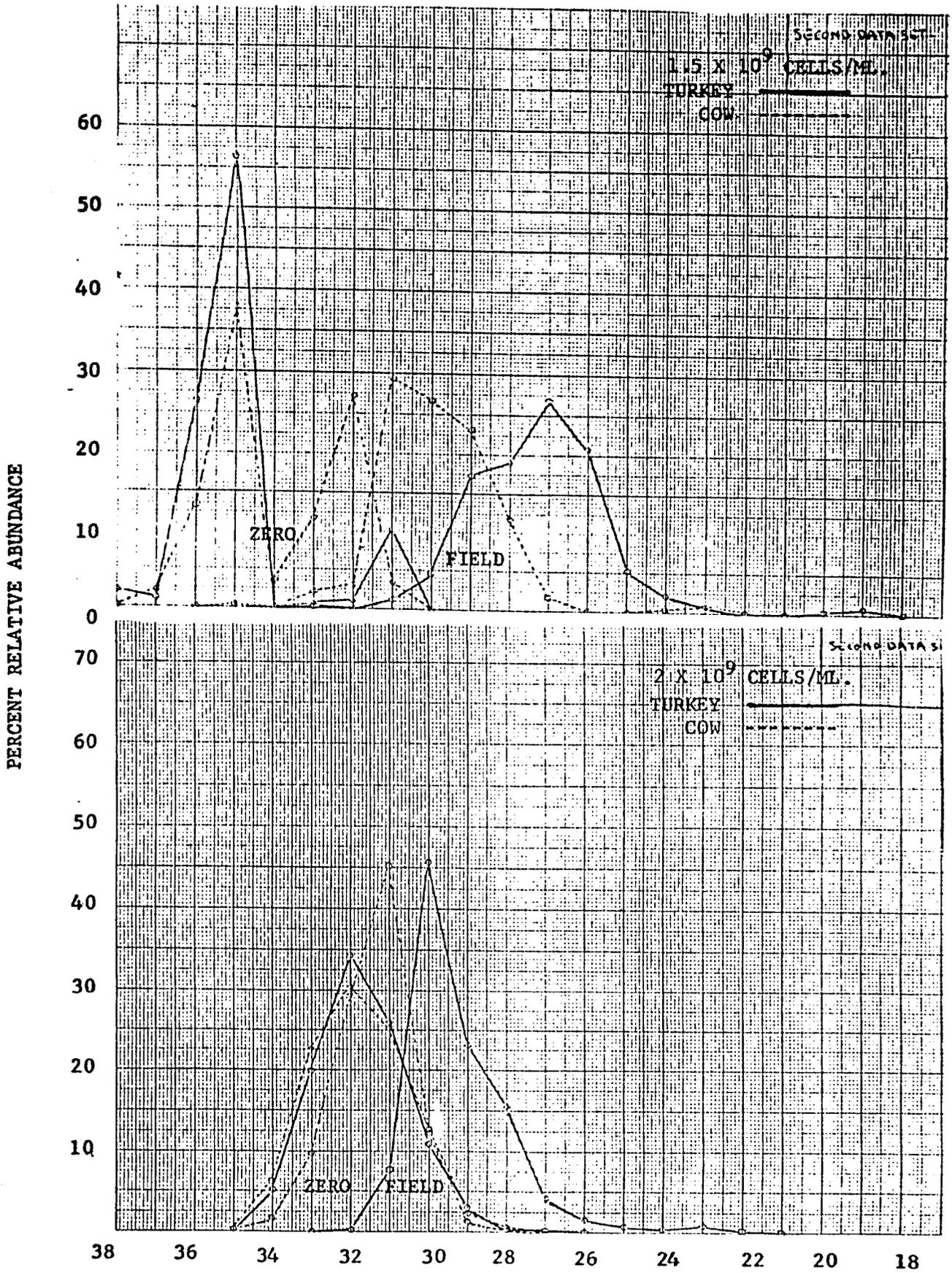


Figure 5. Continuous Particle Electrophoresis (CPE) System:
Collection Bench for Upward Flow.

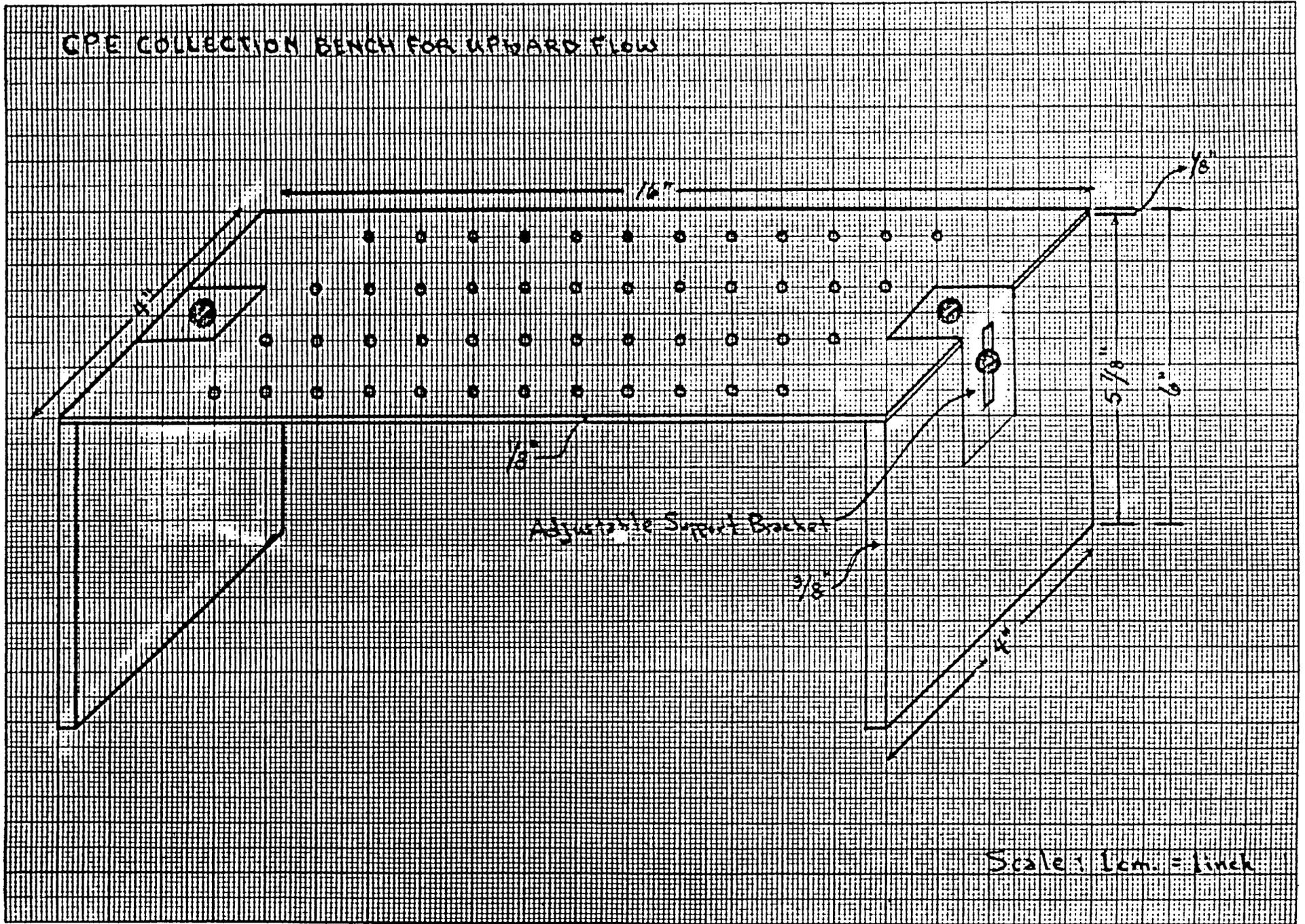
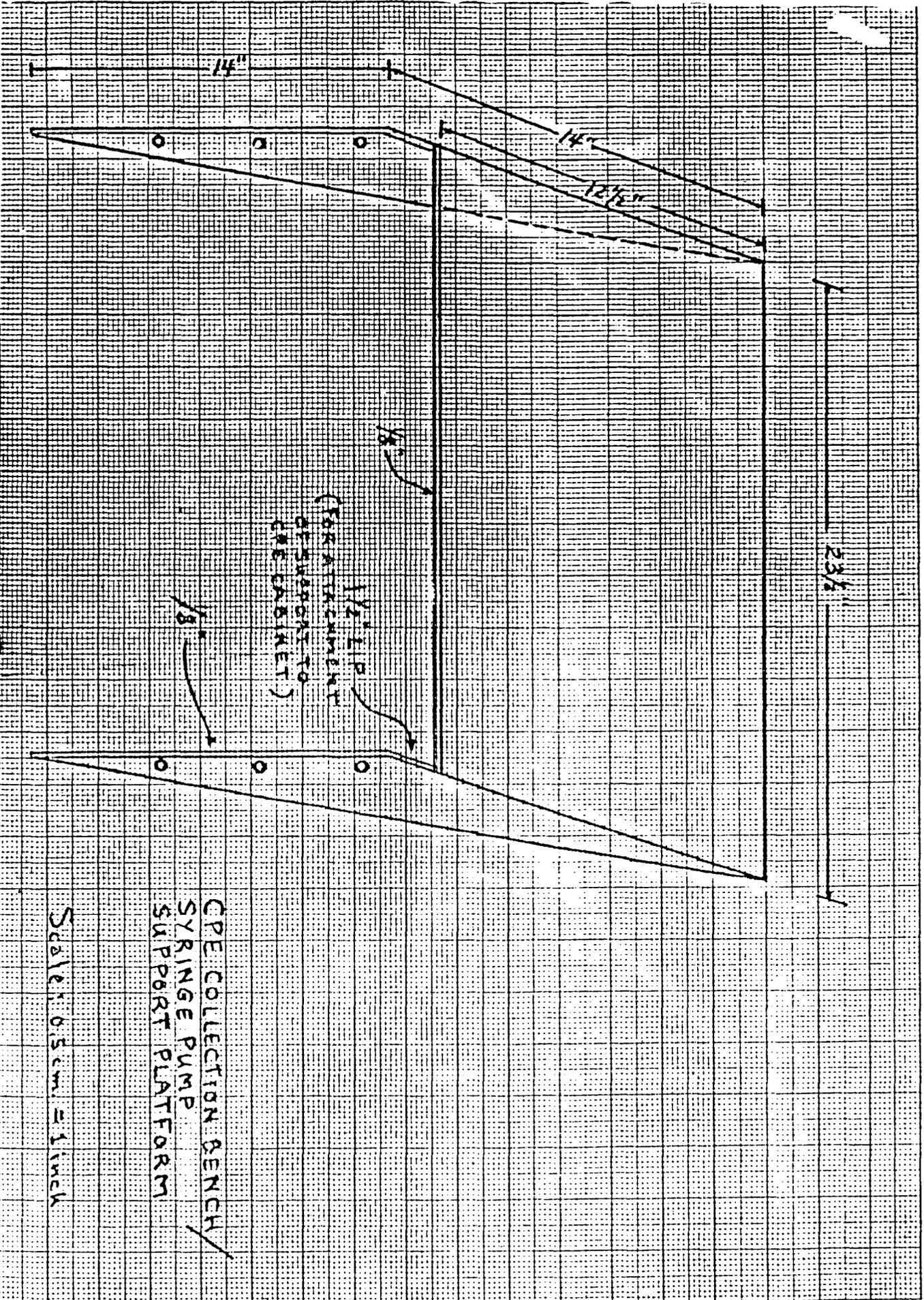


Figure 6. Continuous Particle Electrophoresis (CPE) System: Collection Bench/Syringe Pump Support Platform.



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MARSHALL SPACE FLIGHT CENTER
THE UNIVERSITY OF ALABAMA

MODELING A MAINTENANCE
SIMULATION OF THE GEOSYNCHRONOUS PLATFORM

Prepared by:	A. F. Kleiner, Jr.
Academic Rank:	Professor
University and Department:	Drake University Department of Mathematics and Computer Science
NASA/MSFC: (Office) (Division) (Branch)	Preliminary Design Mission Analysis and Integration Operations Analysis
MSFC Counterpart:	Dr. J. Steincamp
Date:	August 1, 1980
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by

A. F. Kleiner, Jr.
Professor of Mathematics
Drake University
Des Moines, Iowa

ABSTRACT

With the advent of the Space Transportation System it will be possible to service equipment in space. In particular the effect of a maintenance capability for the Geosynchronous Platform (GSP) needs to be examined. Because of the size and complexity of the GSP, analytic results in this area are beyond reach. Hence a simulation study comparing various maintenance routines appears to be the most fruitful approach. The purpose of this report is to outline the basic model which will be used to conduct such a study. Information is included on the computer model and maintenance. A sample system is also modeled.

Acknowledgements

The 1980 NASA/ASEE program has provided me with many benefits both professional and personal. I would like to thank the following people for their assistance this summer: Jim Steincamp, my counterpart, for providing me with an interesting problem and the time to understand it; John Cole, Chief of the Operations Analysis Branch for providing support for my work; Dr. Barfield and Marion Kent for the running of the program; Ronnie Barnes, John Springer and Hank Stoll, fellow participants, for a stimulating carpool and friendship for 8 weeks; Dawn Walton, Carol Byler and Janet Coker for typing this report, and most importantly Mary, Alex and Mimi for making a home away from home for the summer.

1. Introduction

With the advent of the space transportation system NASA will be able to implement a new approach to the reliability of space missions. Rather than build for the mission life from the start it will be possible to visit spacecraft for the purpose of servicing the equipment. The Geosynchronous Platform (GSP) provides a particularly good example of the possibilities of this service for two reasons: (1) GSP will have an extended mission life, 10 to 16 years, requiring heavy redundancy costs if maintenance is not available and (2) GSP will serve the communications industry with projected increases in demand over the course of the mission, thus the ability to increase capability is a valuable option.

An analysis of service for GSP is to be carried out to determine if maintenance is a viable concept and to aid in the scheduling of such if it proves feasible. One approach to this problem is to seek an analytic solution by techniques in decision analysis, such as in [1]. The size of this problem however precludes such an analysis (see for example the "small" sample system in Chapter 3). A numerical approach to the same problem can be developed by means of simulation.

Since the problem of maintenance will arise in other settings besides GSP it is useful to develop a general simulation method. Such a method should allow for the incorporation of special features which arise in individual problems such as the need to increase capacity that is an important part of the GSP. This method should be able to accept different mission types and different maintenance policies and produce data which indicate which policies are better.

This report is a preliminary one on work done to develop such a general method. The method is based on earlier work of Dr. J. Steincamp of the Marshall Space Flight Center. Specific details of the method may be found in [3].

2. Objectives

The immediate objective is to construct a simulation model which can be used for an analysis of maintenance for the GSP. It is required to find out if maintenance is an economic approach and if so how it should be carried out.

A more general objective is to develop a method which can answer the above questions for many different space missions. The method is to accept as input the particular system and be capable of running the system through a mission life with different maintenance policies.

This report contains an overview of such a method. The system model is described and illustrated, the basic concepts of a simulation pass are detailed and sections on failures and maintenance are included.

3. The System Model

3.1 Introduction

An operating, physical system may be considered from many points of view. For the purpose of a maintenance simulation two such points of view are required. When a piece of equipment fails it must be replaced, so its location and accessibility are important. Hence a geometric or physical model of the system which details the replacement relationships among the parts is necessary. More over, in order to carry out a program of part failures on the system, it is necessary to have a model of the system's functional or operational mode.

The operation of the system requires that a physical part fail, the operational effects of this failure be carried out and a determination of the need for maintenance of the system be checked. Hence the physical and operational models must be interrelated. The interrelation will be carried out at the most elementary level of the system by identifying the individual parts which fail, called items, with the lowest level operations carried out by the system, called basic functions. Hence forth the dual nature of these items/basic functions will be emphasized by the name IBF. Thus an IBF can be considered either as a specific physical part which supports a basic function of the system or as a basic operation which is performed by some physical item.

3.2 The Physical Model

Inspace fine tuning of systems will not be available in the foreseeable future. Rather, when a part fails some package which contains the failed part will be replaced. This package may contain other parts which have failed as well as good ones. Such a package is called a module. More generally, a module is a replaceable or stockable unit which may contain both items and other modules (sub modules). Each item belongs to same, smallest module. A module which contains only items is called a simple module, one which contains only other modules is called pure and a module which is neither simple nor pure is called mixed. The smallest module which can be used to replace the failed item is said to contain the item.

In a physical system with planned redundancy there will be many instances of duplication. Identical parts will be used in various places and essentially similar but distinct modules will be used for replacement purposes. Although duplicate items will have the same failure distributions, the randomness of the actual failures requires that each physical part be individually represented in the computer model. As a result replacement must be performed and recorded for each module rather than on the classes of similar types.

On the other hand it is desirable to take advantage of the similarities among the items and modules for efficient input of the system. For instance if only separate types of items and modules must be entered in the computer a good deal of labor can be saved. Also the identification of these duplicates can streamline ordering, packing and inventory procedures and costs. This identification is established by the concept of generic items and modules. The essential features of each type of item and module are entered into the machine and these types are referred to as generic. The specific items are then identified with the basic functions of the operational model to form the actual IBF's which form the base of the system. It is the IBF's which actually fail according to the failure rule prescribed for each generic item. The actual physical instances of the modules are created by the computer as copies of the appropriate generic module. Records of maintenance are kept for specific modules.

One further feature regarding the physical model of the system is useful. When an IBF fails there are several possibilities for replacement. The IBF may already be scheduled for replacement because it is contained in some module which is to be replaced as a previous failure or because its containing module is a submodule of same module scheduled for replacement. In this case the failure is noted but no adjustment need be made to the list of modules to be replaced. On the other hand if the module containing the IBF was not previously scheduled for replacement it is necessary to add it to the list of modules to be replaced. Moreover any module previously scheduled for replacement whose replacement will be included with the new module should be removed from this list. This procedure is accomplished by distinguishing between kitted and packed modules. A packed module is any module which is scheduled for replacement, either of its own right or because it is contained in some such module. A module is kitted only if it is to be replaced in its own right. Thus, if a module not on the packed list must be replaced, it is to be added to the kitted list, any module which will be replaced with it is removed from the kitted list and all such modules are put on the packed list.

3.3 The Operational Model

Each system has one overall mission or function to perform. This function is accomplished by subfunctions each of which contributes something to the operation of the whole system. Usually, each subfunction can be divided further into functions whose composition is the subfunction. This process of dividing functions into lower level functions until functions which are performed by the basic items of the physical model are reached. These functions, which are called basic functions, are identified with the physical items that perform them and form the IBF's upon which the system rests. Thus the operational mode of the system modeled by a treegraph. The nodes correspond to the various functions with the

root node representing the overall system function and the terminal nodes the IBF's.

Each function has a set of attributes which are used for defining the system state, processing failures and keeping records. The tree like structure of the system is defined by the attributes chief and base. The chief of a function is the immediate superior of that function in the tree while the base is the set of immediate subordinates of the function. The duty cycle of a function is that proportion of its chief's operating time which it must operate while the condition indicates if the function exists, is online, in standby or has failed. For the purpose of implementing failures, several attributes such as figure of merit, reserve and a user defined index are useful (these are illustrated in the section on failures). Attributes such as operating life, on line time, number of times failed and time to first failure are useful for record keeping and statistics gathering. Each basic function also has a set of failure distributions which are used to cause failure events (see the section on failures).

3.4 A Sample Model

Some of the preceeding concepts are illustrated by the sample model with function tree as displayed in Figures 1-4 and module definitions as in Table 1. This model is for a simplified GSP type system with 119 IBF's, 5 generic modules and 200 functions. Note that in this model several of the modules contain only one IBF and since there is no nesting of modules the distinction between kitted and packed modules does not arise. Each node of the function tree is identified by name and initial duty cycle modifier. Failure rules are included for the non basic functions. The failure distributions for the IBF's are indicated in Table 2.

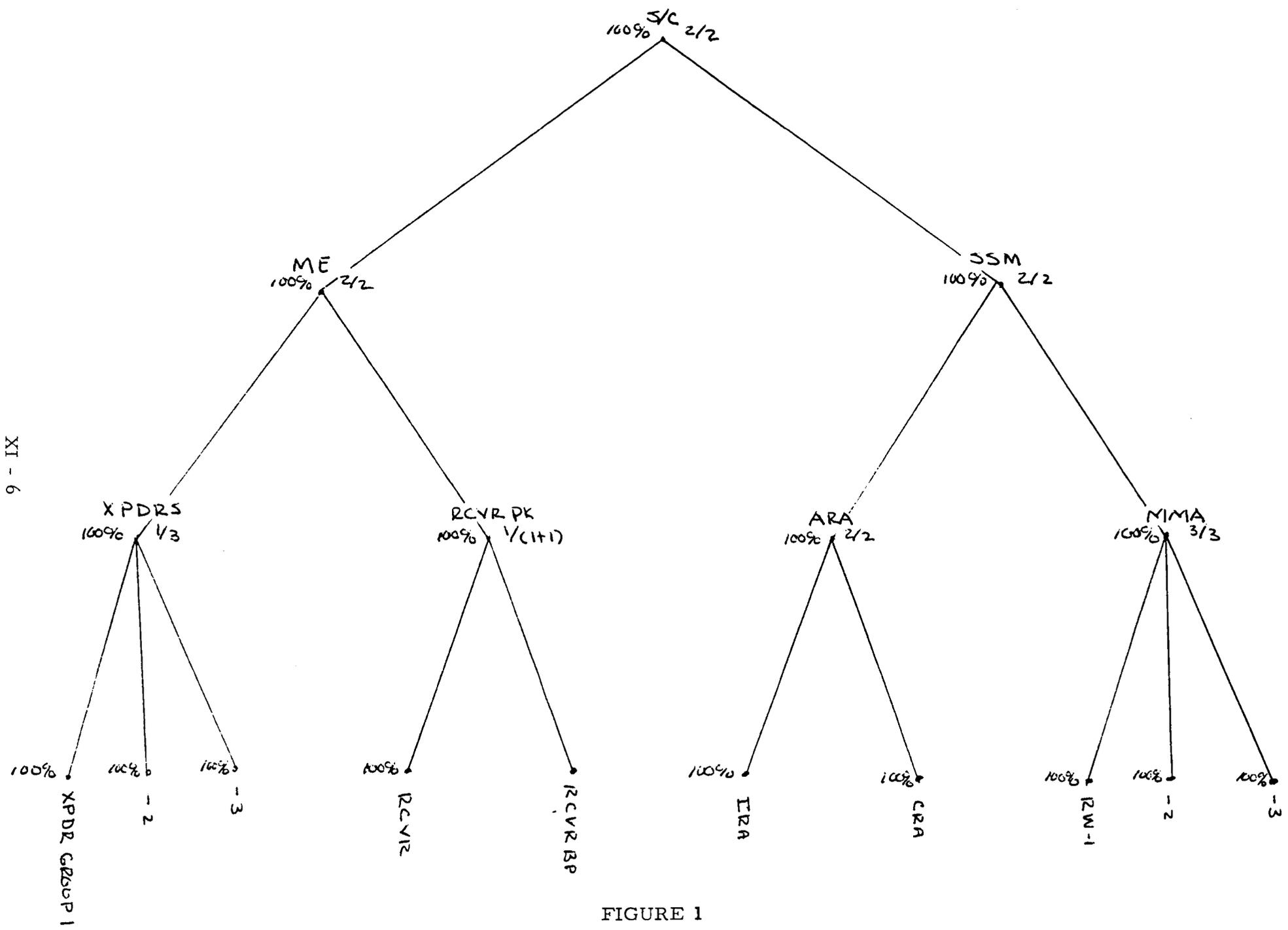
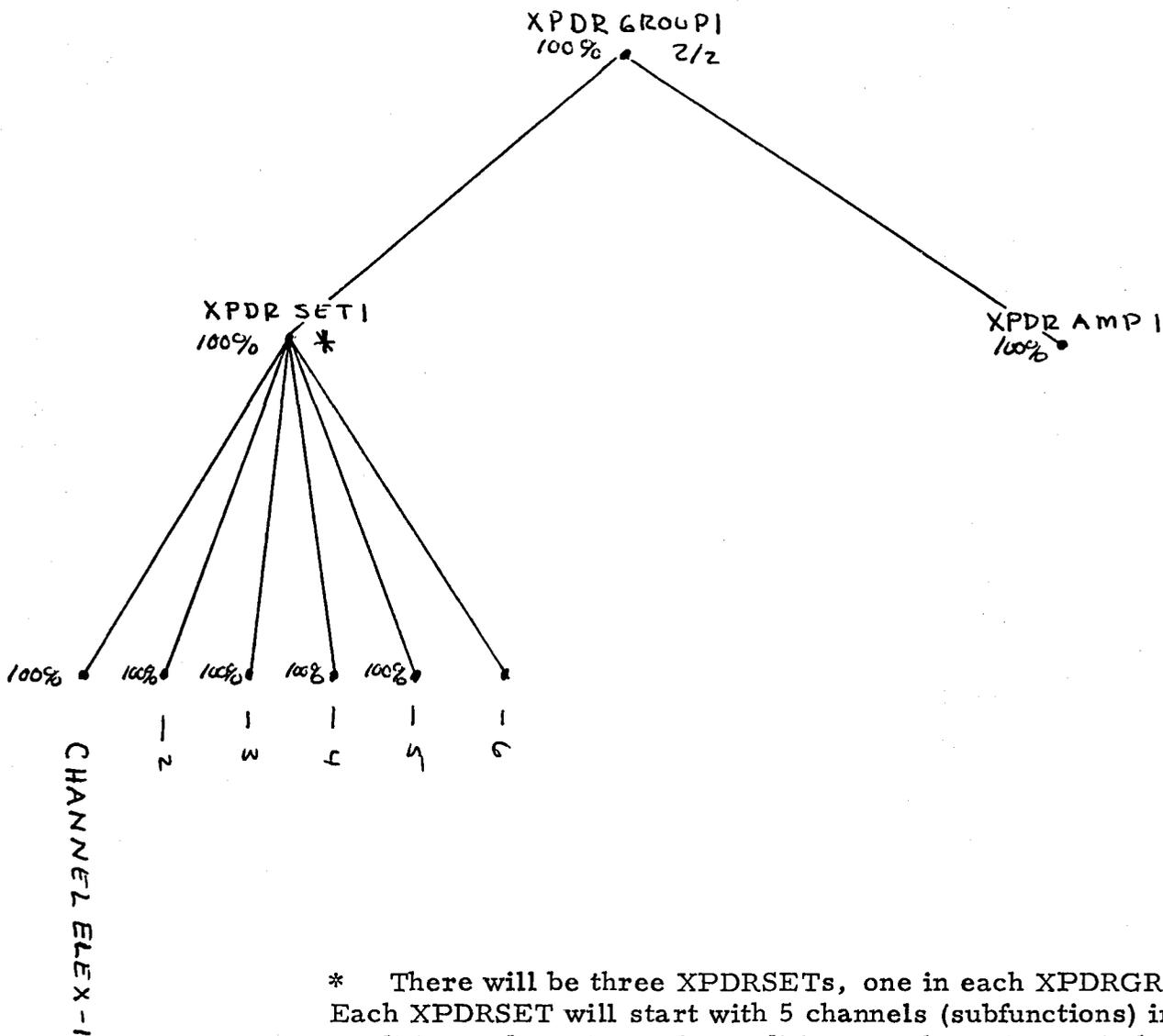


FIGURE 1

PARTIAL FUNCTION TREE - SAMPLE SYSTEM



* There will be three XPDRSETS, one in each XPDRGROUP. Each XPDRSET will start with 5 channels (subfunctions) in on condition and one in ready condition. A channel is switched from ready to on if (a) one of the other 5 subfunctions of its chief fails or (b) a channel fails in some other XPDRSET and no backup is left in that set. The XPDRSET will stay on as long as at least one of its channels is on.

FIGURE 2

XPDR GROUP TREE - SAMPLE SYSTEM

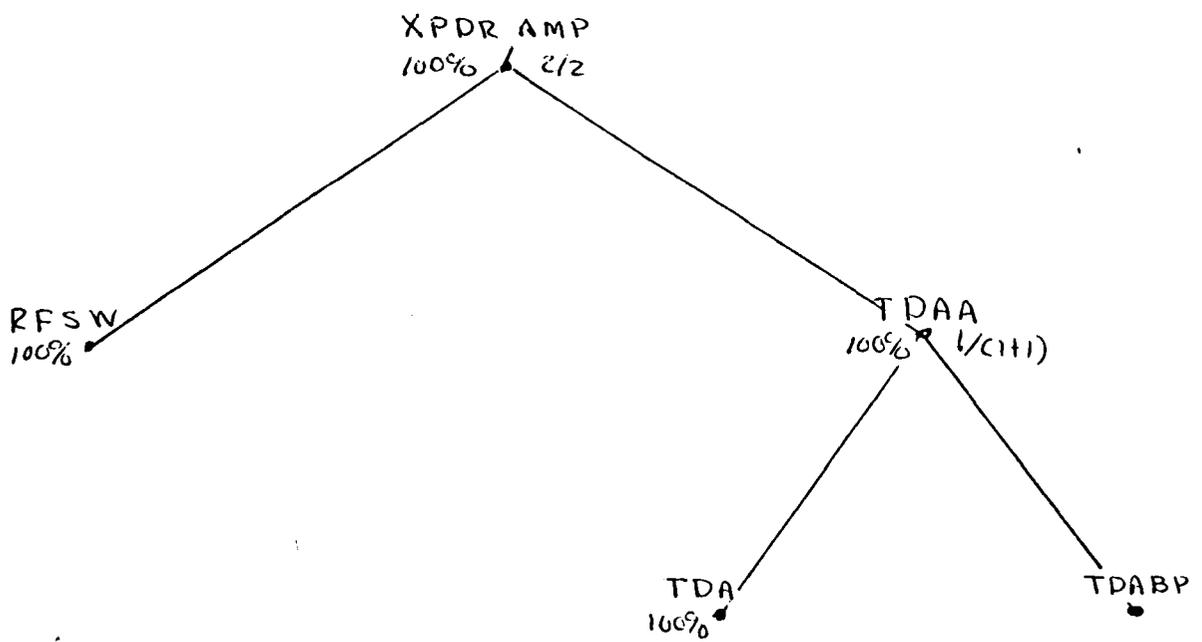
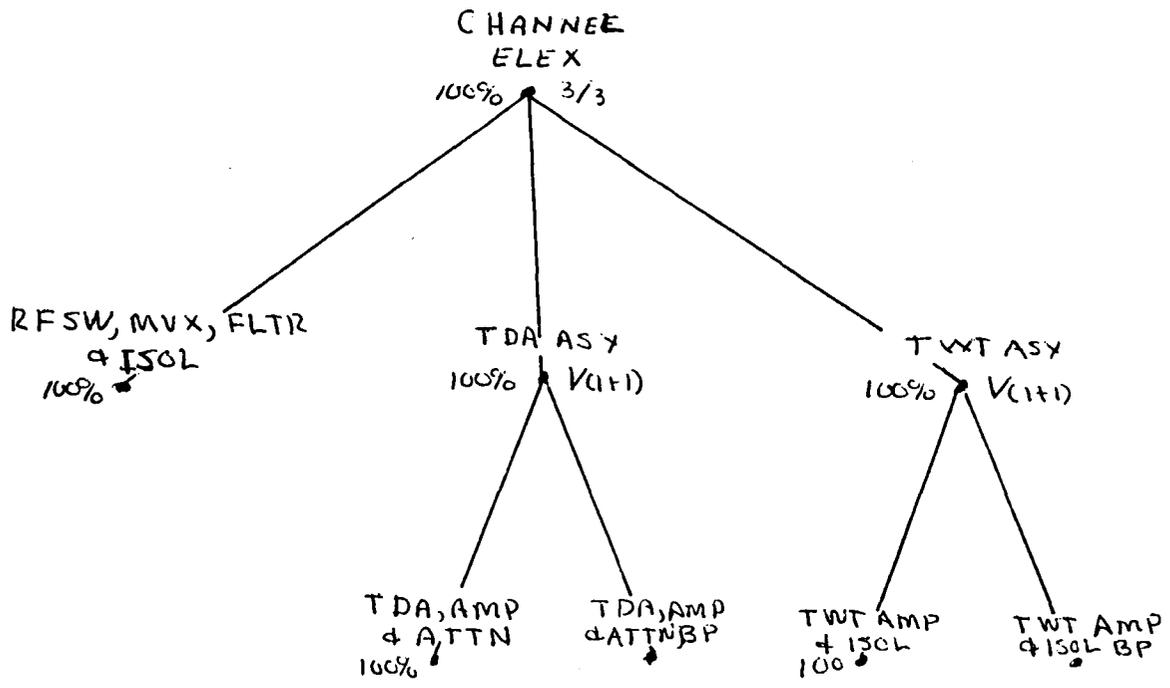
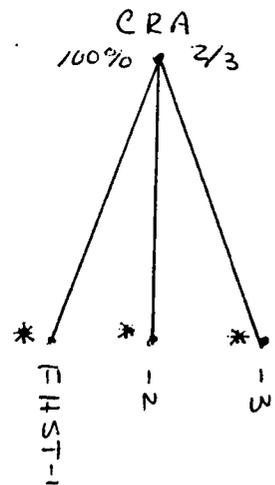
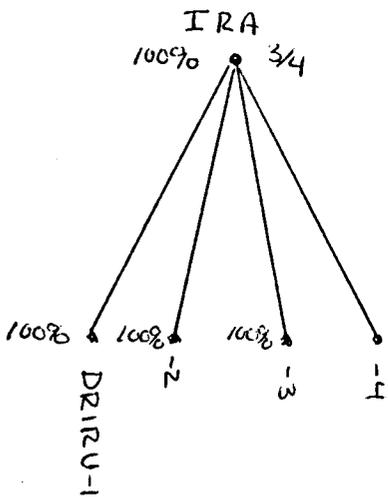
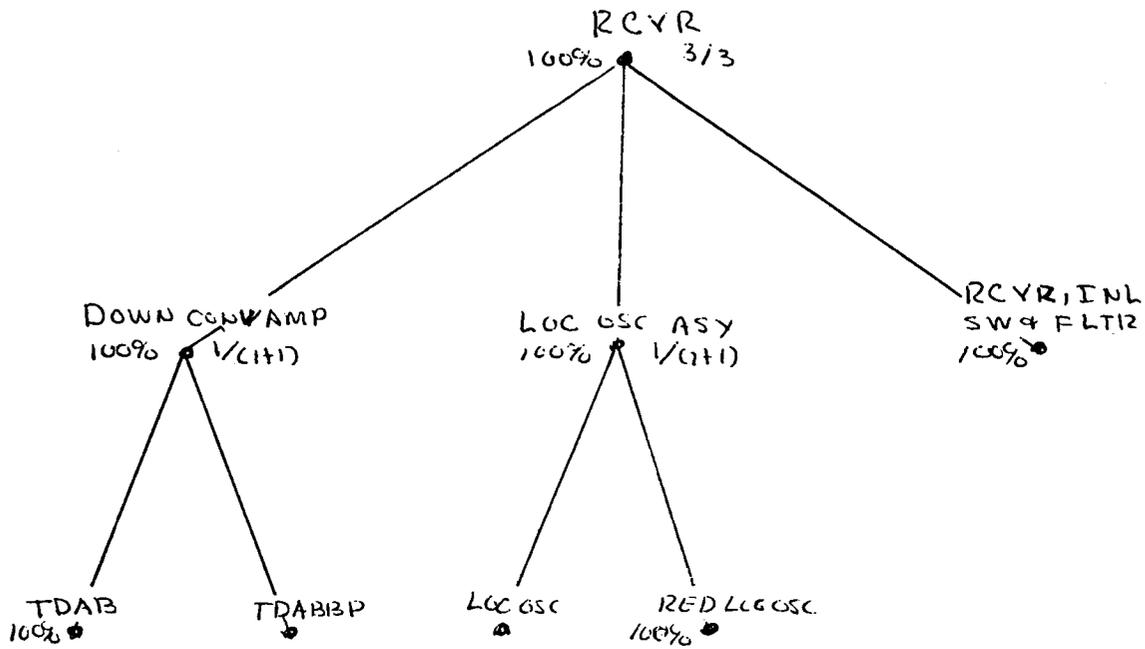


FIGURE 3

XPDR SET DETAILS - SAMPLE SYSTEM



* duty cycle is 66% until first failure then 100%.

FIGURE 4

DETAILS - SAMPLE SYSTEM

MODULE	NUMBER NEEDED	WEIGHT (Kgs)
XPDR GROUP	3	5.2
RCVR PK	1	2.1
IRA	1	3.4
FHST	4	1.2
RW	3	5.1

Payload Capacity of tug is 7.0 Kgs.

Minimum Capability of system is to be 15 channels over a 10 year life.

Desired statistics are

1. time average of on line capacity (# of transponders)
2. time to first maintenance
3. number of times maintenance scheduled due to weight
4. number of times maintenance scheduled due to capability
5. sample mean and variance for the random variables
 - a. pass duration
 - b. tug payload weight

TABLE 1
SAMPLE SYSTEM DEFINITIONS

NAME	NUMBER NEEDED	FAILURE DISTRIBUTION
TDA, AMP & ATTN	36	Expon, .488, .0488
TWT, AMP & ISOL	36	Expon, 2.795, .2795*
RFSW, MVX, FLTR & ISOL	18	Expon, .811, .0811
TDA	6	Expon, .4, .04
TDAB	4	Expon, 1.811, .1181
LOC OSC	4	Expon, 1.6, .16
DRIRU	4	Expon, 15.0
RFSW	3	Expon, .025
FHST	3	Expon, 4.411, .4411
RW	3	Expon, 15.0
RCVR, INL, SW & FLTR	2	Expon, .095,

* Also a drop dead time of 26,298 hours

TABLE 2
IBF DEFINITIONS FOR SAMPLE SYSTEM

4. The Simulation Model

4.1 Introduction

As previously indicated, the complexity of this problem suggests an analysis by simulation. For this purpose, the operation of the system across time is best modeled by a discrete event approach with two basic events - failure and maintenance of the system. Various languages are available for such a simulation. In particular, the language SLAM with its discrete event capability and its built-in use of FORTRAN is appropriate in this context (See [2]).

Each overall simulation run will consist of introducing a particular model of the physical system, together with a maintenance policy, demand function and mission lifetime. The system is then run through many passes, each pass corresponding to one mission and the model is re-initialized before each pass. Statistics are compiled at the end of each pass and after the last pass a report is printed. Items of interest typically include the time to first maintenance, total number of maintenance trips for each pass, average capability of the system, etc. The remainder of this section will treat PASS, the subroutine package which completes each simulation pass. The problem of loading the system model is a significant one and has not yet been fully developed - see the section on recommendations and conclusions.

4.2 PASS

The PASS routine sets the system in initial condition, schedules failures of IBF's, carries out failures, schedules maintenance, when required, performs maintenance, increases capability in response to demand, terminates the mission and keeps records.

Several assumptions about the computer model need to be detailed. The model is to be capable of increasing the system capability in response to demand. This is done by building the model to its full size at the beginning and introducing a condition DOES NOT EXIST for excess equipment. This equipment is then "brought into existence" during maintenance missions as it is needed. Additionally, some procedure must be chosen for terminating a pass. The standard procedure for this method will be to discontinue maintenance when the mission horizon is reached and then to terminate the mission when the system figure of merit drops to some cutoff point.

Table 3 gives the basic flow ideas of PASS and its major subroutines. These ideas are amplified in the pages following Table 3.

PASS

INITIALIZE
SAMPLE
TIME TO FAIL
FAIL
MAINT
DEMAND
MAXTIME
STAT

FAIL

IMPACT
STAT
TIME TO FAIL
SMAINT

MAINT

STORE
HEAL
MODIFIED FAIL
ADDON
SAMPLE
TIME TO FAIL
STAT

DEMAND

TURN ON
UPDATE
SDEMAND
TIME TO FAIL
STAT

TABLE 3

FLOW IDEAS FOR PASS AND ITS MAJOR SUBROUTINES

INITIALIZE	Sets the model into its initial condition at the start of each pass, each IBF is set to its proper starting condition, duty cycles are computed.
SAMPLE	Input is a set of IBF's, output is the time to fail in on-line (TTFO) and standby (TTFS) modes for each IBF in the input set.
TIME TO FAIL	Input is a set of IBF's, output is the time to next failure for this IBF (see section on Failures for details).
FAIL	An event subroutine which carries out the details of a failure (see Table 3 and below).
MAINT	An event subroutine which carries the details of maintenance (see Table 3 and below).
DEMAND	An event subroutine which carries out the details of "creating" new capacity for the GSP (see Table 3 and below).
MAXTIME	When the mission horizon is reached this routine terminates all maintenance and sets a test for mission termination.
STAT	A general routine which uses the SLAM statistics collection capability to record desired statistics.
IMPACT	Uses actions to carry out the effect of failures on the function trees (see section on Failures).
SMAINT	Uses a given maintenance policy to test the need for maintenance and to schedule maintenance when required.
STORE	Keeps the current records of those failed IBF's which will not be replaced during the current maintenance.
HEAL	Similar to initialize, however, all functions, including those which have been added in response to demand, are set to their initial state.
MODIFIED FAIL	A reduced version of FAIL, this routine refails all IBF whose records have been saved in STORE, no records are kept and no new failure times are required.

ADD ON	All equipment which is "delivered" on the current maintenance mission is "brought into existence" and statistics gathering is started.
TURN ON	New equipment is brought on-line in response to increasing demand.
UPDATE	The system definition which is used to HEAL the system for maintenance is brought up to date.
SDEMAND	The known demand curve is used to schedule the next DEMAND.

5. Failures

5.1 IBF Failures

In general, each IBF has a failure distribution with known parameters. Sample time to fail are drawn from this distribution for both on-line (TTFO) and standby (TTFS) modes. The proportion of on-line time (i. e. , the duty cycle), α , is then used to project the time to fail in each mode. The Red Life Method of failure is assumed here (see [4 , page 16]) so that the minimum of these pro-rated times is used as the time to next failure (TTF) of the IBF. Thus,

$$TTF = \text{MIN} \left\{ TTFO/\alpha , TTFS/(1-\alpha) \right\} .$$

Variations of this rule occur when the IBF will not fail in standby mode, $TTFS = \infty$, when it is on, $\alpha = 1$, or off, $\alpha = 0$, over an entire interval or when a drop dead time (DDT) requires that the item must fail after a fixed time. These variations are handled by adding or dropping terms in the preceding formula.

If some other IBF fails first, say after ΔT time units, then the failure times are adjusted by:

$$TTFO = TTFO - \alpha(\Delta T), TTFS = TTFS - (1-\alpha)(\Delta T), DDT = DDT - (\Delta T)$$

where α is the duty cycle of the preceding period. The computation then proceeds as before.

5.2 Failure Propagation

When an IBF fails, a message announcing this failure must be transmitted to its chief. The chief must then react to this message, perhaps issuing orders to its subordinates or send messages further up the function tree. This process must be repeated at each function which receives a message until all effects of the original failure are carried out.

These messages are delivered by means of actions. An action is an internally generated message which includes an ORIGIN (the originating function), a TARGET (the receiving function) and a CMD/CAUSE (a command to change state if the action is directed to a subordinate, a cause if the action is directed to a chief).

Several assumptions are made concerning actions. First among these is the assumption that no function will issue a command to a subordinate unless the subordinate is capable of carrying out the command. Another is that downward actions are always carried out before an upward action. (Actions are filed in a last in - first out set called ACTS).

5.3 A Standard Failure Rule

Let F be a function with a $n/(l + m)$ failure rule. That is, F has $(l + n)$ identical subordinates and requires that n be in an up condition for F to be up. Further, in the initial state l of the subordinates are up ($l \geq n$) and m have condition ready. The first m failures have no effect on the operation of F , the next $l - n$ failures each cause a decrease in the figure of merit of F while the $(l + m - n + 1)$ th failure causes F to fail. The attributes INDEX, FOM and RESERVE are used as follows: INDEX is a user-defined index which assigns to each subordinate of F an integer such that in the initial state the first l subordinates are up and the last m are ready; FOM is the figure of merit of F , if F is up $FOM(F)$ is the number of subordinates which are on, otherwise $FOM(F) = 0$; RESERVE is a pointer which indicates the lowest index of a subordinate in a ready state, when no such subordinates remain, RESERVE is set to zero. Thus, if $FOM(F) \geq N$, and S is a subordinate of F , then $INDEX(S)$ is less than $RESERVE(F)$ if $COND(S) = UP$ and greater than or equal to $RESERVE(F)$ if $COND(S) = READY$. The function F can receive two kinds of notices: a failure notice from a subordinate or a notice to go to a ready state from its chief. The next several tables outline the flow ideas in these two cases.

5.4 A Special Failure Rule

In the sample system of section 3, the XPDRSET's have a special failure rule. Each XPDRSET starts with six channels, five on-line and one standby. When a first channel fails, the standby, if still available, is turned on. When a second fails, the XPDRSET forwards an action to its chief asking that a channel in some other set be turned on if any standbys are left in the system. This can be implemented by using the figure of merit for each XPDRSET to be the number of channels on-line and the reserve to be the number in standby mode. The same meanings are attached to the attributes for each XPDR GROUP while the attributes of ME are the sum of the attributes of all these XPDR GROUPS.

* a failure message has been received by F from subordinate S

IF FOM(F)=0 THEN RETURN	* this failure occurred when F was not up
ELSE	
IF FOM(F)=N THEN GO TO FAIL	* this caused F to fail
ELSE	
IF RESERVE(F)=0 THEN FOM(F)=FOM(F)-1	* no standby left, F does not fail
ELSE	
IF INDEX(S) > RESERVE(F) THEN RETURN	* a stanby beyond the pointer failed
ELSE	
IF THERE IS A T IN BASE(F) WITH	* at least two standbys before the failure
COND(T) = READY AND INDEX(T) >	
RESERVE(F) THEN	
IF RESERVE(F) < INDEX(S) THEN CREATE	* failure was on-line, turn on first standby
AN ACTION AS FOLLOWS	
TARGET (ACTION) = RESERVE(F)	
ORIGIN (ACTION) = F	
CMD (ACTION) = UP	
FILE ACTION IN ACTS	
RESERVE(F) = INDEX OF FIRST MEMBER	* move pointer to next standby
OF BASE WITH COND = READY	
ELSE	
IF RESERVE(F) < INDEX(S) THEN CREATE	* RESERVE(F) is only standby
AN ACTION AS FOLLOWS	* failure was on-line, turn on reserve
TARGET (ACTION) = RESERVE(F)	
ORIGIN (ACTION) = UP	
FILE ACTION IN ACTS	
RESERVE(F) = 0	* no reserves left
RETURN	

. TABLE 4

STANDARD FAILURE RULE

* the FAIL part of the standard failure rule

FAIL:

COND(F) = FAIL

FOM(F) = 0

CREATE AN ACTION AS FOLLOWS

TARGET (ACTION) = CHIEF(F)

ORIGIN (ACTION) = F

CAUSE (ACTION) = FAIL

FILE ACTION IN ACTS

FOR EACH S IN BASE(F) WITH COND = UP

\$ CREATE AN ACTION AS FOLLOWS

TARGET (ACTION) = S

ORIGIN (ACTION) = F

CMD (ACTION) = READY

FILE ACTION IN ACTS

* adjust attributes of F

* send action to chief

* send action to all subfunctions which are up

RETURN

TABLE 5

FAIL

* F is sent an action to go to ready state

```
COND(F) = READY
FOM(F) = 0
FOR EACH S IN BASE(F) WITH COND = UP
$ CREATE AN ACTION AS FOLLOWS
  TARGET (ACTION) = S
  ORIGIN (ACTION) = F
  CMD (ACTION) = READY
  FILE ACTION IN ACTS
RETURN
```

* adjust attribute of F

* send action to all subordinates which are up

TABLE 6

DOWNWARD ACTIONS FOR STANDARD FAILURE RULE

6. Maintenance Policies

6.1 Introduction

It is assumed that demand, D , for the GSP capacity will increase as a function of time and that the demand is known in advance. The minimum acceptable capacity, MC , is typically of the form $D + \text{constant}$ where the constant is a "cushion" designed to avoid penalty costs. The nominal capacity, NC , is a desirable level for ordinary purposes. As time increases $NC - MC$ will typically decrease (refer to Figure 5).

At the initial time of the mission, capacity will be NC (0). As time increases failures will occur. Some of these failures will be in the support system and will not cause any decrease in capacity, others will cause such a decrease. For example, in Figure 5 the broken line gives the actual capacity of the system (AC). Capacity failures occur at points 1 and 3, subsystem failures at 2 and 4.

The intent of the simulation is to minimize the cost of maintenance. In order to do this, two sources of cost are considered: (1) the cost of the delivery system, and (2) the inventory cost of delivering capacity before it is needed. The major cost of the delivery system is independent of payload size, thus a trip with less than full payload is inefficient. There exists one or more size limits on the payload (e.g., weight, volume) and these are grouped under the heading size limit. When the first of these is reached, the payload is full. It is never possible to fly with a payload in which the size limit is exceeded.

The inventory cost is kept small by keeping the capacity between the minimal and nominal capacity. Because penalties occur when the capacity is below the minimum the requirement $AC \geq MC$ will be enforced. Thus, whenever the payload is full or the actual capacity drops to the minimum, a service mission is scheduled. Since it is desirable to keep the payload as large as possible and the capacity as close to nominal capacity as possible, two different policies arise. It is assumed that there exists a supply list which contains, in the order needed, the items necessary to bring the mission to nominal capacity.

6.2 Size Constrained Policy

When a failure occurs the proper replacement parts are loaded and a test is made to determine if the size limit has been reached. If so, the service mission is launched. If the size limit has not been reached, the current value of AC is compared to MC . If $AC = MC$ all available space is filled with modules from the supply list and the mission is launched. Under this policy, a service mission always flies a full payload but no relation between AC and NC can be drawn.

6.3 Capacity Constrained Policy

When a failure occurs the proper replacement parts are loaded and so are those items on the supply list which will bring the capacity up to NC at the time of failure. Size and capacity tests are then checked to see if a mission should be launched at the current time. If a size limit is not reached before the actual capacity drops to MC, the launch then takes place when $AC = MC$. This approach always brings the actual capacity of the system up to the nominal capacity but may produce less than full payloads.

6.4 An Additional Problem

Since failures come at distinct points in time with discrete jumps in capacity (and corresponding payload) it may be that at some point the payload is below the size limit and AC is above MC, however, at the next failure either AC is less than MC or the payload is greater than the size limit. Since minimal capacity is demand plus a cushion, the mission should be flown at once with no change of plans if AC falls below MC. In the case that payload has exceeded the size limit, a problem arises since the size limit is an absolute constraint, the mission cannot be flown if it is violated. In this case, the procedure will be to remove the last package(s) which caused the limit to be exceeded, mark these packages as first on the next service mission and fly the current mission.

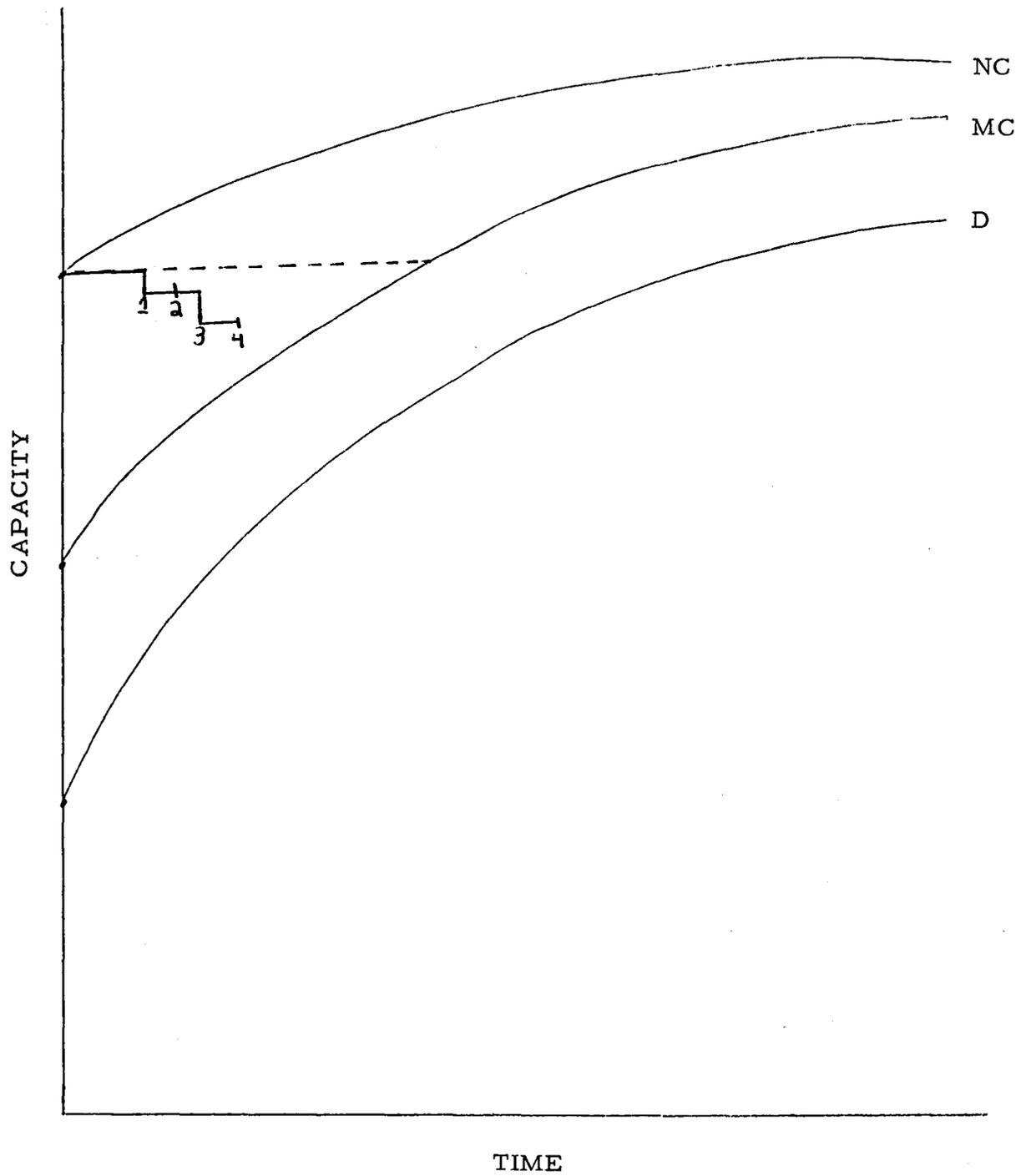


FIGURE 5
CAPACITY CURVES

7. Conclusions and Recommendations

The problem of maintenance of space missions is a serious one and the method outlined herein is a useful tool for approaching this problem. The particular problem of GSP maintenance is a significant one and will be a good test of the method.

Development of tools to solve the GSP problem, starting with the sample system of Chapter 3, will provide significant progress toward the general method.

Among the problems which arise in the implementation of the method one in particular deserves special study. Just as many modules have the same features and can be built into the computer model by the generic module concept, so to many functions one duplicates. A concept of generic function would be useful in this respect (one estimate is that the final GSP model will have 30 identical XPDR GROUPS rather than the 3 of the simple system). In the general case it cannot be assumed that identical basic functions will belong to the same module types. Thus any concept of generic function must allow for the placement of specific IBF's in different modules.

It is recommended that work on this problem continue in two directions. First that a simulation package actually be built and run for GSP and second that work on the general method continue beyond the GSP version of this problem.

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AXIALLY SYMMETRIC MOTION OF A ROTATING STRATIFIED FLUID

PREPARED BY: Benjamin J. Martin, Ph. D.

ACADEMIC RANK: Professor

COLLEGE AND DEPARTMENT: Atlanta University
Mathematics

NASA/MSFC:
(OFFICE) Space Sciences Laboratory
(DIVISION) Atmospheric Sciences
(BRANCH) Fluid Dynamics

MSFC COUNTERPART: Dr. William Fowles

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AXIALLY SYMMETRIC MOTION OF A ROTATING STRATIFIED FLUID

By

Benjamin J. Martin, Ph. D.
Professor of Mathematics
Atlanta University
Atlanta, Georgia

ABSTRACT

A spherical model of the general circulation of the Earth's atmosphere is planned for Spacelab. In order to provide adequate support for the experiment, analytical and numerical studies and simulations must be done. Therefore, an analysis of an earlier effort by Joseph Pedlosky in the area of spherical modeling of geophysical fluid flow is examined. Possible extension of his work is considered and alternative approaches suggested. Recommendations are made for future investigations.

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NOMENCLATURE

Ω	magnitude of the angular velocity vector
R	radius of inner sphere
D	distance between inner and outer spheres
ν	kinematic viscosity
V_0	velocity scale factor
$E = \frac{\nu}{2\Omega D^2}$	Ekman number $(= \frac{1}{2} E_V)$
$\varepsilon = \frac{V_0}{2\Omega R}$	
ΔT	temperature difference between inner and outer spheres
α	coefficient of thermal expansion
κ	thermal diffusivity
$\sigma = \nu/\kappa$	Prandtl number
g	gravitational constant
$\beta = \frac{g\alpha D\Delta T}{4\Omega^2 R^2}$	thermal Rossby number $(= \frac{1}{4} S)$
T	temperature (scaled)
p	pressure (scaled)
ρ	density
$S = D/R$	aspect ratio
u	eastward velocity (scaled)
v	northward velocity (scaled)
w	vertical velocity (scaled)

φ longitude

θ latitude

$$\mu \equiv \sqrt{\frac{1}{2} |\sin \theta|}$$

$$\text{sgn } \theta \equiv \begin{cases} 1, & \theta > 0 \\ -1, & \theta < 0 \end{cases}$$

$$\varphi' \equiv \frac{d\varphi}{d\theta}$$

1. INTRODUCTION.

When the Space Shuttle becomes operational some of the flights will take into orbit a laboratory known as Spacelab. A principal use of Spacelab will be the exploitation of the low gravity environment of the shuttle for scientific and technological investigations. In particular, it will be possible to perform geophysical fluid flow model experiments in true spherical geometry. A radial dielectric body force, which is analogous to gravity, can be achieved over a volume of liquid held between two concentric spheres. However on the Earth's surface the dielectric force cannot be made large enough to dominate the effect of terrestrial gravity. Hence the low-gravity environment of Spacelab is the ideal location for these experiments.

In order to adequately design the experiment for Spacelab, preliminary theoretical and numerical work must be done. Specifically, the scientific design program for the Atmospheric General Circulation Experiment (AGCE) requires that some baroclinic stability studies in spherical geometry be tackled. This can be done only after a basic state has been established. Simplified models must be used since otherwise the equations are non-separable.

The particular task here is the extension of some previous work done in spherical geometry by Joseph Pedlosky to the specific case of interest in AGCE. This involves changing the boundary conditions under which the basic equations are to be solved. For simplicity the linear, axially symmetric steady state solution is sought.

2. THE GOVERNING EQUATIONS.

The general governing equations for an incompressible fluid in dimensional form are

The Momentum Equation

$$\frac{\partial \bar{V}}{\partial t} + (\bar{V} \cdot \nabla) \bar{V} + \Omega \times \bar{V} = -\frac{1}{\rho} \nabla \tilde{p} + \bar{g} + \nu \nabla^2 \bar{V}$$

The Thermodynamic Equation

$$\frac{\partial \tilde{T}}{\partial t} + (\bar{V} \cdot \nabla) \tilde{T} = \kappa \nabla^2 \tilde{T}$$

The Continuity Equation

$$\nabla \cdot \bar{V} = 0$$

and The Equation of State for a liquid

$$\tilde{p} = p_0 (1 - \alpha (\tilde{T} - T_0))$$

Thus we have six equations in six unknowns $\bar{V} = (\tilde{V}_r, \tilde{V}_\theta, \tilde{V}_\varphi)$, \tilde{T} , \tilde{p} , and \tilde{p} .

Note also that the equations are non-linear. For the model under consideration the steady-state axially-symmetric case is of interest so that the operators are two-dimensional involving only r and θ in spherical geometry:

$$(\bar{V} \cdot \nabla) \bar{V} + \Omega \times \bar{V} = -\frac{1}{\rho} \nabla \tilde{p} + \bar{g} + \nu \nabla^2 \bar{V}$$

$$(\bar{V} \cdot \nabla) \tilde{T} = \kappa \nabla^2 \tilde{T}$$

$$\nabla \cdot \bar{V} = 0$$

$$\tilde{p} = p_0 (1 - \alpha (\tilde{T} - T_0)).$$

Introducing non-dimensional variables

$$\tilde{r} = R + \delta R z, \quad \delta = D/R$$

$$\tilde{V}_r = \delta V_0 \omega(\theta, z)$$

$$\tilde{V}_\theta = V_0 \psi(\theta, z)$$

$$\tilde{V}_\varphi = V_0 u(\theta, z)$$

$$\tilde{T} = T_0 + \Delta T z + (2\Omega R V_0 / g \alpha D) T(\theta, z)$$

$$\tilde{p} = p_0 - \rho_0 g D z + \frac{1}{2} g \alpha D \Delta T \rho_0 z^2 + 2\Omega V_0 R \rho_0 P(\theta, z)$$

the equations become

$$\begin{aligned} \epsilon \left[\omega \frac{\partial u}{\partial z} + v \frac{\partial u}{\partial \theta} + \delta u \omega - u v \tan \theta \right] + (\omega \delta \cos \theta - v \sin \theta) \\ = \delta^2 E \left[\frac{1}{\cos \theta} \frac{\partial}{\partial \theta} \left(\frac{\partial u}{\partial \theta} \cos \theta \right) - \frac{u}{\cos^2 \theta} \right] + E \left[\frac{\partial^2 u}{\partial z^2} + 2\delta \frac{\partial u}{\partial z} \right] \end{aligned}$$

$$\begin{aligned} \epsilon \left[\omega \frac{\partial v}{\partial z} + v \frac{\partial v}{\partial \theta} + \delta v \omega + u^2 \tan \theta \right] + u \sin \theta = -\frac{\partial p}{\partial \theta} + 2\delta^3 E \frac{\partial \omega}{\partial \theta} \\ + \delta^2 E \left[\frac{1}{\cos \theta} \frac{\partial}{\partial \theta} \left(\frac{\partial v}{\partial \theta} \cos \theta \right) - \frac{v}{\cos^2 \theta} \right] + E \left[\frac{\partial^2 v}{\partial z^2} + 2\delta \frac{\partial v}{\partial z} \right] \end{aligned}$$

$$\begin{aligned} \epsilon \delta \left[\delta \omega \frac{\partial \omega}{\partial z} + \delta v \frac{\partial \omega}{\partial \theta} - (u^2 + v^2) \right] - \delta u \cos \theta = -\frac{\partial p}{\partial z} + T - 2\delta^4 E \omega \\ + \delta^4 E \left[\frac{1}{\cos \theta} \frac{\partial}{\partial \theta} \left(\frac{\partial \omega}{\partial \theta} \cos \theta \right) \right] + \delta^2 E \left[\frac{\partial^2 \omega}{\partial z^2} + 2\delta \frac{\partial \omega}{\partial z} \right] \\ - 2\delta^3 E \left[\frac{1}{\cos \theta} \frac{\partial}{\partial \theta} (v \cos \theta) \right] \end{aligned}$$

$$\begin{aligned} \beta \omega + \epsilon \left[\omega \frac{\partial T}{\partial z} + v \frac{\partial T}{\partial \theta} \right] = \frac{2\kappa \beta}{V_0 D} + \frac{\delta^2 E}{\sigma} \left[\frac{1}{\cos \theta} \frac{\partial}{\partial \theta} \left(\frac{\partial T}{\partial \theta} \cos \theta \right) \right] \\ + \frac{E}{\sigma} \left[\frac{\partial^2 T}{\partial z^2} + 2\delta \frac{\partial T}{\partial z} \right] \end{aligned}$$

$$\frac{\partial \omega}{\partial z} + 2\delta \omega + \frac{1}{\cos \theta} \frac{\partial}{\partial \theta} (v \cos \theta) = 0$$

For ϵ sufficiently small the above equations can be linearized

$$\delta u \cos \theta - v \sin \theta = S^2 E \left[\frac{1}{\cos \theta} \frac{\partial}{\partial \theta} \left(\frac{\partial u}{\partial \theta} \cos \theta \right) - \frac{u}{\cos^3 \theta} \right] + E \left[\frac{\partial^2 u}{\partial z^2} + 2S \frac{\partial u}{\partial z} \right]$$

$$u \sin \theta = -\frac{\partial p}{\partial \theta} + 2S^3 E \frac{\partial \omega}{\partial \theta} + S^4 E \left[\frac{1}{\cos \theta} \frac{\partial}{\partial \theta} \left(\frac{\partial v}{\partial \theta} \cos \theta \right) - \frac{v}{\cos^3 \theta} \right] + E \left[\frac{\partial^2 v}{\partial z^2} + 2S \frac{\partial v}{\partial z} \right]$$

$$-S u \cos \theta = -\frac{\partial p}{\partial z} + T + S^4 E \left[\frac{1}{\cos \theta} \frac{\partial}{\partial \theta} \left(\frac{\partial \omega}{\partial \theta} \cos \theta \right) \right] + S^2 E \left[\frac{\partial^2 \omega}{\partial z^2} + 2S \frac{\partial \omega}{\partial z} \right] - 2S^4 E \omega - 2S^3 E \left[\frac{1}{\cos \theta} \frac{\partial}{\partial \theta} (v \cos \theta) \right]$$

$$\beta \omega = \frac{2\kappa\beta}{\nu_0 D} + \frac{S^2 E}{\sigma} \left[\frac{1}{\cos \theta} \frac{\partial}{\partial \theta} \left(\frac{\partial \Gamma}{\partial \theta} \cos \theta \right) \right] + \frac{E}{\sigma} \left[\frac{\partial^2 \Gamma}{\partial z^2} + 2S \frac{\partial \Gamma}{\partial z} \right]$$

$$\frac{\partial \omega}{\partial z} + 2S \omega + \frac{1}{\cos \theta} (v \cos \theta) = 0.$$

These are the equations of motion in terms of non-dimensional variables which are appropriate for linear, axially-symmetric, steady-state motion of an incompressible fluid in spherical geometry.

3. THE SOLUTION OF THE SYSTEM.

The system that is used here is the system as derived and simplified by Pedlosky [1] with some obvious corrections and a few notational changes

$$-v \sin \theta + S u \cos \theta = S^2 E \left[\frac{1}{\cos \theta} \frac{\partial}{\partial \theta} \left(\frac{\partial u}{\partial \theta} \cos \theta \right) - \frac{u}{\cos^3 \theta} \right] + E \left[\frac{\partial^2 u}{\partial z^2} + 2S \frac{\partial u}{\partial z} \right]$$

$$u \sin \theta = -\frac{\partial p}{\partial \theta} + S^2 E \left[\frac{1}{\cos \theta} \frac{\partial}{\partial \theta} \left(\frac{\partial v}{\partial \theta} \cos \theta \right) - \frac{v}{\cos^3 \theta} \right] + E \left[\frac{\partial^2 v}{\partial z^2} + 2S \frac{\partial v}{\partial z} \right]$$

$$-S u \cos \theta = -\frac{\partial P}{\partial z} + T + \delta^4 E \left[\frac{1}{\cos \theta} \frac{\partial}{\partial \theta} \left(\frac{\partial \omega}{\partial \theta} \cos \theta \right) \right] \\ + \delta^2 E \left[\frac{\partial^2 \omega}{\partial z^2} + 2\delta \frac{\partial \omega}{\partial z} \right]$$

$$\beta \omega = \frac{\delta^2 E}{\sigma} \frac{1}{\cos \theta} \frac{\partial}{\partial \theta} \left(\frac{\partial T}{\partial \theta} \cos \theta \right) + \frac{E}{\sigma} \left[\frac{\partial^2 T}{\partial z^2} + 2\delta \frac{\partial T}{\partial z} \right]$$

$$\frac{\partial \omega}{\partial z} + 2\delta \omega + \frac{1}{\cos \theta} \frac{\partial}{\partial \theta} (v \cos \theta) = 0.$$

The boundary conditions are

$$(u, v, \omega) = (0, 0, 0) \quad \text{on } z=0 \text{ and } z=1$$

$$T(\theta, 0) = H_L(\theta)$$

$$T(\theta, 1) = H_T(\theta)$$

In order to solve the system with reasonable effort, the solution is decomposed into three parts--the interior motion (the I-subscripted variables), the lower boundary layer corrections (the L-subscripted variables), and the upper boundary layer corrections. Therefore

$$u(\theta, z) = u_I(\theta, z) + u_T(\theta, x) + u_L(\theta, y)$$

$$v(\theta, z) = E v_I(\theta, z) + v_T(\theta, x) + v_L(\theta, y)$$

$$\omega(\theta, z) = E^{1/2} [\omega_I(\theta, z) + \omega_T(\theta, x) + \omega_L(\theta, y)]$$

$$P(\theta, z) = P_I(\theta, z) + \delta E^{1/2} [P_T(\theta, x) + P_L(\theta, y)]$$

$$T(\theta, z) = T_I(\theta, z) + E^{1/2} \sigma \beta [T_T(\theta, x) + T_L(\theta, y)]$$

where $\chi = (1-z)E^{-1/2}$ and $y = zE^{-1/2}$. Since the L and T subscripted variables are significant only near the boundary, it is required that they tend to zero for large y and χ .

The equations for the lowest order correction functions in the respective boundary layers are

Lower Boundary Layer

$$-V_L \sin \theta = \frac{\partial^2 u_L}{\partial y^2} \quad (3.1)$$

$$u_L \sin \theta = \frac{\partial^2 v_L}{\partial y^2} \quad (3.2)$$

$$-S u_L \cos \theta = -S \frac{\partial p_L}{\partial y} + E^{1/2} \gamma \beta T_L \quad (3.3)$$

$$\omega_L = \frac{1}{2} \frac{\partial^2 T_L}{\partial y^2} \quad (3.4)$$

$$\frac{\partial \omega_L}{\partial y} = -\frac{1}{\cos \theta} \frac{\partial}{\partial \theta} (V_L \cos \theta) \quad (3.5)$$

Upper Boundary Layer

$$-V_T \sin \theta = \frac{\partial^2 u_T}{\partial x^2}$$

$$u_T \sin \theta = \frac{\partial^2 v_T}{\partial x^2}$$

$$-S u_T \cos \theta = S \frac{\partial p_T}{\partial x} + E^{1/2} \gamma \beta T_T$$

$$\omega_T = \frac{1}{2} \frac{\partial^2 T_T}{\partial x^2}$$

$$\frac{\partial \omega_T}{\partial y} = \frac{1}{\cos \theta} \frac{\partial}{\partial \theta} (V_T \cos \theta)$$

Similarly equations for the interior solution are obtained to lowest order in E

$$-V_I \sin \theta = \frac{\partial^2 u_I}{\partial z^2} \quad (3.6)$$

$$u_I \sin \theta = -\frac{\partial p_I}{\partial \theta} \quad (3.7)$$

$$T_I = \frac{\partial p_I}{\partial z} \quad (3.8)$$

$$\partial \sigma \beta E^{-1/2} \omega_I = \frac{\partial^2 T_I}{\partial z^2} \quad (3.9)$$

$$\frac{\partial \omega_I}{\partial z} = 0 \quad (3.10)$$

where, following Pedlosky, we have assumed $\delta \ll E^{1/2}$ and have dropped all terms involving powers of δ .

The Boundary Layer Corrections.

The system of equations for the lower boundary layer corrections can be easily solved. First the coupled equations (3.1) and (3.2) are solved. This allows the solution of equation (3.5), which then permits the solution of equation (3.6).

$$u_L(\theta, y) = -u_I(\theta, 0) e^{-\mu y} \cos \mu y$$

$$v_L(\theta, y) = u_I(\theta, 0) e^{-\mu y} \sin \mu y$$

$$\omega_L(\theta, y) = \frac{1}{\cos \theta} \frac{\partial}{\partial \theta} \left[\frac{u_I(\theta, 0) \rho \mu e}{\sigma^2 \mu} e^{-\mu y} \cos \theta \sin(\mu y + \frac{\pi}{4}) \right]$$

$$T_L(\theta, y) = \frac{1}{\cos \theta} \frac{\partial}{\partial \theta} \left[\frac{u_I(\theta, 0) \rho \mu e}{\sigma^2 \mu^3} e^{-\mu y} \cos \theta \sin(\mu y - \frac{\pi}{4}) \right]$$

where the boundary conditions have been used to get

$$u_I(\theta, 0) + u_L(\theta, 0) = 0$$

$$E v_I(\theta, 0) + v_L(\theta, 0) \approx M_L(\theta, 0) = 0$$

$$T_I(\theta, 0) + E^{1/2} \sigma \beta T_L(\theta, 0) \approx T_I(\theta, 0) = H_L(\theta) \quad (3.11)$$

$$\omega_I(\theta, 0) + \omega_L(\theta, 0) = 0$$

$$\text{or} \quad \omega_I(\theta, 0) = - \frac{1}{2 \cos \theta} \frac{\partial}{\partial \theta} \left(\frac{u_I(\theta, 0) \rho g m \theta \cos \theta}{\mu} \right) \quad (3.12)$$

Similarly, for the upper boundary layer corrections we have

$$u_T(\theta, x) = - e^{-\mu x} u_I(\theta, 1) \cos \mu x$$

$$v_T(\theta, x) = e^{-\mu x} u_I(\theta, 1) \rho g m \theta \sin \mu x$$

$$\omega_T(\theta, x) = - \frac{1}{\cos \theta} \frac{\partial}{\partial \theta} \left[\frac{u_I(\theta, 1) \rho g m \theta}{\sqrt{2} \mu} e^{-\mu x} \cos \theta \sin \left(\mu x + \frac{\pi}{4} \right) \right]$$

$$T_T(\theta, x) = - \frac{1}{\cos \theta} \frac{\partial}{\partial \theta} \left[\frac{u_I(\theta, 1) \rho g m \theta}{\sqrt{2} \mu^3} e^{-\mu x} \cos \theta \sin \left(\mu x - \frac{\pi}{4} \right) \right]$$

$$T_I(\theta, 1) = H_T(\theta) \quad (3.13)$$

$$\omega_I(\theta, 1) = \frac{1}{2 \cos \theta} \frac{\partial}{\partial \theta} \left(\frac{u_I(\theta, 1) \rho g m \theta \cos \theta}{\mu} \right) \quad (3.14)$$

The Interior Variables.

Equation (3.10) implies that ω_I is independent of z therefore

$\omega_I(\theta, 0) = \omega_I(\theta, 1)$. Hence $u_I(\theta, 0) = -u_I(\theta, 1)$. Assuming $\omega_I(\theta, 0) = \omega_I(\theta, 1) = \omega_I(\theta)$ to be known the system can be solved beginning with

equation (3.9) working back to equation (3.6):

$$T_I(\theta, z) = \sigma \beta E^{-1/2} \omega_I(\theta) (z^2 - z) + H_T(\theta) z - H_L(\theta) (z - 1)$$

$$u_I(\theta, z) = -\frac{1}{\sin \theta} \left[\sigma \beta E^{-1/2} \omega'_I(\theta) \left(\frac{4z^3 - 6z^2 + 1}{12} \right) + H'_T(\theta) \frac{(z^2 - 1)}{4} - H'_L(\theta) \left(\frac{z^2 - 4z + 1}{4} \right) \right]$$

$$v_I(\theta, z) = \frac{1}{\sin^2 \theta} \left[\sigma \beta E^{-1/2} \omega'_I(\theta) (2z - 1) + H'_T(\theta) - H'_L(\theta) \right]$$

(Note: $p(\theta, z)$ is not of interest except as needed to find the other dynamic variables and thus has been omitted.)

The system is completely solved when $u_I(\theta, c)$ is known (or, equivalently, $\omega_I(\theta)$). $\omega_I(\theta)$ has not yet been found but a second order differential equation for ω_I can be derived

$$\omega_I(\theta) = \frac{\sin \theta}{2 \cos \theta} \frac{d}{d\theta} \left\{ \left[\frac{1}{12} \sigma \beta E^{-1/2} \frac{d}{d\theta} \omega_I(\theta) - \frac{1}{4} (H'_T(\theta) + H'_L(\theta)) \right] \frac{\cos \theta}{\mu \sin \theta} \right\}.$$

This equation has not yet been solved.

4. CONCLUSIONS AND RECOMMENDATIONS.

(a) The technique of Pedlosky is a perturbation technique, however this is disguised in the analysis. The net result is that the order of approximation is different for each variable. It is recommended therefore that each variable be explicitly represented as a perturbation, e.g.,

$\phi = \phi_0 + E^{1/2} \phi_1 + E \phi_2 + \dots$, and all solutions be found to the same order of approximation if possible or as nearly as is possible.

(b) Some of the assumptions of smallness made by Pedlosky are too stringent for AGCE. As many as possible should be relaxed, in particular, $\xi \ll E^{1/2}$.

(c) The author derived the equations of motion for spherical geometry. The resulting equations differ non-trivially from those of Pedlosky. Unless these discrepancies can be adequately explained, all future work should be done using the author's equations.

(d) Some consideration should be given to the non-linear equations, possibly using the technique of Rahm and Walin [2].

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NASA/ASEE SUMMER FACULTY RESEARCH FELLOWSHIP PROGRAM

A RANKING ALGORITHM FOR SPACELAB
CREW AND EXPERIMENT SCHEDULING

Prepared by: Frank H. Mathis and Robert D. Grone

Refer to Section IX for report.

1980

NASA/ASEE SUMMER FACULTY RESEARCH FELLOWSHIP PROGRAM

MARSHALL SPACE FLIGHT CENTER
THE UNIVERSITY OF ALABAMA

PAYLOAD CREW TRAINING COMPLEX (PCTC) UTILIZATION AND TRAINING PLAN
STATUS REPORT

Prepared By: Michael R. Self, Ed.D.

Academic Rank: Assistant Research Professor

University and Department: University of Alabama in Huntsville
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PAYLOAD CREW TRAINING COMPLEX (PCTC) UTILIZATION
AND TRAINING PLAN

by

Michael R. Self, Ed.D.
Assistant Research Professor
University of Alabama-Huntsville
Huntsville, AL

ABSTRACT

The first section describes the physical facilities that comprise the Payload Crew Training Complex (PCTC). The PCTC is located at MSFC, Huntsville, Alabama, and contains the Host Simulator; Experiment Simulators; Spacelab Aft Flight Deck, Experiment Pallet, and Experiment Rack mockups; the Simulation Director's Console; Payload Operations Control Center; classrooms; and supporting soft- and hardware.

The second section of the report establishes the parameters of a training philosophy for Payload Crew Training at the PCTC. Major parameters include: (1) The intent of training is not to be diagnostic, to select the "best" Payload Crew, but should be enhancing to the development of high levels of proficiency in every Payload Crew trainee; (2) The intelligence of each Payload Crew trainee is obviously to be respected. This implies that training at the PCTC, while highly procedural and sophisticated should tend to be less rigid than similar kinds of traditional high technology training; and (3) The function of trainee evaluation will be to determine how well training is being conducted. From the cooperative debriefings between the Payload Crew and PCTC trainers, should come ideas for improving the training sequence.

Section three discusses the development of the training plan. Included are discussions of preassessment, skill identification, procedures, material development, and evaluation options.

INTRODUCTION

NASA finds itself on the threshold of a new era in space travel, namely repeated flights with a reusable Space Transportation System (STS), i.e., the Space Shuttle. Additionally, they have diversified their payload configurations to include a larger portion of the scientific community not traditionally associated with NASA. For the first time too, the European Space Agency (ESA) has primary development responsibility for the Shuttle payload. The result is a new learning opportunity in the co-development of space missions and requires new thinking in training concepts.

One of the major distinctions between past NASA manned flights and this new era has been the changes in training philosophy and requirements. In the past everyone that has flown space missions has been a career astronaut with a scientific or technical background with flight training and experience. Additionally, all of the training tended to be centralized under NASA auspices. Yet in the presently scheduled flights, i.e., Spacelabs 1, 2, and 3, the Principle Investigators (PI) have primary responsibility for training the Payload Crew (PC) in how to conduct the PI's experiments. The PI's are research oriented, typically in a university setting, and literally distributed around the world.

These factors have precipitated the need for decentralized training. The effect of the move to decentralized training has been to: 1) reduce substantially the cost of training to NASA, since NASA does not have to reproduce all of the PI's equipment for training, and 2) add impetus to assuring overall flight competence prior to the actual flight via systems integration training. Concern with this second component is part of the issue addressed in this document. Another facet of the issue is how to best use the Payload Crew Training Complex (PCTC).

The PCTC was established to train the PC on those experiments that require interface with the onboard Spacelab computer, i.e., the Command and Data Management System (CDMS). In establishing the facility many questions arose as to how to achieve maximum utilization from the facility. Some of those questions are addressed herein.

OBJECTIVES

The concern then has been to examine the parameters associated with decentralized training; which, as mentioned previously is a situation unique to the Space Shuttle/Spacelab missions, but a concept that appears to be one that NASA plans to pursue at least for the foreseeable future. The objectives then that derive from this goal are, at present, threefold:

- 1) To describe the PCTC and its training capabilities.
- 2) To define a training philosophy that might guide the PCTC trainers in developing a training plan.
- 3) To describe a procedure for the development and implementation of a training plan for training the PC at the PCTC.

What makes this report a status report is that training has not yet begun, so there are no evaluations of actual training procedures discussed. Furthermore, decisions on the best training approach are still in a deliberative state due to the fact that all of the experiments requiring CDMS interface, while known, have not been broken down into the step-by-step procedures level. This breakdown is basic to skill identification and the subsequent training procedures development.

PCTC

The PCTC is located at MSFC, Huntsville, Alabama. The facility houses the Host Simulator (HS); Experiment Simulators (ES); the Spacelab Aft Flight Deck (AFD), Experiment Pallet, and Experiment Rack mockups; a Simulation Director's Console (SDC), a Payload Operations Control Center (POCC) console; classrooms; and supporting soft- and hardware.

The HS is the computer system that provides the capability to simulate the command, communications, control, and display functions of the Spacelab CDMS. This capability is necessary to execute experiment-unique simulations. It has the following features:

- 1) Two Data Display Systems (DDS) which can be simultaneously operated in support of one mission phase.
- 2) Reconfiguration and turnaround for the support of one training mission to another can be accomplished in eight hours.
- 3) Drives the SDC and POCC consoles for training and to approximate flight conditions.
- 4) Is modular in design and does not preclude expansion in size or computing speed.
- 5) Can simultaneously simulate multiple experiments, consistent with CDMS operating capabilities and provides standard analog-to-digital and digital-to-analog services for interfacing with simple experiment-unique control and display panels.
- 6) Allows for operation in a real time mode so that display time and command responses are commensurate with actual flight conditions.
- 7) Simulates symbolic pictures of the earth, sun, and sky scenes in a standard TV format for training in experiment pointing commands.
- 8) Protects against inadvertent damage to hardware and/or software by any operator or trainee.
- 9) Allows for environmental inputs to experiment models such as orbital state vector, vehicle attitude, and ephemeris of heavenly bodies.
- 10) Allows for detection and accumulation of characteristic data, such as PC errors, response times, and operating times.

The ES's include the hardware and software necessary to simulate experiments through the HS during training. A Data Display Unit (DDU) showing simulated data for experiment operations originates from either an instrument or Experiment Computer Applications Software (ECAS) simulation software package.

Experiment-unique command data is initiated through the DDS-KeyBoard (KB). The CDMS is entered via KB command, item, type, and/or function keys and acted upon, and thus generates the appropriate response within the simulation software. Each ES will send data and accept commands from certain key simulated control and display panels located in the experiment racks onboard the Spacelab mockup. Field of view imagery and alphanumeric data for display on payload-unique Cathode Ray Tubes (CRT's) are provided from the ES. Simulated POCC ground command inputs are available to the crew via the ES. The ES accepts and processes commands from the SDC during simulation. These commands control such simulation circumstances as simulation modes, control sequencing, fault initiation, and time cycling.

The SDC enables the Simulation Director and Assistant Simulation Director to monitor, control, and modify the simulation operation. It is composed of four racks containing two computer terminals, a hard copy machine, four large black and white Closed Circuit TV (CCTV) monitors, and a work surface which encompasses the CCTV camera controls. DDU monitors in rack 4 will display the same DDU displays as the trainee sees in the Spacelab module. Cameras for the TV monitors will be used thus: two inside the Spacelab module, one for the AFD and one for general surveillance of the high bay area. CCTV cameras are controllable in pitch and azimuth from hand controllers on the SDC work surface. Three video tape recorders may be used to record data from any one of several video sources: four CCTV cameras, four DDU's, and three Spacelab CCTV's used with scene generation. Hard copies of data seen on any of the CRT's are available by merely pressing a button on the hard copier after selecting the appropriate CRT.

The PCTC mockups include the Spacelab Module Interior, Shuttle AFD Interior, and associated equipment and pallets for part task activities with the capability for supporting payload crew training for any single Spacelab mission and the capability for reconfiguration to any payload.

TRAINING PHILOSOPHY

A training philosophy is nothing more than an acknowledgement of the guiding ideals within which training will hopefully be implemented. The components of that philosophy may not actually be formally stated again, but they should be reflected in how the training is actually conducted. Philosophy statements describe thoughts or events that typically are extremely difficult to quantify for measurement, yet are important enough to be considered.

The training philosophy for the utilization of the PCTC hopefully incorporates the following parameters:

- 1) The intent of training is not to be diagnostic to select the "best" PC, but should facilitate the development of high levels of proficiency in every PC trainee.
- 2) The intelligence of each PC trainee is obviously to be respected. This implies that training at the PCTC, while highly procedural and sophisticated should tend to be less rigid than similar kinds of traditional high technology training.
- 3) The function of trainee evaluation will be to determine how well training is being conducted. From the cooperative debriefings between the PC and PCTC trainers should come ideas for improving the training sequence to help assure the highest quality of training and the maximum utilization of the PCTC's capabilities.

Specifically, those philosophical considerations include:

- 1) In order to proceed in a logical training sequence, critical skills and critical experiment sequences should be identified to better prepare the PC for the time when he actually encounters them.
- 2) There is a need for PC assessment upon initiation of training at the PCTC. This assessment is necessary for the PCTC trainers to correctly determine how well they are conducting the training. Initial assessment should probably involve assessment of the PC's skills in utilizing the DDU/KB, understanding the Experiment Computer Operating System (ECOS), and communication skills.
- 3) While individual PC decision making ability is a major concern, it is not planned to "assess" those skills during PCTC training. Instead, the goal is to provide training experiences to facilitate those skills. For example, while PCTC training is highly procedural, in selecting the contingencies for use in training, some of the training scenarios are speculative in nature. They have been developed using the best information available, but in some cases the exact flow of the experiment will not be encountered until the actual flight. Therefore, a training goal should be for the PC to remain cognizant of the overall experiment goal(s). The value of this is to be able to periodically step back from the minutia of experiment procedures and conceptually think through what is supposed to be happening. This should enhance the likelihood that the PC would make the best decision when, in flight, he actually encounters that option point in the experiment flow.
- 4) It follows from the above statement that a large segment of learning will not occur until the actual flight because of the sheer uniqueness of space flight; therefore, optimum operations proficiency may be approximated but never achieved. This becomes important in setting achievement criteria for training.

- 5) Training should proceed from the simple to the complex, both in skill acquisition and in experiment training. There is some thought about starting with the more complex because of their criticality and the need for ample training time. However, it is conversely believed that if the PC trainee is allowed to progress from simple to complex, and if an adequate amount of time is allowed for learning each experiment regardless of its complexity, that the PC trainee will be developing the necessary skills to more easily handle the complex experiment when he gets to it. For example, by taking the simple experiments first the trainee is getting familiar and comfortable with the DDU/KB, experiment layout, ECOS, various ECAS/DEP, and the language in which the procedures are written. Then he gathers experience in nominal operations, simple off-nominals, and simple contingencies. All the while he is developing his skill to recognize subtle differences on the DDU displays, learns to discriminate messages, and is enhancing his intuitive knowledge of all the systems. Then when he encounters the more complex experiment, his prior knowledge and experiment experience actually aid in reducing some of the complexity factors of the experiment.
- 6) There should be sufficient training opportunities in the training schedule to allow the PC to work in the various possible dyads that might actually be chosen to fly the various Spacelab missions. Again this is not done to gather information for selection purposes (since the Investigators Working Group (IWG) will actually make the choices), but rather to give the dyads themselves the opportunity to better learn each others style of operation and idiosyncrasies prior to the flight crew selection.
- 7) The focus of training at the PCTC is on CDMS controlled experiments. Non-CDMS experiments will only be addressed as they mesh into the timeline training, i.e., while the PC is conducting a CDMS supported experiment, his operations timeline might call for him to initiate a life sciences experiment, hence, he might walk over to the life-sciences rack and flip a dummy switch and then resume his CDMS procedures.

TRAINING

The intent of this section is to discuss how the PCTC will actually be utilized during training. At present, however, all of the information necessary to make such a plan is incomplete. Therefore, this section represents an estimation of how that training might progress.

Traditionally, training is approached as the compilation of its sequential parts. That sequence is usually thought to be assessment, objectives, materials, procedures, and evaluation. That is generally how training has been approached in the present endeavor.

Assessment in this situation refers to pretraining assessment. Three questions have been explored here; 1) why have pretraining assessment, 2) what type assessment is mandated, and 3) who will do the assessment? The why was partially addressed in the previous philosophy section; specifically, the PCTC trainers need to determine how well they are conducting the training. In order to achieve that goal, it will be necessary to determine where the PC trainee is in his training when he arrives at the PCTC. This will allow the PCTC trainers to assess how the trainee has progressed during training and then devise ways to better conduct future training. The second aspect of the why question deals with determining entry level abilities at skills basic to CDMS operation. At present, five areas have been tentatively identified as being basic to all experiment operations, and as such will comprise the initial assessment/training period at the PCTC. The five areas are:

1. Keyboard operation; skill and location
2. CDMS entry; recognizing different DDU displays, functions, etc.
3. ECOS operations
4. Communications; transmitting accurately and quickly with POCC
5. Timeline operations; getting the total picture of when everything is supposed to run during the mission.

The type of assessment could be group or individual discussions or written exams to determine overall understanding of the various areas. This assessment could precede, run concurrent with, or follow PC trainee demonstration of skill proficiency in those areas requiring manipulative skills. At present, our thinking is that a group of PCTC trainers could do the assessment, supported with statistical records supplied by the HS on key stroke rate, error, and ability to call up various pages and experiments.

The objectives of training would include the development of high operations proficiency and overall experiment competence. Operations proficiency would include the PC's ability to perform the five areas listed previously, their ability to follow experiment procedures in a timely fashion, and the identification of critical sequences in the experiment operations that require special attention, which is partially a function of the PC's decision making abilities.

Once proficiency in the five basic areas has been acquired, two objectives become primary. The first deals with the PC's successful mastery of experiment procedures. The second deals with developing proficiency in the skills necessary to carry out the first objective. These "critical" skills would be above and beyond those skills mastered during training in the five basic areas. At present, however, these skills have not been identified. This condition exists for two reasons. The

first is that the procedures for all the experiments have not been written, which precludes looking at them to identify critical sequences. The second reason is that skill identification has not been prioritized; that is, at present some feel there is questionable doubt as to whether such skills exist, and if they do, they would be developed when the procedures are learned and practiced during training.

At issue too is a philosophical question as to whether training, and hence mission operations, is simple or complex. Is it simple in that all the PC has to do is follow the written procedures which will caution for upcoming critical sequences. The notion here is that all the PC needs to do is develop very high proficiency at knowing and implementing the already spelled out procedures, and then relying on the POCC if questions arise. And while it may be true that the POCC may understand each experiment in more depth than the PC, the POCC has not necessarily developed the procedural knowledge to the degree that the PC has and may be of little substantive help to the PC in some unforeseen situations.

The other view is that good scientific enquiry requires complex thinking skills. In this view the PC has the capability to conduct the experiment as he best sees fit, due to the fact that at that instant he possesses the most advanced conceptual understanding of the entire mission. Scientific enquiry demands sharp decision making skills, and as such these should be addressed during training at the PCTC. If this complex view is ascribed to, then training should afford direct or indirect opportunities to make decisions based on prior knowledge of and training in critical sequences or contingencies in each experiment.

Skill identification can start by asking and answering the following questions, pulled from Ely (1962):

"In the main, the definition of critical task characteristics is most effectively accomplished by considering the consequences of a particular activity if it were accomplished poorly or required excessive time for completion. Asking the following questions with respect to the various tasks is helpful for determining criticality.

Would below-minimum performance:

- o lead to an accident?
- o result directly in mission failure?
- o be impossible to remedy within time constraints or not at all?
- o be difficult to detect because of inadequate information feedback?
- o recur over time in such a way as to produce a cumulative effect?
- o contribute a large proportion of time to the total time required for some larger and critical function?

An affirmative reply to at least one of the above questions implies that the activity in question should be further considered in determining the critical behaviors for measurement."

In order to approach this issue, a skill matrix is being compiled from the available experiment procedures. Actually, two matrices are being developed. Both are modifications of existing skill charts (Rabideau, 1964; and Smode, Gruber, and Ely, 1962). Both are felt will yield useful information and will hopefully validate each other (the original charts and their modifications are in Appendix A). In the final evaluation, the skill matrices might very well show that training is in fact "simple" as described previously, but then at least there would be more support for that position.

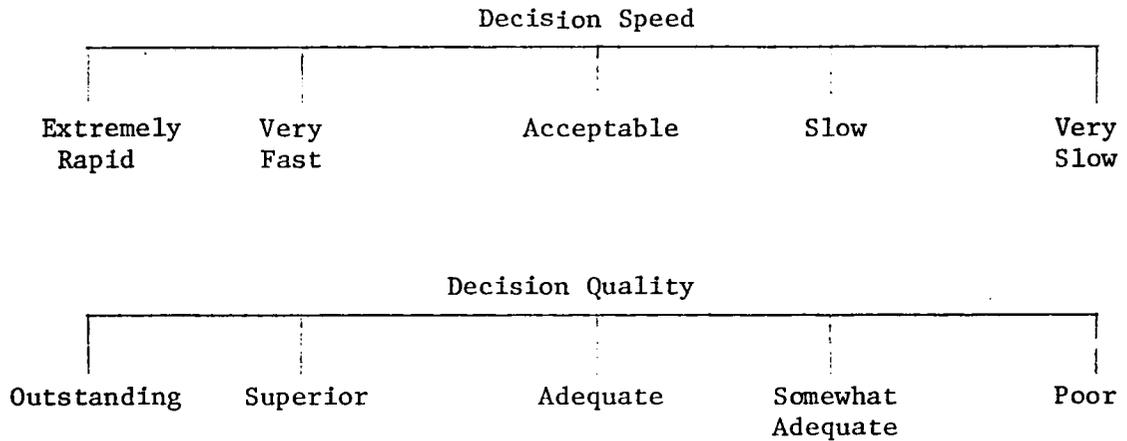
Following skill identification, training procedures can be more fully articulated using the present training definitions. These definitions exist for each experiment and are described in three levels, from A to C. The descriptions progress from a general description (A), to typical off-nominals (B), and finally to step-by-step procedures (C). These skills should then be incorporated into the level A and B experiment descriptions. The level A, B, and C descriptions could then be utilized additionally by; 1) sending out the A and B descriptions as part of the PC's pre-arrival training materials, and 2) using all three levels during training to brief each experiment from and then to train from. "A" descriptions should reflect time of training, the skills addressed, and where the training will take place. "B" descriptions should identify the training personnel, the possible off-nominals, and critical experiment stages; i.e., where contingencies exist or where the experiment could be aborted or drastically effected via operator error. "C" descriptions represent the training guide the PC might follow during training. "C" descriptions list nominal and off-nominal procedures.

After initiation to the experiment during the pretraining briefing and exposure to experiment procedures, but before going to the mockup for practice, the PC should be able to demonstrate the following:

- 1) What is the PC's knowledge of the experiment; communicated verbally?
- 2) What is the PC's knowledge of the possible off-nominals for this experiment and where will they occur?
- 3) What are the PC's perceptions of his anticipated responses to those off-nominals?
- 4) Can the PC verbally walk through the experiment's nominal run?

Evaluation criteria follow from skill identification and should be fully planned for prior to the implementation of training. At this point, four recommendations are made for training:

- 1) Pertinent PCTC trainers should go through the PC training program prior to any PC trainee to debug and improve on the system.
- 2) If the modified Ely chart for skill identification is validated, then his three instrument and criteria measures in Appendix B can be used to develop PCTC evaluation criteria and instruments.
- 3) Additionally, video tapes, PC record books, debriefing questionnaires, and data graphs of performance can be collected to trace training progress
- 4) Two additional types of measures have been located in our review that have application to the current evaluation needs. One is an effectiveness rating scale:



The second is an assessment tool of procedural operations:

<u>Task Step</u>	<u>Error & Consequences</u>
1. Turn power switch to ON position	_____
2. Read external temperature	_____
3.	_____
4. } etc.	_____
5.	_____

The final area addressed in this report is the matter of developing a training schedule. This can not be developed at present until the skill identification is completed. Additional parameters in formulating a schedule would be:

- 1) A need to determine the total time for experiment training, so that a training schedule can be developed.
- 2) How much time is needed for the dyads to work in each of the possible pairs.
- 3) Number of actual training locations within the PCTC for the PC to be trained at.

APPENDIX A

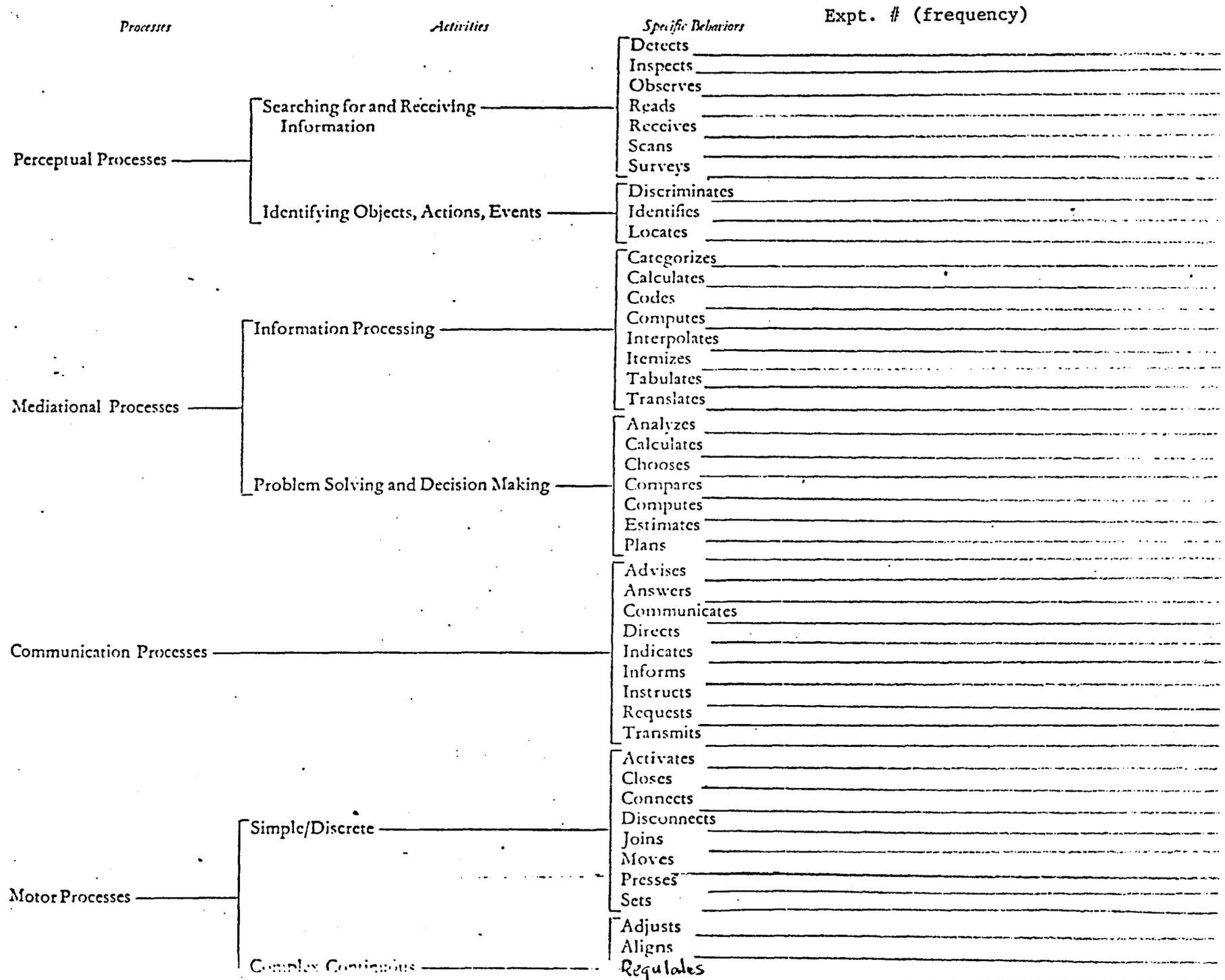
Modified skill identification charts used to identify
PC skills for training definition.

← NASA

→ ESA →

Types of Tasks		EXPERIMENT #
Simple	Follow Routine, Pre-established Checkout Sequence	001
	Position a Set of Discrete/Serial Controls	002
	Obtain Information From Dynamic Displays	003
	Adjust Instrument or Other Equipment Item	005
	Track	008
	Communicate Information	011
	Utilize Status or Reference Data	013
	Directly Observe External Events	014
	Compute	016
	Select Action From Among Alternative Choices	017
	Follow Established Procedures in Inspection, Calibration, and/or Checkout.	019
	Acquire Information From Among Multiple Sources	020
	Make Multiple Control Actions	021
	Locate Malfunctions in Complex Equipment	022
Complex	Remedy, Repair, Fabricate, and/or Assemble Complex Equipment.	023
	Carry Out Computational/Masurement Sequences for Determining Status	024
	Make Decisions Based on Multiple Information Sources	027
		029
Other		033
		034
		300

Modified from Smode, Gruber, and Fly, 1962



Modified from Rabideau, 1964

TABLE 8-2B

Application of Task Data for Error, Frequency, and Achievement Measures
(Based on Smode, Gruber, and Ely, 1962)

	Types of Tasks	Types of Measures																					
		Error Amplitude	Error Frequency	Error Content Analysis	Error Change Over Time	Number of Control Responses and Manipulations	Number of Communications	Number of Personnel Interactions	Number of Diagnostic Checks	Number of Errors	Number of Out-of-Tolerance Conditions	Number of Observations	Number of Verbal or Written Reports	Number of Requests for Information	Per cent of Activities Accomplished	Measures of Achieved Reliability	Measures of Achieved Maintainability	Equipment Failure Rate	Cumulative Response Output	Proficiency Test (Written) Scores	Magnitude Achieved-Steady-State Value	Magnitude Achieved-Changing Value	
Simple	Follow Routine, Pre-established Checkout Sequence	X	X	X				X	X	X				X						X			
	Position a Set of Discrete/Serial Controls		X	X		X			X	X													
	Obtain Information From Dynamic Displays	X		X					X		X										X	X	
	Adjust Instrument or Other Equipment Item	X	X	X		X			X														
	Track	X	X	X	X	X			X	X									X		X	X	
	Communicate Information			X			X	X	X				X	X									
	Utilize Status or Reference Data	X		X					X											X	X	X	
	Directly Observe External Events	X	X	X					X		X										X	X	
	Compute	X	X	X					X												X		
	Select Action From Among Alternative Choices	X	X	X					X											X			
Complex	Follow Established Procedures in Inspection, Calibration, and/or Checkout.	X	X	X				X	X	X				X						X			
	Acquire Information From Among Multiple Sources	X		X					X		X		X									X	
	Make Multiple Control Actions	X	X	X	X	X			X	X													
	Locate Malfunctions in Complex Equipment			X					X	X	X					X	X			X			
	Remedy, Repair, Fabricate, and/or Assemble Complex Equipment.	X	X	X		X			X	X						X							
	Carry Out Computational/Masurement Sequences for Determining Status	X	X	X						X			X							X	X	X	
Make Decisions Based on Multiple Information Sources	X	X	X						X										X				

TABLE 8-2A

Application of Task Data for Latency and Accuracy Measures
(Based on Smode, Gruber, and Ely, 1962)

	Types of Tasks	Types of Measures																								
		Time to Perceive Event	Movement Reaction Time	Time to Initiate Correction	Time to Initiate Activity Following Completion of Prior Activity	Time to Detect Trend of Multiple Related Events	Time to Identify Stimulus	Time to Complete Message, Decision, Control Adjustment	Time to Reach Criterion Value	Time Spent in Activity	Per Cent Time on Target	Time-sharing among Events	Accuracy in Identifying Displays Internal to System	Accuracy in Identifying Displays External to System	Accuracy in Estimating Distance, Direction, Speed, Time	Detection of Change in Stimulus Over Time	Detection of Trend Based on Multiple Related Events	Observation Correctness	Recognition of Signal in Noise	Recognition of Out-of-Tolerance Condition	Accuracy in Control Positioning or Tool Usage	Accuracy in Reading Displays	Accuracy in Symbol-Use, Decision-Making, Computing Among Alternatives	Accuracy in Serial Response	Tracking Accuracy	Communications Accuracy
Simple	Follow Routine Pre-Established Checkout Sequence								X		X	X			X		X		X		X					
	Position a Set of Discrete/Serial Controls							X	X		X									X				X		
	Obtain Information From Dynamic Displays	X					X	X	X			X	X	X	X		X	X	X		X					
	Adjust Instrument or Other Equipment Item							X	X												X				X	
	Track		X	X					X	X	X				X	X					X			X	X	
	Communicate Information							X	X	X															X	
	Utilize Status or Reference Data								X	X																
	Directly Observe External Events	X				X	X		X	X			X	X	X		X	X								
	Compute								X	X														X		
	Select Action From Alternative Choices							X	X	X														X		
	Follow Established Procedures in Inspection, Calibration and/or Checkout				X			X	X	X			X					X		X	X	X	X			
	Acquire Information From Among Multiple Sources				X			X	X	X			X	X	X	X	X	X				X	X			
Make Multiple Control Actions			X	X			X	X	X											X				X		
Locate Malfunctions in Complex Equipment	X	X					X	X	X			X					X		X				X			
Remedy, Repair, Fabricate, and/or Assemble Complex Equipment							X	X	X											X						
Carry out Computational/Measurement Sequences for Determining Status									X					X	X								X			
Make Decisions Based on Multiple Information Sources		X	X	X			X	X	X							X						X	X			

TABLE 8-2C

Application of Task Data for Consumption, Physiological and Behavioral Judgment Measures
(Based on Smode, Gruber, and Ely, 1962)

	Types of Tasks	Types of Measures														
		Fuel/Energy Conservation	Units Consumed in Activity Accomplishment	Rate of Consumption	Operator/Crew Physiological Condition	Operator/Crew Behavioral Condition	Rating of Operator/Crew Performance Adequacy	Rating of Task Performance Adequacy	Estimation of Amount or Degree of Behavior Displayed	Analysis of Operator/Crew Behavior Characteristics	Omission of Relevant Behavior	Occurrence of Nonrelevant Behavior	Description of Out-of-Tolerance Condition	Interview Content Analysis	Self-Report of Experiences	Peer, Self, or Supervisor Ratings
Simple	Follow Routine Pre-established Check-out Sequence						X	X	X	X	X	X		X	X	X
	Position a Set of Discrete/Serial Controls						X	X	X	X	X	X		X	X	X
	Obtain Information From Dynamic Displays						X	X	X	X	X	X		X	X	X
	Adjust Instrument or Other Equipment Item						X	X	X	X	X	X		X	X	X
	Track						X	X	X	X	X	X		X	X	X
	Communicate Information						X	X	X	X	X	X		X	X	X
	Utilize Status or Reference Data						X	X	X	X	X	X		X	X	X
	Directly Observe External Events						X	X	X	X	X	X		X	X	X
	Compute						X	X	X	X	X	X		X	X	X
	Select Action from Among Alternate Choices						X	X	X	X	X	X		X	X	X
Complex	Follow Established Procedures in Inspection, Calibration, and/or Checkout						X	X	X	X	X	X		X	X	X
	Acquire Information from Among Multiple Sources.						X	X	X	X	X	X		X	X	X
	Make Multiple Control Actions						X	X	X	X	X	X		X	X	X
	Locate Malfunctions in Complex Equipment						X	X	X	X	X	X	X	X	X	X
	Remedy, Repair, Fabricate, and/or Assemble Complex Equipment		X	X			X	X	X	X	X	X		X	X	X
	Carry Out Computational/Measurement Sequences for Determining Status						X	X	X	X	X	X		X	X	X
	Make Decisions Based on Multiple Information Sources						X	X	X	X	X		X	X	X	

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1980

NASA/ASEE SUMMER FACULTY RESEARCH FELLOWSHIP PROGRAM

MARSHALL SPACE FLIGHT CENTER
THE UNIVERSITY OF ALABAMA

SOME METHODS IN HIGH ENERGY
COSMIC RAY MEASUREMENT

Prepared By: Julian B. Shand, Jr., Ph.D.
Academic Rank: Professor
University and Department: Berry College
Department of Physics
NASA/MSFC
(Laboratory) Space Sciences
(Division) Space Physics
(Branch) Astrophysics
MSFC Counterpart: T. A. Parnell
Date: August 1, 1980
Contract No.: NGT-01-002-099
(University of Alabama)

SOME METHODS IN HIGH ENERGY COSMIC RAY
MEASUREMENT

By

Julian B. Shand, Jr., Ph.D.
Professor of Physics
Berry College
Mt. Berry, Georgia

ABSTRACT

The response of ion chamber detectors to cosmic rays of energy a few GeV/nucleon and up shows a logarithmic rise with energy. The energy resolution has not been accurately defined. In Ref. 1 a calculation technique has been outlined which gives useful results for cosmic ray protons. The present study has applied the method to much heavier particles and higher energies.

In addition, a balloon flight will soon detect cosmic ray showers. A detector of acoustic waves produced by the showers will be carried, as will emulsion detectors. The present study has involved calibration of a flat plastic scintillation plate which will act as a trigger for the acoustic detector, and will determine the position of exit of the showers in order to expedite visual examination of the emulsions.

INTRODUCTION

Cosmic rays have been investigated since their discovery in 1911. Their flux in space is nearly independent of time and direction. They consist of nuclei, electrons, and electromagnetic radiation. The nuclei are mostly protons (around 90%) and helium (around 9%) but heavy nuclei are present also, being over represented as compared to solar abundances. The energy spectrum extends to the greatest energies known, reaching as much as 10^{20} eV or 1 joule.

The origin of cosmic rays is a mystery, as is their acceleration to ultra high energy. Exploding galaxies, supernovae, pulsars, white dwarfs, and other exotic types of objects have been proposed as sources, but none have been clearly identified.

Measurement of the energy, knowledge of the energy resolution of the measuring devices, and identification of the various nuclear species are important for an understanding of the phenomena. A logarithmic rise in ionization of gas in ion chamber detectors, as cosmic ray energy increases, has long been observed, for energy a few GeV/nucleon and up. The energy resolution has not been accurately defined. In Ref. 1 a procedure has been outlined for calculating resolution for the case of particles of charge 1. This type information is needed also for the heavy nuclei in cosmic rays, and work on this problem is described in the following section.

Flat plates of plastic scintillator can serve several functions in a balloon-borne cosmic ray experiment. Rays passing through a plate produce light in the plate, which can be collected and measured by photomultiplier tubes. The signals from these tubes can indicate the passage of a "shower" of particles produced by collision of a high-energy cosmic ray with a nucleus. These signals can thus be used as a trigger for another detector designed specifically for measuring the energy of such showers. In addition, the relative strengths of the signals from the photomultiplier tubes can be used to determine the place of exit of the particles. This information facilitates examination of nuclear emulsions located next to the plastic scintillator, in order to find tracks in the emulsion created by the particles passing through.

OBJECTIVES

Two main objectives were pursued. The first was to calculate the average energy actually deposited in an ion chamber by an ultra-high energy particle of large mass and charge. The method was to extend a calculational scheme already applied successfully to particles of charge 1.

The second objective was to calibrate a plate of plastic scintillator for measurement of the position of a cosmic ray shower passing through it. The method of calibration was to inject pulses of light at known positions on the plate and record the responses of photomultiplier tubes at the corner of the plate.

BODY OF REPORT

On Relativistic Rise in Ionization

Relativistic charged particles passing through a gas-filled chamber lose energy mainly by excitation and ionization of the gas atoms (Ref. 4). Experiment shows a slow logarithmic increase in energy loss as particle energy increases. This is called the relativistic rise. The rise finally stops and energy loss becomes constant in what is called the Fermi plateau. Examination of the relativistic rise and knowledge of the energy resolution can allow one to determine the energy of the particle. This requires that charge be measured independently by other detectors through which the high energy particles pass. Typically a Cerenkov counter utilizing solid radiators such as lucite plastic is used for charge identification in the ultra-relativistic energy range. Thus for cosmic ray work the particular kind of nucleus striking the detector can be found (Ref. 5).

One way to look at the interaction between an incident particle and the electrons of an absorber is by representing the electromagnetic field of the particle by two terms (see Fano, Ref. 2). One, called the longitudinal term, is merely a static Coulomb interaction. The second term, called the transverse term, involves emission and reabsorption of virtual photons. Fano has treated the interaction in the lowest order (Born) relativistic approximation. The longitudinal interaction is effectively constant at high velocities. The transverse term, however, increases in strength as velocity increases. This is expected since the transverse field increases proportional to γ , where

$$\gamma = (1 - \beta^2)^{-\frac{1}{2}} \text{ and } \beta = v/c .$$

This relativistic rise eventually stops because of polarization of the absorbing medium.

Fano found expressions for differential cross sections for ionization. By integrating over energy he found energy loss. This approach does not agree well with experiment for the case of thin gas absorbers (Ref. 1,3) because the energy loss occurs in a relatively few discrete collisions. Cobb et al. (Ref. 1) have used Fano's results up to a point but instead of integrating over energy have used Monte Carlo techniques and calculated energy losses in discrete collisions many times to get an average energy loss, a spectrum of energy loss, and the amount of relativistic rise. The results are in good agreement with experiment for incident particles of charge 1.

This author has adapted Cobb's method, which came from Fano, to the case of particles of large charge. Several difficulties were encountered:

A. Cobb's use of the letter E in expressions was thought to be ambiguous: it could mean energy transfer to an electron, which would include the binding energy of the electron, as Fano seems to use it: or it could mean the kinetic energy of the electron after collision, as is suggested by Cobb's referring to the Rutherford cross-section as being

$$\frac{2\pi}{mv^2} \frac{1}{E^2}$$

In this expression Rutherford did mean E as kinetic energy of the electron.

B. Cobb's expression for the longitudinal differential cross-section for ionization included the factor (with units 1/energy)

$$\delta(E-E_i)$$

where E_i is the electron binding energy. This has the form of the Dirac delta function, which has meaning when integrated. Since integration is not used in Cobb's method, it is not clear how to use it.

C. Cobb's expression for the dielectric constant ϵ of the absorber gas contains the frequency ω . This seems ambiguous: earlier in his paper the same variable had been used as the frequency of a virtual photon. However, Fano wrote that the ω in the dielectric constant expression is a root of the equation

$$c^2q^2 - \epsilon(\omega) = 0$$

where c is the speed of light and q is momentum transfer to the electron. Since Cobb's treatment follows Fano, an uncertainty in interpretation is introduced.

Consultation with some experts on energy loss and a search of the literature have shed little light on these difficulties. Communication with the authors (Fano, Cobb et al.) will be attempted to resolve the ambiguities. This author has written computer code to calculate cross sections for ionization, using various combinations of interpretations, in order to be able to distinguish the correct ones. The work is still in progress.

Calibrating A Plastic Scintillator for Position Determination

In September 1980, a balloon flight will carry several cosmic ray experiments. Events of the highest energy will be of particular interest. Nuclear emulsions, X-ray film, and calorimetry will measure characteristics of these events. A plastic scintillator can be configured to determine the position of exit of cosmic rays. The rays cause the plastic to emit light. Most of the light is trapped in the thin plate of plastic by total internal reflection. When photomultiplier tubes are optically coupled to the corners of the plate, the tubes measure the amount of light reaching them. The amount varies inversely with the distance to the event, roughly. Several factors complicate the tube response, including edge effects, scratches, crazing of the plastic, inhomogeneity of the scintillating material, and probably others.

Preliminary calibration of the scintillator was done by this author. The method required a light-tight box. The scintillator plate, which measured 44" by 44" by 1/4", lay in the box on white paper which had a grid of horizontal and vertical lines drawn on it, with 10 cm spacing. A photomultiplier tube was attached with optical coupling grease to a specified position at one corner of the plastic plate. The photomultiplier tube was a 3", ten stage EMI, type 9708KB. The output of the tube was amplified and referred to a multichannel analyzer which displayed pulse heights and allowed a determination of average pulse height.

Light pulses were obtained by sending electric pulses at frequency 10Khz to a light-emitting diode (LED). The LED was mounted at the top of a shiny brass tube whose open end fit into a clear plastic disk. The disk, with LED, could be stuck to the plastic scintillator with optical grease and accurately located above an intersection of grid lines on the paper underneath. Thus positions of photomultiplier tube and LED were known to about $\frac{1}{2}$ cm.

After pulses had been injected for 90 seconds and the average pulse height ascertained, the disk with the LED was removed and the plate wiped with dry cotton material and then with the same kind of material moistened with pure ethyl alcohol. All other procedures tried left some residue of film which scattered light out of the plate.

After the cleaning, the LED disk was regreased and re-positioned. In this way a map was generated. Upon completion, the photomultiplier tube was moved to another corner and another map generated. Only one photomultiplier tube was ready at the time, so 3 other "dead" tubes were kept at the 3 other corners to absorb light as would happen in the final setup.

Checks were made of the reproducibility of the reading from a particular point. Deviations ranged from 0 to 12%, averaging about 5%.

A sample of experimental results is shown here. The positions refer to a Cartesian coordinate with the plastic scintillator plate defining the first quadrant, and the photomultiplier tube centered at $x = 5.9$ cm, $y = 5.9$ cm. Pulse heights along an edge and along an approximate diagonal are given. Positions are not integral multiples of 10 cm because visualization is easier than in the system actually used.

Position (cm)			Position		
x	y	Pulse height	x	y	Pulse height
21.8	10	2740	21.8	20	1640
31.8	10	1260	31.8	30	720
41.8	10	740	41.8	40	410
51.8	10	450	51.8	50	280
61.8	10	300	61.8	60	190
71.8	10	240	71.8	70	180
81.8	10	180	81.8	80	160
91.8	10	130	91.8	90	130
101.8	10	110	101.8	100	120

At several positions in the scintillator pulse heights had local maxima or minima. Some of these could be identified with defects mentioned in the first paragraph under calibrating.

CONCLUSIONS AND RECOMMENDATIONS

Concerning relativistic rise in ionization, an attempt should be made to locate the authors of Ref. 1 and correspond with them to resolve the difficulties of interpretation of their paper. In addition, the computer trial and error attempts to find the correct interpretation should be continued in case all else fails.

Calibration of the plastic scintillator as a determiner of position has been done. The experimental error indicates a positional error of nearly 5 cm can be expected. Since the calibration was done with light injected from outside, while the actual use will have light generated inside the plastic, a check of the numbers should be made with all 4 photomultiplier tubes in place, mounted in final form. Energetic radiation which will cause the plastic to scintillate should be injected from outside.

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1980

NASA/ASEE SUMMER FACULTY RESEARCH FELLOWSHIP PROGRAM

MARSHALL SPACE FLIGHT CENTER

THE UNIVERSITY OF ALABAMA

PHOTOCHEMISTRY OF MONODENTATE AND BIDENTATE

CARBONATO COMPLEXES OF RHODIUM (III)

Prepared By:	Peter S. Sheridan
Academic Rank:	Assistant Professor
University and Department:	Colgate University Department of Chemistry
NASA/MSFC:	
(Laboratory)	Space Sciences Laboratory
(Division)	Materials Processing in Space
(Branch)	---
MSFC Counterpart:	Robert J. Naumann
Date:	August 2, 1980
Contract No:	NGT-01-002-099

PHOTOCHEMISTRY OF RHODIUM (III)-CARBONATO COMPLEXES

by

Peter S. Sheridan
 Department of Chemistry
 Colgate University
 Hamilton, New York 13446

ABSTRACT

A scheme for the photochemical fixation of water (eq 1) has been proposed.

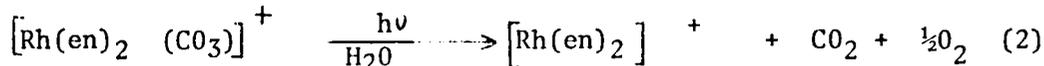


Fuel cells which generate electrical energy using the reverse of eq 1 are used in NASA spacecraft, so the solar-induced regeneration of H₂ and O₂ would allow the same materials to be recycled through the fuel cells, dramatically increasing energy output per pound of fuel carried aloft.

The proposed scheme involved a 5-step reaction sequence; the first step involves the 2-electron reduction of a metal by a coordinated carbonate ligand, with corresponding oxidation of the carbonate to CO₂ and O₂.

Ligand field photolysis of trans- [Rh(en)₂ (H₂O)(CO₃)] ClO₄ (I), and [Rh(en)₂ (CO₃)] ClO₄ (II) have been studied in the solid state and in aqueous solution at various pH values. Both salts are photoinert in the solid phase, but are quite photoreactive in aqueous solution. In solution, the monodentate ion (I) undergoes efficient isomerization to a mixture of cis and trans - [Rh(en)₂ (H₂O)(CO₃)]⁺, presumably with water exchange. A minor pH increase upon photolysis is evidence of inefficient carbonate (CO₃⁼) release, with formation of [Rh(en)₂ (H₂O)₂]³⁺.

In contrast, aqueous solutions of the bidentate carbonato complex, (II) undergo efficient pH decrease upon ligand field photolysis. Changes in the electronic spectrum (200-500 nm) and pH changes indicate that the desired redox is occurring (eq 2)



The pH increase is due to the aqueous behavior of CO₂ (eq 3).



A positive test for released carbonate ion supports this assignment. Work is proceeding to confirm this new photochemical reaction, with emphasis on gaining understanding of the exact nature of the rhodium photoproduct, the details of the photochemical mechanism, and to test the generality of photoredox reactions of bidentate carbonato complexes.

ACKNOWLEDGEMENTS

I wish to thank Dr. Robert Naumann for allowing a stimulating summer of research, without memos or committees; it was great. Mr. Charles Carter and Mr. Marion Kent were instrumental in facilitating my application, and I thank them for their help and support. Ms. Margie Phillips typed this manuscript, for which I am most grateful. Finally, I owe a debt to Dr. Robert Snyder who found room in his Separations Processes Lab for an inorganic photochemist.

NOMENCLATURE

- LMCT - ligand-to-metal-charge-transfer
- en - ethylenediamine ($\text{H}_2\text{NCH}_2\text{CH}_2\text{NH}_2$)
- K - equilibrium constant ($K = 4.3 \times 10^{-7}$ for hydrolysis of CO_2)
- $$\text{CO}_2 + \text{H}_2\text{O} \rightleftharpoons \text{H}^+ + \text{HCO}_3^-$$
- h ν - implies the reaction conditions included the incidence of light
- nm - nanometer

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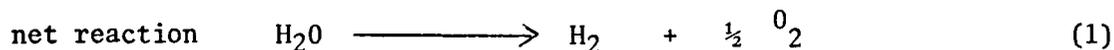
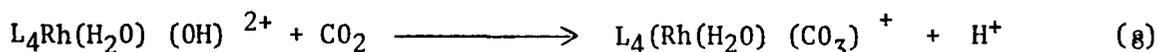
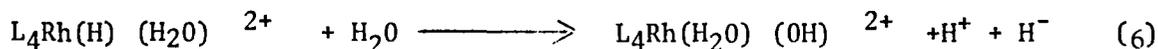
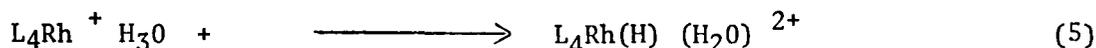
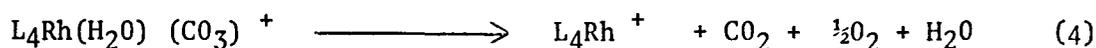
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INTRODUCTION

The photochemical behavior of Rh(III) complexes has attracted considerable interest¹, as a variety of photoinduced aquations and stereochemical rearrangements have been observed. While exceptions exist, Rh(III) amine complexes characteristically undergo efficient photoinduced ligand loss/aquation, with rearrangement to a symmetric trans configuration. There is little evidence of any photoinduced reduction of the Rh(III) center, even upon photolysis of LMCT bands in such complexes as $[\text{Rh}(\text{NH}_3)_5\text{I}]^{2+}$; the instability of the Rh(II) oxidation state and the photoreactivity of the ligand field excited states lead to a predominantly ligand field excited-state chemistry for Rh(III) amines.

Our work on the photochemistry of rhodium complexes has led to a reaction sequence which, on paper, represents the photochemical cleavage of water into H_2 and O_2 . It is a five step reaction sequence (eq 4-8); the net reaction is the desired water fixation.



The first step is the photoinduced oxidation of a d^6 metal center (Rh(III) here) by a carbonate ligand, generating O_2 , CO_2 and a coordinatively unsaturated d^8 complex (eq 4). Oxidative-addition of H_3O^+ then regenerates the Rh(III) center, creating a hydride ligand (eq 5). Photoinduced (or thermal) labilization of this pseudohalide ligand (eq 6) would lead to instantaneous H_2 generation (eq 7). Under the proper pH conditions, reaction of the photo-generated CO_2 (eq 1) and the aquohydroxo complex (eq 8) would regenerate the carbonate starting material (eq 8).

This reaction sequence has many advantages over other currently studied water-photolysis schemes:

a) Each of the steps is either a well-known reaction (eq 7 and 8) or is a reaction for which closely analogous reactions have been studied (eq 4, 5, and 6).

b) No radical intermediates are involved, as the 2-electron transfer of the d^6 - d^8 transition metal system allows complementarity.

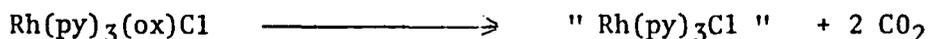
c) The scheme is a self-contained, catalytic cycle, with only water consumed and H_2 and O_2 generated.

d) The scheme is quite flexible, as many variables can be adjusted.

A major difficulty with this scheme is the first step, as there is little precedence for such a photoinduced two-electron transfer. An additional problem is that reports on the photochemistry of carbonato complexes are surprisingly rare.

An apparent two-electron transfer was observed upon irradiation of alcohol solutions of some carbonato complexes of Pt(II)³. In the presence of nucleophiles ($X = C_2H_4, P\phi_3, CO, \phi C \equiv C\phi$) photolysis of $PtL_2(CO_3)$ ($L = P\phi_3, As\phi_3$ or $P(CH_3)\phi_2$) generates PtL_2X_n ($N = 2$ for $X = CO$; $n = 1$ for bulkier ligands), consistent with the in situ formation of PtL_2 . Carbon dioxide was released during the photolysis (one mole per mole of Pt) and alcohols added to the photolysis solutions could be oxidized to aldehydes and ketones, confirming that the carbonato ligand was the two-electron reducing agent.

Such apparent two-electron transfers have also been observed upon photolysis of some oxalato complexes. Some $PtL_2(ox)$ complexes, similar to the carbonato complexes just discussed, photolyze to give CO_2 and similar Pt(0) photoproducts.⁴ Irradiation of $Pd(NH_3)_2(ox)$ was reported⁵ to generate CO_2 and a reactive species identified as $Pd(NH_3)_2$, which decomposed to give Pd black. Poulenc⁶ noted that the yellow $Rh(py)_3(ox)Cl$ solid turned red in sunlight, and Gillard et. al⁷ have shown the reaction involves another photoinduced two-electron transfer



Work at the Space Sciences Laboratory this summer focused on this first step, the photolysis of Rh(III) carbonato complexes. Two complexes were chosen for study - trans - $[Rh(en)_2(H_2O)(CO_3)]^+$ and $[Rh(en)_2(CO_3)]^+$ - as they allowed a study of the affect of monodentate vs. bidentate coordination of the carbonate ion on the resulting photochemistry.

OBJECTIVES

Research was directed toward an understanding of the following questions:

1. What is the photochemical behavior of Rh(III)-carbonato complexes?
2. Do the charge-transfer and ligand field states of these Rh(III) complexes display distinct photochemical reactivity, or does inter-system crossing lead to a common photoreactive state?
3. What affect does monodentate vs. bidentate coordination of the carbonato ligand have on the resulting photochemistry?
4. Are such complexes photosensitive in the solid state, or is an aqueous solution necessary for photoreactivity? In solution, does the pH affect the photochemical reaction.

The objective of the work was to determine if photoinduced two-electron reduction of a Rh(III) center could be induced by photolysis of the appropriate Rh(III)-carbonato complex.

EXPERIMENTAL

A. Preparation of Complexes

trans-Aquochlorobis(ethylenediamine)rhodium(III) Perchlorate

The trans- $[\text{Rh}(\text{en})_2(\text{H}_2\text{O})(\text{CO}_3)]^+$ ion was prepared by the method of Rerek⁸. A sample of trans- $[\text{Rh}(\text{en})_2(\text{Cl})_2]\text{NO}_3 \cdot \text{H}_2\text{O}$ ⁹ was anated with a five fold excess of NaN_3 (under reflux) and the desired trans- $[\text{Rh}(\text{en})_2(\text{N}_3)_2]^{2+}$ was crystallized as the BF_4^- salt upon addition of NaBF_4 . This diazido ion was dissolved in dilute HClO_4 , and NaNO_2 was added; this generated NO^+ , destroying the coordinated azide.



At a neutral pH (adjusted with LiOH) the resulting trans- $[\text{Rh}(\text{en})_2(\text{OH})(\text{H}_2\text{O})]^{2+}$ was precipitated as the perchlorate salt. A solution of this ion was acidified at pH 1 with HClO_4 , and NaHCO_3 slowly added until a pH of 5.80 was reached. The perchlorate salt was precipitated by cooling in ice, followed by dropwise addition of ethanol. The absorption spectrum had λ_{max} at 347 nm ($\epsilon = 105$), consistent with published data.

Carbonatobis(ethylenediamine)rhodium(III) Perchlorate

$[\text{Rh}(\text{en})_2\text{CO}_3]\text{ClO}_4$ was prepared by the method of Harris *et al.*¹⁰ It displayed a single peak in the electronic spectrum at 327 nm ($\epsilon = 250$); this absorption coefficient is less than the literature value, but any impurities were shown not to interfere in the photochemistry. There were no detectable changes in the electronic absorption spectrum of this ion in solutions between pH 1 and pH 13, clearly ruling out significant contamination of either of the monodentate ions, cis or trans- $(\text{Rh}(\text{en})_2(\text{H}_2\text{O})(\text{CO}_3))^+$.

Barium chloride solution

Aqueous solutions of barium chloride were prepared by dissolving $\text{BaCO}_3(\text{s})$ in 1 M $\text{HCl}(\text{aq})$ until effervescence ceased, and a pH of 7 (+1) was reached. The resulting solution was filtered and kept in a sealed flask to avoid CO_2 contamination. It was boiled for ca. 20 minutes before use.

Other reagents were used as supplied by the manufacturers.

B. Photolyses

Photolyses were performed on either solid samples or aqueous solutions of the complexes. Solid samples were finely ground with mortar and pestle, and then stuck on a piece of green tape (1 cm x 3 cm). This tape was suspended in a 1 cm fused silica cell (Beckman), connected to a microvolumeter (*vide infra*).

For the solution studies, a sample of the desired sample was dissolved in the appropriate aqueous solution at room temperature, and placed in a 1 cm fused

silica cell (Beckman). Electronic spectra were recorded with the sample in the same cell.

C. Equipment

The irradiation source was a 1000 watt Mercury-Xenon Lamp (Hanovia Manufacturing Corporation) with compatible power supply and starter (Schoeffel LPS 255 power supply and Model 359 igniter). Photolyses were performed at 28.5 amps at 22 V. Use of the compatible Schoeffel high-intensity monochromator did not allow sufficient intensity to pass to the sample, so wavelength selection was affected with a Corning colored glass filter (C-7-59). This filter effectively absorbed all radiation below 280 nm, but allowed irradiation of the ligand field bands. The absorption spectra of the C-7-59 filter along with the spectra of the two complexes studied is given in Fig. 1.

Electronic spectra were recorded on a Beckman Acta CII spectrophotometer with an external 10 inch recorder (Beckman Model 1005). The pH measurements were done with a PHM 64 Research pH Meter (Radiometer/Copenhagen) with a GK 2321C Combined Electrode. Samples averaged about 3.0 ml for pH measurements.

A microvolumeter was prepared after a design of Stevenson¹⁰. A 1 ml graduated pipet (graduated in 0.01 ml units) was used, and the evolution of a gas was detected by monitoring the flow of water bubbles down the pipet. Photolysis of solid samples of $[\text{Rh}(\text{en})_2(\text{N}_3)_2]\text{BF}_4$, which releases N_2 , was used to check the sensitivity of the microvolumeter.

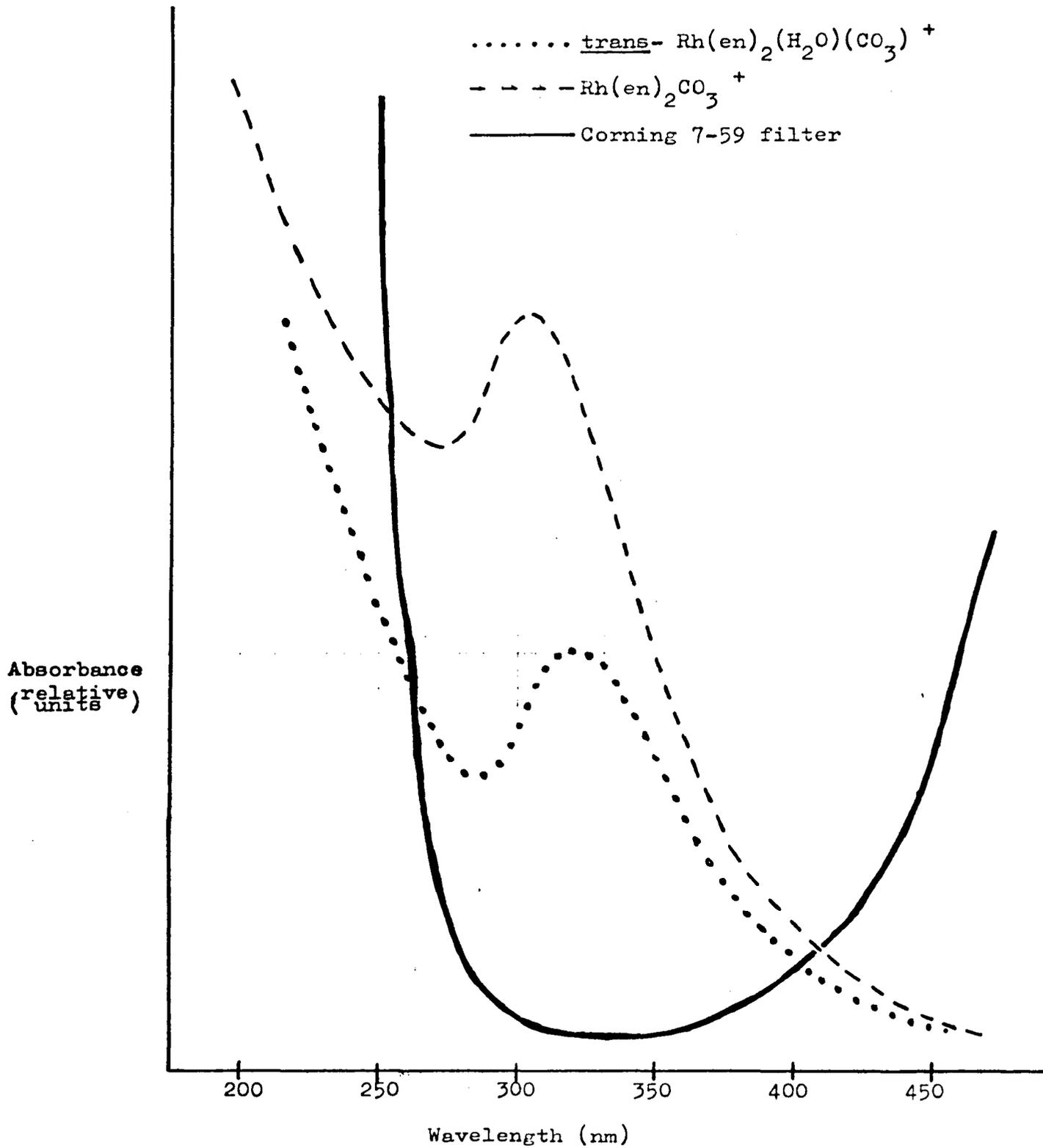


Figure 1

RESULTS AND DISCUSSION

A. Trans- $[\text{Rh}(\text{en})_2 (\text{H}_2\text{O}) (\text{CO}_3)]^+$

1. Results

In the solid state, trans- $[\text{Rh}(\text{en})_2 (\text{CO}_3) (\text{H}_2\text{O})] \text{ClO}_4$ is photoinert. Prolonged photolyses (2 hrs.) with both filtered and unfiltered light did not cause any detectable release of gas. When the photolyzed solid was dissolved in water, it had an electronic spectrum indistinguishable from that of an unphotolyzed sample.

In aqueous solution, and in the absence of light, $(\text{Rh}(\text{en})_2 (\text{H}_2\text{O}) (\text{CO}_3)]^+$ is relatively unreactive. At room temperature and at a solution pH between 6 and 10, there is a minor pH increase and a minor increase in the intensity of the ligand field band. This reaction is presumably the thermal aquation of the carbonate ion, but it was slow enough to have no measurable impact on the photochemical reactions.

Upon photolysis, the electronic spectrum undergoes pronounced changes (Fig. 2). Note that photolysis causes:

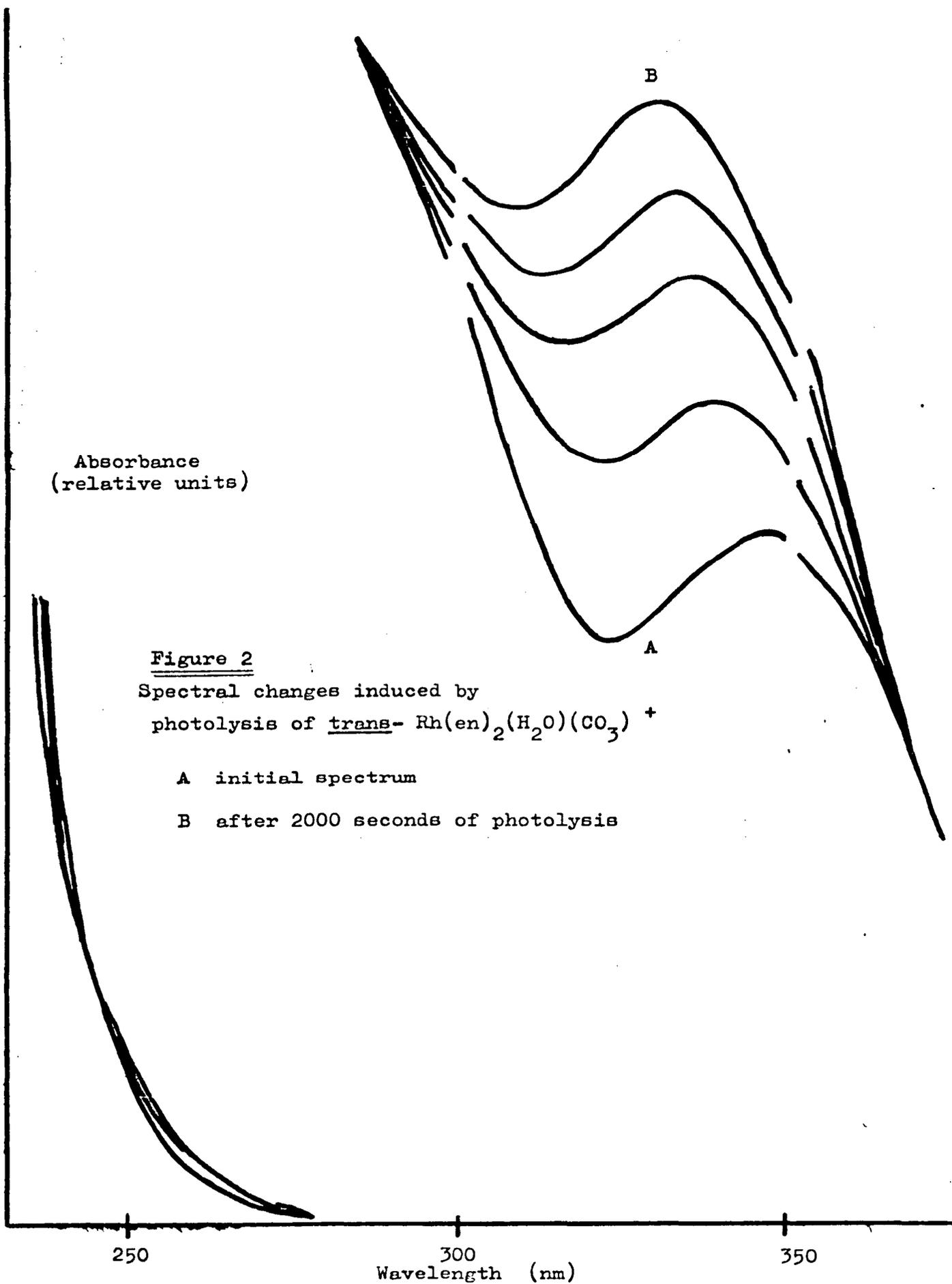
- a) an increase in intensity of the ligand field band,
- b) a blue shift of the peak maximum from 347 nm to about 332 nm,
- c) virtually no shift in the position of the low-energy edge of the charge-transfer band.

There appears to be an isosbestic point near 242 nm, although the spectra are nearly parallel at that point, so the presence of an isobestic is difficult to determine.

In addition to the spectral changes observed in Fig 2., photolysis causes the release of a base into solution. Data from a typical experiment are given in Table 1, where the pH changes observed in an unphotolyzed blank and photolyzed sample are listed. (The pH changes in Table 1 were recorded from the same experiment monitored in Fig 2.).

After extended photolysis, only minor spectral changes could be photoinduced. To determine the isomeric composition of this solution, 2 ml of conc HCl was added to a 3 ml sample of photoproduct, and refluxed for 1 hour. The resulting solution had an electronic spectrum of a mixture of cis and trans - $[\text{Rh}(\text{en})_2\text{Cl}_2]^+$ (ca. 63% cis and 37% trans).

The specific experiment described in Fig 2 and Table 1 was done with unfiltered light, but the same results were obtained using filtered light. The reaction was approximately 5 times more rapid using the more intense unfiltered irradiation, but the nature of the spectral and pH changes were independent of excitation wavelength.



Absorbance
(relative units)

Figure 2
Spectral changes induced by
photolysis of trans- $\text{Rh}(\text{en})_2(\text{H}_2\text{O})(\text{CO}_3)^+$

A initial spectrum

B after 2000 seconds of photolysis

250

300
Wavelength (nm)

350

TABLE 1

total photolysis time (seconds)	sample pH	blank pH
Initial	7.032	7.060
100	7.110	7.070
200	7.184	7.081
300	7.245	7.096
450	7.325	7.129
600	7.376	7.136
800	7.435	7.151
1000	7.470	7.184
1300	7.504	7.216
2000	7.540	7.220

Typical pH changes induced by Photolysis of Trans- Rh(en)₂ (H₂O)(CO₃)⁺

TABLE 2

total photolysis time (seconds)	sample pH	blank pH
Initial	9.905	9.916
100	9.411	9.862
200	8.557	9.789
300	7.729	9.721
400	7.375	9.616
550	7.120	9.508
705	7.002	9.380
900	6.911	9.277
1300	6.691	9.185
2200	6.495	9.086

Typical pH changes induced by Photolysis of Rh(en)₂ (CO₃)⁺

2. Discussion

The available results force the conclusion that photolysis of trans- $[\text{Rh}(\text{en})_2 (\text{CO}_3) (\text{H}_2\text{O})]^+$ causes isomerization (eq⁹).

The cis- $[\text{Rh}(\text{en})_2 (\text{H}_2\text{O}) (\text{CO}_3)]^+$ ion has an absorption maximum at 332 nm, at a significantly higher absorption coefficient than the trans isomer, consistent with the blue shift and increased absorption observed upon photolysis. The lack of change in the charge-transfer edge is also significant, as the intense charge-transfer peak is a LMCT band with carbonate ligand as the donor; the LMCT bands for the analogous $[\text{Rh}(\text{en})_2 (\text{H}_2\text{O})_2]^+$ ions begin at significantly higher energies. As photolysis causes no shift in that edge, it is clear that significant amounts of carbonate ion are not labilized during the photolysis.

The pH changes are not consistent with this proposed reaction, however, as the reaction of eq 9 should not lead to any pH changes. Calculation of the amount of OH^- released after extensive photolysis shows that only a very minor amount of the Rh complex is releasing base to the solution. For example, during the experiment described in Fig 2 and Table 1, 1000 seconds of photolysis causes spectral changes indicating that about 80% of the starting material has undergone photoinduced isomerization. After those same 1000 sec., only about .0026% of the starting material has released an OH^- into solution. We conclude that any pH changes are due to a minor side reaction which is not related to the dominant photoisomerization. The pH increase is apparent as soon as photolysis begins, suggesting that whatever reaction causes the pH increase, it is a primary photochemical process. The rate of pH increase remains relatively constant throughout the photolysis, even when little starting material remains to be photolyzed; this implies that there is also an inefficient base-releasing secondary photochemical reaction. Whether these pH changes are due to inefficient release of carbonate ion or an ethylenedimine chelate is not known at this time.

Further support for the proposed photoisomerization comes from the anation studies. Refluxing the photoproduct in HCl leads to the acid-catalyzed aquation of the carbonate ion, which would be followed by chloride anation of the diaquo ions. Since hydrolysis of the carbonate ion occurs without metal-oxygen bond cleavage, and anations at Rh(III) centers are also known to be stereoretentive, the composition of the resulting mixture of cis and trans- $[\text{Rh}(\text{en})_2 \text{Cl}_2]^+$ would reflect the stereochemical composition of the cis and trans- $[\text{Rh}(\text{en})_2 (\text{H}_2\text{O}) (\text{CO}_3)]^+$ photoproduct mixture.

The best value for this composition - 63% cis and 37% trans-must be considered a preliminary result; further work will be done to determine whether photolysis of trans- $[\text{Rh}(\text{en})_2 (\text{H}_2\text{O}) (\text{CO}_3)]^+$ generates a photostationary state of that composition. Such a result, which available information supports, would imply that the dominate photochemical reaction of cis- $[\text{Rh}(\text{en})_2 (\text{H}_2\text{O}) (\text{CO}_3)]^+$ is isomerization back to the trans isomer.

The reactions described above occurred upon irradiation with filtered (ligand field) and unfiltered (ligand field and charge-transfer) light. This suggests a single photoreactive state; since no evidence of photoinduced redox behavior was observed, a ligand field photoreactive state is indicated.

(Presumably, photoinduced water exchange accompanies isomerization, although the necessary isotope exchange studies have not been done to verify this assumption). There is no evidence of CO₂ release, which would suggest a photoredox process, even upon excitation of the LMCT bands. The rate of photoreaction was markedly increased when the LMCT bands were irradiated, implying that intersystem crossing, from the initially populated charge-transfer states to the photoreactive ligand field state, is efficient.

In summary, the photochemical behavior of the monodentate carbonate complexes is reminiscent of the behavior of analogous haloamine complexes of Rh(III). The LMCT states are photoinert, while the ligand field states are photoreactive, leading to ligand labilization and aquation, with concomitant stereochemical rearrangement about the Rh(III) center.

B. [Rh(en)₂ (CO₃)]⁺

1. Results

In the solid state, [Rh(en)₂ (CO₃)] ClO₄ is photoinert. Prolonged photolysis (2 hrs) with both filtered and unfiltered light did not cause any detectable release of gas, and when a sample so photolyzed was dissolved in water, it had an absorption spectrum indistinguishable from that of an unphotolyzed sample.

In aqueous solution, and in the absence of light, [Rh(en)₂ (CO₃)]⁺ is very inert. At room temperature and at a solution pH between 6 and 10, there was no detectable change in the absorption spectrum for over 48 hours. No pH changes, beyond those observed in the blank due to absorption of atmospheric CO₂, were observed in aqueous solutions of [Rh(en)₂ (CO₃)]⁺.

Upon photolysis, the electronic spectrum undergoes pronounced changes (Fig. 3). Note that photolysis causes:

- a) a decrease in intensity of the ligand field band, with virtually no change in wavelength maximum
- b) a minor shift in the low-energy edge of the charge-transfer band to higher energy.

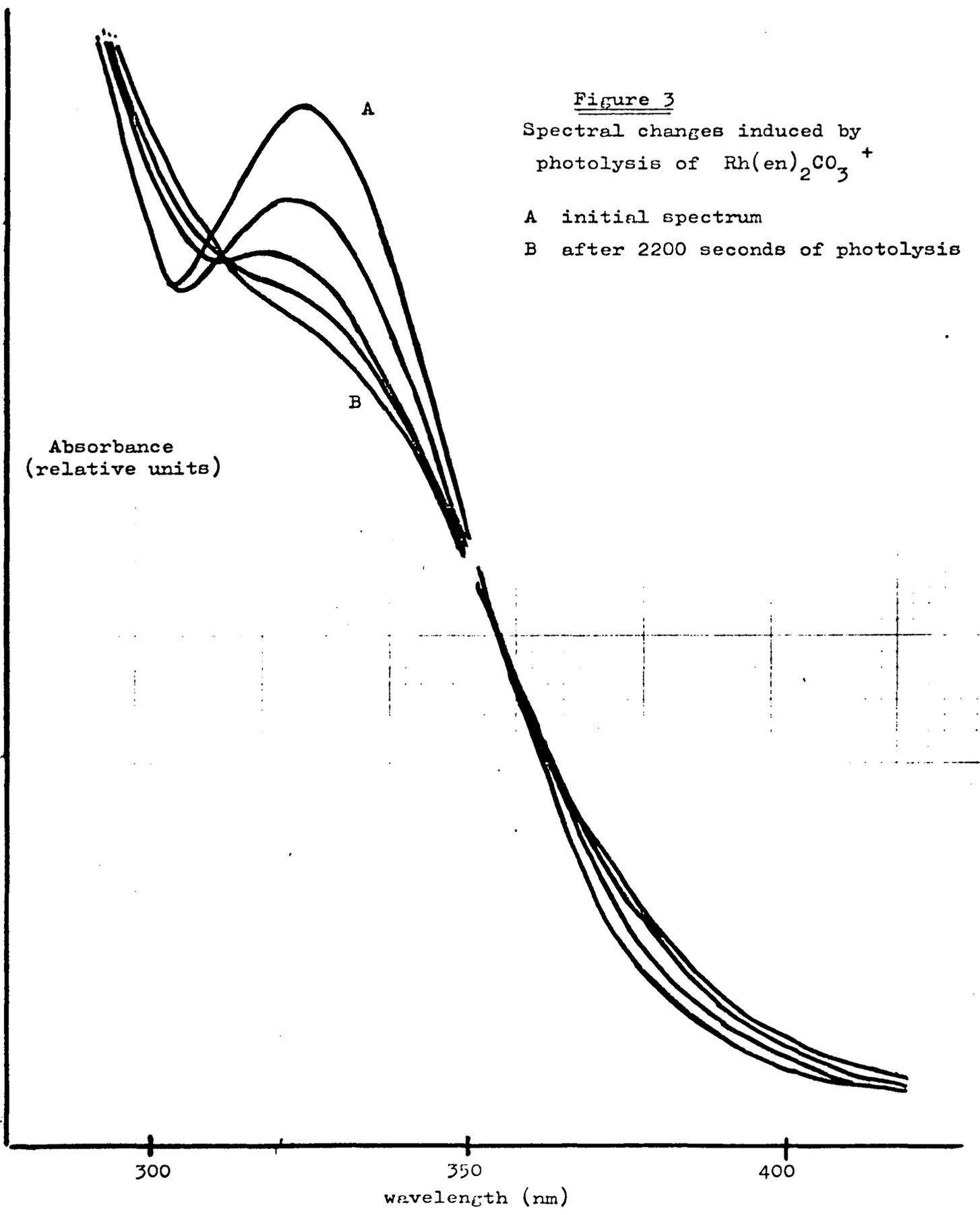
Isosbestic points are not observed in the spectrum during the initial stages of the photolysis, but an isosbestic point does develop at about 314 nm after the initial stages of the photolysis are complete.

In addition to the spectral changes observed in Fig 3, photolysis causes a marked decrease in the solution pH. Data from a typical experiment are given in Table 2. The pH changes recorded in Table 2 were observed from the same experiment monitored in Fig 3.

The presence of photoproduct carbonate ion in solution was checked by photolyzing a sample of [Rh(en)₂ (CO₃)]⁺ in a solution containing Ba²⁺

Figure 3
Spectral changes induced by
photolysis of $\text{Rh(en)}_2\text{CO}_3^+$

A initial spectrum
B after 2200 seconds of photolysis



ions; formation of $\text{BaCO}_3(\text{s})$ would signal the presence of CO_3 in solution. (A stream of $\text{N}_2(\text{g})$ was bubbled through a slightly acidic solution of $[\text{Rh}(\text{en})_2\text{CO}_3]^+$ for 30 min. This solution was made basic (pH 10) with $\text{LiOH}(\text{s})$, and bubbling was continued for another 15 min. This solution volume was doubled with the addition of a freshly boiled BaCl_2 solution (0.5 M), with N_2 bubbling continued throughout these manipulations. Any $\text{BaCO}_3(\text{s})$ which formed was allowed to precipitate and a clear solution was put into a closed, fused silica cell for photolysis. The formation of a precipitate was monitored visually, or by recording the absorption spectra at 550 nm, where only scattered light could lead to absorption.) Photolysis with unfiltered light rapidly led to precipitate formation (presumably $\text{BaCO}_3(\text{s})$), which did not occur in a blank sample held in the dark. Photolysis with the filtered light did not lead to immediate precipitate formation, but prolonged photolysis did lead to a measurable cloudiness. We attribute this to the less efficient formation of $\text{BaCO}_3(\text{s})$.

In an effort to determine the stereochemical composition of the photoproduct, a sample of $[\text{Rh}(\text{en})_2\text{CO}_3]^+$ was photolyzed with unfiltered radiation for 3200 sec., and the photoproduct solution was then diluted with an equal volume of conc HCl. The solution immediately became yellow, and displayed a broad absorption band with local maxima at about 355 nm and 325 nm. Refluxing this solution for about 1 hour led to a yellow solution without a peak at 355 nm, but a more intense, clean peak centered at 331 nm; this peak had an absorption coefficient of about $3 \times 10^3 \text{M}^{-1}\text{cm}^{-1}$.

2. Discussion

The photochemical reactivity of $[\text{Rh}(\text{en})_2\text{CO}_3]^+$ has not yet been clearly elucidated; additional work is necessary to fully explain the phenomena observed. The following hypothesis, therefore must still be considered tentative.

Direct labilization of carbonate or ethylenediamine ligands can be ruled out as primary photochemical processes, as such reactions would lead to pH increases, not the observed pH decrease. Photoinduced isomerization, as observed for the monodentate complex (vide supra), may be occurring, but cannot account for the dramatic pH decrease; in addition, the photoinduced spectral changes do not support the idea of an isomerization to some other Rh(III) tetramine complex. The observed increase in acidity, along with the detection of free CO_3^{2-} in solution, strongly implies that CO_2 is a primary photoproduct. The $\text{H}^+(\text{aq})$ and $\text{CO}_3^{2-}(\text{aq})$ result from the rapid hydrolysis of the acidic oxide.

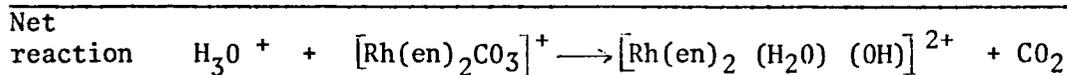
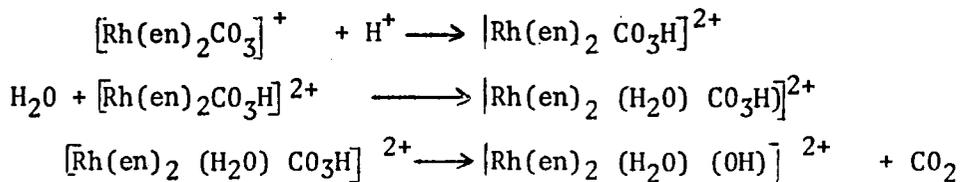


Independent detection of H^+ and CO_3^{2-} in the photolyzed solutions supports the supposition of photogeneration of CO_2 .

Two possible routes for CO_2 release seem likely:

- a) Decarboxylation of coordinated CO_3^{2-} to generate CO_2 and OH^-
- b) oxidation of coordinated CO_3^{2-} to give CO_2 and $\frac{1}{2} \text{O}_2$.

In case a), observed for the thermal reaction of the analogous Co(III) complex, the following reaction sequence would be anticipated:



This process does release the weak acid, CO_2 , but consumes the strong acid H_3O^+ . Thus, the observed pH decrease would not be anticipated if such a process were occurring. In addition, reaction of the photoproduct of such a sequence with HCl would lead to cis or trans- $[\text{Rh}(\text{en})_2\text{Cl}_2]^+$; the observed product, with a single, intense peak at 331 nm is neither of the $[\text{Rh}(\text{en})_2\text{Cl}_2]^+$ isomers.

Possibility b), the two electron oxidation of the carbonate ligand, is more consistent with the available information (and is the more fascinating photochemical reaction). Assuming concomitant two-electron reduction of the Rh(III) center, a reaction of the form (eq 10)



would be occurring.

Reaction 10 is consistent with a variety of observations. Plots of "moles of H^+ released" as a function of photolysis time showed two distinct patterns, depending upon the initial pH (Fig. 4). At pH 6, the rapid initial generation of H^+ becomes less pronounced as the photolysis proceeds; at pH 10, the initial photolysis period generates only minor amounts of H^+ , followed by a more rapid rate of H^+ generation. This is consistent with the release of a weak acid, as at pH 10 the $[\text{HCO}_3^-] / [\text{CO}_2]$ ratio would be four orders of magnitude larger than at pH 6.



$$K = \frac{[\text{H}^+][\text{HCO}_3^-]}{[\text{CO}_2]} \quad (12)$$

$$\frac{K}{[\text{H}^+]} = \frac{[\text{HCO}_3^-]}{[\text{CO}_2]} \quad (13)$$

Such a large concentration of bicarbonate ion would act as a buffer, resisting the generation of H^+ ions upon the addition of CO_2 to the solution. As shown in Fig. 4, H^+ increase was initially quite slow, until the buffering action of the bicarbonate ion was overcome. At pH 6, the bicarbonate ion concentration would be less formidable, and little buffering action would be apparent. If we consider that for reaction 11,

$$[\text{H}^+] = [\text{HCO}_3^-] \quad \text{and} \quad [\text{CO}_2] = [\text{CO}_2]_{\text{initial}} - [\text{H}^+]$$

we can write

$$K = \frac{[\text{H}^+]^2}{[\text{CO}_2]} = \frac{[\text{H}^+]^2}{[\text{CO}_2]_{\text{initial}} - [\text{H}^+]}$$

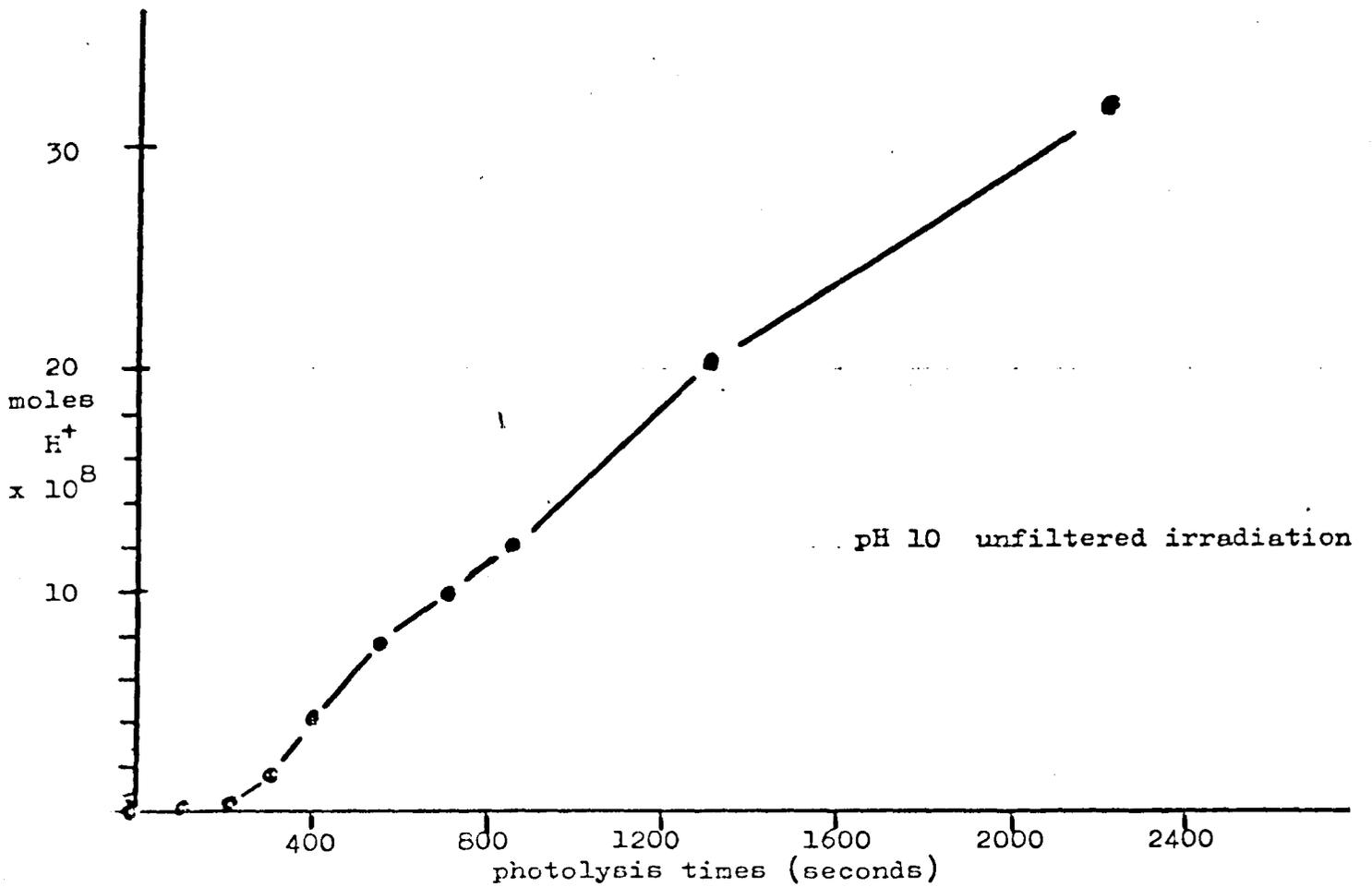
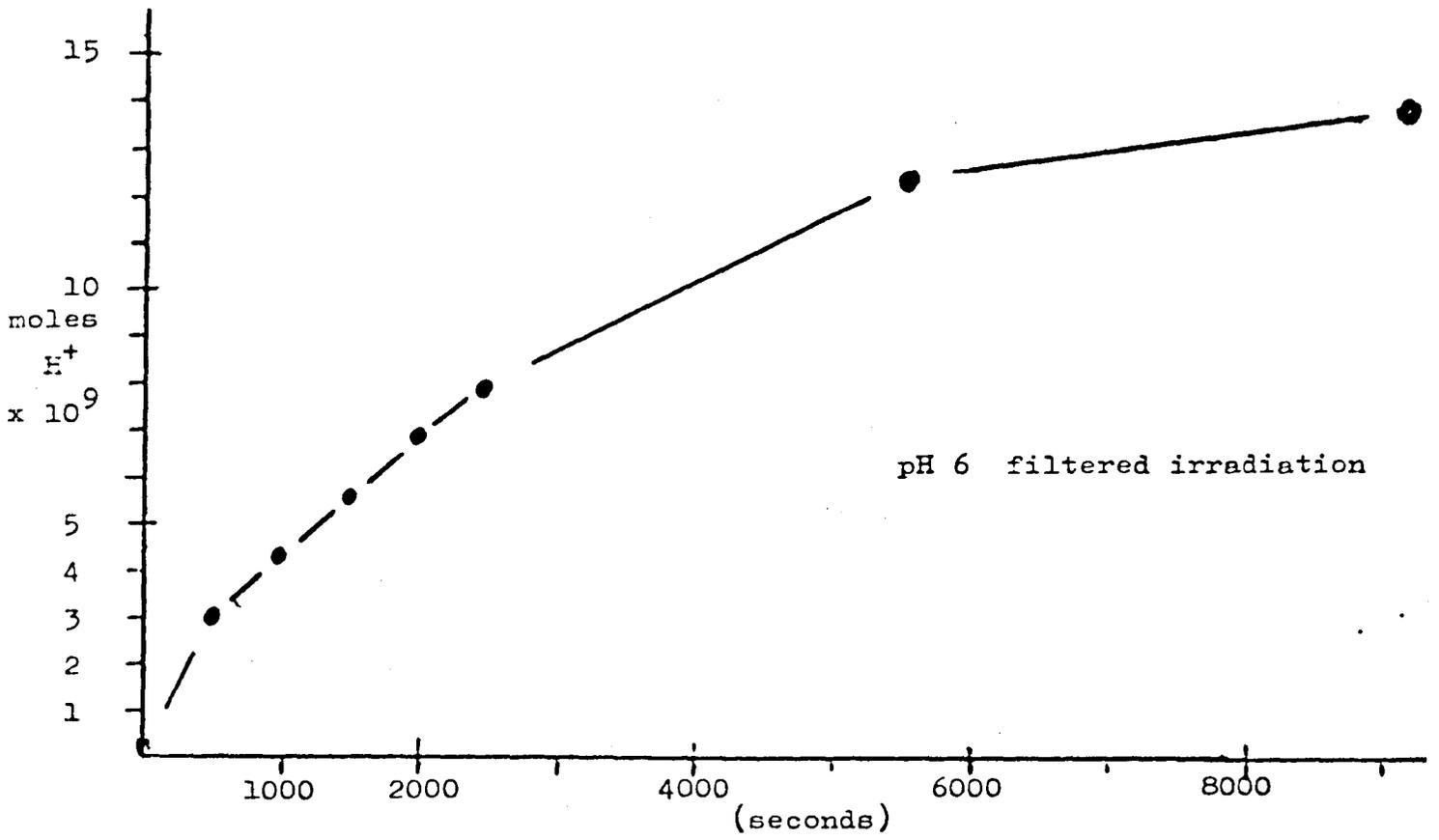


Figure 4

Solving for H^+ , one obtains:

$$[H^+] = \left\{ K ([CO_2]_{\text{initial}} + \frac{1}{4}K^2) \right\}^{1/2} - \frac{1}{2}K \quad (14)$$

Evaluating for H^+ at increasing levels of $[CO_2]_{\text{initial}}$ (representing the CO_2 generated by photolysis) gives a curve which closely models the pH 6 curve of Fig. 4.

After 2200 seconds of unfiltered irradiation of $[Rh(en)_2CO_3]^+$, the spectrum shows that a significant amount of the starting material has photochemically reacted. The data from Table 2 can be used to show that the H^+ released at that time is equivalent to about 10% of the total Rh(III) concentration. This, again, is consistent with the photogeneration of a weak acid (CO_2). A more quantitative assessment is not yet possible, as the exact nature of the Rh-containing photoproduct is not known.

The nature of the Rh-containing photoproduct is a major mystery remaining after this summer's work. The observed spectral changes do not suggest the formation of any obvious Rh(III) tetraamine species, and refluxing the photoproduct in HCl leads to an intensely yellow species which has, as yet, escaped identification. The spectral changes are consistent with the formation of the same product observed upon BH_4^- reduction of cis- $[Rh(en)_2Cl_2]^+$, suggestive of a two-electron reduction of the Rh(III) center, but further work would be necessary to corroborate this supposition.

In summary, work this summer has led us to the tantalizing hypothesis that ligand field and charge-transfer photolysis of a bidentate carbonate complex of Rh(III) leads to the two-electron oxidation of the carbonate ligand, with complementary reduction of the Rh(III) center. This result, if borne out by further studies, would represent a new type of photochemical reaction, and would encourage further work on the water-photolysis scheme (eq 4-8).

CONCLUSIONS AND RECOMMENDATIONS

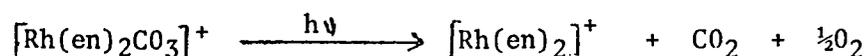
1. Conclusions

a) The monodentate carbonato complex, trans- $[\text{Rh}(\text{en})_2 (\text{H}_2\text{O}) (\text{CO}_3)]^+$ efficiently undergoes photoisomerization:

$$\text{trans-} [\text{Rh}(\text{en})_2 (\text{H}_2\text{O}) (\text{CO}_3)]^+ \xrightarrow{h\nu} \text{cis-} [\text{Rh}(\text{en})_2 (\text{H}_2\text{O}) (\text{CO}_3)]^+$$

There is no evidence of electron-transfer upon either ligand field or charge-transfer excitation. A very inefficient base-releasing photochemical reaction also occurs, but its nature has not been characterized. Preliminary evidence implies that the cis-aquocarbonato photoproduct undergoes photoinduced isomerization back to trans- $[\text{Rh}(\text{en})_2 (\text{H}_2\text{O}) (\text{CO}_3)]^+$.

b) Ligand field and charge-transfer photolysis of the bidentate carbonato complex, $\text{Rh}(\text{en})_2 \text{CO}_3^+$ causes the release of CO_2 into solution by a reaction tentatively described as:



If subsequent studies corroborate the occurrence of a two-electron transfer, it will represent a new class of photochemical reaction. Such a reaction is the crucial first step in a water-photolysis scheme (eq 4-8), so confirmation of an efficient two-electron transfer, induced by a single photon, will open a new area of transition metal photochemistry.

2. Recommendations

This summer's study has shown that a promising area of transition metal photochemistry can be found in the study of bidentate carbonate complexes. The nature of the Rh-containing photoproduct in the photolysis of $[\text{Rh}(\text{en})_2 \text{CO}_3]^+$ must receive immediate attention, and the design of future experiments depends on the results of that study. Available evidence strongly suggests that a two-electron redox process is involved, so the next steps must include:

a) a study to determine the factors which maximize the efficiency (quantum yield) of this redox reaction.

b) Studies must be initiated to increase the amount of the solar spectrum absorbed by the photoreactive complex. A logical approach would be to look at some substituted bipyridyl ligands or other aromatic ligands with extensive absorption in the visible and near-IR portions of the spectrum. The use of intensely colored sensitizers should also be studied.

c) Parallel studies should explore the detailed chemistry of the water-soluble complexes of Rh(I). The nature of the four-coordinate d^8 complexes, and the proposed d^6 hydrides must be determined. The thermal and photochemical properties of a variety of such hydride complexes must be studied to determine the conditions which would lead to a photoreactive hydride complex.

In summary, the discovery of a two-electron redox reaction, induced by a single photon, suggests that it may be possible to photochemically generate H_2 from water without the complications inherent in the hydrogen atom radical. This should encourage further study of a carbonate-based system for the photochemical cleavage of water.

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MARSHALL SPACE FLIGHT CENTER
THE UNIVERSITY OF ALABAMA

CALCULATIONS AND MEASUREMENTS IN SUPPORT OF THE
TRIGLYCINE SULFATE ZERO GRAVITY GROWTH EXPERIMENT

Prepared By: John M. Springer, Ph.D..

Academic Rank: Research Associate

University and Department: Fisk University
Department of Physics

NASA/MSFC:
(Laboratory) Space Sciences
(Division) Materials Processing
(Branch) Solid State

MSFC Counterpart: Roger Kroes, Ph.D.

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By

John M. Springer, Ph.D.
Research Associate in Physics
Fisk University
Nashville, Tennessee

ABSTRACT

The solution growth of crystals of triglycine sulfate (TGS) in a zero gravity environment is a materials processing experiment planned for an early Spacelab mission. This experiment has the twin objectives of both studying the growth process by optical means and also returning the grown crystals to earth for characterization.

This paper describes two areas of investigation which are required for maximum utilization of the flight experiment results. The first area studied is the use of an ultramicroscope to characterize structural defects in the grown TGS crystals. Mie theory has been applied to the light scattering expected from idealized crystal defects in order to predict the sensitivity of ultramicroscopy in the detection of small defects.

The second area of investigation has involved the measurement of optical properties of TGS solutions in water. This data is needed in order to interpret interferometric studies of mass transport through the TGS solution during the growth phase of the flight experiment. In particular, refractive index data has been obtained for a wide range of temperatures and concentrations of TGS solutions. Absorbance measurements in the ultraviolet have also been made for the same samples.

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INTRODUCTION

One of the materials processing experiments that will be flown aboard an early Spacelab mission is the solution growth of triglycine sulfate (TGS). This experiment has two objectives, one being the observation of the crystal growth process itself by optical techniques such as Schlieren photography and interferometry, the other being the return of the zero-gravity grown crystals to earth for testing and comparison with their earth-grown counterparts.

This report describes two studies performed which are related to the TGS growth experiment. The first part discusses the use of a light scattering technique called ultramicroscopy to characterize triglycine sulfate crystals. The theory of Mie for the scattering of light by spheres is used to indicate the sensitivity of the ultramicroscope in the detection of volume defects. Although based on an idealization of the actual defects that would be present in a real crystal, this analysis is useful in analyzing the effectiveness of this technique.

The second part of the report presents data on the optical properties of solutions of TGS in water. This information on the refractive index and optical absorbance of TGS in solution is required for the analysis of the Schlieren and interferometric data which will be obtained during the crystal growth experiment on Spacelab 3.

I. Crystal Characterization by Ultramicroscopy

A. General

Crystal characterization involves describing a crystal's structure and composition in such a way that those of its properties of relevance to the study at hand are well enough known to predict in principle the behavior of the crystal. One of the important features of a crystal are its departures from the perfect array of atoms by which its structure is commonly described. These departures from ideality, called defects, exist in every real crystal and range from the simple absence of a single atom from the lattice to dislocations involving the entire crystal.

The scattering of electromagnetic radiation incident on a crystal is one tool for detecting defects. The scattering and diffraction of X-rays yields information on structural features whose size is on the order of the lattice spacing. In the case of optically transparent crystals, the scattering of visible light can also reveal structural defects in a crystal, though not with the detail of X-ray topography. For a crystal whose optical properties are germane to its application, however, the measurement of visible light scattering is essential for its proper characterization.

One method being implemented to study TGS crystals by light scattering is that of the ultramicroscope. In this technique, illustrated in Figure 1, a beam of light focussed to a small diameter is used to illuminate the crystal. As the light passes through the crystal, it is scattered by inhomogeneities in the crystal's refractive index, such as are caused by certain types of defects. In order to observe the light scattered from the defects, a microscope is used to collect the light scattered at right angles to the beam. Small, isolated defects show up as points of light in the dark surrounding field of view. Regions of a crystal where the defect density is quite high will show nebulous or diffuse scattering regions. As described by Vand,¹ a small He-Ne laser makes an excellent source for the ultramicroscope. This method has already been used to study the relation between defects in TGS and its dielectric properties.²

A diagram of an ultramicroscope being constructed for the study of TGS is shown in Figure 2. This ultramicroscope will use photoelectric rather than photographic detection of scattering centers, and will be computer driven so a three dimensional map of the light scattering defect may be constructed.* In the remainder of this section of the report, an analysis is described in which the expected sensitivity of this ultramicroscope to defects is calculated. This sensitivity will be seen to depend on defect size and relative effective refractive index.

* This is being constructed at Fisk University with the support of NASA Grant #NSG-8060.

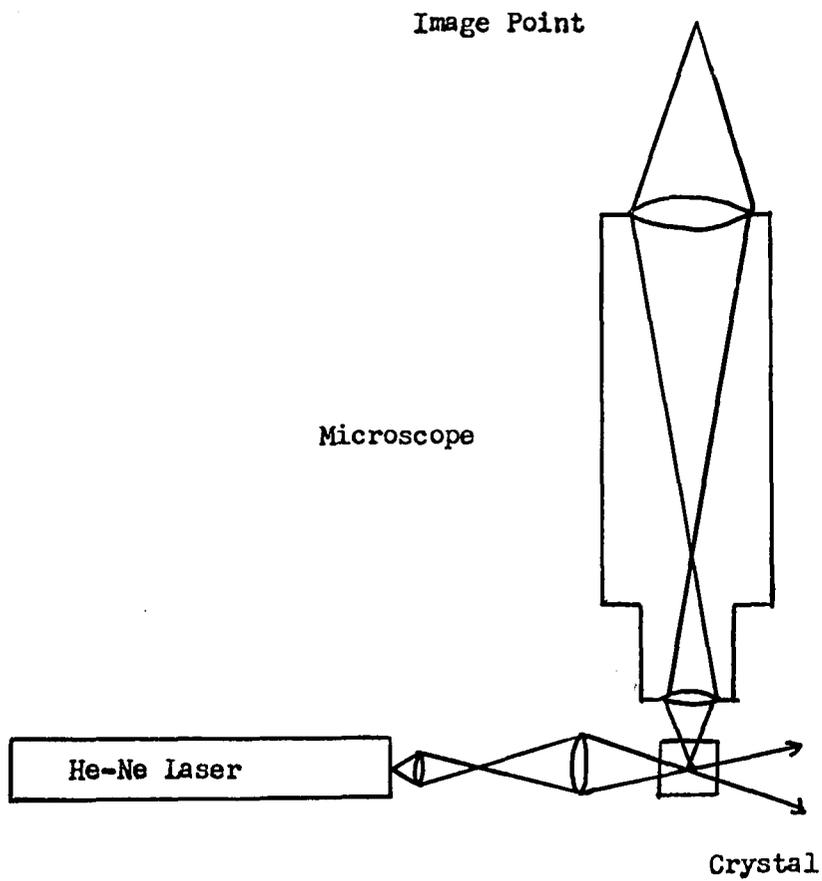


FIGURE 1: Schematic diagram of ultramicroscope used to observe defect scattering.

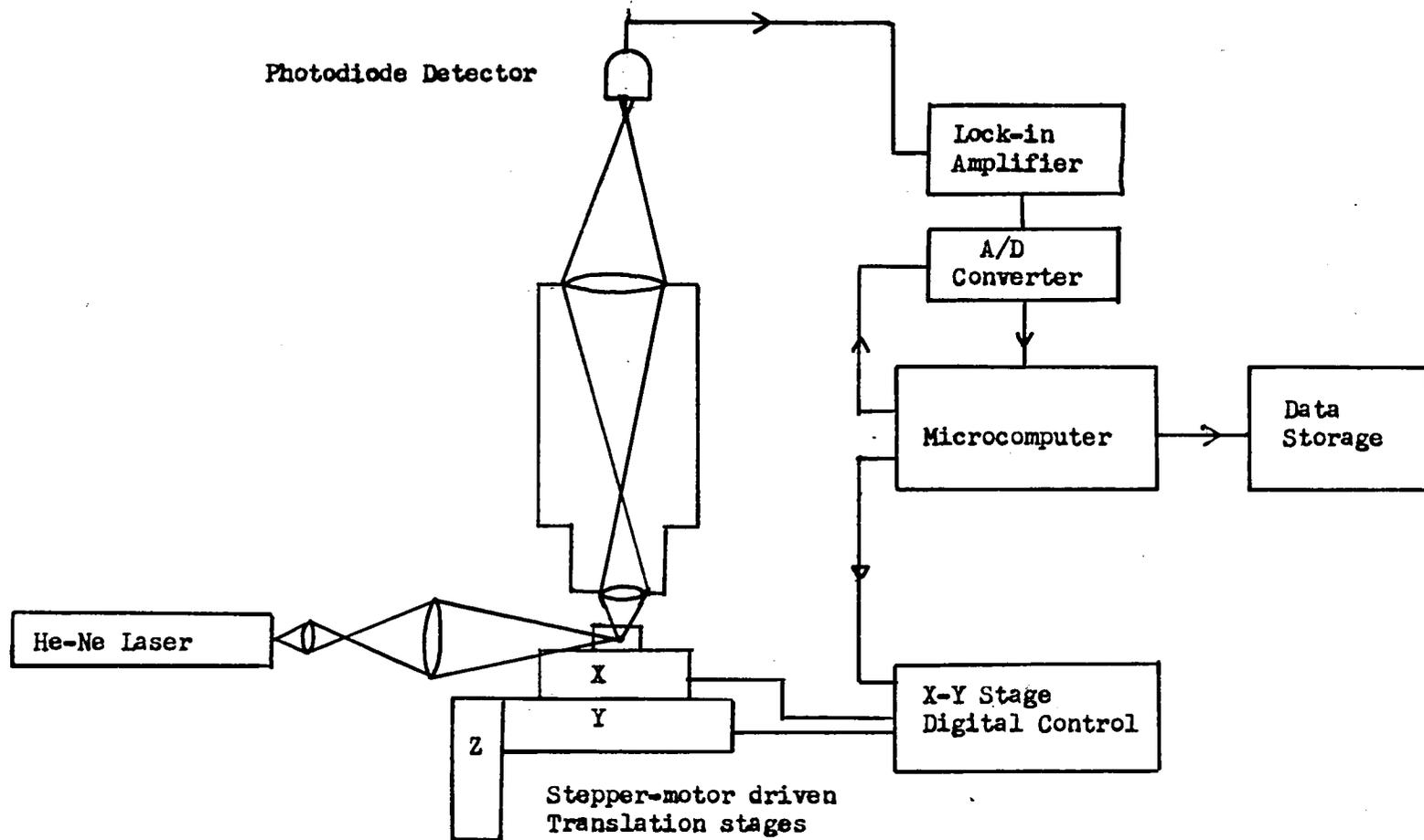


FIGURE 2: Automated ultramicroscope

B. Discussion

The model "defect" which has been used in this study is that of a homogeneous sphere of diameter D and refractive index n . This spherical defect is assumed to be surrounded by the undisturbed TGS lattice. The absorption coefficient of the defect is assumed to be zero, so the refractive index is a real number. It is illuminated in the ultramicroscope by a plane electromagnetic wave of wavelength 632.8 nm in air.

The theory of scattering of electromagnetic waves by spheres is a classical problem, and was solved analytically by Mie.³ His solutions involve infinite sums of complex functions, but several sets of tables are available in the literature which make the evaluation of the solutions for particular cases straightforward, if somewhat tedious.^{4,5,6} Computer programs for this purpose have also been written, and would be useful if a large number of cases must be calculated.

Figure 3 defines the geometry of the problem. According to the treatment by Boll,⁷ the scattered light intensity is represented by two intensity functions i_1 and i_2 , given by

$$i_1 = \frac{\lambda^2}{8\pi^2} \left| \sum_{n=1}^{\infty} \left\{ A_n \frac{dP_n(x)}{dx} + B_n \left[x \frac{dP_n(x)}{dx} - (1-x^2) \frac{d^2 P_n(x)}{dx^2} \right] \right\} \right|^2 \quad (1)$$

and

$$i_2 = \frac{\lambda^2}{8\pi^2} \left| \sum_{n=1}^{\infty} \left\{ A_n \left[x \frac{dP_n(x)}{dx} - (1-x^2) \frac{d^2 P_n(x)}{dx^2} \right] + B_n \frac{dP_n(x)}{dx} \right\} \right|^2 \quad (2)$$

where $P(x)$ is a Legendre polynomial of order n , with $x = -\cos(\theta)$. θ is the scattering angle from the incident direction. The coefficients A_n and B_n are complex Ricatti-Bessel and Hankel functions, and for the values to be considered here can be obtained from tables.⁴ These coefficients depend on only two physically relevant parameters for their evaluation, a size parameter.

$$\mathcal{L} = \frac{\pi D}{\lambda} \quad (3)$$

where D = the scattering particle diameter

λ = the wavelength of the incident light in the surrounding medium

and m , the refractive index of the scattering particle relative to the surrounding medium.

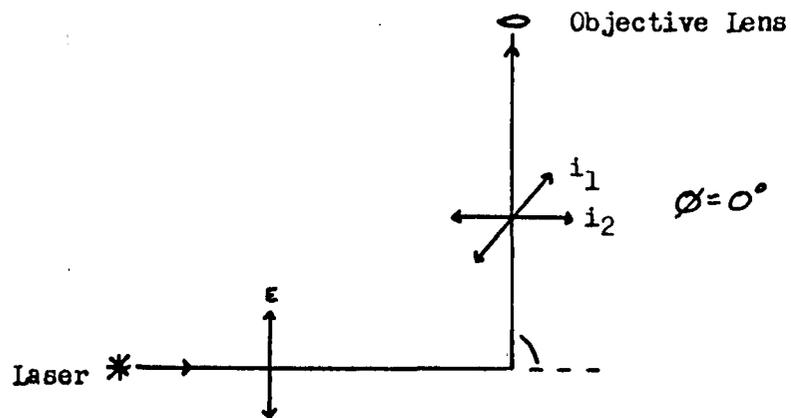
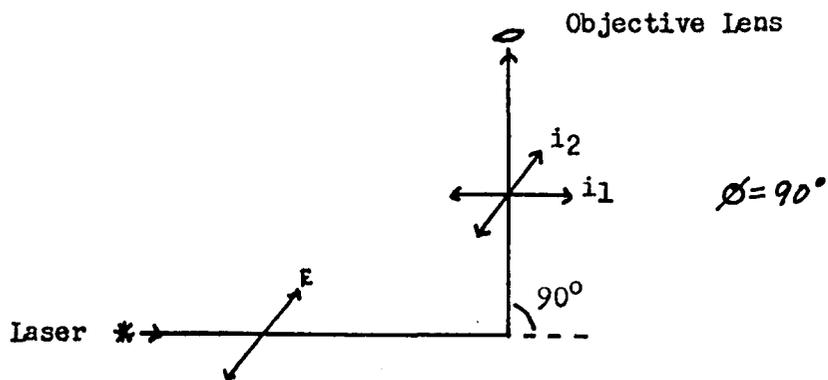


FIGURE 3: Orientation of the intensity functions for right angle scattering.

For plane-polarized incident light, the intensity function i_1 is the component of scattered light polarized perpendicular to the electric vector of the incident beam, and i_2 represents the component of scattered light polarized parallel to the plane formed by the electric vector of the incident beam and its direction of propagation. The resulting scattered intensity per unit incident intensity and unit solid angle is for any direction θ, ϕ given by

$$I = \frac{\lambda^2}{4\pi^2} \left[(\cos^2 \phi) i_1 + (\sin^2 \phi) i_2 \right] \quad (4)$$

In the ultramicroscope, the scattered light collected by the objective lens is nominally at a scattering angle $\theta = 90^\circ$. If the incident light on the crystal is polarized with its electric vector perpendicular to the plane formed by the incident and scattered light directions of propagation, then $\phi = 90^\circ$ also, so the total scattered intensity per unit solid angle is

$$I_{\alpha} = I_0 \frac{\lambda^2 i_2}{8\pi^2} \quad (5)$$

where I_0 is the incident intensity (W/m^2).

A typical objective used on the microscope will have a numerical aperture of 0.25. Taking into account the refraction that occurs at the surface of the TGS crystal as the scattered light passes into the air, an objective lens of N. A. 0.25 will collect a cone of light with a half-angle of 9° in the crystal, corresponding to a solid angle of 0.00612. So for this case, the scattered power entering the objective lens will be

$$P_s = I_0 \frac{\lambda^2}{8\pi^2} (6.12 \times 10^{-3}) i_2 \quad (6)$$

In the ultramicroscope the detector is a photodiode with an active area which is larger than the image of the smaller defects with which this analysis is concerned, so the scattered light power collected by the objective lens will all fall on the detector when the defect is centered in its field of view. P_s , therefore, is the relevant value which must be calculated and compared with the sensitivity of the detector and its amplifier.

C. Calculations

The relevant physical parameters of a defect are its diameter D and refractive index n . TGS in solution at its saturation point at 25° has a refractive index of approximately 1.38. The crystalline solid has three principal indices which range from 1.484 to 1.584.⁸ Values of the relative refractive index m could be expected to vary typically from 0.6 for a void with absolute index 1 to 0.93 for a volume defect consisting of uncrystallized solution. The actual values of m could of course be close to 1 for small distortions of the lattice. Values of the size parameter which are available in Boll's⁷ tables range upward from 1 to 120. Since only small defects are of interest in finding the limit of detectable scattering, values of from 1-10 were used in the calculations. In triglycine sulfate, this corresponds to a diameter of from approximately 1.2×10^{-7} meters to 1.2×10^{-6} meters when the wavelength of the incident light is 632.8nm.

The laser which will be used is a 5mW He-Ne laser, which can be focussed to a 20×10^{-6} meter diameter in the crystal. This gives an average beam intensity of $I_0 = 1.6 \times 10^7$ W/m². Using a refractive index of 1.58 for the crystalline TGS, the wavelength of the laser light in the crystal is $\lambda = 400$ nm. These values give a scattered power from equation 6 of

$$P_s = I_0 \frac{\lambda^2 (6.12 \times 10^{-3})^2}{8\pi^2} i_1 = (1.98 \times 10^{-10} \text{ Watts}) i_2 \quad (7)$$

Representative values of the scattered power collected by the ultramicroscope for several values of i_1 and m are shown in Table 1. The effective scattering defect size D is also given.

An evaluation of these values in terms of photoelectric detection can be made by comparison with the noise figure for a photodiode which is being evaluated in the ultramicroscope. The photodiode, a model PN-4QA from United Detector Technology, has a noise equivalent power (NEP) of 10^{-14} Watts. For spherical defects with a relative refractive index near unity (0.93), defects of this type would have to exceed 300nm to be above the noise level of the detector. For voids, where the refractive index would be around 0.6, a defect somewhat small (100nm) would scatter sufficiently to be observed.

TABLE 1

Power scattered at 90° from spherical defects for plane polarized incident beam of wavelength 400 nm and intensity $1.6 \times 10^7 \text{ W/m}^2$. Power is in Watts.

α	Defect Diameter (nm)	Relative Refractive Index m					
		0.6	0.7	0.75	0.8	0.9	0.93
1	127	6.2×10^{-14}	2.9×10^{-15}	1.5×10^{-15}	6.6×10^{-16}	4.7×10^{-17}	1.2×10^{-17}
2	250	2.7×10^{-12}	1.2×10^{-12}	7.5×10^{-13}	3.2×10^{-13}	2.8×10^{-14}	7.5×10^{-15}
3	381					4.1×10^{-14}	1.1×10^{-13}
4	508					4.9×10^{-14}	1.3×10^{-13}
5	635					8.9×10^{-14}	3.2×10^{-13}
10	127					2.0×10^{-12}	1.6×10^{-12}

II. Optical Measurements

Refractive index and ultraviolet absorbance measurements were made on solutions of triglycine sulfate in distilled water at concentrations ranging from 0.25 grams-TGS/100 grams-H₂O to 32 grams-TGS/100 grams-H₂O. Samples of various concentrations were obtained by making an initial solution of 32g/100g, dividing this in half and diluting one part to make a 16g/100g solution, and then repeating this process to form a series of samples each one-half as concentrated as its predecessor. The resulting solutions were stored in sealed bottles. The samples were weighed during their preparation to a precision of ± 0.01 grams.

Refractive index measurements were made using a Billingham and Stanley Model 60/ED precision Abbe refractometer. The sample and prism temperature was controlled by water circulated from a water bath in which the temperature was regulated to ± 0.01 degree centigrade. The actual refractometer head temperature versus bath temperature was determined by calibration to compensate for heat loss in the connecting tubing. Temperature measurement was made using a Cole-Parmer Model 8502-25 thermistor thermometer, and a Yellow Springs Inc. Model 703 thermistor probe. A sodium lamp was used to illuminate the refractometer, so the values obtained are for the sodium D₁ line, 589.6 nm. Values calculated from the critical angle measurements using the refractometer are given in Table 2. Graphs of n_d vs temperature for various concentrations and n_d vs concentration for different temperatures are shown in Figures 4 and 5.

Transmission curves in the range of 200 - 300 nm were measured for the same samples used in the refractive index study. The curves were recorded on a Model 701 Heath/Schlumberger single-beam spectrophotometer. All measurements were made for a 10 mm path length. This data will be used in experimentally determining mass diffusion coefficients for TGS in water. For this purpose, a graph of the wavelengths at which the present transmission of the solutions reaches 50% is useful. This graph is shown in Figure 6.

TABLE 2: Refractive Index Data for Triglycine Sulfate Solution - (n_d)

Concentration g/100g H ₂ O	Solution Temperature - Degree Celsius							
	24.98	29.66	34.36	39.28	43.95	48.75	53.45	58.23
0.25	1.33290	1.33236	1.33170	1.33100	1.33018	1.32938		
0.50	1.33312	1.33252	1.33191	1.33114	1.33052	1.32952	1.32858	1.32762
1.00	1.33381	1.33326	1.33259	1.33184	1.33108	1.33020	1.32929	
2.00	1.33541	1.33476	1.33411	1.33337	1.33253	1.33176	1.33085	1.32989
4.00	1.33853	1.33791	1.33721	1.33643	1.33560	1.33471	1.33380	1.33292
8.00	1.34408	1.34343	1.34274	1.34190	1.34101	1.34009	1.33918	1.33819
16.00	1.35430	1.35352	1.35270	1.35188	1.35011	1.35011	1.34917	1.34815
32.00	1.37116	1.37038	1.36948	1.36879	1.36779	1.36681	1.36590	1.36494

FIGURE 4: Refractive index vs temperature for solutions of TGS in water. Each curve is marked with the concentration of the solution in grams of TGS per 100 grams water.

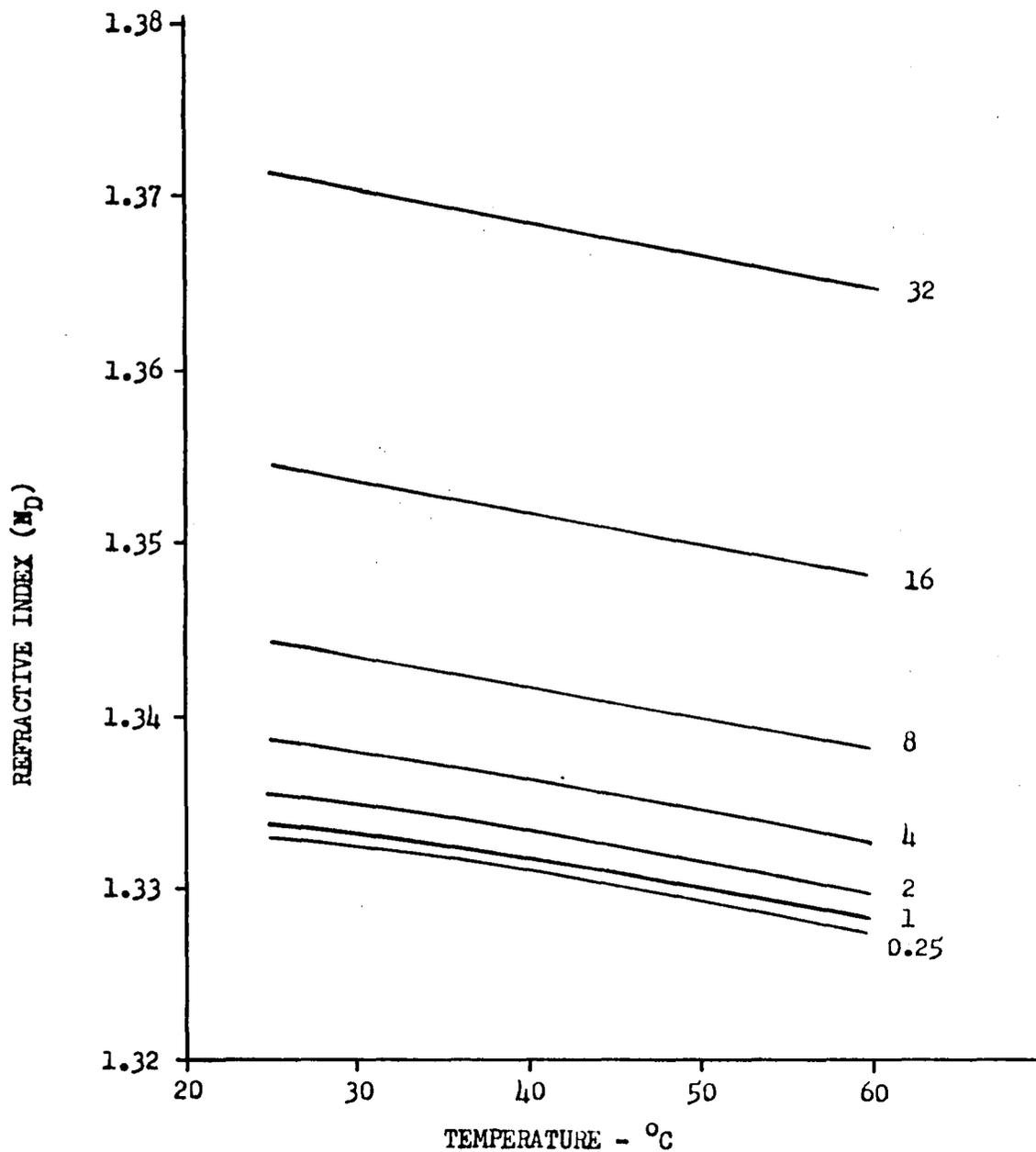


FIGURE 5: Refractive index vs concentration for solutions of TGS in water.

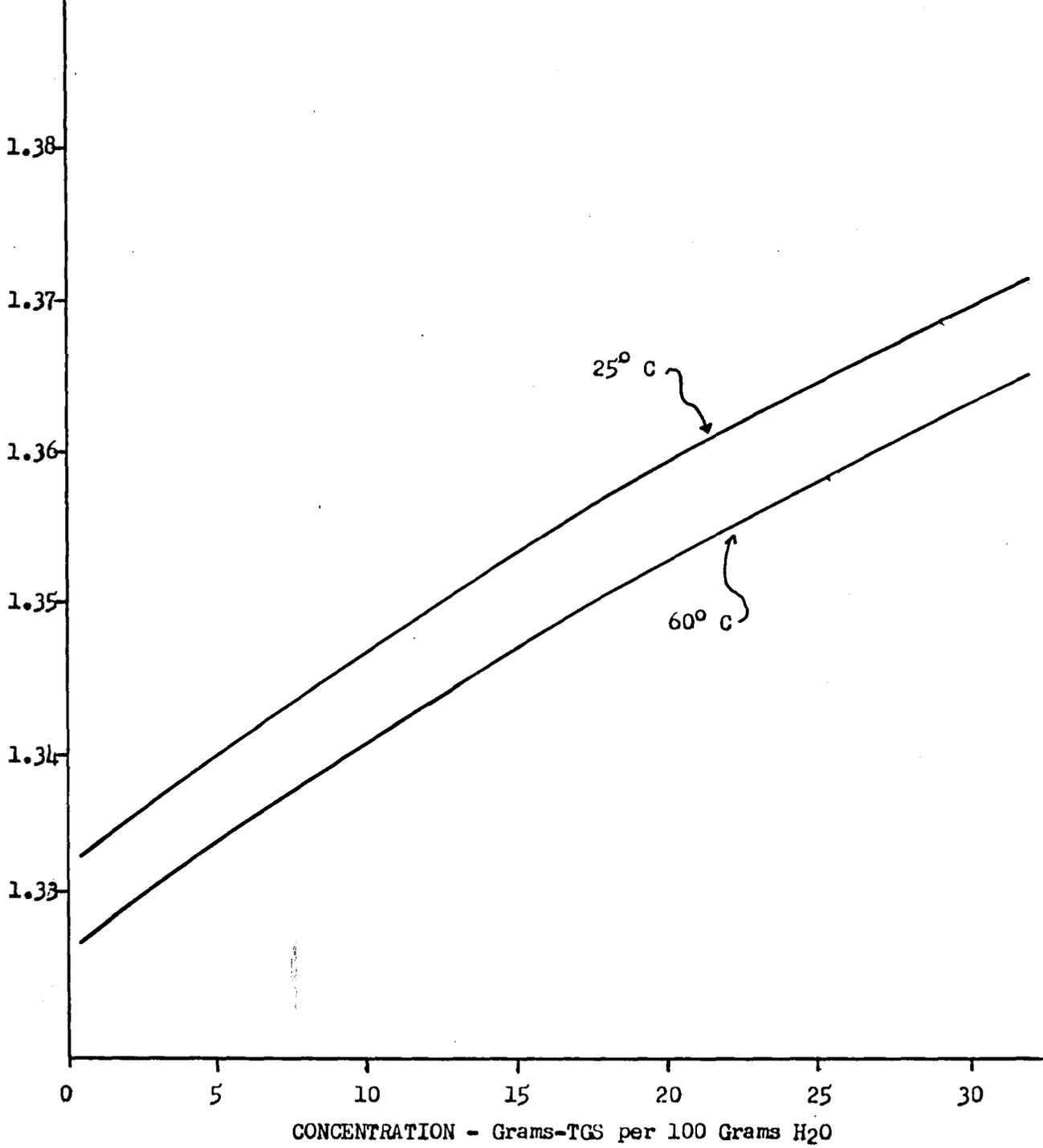
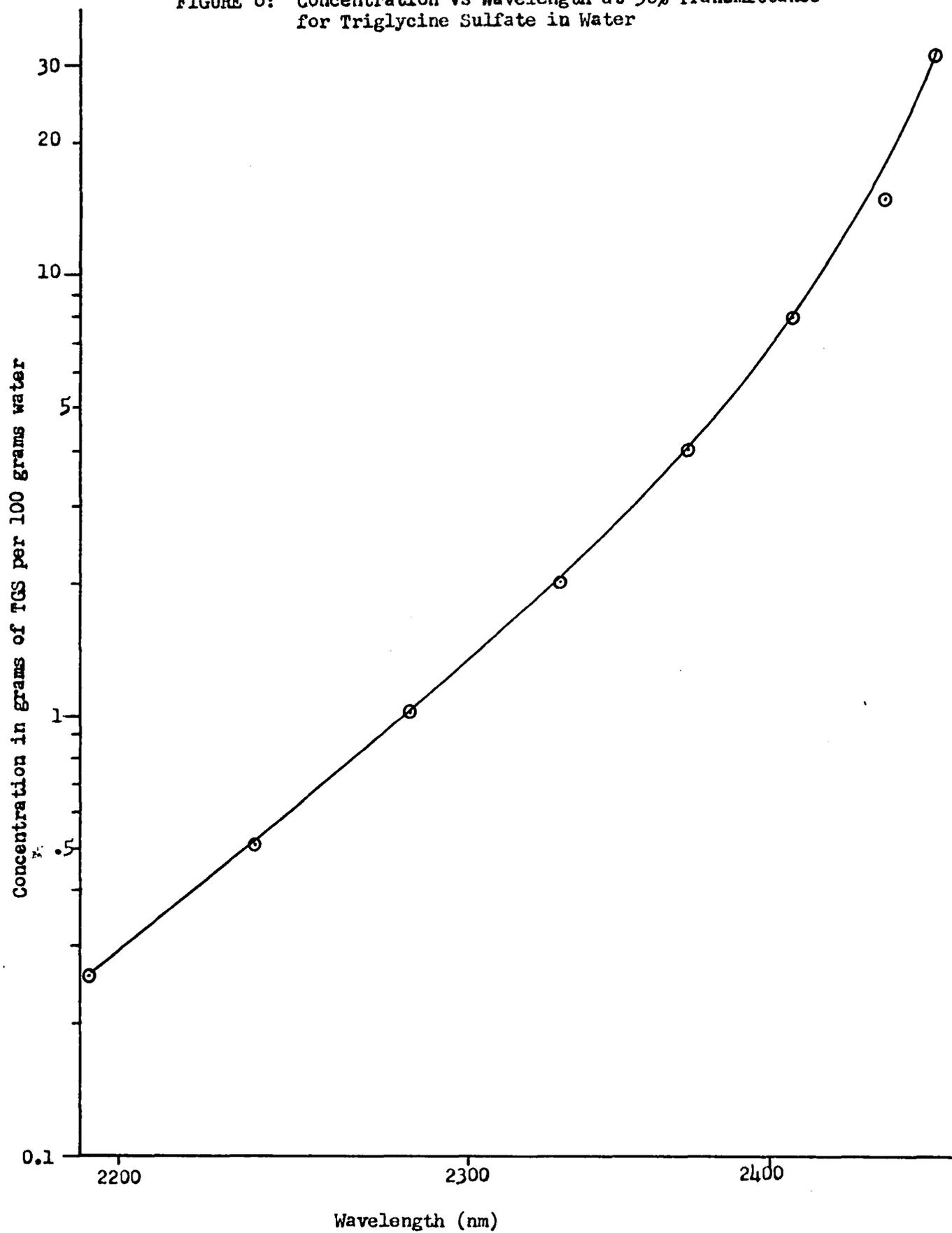


FIGURE 6: Concentration vs Wavelength at 50% Transmittance for Triglycine Sulfate in Water



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MARSHALL SPACE FLIGHT CENTER
THE UNIVERSITY OF ALABAMA

FLUID DYNAMIC ANALYSIS OF THE SPACE SHUTTLE MAIN ENGINE
HIGH PRESSURE OXIDIZER TURBOPUMP SLINGER SEAL

Prepared By:	James S. Voss, Captain
Academic Rank:	Assistant Professor
University and Department	United States Military Academy Department of Mechanics
NASA/MSFC: (Laboratory) (Division) (Branch)	Structures and Propulsion Propulsion Turbomachinery and Combustion Devices
MSFC Counterpart:	Loren Gross
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Contract No.:	NGT-01-002-099 (University of Alabama)

FLUID DYNAMIC ANALYSIS OF THE SPACE SHUTTLE
MAIN ENGINE HIGH PRESSURE OXIDIZER TURBOPUMP
SLINGER SEAL

By

Captain James S. Voss
Assistant Professor of Mechanics
United States Military Academy
West Point, New York

ABSTRACT

An analytical study has been conducted to clarify the details of the flow on the bladed side of a centrifugal type dynamic shaft seal utilized to contain liquid oxygen in the Space Shuttle Main Engine high pressure oxidizer turbopump.

The governing equations are solved to predict the pressure and temperature gradients and to aid in investigating the nature of the liquid-vapor interface.

Recommendations for design improvements are discussed.

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NOMENCLATURE

- a - Axial clearance between seal and housing (in)
- b - Blade height (in)
- C - Constant pressure specific heat (BTU/lbm-⁰R)
- C_m - Moment coefficient (one side), $\tau / \frac{1}{2} \rho \omega^2 r^5$
- C_{ms} - Smooth disk moment coefficient (one side)
- C_{mv} - Vaned disk moment coefficient
- E_c - Eckert Number, $\omega^2 R^2 / C T_0$
- h - Enthalpy (BTU/lbm)
- k - Thermal conductivity (BTU/ft.-hr.-⁰R)
- K - Fluid angular velocity ratio, β / ω
- \dot{m} - Mass flow rate (lbm/sec)
- P - Pressure (lbf/in²)
- \bar{P} - Dimensionless pressure, $p / \rho \omega^2 R^2$
- Pr - Prandtl number, $C \mu / k$
- r - Radial coordinate, radial distance (in)
- \bar{r} - Dimensionless radial coordinate, r / R
- R - Disk radius (in)
- R_{int} - Liquid-vapor interface radius (in)
- Re - Reynolds number, $\omega R^2 / \nu$
- T - Temperature (⁰R)
- \bar{T} - Dimensionless temperature, T / T_0
- T_0 - Seal fluid flow inlet temperature (⁰R)
- u - Radial velocity component (ft/sec)
- \bar{u} - Dimensionless radial velocity component, $u / R \omega$

v	-	Tangential velocity component (ft/sec)
\bar{v}	-	Dimensionless tangential velocity component, $v/R\omega$
w	-	Axial velocity component (ft/sec)
\bar{w}	-	Dimensionless axial velocity component, $w/R\omega$
z	-	Axial coordinate, axial distance (in)
\bar{z}	-	Dimensionless axial coordinate, z/R
β	-	Angular velocity of fluid (rad/sec)
μ	-	Viscosity (lbm/ft-sec)
ν	-	Kinematic viscosity (ft ² /sec)
ρ	-	Density (lbm/ft ³)
τ	-	Torque (ft-lbf)
ω	-	Angular velocity of disk (rad/sec)
--- _{1,2}	-	Subscripts denoting radial location on disk

INTRODUCTION

In high pressure oxidizer turbopumps it is necessary to have effective sealing systems to prevent contact between the turbine hot gas and the oxidizer being pumped. The centrifugal dynamic shaft seal, or slinger seal, can be a valuable part of such a sealing system.

In a slinger seal, a disk shaped impeller with vanes on one side and the other side smooth, rotates with the shaft to be sealed inside a close fitting, but non-contacting housing, as shown in Fig. 1. The liquid to be sealed is centrifuged to the outer periphery of the rotating disk, but because the vaned side has a steeper pressure gradient [1] there will be a resulting net pressure difference across the seal which limits the fluid flow. A gas introduced to the system or vaporized liquid will form an interface with the liquid on the vaned side of the seal.

The primary seal in the Space Shuttle Main Engine high pressure oxidizer turbopump uses this type of slinger with a downstream labyrinth as shown in Fig. 2. The actual seal is modeled by the simplified geometry indicated by Fig. 3.

Previous investigations have primarily been concerned with the more general topic of rotating disks. The theoretical work on free rotating disks done by von Kármán [2], which was corrected by Cochran [3], and the analysis of the enclosed rotating disk by Schultz-Grunow [4] are fundamental to many of the later studies. Daily and Nece, with their experimental work on enclosed rotating disks [5], [6], [7], laid a base for much of the theoretical [8], [9] as well as experimental [10], [11], [12] work that followed.

A limited number of researchers have studied the utilization of enclosed rotating disks as centrifugal seals, with most of their work being experimental or the correlation of experimental data with simple models [13], [14], [15], [16], [17], [18]. A notable exception is the theoretical analysis of smooth slinger seals by Rosenthal and Reshotko [19], [20].

OBJECTIVES

The purpose of this paper is to analyze the dynamics of the fluid on the vaned side of a slinger seal. The governing equations are established and reduced to a form solvable by numerical methods. Time constraints prevent the complete solution by matching of the boundary conditions and numerical analysis so this will be done as a future task.

Some of the important thermodynamic design considerations for this type seal are the radial pressure distribution, temperature distribution, and the liquid-vapor equilibrium interface. A solution is found for the radial pressure distribution and in order to look at the liquid-vapor interface a simplified analysis is used to obtain a restrictive temperature distribution.

DEVELOPMENT OF EQUATIONS

The fluid will be treated as an incompressible, Newtonian fluid where body forces and radiation effects are negligible. Temperature variations will generally be small so the viscosity, thermal conductivity and specific heats can be treated at constants for a particular temperature range [21]. The flow is steady and axisymmetric. Using these assumptions and the usual notation for rotational flow, the governing equations are:

Continuity

$$\frac{\partial u}{\partial r} + \frac{u}{r} + \frac{\partial w}{\partial z} = 0 \quad (1)$$

Radial Momentum

$$u \frac{\partial u}{\partial r} - \frac{v^2}{r} + w \frac{\partial u}{\partial z} = -\frac{1}{\rho} \frac{\partial p}{\partial r} + \nu \left[\frac{\partial^2 u}{\partial r^2} + \frac{\partial}{\partial r} \left(\frac{u}{r} \right) + \frac{\partial^2 u}{\partial z^2} \right] \quad (2)$$

Tangential Momentum

$$u \frac{\partial v}{\partial r} + \frac{uv}{r} + w \frac{\partial v}{\partial z} = \nu \left[\frac{\partial^2 v}{\partial r^2} + \frac{\partial}{\partial r} \left(\frac{v}{r} \right) + \frac{\partial^2 v}{\partial z^2} \right] \quad (3)$$

Axial Momentum

$$u \frac{\partial w}{\partial r} + w \frac{\partial w}{\partial z} = -\frac{1}{\rho} \frac{\partial p}{\partial z} + \nu \left[\frac{\partial^2 w}{\partial r^2} + \frac{1}{r} \frac{\partial w}{\partial r} + \frac{\partial^2 w}{\partial z^2} \right] \quad (4)$$

Energy

$$\rho C \left(u \frac{\partial T}{\partial r} + w \frac{\partial T}{\partial z} \right) = k \left(\frac{\partial^2 T}{\partial r^2} + \frac{1}{r} \frac{\partial T}{\partial r} + \frac{\partial^2 T}{\partial z^2} \right) + \mu \left[2 \left(\frac{\partial u}{\partial r} \right)^2 + 2 \left(\frac{u}{r} \right)^2 + 2 \left(\frac{\partial w}{\partial z} \right)^2 + \left(\frac{\partial v}{\partial z} \right)^2 + \left(\frac{\partial u}{\partial z} + \frac{\partial w}{\partial r} \right)^2 + \left(\frac{\partial v}{\partial r} - \frac{v}{r} \right)^2 \right] \quad (5)$$

Equations (1)-(5) can be non-dimensionalized by using the following substitutions:

$$\bar{r} = \frac{r}{R}, \bar{z} = \frac{z}{R}, \bar{u} = \frac{u}{\omega R}, \bar{v} = \frac{v}{\omega R}, \bar{w} = \frac{w}{\omega R}, \bar{p} = \frac{P}{\rho \omega^2 R^2}, \bar{T} = \frac{T}{T_0}$$

where T_0 is the fluid inlet temperature.

Using these expressions the differential equations become in dimensionless form:

Continuity

$$\frac{\partial \bar{u}}{\partial \bar{r}} + \frac{\bar{u}}{\bar{r}} + \frac{\partial \bar{w}}{\partial \bar{z}} = 0 \quad (6)$$

Radial Momentum

$$\bar{u} \frac{\partial \bar{u}}{\partial \bar{r}} - \frac{\bar{v}^2}{\bar{r}} + \bar{w} \frac{\partial \bar{u}}{\partial \bar{z}} = - \frac{\partial \bar{p}}{\partial \bar{r}} + \frac{1}{\text{Re}} \left[\frac{\partial^2 \bar{u}}{\partial \bar{r}^2} + \frac{1}{\bar{r}} \frac{\partial \bar{u}}{\partial \bar{r}} - \frac{\bar{u}}{\bar{r}} + \frac{\partial^2 \bar{w}}{\partial \bar{z}^2} \right] \quad (7)$$

Tangential Momentum

$$\bar{u} \frac{\partial \bar{v}}{\partial \bar{r}} + \frac{\bar{u}\bar{v}}{\bar{r}} + \bar{w} \frac{\partial \bar{v}}{\partial \bar{z}} = \frac{1}{\text{Re}} \left[\frac{\partial^2 \bar{v}}{\partial \bar{r}^2} + \frac{1}{\bar{r}} \frac{\partial \bar{v}}{\partial \bar{r}} - \frac{\bar{v}}{\bar{r}^2} + \frac{\partial^2 \bar{v}}{\partial \bar{z}^2} \right] \quad (8)$$

Axial Momentum

$$\bar{u} \frac{\partial \bar{w}}{\partial \bar{r}} + \bar{w} \frac{\partial \bar{w}}{\partial \bar{z}} = - \frac{\partial \bar{p}}{\partial \bar{z}} + \frac{1}{\text{Re}} \left[\frac{\partial^2 \bar{w}}{\partial \bar{r}^2} + \frac{1}{\bar{r}} \frac{\partial \bar{w}}{\partial \bar{r}} + \frac{\partial^2 \bar{w}}{\partial \bar{z}^2} \right] \quad (9)$$

Energy

$$\begin{aligned}
 \bar{u} \frac{\partial \bar{T}}{\partial \bar{r}} + \bar{w} \frac{\partial \bar{T}}{\partial \bar{z}} = \frac{1}{Pr Re} \left(\frac{\partial^2 \bar{T}}{\partial \bar{r}^2} + \frac{1}{\bar{r}} \frac{\partial \bar{T}}{\partial \bar{r}} + \frac{\partial^2 \bar{T}}{\partial \bar{z}^2} \right) + \\
 \frac{Ec}{Re} \left[2 \left(\frac{\partial \bar{u}}{\partial \bar{r}} \right)^2 + 2 \frac{\bar{u}^2}{\bar{r}^2} + 2 \left(\frac{\partial \bar{w}}{\partial \bar{z}} \right)^2 + \left(\frac{\partial \bar{v}}{\partial \bar{z}} \right)^2 + \left(\frac{\partial \bar{u}}{\partial \bar{z}} \right)^2 + \frac{\partial \bar{u}}{\partial \bar{z}} \frac{\partial \bar{w}}{\partial \bar{r}} + \right. \\
 \left. \left(\frac{\partial \bar{w}}{\partial \bar{r}} \right)^2 + \left(\frac{\partial \bar{v}}{\partial \bar{r}} \right)^2 - \frac{\bar{v}}{\bar{r}} \frac{\partial \bar{v}}{\partial \bar{r}} + \frac{\bar{v}^2}{\bar{r}^2} \right] \quad (10)
 \end{aligned}$$

where

$$Re = \frac{\omega R^2}{\nu}, \quad Pr = \frac{\mu C}{k}, \quad Ec = \frac{\omega^2 R^2}{C T_o} \quad (11)$$

Equations (6)-(9) can be solved independent of equation (10) and are to be done by numerical analysis. The resulting flow field solution can then be used to solve equation (10) for the temperature distribution. The entire solution of these equations will be completed at a later time.

RADIAL PRESSURE DISTRIBUTION

The sealing capability of a slinger will be determined by the radial pressure gradient. The exact nature of this gradient will be quite complex due to the complicated flow across the disk. As an example, see reference 22 for a detailed look at secondary flows in a centrifugal impeller. Other inter-related factors are the presence of a liquid-vapor interface [18], pressure variation between vanes [23], and fluid throughflow [24], although the experimental results of Due [25] indicated that there was no significant dependence on flow coefficient.

A simplified approach is necessary so it is assumed that once steady state operation is reached the fluid in any thin annulus moves as a solid mass rotating about its own axis with angular velocity $K\omega$ where K is the ratio of the fluid angular velocity, β , to the disk angular velocity, ω . The radial velocity will be small compared to the tangential velocity and will be neglected, so the radial momentum equation (2) becomes:

$$\frac{1}{r} v^2 = \frac{1}{\rho} \frac{\partial p}{\partial r} \quad (12)$$

Since the pressure is a function of the radius only, this equation reduces to an ordinary differential equation. Then substituting $K\omega r$ for the tangential velocity results in:

$$\frac{dp}{dr} = \frac{\rho (K\omega r)^2}{r} \quad (13)$$

Integrating between any two radial points on the face of the slinger yields:

$$\Delta p = p_2 - p_1 = \frac{\rho}{2} K^2 \omega^2 (r_2^2 - r_1^2) \quad (14)$$

Other investigators [5], [12], [19], have obtained the same result by different approaches and their close correlation with their experimental results verifies its general applicability.

The value of K must be determined to solve for the pressure gradient, but K will be influenced by several variables including geometry of the seal, solidity, clearances, Reynolds number and radius. Since the fluid between the vanes on a slinger will rotate essentially with the slinger angular velocity, $K \approx 1$, but the fluid in the gap between the vanes and the housing will rotate at some lesser angular velocity. The regions could be considered separately but in reference 25 Due established equations for K by correlating experimental data to equation (14) using a statistical regression analysis. The applicable equation from Due is:

$$K = 0.951 - 0.282 \frac{r}{R} + 0.2137 \frac{b}{a+b} + 0.175(b) \quad (15)$$

where R is the seal radius, a is the seal to housing gap width, b is the blade height, and r is the particular radial distance. Average values for the subject seal are: a = 0.08 in., b = 0.25 in., and R = 2.24 in. Using these values in equations (14) and (15) the radial pressure distribution can be determined.

TEMPERATURE DISTRIBUTION

Due to the rotation of the slinger, energy is imparted to the fluid, manifested as a temperature rise and eventually as a phase change from liquid to vapor. This phase change causes a drastic change in the density of the fluid and a resulting smaller radial pressure change. To account for this effect, the radial temperature distribution must be known.

A simple control volume analysis will be used to look at the enthalpy increase from one radial plane to another. Knowing the enthalpy and pressure defines the state at a point so the temperature, density or any other required properties can be found using a standard computer code [27] or from tables or charts [28]. The process will be assumed to be steady state, steady flow [26] and the radial distance between inlet and exit planes will be small so the change in kinetic and potential energies will be negligible. Heat transfer to the walls will be considered negligible. Continuity requires the mass flow at the inlet to equal that of the exit and with the other conditions as above, the one dimensional energy equation will be:

$$\dot{m} h_1 = \dot{m} h_2 + \dot{W} \quad (16)$$

where \dot{m} is the fluid mass flow through the seal, h is the enthalpy at a particular radius and \dot{W} is the shaft power absorbed by the fluid.

$$\dot{W} = -\tau \omega \quad (17)$$

with torque being defined for one side of a disk [5], [12] as:

$$\tau = \frac{C_m \rho \omega^2 (r_2^5 - r_1^5)}{4} \quad (18)$$

so the change in enthalpy becomes:

$$h_2 - h_1 = \frac{C_m \rho \omega^3 (r_2^5 - r_1^5)}{4 \dot{m}} \quad (19)$$

The moment coefficient C_m must be determined. For the smooth side of the seal the value of C_{ms} is found to be 0.0008 for $Re = 10^8$ using the empirical

equations from reference 8. For this seal the flow is between regimes 3 and 4 as defined by Daily and Nece [5] so an average value of C_{ms} from the two applicable equations is used. The disk is thin so no contributions due to the tip are considered.

For the vaned side of the disk the moment coefficient will be greater and will be approximated by the method derived by Thew [15] where

$$C_{mv} = C_{ms} + 1.2 \frac{b+a}{R} (1-K)^2 \left[\left(\frac{a+b}{0.8a} \right) - 1 \right]^2 \left[1 - \left(\frac{R_{int}}{R} \right)^4 \right] \quad (20)$$

The pressure and enthalpy distributions can now be calculated, yielding any other thermophysical properties needed at a radial point. The velocity ratio is found to be essentially constant for the smooth side, $K = 0.44$, but varies with radius on the vaned side so C_m also varies as shown. The mass flow of 0.13 lbm/sec and slinger angular velocity of 2932 rad/sec are obtained from engine balance data [30]. On the vaned side it is necessary to assume a value of R_{int} , determine the point at which the fluid begins vaporizing, and use this as the new assumed value of R_{int} , continuing until the assumed and determined values converge. For this seal a radius of 2.2 in. is where vaporization, therefore the interface begins.

Using the property values from reference 29 (their station 20) as being at the root of the smooth side of the slinger ($r = 1.7$ in.) the radial property variations are calculated and summarized in Table 1.

CONCLUSIONS AND RECOMMENDATIONS

A thorough literature survey has been completed with the most applicable work listed beginning on page XVIII-16. A set of non-dimensional partial differential equations have been established which describe the flow on the face of a slinger seal. The pressure and temperature distributions for the Space Shuttle Main Engine high pressure oxidizer turbopump primary slinger seal have been described and the location of the liquid-vapor interface determined.

The radial pressure distribution equation (14) should provide good results based on the experimental work confirming it, but the analysis of the temperature variation is approximate at best and too many simplifying assumptions are made to trust its validity. The smooth side values for C_m are sufficiently grounded with experimental work but the equation for C_{mv} has not been investigated enough to verify its generality. Due to these factors and the nature of the initial data used from reference 29, it is believed that this simplified analysis can be used only to predict tendencies.

Further work needs to be done on vaned slinger seals. Solution of the governing equations and matching of all boundary conditions should be done to get a complete picture of the flow involved and to predict more accurate pressure and temperature distributions. An experimental program should be conducted to determine actual pressure and temperature values to correlate with the analytical work. No previous experimental work could be found through the literature survey which deals with liquid oxygen, high rotational speeds as found in modern turbo-pumps, or the energy transfer involved with the production of a liquid-vapor interface.

Recommendations based on the literature survey that could improve the sealing performance of the Space Shuttle Main Engine high pressure oxidizer turbopump slinger are:

- include circumferential tangs on the vaned side of the seal with matching grooves in the housing to stabilize the liquid-vapor interface, thereby reducing leakage [14] .
- increase the number of vanes to 24 which has been established as optimum [14] .
- utilize a non-wetting coating on the housing opposite the vaned side of the seal to reduce secondary flow that enhances leakage. [20] .

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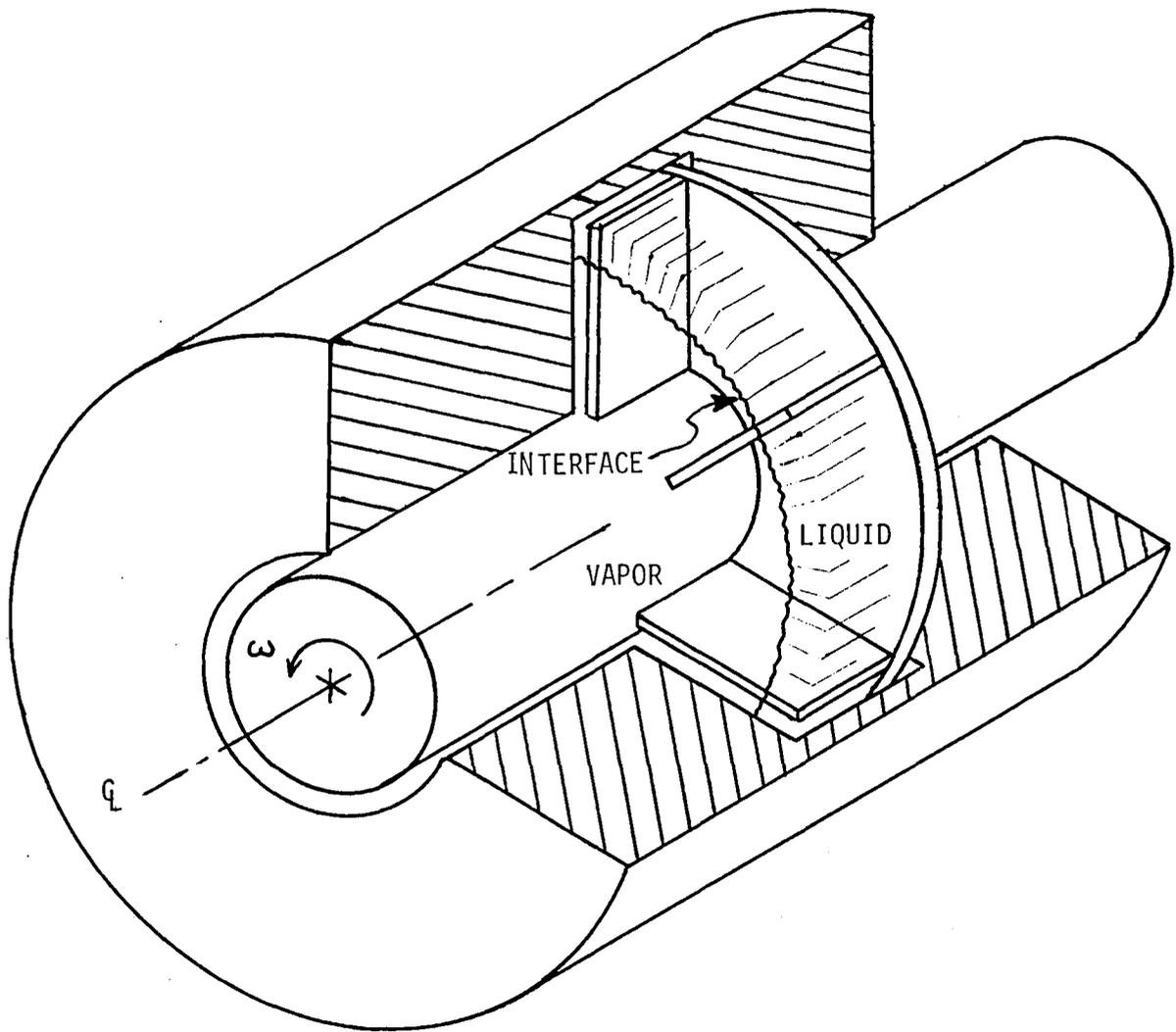


Figure 1. Cutaway view of slinger seal model

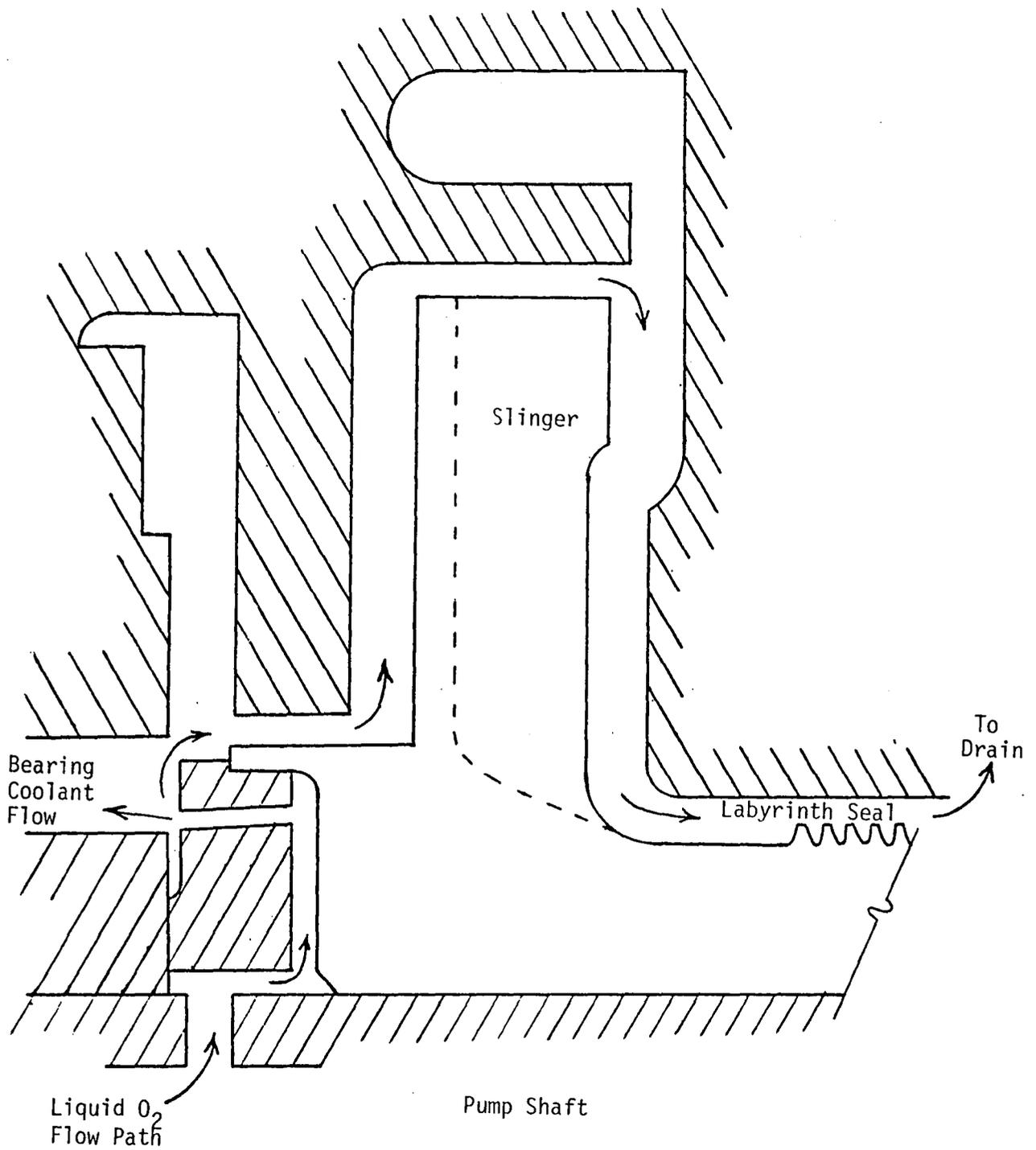


Figure 2. Actual slinger configuration

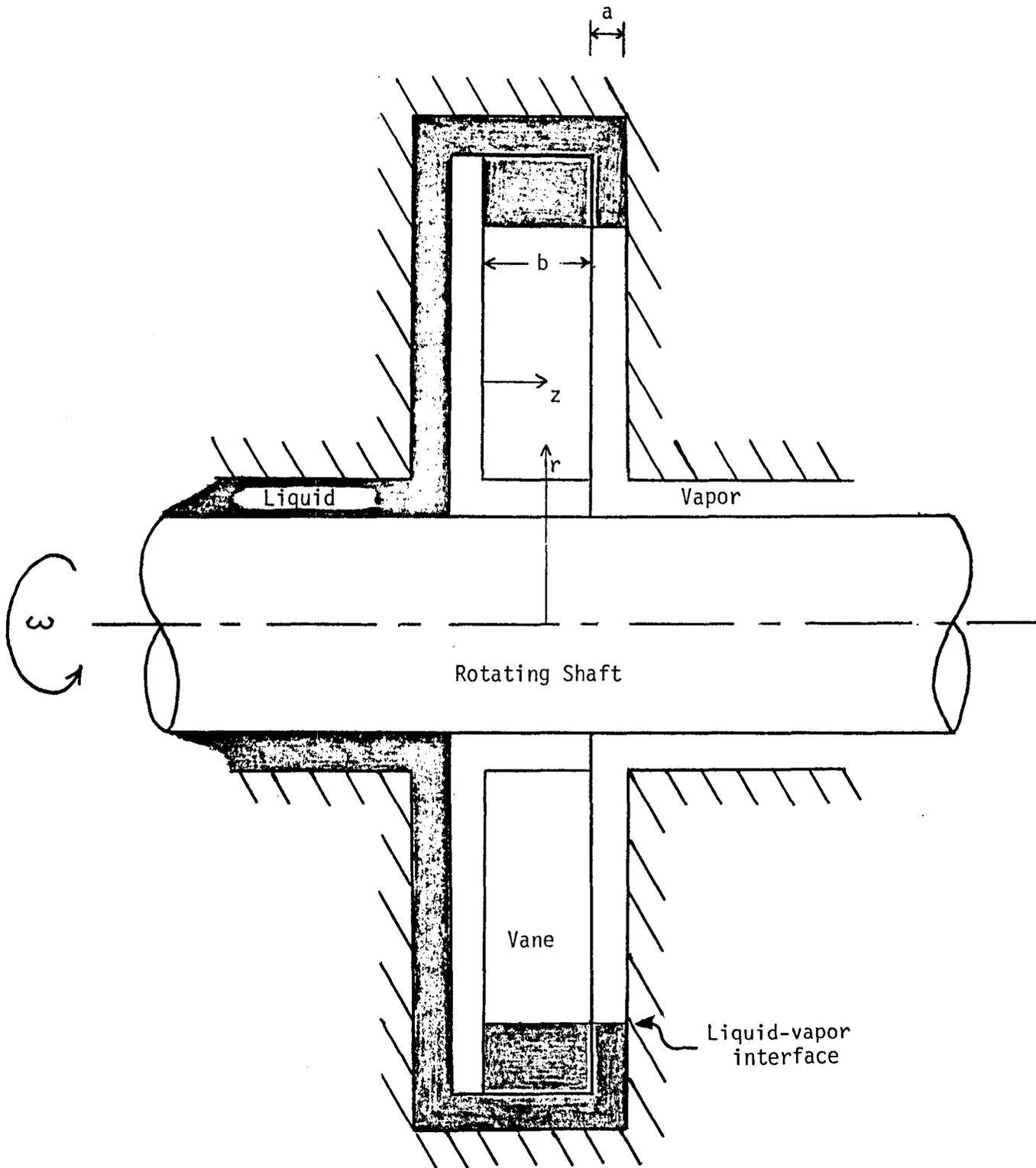


Figure 3. Slinger model with notation.

TABLE 1
SLINGER RADIAL PROPERTY DISTRIBUTIONS

Smooth Side

Radius (in.)	Velocity Ratio	Moment Coefficient	Enthalpy (BTU/lbm)	Pressure (lbf/in ²)	Temperature (°R)	Density (lbm/ft ³)
1.7	.44	.0008	-34.3	334	216	61.2
1.8	.44	.0008	-32.5	361	219	60.5
1.9	.44	.0008	-30.3	388	223	59
2.0	.44	.0008	-27.5	416	230	58
2.1	.44	.0008	-24.3	446	237	56
2.2	.44	.0008	-20.6	476	245	54
2.24	.44	.0008	-18.9	488	247	53

Vaned Side

Radius (in.)	Velocity Ratio	Moment Coefficient	Enthalpy (BTU/lbm)	Pressure (lbf/in. ²)	Temperature (°R)	Density (lbm/ft ³)	Quality
2.24			-18.9	488	247	53	0.0
2.22	.88	.0040	-14.8	464	254	51	0.0
2.20	.88	.0039	-11.2	442	258	48	0.0
2.18	.88	.0037	- 8.0	421	255	42	0.1
2.16	.89	.0036	- 5.4	402	253	35	0.2

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THE UNIVERSITY OF ALABAMA

VALIDATION OF THE SOLAR HEATING AND COOLING HIGH
SPEED PERFORMANCS (HISPER) COMPUTER CODE

Prepared By:	Donald B. Wallace, Ph. D., P. E.
Academic Rank:	Associate Professor
University and Department	The University of Alabama in Huntsville Department of Mechanical Engineering
NASA/MSFC: (Laboratory) (Division) (Branch)	Systems Analysis and Integration Systems Engineering Systems Requirement & Definition
MSFC Counterpart:	Robert F. Elkin
Date:	August 1, 1980
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By

Donald B. Wallace, Ph.D., P. E.
Associate Professor of Mechanical Engineering
The University of Alabama in Huntsville
Huntsville, Alabama 35809

ABSTRACT

Computer simulations of solar energy systems are used to evaluate system performance, size system components and predict fossil fuel savings. The solar heating and cooling high speed performance (HISPER) computer code was previously developed to give a quick and accurate predictions. As a simplification of the TRNSYS program, it achieves its computational speed by not simulating detailed system operations or performing detailed load computations.

In order to validate the HISPER computer code for air systems the simulation was compared to the actual performance of an operational test site. The site selected was the Home Builders Association of Huntsville office building in Huntsville, Alabama. Solar insolation, ambient temperature, water usage rate, and water main temperatures from the data tapes for the site were used as input for the HISPER program.

The HISPER program was found to predict the heating loads and solar fraction of the loads with errors of less than ten percent. Good correlation was found on both a seasonal basis and a monthly basis. Several parameters (such as infiltration rate and the outside ambient temperature above which heating is not required) were found to require careful selection for accurate simulation.

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INTRODUCTION

Solar energy collection systems are installed on dwellings in order to reduce the occupant's dependence on conventional forms of energy for heating and cooling. Due to the diffuse nature of solar energy and the variability in the weather, it is impractical to install a system large enough to provide all of the energy needs. A back-up system, using conventional energy, must be provided for supplying energy during long periods of cloudy weather. The solar energy system is ideally sized to maximize the benefit to cost ratio (the ratio of annual solar energy provided divided by the life cycle cost of the solar energy system). Because of the large number of design parameters and the weather variability, the design optimization problem is a difficult one.

Computer programs have been developed to help predict solar heating and cooling (SHAC) performance. One widely used program is TRNSYS which was developed at the University of Wisconsin Solar Energy Laboratory [1]*. This is a general purpose program that allows the user to simulate a system by defining the performance of and the interaction between various types of components (collectors, storage, pumps, control logic, etc.). TRNSYS gives an accurate indication of the model performance but it does require a significant amount of computer time to run. In a design situation it is more desirable to have an analysis tool which can quickly and economically give reasonable answers. This allows more design iterations to be analyzed.

In response to the need for a rapid solar simulation computer program, Northrup Services, Inc. (under contract to NASA) developed HISPER - a high speed performance program [2]. HISPER is basically a simplification of the TRNSYS program in which some detailed system operation and load calculations are not performed (such as room temperature oscillations and ductwork heat losses). When HISPER was compared to TRNSYS for identical cases, the predicted system performance differed by less than 7 percent while computer time required by TRNSYS was approximately 30 times greater than the time required by HISPER.

The usefulness of any simulation is dependent on how well it predicts the performance of the actual system. This study provides this validation of the HISPER program by comparing the predicted results using actual weather data

*numbers in brackets refer to references at the end of this paper.

with the measured performance of the site during this same period. The Home Builders Association of Huntsville office building was selected as the site to be simulated.

This report first describes the Home Builders Association office building and its operation over a period from September, 1978 to April, 1979. Next, the basic organization of the HISPER program and the modifications necessary to run actual weather data is presented. Third, the simulation input data is discussed. The fourth part is a discussion of the results of the study including the effects of several critical design parameters. Finally the results of this study are discussed and recommendations made for the use of HISPER and for topics which need further study.

OBJECTIVES

The primary objective of this study are to verify the general computational philosophy of the HISPER program by demonstrating that the program adequately predicts the performance of actual solar energy systems. This verification will result in more confidence in HISPER for use as a design tool.

Secondary objectives include: identification of the effects of parameter values on the simulation performance, identification of any desirable changes in the HISPER logic, and the establishment of a post-design review procedure in which the actual performance of a site can be compared with the predicted design value.

OPERATIONAL TEST SITE

Site Description:

The operational test site selected for use in this study is the Home Builders Association of Huntsville office building located in Huntsville, Alabama (see Figure 1). This building has two stories with a total floor area of approximately 210 square meters (2270 square feet). The building was designed by IBM (under contract to NASA) to be a single family dwelling but is being used as an office building at this location.

The south facing roof is sloped at 45° and contains 30 solar collectors (manufactured by Solar Energy Products) for a gross area of 67 square meters (720 square feet). These collectors are single glazed and use air as the energy transport medium.

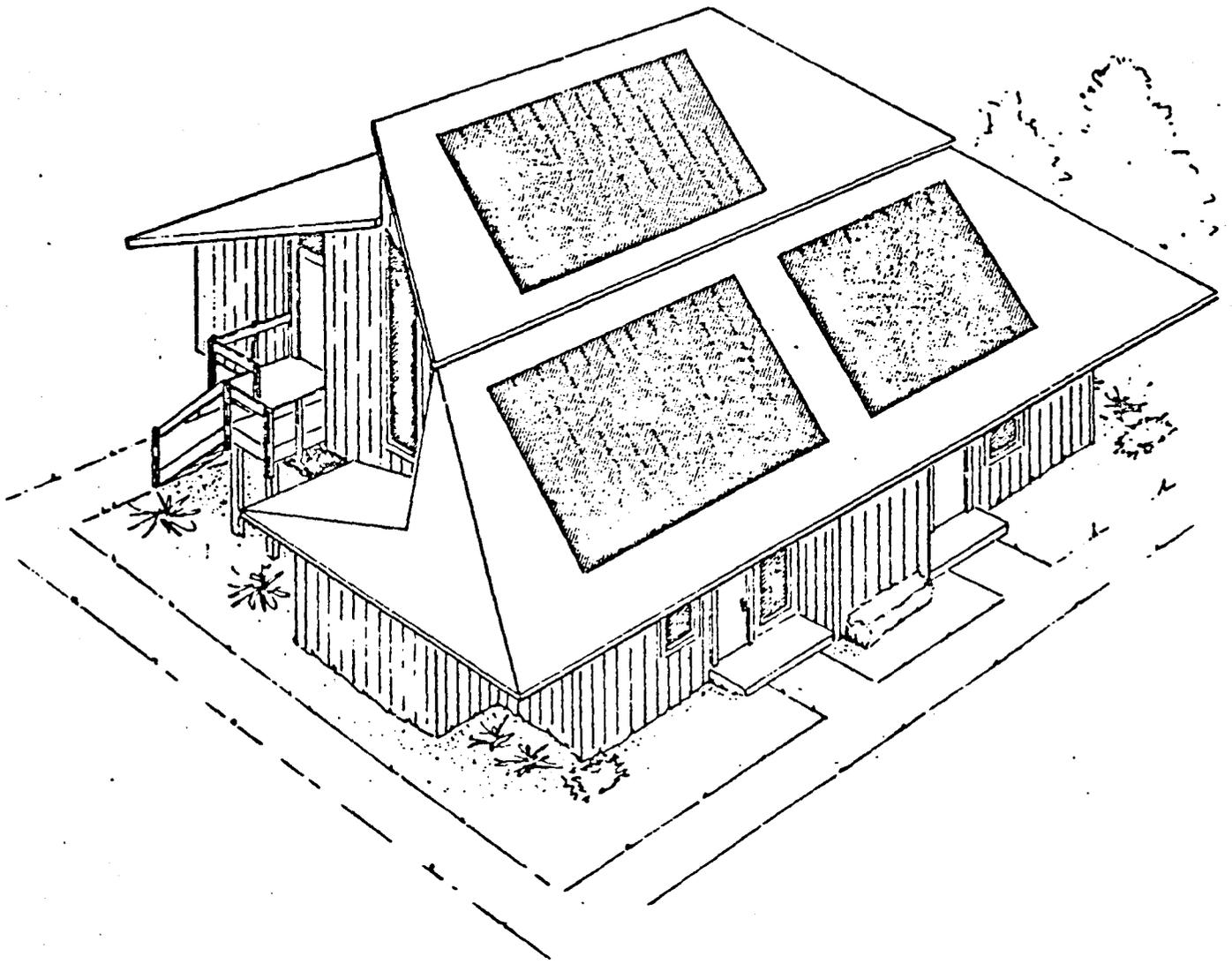


Figure 1. Office Building for the Home Builders Association of Huntsville, Alabama

Figure 2 shows a schematic of the solar energy collection system. Air heated in the solar collectors first passes through an air/water heat exchanger so that the domestic water going to the hot water heater (not shown) may be preheated. Then the hot air is directed by the air handles either to the house or to the pebble bed storage, depending upon the energy needs of the house at that time. The pebble bed storage has a volume of 14 cubic meters (500 cubic feet) and contains 23 metric tons (25 tons) of river rock.

During periods when no solar energy is being collected, house air may be passed through the pebble bed thus providing energy to the house. If the temperature of the pebble bed storage is too low (below 32°C or 90°F), a Westinghouse air-to-air heat pump provides the heat demanded by the house.

A complete description of the building and the solar energy system design and operation is given in various reports prepared by the designers - the IBM Corporation Federal Systems Division [3, 4, 5].

Operational Test Site Data:

The Home Builders Association office building was equipped with 45 sensors to obtain data for performance evaluation. These include 27 temperature sensors, 7 electric power recorders, 5 flow rate sensors, a solar insolation indicator, and 5 others. Each of these sensors is scanned at 320 second intervals (5 minutes and 20 seconds) and their average value is recorded by a data logger. All of this information is then put on to magnetic tapes.

This data is used by IBM to prepare a monthly performance summary for the site. The performance summary includes the climatic conditions, house heating and hot water loads, and solar collection information. The monthly summaries also include any known problems (such as malfunctioning equipment) which occur during the month.

The period chosen for the comparison of HISPER and the actual system was October 1978 through April, 1979. This was chosen since both the 5 minute recorder data (on magnetic tape) and the monthly reports were available. Summer months were not included since the real system is put in "Summer Mode" in which the pebble bed storage is not used. HISPER does not have the capabilities of the "Summer Mode".

A review of the information on the data tapes indicated that the data tapes could not be used directly with the HISPER program. First, periodic sensor malfunctions were discovered (such as ambient temperature going from about 29°C (84°F) to -20°C (-4°F) for a period of about 45 minutes and then back again). Secondly, gaps in the data of up to 30 hours was observed on several occasions. These problems were overcome by correcting the obviously erroneous data and inserting reasonable values for the missing data. This data manipulation will result in a small discrepancy in the IBM Monthly Reports and the HISPER predictions. Since the total amount of bad or missing data is

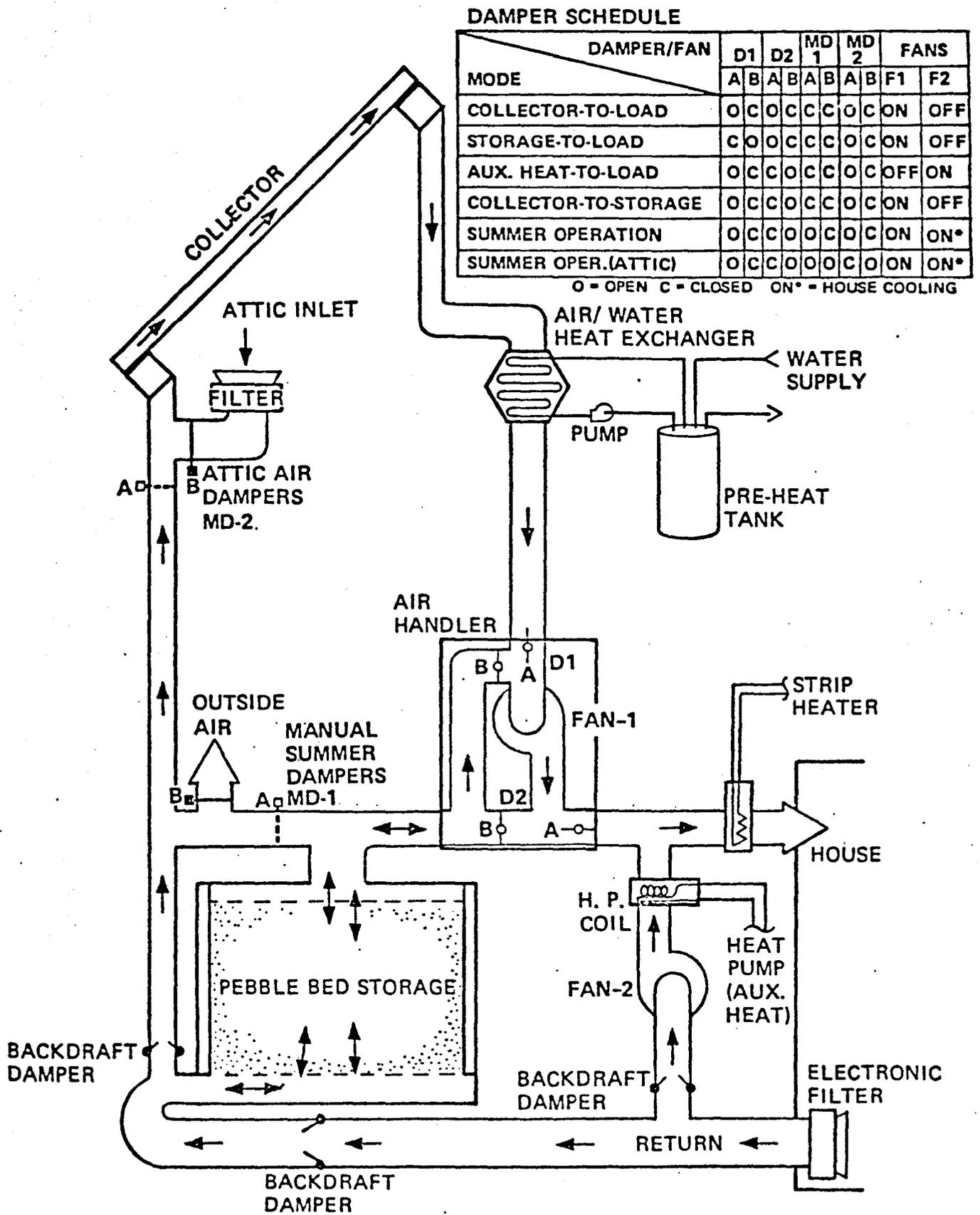


Figure 2. Actual System Configuration

very small, the discrepancy is insignificant.

Site Operation:

A review of the test site data revealed several areas which could lead to significant errors. These relate to hot water usage and temperature control of the house.

Since the site is being used as an office building rather than a residence, the demand for hot water was quite low. When the hot water tank was set at 63°C (145°F), before November 22, 1978, an average of only 30kg (7.8 gallons) of hot water were used daily. After the hot water temperature was lowered to 51°C (123°F) on November 22, 1978, the hot water demand rose to an average of 36kg (9.5 gallons) a day. Very little water was used on the weekends.

The temperature of the water coming out of the water main fluctuates over the period of the year. Water main temperature varies from a high in August at about 25°C (77°F) to a low in February at about 9°C (48°F). This variation probably results from the water being transported in mains that are not buried deep enough to be unaffected by seasonal temperature variations.

Another problem is illustrated in Figure 3. March 24th and 25th were cold weekend days following a rather mild week (high ambient temperatures). Apparently someone had shut off the thermostat before leaving work on Friday so that when the workers returned on Monday, the building temperature was 8°C (46°F). There is other evidence that the thermostat settings had been adjusted over the period of investigation.

Figure 3 also shows that a significant variation in room temperature does occur on a daily basis. This may be due to a direct solar gain either through the windows or through the walls. The resulting heating in the occupied area will reduce the amount of solar or auxiliary energy required.

HISPER MODIFICATIONS

The HISPER simulation program consists of a main program and three fundamental subroutines: MODEL, WETHR, and SOLAR. The purpose of each of these subprograms is:

1. MAIN - The main program reads the system parameters, performs the time integration, and controls the output of the program. The main program calls subroutine MODEL.

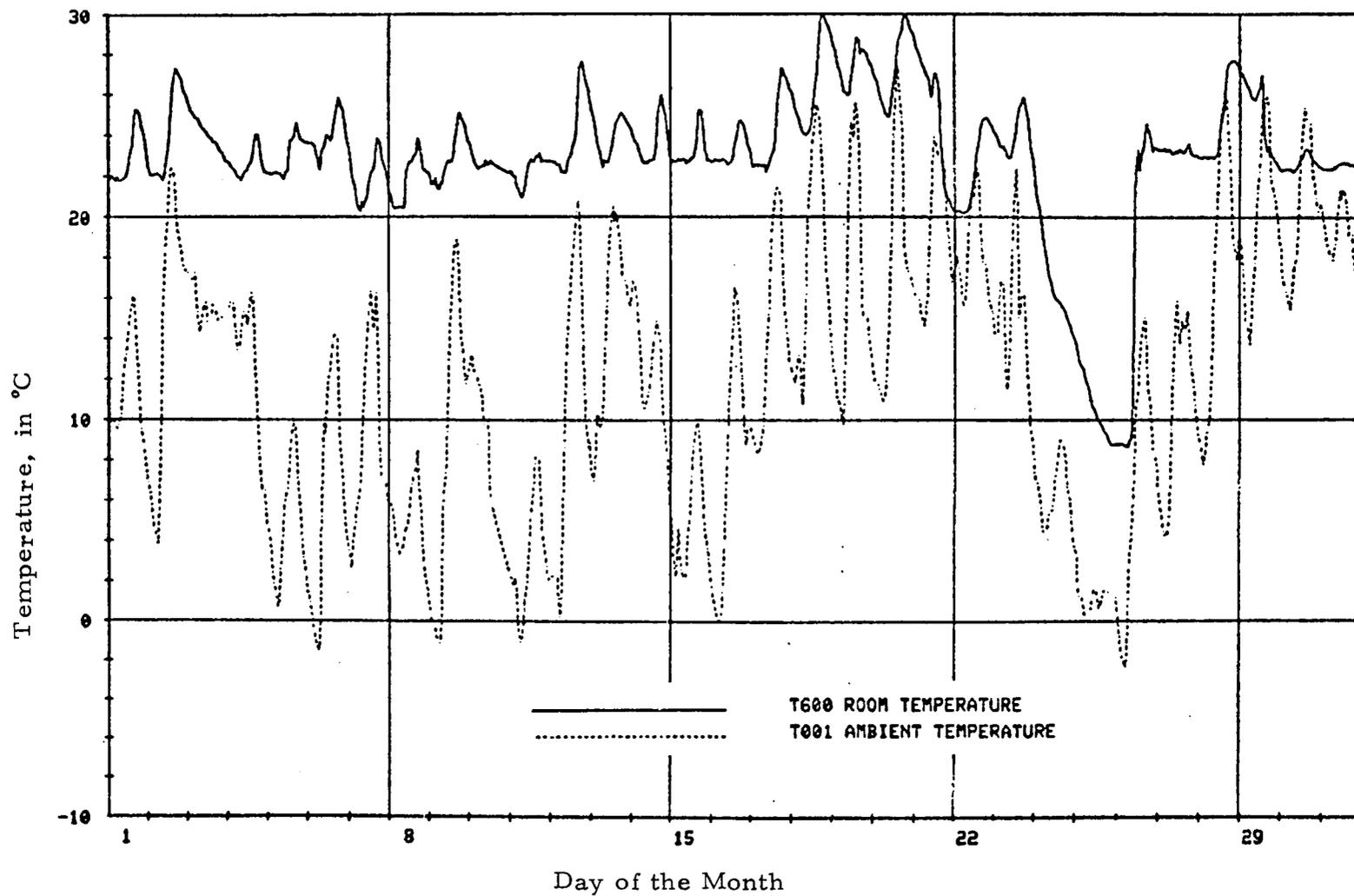


Figure 3. Ambient and Room Temperatures in March, 1979

2. MODEL - This subroutine calculates all of the energy flows in the system model in a given increment of time. It calculates collector performance, the dwelling heating load, the hot water load, and the flow of energy in the solar system (including in to and out of storage). It calls WETHR and SOLAR.
3. WETHR - This subroutine provides ambient temperature and horizontal solar insolation to MODEL. These values are read from weather tapes containing typical weather data.
4. SOLAR - This subroutine receives the total hourly horizontal insolation output by the WETHR subroutine and converts it to total insolation incident on the collector surface using one of four user determined methods.

The basic HISPER procedure, with the exception of WETHR, was not changed significantly. Some control logic was added to MODEL and different forms of output were provided for in MAIN. Subroutine SOLAR was not altered at all. Since the climatic data was from a new source, WETHR had to be changed completely.

HISPER was designed to read weather tapes which contain typical climatic data for a particular location. In order to validate HISPER predictions for an actual operational test site, it was necessary to modify HISPER to read the actual climatic conditions at that site. Sensors at the Home Builders Association office building include an ambient temperature sensor and a pyranometer to measure total solar insolation in the collector plane.

HISPER uses both insolation on a horizontal surface (for load calculations) and insolation on the collector plane (for solar collector performance) in its simulation procedure. Normally the horizontal insolation is provided by the weather tapes and HISPER uses one of four standard techniques to estimate the tilted insolation from the horizontal value (see references [1, 6, 7]). Since the site data includes only tilted insolation data, HISPER's WETHR subroutine had to be modified to reverse the standard procedures and predict the horizontal insolation value which will produce the observed tilted value when SOLAR is called.

HISPER is designed to simulate a typical domestic dwelling so the assumed hot water load represents what a normal family might use each day. Since the system being studied is an office building, the HISPER predicted daily usage (224kg -- 59 gal) does not correspond well to the actual usage of about 33kg (8.7 gallons) per day. Therefore HISPER was modified to read (in WETHR) actual water usage and true water main temperature from the site data.

The control logic used in HISPER is designed to provide the maximum amount of energy to the loads as possible. The main control features include:

1. Turning on the collector when ever energy can be gained.
2. Energy is extracted by the hot water preheat tank whenever the collector is on and the outlet temperature is greater than the preheat tank temperature.
3. Energy is provided to the house whenever the house has a load and the air inlet temperature, either from the collector or storage, is greater than the house temperature (returns air temperature is assumed to be room temperature).
4. Energy is not allowed to go to storage if the temperature from the collector is less than the temperature at the top of the storage bed.

In order to study the effects of actual control settings on the HISPER Simulation performance, the following features were added to the MODEL subroutine control logic.

1. A minimum temperature difference between the collector plate temperature and the bottom of the pebble bed storage bin before the collector would collect energy (note: the plate temperature was based on steady state flow conditions) Actual site required a 15°C (28°F) temperature difference [8].
2. A minimum temperature difference between the collector outlet temperature and the preheat tank for domestic hot water before energy could be extracted from the air and be put in the preheat tank. Actual site required 11°C (20°F) to turn on and no less than 2°C (3°F) before turning off [8].
3. A minimum temperature at the top of the pebble bed storage was necessary for the storage to be allowed to provide energy to the house. Actual site required a temperature from the storage of at least 32°C (90°F). [8].
4. An air return temperature from the house that was lower than the nominal house temperature. Review of data tapes indicates that the return air is approximately 5°C (9°F) below room temperature. This is probably due to thermal stratification in the rooms.

The control logic as modified by these four features will be referred to as the "modified control logic."

SIMULATION INPUT DATA

A variety of sources was used to develop the input data for the HISPER simulation program. The attempt was to make the model for the Home Builders of Huntsville office building as realistic as possible.

The manufacturer's data for the sunworks solar collectors was used since other tests of the collectors by Wyle Laboratories showed good correlation with the predicted performance. Manufacturers data was also used for the air to water heat exchanger used for preheating water.

Flow rates in the air ducts were taken from actual tests performed by IBM after the construction of the test site.

Internally generated heat loads were assumed to be 2890 kJ/hr. This corresponds to the average heat generation at 10 people, lights, 2 refrigerators, and three coffee pots. The storage losses (since the storage bed is in the building, is considered to be part of the internally generated heat.

The parameters used for heat load calculations are primarily based on a design analysis done at NASA/MSFC in 1977. After the Home Builders Association building was constructed, tests were conducted to determine actual heat loss coefficients. The results of the study indicate that the building load demand is commensurate with the 720 BTUH/°F design building loss coefficient-area product (UA).

The only significant difference between the NASA/MSFC design values and the values used in the HISPER simulation was for infiltration. The NASA/MSFC value was one half air change per hour which assumes very tight construction. The tests performed on the building were conducted at night with no employees present at the office building. ASHRAE [9] recommends design infiltration rates from one half to two air changes per hour, depending on the number of windows and doors in the building. Since the building is being used as an office, with the possibility of many people entering and leaving during the day, an air infiltration rate of 1.5 air changes per hour was used.

SIMULATION RESULTS

HISPER was designed to give a reasonable prediction of solar performance of a home. This means that it should be able to predict the loads and the amount of solar energy delivered to the loads. The accuracy of the predictions was investigated with respect to several of the input parameters: infiltration rate, the outside ambient temperature above which the house is assumed to require no heat (TLH), and the effect of control logic. The domestic hot water predictions were also investigated.

Effect of TLH: (Temperature above which no energy is required by the house)

A house does not always require energy whenever the outside ambient temperature is less than the house temperature. This is because any house has some direct solar gain and a large amount of thermal capacitance. The direct solar gain satisfies some of the current house demand as well as being absorbed in the house for use at night. This effect can be seen in the room temperature fluctuations in Figure 3.

The data tapes for the Home Builders Association building were searched to determine what percentage of the time the heating/cooling system was on as a function of the outside temperature. Figure 4 shows this relationship. The increase in the percentages for the higher temperatures is due to cooling, not heating. From Figure 4, it is obvious that if the house is set at 22°C (72°F), it actually does not require significant heating until well below that value, at say 10°C.

Initial simulations were run assuming heating was not required over an ambient temperature of 10°C, i. e. TLH 10°C (50°F). The HISPER results are shown on a monthly load basis in Figure 5 and on a daily load basis in Figure 6 (a). An infiltration rate of 1.1 was assumed so that the total house load for the period was correct. This value of TLH results in HISPER under predicting the actual loads on cold days (heavy loads) and over predicting on warm days (light loads).

A value of TLH of 7°C (45°F) and an infiltration rate of 1.5 air changes an hour was then used to produce the results in Figure 5 and Figure 6 (b). These values result in a very good load prediction on both a monthly and daily basis.

Effects of Control Logic:

Once the actual loads were matched through the appropriate selection of TLH, the portion of the total load provided by solar energy was investigated. Figure 7 shows that a good correlation was achieved on a monthly basis.

A closer examination of component performance revealed that the temperature at the top of the storage was higher after periods of heavy load than HISPER predicted. (See Figures 8 and 9). This was due to the actual control logic which stopped energy withdrawal at temperatures of less than 32°C (90°F). The storage bed model of HISPER is programmed to supply, on demand, energy to the building anytime the temperature at the top of the storage is greater than the thermostat setting.

The temperature at the bottom of the pebble bed storage at the actual site often reached temperatures 5°C (9°F) below the nominal room temperature setting. This is probably due to the return air from the house being at a lower temperature than the room.

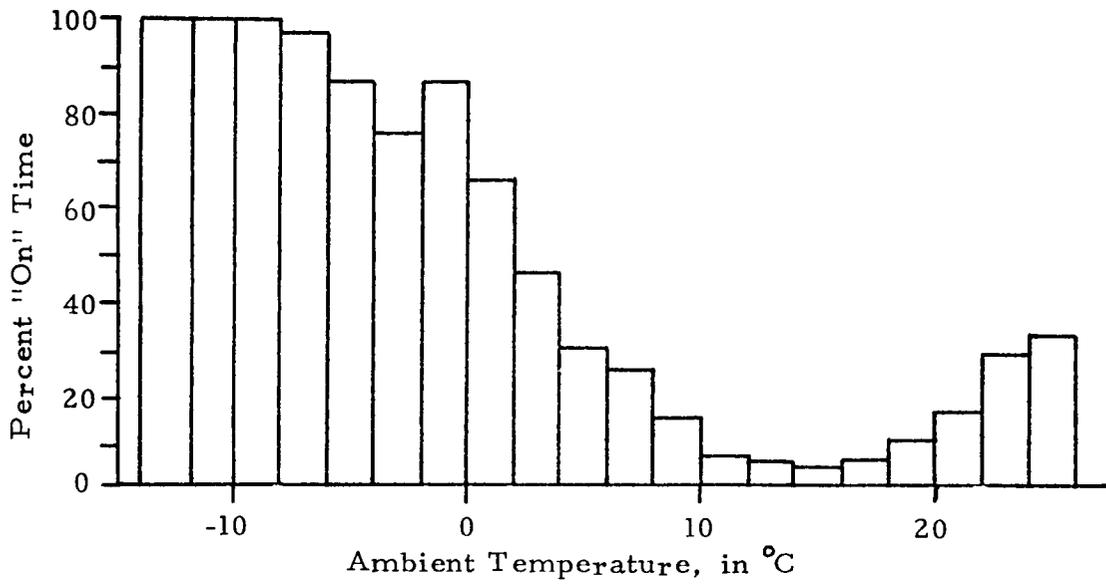


Figure 4. Heating/Cooling System "on" Time

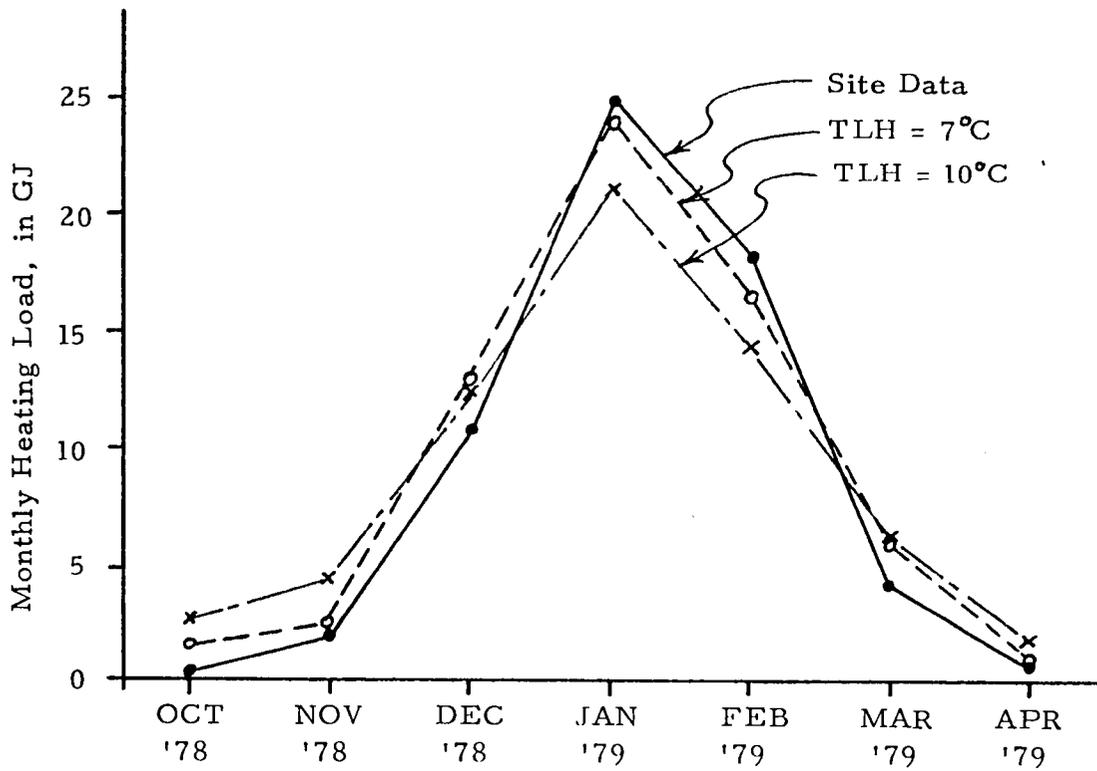
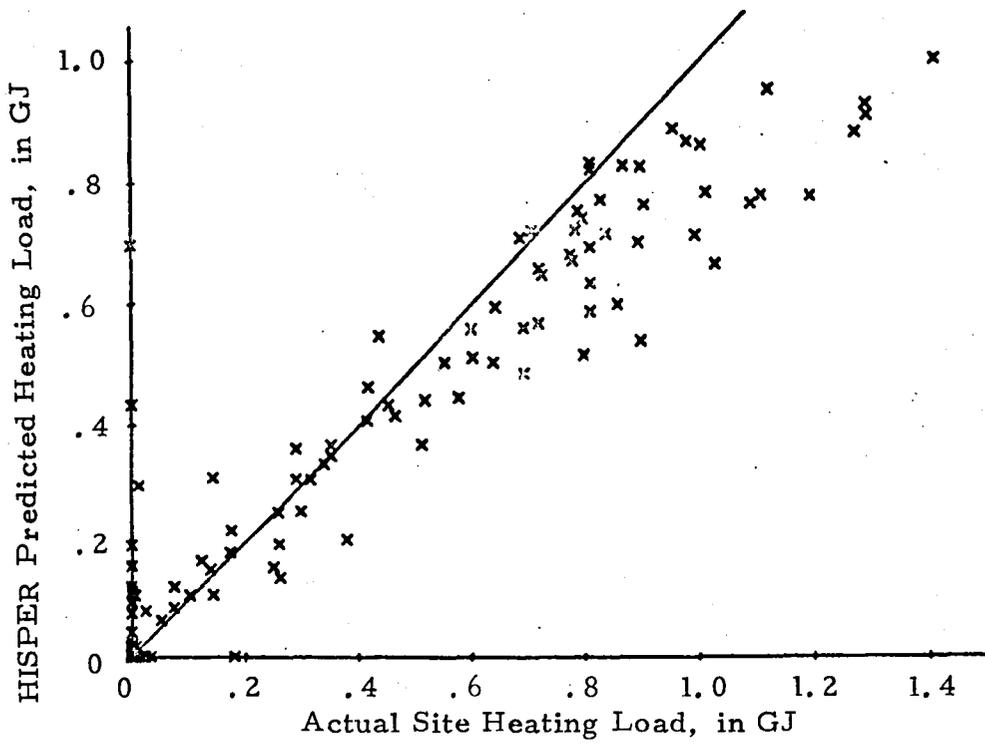
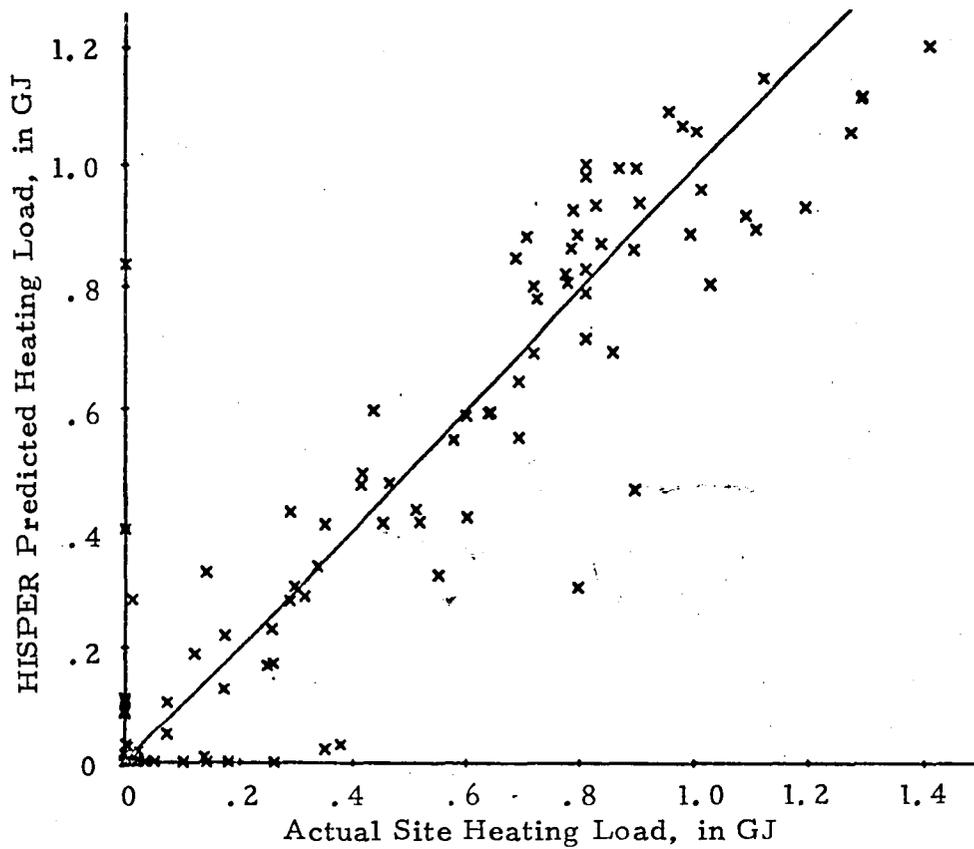


Figure 5. Predictions of Monthly Heating Load



(a) TLH = 10 °C



(b) TLH = 7 °C

Figure 6. Actual Versus Predicted Daily Heating Loads

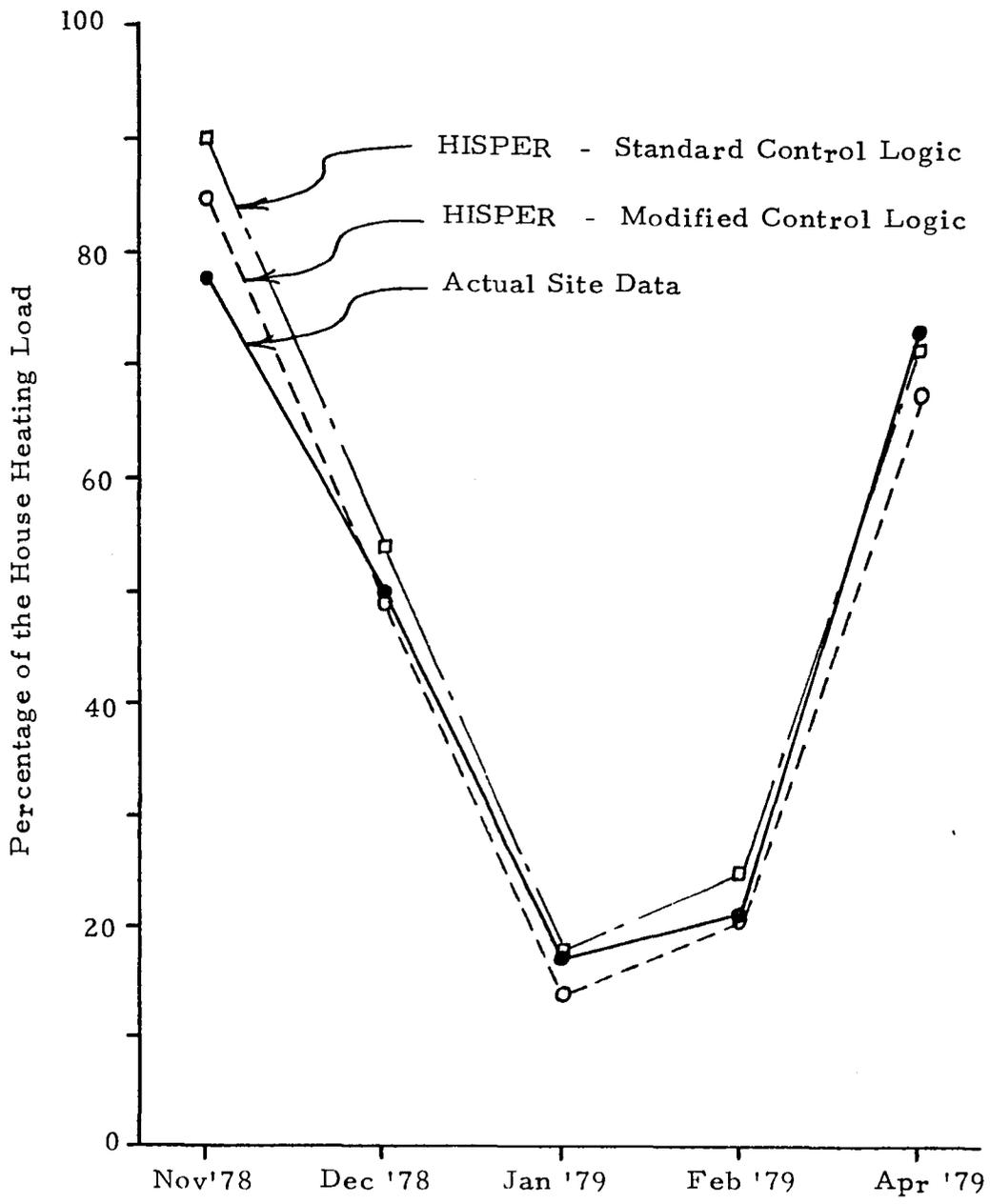
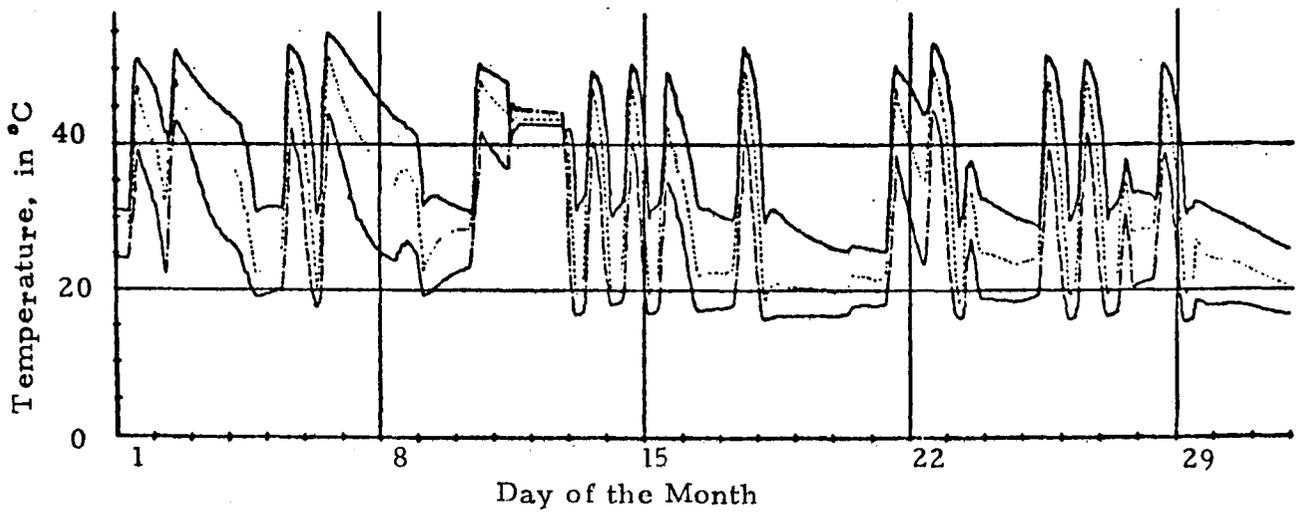
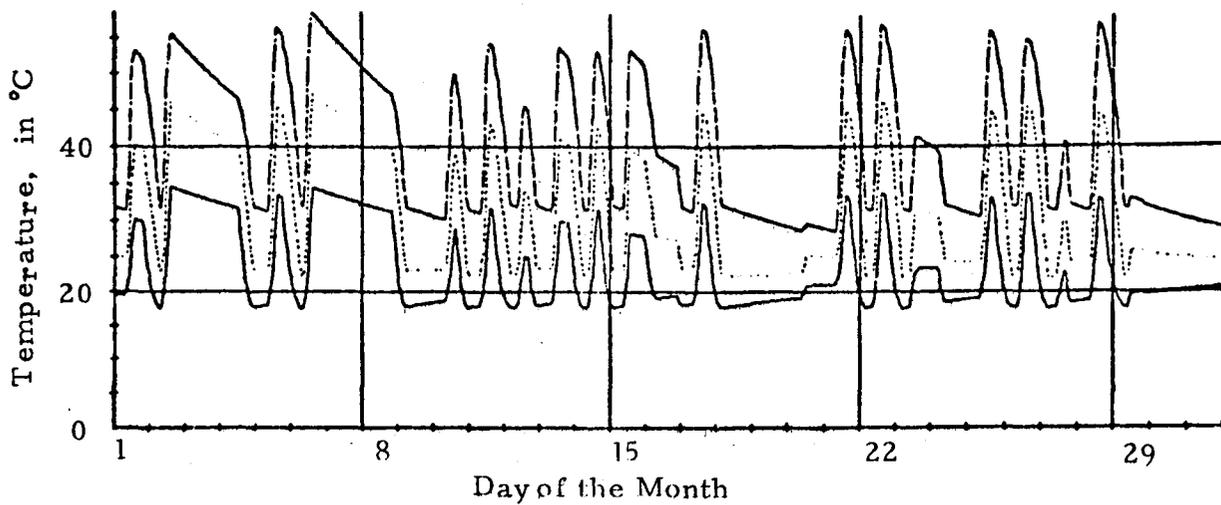


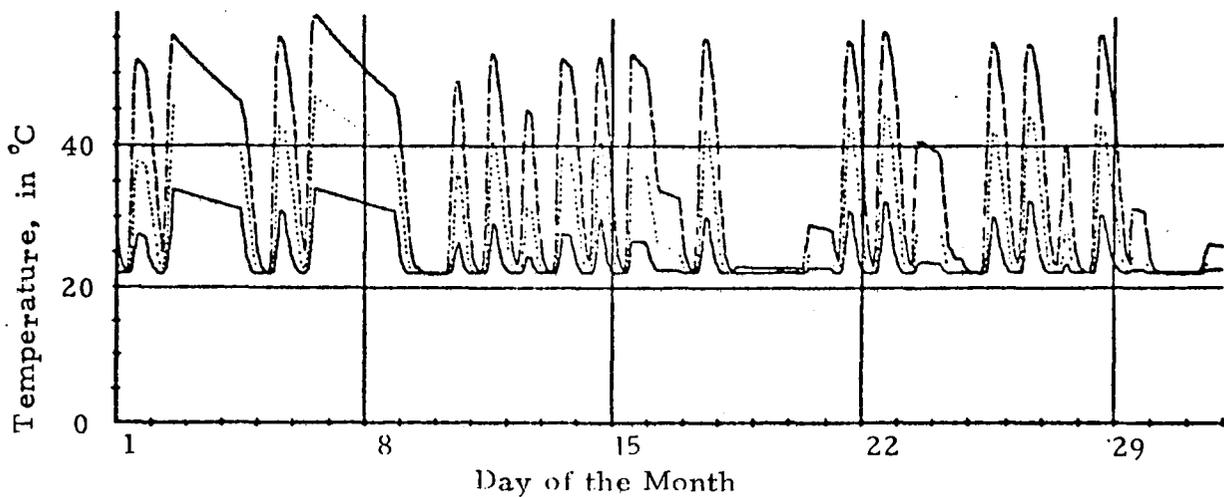
Figure 7. Solar Percent of House Heating Load



(a) Actual Test Site Data

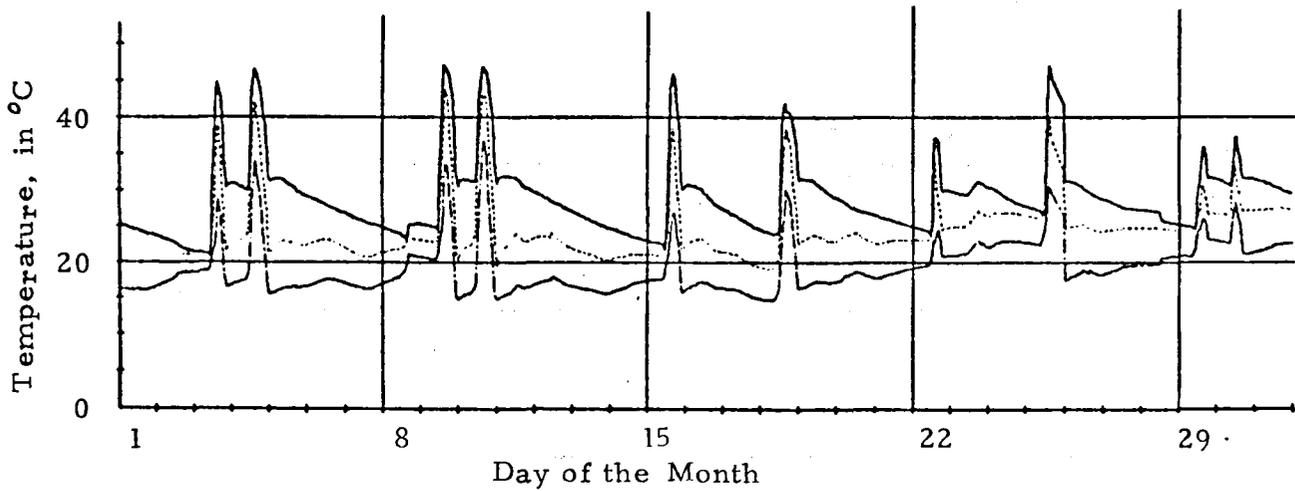


(b) HISPERS Prediction (Modified Control Logic)

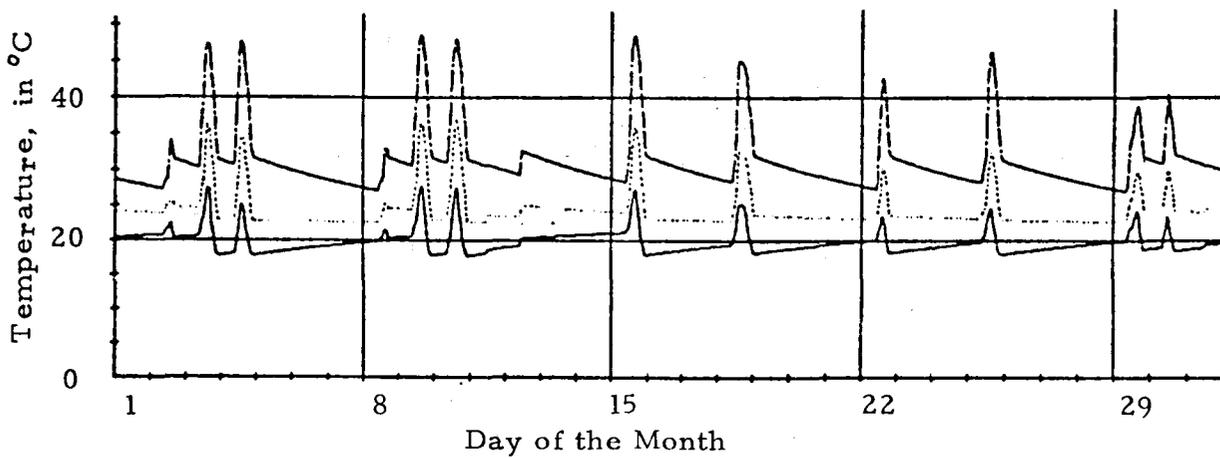


(c) HISPERS Prediction (Standard Control Logic)

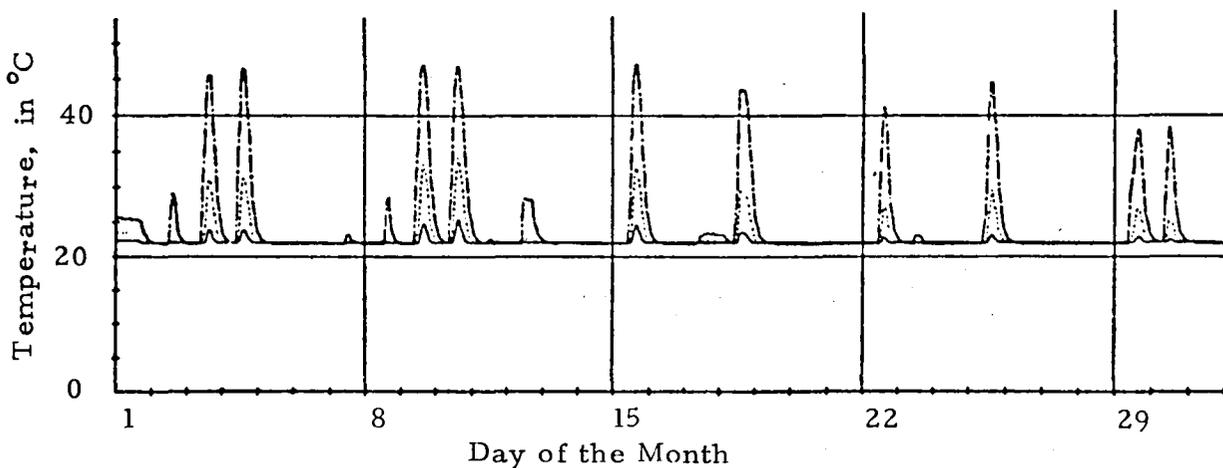
Figure 8. December Storage Temperatures (Top, Middle, & Bottom)



(a) Actual Test Site Data



(b) HISPER Prediction (Modified Control Logic)



(c) HISPER Prediction (Standard Control Logic)

Figure 9. January Storage Temperatures (Top, Middle, & Bottom)

The four changes to the "control logic" detailed in the HISPER Modification section were made with the following values:

1. Minimum temperature difference between collector plate and storage bottom: 7°C (13°F)
2. Minimum temperature difference between collector output temperature and preheat tank temperature: 5°C (9°F)
3. Minimum temperature from storage to house: 32°C (90°F)
4. House return temperature: 5°C (9°F) below room temperature.

The effects of this modified control logic can be seen in Figures 7, 8 and 9. The temperature difference criteria for turning on the collector and preheat tank heat exchanger did not have a great effect on the results. The minimum temperature to the house requirement reduced the solar percentages since the storage could no longer be "drained." Lowering the return temperature from the house allowed the collectors to operate more efficiently, collect more energy and thus deliver more to the house. The last two effects tend to counteract one another thus resulting in still a very good prediction of solar percentages on a monthly basis (Figure 7). Figures 8 and 9 show that these changes do result in a significant improvement in the simulation of the pebble bed storage behavior.

Hot Water Percentages:

Hot water loads were accurately predicted since almost all of the information required for the load calculation was read from the data tapes. The percentage of the domestic hot water was not predicted well by HISPER. HISPER predicted a larger solar fraction than actually existed.

The inaccurate prediction is a result of several factors. First, the water usage demand is so low in this application that energy losses are of a much greater magnitude than energy used in heating the water actually used. Therefore any error in heat loss parameters will have a large effect on the solar percentage.

The second effect is that HISPER only simulates heat lost from the preheat tank, not the standard hot water tank. Due to the low usage rate, most of the solar provided heat will be lost before it is used.

The solar percentages of the hot water load for the actual site and the HISPER predictions are shown in Table 1.

Table 1. Solar Percent of Hot Water Load

Month	Actual Site Percentage	HISPER Predicted Percentage
October, 1978	52	99
November, 1978	49	90
December, 1978	64	91
January, 1979	50	68
February, 1979	43	69
March, 1979	59	100
April, 1979	64	100

CONCLUSIONS AND RECOMENDATIONS

The major conclusion is that the HISPER solar simulation program seems to be a sufficiently accurate design tool. Errors of less than 10 percent were observed when comparing predicted results to the actual results. One caution, however, is that parameters such as infiltration and TLH (the temperature above which no heating is required) must be carefully selected to achieve good monthly predictions.

Although HISPER predictions have been validated against actual performance, several areas were discovered which need further attention:

1. The value of 7^oC for TLH (temperature above which heating is not required) seems lower than the site data indicates it should be (Figure 4). This should be investigated for this site and others.
2. The validation process should be carried out for several other operational test sites.
3. The four control options previously discussed should be added to HISPER. This will give HISPER greater flexibility but will not significantly increase computer time required for simulation.
4. The water main temperature was observed to vary from 8^oC to 25^oC over the period of a year. This variation could significantly affect the predicted hot water load since HISPER currently assumes a constant water main temperature.

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MARSHALL SPACE FLIGHT CENTER
THE UNIVERSITY OF ALABAMA

INCIPIENT FAILURE DETECTION (IFD) OF
ANTI-FRICTION BALL BEARINGS

Prepared By:	David Y. Wang, Ph.D.
Academic Rank:	Associate Professor
University and Department:	University of Wisconsin-Plattville Civil Engineering Department
NASA/MSFC: (Laboratory) (Division) (Branch)	Systems Dynamics Structural Dynamics Environmental Analysis
MSFC Counterpart:	Luke A. Schutzenhofer Jess H. Jones
Date:	August 1, 1980
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INCIPIENT FAILURE DETECTION (IFD) OF BALL BEARINGS

by

David Wang, Ph.D.
Associate Professor of Civil Engineering
University of Wisconsin - Platteville
Platteville, Wisconsin

ABSTRACT

Because of the immense noise dissipated during the operation of a large engine such as the SSME, the unique ball bearing signals are often buried by the overall signature of the machine. As a result, the most commonly used technique, pattern recognition (1)*, to detect and predict ball bearing failures is rendered useless.

In order to retrieve the buried bearing defect signals from the background noises, a survey of two improved techniques was conducted.

Signal averaging method (2) averages n^{**} signals over a time duration equaling the known period of the ball bearing defect frequency. With this method, the desired ball bearing defect signal will be enhanced over the noise by a factor of \sqrt{n} .

A second method is to perform a further frequency analysis on the logarithmic spectrum. The purpose of this operation is to detect and separate different families of harmonics and sidebands, thereby giving information about modulating frequencies.

The above mentioned alternate IFD methods were carried out with a Norland 3001 waveform and data system. The obtained results are then compared with the power spectral density (PSD) of the overall machine signals and conclusions are made accordingly.

* Number within the parentheses refers to the references at the back of the report.

** Number of samples taken.

ACKNOWLEDGEMENTS

The author wishes to acknowledge contributions made to this project by his counterparts at NASA, Dr. L. A. Schutzenhofer and Mr. J. H. Jones. The author also wishes to express his appreciation to those of the Environmental Analysis Branch who provided help in preparing this report.

NOMENCLATURE

$f(t)$	- complex signal
$S(t)$	- periodic signal
$n(t)$	- noise
S/N	- signal to noise ratio
t_k	- beginning time of the kth repetitive signal
t_s	- sample time interval
T	- period
σ	- rms of noise $n(t)$
f_o	- ball pass frequency over outer race
f_i	- ball pass frequency over inner race
f_s	- ball spin frequency
D_w	- ball diameter
d_m	- bearing pitch diameter
α	- contact angle
N	- number of rolling elements
R	- shaft rotative speed in Hz

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INTRODUCTION

Every operating machine will generate a combination of sound and vibration in the form of 'signatures' which are unique to the devices being evaluated. The ability to monitor, analyze, and use this signature to locate vibratory sources and predict future failures is the goal of an IFD system. Pattern recognition is the most commonly used technique in evaluating ball bearing conditions. It is performed by spectrum analyzing the output of an accelerometer signal over a specified frequency range that includes the ball pass frequencies. An abnormal increase in amplitude of this 'signature' over some duration of time indicates deterioration of ball bearings.

During the operation of a large machine such as the SSME, the bearing 'signature' in the form of electronic signals is often buried by the large background noise. Therefore, it is necessary to extract the bearing defect signals from the combined signal before an evaluation of the bearings can be made. This report describes a signal averaging or summation technique to extract the repetitive bearing defect signal from a complex signal with a high level background noise.

A cepstrum (3) type of signal analysis was also carried out. The advantage of cepstrum over a spectrum analysis is that it can separate families of periodic sidebands around some fundamental frequencies. The purpose of this cepstrum analysis is to cross check its results with that obtained using the signature averaging method. The possibility of using cepstrum in evaluating ball bearing conditions was also explored.

OBJECTIVE

The objective of this project is to seek a data reduction process in order to extract pertinent information from the time record of SSME during its tests. The extracted information is, then, used to predict condition of ball bearings installed in the high pressure oxygen turbopump (HPOTP) of the SSME. Of numerous available methods, signal averaging or summation analysis (3), (7), were chosen to be examined in this report.

SIGNATURE EXTRACTION

A signal with known periodicity, T , is buried deep in noise. To extract that signal by averaging or summation, the total signal plus noise is divided into m segments of duration T . Each segment is sampled at a number of points during every period. The samples are stored in a data storing device such as a magnetic tape which algebraically adds together all m samples.

A repetitive event of period T will always be added in the same sample location. The resulting sum, thus obtained, will be m times the average amplitude of that event. Events which do not repeat with the same period will result in a sum proportional to \sqrt{m} , therefore, the desired signal has been enhanced over the noise by a factor of \sqrt{m} .

The above described signal averaging process can be described as follows. Signal $f(t)$ is

$$f(t) = S(t) + n(t) \text{ -----} \quad (1)$$

the signal at k th repetition starts at time instant t_k , if taking $t_1 = 0$, is

$$\begin{aligned} f(t_k + it_s) &= S(t_k + it_s) + n(t_k + it_s) \\ &= S(it_s) + n(t_k + it_s) \text{ ----} \end{aligned} \quad (2)$$

Since noise, $n(t)$, is random and the m segments of samples are independent, summation of the m noises will yield a mean square value of $m\sigma^2$ and a rms value of $\sqrt{m}\sigma$. Hence, after adding the m segments of the signal, equation (2), which indicates values at the i th location of the signal $f(t)$ within the period T becomes

$$\begin{aligned} \sum_{k=1}^m f(t_k + it_s) &= mS(it_s) + \sum_{k=1}^m n(t_k + it_s) \\ &= mS(it_s) + \sqrt{m}\sigma \end{aligned}$$

Therefore, the signal to noise ratio, after m summations, is

$$\left(\frac{S}{N}\right)_m = \frac{mS}{\sqrt{m}\sigma} = \sqrt{m} \left(\frac{S}{N}\right)_m$$

A Norland 3001 waveform analyzer was used to carry out the signal extraction procedures described above. There are two ways to accomplish signal extraction. One is to stack segments of signal of given period T by shifting the signal segment of duration T , without any gap, m times successively. Another way is

to convolute the signal, $f(t)$, with a train of m units impulses at a time delay of the given period T . The latter procedure is described by the following operation.

$$a(t) = \int_{-\infty}^{\infty} f(t - \epsilon) \sum_{k=1}^m \delta(\epsilon - kT) d\epsilon$$

$$= \sum_{k=1}^m f(t - kT)$$

CASE HISTORY AND DATA COLLECTION

Failures of bearings within the HPOTP were experienced during tests of the SSME. Spallings, damage cuts, pittings, and discoloration appear on both inner and outer races, rolling elements, and cages. The most serious failure seems to occur at bearings closest to the turbine of HPOTP. It was suspected from evidences of damages, that these failures were caused by excessive loading in the axial direction of the turbopump during the starting stage of the test. In order to be able to predict the oncoming bearing as well as other types of failures, data were collected from several strategically located accelerometers. These signals were usually low pass filtered at 10 KHz or under before storing on magnetic tapes for later analysis. Because of the low pass filtering of data, possibility of adopting various IFD techniques in the higher frequency range (30 KHz and higher) as diagnostic instruments is precluded.

The collected data in the form of time record is time averaged or summed and then transformed to the frequency domain as power spectral density (PSD) diagrams. This allows one to relate specific periodic occurrence of the bearing to specific frequencies in the data.

In order to analyze the obtained results, ball pass frequencies over a defect were computed according to the following equations.

$$f_o = \frac{N}{2}R(1 + \frac{D_w}{d_m}\cos\alpha)$$

$$f_i = \frac{N}{2}R(1 - \frac{D_w}{d_m}\cos\alpha)$$

$$f_s = R\frac{d_m}{D_w}(1 - (\frac{D_w}{d_m})^2 \cos^2\alpha)$$

For the given bearing configuration, various ball pass frequencies and other relevant data (at 100% SSME power) are listed as follows:

$$f_o = 2570 \text{ Hz}, f_i = 3530 \text{ Hz}, f_s = 2920 \text{ Hz}$$

$$N = 13, \quad R = 469 \text{ Hz}$$

Static capacity of the bearing is 3050 lbs.

DATA ANALYSIS

Results obtained from the test carried out in the Spring of 1980 are presented in figure 1 through figure 11. Figure 1 is a collection of three PSD's for the initial, middle, and final stage of the bearing life span. Narrow band spikes at shaft rotative speed and turbine blade frequencies appear in the diagram at 469 Hz, 1836 Hz and 3672 Hz, as expected. Note the growth of the narrow band spike at 2570 Hz region which indicates a possible fault on the outer race. Defects on the inner race and rolling elements are not obvious in this particular figure.

Figure 5 through figure 10 are the so-called zoom PSD's at the same time instants as those in figures 1 through figure 4. It is more desirable than the normal PSD's because it provides a considerably finer resolution over the portion of the spectrum that we are interested in. Again, possible defects on the outer race appear on these spectra in the neighborhood of 2600 Hz region. Finally, a useful technique in detecting the deterioration of bearings with respect to time is presented in the form of a three dimensional PSD with time as the third parameter.

CONCLUSIONS AND RECOMMENDATIONS

A signal averaging or summation process has been described that is used to extract the bearing defect signals from the time record of SSME. It appears that this procedure can also be applied to the zoom and three-dimensional PSD for finer resolution and better presentation of the deterioration history of ball bearings. An electronic equipment of black box type based on this principle could be constructed and adopted as part of an "on-line" IFD system. One drawback in applying this technique is that the period, T , of a repetitive signal cannot be absolutely time-locked in an analog to digital converter because of the limitation of the sampling time interval. The result of this deficiency is the tendency of the summed time signal being smeared gradually. Signal to noise gain will be decreased accordingly.

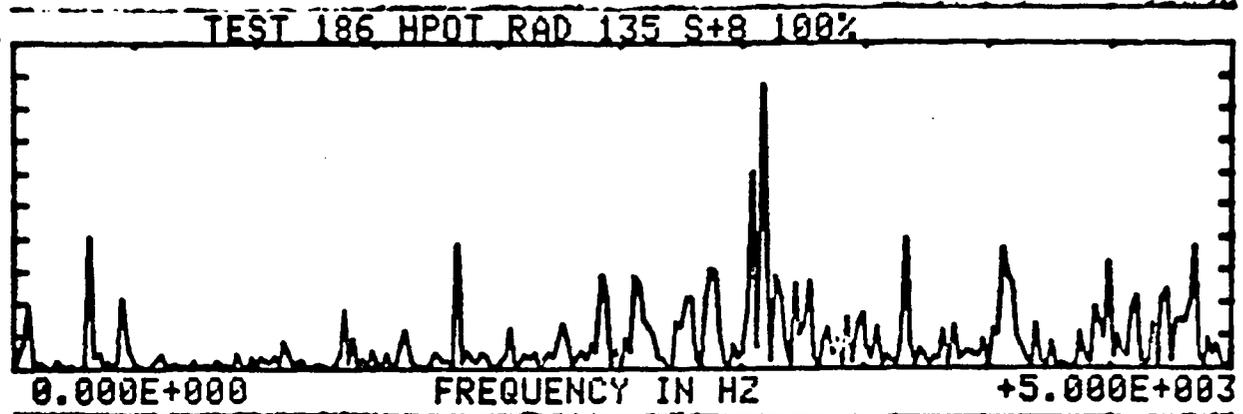
Even though other techniques such as "acoustic emission" (10) and "shock pulse" have been proposed, they have not been proven successful. One alternative method in ball bearing defects detection is to utilize the bearing high frequencies (13), (14), especially for large machines. Natural frequencies of ball bearings are usually far beyond that of the noise level, therefore, less noise interference to the bearing signal. In order to adopt this method of approach, real time signal should be collected without any low pass filtering.

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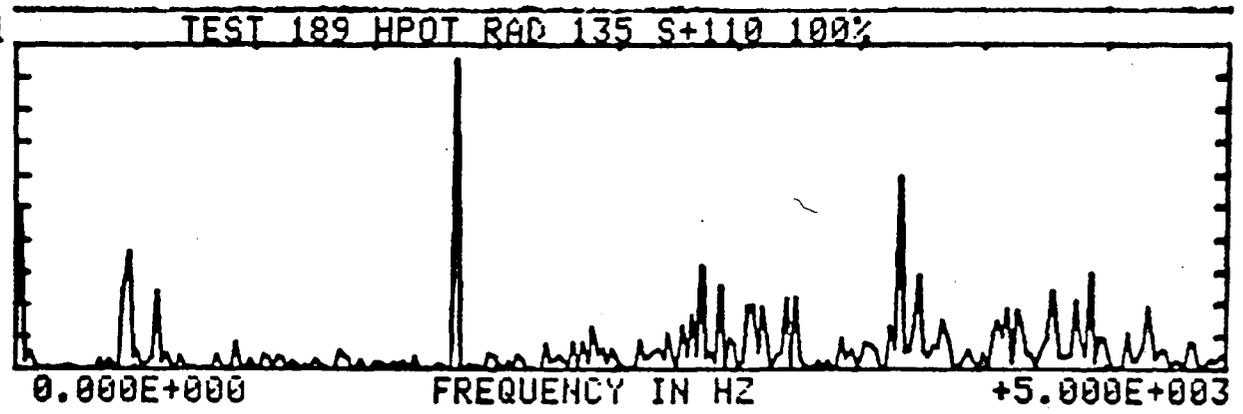
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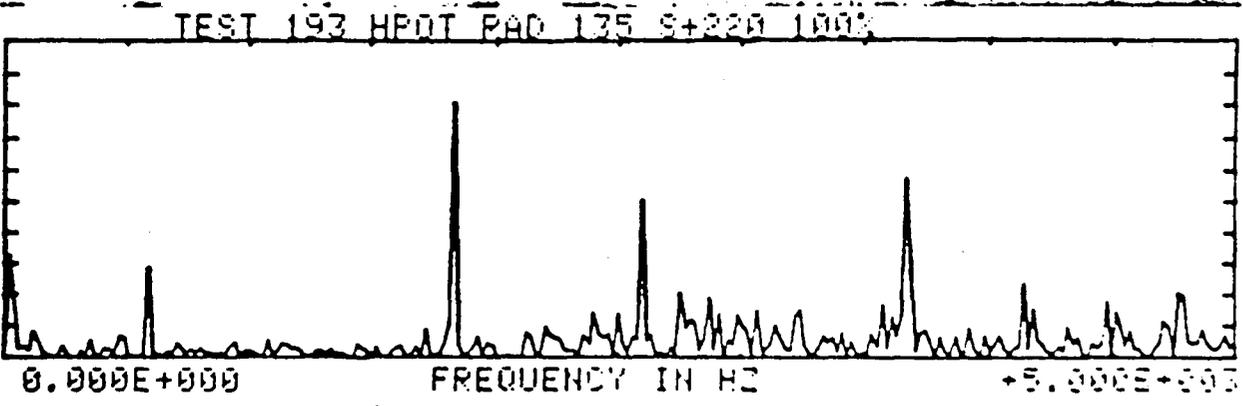
+2.000E-001
 3886 0.17 G
 3847 0.12
 312 0.08 S
 3672 0.08
 COMPOSITE Q
 7.37 /
 3.54 H
 BW=19.531 Z
 8-9 0.0



+4.000E-001
 1836 0.38 G
 3672 0.24
 20 0.20 S
 469 0.14
 COMPOSITE Q
 8.91 /
 4.77 H
 BW=19.531 Z
 8-9 0.0



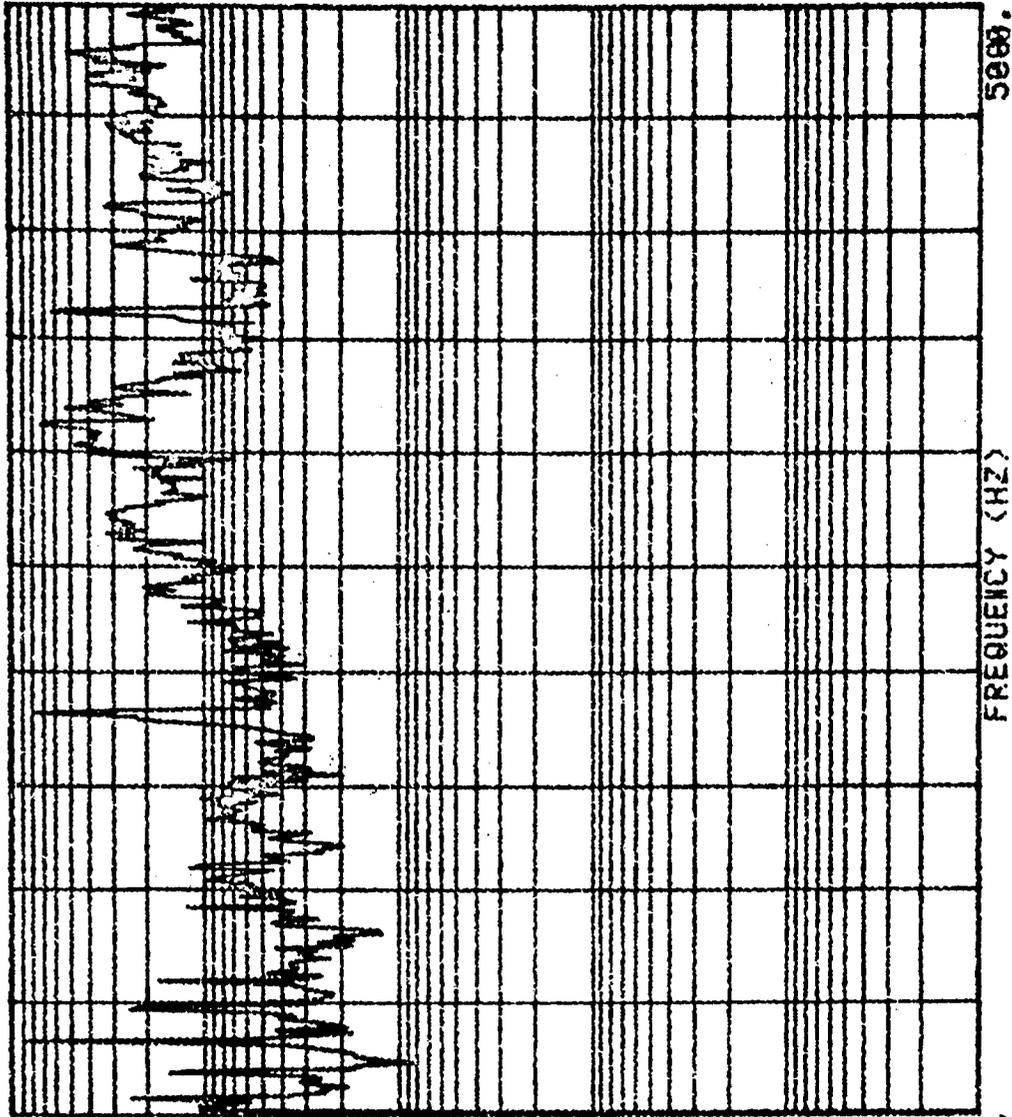
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 1836 0.49
 3672 0.29
 2598 ✓ 0.25
 20 0.16
 COMPOSITE
 10.07
 5.23
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 8-9 0.0



NARROW BAND FREQUENCY ANALYSIS - INITIAL, MIDDLE, AND FINAL STAGES

Figure 1

186 HPOT RAD 175 ACC S+ RUN



+1.000E-006
313 0.0093
1225 0.0778
3125 0.0725
3138 0.0693

F S R / H Z

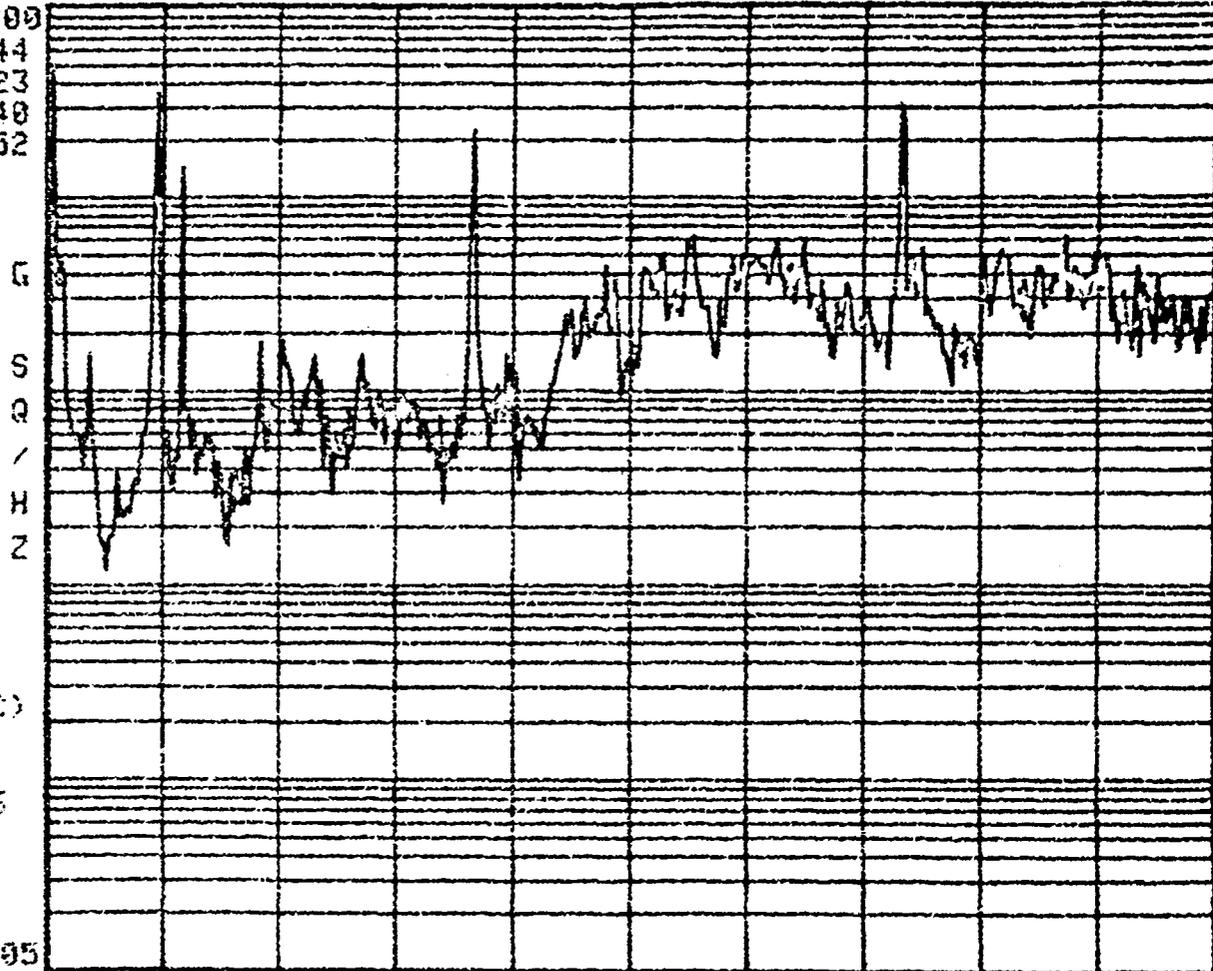
SLICE TIME (SEC)
1.28
BW=12.5
NO. SPECIRA =16
COMPOSITE
8.09735

+1.000E-006
8-9 6.

PSD (INITIAL STAGE)

189 HPOT RAD 135 ACC S+ 110

	+1.000E+000
13	0.4644
463	0.3523
3663	0.3148
3675	0.2552



SLICE TIME (SEC)
1.28
BW=12.5
NO. SPECTRA =16
COMPOSITE
11.19117

2-9 +1.000E-095
0.

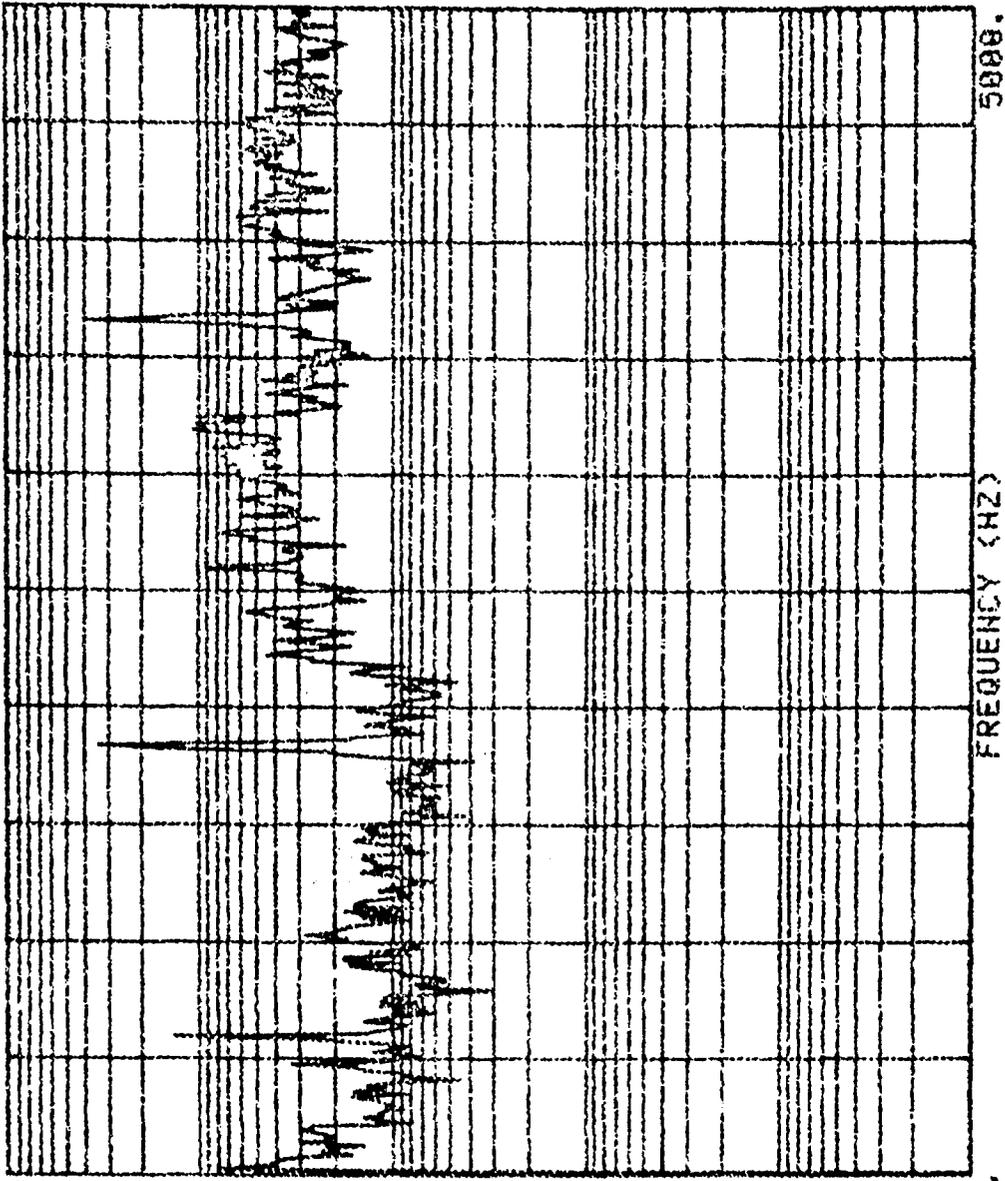
FREQUENCY (HZ)

5000.

PSD (MIDDLE STAGE)

XX-16

193 HPDT RAD 135 ACC S+ 220



3653
1838
3575
1825

+1.000E+000
0.3777
0.3367
0.1800
0.1611

G S Q / H Z

SLICE TIME (SEC)
1.28
SN=12.5
NO. SPECTRA =16
COMPOSITE
12.00045

+1.000E-005
8-9

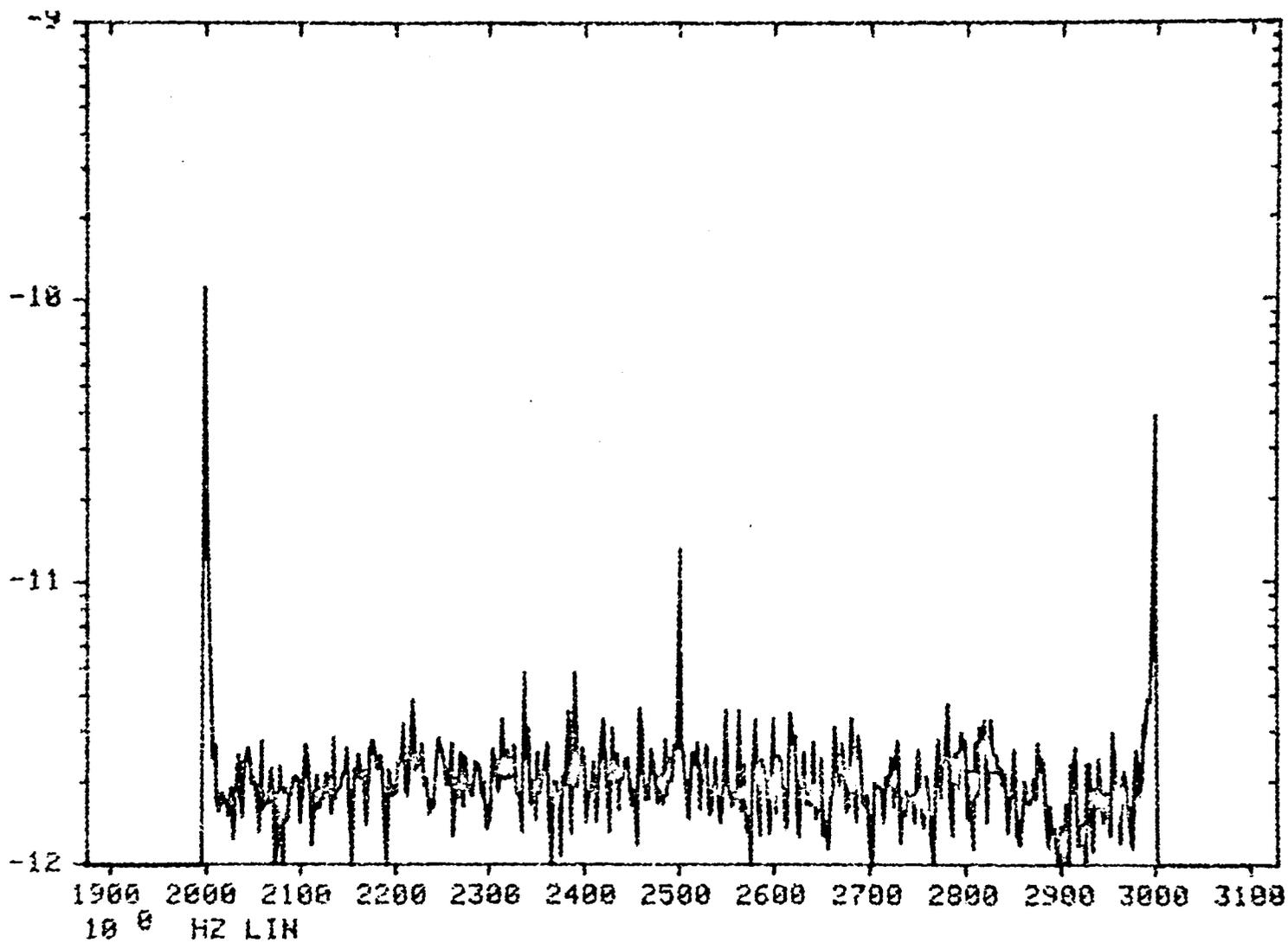
FREQUENCY (HZ)

5000.

PSD (FINAL STAGE)

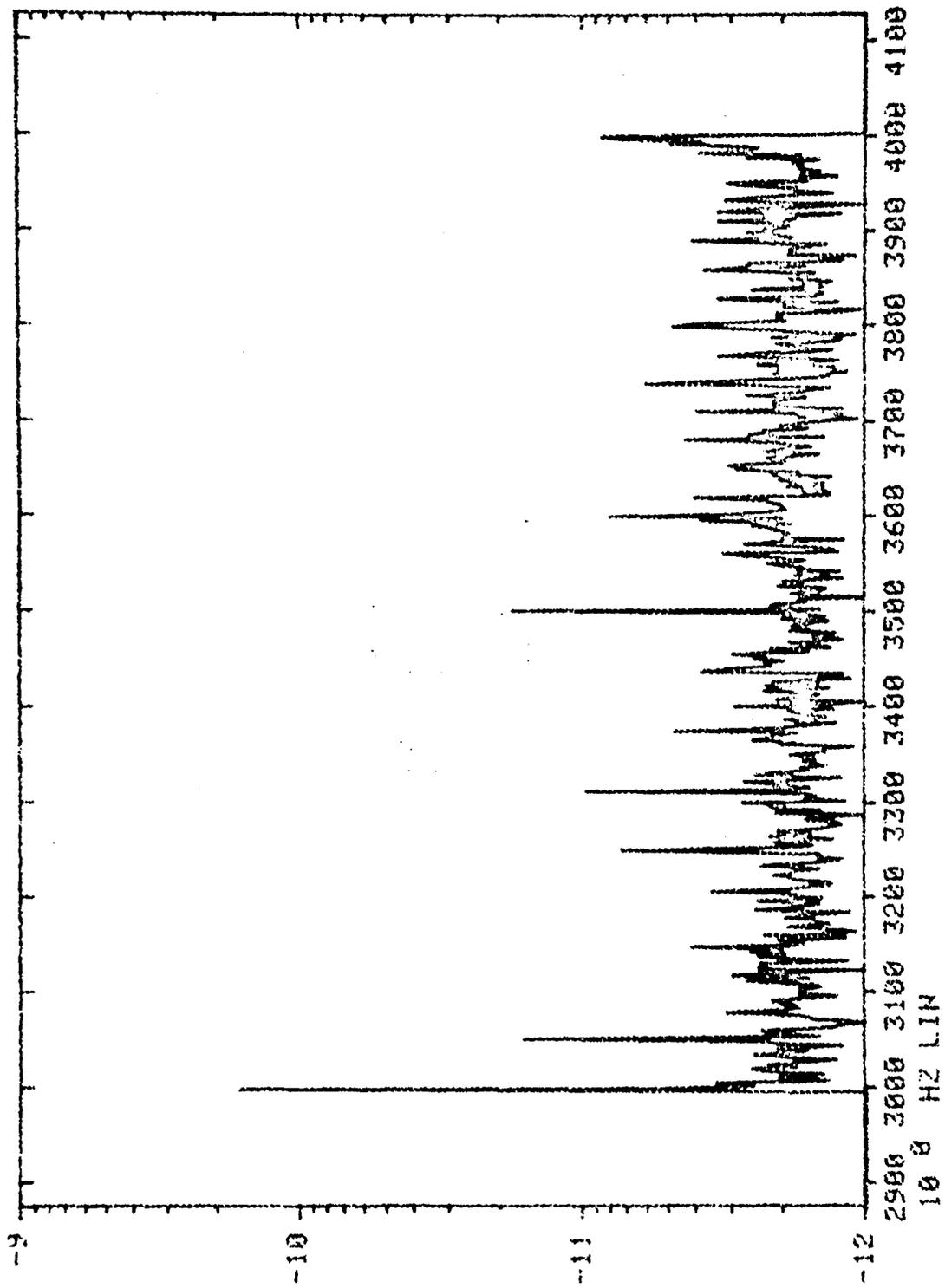
189

XX-18

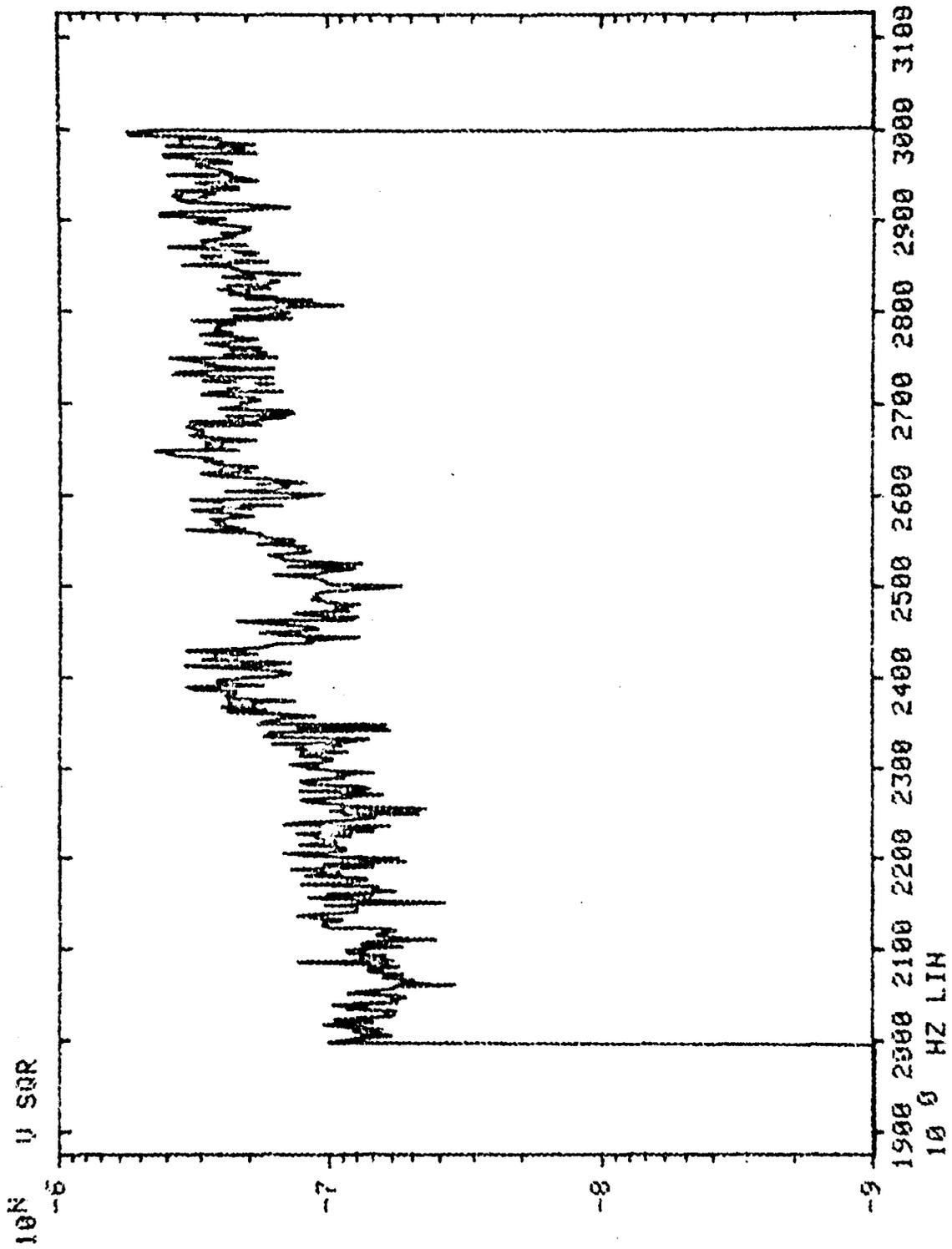


ZOOM (INITIAL STAGE)

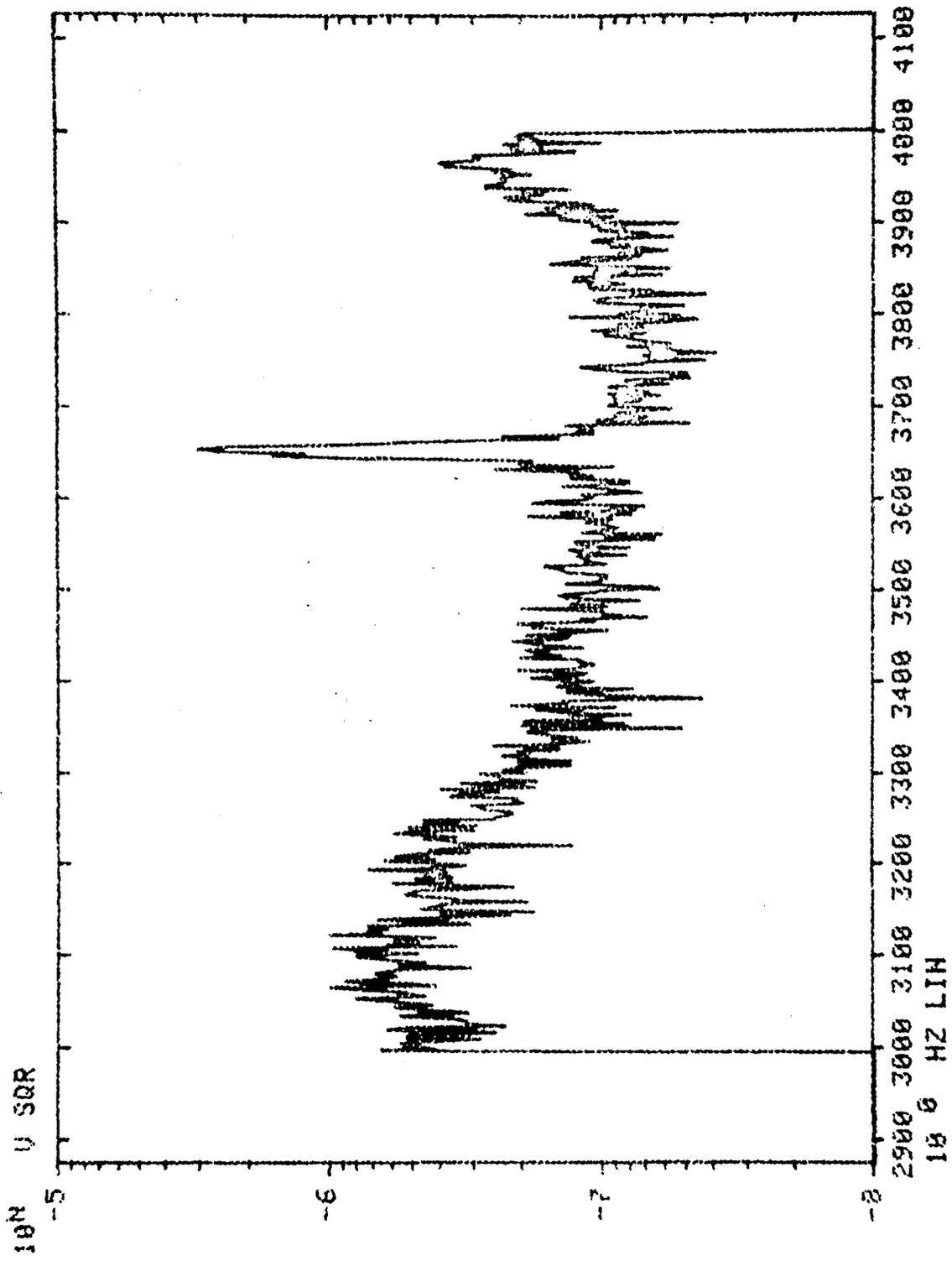
189



ZOOM (INITIAL STAGE)

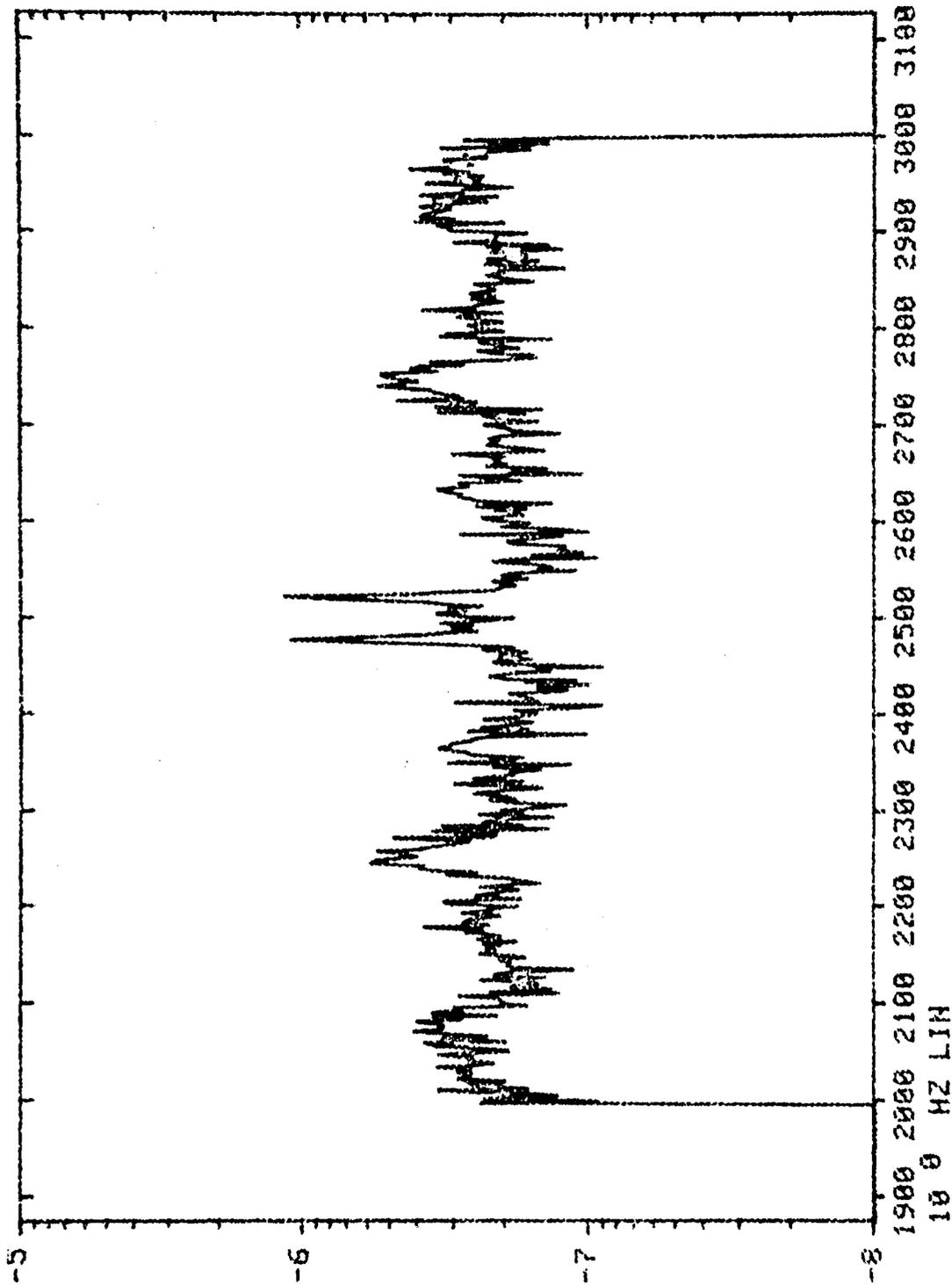


ZOOM (MIDDLE STAGE)

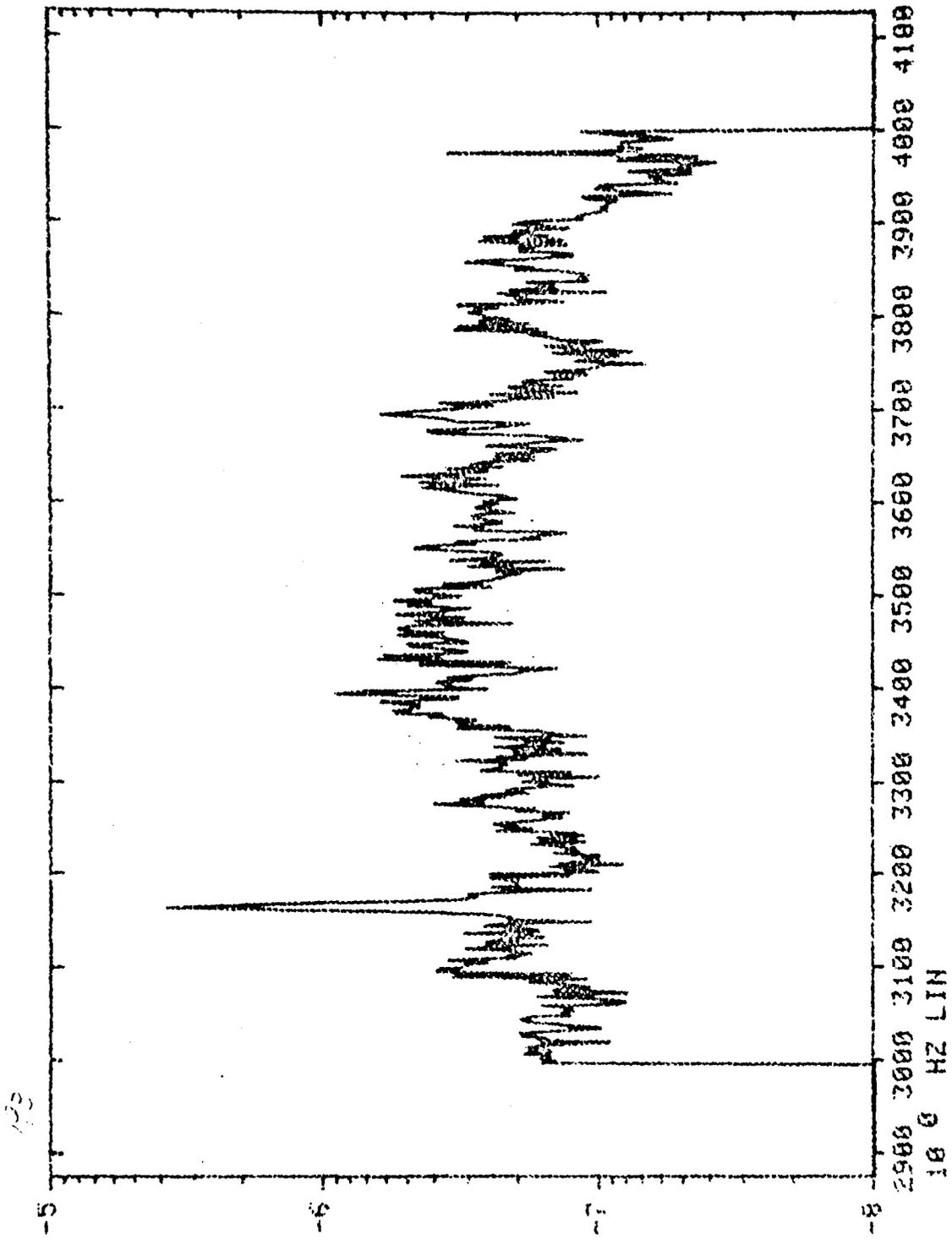


ZOOM (MIDDLE STAGE)

193, St220



ZOOM (FINAL STAGE)



ZOOM (FINAL STAGE)

Initial Plot Time:

18 HOURS
6 MINS
46.0000 SECS

TEST #189

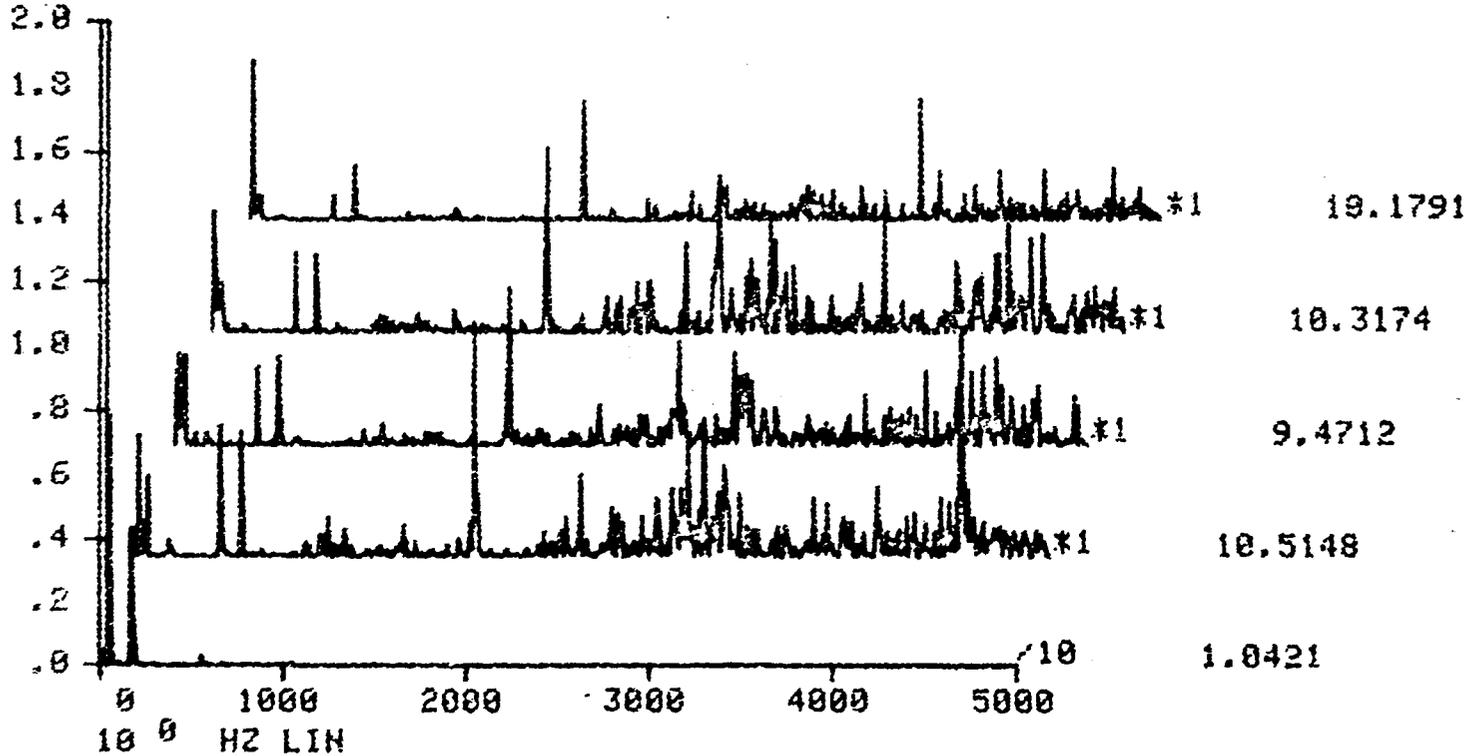
HPOT RAD 135

Final Plot Time:

18 HOURS
7 MINS
18.0000 SECS

Plot Increment: 8.0000 SECS

10⁰ (G's or PSI) SQR per Hz



XX-24

THREE DIMENSIONAL PSD

1980

NASA/ASEE FACULTY RESEARCH FELLOWSHIP PROGRAM

MARSHALL SPACE FLIGHT CENTER
THE UNIVERSITY OF ALABAMA

PULSED EDDY CURRENT TESTING

Prepared By: Gary L. Workman, Ph.D.
Academic Rank: Associate Professor
University and Department: Athens State College
Department of Scientific Technologies
NASA/MSFC:
(Laboratory) Materials and Processes
(Division) Engineering Physics
(Branch) Nondestructive Testing
MSFC Counterpart: J. M. Knadler, III
Date: August 1, 1980
Contract No.: NGT-01-002-099
(University of Alabama)

PULSED EDDY CURRENT TESTING

BY

Gary L. Workman, PH.D.
Associate Professor of Scientific Technologies
Athens State College
Athens, Alabama 35611

ABSTRACT

The traditional nondestructive testing technique used for inspecting welds in large space structures, such as the "External Tank", has been radiographic testing, in conjunction with surface inspection methods, such as dye penetrant testing. In order to improve upon the economics and reliability of these critical inspections, new approaches to the test methods are under consideration. One approach is to replace the very expensive radiographic inspection with an automated eddy current testing technique. A description of pulsed eddy current and its relationship to multi-frequency techniques is presented here, as well as some preliminary results obtained from observing pulsed waveforms with apparatus and algorithms currently in use for ultrasonic testing of welds. It can be shown the pulsed eddy current techniques can provide similar results.

Eddy current testing has primarily been used for sorting of materials due to differences in conductivities, detecting differences in surface due to corrosion on cracks, and in some instances to detect flaws below the surface. Manual operation of the test proves to be very unreliable due to such test parameters as lift off, non-uniform conductivity, depth of penetration, surface treatment, and complex geometries of the test object. The use of a pulsed eddy current techniques can eliminate some of the noncritical parameters affecting the eddy current signals and facilitate in the detection of critical parameter such as flaws, subsurface voids, and corrosion.

Acknowledgements

I wish to acknowledge Mr. John M. Knalder, III, for allowing me to conduct the research this summer and Mr. Ken Woodis for assisting me in setting up the computer interfaces. Also Mr. Ben Bankston and Mr. Claude Williamon, for implementing experimental fixtures. I also thank Miss Pearline Stone for typing the report.

Introduction

The nondestructive inspection of large space structures requires that a large part of the inspection procedure be automated in order to reduce human errors and to reduce the amount of time and money spent in this critical phase of production. Since a large number of the procedures used for inspecting the External Tank are concerned with determining flaws in welds, there is a need to develop an inspection technique, which can be automated, to determine flaws in welds and structures with complex geometries. The technique, whereby one generates eddy current in a metallic material and observes the changes in the circuit parameters due to material differences, has been chosen as one possible approach. The principles of eddy current testing has been treated in a number of publications (1-3). Of particular interest to the researcher is the book by Libby¹ and the work of Dodd³.

Eddy currents are generated in metallic specimens by alternating currents in coils in close proximity to the material. Some of the properties associated with eddy currents are:

1. They constitute a closed circuit; hence, in the material, they usually have a circular configuration
2. They penetrate into the material with decreasing

amplitude away from the surface.

3. Variations in part distances introduces electrical phase differences.
4. The magnitude of eddy current flow is proportional to the current in the test coil.
5. Both the electrical conductivity and the magnetic permeability affect the eddy current flow in the material.
6. Discontinuities which affect the eddy current flow are easily detected as the eddy current redistribute themselves around the discontinuity.

These general properties have enabled eddy current techniques to be useful for a number of applications. Some of the most notable are:

1. Measurements of electrical conductivity
2. Measurements of magnetic permeability
3. Sorting of metallic materials
4. Measurements of distances between components
5. Thickness determinations
6. Detections of voids on discontinuities in metallic materials

A number of commercial eddy current instruments are available for these applications. The reliability and sensitivity of each test depends upon many parameters, for instance:

1. Coil-to-conductor spacing and geometry
2. Choice of frequency for a particular conductor
3. Orientation of defects to coil
4. Sample geometry
5. Coil characteristics

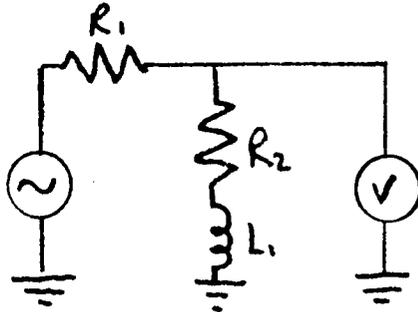
Such a large number of parameters affecting an eddy current measurement accounts for the range and flexibility of the technique, as well as its pitfalls. For instance, in the determination of flaws in welds, one has to keep all other parameters constant so that only the interaction of the eddy currents with a flaw would be observed as a signal. Even with a judicious laboratory arrangement, this is very difficult to accomplish. When testing complex geometries, in a production environment, it is practically impossible to maintain that many parameters constant. Also, most materials exhibit minute changes in composition which can affect the eddy current measurements and this can be misinterpreted as a flaw. Since it can be so difficult to make a single eddy current measurement and interpret the results correctly, many researchers in the field are developing methods which take a number of measurements and solve simultaneously for different parameters which can change. The use of computers in both hardware control and data acquisition and analysis allows these approaches to become easily automated.

Objectives

The primary objectives of this work is to determine the feasibility of using pulsed eddy current methods for nondestructive testing. As described in the report there are many ways of handling the data received from the eddy current test. Our approach has been to pulse an eddy current probe, use present FFT data processing and determine if the information contained in the frequency domain does allow one to solve for the properties of the material under inspection.

Multiparameter Methods

The electrical quantities of interest in an eddy current measurement are usually a voltage or current and its phase relationship to the driving voltage or current. Eddy current circuits are most easily represented as simple RLC circuits and their signal responses represented as impedance of the material being inspected. Hence impedance plane plots are most often used to illustrate and differentiate between the different parameters affecting the eddy current measurement. For example, if we consider a simple coil and driver arrangement such as



then if $V_0 = V_0 \cos \omega t$ (i.e. a sinusoidal waveform) then

$$V = V_0 \left(\frac{Z}{R+Z} \right) = V_0 \left| \frac{Z}{R+Z} \right|$$
$$= V_0 \sqrt{\frac{(R_2 + 2\pi f L_1)^2}{(R_1 + R_2 + 2\pi f L_1)^2}}$$

Notice that the measurement of a single voltage can determine only a single parameter or the combined effect of several parameters. In order to differentiate between the

parameters in an eddy current measurement, one has to synthesize an excitation signal which then can be decomposed into frequencies affected by the desired parameters. One approach is to use multifrequency excitation from which we can extract a voltage and a phase value for each frequency. For example, if we use two frequencies, then we have four measurements for each test and

$$\begin{aligned}
 V_1 &= P_{11} A_1 + P_{12} A_2 + P_{13} A_3 + P_{14} A_4 \\
 V_2 &= P_{21} A_1 + P_{22} A_2 + P_{23} A_3 + P_{24} A_4 \\
 V_3 &= P_{31} A_1 + P_{32} A_2 + P_{33} A_3 + P_{34} A_4 \\
 V_4 &= P_{41} A_1 + P_{42} A_2 + P_{43} A_3 + P_{44} A_4
 \end{aligned}$$

where A_1, A_2, A_3, A_4 , represent the material properties as coefficients in response to the eddy current source and each P_{ij} represents the magnitude of that effect in each measurement. V_1 through V_4 represents either a voltage or phase measurement. This procedure has been developed and successfully demonstrated by Dr. Dodd and his Union Carbide Corp. in Oak Ridge(4,5). Although the parameters A_1 through A_4 appear to be linearly independent, experimentally it can often be difficult to determine unique solutions and thus to identify such effects as lift-off, voids, or corrosion. However, their approach was to synthesize known types of defects, liftoff's, etc into the laboratory calibration. Then, during an inspection procedure, their system uses a pattern recognition approach to go back and fit the experimental data

to the calibration data in order to determine what responses are contained in each measurement. To date, this approach has worked very well in the inspection of tubing used in nuclear reactor heat exchangers.

The frequencies chosen for this approach use a high frequency signal for surface effects and a lower frequency (say 1/10) for multilayer identification, and even another lower frequency (say 1/100) for detection of voids farther away from the surface.

Pulsed Eddy Current Techniques

The multifrequency technique provides a workable solution to the eddy current testing problem. The major drawback to the method is that it is expensive. A multifrequency head using several coils will prove to be extremely costly. In addition to the data acquisition circuit, precise frequency filters are required to maintain a reliable signal-to-noise ratio for each phase and amplitude measurement. A promising technique, which should prove to be less expensive, is to use a pulsed excitation rather than the continuous alternating current as in the previous work. In reality pulsed eddy current and multifrequency eddy current provide similar information. For instance, a short duration pulse can be transformed into the frequency domain and the resulting wave form consists of a large number of frequencies. Hence, we can also produce multifrequency excitation using a pulse with a finite time period.

Pulsed eddy current data can be obtained in two different ways, although the two approaches are essentially equivalent. The pulsed eddy current experiment can be performed in a manner similar to the multifrequency experiment discussed above, the only deviation is using a finite pulse instead of several continuous wave frequencies. The interaction of the coil with the eddy currents is still dependent on the inherent frequency of interaction. Hence, one would still measure amplitudes and phases of the resulting signals and perform the data reduction in exactly the same way as before. The two methods are equivalent in that same sense.

The other approach one might take in using pulsed eddy currents is to actually transform the time domain data into the frequency domain and use the multifrequency analytical approach with these transformed values. Our approach this summer has been to obtain data in this mode in order to determine the feasibility of the approach. A number of algorithms currently in use for ultrasonic imaging applications are applicable, as well as a hardwired Fast Fourier Transform Analyzer (6). Hence we felt that this approach might proceed the quickest for a summer's work.

Experimental Results

All of the measurements taken in this work were obtained with a Microtek model 101 ultrasonic pulsing system driving a Kervonics reflection eddy current coil. The pick-up coil signals were digitized by a 8-bit A/D and stored in a Biomation Transient Analyzer.

Model #8100. The data was then transferred to a PDP11/45 computer and analyzed via software generated by contractor personnel. An example of the data obtained here is shown in Figure 1-4. The pick-up signal itself is shown in Figure 1 and its FFT is shown in Figure 2. These observations were obtained statically with a Kervonics 40A probe (pick-up coil radius=0.7 cm) on aluminum.

Lift-off simulation is shown in Figure 3 and the resulting waveform when over a 20 mil surface defect is shown in Figure 4.

Note the changes which occur for the frequency domain data, particularly at 0.5, at 1.3, and at 3 MHz. In all three cases the pick-up signal changes somewhat but not as quantitative as given in the frequency data. Obviously then the analysis of a three frequency probe at 0.5, 1.3, and 3 MHz should provide the same type of information, but would require three frequencies with separate pick-up coils. Since the scanning mechanism, which was expected to be used in this work, never did arrive; all the measurements were made using a specially improvised mount for the probe. Since no motion occurred during the measurement, all tests were static.

Conclusions and Recommendations

The pulsed eddy current technique can differentiate between various materials properties using FFT analysis. Since we were not able to obtain data while scanning over a specimen, that should be the next objective in this project. Also, in view of the information published by Dr. Dodd and his associates, the following recommendations can be made.

1. Improve the resolution of the A/D converter presently being used by replacing it with at least a 12-bit A/D converter.
2. Use waveform synthesis techniques to generate a pulse which contains those frequencies of interest for a particular inspection. A micro-processor generated pulse can do this.
3. Implement the pattern recognition algorithms at Marshall Space Flight Center. Upon request most of his software is available.

If pulsed eddy current is to be developed at MSFC, a full time effort should be provided. The nondestructive testing requirements for large spacecraft will be more easily implemented using either a pulsed or multi-frequency method.

.01 μ SEC.
0000 DELAY

R A W D A T A

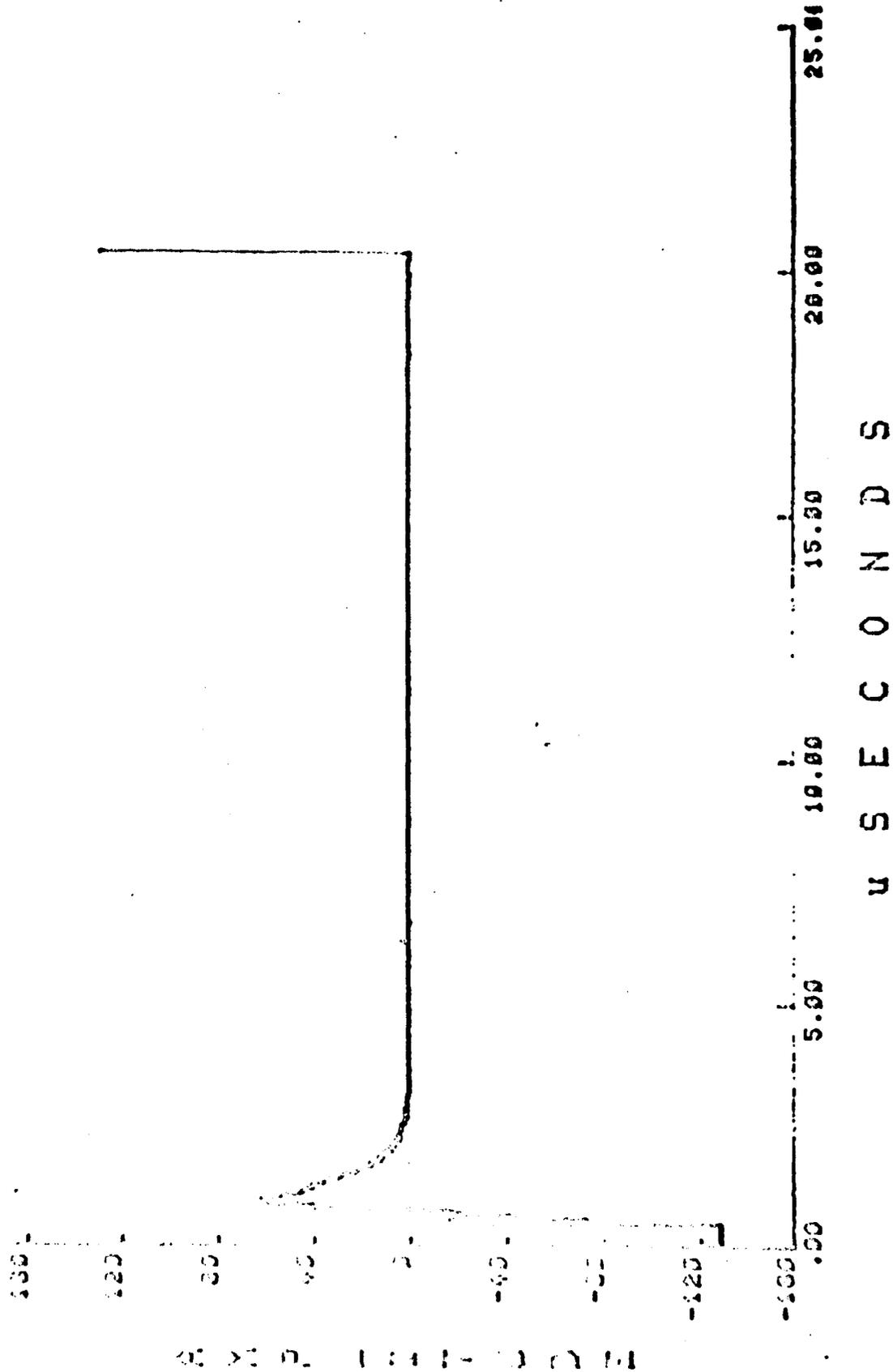
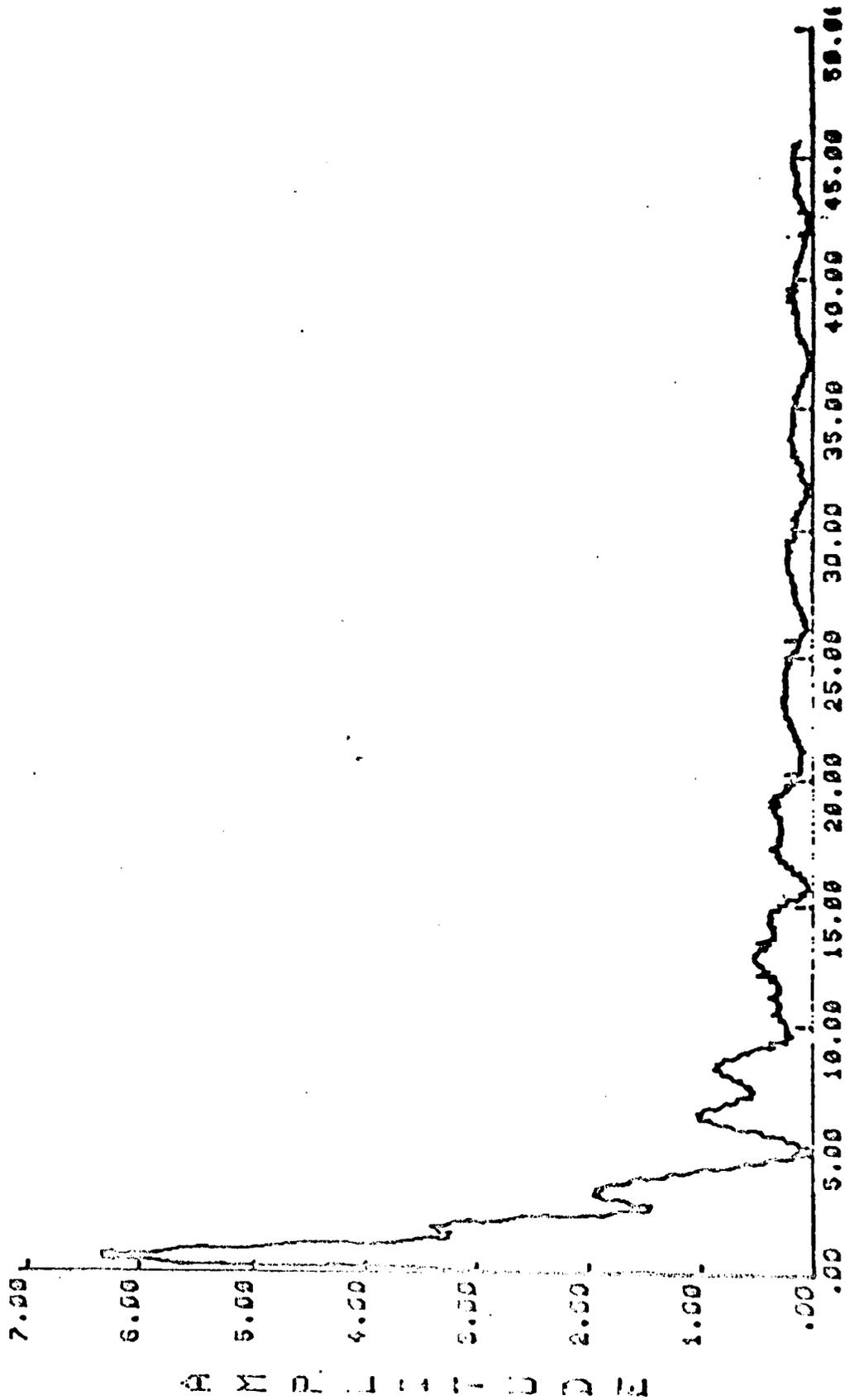


Figure 1: Signal from Pick-Up Coil

.01 USEC.
0000 DELAY

M A G N I T U D E F F T



F R E Q U E N C Y M H Z

Figure 2: FFT Signal for Aluminum

.01 USEC.
8000 DELAY

M A G N I T U D E F F T

XXI-13

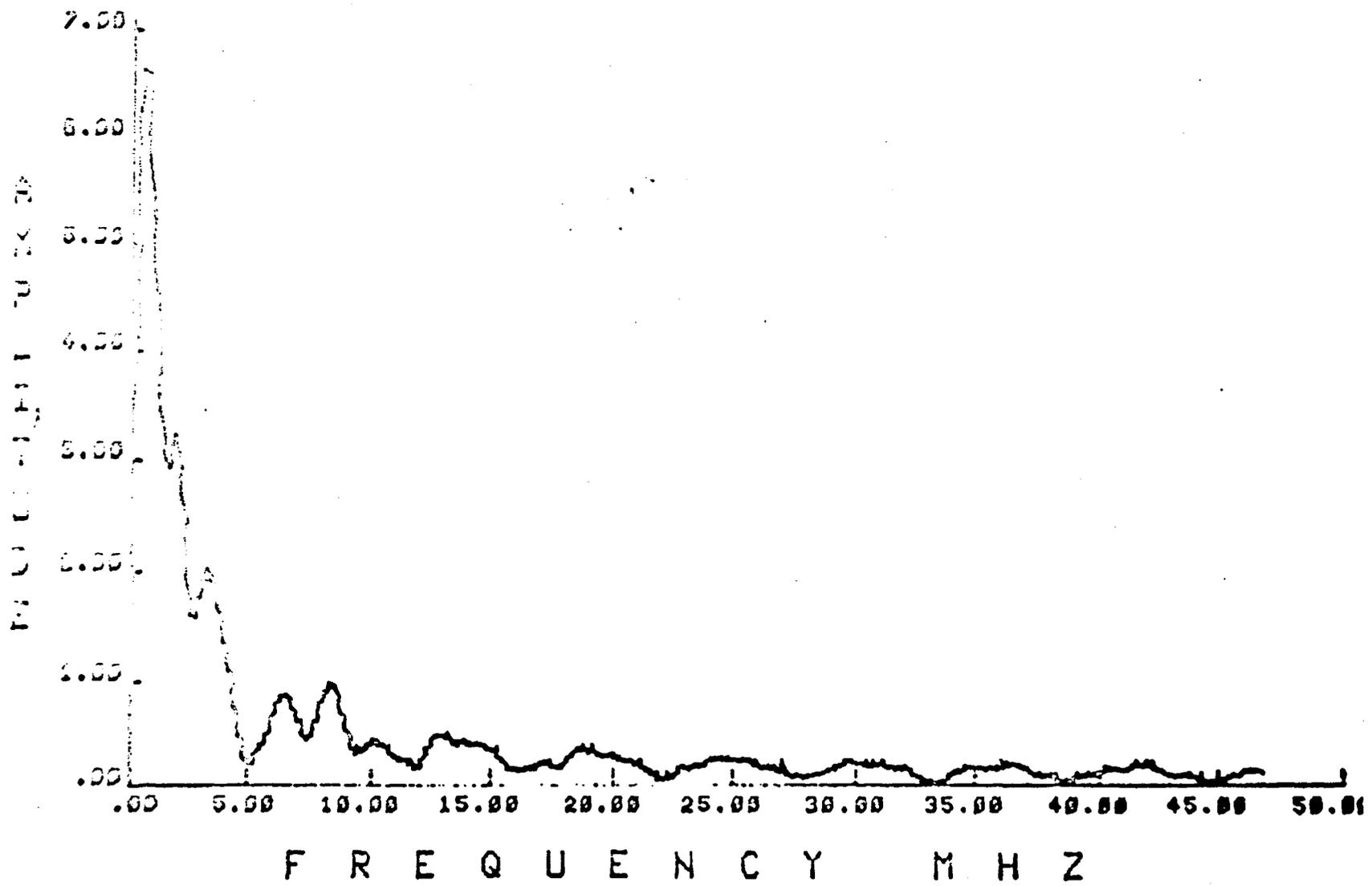
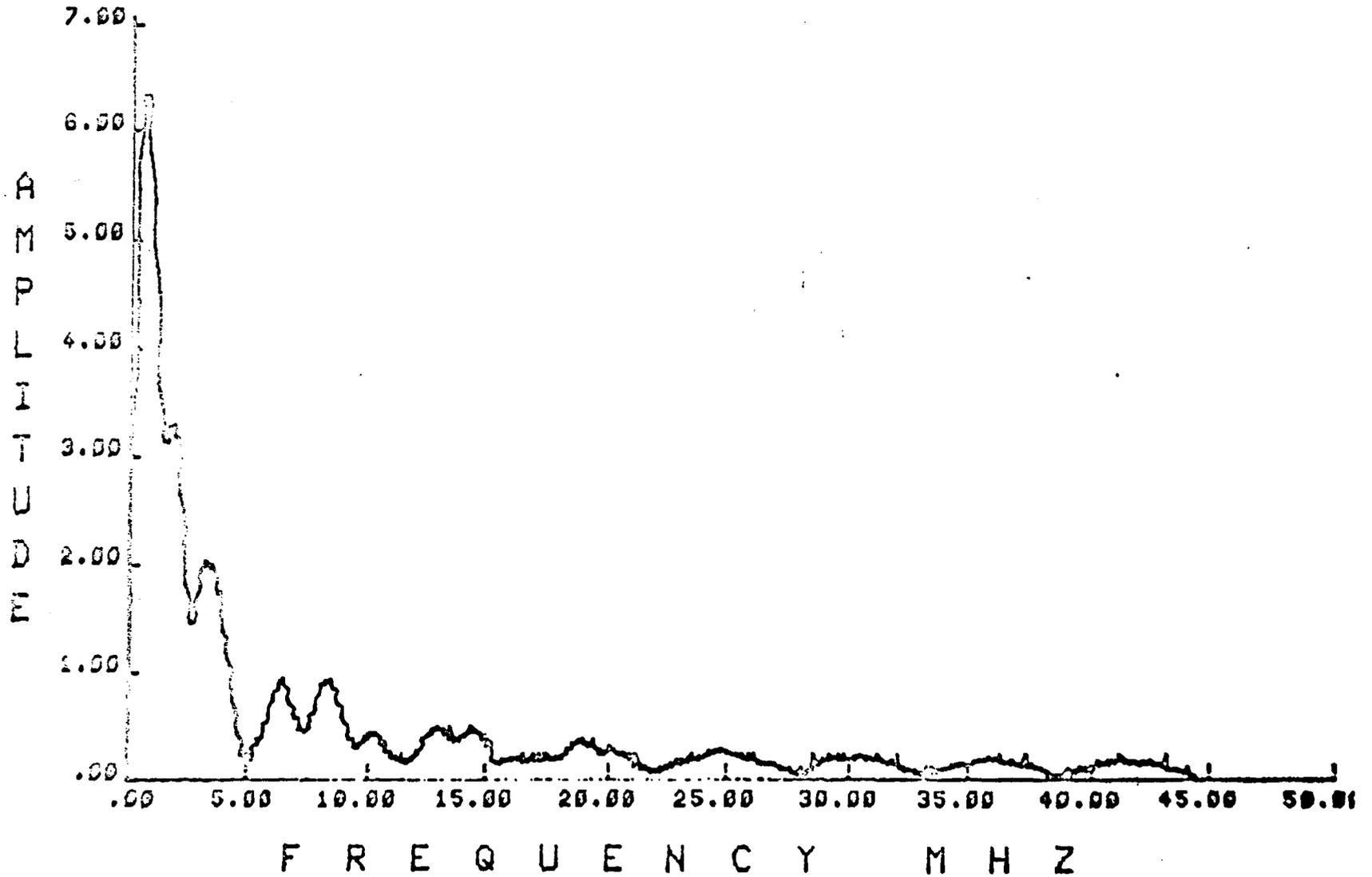


Figure 3: FFT Signal for Lift-Off

.01 USEC.
0000 DELAY

M A G N I T U D E F F T



XXI-14

Figure 4: FFT Signal for 30 Mil Defect

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1980

NASA/ASEE SUMMER FACULTY RESEARCH FELLOWSHIP PROGRAM

MARSHALL SPACE FLIGHT CENTER
THE UNIVERSITY OF ALBAMA

STABILITY OF THE STRATIFIED CYLINDRICAL ANNULUS FLOW

Prepared By:	Basil N. Antar, Ph.D.
Academic Rank:	Associate Professor
University and Department:	University of Tennessee Space Institute Department of Engineering Science and Mechanics
NASA/MSFC: (Laboratory) (Division) (Branch)	Space Sciences Atmospheric Sciences Fluid Dynamics
MSFC Counterpart:	W. W. Fowlis
Date:	August 1, 1980
Contract No.:	NAT-01-002-099 (University of Alabama)

STABILITY OF THE STRATIFIED CYLINDRICAL ANNULUS FLOW

by

Basil N. Antar
Assistant Professor of Engineering
Science and Mechanics
The University of Tennessee Space Institute
Tullahoma, Tennessee 37388

ABSTRACT

The linear stability analysis for the stratified flow between two rotating circular cylinders is formulated. Two approaches for the stability analysis are presented. The first approach results in an algebraic eigenvalue problem, while the second results in an initial value problem for the perturbation function. The advantages and disadvantages of both approaches are discussed and a preferable numerical solution technique is outlined.

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The linear stability analysis for the stratified flow between two rotating circular cylinders is formulated. Two approaches for the stability analysis are presented. The first approach results in an algebraic eigenvalue problem, while the second results in an initial value problem for the perturbation function. The advantages and disadvantages of both approaches are discussed and a preferable numerical solution technique is outlined.

INTRODUCTION

It is generally believed that the existence of fluctuations in the circulation of the atmosphere can be attributed to the instability of the stationary dynamical state without fluctuations. Partial confirmation of this belief has recently come from detailed analysis of the numerical general circulation models. However, no exact laboratory experiments have been conducted of the atmospheric circulation in order to test this hypothesis. The absence of such a laboratory model is basically due to the difficulty arising from the large earth gravity field which is ever present in any terrestrial experiment. Recently hopes for the possibility of setting up an experiment model of the earth's global circulation has been rekindled again through the availability of the gravity free environment of the forthcoming missions of Spacelab. Indeed an experiment to model and study atmospheric fluctuations arising from stability considerations in a true spherical geometry is being considered by NASA for a future Spacelab Mission [1].

The existence of fluctuations in the flow inside a rotating cylindrical annulus in which the inner cylinder is cooled while the outer cylinder is heated have been observed experimentally by Fultz et al. [2] and Fowlis and Hide [3]. These fluctuations were found to resemble in a remarkable way those that are observed in the atmosphere although the geometries are different. Due to this striking resemblance between the waves in the annulus experiments and the waves in the atmosphere, the flow field in the rotating cylindrical annulus has been studied extensively (see Hide and Mason [4] for an up to date discussion on the annulus problem).

One of the major findings common to both the analytical and experimental studies of the annulus problem is that waves of different azimuthal wave lengths appear only for specific value of the governing parameters of the problem. This fact is usually shown by plotting a curve in the Taylor-Froude number plane which separates the wave regime from the no-wave (symmetric) regime. Such a curve has been obtained experimentally by Fowlis and Hide [3]. However, in the theoretical treatment of the exact annulus problem only few points on that curve have been obtained, originally by Williams [5] and subsequently by Quon [6]. These theoretical confirmations

have been obtained through the complete, three-dimensional, numerical solution of the governing equations for the annulus geometry. These numerical solutions of the exact problem consume excessive amounts of computer time to perform. For this reason the regime diagram which was obtained experimentally have not been reproduced in full through numerical solutions, since the amounts of computer time needed to perform these calculations are beyond the scope of present day computers.

In this report we will outline a different numerical method for obtaining the regime diagrams for the annulus problem. Since the existence of the observed fluctuations in the annulus is due to instability of the basic dynamical state, it then seems reasonable that a stability analysis of the basic state, symmetric annulus flow should predict under what conditions the fluctuations will appear. Now, it is well known that stability analysis requires far less calculations than the total solution of the governing system of equations. Hence, such an analysis may be efficiently used to generate the desired regime diagrams. In this report two methods of stability analysis will be presented and the advantages and disadvantages of each method will be discussed in detail. Furthermore, a recommendation will be given on the best numerical implementation of the techniques in question.

STABILITY ANALYSIS

As indicated in the introduction there are basically two analytical approaches that may be used to verify the experimental regime diagrams. The first approach is to integrate the initial-boundary value problem starting from a state of no motion and following the solution in time. This approach requires the solution of the full governing equation in three-dimensions and for long enough time to discern whether fluctuations will appear or not. However, the computation time that may be necessary to implement this approach could be prohibitive. Quon [6] has indicated that as much as 150 hours of computer time on the Univac 1108 is necessary to establish only one point in the regime diagram using this approach.

The alternative approach which is subsequently outlined is to use linear stability analysis to establish the regime diagrams. This approach may be implemented in one of two ways both of which will be discussed at length.

The linear stability analysis starts by first formulating the governing equation for the annulus flow in cylindrical geometry. These equations are the Navier-Stokes equations, the energy equation and the mass conservation equation. The Navier-Stokes equations in cylindrical geometry in a rotating reference frame, are the following:

$$u_t + u u_r + \frac{v}{r} u_\phi + w u_z - \frac{v^2}{r} - 2\Omega v = -\frac{1}{\rho} p_r + \nu \left(\nabla^2 u - \frac{2}{r^2} v_\phi - \frac{u}{r^2} \right) \quad (1)$$

$$v_t + u v_r + \frac{v}{r} v_\phi + w v_z + \frac{u v}{r} + 2\Omega u = -\frac{1}{\rho} p_\phi + \nu \left(\nabla^2 v + \frac{2}{r^2} u_\phi - \frac{v}{r^2} \right) \quad (2)$$

$$w_t + u w_r + \frac{v}{r} w_\phi + w w_z = -\frac{1}{\rho} p_z + \nu \nabla^2 w + \alpha g T \quad (3)$$

where

$$\nabla^2 f = f_{rr} + \frac{1}{r^2} f_{\phi\phi} + f_{zz} + \frac{1}{r} f_r$$

and

$$f_a = \partial f / \partial a$$

In Eq. (3) the Boussinesq approximation has been used. In the above equations u , v and w are the radial (r), azimuthal (ϕ) and axial (z) velocity components, respectively. p and ρ are the pressure and density; ν is the kinematic viscosity; α is the volumetric expansion coefficient and T is the temperature. Ω is the angular velocity of both cylinders. In conjunction with above system the energy and mass conservation equations are needed which are given by

$$T_t + u T_r + \frac{v}{r} T_\phi + w T_z = \kappa \nabla^2 T \quad (4)$$

$$u_r + \frac{1}{r} v_\phi + w_z + \frac{u}{r} = 0 \quad (5)$$

where κ is the thermal diffusivity of the fluid.

Next, all the field variables are separated into a stationary mean part and a perturbation part in the following way:

$$\begin{aligned} u &= u_0 + u' \\ v &= v_0 + v' \\ w &= w_0 + w' \\ p &= p_0 + p' \\ T &= T_0 + T' \end{aligned}$$

When the above variables are substituted into the governing system of equations (1)-(5) and using the fact that the mean variables (f_0) themselves satisfy equations (1)-(5) a system of equations governing the perturbation functions results. However, this latter system of equations is as complicated as the original system with the added complexity of having the basic state functions appearing as coefficients in the equations. This means that in order to solve for the perturbation functions one needs to solve for the basic state first.

If it is further assumed that the perturbation functions are very small, i.e. infinitesimal then it is possible to linearize the system of equation for the perturbations. These equations will be of the following form after dropping the primes:

$$\begin{aligned} u_t + u_0 u_r + \frac{v_0}{r} u_\varphi + w_0 u_z + u u_{0r} + w u_{0z} - \frac{2v_0 v}{r} - 2\Omega v \\ = -\frac{1}{\rho} p_r + \nu \left(\nabla^2 u - \frac{2}{r^2} v_\varphi - \frac{u}{r^2} \right) \end{aligned} \quad (6)$$

$$\begin{aligned} v_t + u_0 v_r + \frac{v_0}{r} v_\varphi + w_0 v_z + u v_{0r} + w v_{0z} + \frac{1}{r} (u_0 v + u v_0) + 2\Omega u \\ = -\frac{1}{\rho r} p_\varphi + \nu \left(\nabla^2 v + \frac{2}{r^2} u_\varphi - \frac{v}{r^2} \right) \end{aligned} \quad (7)$$

$$w_t + u_0 w_r + \frac{v_0}{r} w_\varphi + w_0 w_z + u w_{0r} + w w_{0z} = -\frac{1}{\rho} p_z + \nu \nabla^2 w + \alpha g T \quad (8)$$

$$T_t + u_0 T_r + \frac{v_0}{r} T_\phi + w_0 T_z + u T_{0r} + w T_{0z} = \kappa \nabla^2 T \quad (9)$$

$$u_r + \frac{u}{r} + \frac{1}{r} v_\phi + w_z = 0 \quad (10)$$

In the above equations the stationary basic state variables are functions of r and z alone are assumed to be known everywhere.

EIGENVALUE PROBLEM

We continue with the stability analysis by looking for solutions to Eqs. (6)-(10) for the perturbation functions with the known stationary basic state for some known initial small values for the perturbations. The solution will proceed by first formulating the eigenvalue problem. Eqs. (6)-(10) can be simplified by noting that they are linear and the coefficients are independent of the azimuthal direction ϕ . Under these conditions the perturbation functions admit the following form of solution:

$$[u, v, w, \tau, \rho] = [\hat{u}(r, z), \hat{v}(r, z), \hat{w}(r, z), \hat{\theta}(r, z), \hat{\pi}(r, z)] \times \exp[ik(\phi - ct)] \quad (11)$$

The solution form (11) represents a wave of wavelength $2\pi/k$ which is propagating in the azimuthal direction ϕ with a speed c . Upon substituting the functional form (11) into the perturbation equations (6)-(10), the following equations result

$$-ikc\hat{u} + u_0\hat{u}_r + \frac{v_0}{r} ik\hat{u} + w_0\hat{u}_z + \hat{u}u_{0r} + \hat{w}u_{0z} - 2\frac{v_0}{r}\hat{u} - 2\Omega\hat{u} = -\frac{1}{\rho}\hat{p}_r + \nu(\nabla^2\hat{u} - \frac{2k}{r^2}\hat{u} - \frac{\hat{u}}{r^2}) \quad (12)$$

$$-ikc\hat{v} + u_0\hat{v}_r + \frac{1}{r}ikv_0\hat{v} + w_0\hat{v}_z + \hat{u}v_{0r} + \hat{w}v_{0z} + \frac{1}{r}(u_0\hat{v} + \hat{u}v_0) + 2\Omega\hat{u} = -\frac{ik}{\rho}\hat{p} + \nu(\nabla^2\hat{v} + \frac{2k}{r^2}\hat{u} - \frac{\hat{v}}{r^2}) \quad (13)$$

$$-ikc\hat{w} + u_0\hat{w}_r + \frac{ikv_0}{r}\hat{w} + w_0\hat{w}_z + \hat{u}w_{0r} + \hat{w}w_{0z} = -\frac{1}{s}\hat{p}_z + v\nabla^2\hat{w} + \alpha g\theta \quad (14)$$

$$-ikc\theta + u_0\theta_r + \frac{ikv_0}{r}\theta + w_0\theta_z + \hat{u}T_{0r} + \hat{w}T_{0z} = \kappa\nabla^2\theta \quad (15)$$

$$\hat{u}_r + \frac{\hat{u}}{r} + \frac{ikv}{r}\hat{u} + \hat{w}_z = 0 \quad (16)$$

In the present work the perturbation functions will be zero at the boundaries.

Now Eqs. (12)-(16) may be written in the following operator form:

$$\mathcal{L}_1 \begin{Bmatrix} \hat{u} \\ \hat{w} \\ \hat{p} \\ \theta \end{Bmatrix} = c \mathcal{L}_2 \begin{Bmatrix} \hat{u} \\ \hat{w} \\ \hat{p} \\ \theta \end{Bmatrix} \quad (17)$$

where \mathcal{L}_1 and \mathcal{L}_2 are linear partial differential operators which involve the basic state u_0, v_0, w_0, T_0 and k . Since Eq. (17) is homogeneous with homogeneous boundary condition then it forms an eigenvalue problem for the eigenvalue c .

Normally the system (17) is solved through the method of separation of variables which will reduce the eigenvalue problem to an ordinary differential eigenvalue problem. Such a problem may be solved either by the shooting method or the matrix method (see Antar [7] for a discussion on the application of these techniques). However, in the present case we can not use the method of separation of variables since the coefficients in the operators are not separable. Thus, the partial differential eigenvalue problem (17) has to be solved as it stands.

A logical way to handle the system (17) is to use the finite difference method especially since the coefficients are given as a set of numbers at specific points in the domain. This is due to the fact that the basic state solution is obtained through a finite difference method. Without going too much into the details of the finite differencing procedure, if the derivatives in the operators \mathcal{L}_1 and \mathcal{L}_2 are

approximated by finite differences, the equation (17) will be transformed to the following algebraic problem

$$[A]\{\bar{x}\} + c[B]\{\bar{x}\} = 0 \quad (18)$$

where [A] and [B] are banded matrices and \bar{x} is a vector representing the value of the field variables at the discrete points in the solution domain. Taking the inverse of [B] in (18), the equation may be rewritten as

$$\{[\Delta] - c[I]\}\{x\} = 0 \quad (19)$$

where $[\Delta] = [B]^{-1}[A]$. Clearly (19) then is an algebraic eigenvalue problem which may be solved through matrix manipulations.

Although in principle the technique is straight forward, in reality the transformation of Eq. (17) into Eq. (18) is quite involved and will not be discussed here. The major difficulty in formulating Eq. (19) is that the vector \bar{x} is very large. In fact if one chooses to discretize Eq. (17) through 10 points in each of the z and r directions the vector x will have 400 elements. However, we know from experience that 10 points for the discretization is not enough and a reasonable choice will be 50₄ points. For this latter value the vector x will possess 10⁴ elements. This size of the vector will make the solution of the eigenvalue problem very lengthy. Another approach which might be more suitable for the present problem is discussed next.

INITIAL VALUE PROBLEM

The second alternative for obtaining a solution to the perturbation equations (6)-(10) is to assume a solution which possesses a sinusoidal form in the azimuthal direction, ϕ , i.e.

$$f(r, z, \phi, t) = \hat{f}(r, z, t; k) \sin k\phi \quad (20)$$

This form of the solution is admissible due to the linearity of the equations and also the independence of the coefficient function of the azimuthal direction ϕ . Upon substituting the functional form (20) into eqs. (6)-(10), the following linear system of equations results:

$$\begin{aligned} \hat{u}_t + u_0 \hat{u}_r + ik \frac{v_0}{r} \hat{u} + w_0 \hat{u}_z + \hat{u} u_{0r} + \hat{w} u_{0z} - \frac{z v_0}{r} \hat{u} - z \Omega \hat{u} \\ = -\frac{1}{\xi} \hat{p}_r + \nu \left(\nabla^2 \hat{u} - z \frac{k}{r^2} \hat{u} - \frac{\hat{u}}{r^2} \right) \end{aligned} \quad (21)$$

$$\begin{aligned} \hat{v}_t + u_0 \hat{v}_r + ik \frac{v_0}{r} \hat{v} + w_0 \hat{v}_z + \hat{v} v_{0r} + \hat{w} v_{0z} + \frac{1}{r} (u_0 \hat{v} + \hat{u} v_0) + z \Omega \hat{v} \\ = -\frac{ik}{\xi r} \hat{p} + \nu \left(\nabla^2 \hat{v} + z \frac{k}{r^2} \hat{v} - \frac{\hat{v}}{r^2} \right) \end{aligned} \quad (22)$$

$$\hat{w}_t + u_0 \hat{w}_r + ik \frac{v_0}{r} \hat{w} + w_0 \hat{w}_z + \hat{u} w_{0r} + \hat{w} w_{0z} = -\frac{1}{\xi} \hat{p}_z + \nu \nabla^2 \hat{w} + \alpha g \theta \quad (23)$$

$$\theta_t + u_0 \theta_r + ik \frac{v_0}{r} \theta + w_0 \theta_z + \hat{u} T_{0r} + \hat{w} T_{0z} = \kappa \nabla^2 \theta \quad (24)$$

The set (21)-(24) constitute a linear partial differential set depicting an initial value problem. Thus a solution to the system may be obtained once the proper initial conditions are given. The initial conditions in this case are given again as a wave function with a specified wavenumber k and a specified amplitude function.

The stability or instability of the perturbations is resolved by examining the behavior of the amplitude functions

with time. If for a given wavenumber k , the amplitudes appear to grow with time, then the perturbations are said to be unstable. If, on the other hand, the perturbation amplitudes seem to decay with time then the perturbations are said to be stable. As for the eigenvalue problem, however, the sign of the imaginary part of c determines stability or instability. In this case if the sign of c_j is negative, then the perturbations are said to be stable, otherwise it is unstable.

Now, due to the complexity of the coefficients of the system (21)-(24), no closed form solution is possible. Thus one has to resort to numerical solutions. In this case it seems logical to use finite difference techniques for both the time and the two spatial directions. Conceptually, a numerical solution for this system should be obtained without difficulty due to the linearity of the system.

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ON ROSEN'S THEORY OF GRAVITY
AND COSMOLOGY

Prepared By:	Dr. Ronnie C. Barnes
Academic Rank:	Associate Professor
University and Department:	Lambuth College Physical Science Department
NASA/MSFC: (Laboratory) (Division) (Branch)	Space Sciences Cryogenics Gravitation
MSFC Counterpart:	Dr. Peter Eby
Date:	August 1, 1980
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ON ROSEN'S THEORY OF GRAVITY AND COSMOLOGY

By

Dr. Ronnie C. Barnes
Associate Professor of Astronomy
Lambuth College
Jackson, TN 38301

ABSTRACT

Formal similarities between general relativity and Rosen's bimetric theory of gravity have been used to analyze various bimetric cosmologies. The following results have been found:

- (1) Physically plausible model universes which have a flat static background metric, a Robertson-Walker fundamental metric and which allow co-moving coordinates do not exist in bimetric cosmology.
- (2) It is difficult to use the Robertson-Walker metric for both the background metric and $g_{\mu\nu}$ and require that $g_{\mu\nu}$ and $\gamma_{\mu\nu}$ have different time dependences because there are not enough bimetric field equations to simultaneously fix the time dependences.
- (3) A consistency relation for using co-moving coordinates in bimetric cosmology was derived.
- (4) Certain spatially flat bimetric cosmologies of Babala (1975) were tested for the presence of particle horizons.
- (5) An analytic solution for Rosen's (1978) $k = +1$ model was found.
- (6) Rosen's singularity free $k = +1$ model arises from what appears to be an arbitrary choice for the time dependent part of $\gamma_{\mu\nu}$.

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I. INTRODUCTION

In 1973 and 1974 Nathan Rosen proposed a bimetric theory of gravitation. The origin of its name arises from Rosen's assumption that two tensor fields ($g_{\mu\nu}$ and $\gamma_{\mu\nu}$) exist at each point (x^1, x^2, x^3, x^4) of spacetime. Rosen (1973, 1974, 1978) has given the following interpretations for $g_{\mu\nu}$ and $\gamma_{\mu\nu}$:

As in general relativity, $g_{\mu\nu}$ is the fundamental metric tensor of Riemannian geometry. The motion of material bodies and photons is determined by $g_{\mu\nu}$. Moreover, those aspects of spacetime which can be observed by using clocks and measuring rods are contained in $g_{\mu\nu}$.

On the other hand, $\gamma_{\mu\nu}$ may be thought of as representing the spacetime metric that one would obtain if all the mass-energy in the universe was removed. $\gamma_{\mu\nu}$ is called the "background" metric. According to Rosen (1978), " $\gamma_{\mu\nu}$ describes a geometry of spacetime not directly observable at present."

Bimetric gravitation has a number of interesting features:

1. It is a metric theory which, if suitable boundary conditions are chosen, agrees exactly with general relativity in the post-Newtonian limit (Lee, et. al. 1976).
2. It does not appear to predict the existence of black holes (Rosen 1973, 1974).
3. Considerably larger (5-6X) masses for neutron stars are allowed by bimetric gravitation than by general relativity (Rosen and Rosen 1975).
4. If the background metric is assumed to be Minkowskian ($\gamma_{\mu\nu} = \eta_{\mu\nu}$), then it is possible to express gravitational energy and momentum density via a conserved tensor and the field equations and resulting calculations are simpler than in general relativity (Rosen 1973, 1974).
5. Homogeneous bimetric cosmological models which have acceptable values for the Hubble parameter H_0 and the deceleration parameter q_0 have been found and studied by Babala ($k=0$, 1975), Goldman and Rosen ($k=-1$, 1977), and Rosen ($k=+1$, 1978).
6. At the present time the only observation known to pose difficulties for Rosen's theory is the period change $|\dot{P}/P| = 1.2 \times 10^{-7} \text{ yr}^{-1}$ determined by Taylor et.al. (1976) for the binary pulsar PSR 1913 + 16. Will and Eardley (1977) analyzed binary star motion using

bimetric theory and, assuming that gravitational radiation from a source is described by a retarded potential solution of the wave equation, have shown that bimetric gravitation allows dipole radiation. (However, also see Rosen 1978b for a different view.) Will and Eardley concluded that Rosen's theory is consistent with observations only if PSR 1913 + 16 contains either (a) two very heavy neutron stars of nearly equal mass or (b) a very light neutron star whose companion is a rapidly rotating white dwarf or helium main sequence star.

II. MOTIVATION--COSMOLOGICAL HORIZONS

One of the most intriguing predictions of general relativity is its prediction of "event horizons" in spacetime. These are boundaries which separate observable events from those not observable. The best known example of an event horizon is that which surrounds a spherical black hole and which happens to arise at the same value of the radial coordinate, $r = 2GM/c^2$, for which the Schwarzschild metric

$$ds^2 = \left(1 - \frac{2GM}{rc^2}\right) dt^2 - \frac{dr^2}{1 - 2GM/rc^2} - r^2 d\theta^2 - r^2 \sin^2 \theta d\phi^2 \quad (1)$$

has a singularity.

However, the existence of an horizon in spacetime is independent of the choice of a coordinate system and, in fact, it is possible to have event horizons even when spacetime is perfectly homogeneous. Rindler (1956), in his classic paper on cosmological horizons, showed that homogeneous cosmology permits universes with particle horizons, event horizons or both.

The presence of particle horizons (a surface in 3-space which divides all fundamental particles into two classes: those which have been observed by a given "fundamental observer" and those which have not been observed) has troubled cosmologists for the following reason. Particle horizons represent regions which have been and still are causally disconnected from other parts of the universe. However, according to the precepts of canonical cosmology, all fundamental observers must find identical physical conditions (of course disregard small scale local inhomogeneities) at the same instant of cosmic time. The obvious and difficult question is: How do regions which are causally disconnected achieve such a remarkable degree of physical identity?

Of course, one can always postulate special initial conditions, but this approach seems more in the spirit of theology than of

science. A more challenging approach is to assume that the universe began with anisotropies present and that through the actions of certain mechanisms homogeneity naturally arose. (Misner, Thorne and Wheeler 1973, ch. 30). For this approach to be successful it is essential that the following two requirements be met:

- (1). A satisfactory mechanism for converting initial anisotropies into isotropy be found.
- (2). Whatever the mechanism, there must be sufficient time to create the degree of homogeneity and isotropy now observed.

It is possible to use the 3°K microwave background radiation to place some constraints on the time available for producing homogeneity (Misner, Thorne and Wheeler 1973, ch. 30). These constraints are so severe--the universe must be already homogeneous as early as the epoch $z = 1000$ (only 600,000 yr after creation in an Einstein-de Sitter model) that no entirely satisfactory chaotic cosmology has yet been found.

Therefore, because particle horizons pose an important problem for cosmology, it is important to analyze bimetric cosmology in order to discover if these horizons exist in bimetric models as they do in general relativity.

III. BIMETRIC COSMOLOGY WITH THE ROBERTSON-WALKER METRIC.

Nowadays it is popular to use Robertson-Walker spacetime

$$ds^2 = dt^2 - \frac{R^2(t)}{(1+kr^2/4)^2} (dr^2 + r^2 d\theta^2 + r^2 \sin^2\theta d\phi^2) \quad (2)$$

for cosmological investigations. Here (r, θ, ϕ) form a set of "co-moving" spherical coordinates, t is cosmic time, $R(t)$ is the scale factor function, and k is the constant of space curvature.

Rindler (1956) derived equations which allow one to test for horizons in Robertson-Walker spacetime. In models which have $R = 0$ a particle horizon will exist at time t_0 if

$$\int_0^{t_0} \frac{dt}{R(t)} \quad \text{converges.} \quad (3)$$

Some cosmologies never pass through $R = 0$ and begin their motion as $t \rightarrow -\infty$. In these models a particle horizon will exist at t_0 if

$$\int_{-\infty}^{t_0} \frac{dt}{R(t)} \quad \text{converges.} \quad (4)$$

For an event horizon there is only one test, viz.,

$$\int_{t_0}^{\infty} \frac{dt}{R(t)} \quad \text{converges.} \quad (5)$$

Although the above tests are, at least in principle, straightforward there do arise problems when one attempts to use them in bimetric cosmology. The principle difficulty arises from the

from the fact that none of the published bimetric cosmological models are consistent with the Robertson-Walker line element for $g_{\mu\nu}$ and, at the same time, consistent with the form used for $\gamma_{\mu\nu}$ in solving the bimetric field equations. For example, Babala (1975) initially assumes

$$ds_{\mathbf{I}}^2 = g_{\mu\nu} dx^\mu dx^\nu = e^{2\bar{\Phi}} dt^2 - e^{2\Psi} (dr^2 + r^2 d\theta^2 + r^2 \sin^2 \theta d\phi^2) \quad (6)$$

$$ds_{\mathbf{II}}^2 = \gamma_{\mu\nu} dx^\mu dx^\nu = dt^2 - dr^2 - r^2 d\theta^2 - r^2 \sin^2 \theta d\phi^2 \quad (7)$$

where $\bar{\Phi}$ and Ψ are functions of t . By redefining the time coordinate it is possible to transform eq. (6) into Robertson-Walker form

$$\tau \equiv \int e^{\bar{\Phi}} dt \quad ; \quad ds_{\mathbf{I}}^2 = d\tau^2 - e^{2\Psi(\tau)} (dr^2 + r^2 d\theta^2 + r^2 \sin^2 \theta d\phi^2). \quad (8);(9)$$

From equations given by Babala it is easy to show that his models require

$$\tau = e^{\bar{\Phi}_0} \tanh t + k \quad (10)$$

where $\bar{\Phi}_0$ and k are obtained from initial conditions. The simplest case to analyze is $k = \bar{\Phi}_0 = 0$ and then one finds

$$\tau = \tanh t \quad -1 < \tau < 1 \quad (11)$$

$$t = \frac{1}{2} \ln \frac{1+\tau}{1-\tau} \quad (12)$$

$$e^{\Psi(\tau)} = e^{\Psi_0} \frac{1}{1-\tau^2} \left(\frac{1+\tau}{1-\tau} \right)^{\lambda'} \quad (13)$$

where λ' is a constant of integration related to the value of $\dot{\Psi}_0$. According to Babala, expanding models are given by $\lambda' > 1$, $\lambda' = 1$, $-1 < \lambda' < 1$. The simplest case to study is the one with $\lambda' = 1$ because eq. (13) simplifies to form easy to integrate.

If the Rindler tests are applied, then one finds that this model contains both a particle horizon and an event horizon at τ_0 because both

$$\int_{-\infty}^{\tau_0} \frac{dt}{R(t)} \rightarrow \int_{-1}^{\tau_0} e^{-\Psi_0} (1-\tau)^2 d\tau = \frac{e^{-\Psi_0}}{3} (8 - (1-\tau_0)^3) \quad (14)$$

$$\int_{\tau_0}^{\infty} \frac{dt}{R(t)} \rightarrow \int_{\tau_0}^1 e^{-\Psi_0} (1-\tau)^2 d\tau = \frac{e^{-\Psi_0}}{3} (1-\tau_0)^3 \quad (15)$$

converge.

However, it is possible to find bimetric models without particle horizons. An example is afforded by choosing $\lambda = 2$. In this model the particle horizon integral logarithmically diverges as $\tau \rightarrow -1$.

$$\int_{-1}^{\tau_0} e^{-\psi_0} \frac{(1-\tau)^3}{1+\tau} d\tau = \left[8 \ln(1+\tau) - 8(1+\tau) + 2(1+\tau)^2 - \frac{1}{3}(1+\tau)^3 \right] e^{-\psi_0} \Big|_{-1}^{\tau_0} \quad (16)$$

The difficulty which results when the time coordinate is re-defined lies in the fact that while eq. (8) succeeds in converting eq. (6) to Robertson-Walker form it also forces the background metric to be time dependent. However, in deriving eqs. (10)-(13), gotten by substituting eqs. (6) and (7) into the bimetric field equations, Babala assumed that $\gamma_{\mu\nu}$ was independent of time. Bimetric cosmologies published by Goldman and Rosen (1977) and Rosen (1978) are subject to the same criticism. However, in the latter paper Rosen assumes that $\gamma_{\mu\nu}$ is time dependent at the outset and that $g_{\mu\nu}$ is given by eq. (6). In this case the effect of making a new time coordinate via eq. (8) is to change the time dependence of $\gamma_{\mu\nu}$ away from the form used in setting up the field equations.

The preceding considerations suggest that the Robertson-Walker line element be introduced at the start rather than making an attempt to transform to it at a later stage. Unfortunately, this procedure does not work if one uses a flat, static background metric. On substituting eqs. (2) and (7) into the bimetric field equations (see Rosen 1973, 1974 for more detail):

$$N_{\mu\nu} = \frac{1}{2} \gamma^{\alpha\beta} g_{\mu\nu|\alpha\beta} - \frac{1}{2} \gamma^{\alpha\beta} g^{\lambda\rho} g_{\mu\lambda|\alpha} g_{\nu\rho|\beta} = -8\pi\kappa \left(T_{\mu\nu} - \frac{1}{2} g_{\mu\nu} T \right) \quad (17)$$

and evaluating the (4,4) component one finds

$$0 = -4\pi\kappa\rho \quad (18)$$

for a matter dominated universe with negligible pressure. Apparently, only empty Robertson-Walker bimetric cosmologies are allowed by using eqs. (2) and (7).

The simplest way to produce non-empty bimetric Robertson-Walker models is to let $\gamma_{\mu\nu}$ be time dependent. A straightforward method would be to let $\gamma_{\mu\nu}$ have a Robertson-Walker form but with a different time dependence than $g_{\mu\nu}$, viz.,

$$ds_{\text{I}}^2 = dt^2 - \frac{R^2(t)}{(1+kr^2/4)^2} [dr^2 + r^2 d\theta^2 + r^2 \sin^2 \theta d\phi^2] \quad (19)$$

$$ds_{\text{II}}^2 = dt^2 - \frac{S^2(t)}{(1+kr^2/4)^2} [dr^2 + r^2 d\theta^2 + r^2 \sin^2 \theta d\phi^2] \quad (20)$$

However, this approach is not completely satisfactory because

there are not enough field equations to simultaneously determine both $R(t)$ and $S(t)$.

Rosen (1978) made an interesting attempt to fix the time dependence of $\gamma_{\mu\nu}$ by using the cosmological principle. It is well known, e.g., see Robertson and Noonan 1968, ch. 14, 347, that the only non-static metric which satisfies the perfect cosmological principle is the spatially flat de Sitter line element

$$ds^2 = dt^2 - e^{\alpha t} (dr^2 + r^2 d\theta^2 + r^2 \sin^2 \theta d\phi^2) \quad (21)$$

When $\gamma_{\mu\nu}$ is identified with de Sitter's metric it is possible to solve Rosen's field equations for $R(t)$. Rosen, however, rejected the resulting model because it has a big-banglike singularity at $t \rightarrow \infty$. Assuming that the time dependence of $\gamma_{\mu\nu}$ is $\cosh t/a$ rather than $\exp 2t/a$ and that both $g_{\mu\nu}$ and $\gamma_{\mu\nu}$ are spacetimes of constant positive curvature, Rosen was able to derive an expanding, closed and singularity free universe.

However, it should be pointed out that these goals were reached by making what seems to be an arbitrary choice for $S(t)$. Also Rosen fails to point out that if $S(t) = \cosh t/a$, then $\gamma_{\mu\nu}$ no longer satisfies the perfect cosmological principle because the Hubble parameter of this metric

$$\dot{S}/S = \frac{1}{a} \tanh t/a \quad (22)$$

is not independent of time.

During the course of this fellowship the author attempted to develop a satisfactory method for fixing the time dependent part of $\gamma_{\mu\nu}$. One approach, which looked promising, was suggested by the fact that in general relativity one can start with a rather general spacetime

$$ds^2 = e^{\nu(r,t)} dt^2 - e^{\mu(r,t)} (dr^2 + r^2 d\theta^2 + r^2 \sin^2 \theta d\phi^2) \quad (23)$$

and derive the Robertson-Walker line element by making use of a "consistency relation" which stems from the use of co-moving coordinates (McVittie, 1965, ch.8).

Since no published work about bimetric consistency relations now exists, the author hoped to (1) derive a consistency relation for the use of co-moving coordinates and (2) use it, if possible, to pin down the time dependence of $\gamma_{\mu\nu}$.

The first step was accomplished for the following case. If one assumes that $\gamma_{\mu\nu}$ and $g_{\mu\nu}$ generate spherically symmetric spacetimes

$$ds_{\text{I}}^2 = g_{\mu\nu} dx^\mu dx^\nu = e^{\nu(r,t)} dt^2 - e^{\mu(r,t)} (dr^2 + r^2 d\theta^2 + r^2 \sin^2 \theta d\phi^2) \quad (24)$$

$$ds_{\text{II}}^2 = \gamma_{\mu\nu} dx^\mu dx^\nu = dt^2 - e^{\lambda(r,t)} (dr^2 + r^2 d\theta^2 + r^2 \sin^2 \theta d\phi^2) \quad (25)$$

and then substitutes these into the bimetric field equations, then one finds the following non-vanishing components

$$N_{11} = \frac{1}{2} \left[\mu_{rr} - \lambda_{rr} + (\mu_r - \lambda_r) \left(\frac{2}{r} + \frac{1}{2} \lambda_r \right) \right] e^{\mu-\lambda} + \frac{1}{2} \left[\lambda_{tt} - \mu_{tt} + \frac{3}{2} (\lambda_t - \mu_t) \right] e^\mu + \frac{1}{8} \lambda_t^2 e^{2\mu-\lambda-\nu} - \frac{1}{8} \lambda_t^2 e^{\lambda+\nu} \quad (26)$$

$$N_{22} = r^2 N_{11} \quad (27)$$

$$N_{33} = r^2 N_{11} \sin^2 \theta \quad (28)$$

$$N_{44} = -\frac{1}{2} \left[\nu_{rr} + \left(\frac{2}{r} + \frac{1}{2} \lambda_r \right) \nu_r \right] e^{\nu-\lambda} + \frac{1}{2} \left[\nu_{tt} + \frac{3}{2} \lambda_t \nu_t \right] e^\nu + \frac{3}{8} \lambda_t^2 \left[e^{\mu-\lambda} - e^{\lambda+2\nu-\mu} \right] \quad (29)$$

$$N_{14} = \frac{1}{4} \left[\lambda_{tr} + \lambda_t (\nu_r + \lambda_r - \mu_r) \right] e^\nu - \frac{1}{4} \left[\lambda_{tr} - \lambda_t (\nu_r + \lambda_r - \mu_r) \right] e^{\mu-\lambda} \quad (30)$$

The use of co-moving coordinates requires that, in spherically symmetric spacetime,

$$T^{14} = (\rho + p) \frac{dx^1}{ds} \frac{dx^4}{ds} - g^{14} p \quad (31)$$

vanish since $\frac{dx^1}{ds} = 0$ and $g^{14} = 0$. However, these conditions also require $T^{14} - g^{14} T/2 = 0$ and that means $N_{14} = 0$. Thus, the bimetric consistency relation is

$$\left[\lambda_{tr} + \lambda_t (\nu_r + \lambda_r - \mu_r) \right] e^\nu = \left[\lambda_{tr} - \lambda_t (\nu_r + \lambda_r - \mu_r) \right] e^{\mu-\lambda} \quad (32)$$

For comparison, the analogous general relativistic equation is

$$\mu_{rt} = \frac{1}{2} \mu_t \nu_r \quad (33)$$

As outlined in McVittie's textbook, it is possible to integrate eq.(33) and, by requiring that the (1,1) and (2,2) components of the Einstein tensor yield the same cosmological fluid pressure, one arrives at the Robertson-Walker line element.

Unfortunately, when the same procedure is followed in bimetric theory, one discovers that N_{11} and N_{22} already give the same differential equation for pressure. Thus, unlike general

relativity, the consistency relation for co-moving coordinates does not lead to appreciable progress and, in particular, no new information is gained about the time dependence of $\gamma_{\mu\nu}$.

IV. AN ANALYTIC SOLUTION FOR ROSEN'S 1978 $k=+1$ MODEL

As mentioned in the previous section, Rosen (1978) found a closed space expanding model which does not have a singularity. During the course of this summer's work the author found an analytic solution to the bimetric field equations of this model.

$$\ddot{\Phi} + \frac{3}{a} \left(\tanh \frac{t}{a} \right) \dot{\Phi} + \frac{3}{a^2} \tanh^2 \frac{t}{a} \sinh (2\Psi - 2\Phi) = -4\pi e^{\Phi+3\Psi} (\rho + 3p) \quad (34)$$

$$\ddot{\Psi} + \frac{3}{a} \left(\tanh \frac{t}{a} \right) \dot{\Psi} - \frac{1}{a^2} \tanh^2 \frac{t}{a} \sinh (2\Psi - 2\Phi) = 4\pi e^{\Phi+3\Psi} (\rho - p) \quad (35)$$

$$\dot{\rho} + 3(\rho + p) \left(\dot{\Psi} + \frac{1}{a} \tanh \frac{t}{a} \right) = 0 \quad (36)$$

If one investigates a zero pressure model and sets $g_{44} = 1$, then these equations simplify and it is possible to use the integral of eq.(36)

$$\rho e^{3\Psi} \cosh^3 \frac{t}{a} = A \quad ; A \text{ is a constant of integration,} \quad (37)$$

to set up a single second order differential equation which is linear in Ψ

$$\ddot{\Psi} + \frac{3}{a} \dot{\Psi} \tanh \frac{t}{a} = \frac{8\pi A}{3} \operatorname{sech}^3 \frac{t}{a} \quad (38)$$

By using the classical variation of parameters method it is possible to arrive at the following general solution to eq.(38):

$$\Psi(t) = \left(\frac{8\pi A}{3} \frac{t}{a} + \frac{1}{2} aB \right) \left[\operatorname{sech} \frac{t}{a} \tanh \frac{t}{a} + \tan^{-1}(\sinh t/a) \right] + \frac{8\pi A}{3} \operatorname{sech} \frac{t}{a} - \frac{8\pi A}{3} \int \tan^{-1}(\sinh \frac{t}{a}) \frac{dt}{a} + C \quad (39)$$

In this solution B and C are constants of integration and the integral can be carried out by expanding $\tan^{-1}(\sinh t/a)$ in a power series.

Although eq.(39) is not as simple as one would like, it is nevertheless, sufficiently simple to analyze for possible singularities in cosmic time. By letting $t \rightarrow -\infty$, $t \rightarrow 0$ and $t \rightarrow \infty$ and looking at each term in (39) one can verify that Rosen's conclusion about the absence of a singularity in this model is correct.

V. SUMMARY

Because the formalism of Rosen's bimetric theory is so similar to general relativity one would think that it would be a simple matter to set up bimetric cosmology by using the Robertson-Walker metric for $g_{\mu\nu}$. However, if such is attempted, then one finds that there are too few bimetric field equations to simultaneously fix the time dependence of $g_{\mu\nu}$ and $\gamma_{\mu\nu}$.

Other results are:

- (1) Certain flat bimetric cosmologies of Babala have particle horizons; others do not. Apparently, the bimetric field equations (by themselves) are not strong enough to prevent the occurrence of cosmological particle horizons, although strangely enough, they do exclude black holes.
- (2) An empty universe results if one tries to use a flat background metric along with a Robertson-Walker fundamental metric.
- (3) Rosen's singularity free universe of positive spatial curvature (Rosen 1978) arises from what seems to be an arbitrary choice for the time dependent part of $\gamma_{\mu\nu}$.
- (4) A consistency relation for using co-moving coordinates in bimetric cosmology was derived.
- (5) An analytic solution was found for the field equations which describe Rosen's (1978) universe of positive space curvature.

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NASA/ASEE SUMMER FACULTY RESEARCH FELLOWSHIP PROGRAM

MARSHALL SPACE FLIGHT CENTER
THE UNIVERSITY OF ALABAMA

MEASUREMENTS OF THE SAMPLING VOLUME OF AN
INFRARED LASER DOPPLER VELOCIMETER FOR THE
STUDY OF FEASIBILITY OF SIMULTANEOUS
THREE-DIMENSIONAL VELOCITY PROFILE MAPPING

Prepared By:	Robert L. Bond, Ph.D.
Academic Rank:	Associate Professor
University and Department:	University of Arkansas Department of Electronics and Instrumentation
NASA/MSFC: (Laboratory) (Division) (Branch)	Electronics and Control Optical and R.F. Systems Optics
MSFC Counterpart:	James W. Bilbro
Date:	August 1, 1980
Contractor No.:	NGT-01-002-099 (University of Alabama)

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By

Robert L. Bond
Associate Professor of Electronics & Instrumentation
University of Arkansas
Little Rock, Arkansas

ABSTRACT

This project is a continuation and configuration of the experimental work performed during the summer of 1979. Data obtained after the submission of last year's report gave strong indication that it would be possible by spatial masking to code information onto an outgoing laser beam which would yield three-dimensional velocity information when analyzed at an interferometer.

The work continued into this year has involved detailed, high resolution incoherent mapping of the sampling volume to determine the three-dimensional profile of the volume of the beam from which information will be obtained by scattering.

ACKNOWLEDGEMENTS

I wish to express my gratitude to Mr. James W. Bilbro for his continuous support and many suggestions offered throughout the performance of this work. I also appreciate Mr. Bilbro's assignment of Miranda James to assist me during the data-taking phase of this work. Miranda has substantially facilitated this normally time-consuming and tedious process. I also wish to thank Dr. Bob Barfield for his very competent management of the Summer Faculty Program.

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INTRODUCTION

Marshall Space Flight Center has been involved in the application of Laser Doppler Velocimeters (LDVs) since their inception by Yeh and Cummins¹ in 1964. Subsequent research has applied LDVs to the study of aircraft wake vortices,^{2,3} atmospheric turbulence measurements,⁴ remote intensity fluctuations,⁵ dust devil measurements,⁶ and presently to severe storms.⁷

Except for the earliest work in wind tunnels, most applications have used CO₂ lasers to increase the signal-to-noise ratio - an advantage resulting from the atmospheric window at 10.6 micrometers. Present work involves pulsed CO₂ laser to further increase the signal-to-noise ratio. Future work will utilize the advantages of off-axis telescopes and, possibly, lasers operating at wavelengths which more nearly match atmospheric windows.

Each LDV has certain limitations on the velocity vectors which it can measure. The early LDV of Yeh and Cummins, for example, could easily measure flow velocities perpendicular to the incident optical plane. However, it could not measure velocity components along the optical axis. Subsequent systems, such as those used to measure aircraft wake vortices, measure the components along the optical direction at the expense of the orthogonal components. A need exists for an LDV which is capable of reading all orthogonal velocity components. Such an LDV could be used to map the three-dimensional characteristics of a remote flow field.

Several developments in LDVs point to the possibility of coding sufficient information into a laser beam so that the returned radiation will contain three-dimensional velocity component data. These developments include the classical heterodyne, fringe, and the more recent time-of-flight LDVs. It is possible that a combination of these approaches would yield the three-dimensional information necessary to map a three-dimensional flow field. The most likely impediments appear to be instrumentation artifacts which may obscure the signal of one or more velocity components or produce an unacceptable signal-to-noise ratio.

OBJECTIVES

The purpose of this work has been to obtain a detailed understanding of the sampling volume of the infrared laser Doppler velocimeter assembled during the 1979 Summer Faculty Fellowship Program. Detailed, high resolution maps were to be made of both the coherent and incoherent sampling volumes. The response of the sampling volume to spatial masking within the telescope was also to be determined. The response of the system to various objects moving through the sampling volume was also to be determined.

EXPERIMENTS

The infrared laser Doppler velocimeter (Figure 1) assembled during the Summer Faculty Fellowship Program last summer was returned to its operational status. The system was improved by modifying the external mirrors to permit mechanical rather than electrical positioning. This step was necessary to maintain more precise alignment of the entire laser Doppler velocimeter (LDV).

The detailed study of the coherent sampling volume yielded essentially the same results as those obtained last summer. The double maximum present last summer in the Doppler Signal Power vs. Secondary Mirror Position curves was not as pronounced. The cause of this phenomenon was isolated to the z-scan mechanism of the secondary mirror. As the secondary mirror is moved along the optical (z) axis, the focal point projected onto the detector assembly (27-50 feet away) moves erratically about in the x-y plane of the detector. Of course, similar wandering of the beam occurs relative to a fixed position on the rotating sandpaper wheel.

Ironically, a similar phenomenon occurs when the rotating sandpaper wheel located on a carriage on the optical bench is moved along the z-axis of the beam. In this case the "wandering of the beam" is only apparent. A fixed point on the rotating wheel is actually moving in the x-y plane as the z-traverse is made. Obviously, data entailing movement of both the wheel and secondary mirror contains a compound error resulting from both phenomena.

Data taken by scanning the secondary mirror, first in one direction, then in the reverse direction, is reasonably repeatable but is not symmetric relative to the maximum power point in the scan. This asymmetry is in part due to the x-y motion introduced into the outgoing beam.

The same data taken by scanning the rotating wheel along the optical axis, first in one direction, then in the reverse direction, is approximately as repeatable but with much less asymmetry.

When the signal power (as measured by a spectrum analyzer) was plotted against the voltage (as measured on an oscilloscope) there were discontinuities on the semi-log plot. The causes of these points which did not fall on the straight line plot were never determined. There is a remote possibility

that they resulted from an ailing Honeywell specturm analyzer which failed to function after this data was taken.

The incoherent data verified and accentuated the conclusions arrived at from the coherent data - namely that the beam wanders about in the x-y plane as it rotates in an irregular arc about the optical axis. An example of the phenomenon is as follows. With the secondary mirror adjusted to focus at the near end of the optical bench let us assume the focal point is on the optical axis. As the secondary mirror is adjusted to move the focal point toward the far end of the optical bench, the beam first swings toward the left as it swings upward and finally swings back toward the right as it reaches its maximum height at the far end of the optical bench. This general pattern was repeated for many experiments. Occasionally there was some fine detail in the arc but the trend remained the same. It was determined that the relationship between the secondary mirror position and the focal point position on the optical axis was highly repeatable and agreed with theoretical predictions.

The incoherent sampling volume was measured by moving an aperture throughout the double cone defined by the focused beam from the laser. Figure 2 is a traverse in the x-axis of the laser beam without secondary mirror or wire filter. This x-axis profile is the standard for all additional work which was taken with the beam focused between 27 and 50 feet by the secondary mirror. A 500 μm aperture was selected for the traverse of the unfocused beam to keep the power density within the limits of the pyroelectric detector and lock-in amplifier used for power detection. Apertures of suitable size were always used for taking data since neutral density filters operable at 10.6 μm were not available. Attempts to use existing flats of Ge, CaF_2 or BaF_2 were futile because of significant artifacts introduced into the data via diffraction and multiple reflections. Notice the asymmetry in the beam which is present to some extent in all data.

Initial data was taken with a 1.0 mm aperture as was done last year. It was determined that this aperture did not produce sufficient resolution for a laser beam that has a theoretical focal diameter of approximately 40 mil. The resolution would be particularly dismal if one were interested in detecting auxillary maxima about the center of the beam. As a result, the 1.0 mm aperture attached to a Coherent Radiation power head was replaced with pinholes of either 25 or 50 μm attached to a pyroelectric detector. Depending upon the laser

output power, the 50 μm aperture was frequently so large that it produced slight saturation of the pyroelectric detector or lock-in amplifier. Therefore, much of the data was taken with the 25 μm aperture when the laser power was too large for the other size aperture. The Coherent Radiation power head could not be used because it did not have sufficient sensitivity with small apertures and because its frequency response was below the 4 Hz minimum response of the lock-in amplifier and accompanying rotating aperture. Though data were taken at many z-axis positions, those which follow were taken at a nominal 47 feet.

Figure 3 is a trace along the optical axis obtained by moving the aperture along the optical bench with the secondary mirror set for a nominal distance of 47 feet. Notice the asymmetry relative to the nominal focal plane of the nominal focal plane is caused by variation in time of the x-y position of the focal point. The beam was originally positioned in x, y and z to provide maximum power at the nominal focal plane.

Figure 4 compares the trace along the z-axis taken with and without the wire filter in place. There is a natural reduction in total power when the wire is in place but otherwise the two traces are similar.

Additional information was obtained by x and y traverses of the sampling volume at the nominal focus and at distances before and after the nominal focal plane. Figure 5 compares the x-axis traverse taken at the nominal focal plane with and without the wire filter in position. Figure 6 presents the same information for the y-axis. Other than the normal reduction in power produced when the wire is introduced, the two traces are similar for both the x and y traverses.

Figure 7, in combination with Figure 8, represents several x traverses in the nominal focal plane and in several planes before and after the nominal focal plane. Figure 7 compares the x trace in the nominal focal plane to the x traces taken at nominal half (5 in.) and quarter (7 in.) power points before focus. Figure 8 compares the x trace in the focal plane to the x trace taken at nominal half power (3 in.) and quarter power (4 in.) points after focus.

Figure 9 and 10 present corresponding traces for the y axis.

The introduction of the wire filter just prior to the secondary mirror of the telescope splits the outgoing laser beam into two distinct nearly equal components - one left and one right. Each component is made up of a large lobe of radiation separated into several smaller diffraction lobes whose intensity decreases rapidly as one progresses from the z axis outward - to right for right hand lobe - to left for left hand lobe. Data indicates that the x and y tracings taken at the nominal focal plane for the whole beam is not the simple intensity sum of the two individual half beams. It is speculated that the system is sufficiently aberrated that the individual lobes each form a non-detailed blur in the region of the focal plane. When the two lobes are simultaneously present at the focal plane, they coherently interfere to form an interference pattern with its maximum on axis and usually with side lobes.

Figure 11 presents a trace along the x axis at the nominal focal plane for the total beam and for the left and right hand halves of the beam. Either the right or left half of the beam is removed by placing a mask near the telescope. The edge of the mask coincides with the center of the shadow of the wire filter. The introduction of only one half the beam at a time is guaranteed by viewing the pattern with liquid crystal paper.

Figure 12 presents a trace along the y axis. The x-y position of the focal point has moved in this plot.

Figure 13 and 14 present the same data for a nominal half power point 2.5 inches after the nominal focal plane.

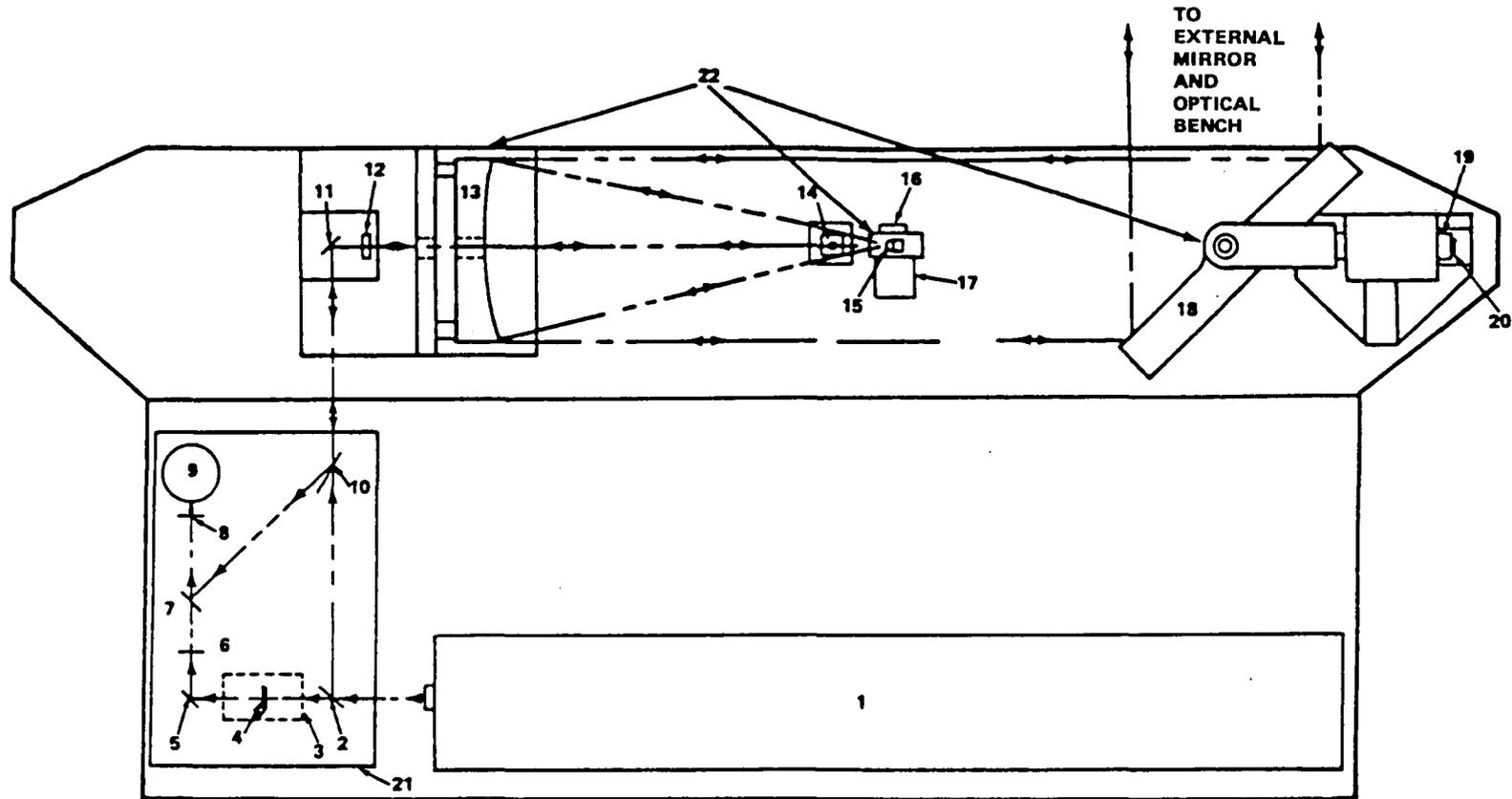
The normal sampling interval for the x, y traverses is 2.5 inches, which provides that there is no overlap of sample areas whether a 25 or 50 μm aperture is used. For the data with the unfocused beam a 500 μm (approximately 20 mil) aperture sampled every 25 mils.

CONCLUSIONS AND RECOMMENDATIONS

The incoherent sampling volume of the infrared laser Doppler velocimeter assembled in the 1979 Summer Faculty Fellowship Program was found to be heavily dependent upon the configuration of the outgoing laser beam. In general when only half the beam is allowed to focus at the focal plane, the x, y traverses indicate a broader beam with nominal side lobes. However, if the total beam is permitted to impinge on the focal plane there is substantial narrowing of the beam with increased maximum intensity and an increase in side lobes.

The double maximum indicated by the coherent sampling volume appears to be an artifact of the scanning process - not a fundamental entity of the sampling volume.

It is not known at this point how significant any of these effects are on the actual Doppler signal and its analysis. It is recommended that this information be obtained by first "calibrating" the laser output followed by automated data taking procedures to minimize the very apparent time factor which permeates all the data.



- | | |
|---------------------|--------------------------------------|
| 1. LASER | 12. QUARTER WAVE PLATE |
| 2. BEAM SPLITTER | 13. PRIMARY MIRROR |
| 3. BRAGG CELL | 14. SECONDARY BACKSCATTER ATTENUATOR |
| 4. ATTENUATOR | 15. SECONDARY MIRROR |
| 5. 45° MIRROR | 16. SERVO MOTOR |
| 6. HALF WAVE PLATE | 17. SHAFT ENCODER |
| 7. BEAM SPLITTER | 18. SCANNING MIRROR |
| 8. FOCUSING LENS | 19. SERVO MOTOR |
| 9. DETECTOR | 20. SINE COSINE POT |
| 10. BREWSTER WINDOW | 21. INTERFEROMETER |
| 11. 45° MIRROR | 22. TELESCOPE |

FIGURE 1. OPTICAL BLOCK DIAGRAM

6·AIXX

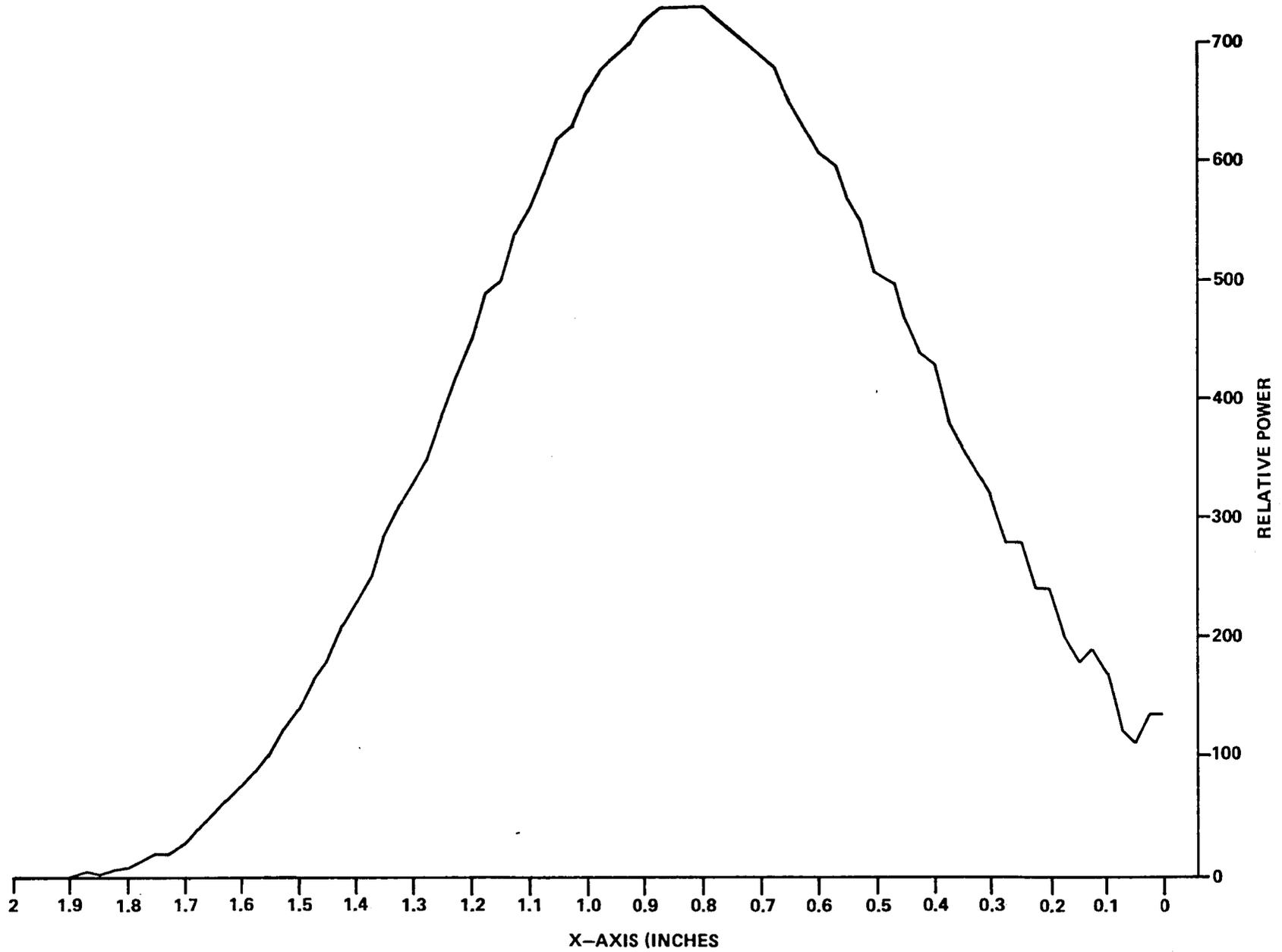


FIGURE 2. TRACE ALONG X-AXIS THROUGH UNFOCUSED LASER BEAM -- 500 μ m APERTURE

XXIV.10

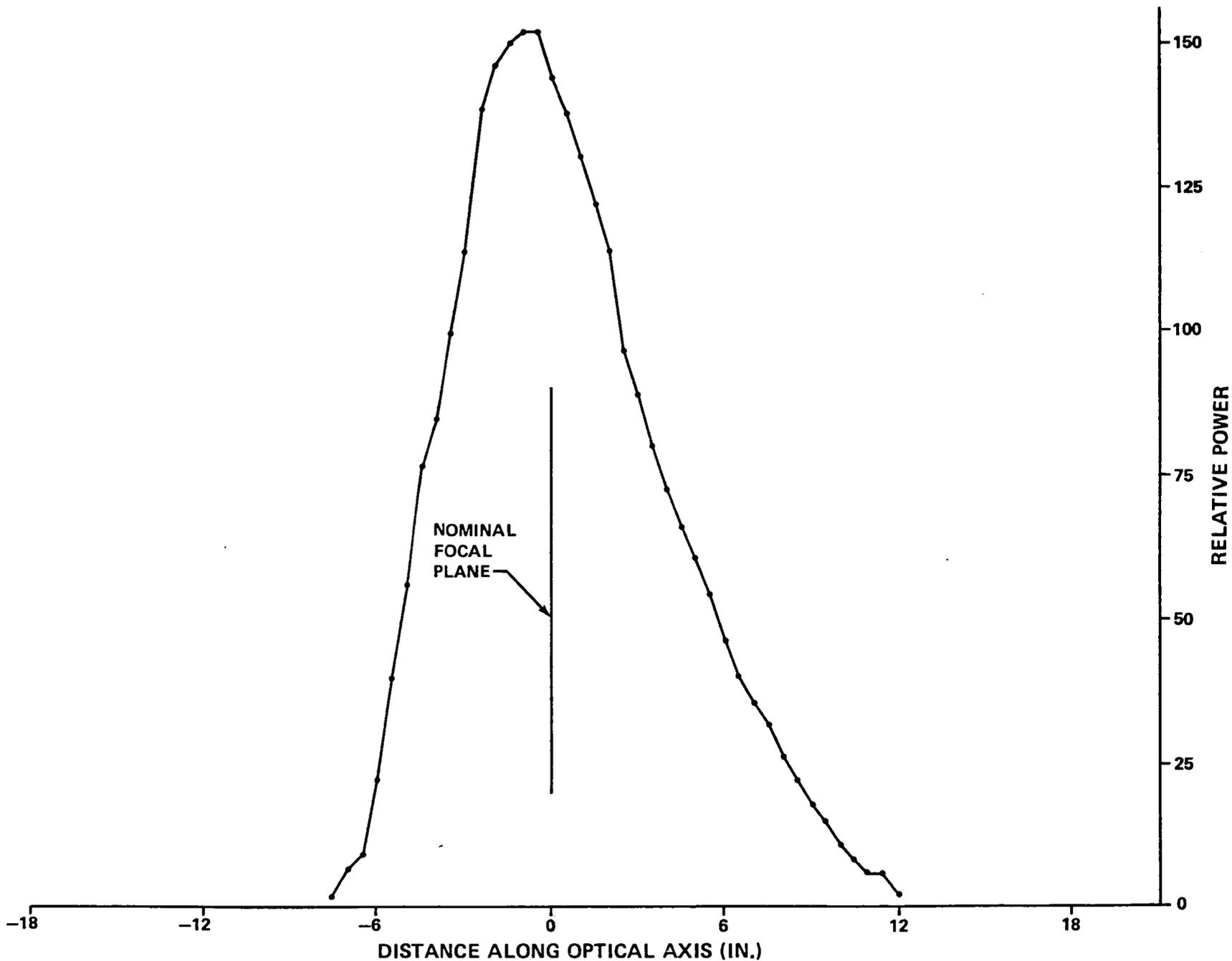


FIGURE 3. TRACE ALONG z-AXIS OF BEAM FOCUSED AT 47 FEET - 25 μ m PINHOLE

XXIV.11

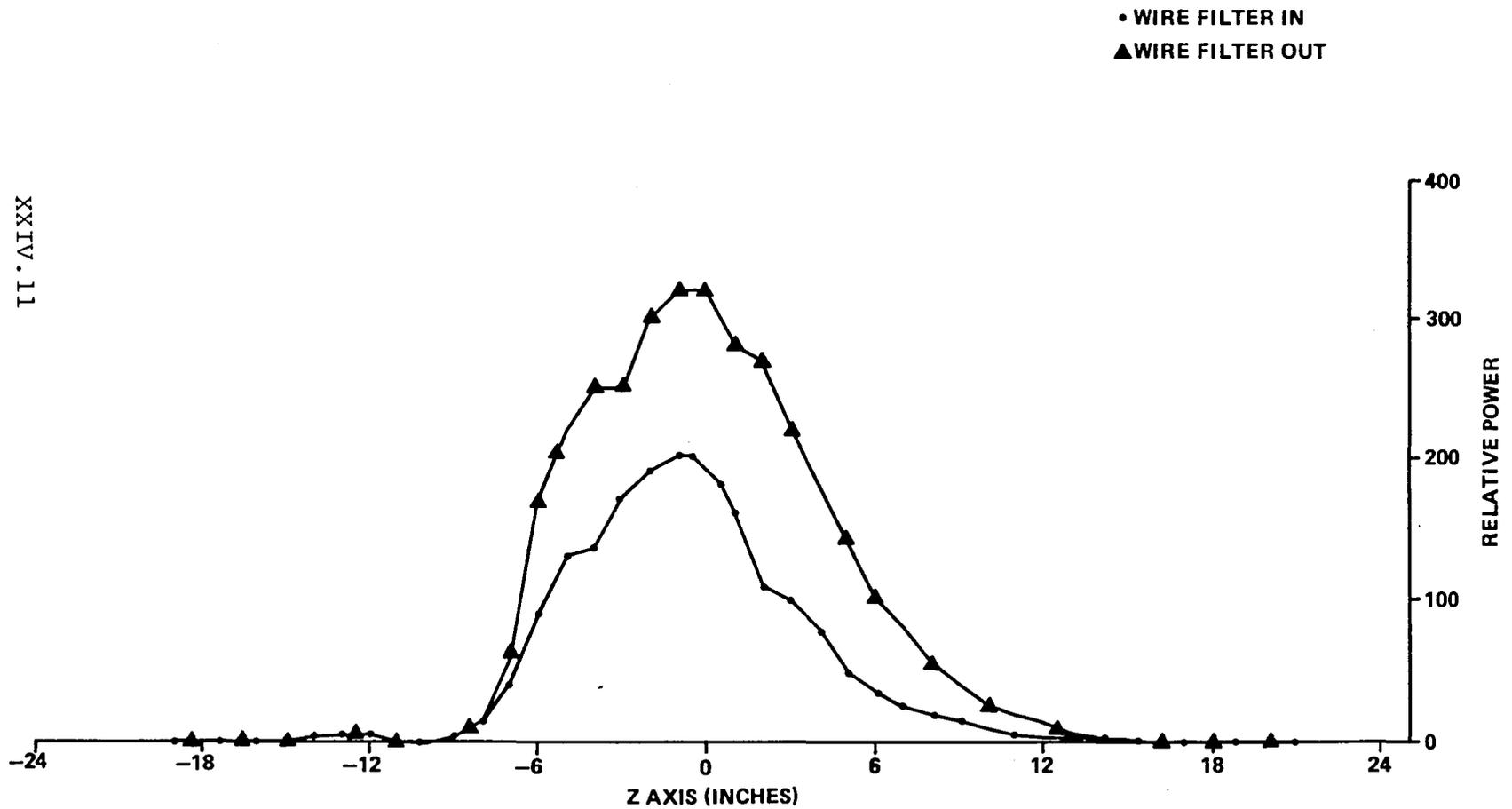


FIGURE 4. TRACE ALONG z-AXIS OF BEAM FOCUSED AT 47 FEET --
COMPARISON WITH AND WITHOUT WIRE FILTER -- 50 μ m PINHOLE

XXIV.12

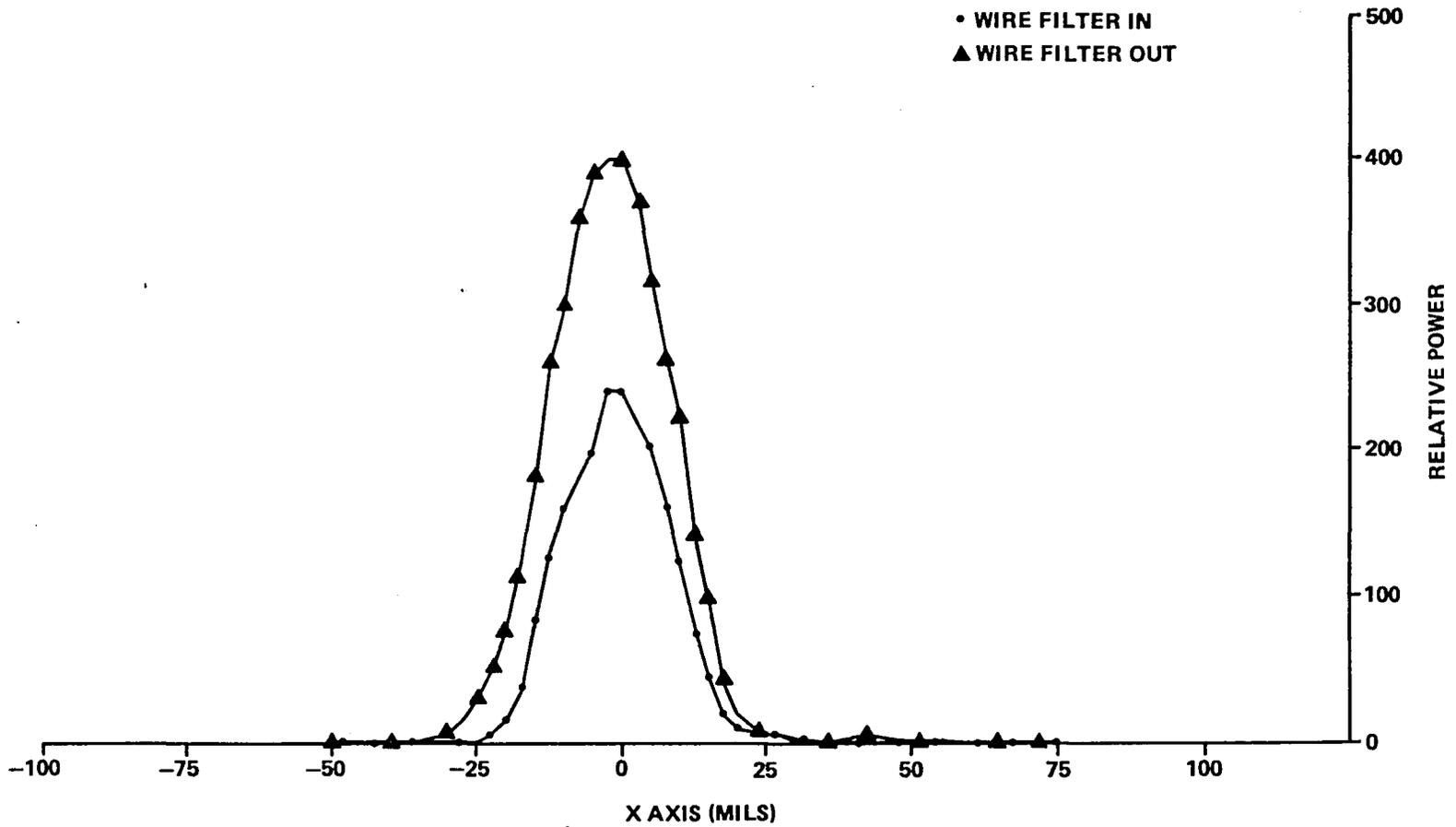


FIGURE 5. TRACE ALONG x-AXIS -- COMPARISON WITH AND WITHOUT WIRE FILTER -- 50 μ m PINHOLE

XXIV.13

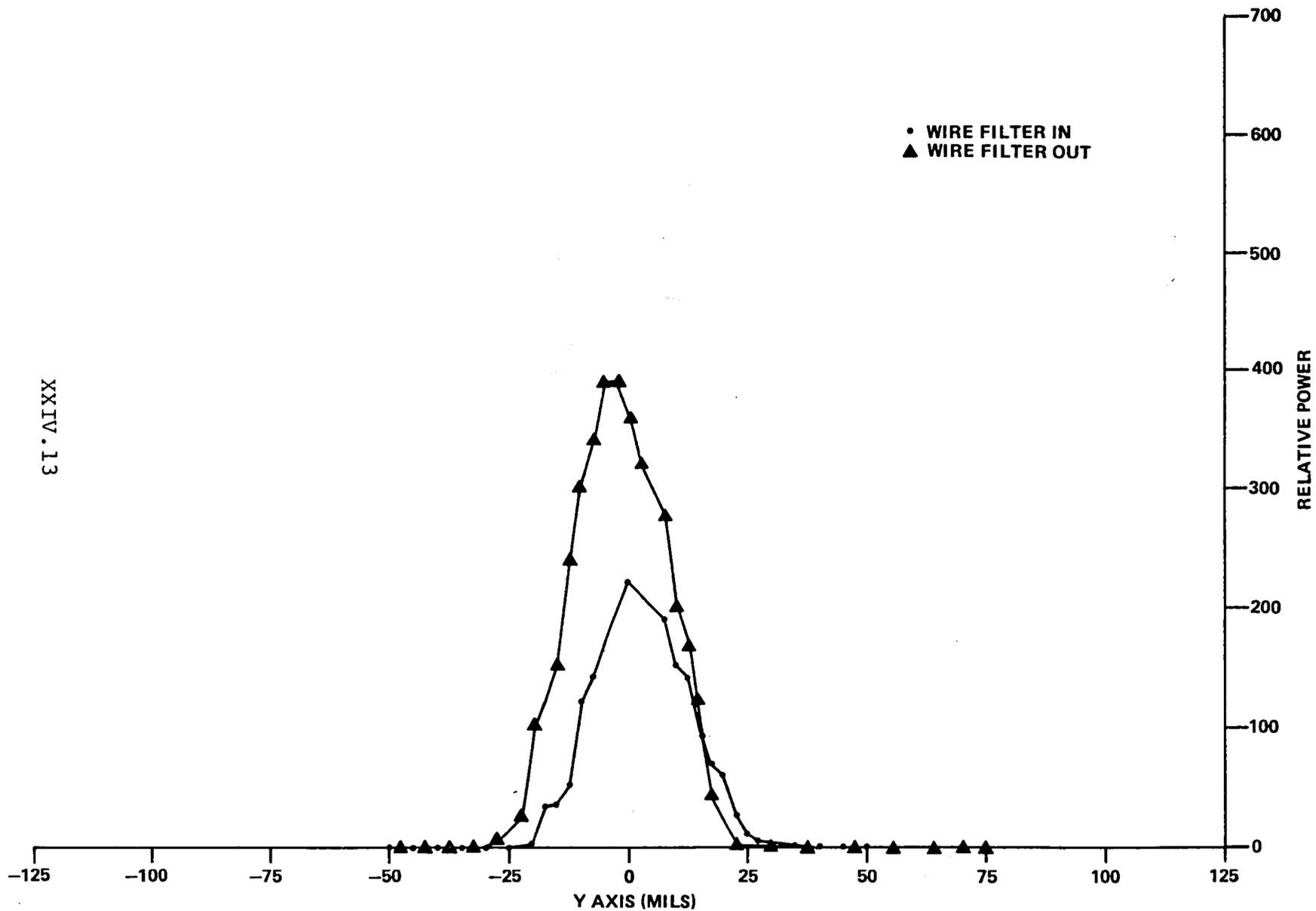


FIGURE 6. TRACE ALONG y-AXIS -- COMPARISON WITH AND WITHOUT WIRE FILTER -- 50 μ m PINHOLE

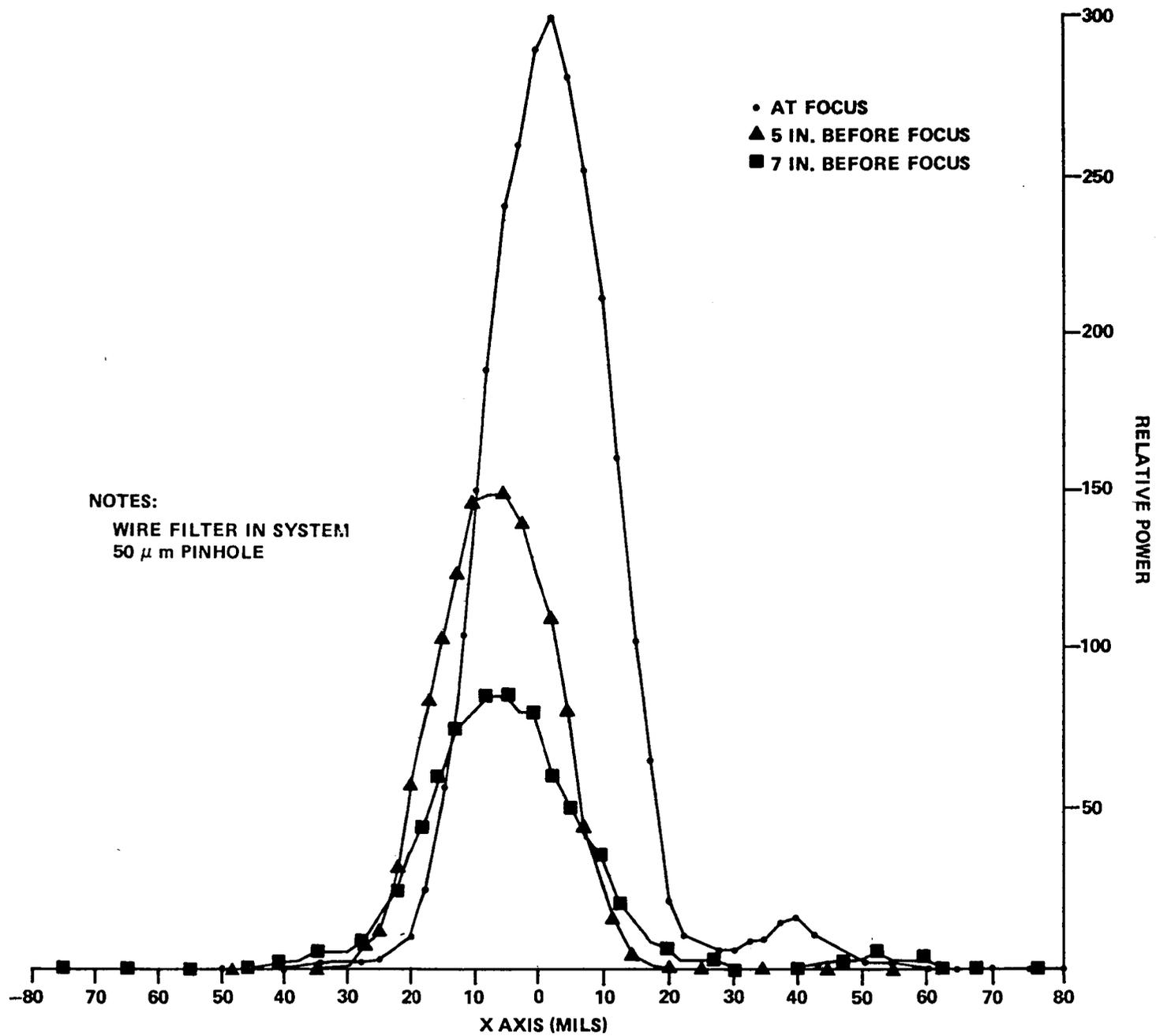


FIGURE 7. COMPARISON OF BEFORE FOCUE TRACES IN x-AXIS

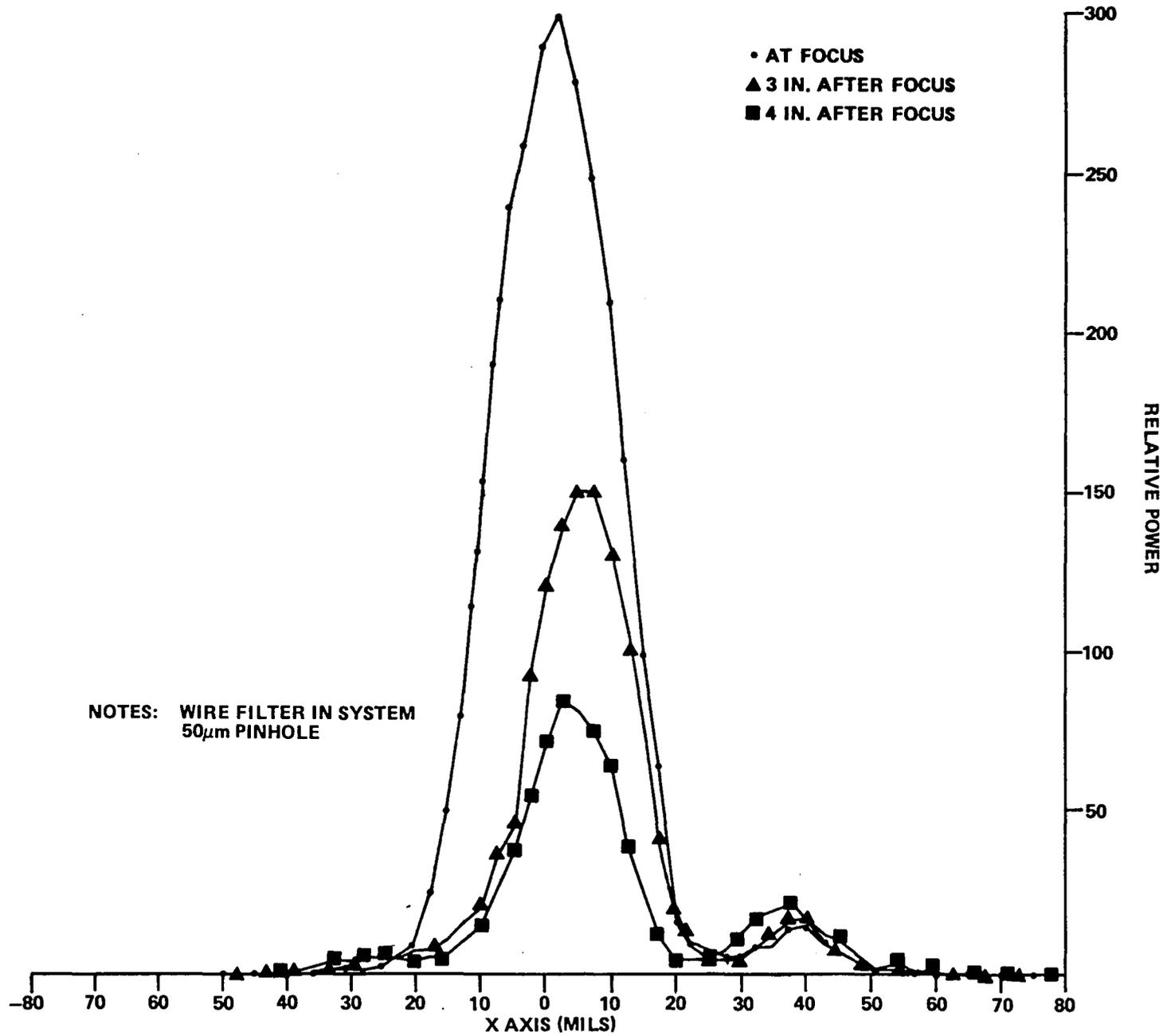


FIGURE 8. COMPARISON OF AFTER FOCUS TRACES IN x-AXIS

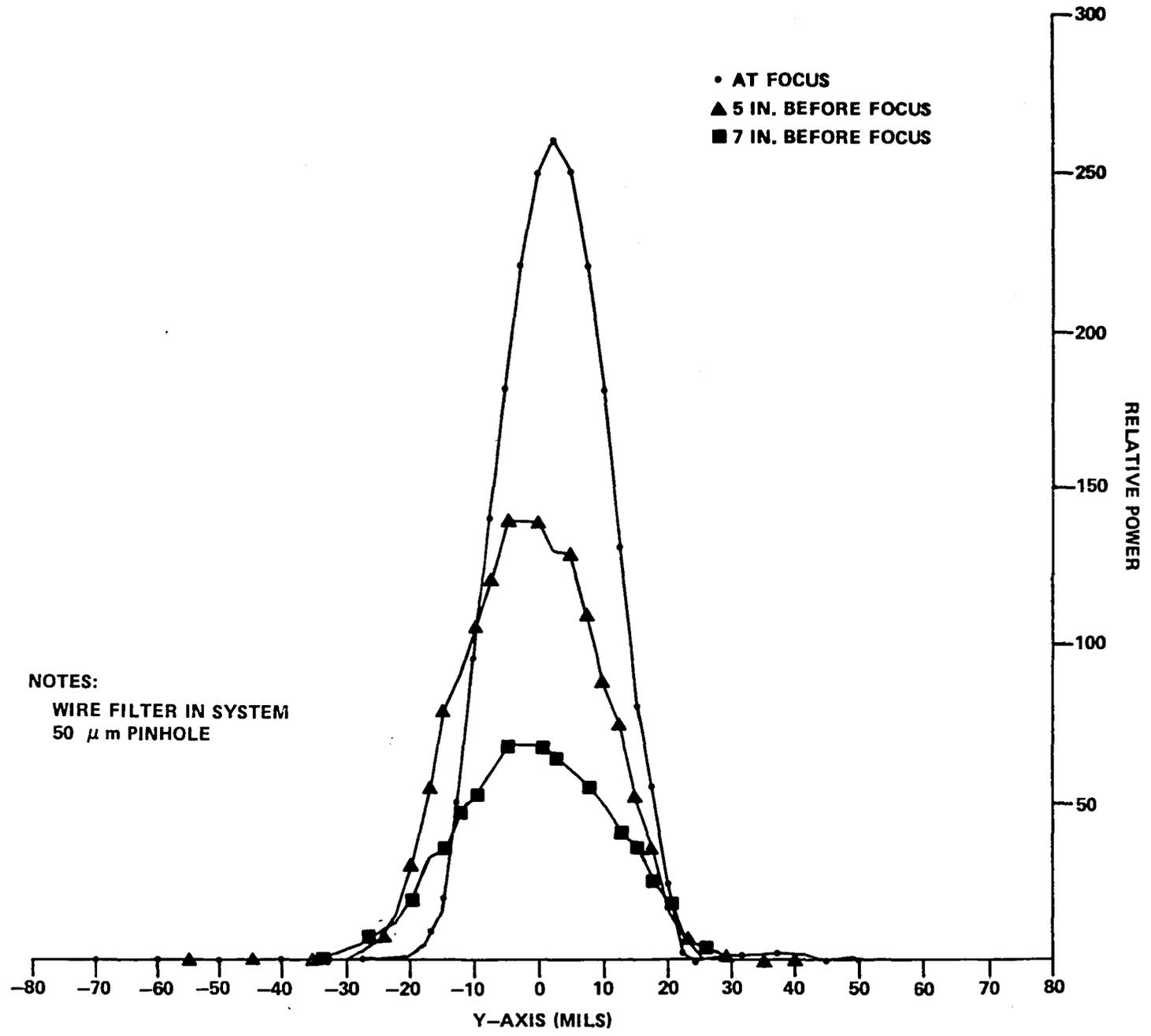


FIGURE 9. COMPARISON OF BEFORE FOCUS TRACES IN x-AXIS

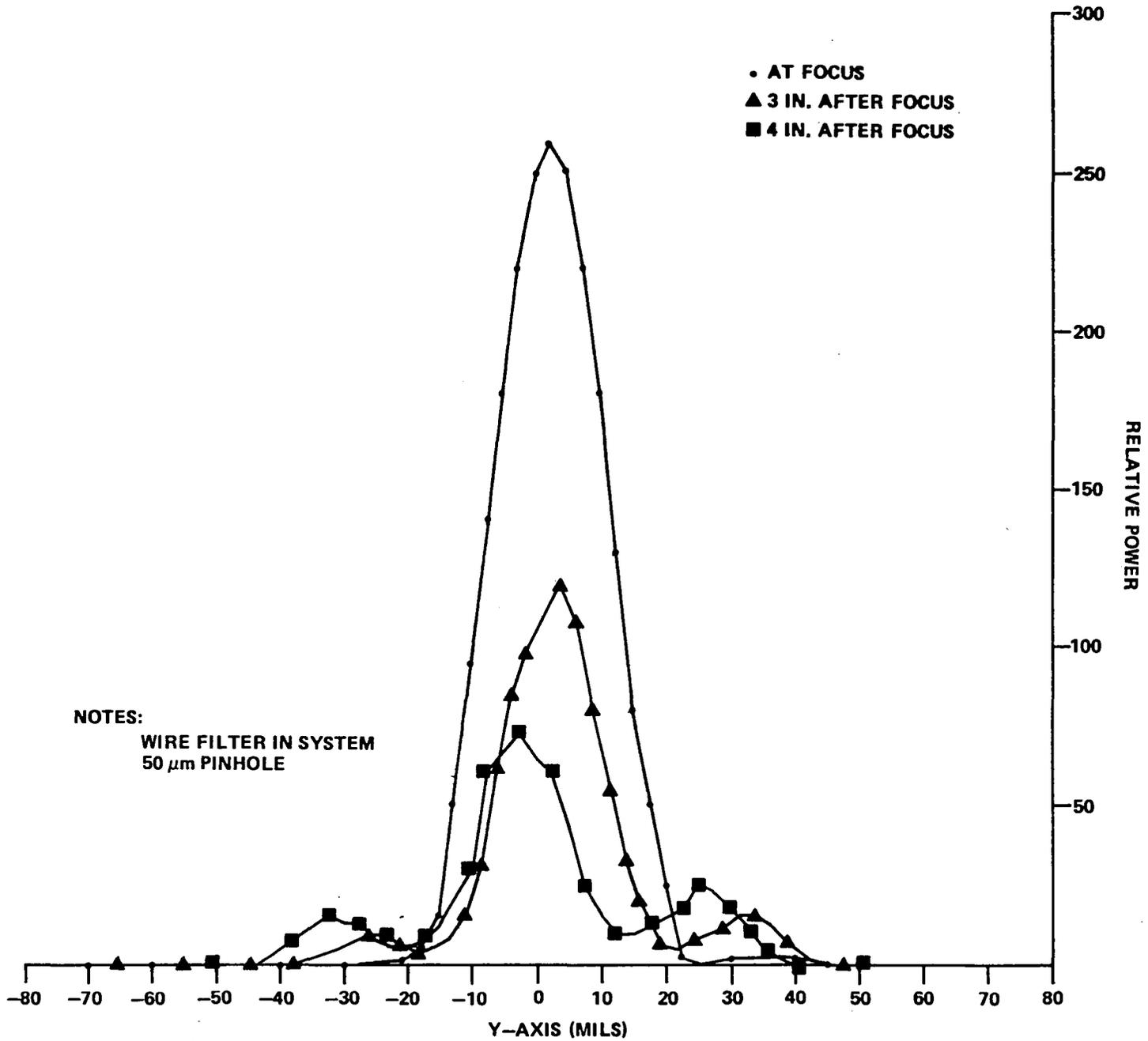


FIGURE 10. COMPARISON OF AFTER FOCUS TRACES IN y-AXIS

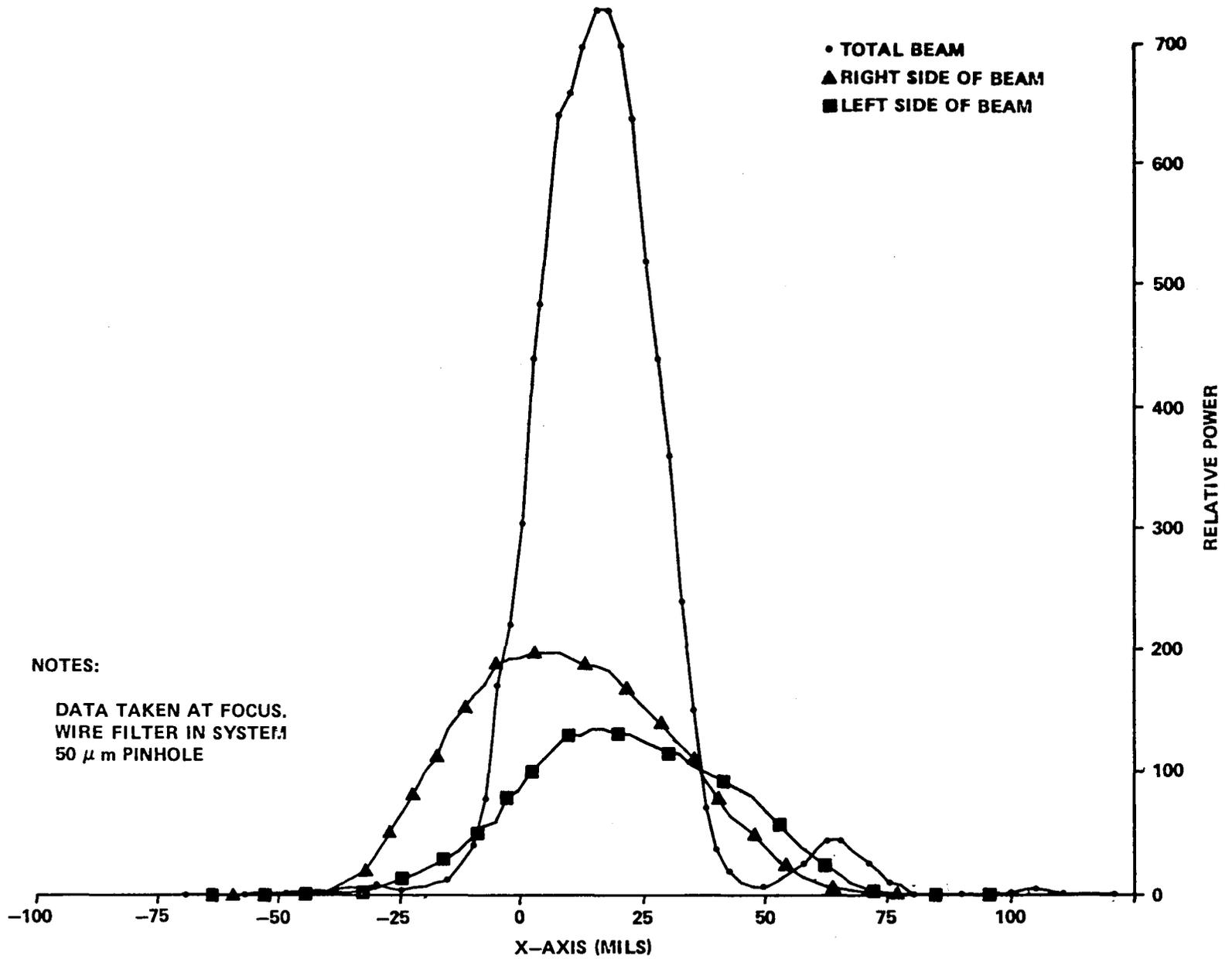


FIGURE 11. TRACES (x) IN FOCAL PLANE FOR THE TOTAL BEAM AND FOR LEFT AND RIGHT HALVES

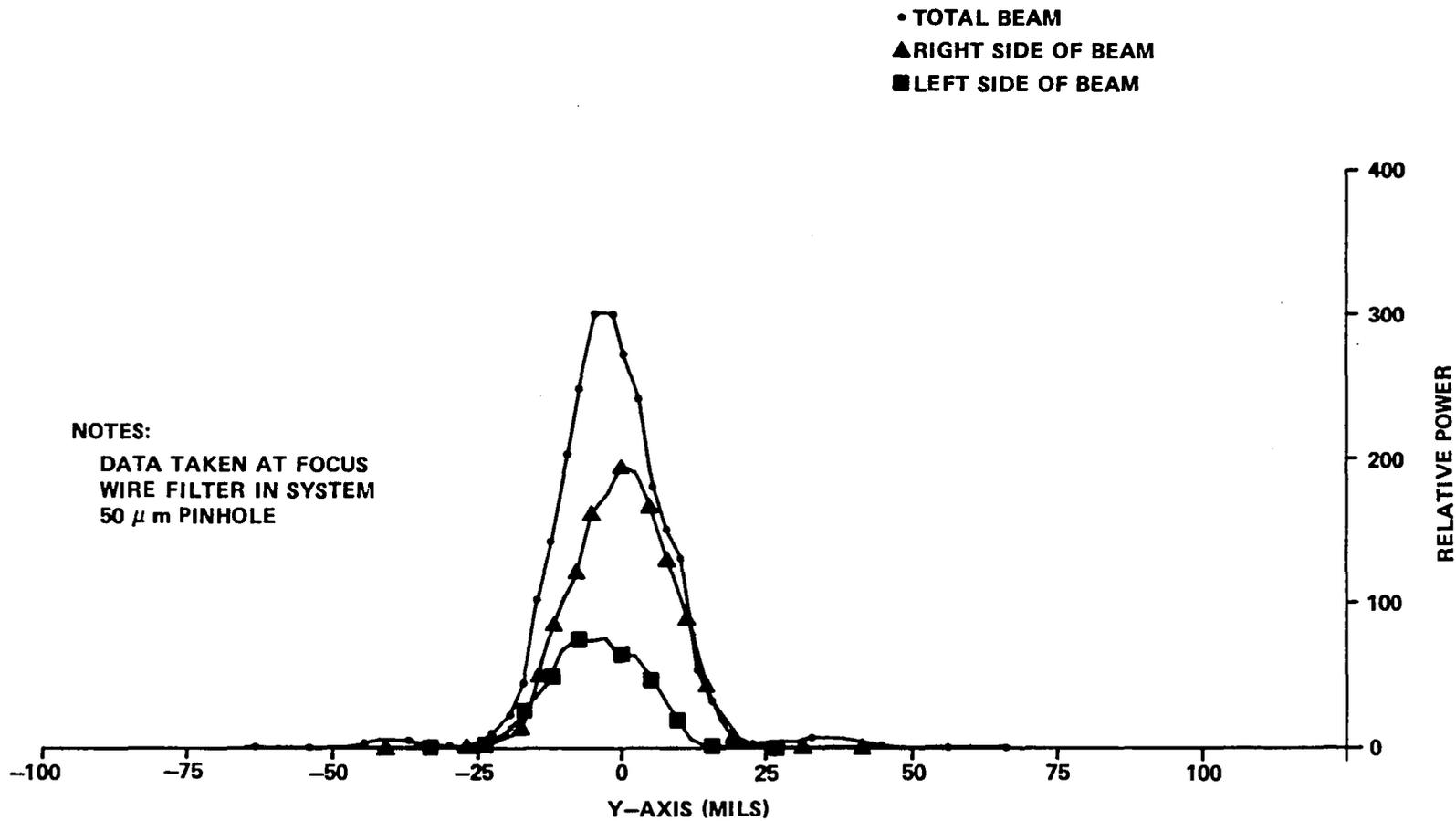


FIGURE 12. TRACES (y) IN FOCAL PLANE FOR THE TOTAL BEAM AND FOR LEFT AND RIGHT HALVES

XXIV.20

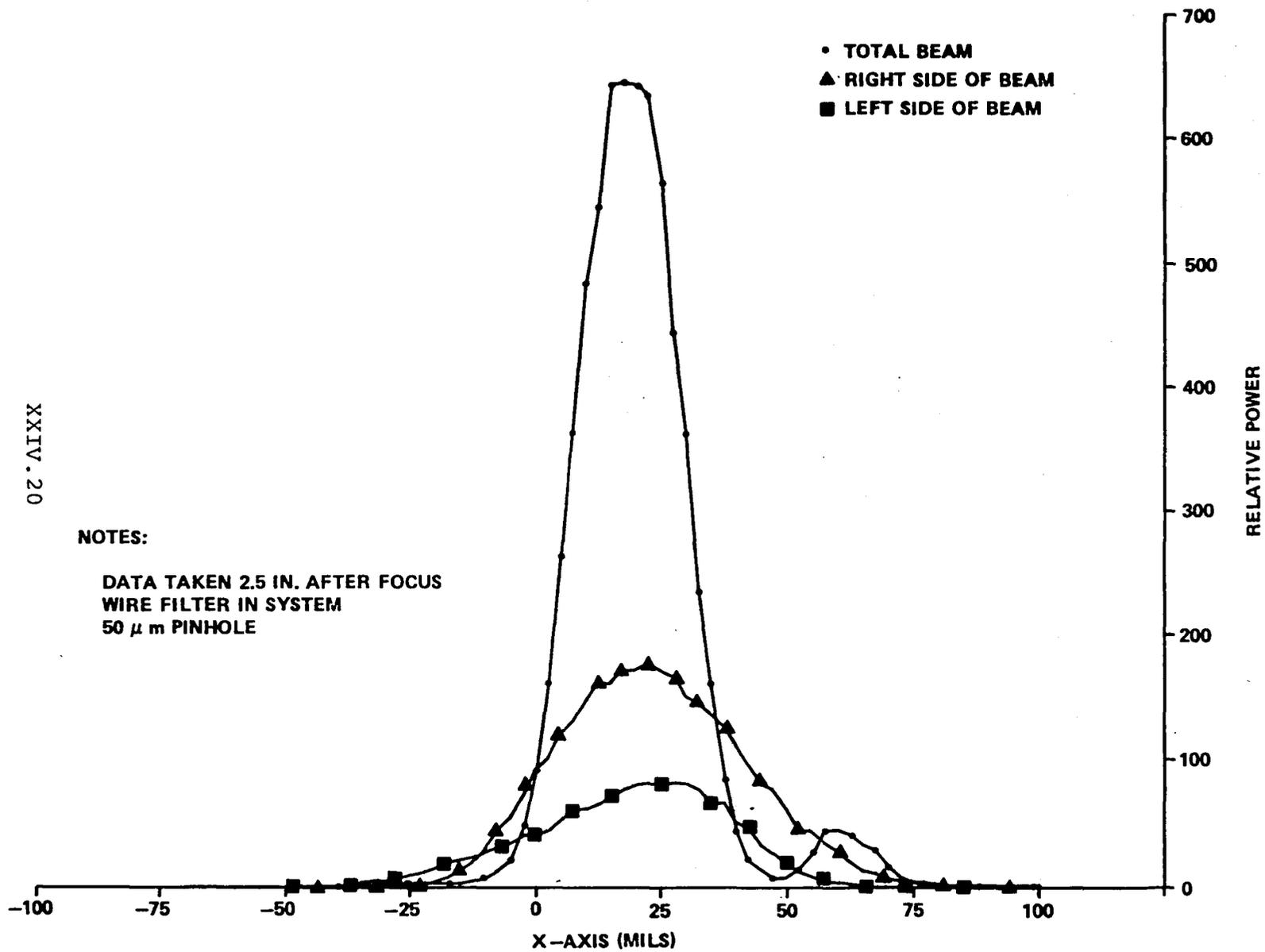


FIGURE 13. TRACES (x) 2.5 INCHES AFTER FOCUS FOR THE TOTAL BEAM AND FOR LEFT AND RIGHT HALVES

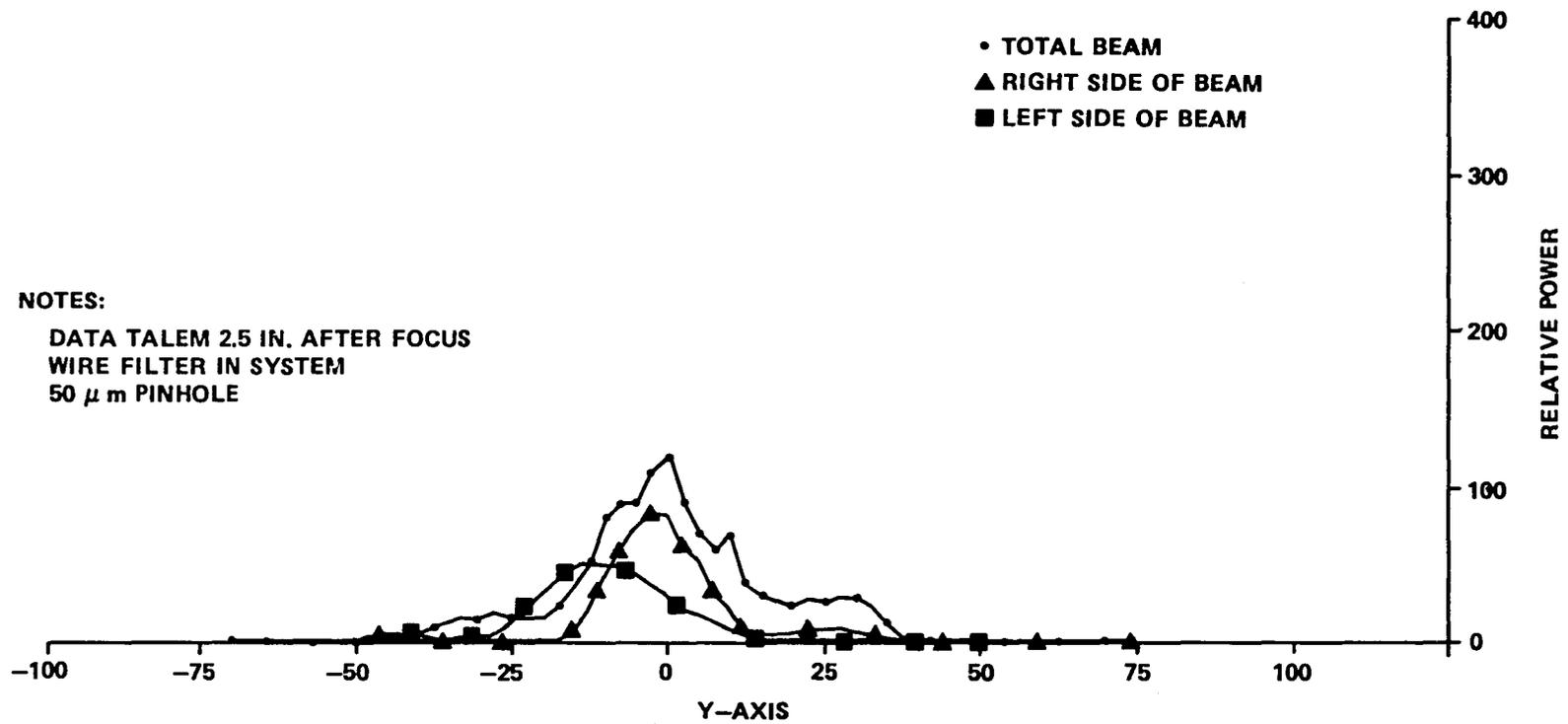


FIGURE 14. TRACES (y) 2.5 INCHES AFTER FOCUS FOR THE TOTAL BEAM AND FOR LEFT AND RIGHT HALVES

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A MONTE CARLO APPROACH TO
TOLERANCE ANALYSIS OF A NEAR-DIFFRACTION
LIMITED OPTICAL IMAGING SYSTEM

Prepared By:	W. Harold Davenport
Academic Rank:	Assistant Professor
University and Department:	Columbus College Department of Mathematics
NASA/MSFC: (Laboratory) (Division) (Branch)	Electronics and Control Optical and R.F. Systems Optics
MSFC Counterpart:	Donald B. Griner
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OF A NEAR-DIFFRACTION LIMITED
OPTICAL IMAGING SYSTEM

by

W. Harold Davenport, Ph.D.
Assistant Professor of Mathematics
Columbus College
Columbus, Georgia

ABSTRACT

Using the theory of Don Knuth, a computer program has been developed to generate random numbers on the Sigma V computer. A subroutine gives a random real number from a uniform distribution on the interval (0,1). A normal distribution with mean zero and variance 1 is also obtained. In my NASA/ASEE report for 1979, a statistical description of 5 possible errors is given. These 5 errors are S_1 which is X-axis tilt; S_2 which is Y-axis tilt; S_3 which is X-decenter; $S_4 =$ Y-decenter; and S_5 which is despace.

Combining the above results a quality function, $Q(S_1, S_2, \dots, S_5)$, is obtained via a computer program. Values of Q^2 are given in fractional wavelengths. The (variables) errors S_1, \dots, S_5 are assumed to be independent of each other with possibly different statistical descriptions. Consequently, Q is a linear function.

ACKNOWLEDGEMENTS

The author would like to express his gratitude to Dr. Bob Barfield for directing a very successful summer program. I would also like to express my appreciation to my MSFC counterpart, Don Griner, for his consultations and guidance during the summer; to Charlie Jones for the many technical discussions concerning this project; and to Julie Taylo for her programming assistance.

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I. INTRODUCTION

A Monte Carlo approach is used to determine wavefront error on a two mirror telescope. A computer program that traces N rays through an optical system and calculates the root mean square of the optical path difference (RMSOPD) is used for this analysis. An expanded version of this program will also give the Zernike coefficients. In this study only five errors are used, X-tilt, Y-tilt, X-decenter, Y-decenter, and despace of the secondary mirror. A statistical description is obtained for these five errors. Assuming a uniform probability of any one of these errors, we obtain a wavefront error.

The critical ingredient in a Monte Carlo method is a good random number generator. Knuth has developed a theory for an algorithm that generates a uniformly distributed set of pseudo-random numbers. This algorithm was programmed in FORTRAN IV and used to obtain a uniform distribution of random numbers on (0,1) and a normal distribution of mean zero and variance of 1.

II. OBJECTIVES

To develop a statistical simulation that uses the Monte Carlo techniques in the form of a computer program. The RSS method of tolerance analysis then is to be compared with the Monte Carlo method.

III. TOLERANCE ANALYSIS OF A NEAR-DIFFRACTION LIMITED OPTICAL SYSTEM

A near-diffraction limited optical imaging system is an optical system designed such that root mean square of the optical path difference (RMSOPD) is less than or equal $.25 \lambda$. In this analysis the wavelength (λ) used was 632.8 nm. A ray trace computer program written in FORTRAN IV for the Sigma V computer was used. This program can trace N ($N = 101$ to 1000) rays through an optical system and calculates RMSOPD. The optical path difference (wavefront errors) is obtained by subtracting the chief ray length from the remaining ray lengths. The chief ray is always traced by the program; ignoring the central obscuration it is used as our reference. A spherical wavefront on the image plane (focal plane) is used as the reference. The optical system is a modified Cassegrain or Ritchey-Chretien design. The exit pupil for this design is back of the secondary mirror (see Figure 1). The entrance pupil is the primary mirror.

The ray trace program has a graphics capability of plotting the spot. Figure 2 is a spot diagram of the program when no aberrations are present. Page XXV-10 gives the output that corresponds to this case. Recall that aberration describes the lack of homocentricity in a pencil of rays (input) or the deviation of a wave surface from

a spherical form. In Figure 3 we have a spot diagram with 0.1 arc min of tilt about the X-axis. The output that corresponds to this case is on page XXV-9.

The primary lens (mirror) will remain fixed. The secondary has six degrees of freedom but is symmetric with respect to rotation; hence, we only consider:

- (i) X-axis tilt (S_1)
 - 0.95 to 0.95 arc min
 - 1.7 to 1.7 arc min (centroid correction)
 - 2.9 to 2.9 arc min (best chief ray)
- (ii) Y-axis tilt (S_2)
 - Same range of numbers as X-axis tilt
- (iii) X-axis decenter (S_3)
 - 165 to 165 μm
 - 300 to 300 μm (centroid correction)
 - 530 to 530 μm (best chief ray)
- (iv) Y-axis decenter (S_4)
 - Same range of values as X-axis decenter
- (v) Despace (S_5)
 - 10.9 to 10.9

When the optical system is perturbed by one of the above five perturbations, it is the exit pupil wavefront that is operated upon.

IV. MONTE CARLO METHOD

A random number generator using the theory of Don Knuth (see reference 7) is used to give a uniform distribution on (0,1). Also see reference 2, page 240f, for a FORTRAN listing of this random number computer program. Let $Q = Q(S_1, S_2, \dots, S_5)$ be the quality function where S_i , $i = 1, \dots, 5$ are the errors given above. A Monte Carlo approach is used to obtain Figure 2. A random number is selected from the random number generator corresponding to a uniform distribution. This number is then scaled to the maximum range of the possible error for S_i . The five errors S_i are assumed to be independent of each other. Hence, one to five different uniformly distributed random numbers are obtained to give the histogram of Figure 2. The data for each of the histograms was formatted as follows:

Run No.	S_1	S_2	S_3	S_4	S_5	RMSOPD	AVG. RMSOPD
1							
2							

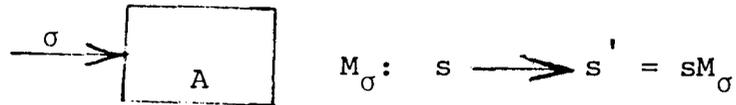
One thousand runs were made and the average RMSOPD for these 1000 runs was plotted on the Y-axis.

One thousand runs were made where X-Y tilt and X-Y decenter were given an equal probability of occurring. The tolerance range used for each of these four errors was the range that would give a $.25 \lambda$ RMSOPD wavefront error. If we take the root sum square (RSS) of these four values we get 0.5λ . From Figure 4 we see that the average RMSOPD is 0.302λ . Hence, this would imply for this scenario that the RSS technique is a more rigid tolerance than a Monte Carlo technique. So an optical system designed with respect to the RSS method would be better than one designed relative to the tolerances given by a Monte Carlo method.

Taking the RSS of two errors with a value of $.25 \lambda$ we get 0.354λ . From Figure 4 we see that the Monte Carlo method used on any combination of errors X-tilt, Y-tilt, X-decenter, and Y-decenter taken two at a time will give a range of $.158 \lambda$ to $.226 \lambda$.

V. AN OPTICAL SYSTEM VIEWED AS AN AUTOMATON

Let $A = (S, \Sigma, M)$ be an automaton



$\Sigma = \{\sigma / \sigma = \text{a unit of measure of X-axis tilt of the secondary mirror in arc min}\}$.

Let N be some positive integer such that $\sigma^N = 0$. To obtain $\sigma^2 = \sigma\sigma$ we define σ^2 to be two units of tilt, e.g., let $\sigma = 1$ arc min then σ^2 is two arc min, etc. With this binary operation Σ makes a semigroup with zero which acts like an identity. Note Σ is semigroup isomorphic to $(Z_N, +)$. $(Z_N, +)$ is the semigroup of integers modulo N with respect to addition.

$S =$ states of the system which will be the Zernike coefficients A_1, \dots, A_k in the polynomial approximation. $W(X, Y) = A_1 P_1 + \dots + A_k P_k$. Note: k is to be a fixed positive integer; $P_i, i=1, 2, \dots, k$, are the Zernike polynomials.

Now for $M_{\sigma} \in M, \sigma \in \Sigma$, define

$$M_{\sigma}: \begin{array}{l} S \longrightarrow S \\ A_i \longrightarrow A_i M_{\sigma} = A_i \end{array}$$

$$\text{so } M_{\sigma} \sim \begin{pmatrix} A_1 & \dots & A_k \\ \vdots & & \vdots \\ A_1 & \dots & A_k \end{pmatrix}$$

Choose P_1, \dots, P_k such that they are orthogonal with respect to the inner product

$$(P_i, P_j) = \int_{x^2+y^2 \leq 1} P_i P_j \quad .$$

Thus, if $M_{\tau} = \begin{pmatrix} A_1 & \dots & A_k \\ \vdots & & \vdots \\ A_1 & \dots & A_k \end{pmatrix}$ then

$$M_{\sigma} M_{\tau} = \begin{pmatrix} A_1 & \dots & A_k & \dots \\ \vdots & & \vdots & \dots \\ A_1 & \dots & A_k & \dots \\ \vdots & & \vdots & \dots \end{pmatrix}$$

$$\text{where } A_i' = A_1 + A_i$$

VI. RESULTS AND DISCUSSION

The usual method of combining wavefront errors is the root sum square (RSS). For example, using the five errors above and combining them as follows:

$$\left(S_1^2 + S_2^2 + S_3^2 + S_4^2 + S_5^2 \right)^{\frac{1}{2}}$$

Also, another method is to combine those by a Monte Carlo technique. If S_5 , despace, is not considered then we obtain Figure 4. Hence, an optical system designed to operate within the tolerance criterion of the RSS method would also satisfy the tolerance conditions of the Monte Carlo method.

The maximum bounds of the error on focus were approximately -10 to 10 μm . The focus proved to be a critical error in this analysis and further study is required.

Problems to consider:

1. Develop algorithms for an optical system considered as an automaton.
2. Extend this Monte Carlo method of tolerance analysis to more complicated optical systems.
3. Develop an algorithm and program it in FORTRAN for a Zernike coefficient program (see Davenport, NASA/ASEE summer report, 1979).

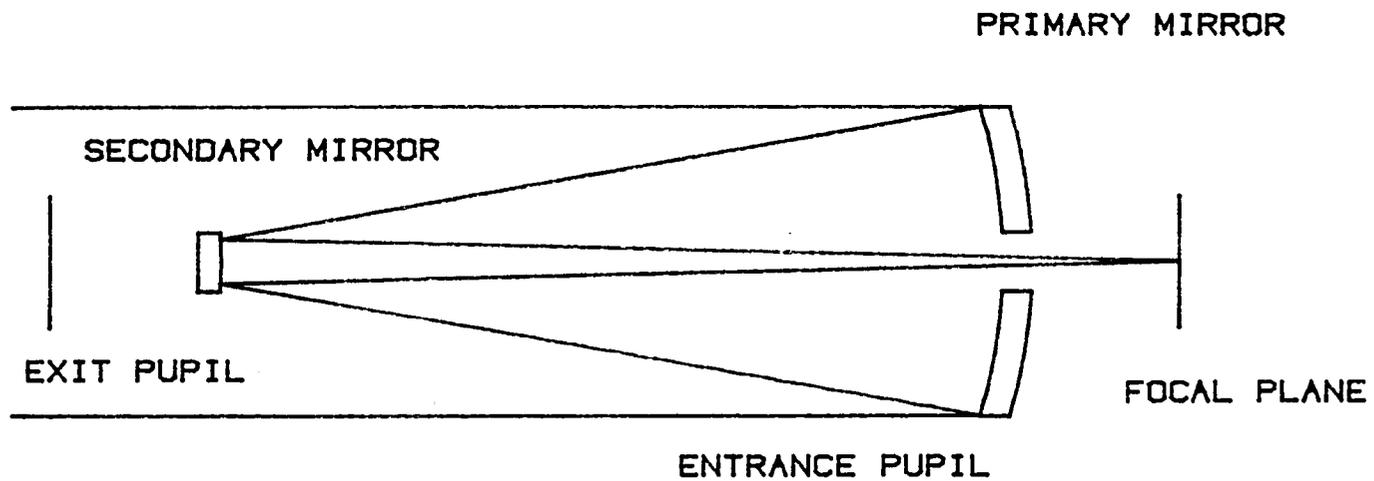


FIGURE 1 RITCHEY-CRETIAN OPTICAL SYSTEM

ST 2.4M CONFIGURATION
SPOT DIAGRAM AT SURFACE # 5 RMS OPD = .00001

CENTROID AT X = .00000000E 00
Y = .00000000E 00
Z = .00000000E 00

BOX IS .00000200 CM SQUARE CENTERED ON CENTROID

PAUSE

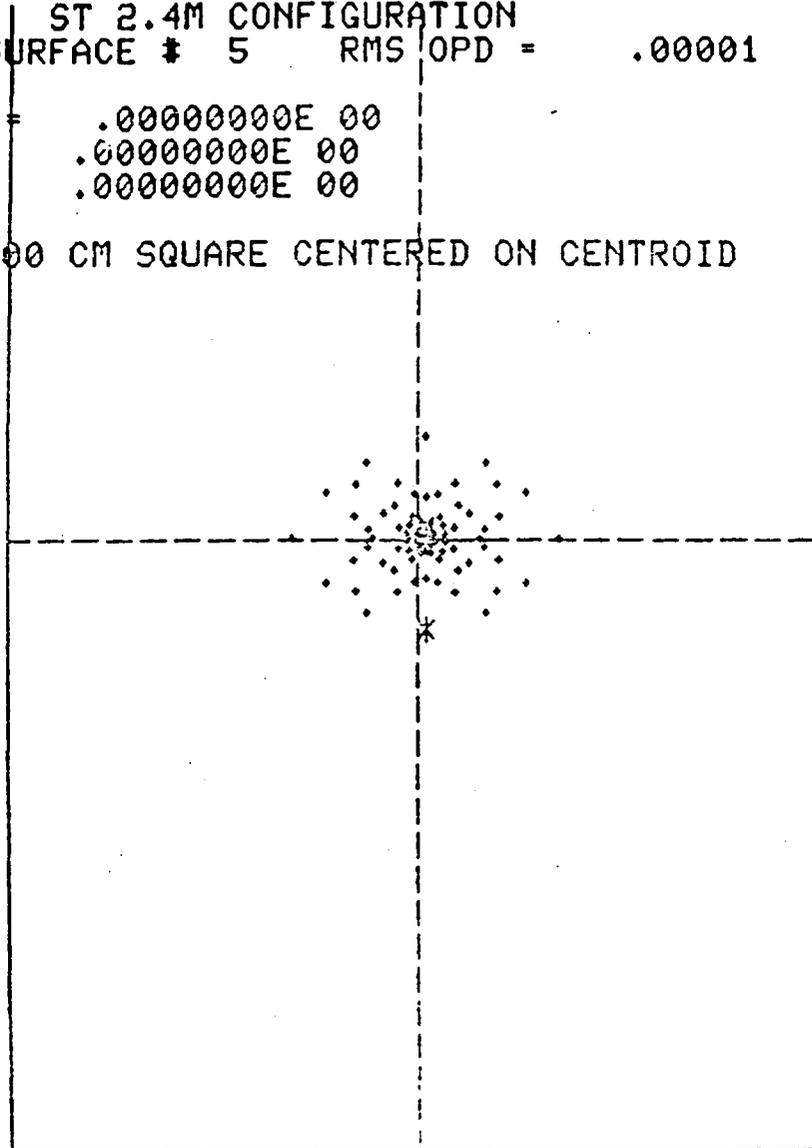


Figure 2. No Aberrations

ST 2.4M CONFIGURATION
SPOT DIAGRAM AT SURFACE # 5 RMS OPD = .24316

CENTROID AT X = $-.33293707E\ 00$
Y = $.00000000E\ 00$
Z = $.00000000E\ 00$

BOX IS $.00800000$ CM SQUARE CENTERED ON CENTROID

PAUSE

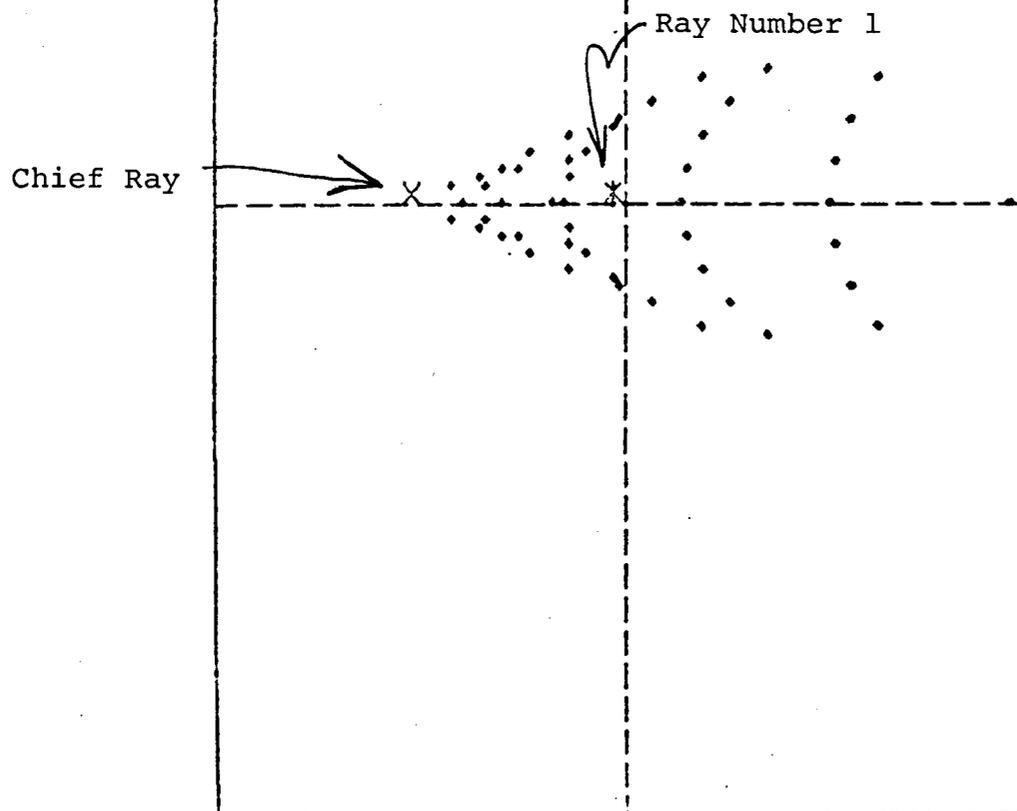


Figure 3. 0.1 Arc Min Tilt About X-Axis

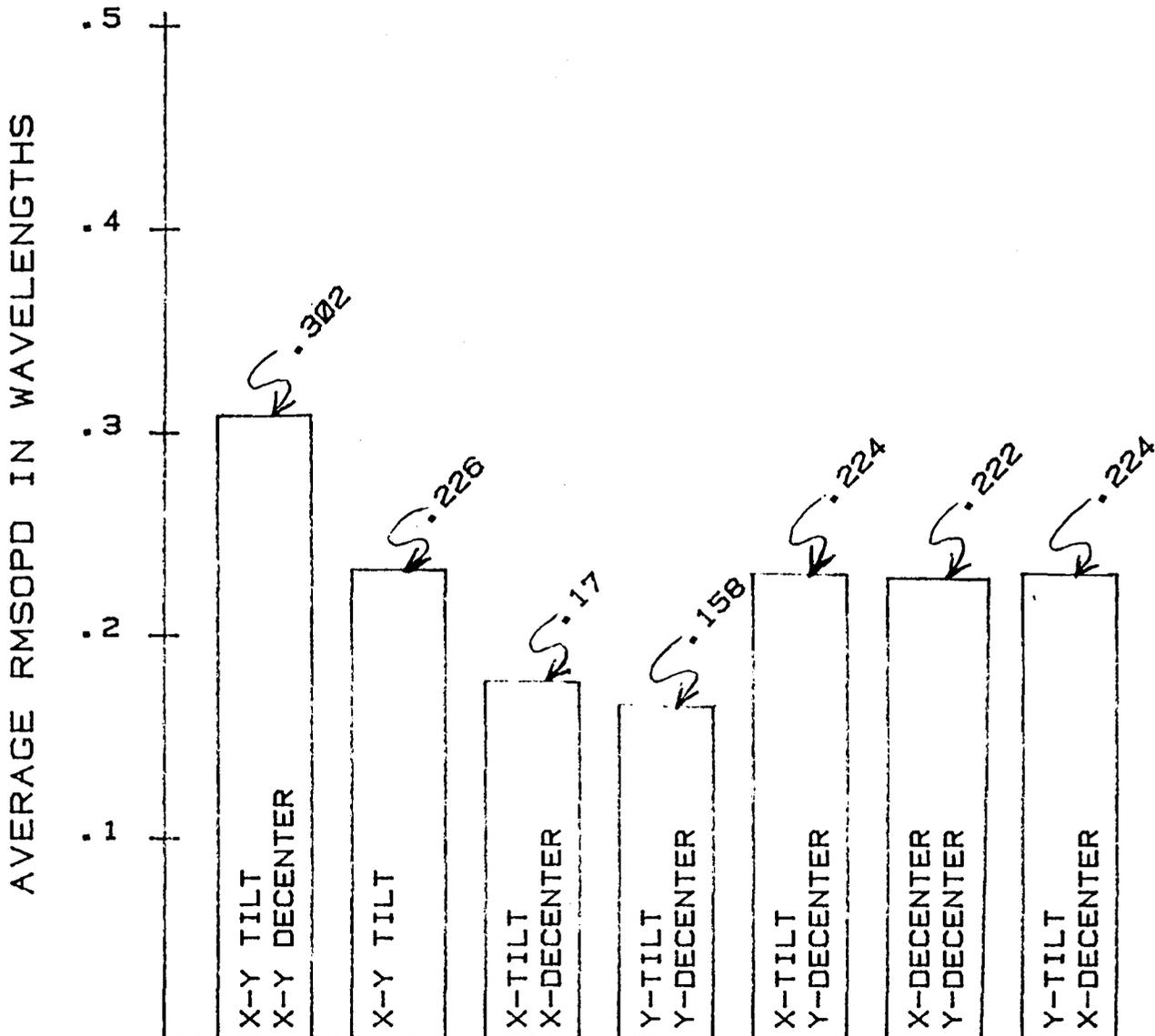


FIGURE 4 HISTOGRAM OF AVERAGE RMSOPD UNIFORM DISTRIBUTION OF 1000 POINTS.

ST 2.4M CONFIGURATION, FIELD PT. .100, .000
101 RAYS

RMS OPD .2432
STREHL -1.334
CENTROID:
X -.3329
Y .0000
Z .0000
CHIEF RAY:
X -.3351
Y .0000

RANK 10
P1 = -.1079021E-01

ZERNIKE COEFFICIENTS

P2	P3	P4	P5	P6
X TILT	Y TILT	FOCUS	ASTIG(45)	ASTIG(0)
-1.46027	-.264879E-07	-.496520E-02	.163329E-18	-.642378E-04
P7	P8	P11	P12	
Y COMA	X COMA	3D-ORDER	5TH-ORDER	
.182050E-07	-.495005	.141003E-04	.170063E-05	

DESIGN PARAMETERS

#	D	E	R	QN
1	10.00000000000000	-1.00228977203300	-1102.500000000000	-1.00000
2	DUMMY			
3	-490.00000000000000	-1.49599512243700	-135.464362850970	-1.00000
4	DUMMY			
5	640.00000000000000	-1.0000000000000000	1000000000.00000	1.00000

R1 = 120.000 R4 = 699.5070000000000
EE = .340000 FL = 5760.00
QL0 = .632800E-04 NRAS = 13

STOP 0

Table 1. Zernike Coefficients with 0.1 Arc Min Tilt About the X-Axis

ST 2.4M CONFIGURATION, FIELD PT. .000, .000
101 RAYS

RMS OPD .7330E-05
STREHL 1.000
CENTROID:
X .0000
Y .0000
Z .0000
CHIEF RAY:
X .0000
Y .0000

RANK 10
P1 = .3218636E-04

ZERNIKE COEFFICIENTS

P2	P3	P4	P5	P6
X TILT	Y TILT	FOCUS	ASTIG(45)	ASTIG(0)
-.409722E-12	-.708786E-11	.308430E-04	-.847167E-21	.298539E-12
P7	P8	P11	P12	
Y COMA	X COMA	3D-ORDER	5TH-ORDER	
.251956E-12	.129803E-11	.116445E-04	.163000E-05	

DESIGN PARAMETERS

#	D	E	R	QN
1	10.00000000000000	-1.00228977203300	-1102.500000000000	-1.00000
2	DUMMY			
3	-490.00000000000000	-1.49599512243700	-135.464362850970	-1.00000
4	DUMMY			
5	640.00000000000000	-1.00000000000000	100000000.00000	1.00000

R1 = 120.000 R4 = 699.5070000000000
EE = .340000 FL = 5760.00
QL0 = .632800E-04 NRAS = 13

STOP 0

XXV-10

Table 2. Zernike Coefficients With no Aberrations

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THE UNIVERSITY OF ALABAMA

POWER DISTRIBUTION FOR ELECTRON BEAM WELDING

Prepared by:	Eugene Edwards, M.S.E.E.
Academic Rank:	Assistant Professor
University and Department:	Alabama A. & M. University Department of Engineering and Engineering Technology.
NASA/MSFC: (Laboratory) (Division) (Branch)	Materials and Processes Process Engineering Metals Processes
NASA Counterpart:	Arthur C. Nunes, Ph.D.
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POWER DISTRIBUTION FOR ELECTRON BEAM WELDING

By

Eugene Edwards, M.S.E.E.
Assistant Professor of Electrical Engineering
Alabama A&M University
Normal, Alabama

ABSTRACT

Electron beams are being used to apply heat to metals for welding purposes.

In this report, the power distribution in the beam is analyzed. Experimental and digital computer techniques are used to evaluate the radial distribution of power detected by a wire probe circulating through the beam. Analytical models developed at MSFC of the Electron Beam-Work-piece interaction contain two weld loss parameters in a function of the radial power distribution. The results from this research are intended to yield information on the mechanism by which these losses take place.

Additional information on the second loss mechanism is provided by measuring metal weight losses during welding. The power wasted on these weld losses can thus be estimated and distributed to obtain a more accurate weld penetration computation.

ACKNOWLEDGEMENTS

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NOMENCLATURE

AV	- accelerating voltage
BC	- beam current
BF	- beam focus
D	- diameter of the irradiated target
F	- in welder's computer program, focus
FIL	- filament
I	- beam current
I_e	- emission current
°K	- degrees Kelvin
L	- length of the wire probe
P	- perveance
P_c	- power lost to the crucible by conduction
P_d	- power dissipated
P_e	- power loss for electrons not reaching the target
P_{eff}	- effective power
P_g	- power of the gun
P_i	- power developed in the electron beam
P_1, P'_1	- convergence points
P_L	- power loss due to latent heat
P_n	- loss of power to the evaporant
P_r	- power lost by heat radiation
P_v	- power lost by electrons not striking the target
P_x	- loss of power due to X-ray production
POT	- potentiometer
R	- brightness of the beam
T	- cathode temperature
V_a	- acceleration voltage
d_a	- cathode diameter at point (a)
d_f	- beam's spot diameter
d_k	- cathode diameter at k
d_m	- smallest diameter of the beam
e	- electron charge of 1.6×10^{-9} coul.
eV_0	- probable thermal energy

- j_F - current density of the beam at the target
- j_k - current density for beam current (J) at the cathode
- k - constant, such as the k^{th} term, also K, as a constant
- k_b - Boltzman constant
- p - power density at the target
- p'_F - high power density
- α - alpha, space-charge-limited current flow function
- α_L - aperture at the target
- γ - gamma, aperture angle of marginal rays between the anode
- $\bar{\gamma}$ - aperture of marginal rays at the position of the smallest beam cross section
- Γ - gamma, thickness of the wire probe
- ω - omega, width of the wire probe
- ϕ - phi, angle of current signal for computer input
- π - pi, ≈ 3.14
- ρ - rho, beam target distance from motor-plate to some point on the probe
- θ - theta, aperture angle of the cathode beam generator

OBJECTIVES

The objective of this study is to investigate the power distribution of the electron beam in order to understand the loss mechanisms associated with the beam-workpiece interaction. Another objective of this research is to apply techniques that will yield distribution and profiles of the beam such that power losses may be carefully examined.

INTRODUCTION

Electron Beam Welder (Sciaky Bros. Inc., Chicago, Illinois)

In an electron beam process, the part is welded by the Kinetic energy of a dense beam of accelerated electrons. As the electrons strike the workpiece, their Kinetic energy is converted to the thermal energy causing the metal to melt and fuse.

The device which emits, accelerates, and focuses the electrons on the workpiece is referred to as the electron beam gun. The gun can be mounted on a carriage to provide several axes of motion, or it can be held stationary while the work is driven, or both the gun and the work can be driven. See Figure 1 below for a simplified illustration.

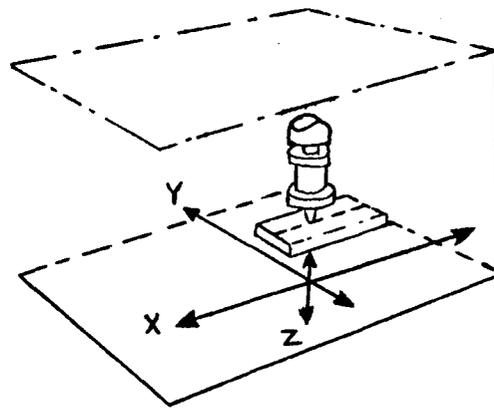


FIGURE 1 - Illustration of Travel Direction

The gun assembly is housed in a chamber which is pumped down to a fine vacuum level, usually to about 1×10^{-4} torr. or less. By operating in a vacuum the life of the filament (Figure 2) is protected and the electrons can be focused into a fine dense beam. The filament which is heated to a high temperature would be destroyed if exposed to air at the operating temperature. If there were a sufficient concentration of loose molecules in the chamber, the electrons would collide with these molecules. This would cause the beam to be diffused and the electrons to lose some of their Kinetic energy.

In Figure 2, the voltage of the bias allows the beam current to be changed independent of the accelerating potential (within the operating limits of the gun). Irrespective of the bias level, the maximum beam current cannot exceed the maximum current range of the gun. This range is determined by the mating elements installed in the gun. Maximum current is obtained at 0 bias. As the bias is increased, the current is reduced.

XXVI-10

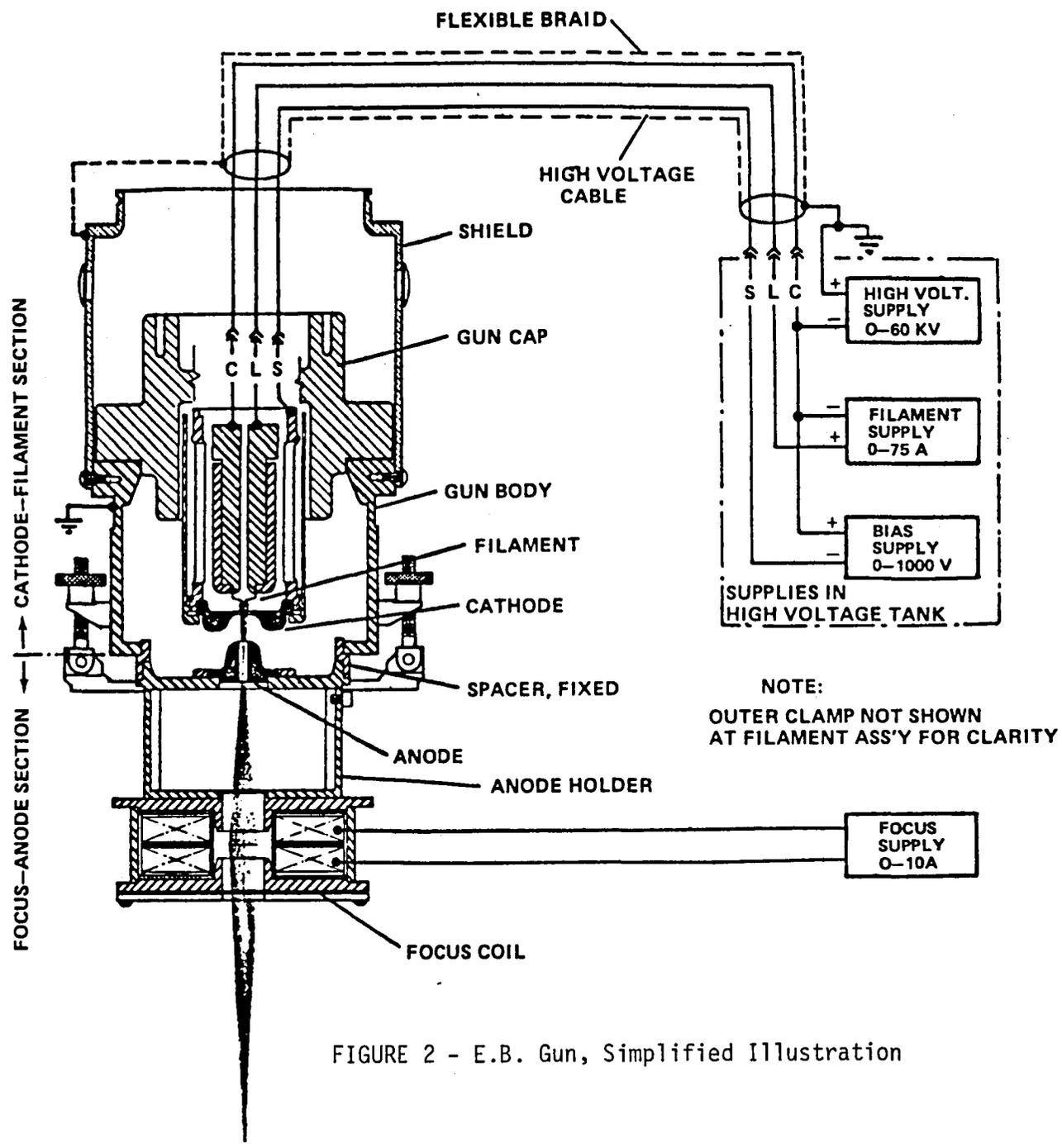


FIGURE 2 - E.B. Gun, Simplified Illustration

The spacing between the filament, cathode, and anode determines the internal resistance of the gun. This resistance is non-linear and varies directly with the distance between these elements. As the distance is increased, the beam current is decreased.

The model E-S9460 gun has 2 major ranges; 250 mA and 500 mA. A mating filament and cathode are provided for each range. In each of these 2 ranges, additional ranges can be obtained by inserting a spacer between the 2 main sections. The same anode is used in both major ranges.

Ranges of Gun E-S9460

Range : 500 mA	Range : 250 mA
Filament: C-K2501	Filament: C-G8615
Cathode : B-K2501	Cathode : B-H844
**Anode : B-M5740	**Anode : B-M5740

For additional ranges use:
350 mA W/0.050" spacer

For additional ranges use:
150 mA W/0.100" spacer
100 mA W/0.200" spacer
50 mA W/0.500" spacer

For application requiring shallow penetration with narrow welds at low currents, a small hole cathode is used.

Electron beam welds possess several unique characteristics:

- (1) Welds are produced in many cases 15 to 20 times narrower than comparable TIG/MIG welds. Notice the dotted isothermal lines of typical Tungsten Arc welding compared to Electron Beam welding (Figure 3).

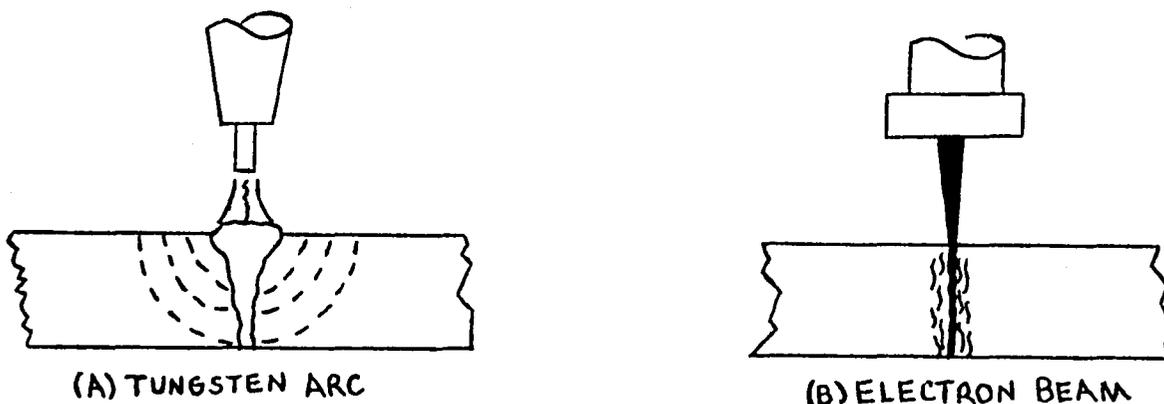


FIGURE 3 - Weld Illustrations

**The same cathode is used in both major ranges.

- (2) Less thermal energy is transferred to the unmelted base metal and thus reducing residual stresses, warping, and increasing the strength of the weld.
- (3) Absorption of atmospheric gasses by the weld is eliminated by the vacuum system, thus highly oxidizable materials can be welded (Zirconium, Beryllium, etc.).

POWER RELATIONS

The output power from the electron beam gun follows the general power formula of the form

$$P_g = V_a I_e, \quad (1)$$

where, P_g is the gun power

V_a is the acceleration voltage

and I_e is the emission current.

Power, P_d , is lost as the electrons are taken off of the anode. This dissipated power must be subtracted from the gun power.

Since all of the electrons will not reach the welding target the gun power becomes even smaller. The beam focus plays an important role in allowing more electrons to reach the target. Now, the effective power becomes

$$P_{eff} = P_g - P_d - P_e, \quad (2)$$

where P_e is the power lost due to electrons not reaching the target.

For certain applications, a low precision gun with a diffused output beam may allow a little more than 50% of the emitted electrons to reach the target [1].

Energy is lost when the electrons collide with system components and target samples. Smith, reference [1], attribute some of this energy loss to:

- o excitation and ionization of electrons, by electron-electron interaction, and displacement of atoms with the lattice by energy transferred to individual atoms. This occurs at energies of a few-hundred KEV.
- o X-ray production by resonance absorption occurs at the high energy range. These losses are minor.
- o inadequate focusing method creating minor loss of electrons from the beam.

- o vapor between the welding sample and gun can cause minor losses due to collisions.
- o back scatter of electrons reflected produces the main source of loss (25%).

It is generally safe to assume that the effective power that is converted to heat the sample is

$$P_{\text{eff}} = 0.60 P_g. \quad (3)$$

Note that equation Eq. (3) is approximately the equation for average power. Information on evaporation losses will be presented later in this paper.

The important relationship between beam power and quality of the weld penetration can be obtained for various metals, metal sizes, and welding speeds. A computer routine is available that gives results similar to those shown in Figure 4 (NASA-MSFC).

A summary of the power-energy distribution is given by Hunt and Hughes [2] as the process heat balance:

$$P_i = P_v + P_r + P_L + P_n + P_x + P_c, \quad (4)$$

where, P_i is the power developed in the electron beam by acceleration to the anode voltage.

P_v is the power lost by electrons striking other than the target; e.g., filament-to-anode current (negligible).

P_r is the power lost by heat radiation from the liquid evaporant surface (hard to determine).

P_L is the power loss due to the latent heat of evaporation for the evaporant (available for most materials).

P_n is the loss of power to the evaporant by ionization and secondary electron generation (due to vapor density and electron back-scattering).

P_x is the power loss due to production of X-rays (negligible).

P_c is the power loss to the crucible by conduction (very high).

Table I (reference [2]) on page XXVI-15 shows data on power requirements of typical sources evaporating various materials from a 4-inch-diameter water cooled copper crucible, fed from the bottom with rods of evaporant. This data implies that there appears to be an optimum beam density above and below which evaporation rate decreases for a given level of electron beam power at the evaporant surface.

2219 Aluminum--Plate Thickness .75 Inches
50% Nonconductive Losses--10% to Nailhead
.08 Inch Root Width

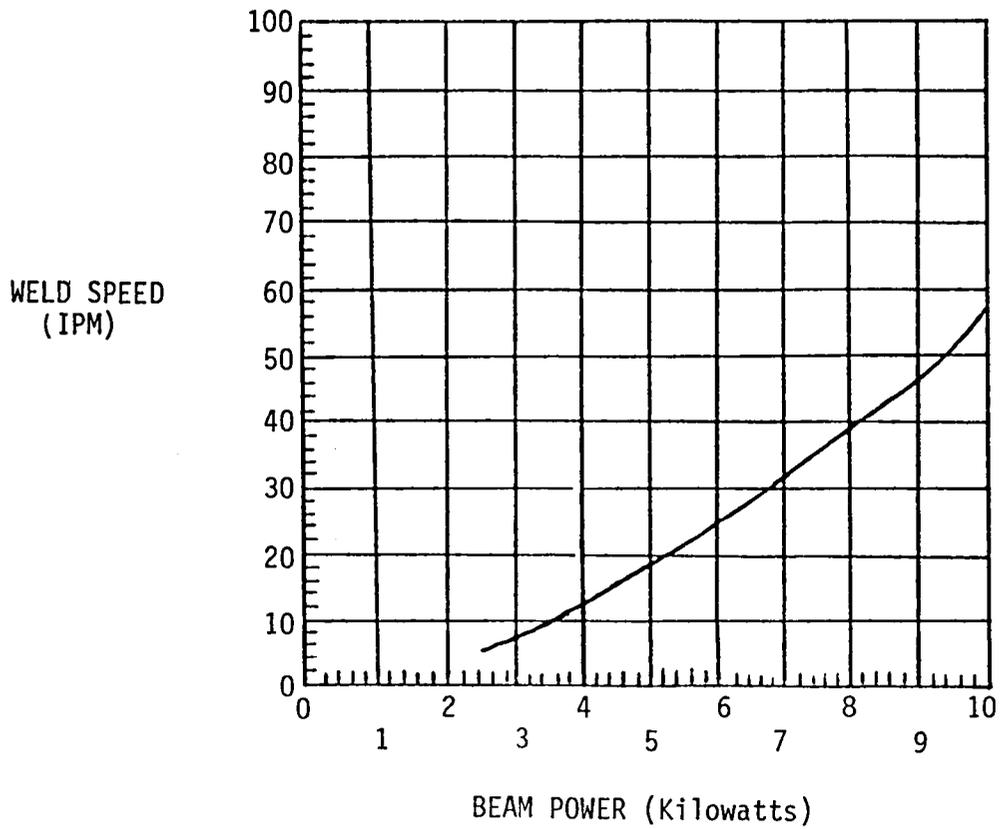


FIGURE 4 - Typical Minicomputer Output on Beam Power and Weld Speed

TABLE 1

<u>Material</u>	<u>Power In</u>	<u>Evap. Rate</u>	<u>Specific Power Requirement</u>
Zirconium	72kw	2.57 lb/hr	28.0 kwh/lb
20% Cr S.S.	70kw	9.38 lb/hr	7.5 kwh/lb
Ti-6Al-4V	70kw	5.17 lb/hr	13.5 kwh/lb
Aluminum-Bronze	80kw	8.96 lb/hr	13.1 kwh/lb
Inconel 600	70kw	5.06 lb/hr	13.8 kwh/lb

TABLE 1 - Power Relations for Evaporation Loss

EQUATIONS OF THE BEAM CURRENT AND POWER DENSITY

When the E.B. gun is operating, its beam current follows the equation [3]

$$I = P V_a^{3/2} \quad (5)$$

where, V_a is accelerating voltage and P is the perveance. Schiller, Heisig, and L nk [3] used the equation below for perveance (P) as well as the following equations including a deviation of the power density (p_f):

$$P = \text{perveance} = 15 \times 10^{-6} \frac{(1 - \cos \theta)}{\alpha^2}, \quad (6)$$

where, α is a function of the space-charge-limited current flow depending on the relation,

$$\alpha = f \frac{d_k}{d_a}, \text{ between the cathode diameter}$$

d_k and the anode bore diameter d_a . The f implies function of.

θ is the aperture angle of the cathode beam generator or the convergence angle of marginal rays in the beam generator (See Figure 5).

$$eV_0 = \text{probable thermal energy} = k_b T = \frac{T}{11,608} \text{ ev}, \quad (7)$$

where,

V_0 is thermic electron potential

e is the charge of 1.6×10^{-19} coul.

k_b is the Boltzmann constant in $\frac{\text{ev}}{\text{deg.}}$

T is the cathode temperature $^\circ\text{K}$.

A Maxwellian energy distribution is assumed with usual emission temperatures, perhaps 3000°K .

$$R = \text{brightness of the beam} = \frac{j_k V_a}{\pi V_0} = \frac{j_k e V_a}{\pi k_b T_K}, \quad (8)$$

where,

j_k is the current density for beam current (J) at the cathode.

$$p = \text{power density at target} = j'_F \cdot V_a \quad (9)$$

where,

j'_F is the current density of the beam at the target and equals $R\pi\alpha_L^2$. Here, α_L is the aperture at the target.

An arrangement of Equations (7), (8), and (9) yields (a high power density p_f)

$$p_f = \frac{j_k}{V_0} \alpha_L^2 \cdot V_a^2 \quad (10)$$

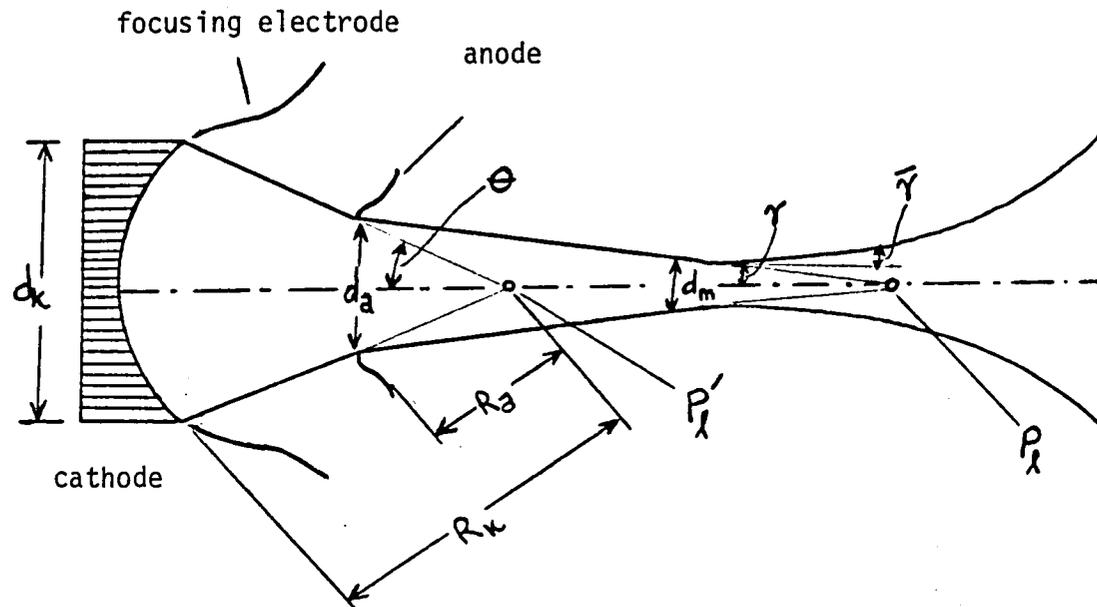


FIGURE 5 - Cross Section of a Spherical Beam Generation System of Pierce-type (Schematically)

- d_k = diameter of cathode
 d_a = diameter of anode bore
 θ = aperture angle of the beam generator - convergence angle of marginal rays in the beam generator
 γ = aperture angle of marginal rays behind the anode, considering scattering effect
 $P_1' \quad P_2$ = convergence points
 d_m = smallest diameter of the beam consider the intrinsic space charge
 $\bar{\gamma}$ = aperture of marginal ray at the position of the smallest beam cross section

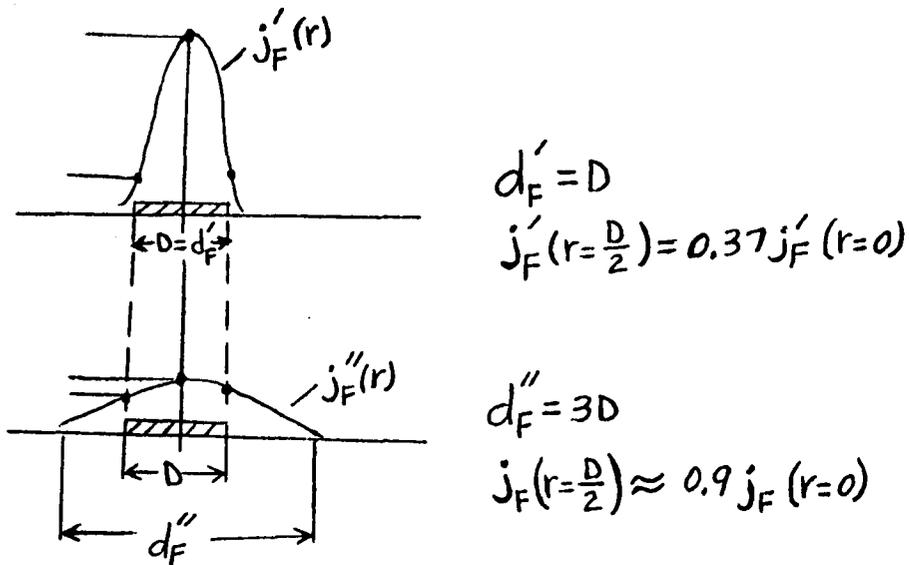


FIGURE 6 - Reduction of the Current Density Difference Between Center and Margin of an Irradiated Area of Diameter D by an Increased Electron Beam Diameter d''_F

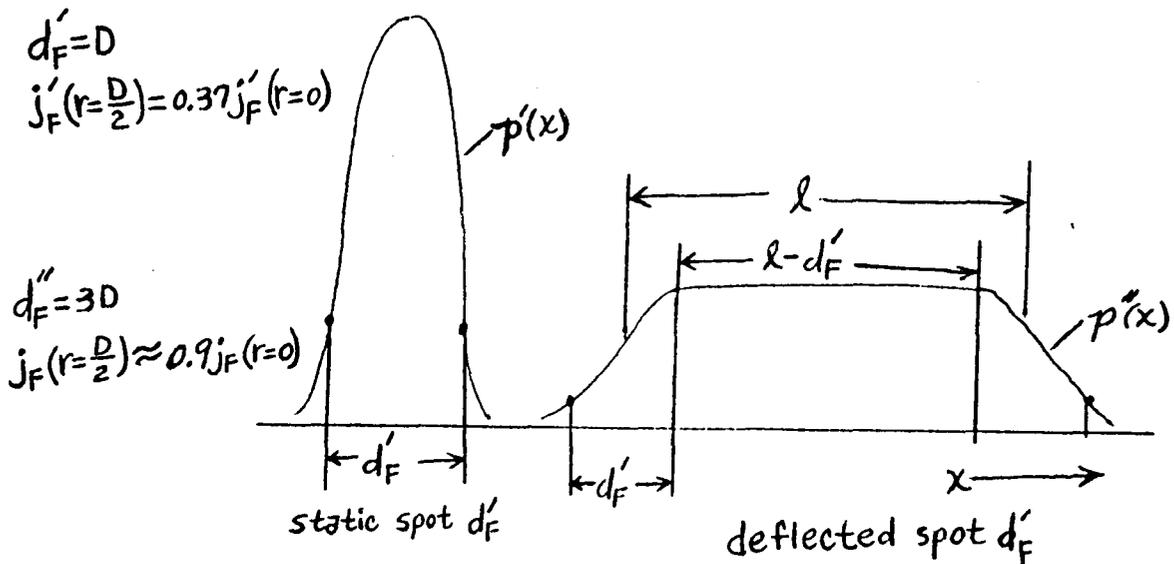


FIGURE 7 - Creating a Constant Power Density $p''(x)$ on a Length $l - d'_F$ by Time Linear Deflection of an Electron Beam d''_F over the Length l . (The range over which power density p'' decreases to $0.63 p''(x=0)$ is $= d'_F$)

Reference [3] also used Figures 6 and 7 to show that time-linear deflection is one coordinate results in a power density distribution (deflection $X = \pm .5$) of constant values in certain ranges. Power outside this range decreases and is a power loss. The term d_F is the beam's spot diameter and D is the target diameter (irradiated).

TECHNIQUES FOR MEASURING AND EVALUATING THE ELECTRON BEAM

This section explains the experimental and computer aided approaches used to evaluate the beam configuration.

The initial problem task for this section is that of determining what available method and apparatus to use to take measurements of the beam. The rotating probe (or wire) approach was chosen. This method utilizes a small heat protected motor to turn the wire probe through the beam. The signal obtained on the probe is fed into a storage oscilloscope. Pictures of the signal for various operating conditions of the system are taken from the scope and analyzed. The technique has the following advantages:

- (a) simple
- (b) apparatus available and not refined
- (c) beam current is measured
- (d) beam diameter is obtainable

The wire probe did not obtain appreciable damage from the beam and adequately allowed suitable measurements to take place.

Adjusting the accelerating voltage (thus power output) appears not to change the shape nor the general dimensions of the beam.

The experimental setup used to measure the beam is shown in Figure 8. The oscilloscope was used to monitor and record not only the beam signals but general signals (noise) in the vacuum. During pump down the oscilloscope was triggered by noise signals equivalent to those shown in Figure 9.

Since this electron beam welding machine receives its instructions from a computer controlled system, the sample computer program for a typical set of data inputs is shown on page XXVI-23.

When the beam comes into contact with the rotating wire probe, the electrons that are not backscattered or secondary-emitted are received as the current of the beam (perhaps 50%). The current through the wire has the common variation in magnitude at various angles of emergence. The surface contour of the current is obtained from the electron distribution. Care must be taken in determining the needed current signal configuration and associate information instead of bad data. The collecting wire probe has a symmetrically fixed amount of wire extending from two sides of the plate affixed to the

VACUUM CHAMBER

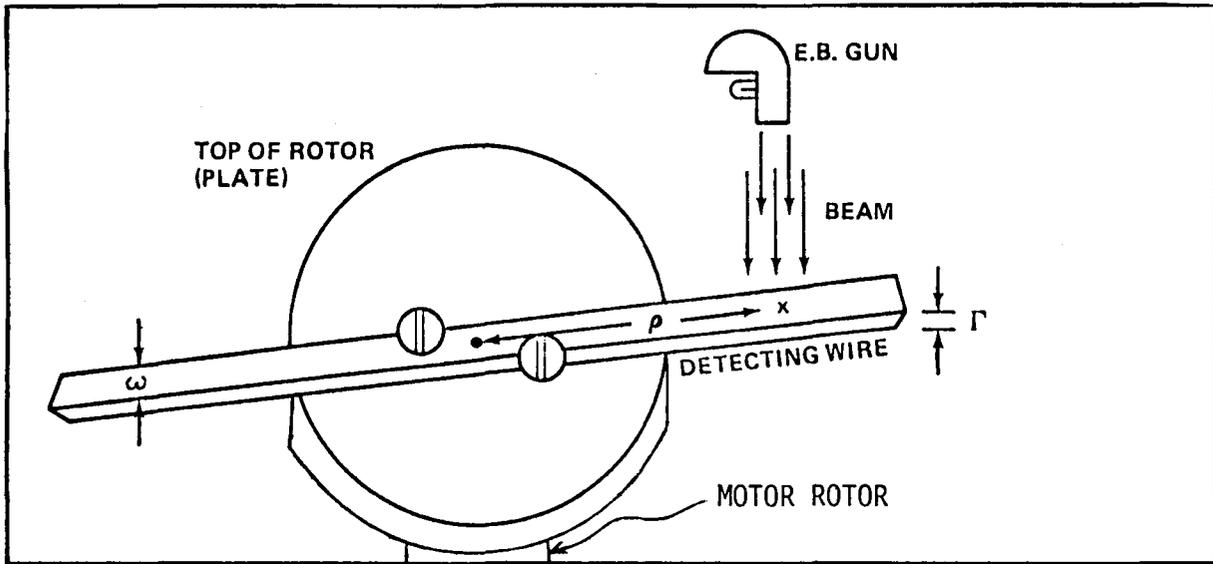
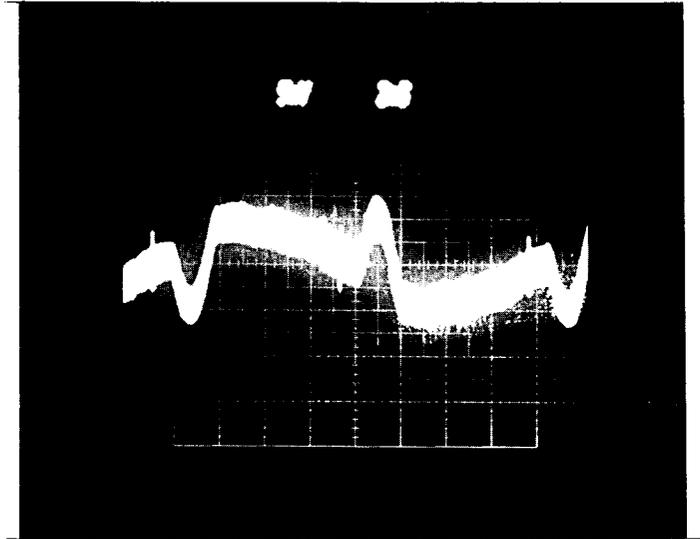


FIGURE 8 - EXPERIMENTAL SETUP

5mv

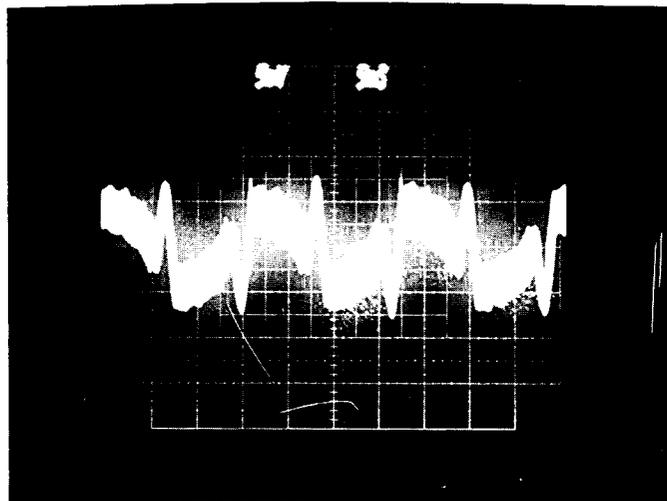
2ms



(a) Noise in the E.B. Welder Recorded at 2 Milliseconds Fullscale (above)

5mv

5ms



(b) Noise in the E.B. Welder Recorded at 5 Milliseconds Fullscale (above)

FIGURE 9 - Signals Triggering the Scope Due to Noise Present During Pumping of Vacuum Chamber

TYPICAL DATA RESULTS

ρ = 2.833 INCHES

ω = .048 INCHES

Γ = .003 INCHES

BF (60 KV) 5.00

FIL 3.50

AV 30 KV

BC 50 MA

THE SAMPLE COMPUTER PROGRAM FOR THE WELDER IS SHOWN BELOW.

FILE RECORD
NASA EB WELDER

FILE NO. 110

BEAM CROSS SECTION STUDY PROGRAM

PAGE 1

(BEAM CROSS SECTION STUDY PROGRAM)

OVE POT 0

G1APT X0 Y0 F500 M0

FIL350 F20

BF500 F28

AV300 BC50 F20

AV300 BC50 F1000

AV0 BC0 F20

A TABLE OF COMPUTER INPUT DATA (σ , I, ETC...) IS DEVELOPED FROM FIGURE 11 FOR USE ON THE DIGITAL MINICOMPUTER.

rotor of the motor (as was shown in Figure 8). The motor was shielded from the beam and from heat by a thin sheet of metal. The current collected was sent through a load resistor and the output from this load to the oscilloscope.

In order to adequately measure the beam current, the wire probe dimensions were carefully measured (Figure 10). The beam was also precisely applied to a target area of the probe at a chosen radius from the center of the plate of the motor.

Figure 11 shows the typical current signal and can be plotted versus angle position. With the given beam characteristics or welding operation status, the signal profile is always of a similar shape. To avoid possible aggravation of the signal, it should be saved on the scope as soon as a clear configuration is observed. The scope will automatically trigger to the signal prior to being available for saving the signal. Interferences during signal generation, such as surfaces disturbances not allowing adequate distribution of beam electrons, were considered minor.

Two cases representing the output current signal are in Figures 12 and 13. The first case (Fig. 12) is the peak beam current for two pulse signals. The second case (Fig. 13) is an expanded version of one of the pulse signals distributed with a wider profile. Several sweep time divisions of the scope were investigated for many conditions until satisfactory signals were realized.

The current vs. probe angle was plotted and used as inputs for the computer routine that calculates the current density per unit radius. Additional inputs to the computer included (a) the distance from the center of the plate of the motor to a target point on the wire, (b) the width of the wire probe (receptor), (c) the total beam current, and (d) the number of current-angle data points.

As can be seen from Table 2 (page XXVI-28) and Figures 14 and 15, a current density profile can be obtained. The radial distribution is modulated by a strong periodic error signal of a sort known to arise from small errors (about 0.1%) in data. Work is continuing to obtain current density profiles.

The measurements taken from this experimental setup show that much more beam power is lost than can be explained from measurements taken from the wire probe, unless very large amounts of energy are carried away per secondary emission or backscattered electrons. This justifies the need to continue investigating the evaporation power loss and other power losses accounting for this reduction of total beam power. This phenomenon occurs quite often when samples are being welded. Some information is available on secondary emission losses from a 1974 study conducted at NASA-MSFC.

TUNGSTEN RIBBON (Wire # 2)

$L \approx 8''$

Γ = Thickness .003"

ω = Width .048"

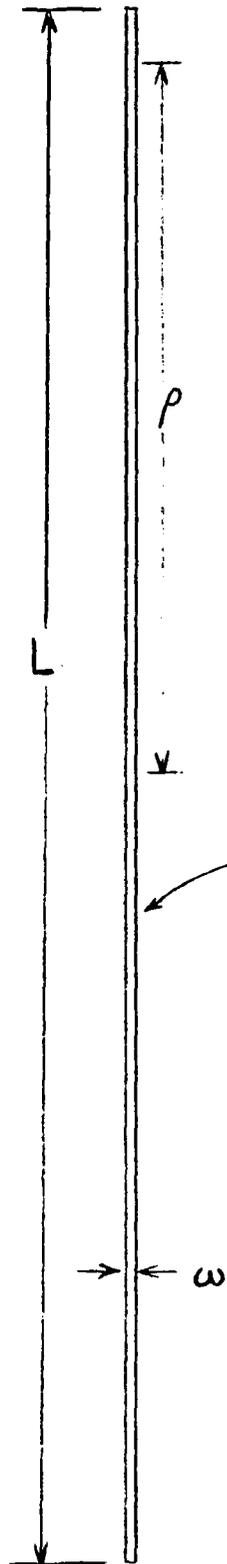
ρ = Beam Target Distance 3-3/4"

WELDER OPERATION STATUS:

Beam Focus - BF (60 Kv) 5.00

Filament Setting - Fil 3.50

Accel. Voltage, Beam Current - 30 Kv,
BC 50 ma



Actual size of the Wire Probe (Γ not shown)

FIGURE 10 - Wire Probe Characteristics

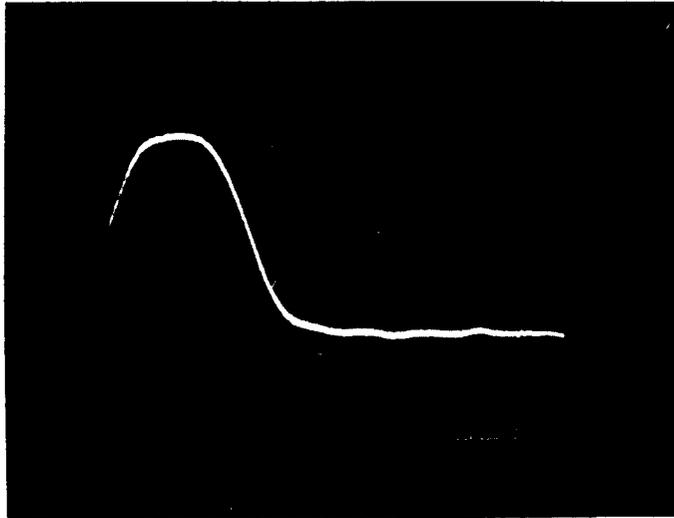


FIGURE 11 - Typical Beam Current Configuration (actual photo)

EXPERIMENTAL RESULTS

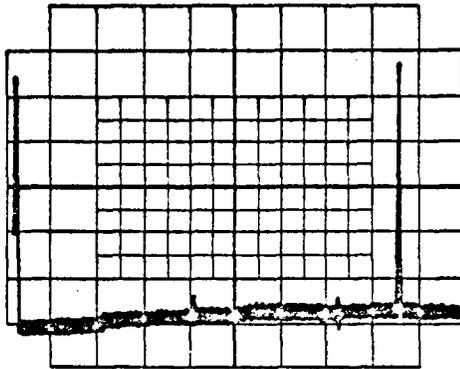


FIGURE 12 - Peak Beam Current for Two Pulse Signals

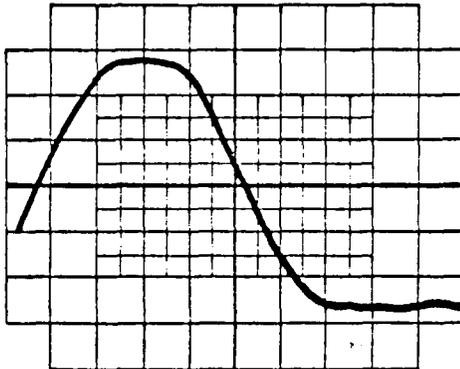


FIGURE 13 - Single Pulse Signal Distributed

THE CHARACTERISTICS OF THE TUNGSTEN WIRE ARE

- ρ , THE DISTANCE FROM THE CENTER OF THE MOTOR PLATE TO A TARGET POINT ON THE WIRE
- ω , THE WIDTH OF THE WIRE
- Γ , THE THICKNESS OF THE WIRE

THE E.B. WELDER WAS COMPUTER CONTROL FOR AN OPERATION STATUS FOR VALUES OF

- BF, BEAM FOCUS
- FIL, FILAMENT SETTING
- AV, ACCELERATING VOLTAGE
- BC, BEAM CURRENT

TYPICAL DATA RESULTS ARE SHOWN ON THE NEXT PAGE.

TABLE 2

<u>ANGLE</u> <u>(DEG)</u>	<u>CURRENT</u> <u>(MA)</u>	<u>RADIUS</u> <u>(IN)</u>	<u>CURRENT DENSITY</u> <u>(AMPS/SQ IN)</u>
.017	26.32	8.40541E-04	-832.032
.031	28.15	1.53275E-03	1153.54
.053	28.97	2.62051E-03	-571.086
.074	31.79	3.65883E-03	301.177
.093	33.61	4.59827E-03	.0935703
.11	35.43	5.43882E-03	214.231
.134	37.25	6.62548E-03	-3.99142
.154	39.08	7.61436E-03	104.89
.178	40.89	8.80103E-03	75.7133
.211	42.72	.0104327	66.7768
.248	44.54	.0122622	-57.2598
.266	46.36	.0131522	246.266
.327	48.18	.0161683	6.38338
.386	50	.0198856	45.3961

Ratio of Total Current Calculated to Total Beam Current.

1

TABLE 2 - Digital Computer Results on Current Density

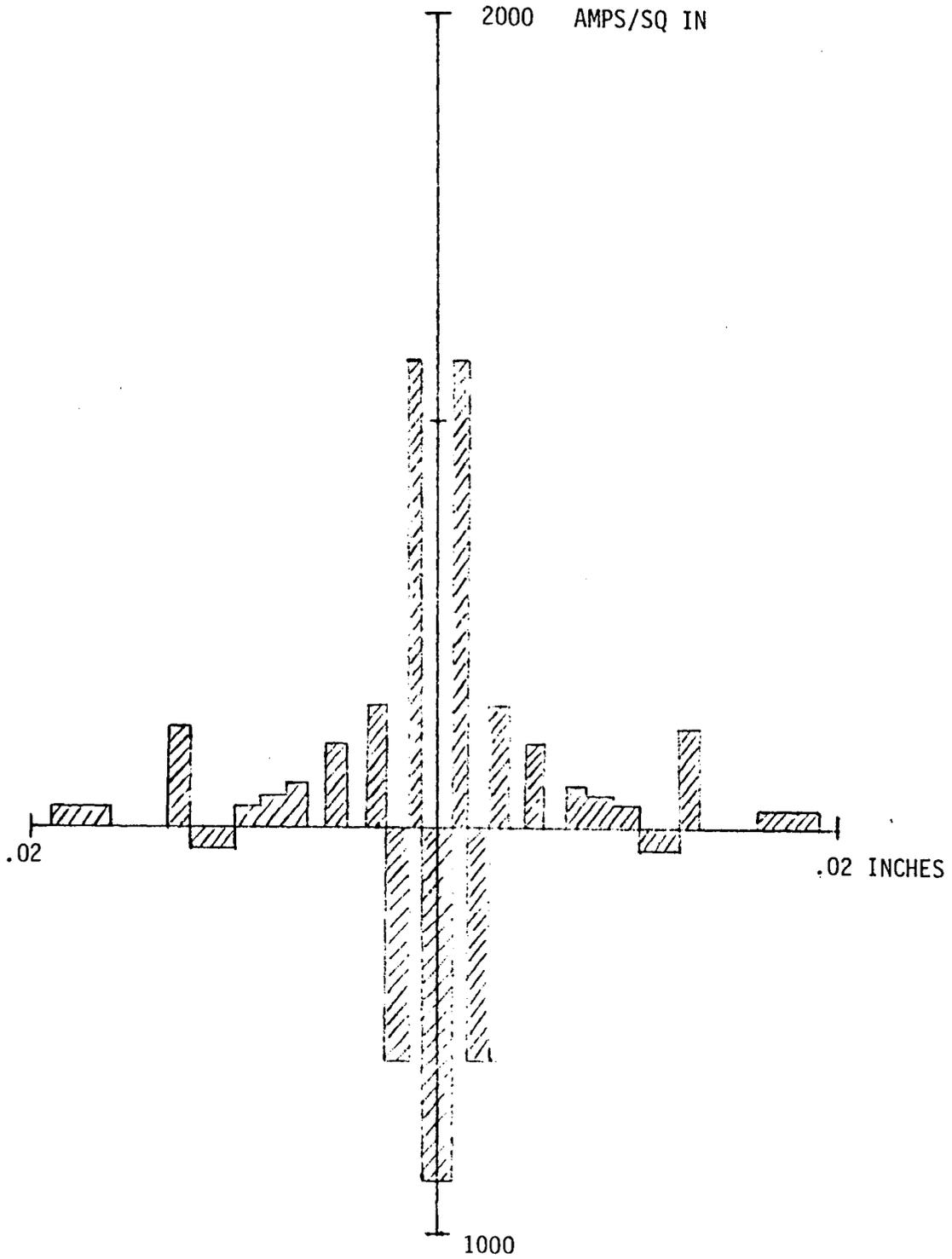


FIGURE 14 - Radial Current Distribution (2 dimensions)

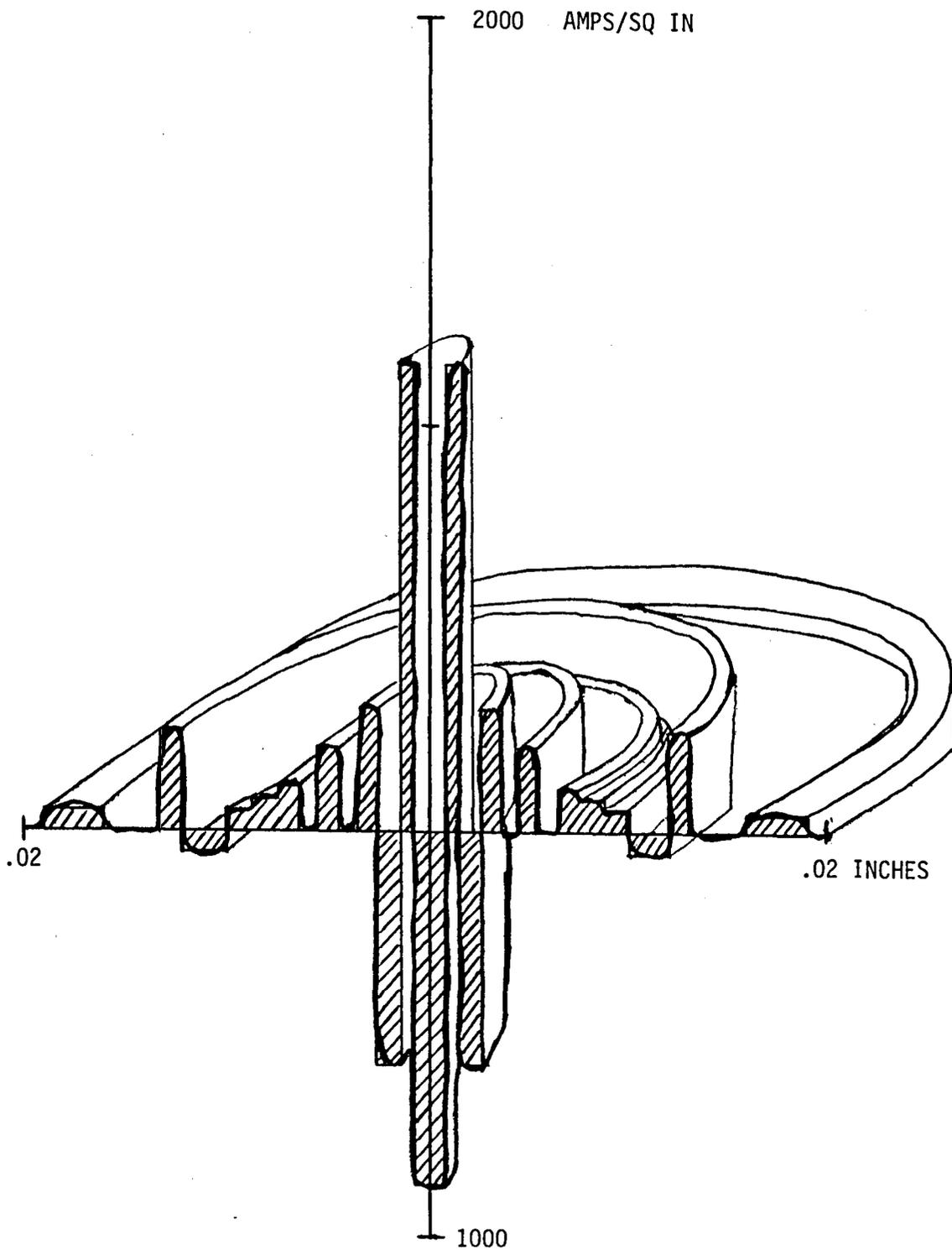


FIGURE 15: Radial Current Distribution (profile)

CONCLUSIONS

At the present time we are able to capture by experimental techniques the current configuration for an electron beam. The computer routines allow us to theoretically determine the beam characteristics on the workpiece for a given power profile. Techniques developed and described in this report can be valuable for gathering information needed to assess the nature of weld power losses.

Future work should be done in order to relate accurately the total beam characteristics and power distribution to weld geometry. However, we have successfully completed the experimental task of obtaining the signal configuration which determines the cross sectional power distribution in the electron beam.

The results of Figures 11 and 13 compare with those obtained by Sayegh and Dumonte [4] [5].

An analysis for obtaining the radial current density and the profiles in Figures 14 and 15 is available but small errors in data (or deviations from a radial distribution) apparently cause strong periodic error modulations in the radial distribution.

RECOMMENDATIONS

1. Techniques for obtaining more information on electron reflection should be investigated.
2. Controlled use of the welding system such that the time wasted in equipment venting and pump down is minimized.
3. Advanced studies should be done on the trajectories of the beam (both gaussian and nongaussian type).
4. Curvilinear distribution analyses should be considered for further study.
5. Relate the diameter of the beam and the power distribution on a welding sample in items 3 and 4 above.
6. Determine if the target surfaces cause any disturbance in the distribution of the beam's electrons on the wire probe.
7. Determine the optimum setting for the E.B. welder in order to obtain the optimum power density profile.

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1980

NASA/ASEE SUMMER FACULTY RESEARCH FELLOWSHIP PROGRAM

MARSHALL SPACE FLIGHT CENTER
THE UNIVERSITY OF ALABAMA

MEASUREMENT OF VELOCITY FIELDS IN FLUID DYNAMICS EXPERIMENTS

Prepared by:	Daniel E. Fitzjarrald
Academic Rank:	Research Associate
University and Department	Geophysical Fluid Dynamics Institute Florida State University
NASA/MSFC: (Laboratory) (Division) (Branch)	Space Sciences Laboratory Atmospheric Sciences Fluid Dynamics
MSFC Counterpart:	George H. Fichtl
Date:	August 6, 1980
Contract No.:	NGT-01-002-099 (University of Alabama)

MEASUREMENT OF VELOCITY FIELDS IN FLUID DYNAMICS EXPERIMENTS

by

Daniel E. Fitzjarrald
Geophysical Fluid Dynamics Inst.
Florida State University
Tallahassee, FL

ABSTRACT

In many fluid dynamics experiments in which the motion has both spatial and temporal variations, it is necessary to measure the entire two-dimensional field of velocity at an instant. Two methods to accomplish this measurement have been investigated. In both cases the emphasis has been on using a microcomputer-video system in the analysis of data. The first method is the traditional one in which tracers are introduced into the fluid and their position compared at two closely spaced times. The second method involves scattering coherent light in the fluid and obtaining motion by analyzing the multiple-exposed speckle pattern recorded on photographic film. Both of these velocity field methods can be used in the geophysical fluid flow and space processing experiments in progress at NASA.

1. Introduction

In many fluid dynamics experiments both spatial and temporal variations of the flow patterns are important. This is particularly true in experiments modelling geophysical flows and in space-processing experiments, both in progress at NASA. With conventional instrumentation the experimentalist must rely on the use of fine wire probes, laser-Doppler measurements, or other similar techniques that are all limited to a single point in space. It is possible to move the measuring point around in the fluid, but the technique is never satisfactory when there are significant time variations in the flow field.

Streak photography has often been used in such cases, i.e., taking photographs of the displacement of tracers in the velocity field. Data analysis is quite cumbersome, however, whenever there is much variation in the flow field, or when a dense coverage of data points is required.

It is the goal of this summer's research to investigate a promising alternative to conventional velocity field measurement. The technique to be investigated will be termed light-speckle techniques, and all are closely related to streak photography. The key in each case is that the tracers and time intervals are chosen so that optical correlation techniques can be used to recover velocities from the data.

2. Light-Speckle Velocity Field Measurement

Although there are several variations in details, these techniques are all basically the same. Light from scatterers that move with the fluid is recorded at closely-spaced time intervals on photographic film. The optical pattern from the first instant is slightly changed by the fluid motion at succeeding time intervals. Thus, when the multiple-exposed negative is interrogated by a narrow beam of coherent light, the correlation between patterns recorded at succeeding times is indicated by a pattern of regular fringes. The spacing of the fringes tells the amount of displacement, i.e., the time interval and the orientation of the fringes tells the direction of displacement. Thus, by illuminating the fluid with

a thin sheet of light and viewing it at a right angle to the plane of illumination, an entire two-dimensional field of velocity can be obtained at an instant.

2.1 Generating the Optical Pattern

The optical pattern that is to be displaced by the fluid can be generated either by using the coherent nature of laser light with very small scatterers, or with white light and larger scatterers. The former method has been used by Simpkins and Dudderer (1978) and is based on the fact that laser light appears grainy when diffusely reflected off a surface. The grainy appearance is caused by constructive and destructive interference of coherent light scattered off roughness elements the same size as the wavelength. These "objective speckles" are a function of the size of the scatterers, while the "subjective speckles" seen by the observer or recorded on film are dependent on the aperture size of the lens through which they are viewed. When the displacement of the recorded subjective speckles due to fluid motion is between one and twenty speckle diameters the correlation will yield a good fringe pattern. The fringes will also be enhanced if more than two optical patterns are recorded on the same negative.

A useable optical pattern can also be obtained by using larger scatterers illuminated by white light (Chiang and Asundi, 1979), or by a laser (Meynart, 1980). In the latter case the spatial coherence of the laser light is used only to obtain a thin light sheet of high power density. Again, multiple exposures increase the visibility of the fringe.

2.2 Interrogating the Negative

Once the multiple-exposed negative has been obtained, there are two methods to recover the velocity field. Illuminating the negative point-by-point with a small coherent beam yields fringes that come from a small portion of the flow field where the velocity was constant. The negative is thus interrogated at each point in the field that the velocity is required.

A whole-field method of analysis can be obtained by illuminating the entire negative and using a spatial filtering setup (see Meynart, 1979) that consists of two lenses (Fourier transformers) and an off-axis pinhole

(spatial filter). The result is the whole velocity field described in isolines of the component of velocity depending on the orientation of the spatial filtering hole.

There are advantages to both these interrogation methods. The first yields results that are more precise, with the limits of precision depending on the quality of the fringes that are obtained. The whole-field method is simpler and much quicker.

3. Use of Microcomputer in Analysis

In order to speed up analysis to make possible the evaluation of fast-changing geophysical and space-processing fluid flows, it is desirable to automate the data analysis as much as possible. Using a microcomputer-TV system it should be possible to accomplish the point-by-point interrogation of the multiple-exposed negatives automatically. The microcomputer will be programmed to count the fringes, determine orientation, and then move the film transport on to the next position. In addition, the TV system can determine velocity by simply subtracting successive positions of large scatterers to serve as a check on the velocities obtained using the speckle techniques.

4. Planned Work and Work Accomplished to Date

At this writing the experimental work is just getting into progress. Multiple exposures have been taken in a small water flume and thermal convection cells illuminated by a 1 watt CW argon laser. The pulse repetition rate and flow rate have been varied to determine optimum conditions for obtaining good fringes.

In addition considerable program development work has been accomplished on the TV-microcomputer system. Picture enhancement and pattern recognition programs have been written and debugged, so that they can be put together to measure fringes and track individual scatterers.

5. Conclusions

The light-speckle velocity field techniques described above offer tremendous possibilities toward the understanding of geophysical and space-processing fluid flow experiments. The ability to obtain an entire two-dimensional velocity field at an instant makes these techniques extremely powerful.

The experiments and computer program development in progress should demonstrate the utility of these light-speckle techniques.

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ON THE CONTROL AND STABILITY OF THE
PINHOLE-OCCULTER FLEXIBLE BOOM FACILITY

PREPARED BY: Michael Greene, PhD
ACADEMIC RANK: Assistant Professor
COLLEGE AND DEPARTMENT: University of Alabama
Department of Electrical Engineering
NASA/MSFC:
(OFFICE) Preliminary Design
(DIVISION) Subsystems Design
(BRANCH) Navigation and Control
MSFC COUNTERPART: J. R. Parker
DATE: July 15, 1980
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(University of Alabama)

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ON THE CONTROL AND STABILITY OF THE
PINHOLE-OCCULTER FLEXIBLE BOOM FACILITY

By

Michael Greene, PhD
Assistant Professor of Electrical Engineering
University of Alabama
Tuscaloosa, AL

Abstract

The dynamics and control of the Shuttle based, gimbal mounted, boom pointing system for use in the x-ray pinhole experiments were studied. The overall system stability was studied, and root locus compensation was designed to achieve stability and pointing stability requirements. This single axis control study found the compensated system to be stable and capable of pointing to within 6.5 arc sec while the Shuttle rocks in its 0.1° deadband. The system can follow step (position) inputs with zero error and ramp (velocity) inputs with a steady state error coefficient of 4.5 sec^{-1} .

Introduction

The pinhole-occulter system is a Space Shuttle based experiment for the production of hard X-ray images taken primarily from the Sun. The system is basically a pinhole camera utilizing a deployable 50-m flexible boom for separating the pinhole from the recording devices located in the Shuttle as seen in Figure 1. At the distal end of the boom from the Shuttle is a 50 kg mask containing two sets of pinholes and two chronograph shields as seen in Figure 2. At the proximal end the detectors are located and mounted, along with the boom, to a gimbal pointing system (either IPS or AGS) mounted in the Shuttle Payload Bay.

The mask of Figure 2 must be pointed at the X-ray source with a pointing stability of less than 10 arc sec to align the axes of the detectors with the pinholes and shields. Failure to do so will result in a blurring of the images on the detectors and a loss of resolution. Being a Shuttle based experiment, the system will be subjected to the disturbances of the Shuttle. The worst of these is thruster firing for orbit correction; Shuttle uses a bang-bang thruster control system to maintain orbit to within $\pm 0.1^\circ$. Other disturbances include man motion, motion induced by other systems, and gravity gradient torques.

The control system of the pointing mount senses both position and velocity of the mask tip and uses these to accurately point the boom. The AGS employing perfect sensors was used in this study of the single axis (y-axis) control system. The extension of this study to a three-axis system will be the topic of a later study.

Background

The structural analysis of the boom [1,2,3] revealed the seven mode shapes and natural frequencies shown in Table 1. These natural frequencies varied with both boom length and tip mass as seen in Figure 3. For a 50-m boom with a 50 kg tip mass, the first two natural frequencies are 1 rad/sec for deflections about the x and y axes, respectively. The third natural frequency is 17.1 rad/sec for z axis motion. For x and y motions the next two natural frequencies are 17.4 rad/sec and 59.9 rad/sec, respectively.

For y-axis control (torques about y-axis), the equations of motion for the boom, assuming a damping factor of 10 percent, are [1]:

$$\ddot{\eta}_1 + .2 \dot{\eta}_1 + \eta_1 = - 3687.93 \ddot{\theta}_2 \quad (1)$$

$$\ddot{\eta}_2 + .2 \dot{\eta}_2 + \eta_2 = 0 \quad (2)$$

$$\ddot{\eta}_3 + 3.42 \dot{\eta}_3 + 292.4 \eta_3 = - .018396 \ddot{\theta}_2 \quad (3)$$

$$\ddot{\eta}_4 + 3.46 \dot{\eta}_4 + 300 \eta_4 = 1250.734 \ddot{\theta}_2 \quad (4)$$

$$\dot{\eta}_5 + 3.46 \eta_5 + 300 \eta_5 = 0 \quad (5)$$

$$\ddot{\eta}_6 + 11 \dot{\eta}_6 + 3014 \eta_6 = 52.3905 \ddot{\theta}_2 \quad (6)$$

$$\dot{\eta}_7 + 11 \eta_7 + 3014 \eta_7 = 0 \quad (7)$$

$$\theta_{\text{TIP}} = 2.895 \times 10^{-4} \eta_1 - 8 \times 10^{-10} \eta_3 + 5.31 \times 10^{-5} \eta_4 - 3 \times 10^{-5} \eta_6 + \theta_2 \quad (8)$$

Taking the Laplace transform of equations (1) through (7) and inserting into equation (7), we obtain:

$$\theta_{\text{TIP}}(S) = \left[\frac{-1.07S^2}{S^2 + .2S + 1} + \frac{1.5 \times 10^{-11} S^2}{S^2 + 3.42 S + 292.4} + \frac{.07S^2}{S^2 + 3.465 + 300} + \frac{1.57 \times 10^{-3} S^2}{S^2 + 11S + 3014} + 1 \right] \theta_2(S) \quad (9)$$

Ignoring terms 2 and 4 and reducing, the ratio of input motion to output motion for the boom is obtained:

$$\frac{\theta_{\text{TIP}}(S)}{\theta_2(S)} = \frac{-.07(S^2 - 2.865 - 14.28)}{S^2 + .2S + 1} + \frac{.07S^2}{S^2 + 3.48S + 300} \quad (10)$$

$$= \frac{-.0289(S^3 + 688.17S^2 - 2194.12S^1 - 10380.62)}{S^4 + 3.67S^3 + 301.69S^2 + 63.47S^1 + 300} \quad (11)$$

In its factored form, this transfer function for the boom is:

$$G_B(S) = \frac{-0.0289 (S + 2.63) (S - 5.88) (S + 671)}{(S^2 + .2S + 1) (S^2 + 3.46S + 300)} = \frac{N_B(S)}{D_B(S)} \quad (12)$$

The block diagram for the system is shown in Figure 4. The pointing mount uses a position plus integral control with a velocity feedforward [4]. For the sake of clarity, the velocity feedforward will be considered separately from the forward transfer function of the Gimbal Pointing System which is:

$$G_G(S) = \frac{K_P (S + 1.2)}{S} \quad (13)$$

The velocity feedforward and the boom comprise a loop which can be reduced to a new forward transfer function, $G_{BS}(S)$, to treat the system as a unity feedback system as seen in Figure 5. The new forward transfer function is given by:

$$G_{BS}(S) = \frac{G_B(S)}{1 + K_R S G_B(S)} = \frac{N_B(S)/D_B(S)}{1 + K_R S N_B(S)/D_B(S)} \quad (14)$$

$$G_{BS}(S) = \frac{N_B(S)}{D_B(S) + K_R S N_B(S)} \quad (15)$$

From equation (15) we can see that the open loop eigenvalues can be arbitrarily set by adjustment of the rate gain, K_R . The closed loop eigenvalues will then be set by means of the compensator, $G_C(S)$, and the forward gain, K_P .

Results

The open loop eigenvalues were set by adjustment of the rate feedforward gain, K_R . The plot of these roots (eigenvalues) as K_R is increased from zero is shown in Figure 6. At a gain between 0.1 and 0.2, the system goes unstable. At a gain of 0.07 the open loop roots are:

$$S_{1,2} = -.14 \pm j .98$$

and,

$$S_{3,4} = -1.23 \pm j 17.38 .$$

Yielding the new transfer function:

$$G_{BS}(S) = \frac{-.02896 (S + 2.63) (S - 5.88) (S + 671)}{(S^2 + 28S + .99) (S^2 + 2.46S + 303)} . \quad (16)$$

The major result of this rate compensation (which is built into the pointing mount) is to increase the damping ratio of the first quadratic root and decrease the damping of the second. This gives the basic system transfer function:

$$G_{SYS}(S) = \frac{K^1 (S + 1.2) (S + 2.63) (S - 5.88) (S + 671)}{S (S^2 + .28S + .99) (S^2 + 2.46S + 303)} . \quad (17)$$

Setting $G_c(S) = 1$, we can again plot the root locus for the system as K^1 is varied ($K^1 = -.02896 K_p$). This plot is shown in Figure 7. At a gain between -0.2 and -0.03 the system goes unstable and oscillates; the system is unstable for all values of positive gain. At a gain of -0.2 the dominate roots have a time constant of about 8 seconds and would be extremely underdamped with $\xi = 0.01$ ($S_{1,2} = -0.13 \pm j 12.3$). This is not acceptable; compensation is required.

The compensation chosen was:

$$G_c(S) = \frac{(S^2 + 2.46S + 303) (S + 2)}{(S + 671) (S + 10)^2} . \quad (18)$$

This compensator cancels the open loop roots at $s = -1.23 \pm j 17.38$ and the zero at $S = -671$. If the compensator does not cancel the complex roots exactly it is better to undersize the real part of the compensator roots than oversize them. The response will contain terms of $Ae^{-1.23t} \cos(17.38t + \phi)$ but A will be extremely small due to the proximity of the zeroes to these roots.

The basic idea of the compensation is to cause the roots nearest the imaginary axis (first mode) to move to the left as gain is increased while higher order modes (roots) move to the right. This allows for a high value of the gain which produces low steady state ramp errors and a wide passband. Ideally a type 2 system is desired but due to the zero in the right half s-plane, this is impossible.

Using the compensator of equation (18), a new G_{total} is obtained:

$$G_{total}(S) = \frac{K^1 (S + 1.2) (S + 2) (S + 2.6) (S - 5.88)}{S (S^2 + .28S + .99) (S + 10)^2} = \frac{K^1 N(S)}{D(S)} \quad (19)$$

With unity feedback, the closed loop control ratio is:

$$\frac{C}{R} = \frac{G}{1 + G} = \frac{K^1 N(S)/D(S)}{1 + K^1 N(S)/D(S)} \quad (20)$$

which reduces to

$$\frac{C(S)}{R(S)} = \frac{K^1 N(S)}{D(S) + K^1 N(S)} \quad (21)$$

The plot of the closed loop roots is seen in Figure 8. At a gain of approximately -16, system becomes unstable. At a gain of -12, the dominant root is $S = -0.75$ and the ramp error coefficient is $K_1 = 4.5 \text{ sec}^{-1}$. For a ramp input of 0.01° , the steady state error for the system would be 8 arc sec.

The error response of the system to the Shuttle's thruster firing and drift due to gravity gradient is seen in Figure 9. The peak error response is 3.6 arc sec as the Shuttle drifts through its limit cycle and 6.5 arc sec when one thruster is firing. The maximum gimbal torque required to oppose the thruster acceleration is 17.9 NT-m while the maximum torque during drift is 8.3 NT-m.

The step response of the system to a 1° step is shown in Figure 10. Within 5 sec of the step, the tip angle is within 104 percent of steady state. The peak overshoot is 42 percent occurring at 7 sec. The response is within ± 1 percent of steady state within 16 sec. Steady state error to step commands is zero. The torque required at the gimbal motors for this response exceeds the rated 33 NT-m during the first 12 sec. The torque required to initially accelerate the boom is up to 182 NT-m/degree. This limits step commands to less than 0.18 degree.

The frequency responses for the boom and rate compensated boom are seen in Figure 11. The responses of the fixed boom is a series of single order zeroes and quadratic poles starting at 1 radian/sec (.16 Hz). The boom is itself stable having an 18 dB gain margin and a 35° phase margin. The effect of the gimbal system rate compensation is to increase the damping ratio of the first mode and decrease the damping of the second. At a $K_r = 0.07$, stability is unaffected.

The frequency response of the gimbal mounted boom system is shown in Figure 12 for a gain of 16. At this gain, the system is unstable having a gain margin of -50 dB. And a phase margin of nearly -90° . For a stable system, the gain would have to be reduced drastically and would yield a slow, low bandpass system with high ramp following errors.

The frequency response of the compensated system is shown in Figure 13 for a gain of 16. At this value of gain, the system is just unstable with both a phase margin and a gain margin of zero. Reducing the gain of 2.4 dB to a gain of 12, the system becomes stable and has the closed loop frequency response of Figure 14. The system bandpass is 2.5 Hz and it will follow up to 0.2 Hz signal without error or phase shift. This is the basic reason the system can tolerate the drift of the Shuttle in its deadband and the firing of one thruster. The drift of the Shuttle is very nearly a 0.01 Hz signal while the turnaround at the deadband is like a 1 Hz signal. The system response to thruster firing is not frequency (dynamics) limited but torque limited by the gimbal motors.

Conclusions and Recommendations

As compensated, the gimbal mounted boom system is stable with fast dynamics and low following errors. The system is relatively unaffected by the drift of the Shuttle in its deadband and by one thruster firing (maximum error 6.5 arc sec). The system is limited by the torque produced by the gimbal motors not by dynamics.

This study only considered motions about the y-axis, not x-axis or z. Motion about the x-axis should be very similar to that of the y-axis except modes shapes 2 and 5 would predominate. The control system for x-axis should be nearly identical to that of the y-axis. Motion about the z-axis is twisting motion of primary the third mode (see Table 1). This control system should be much simpler than either x or y control.

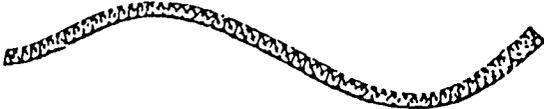
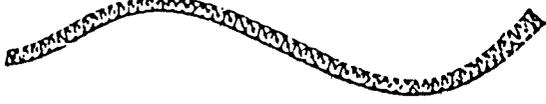
This study assumed perfect motion detectors for the feedback loops. Good detectors [6] are available for x, y, and z motions but their sensitivities and accuracies and sensitivities need to be added to the control simulation as disturbance.

The dynamic as well as probabilistic characteristics of each of these disturbances as well as man motion need to be characterized and incorporated into the control model to determine the pointing accuracies and pointing stability of the gimbal mounted boom. Future work should include a stochastic analysis of the control system concerning probabilistic disturbances and the minimization of their effects on pointing stability. The results of such a study would be a three-axis model of the system compensated for stability as well as compensated for disturbance induced pointing variations.

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5. D'Azzo, J. J. and Houpis, C. H. "Linear Control System Analysis and Design, Conventional and Modern." McGraw-Hill, New York, N. Y., 1975.
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TABLE 1. PINHOLE BOOM FREQUENCIES AND MODE SHAPES

L = 50 METERS, TIP MASS = 505.7 KG	
FREQ (HZ)	MODE SHAPE
1	
1	
17.1	
17.4	
17.4	
54.9	
54.9	

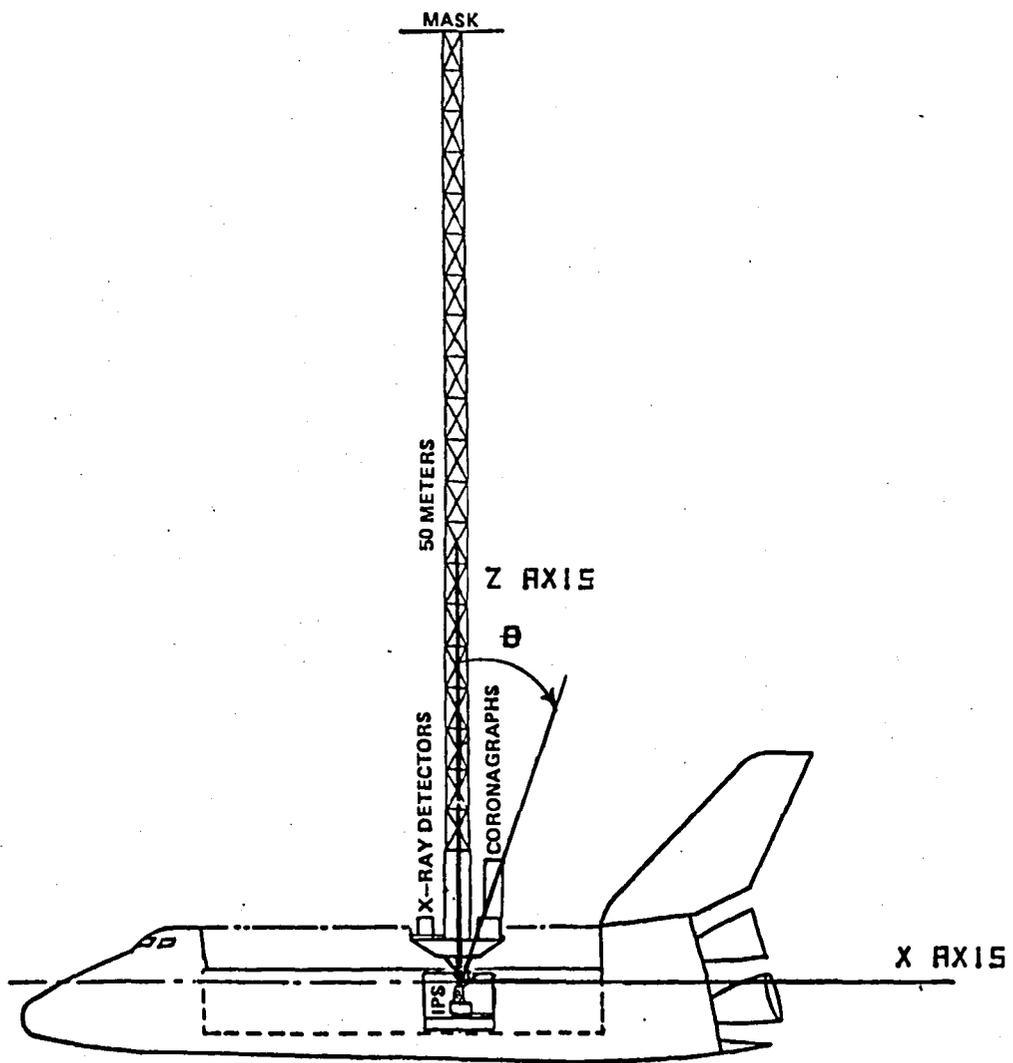
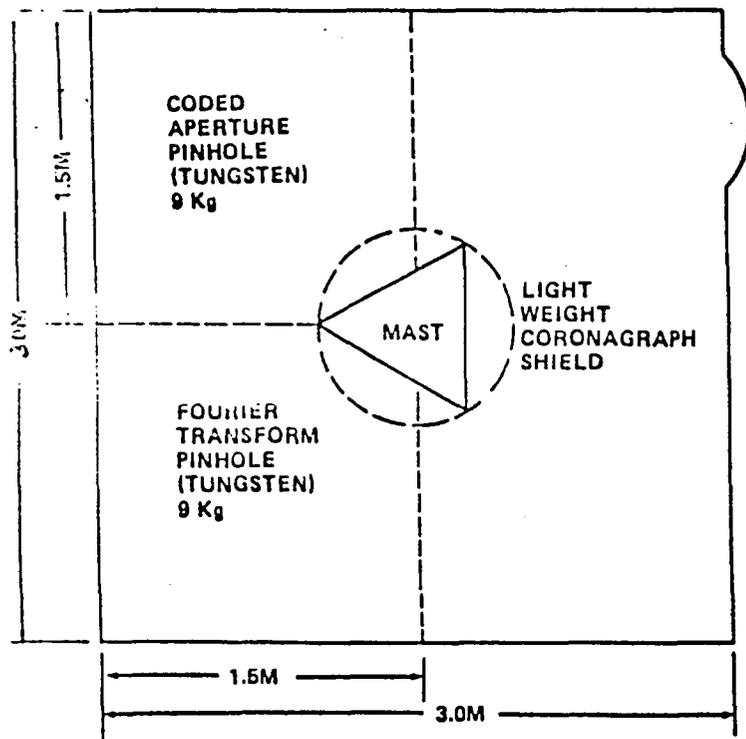


Figure 1. Pinhole occulter facility.

CONFIGURATION

MASK



DETECTORS

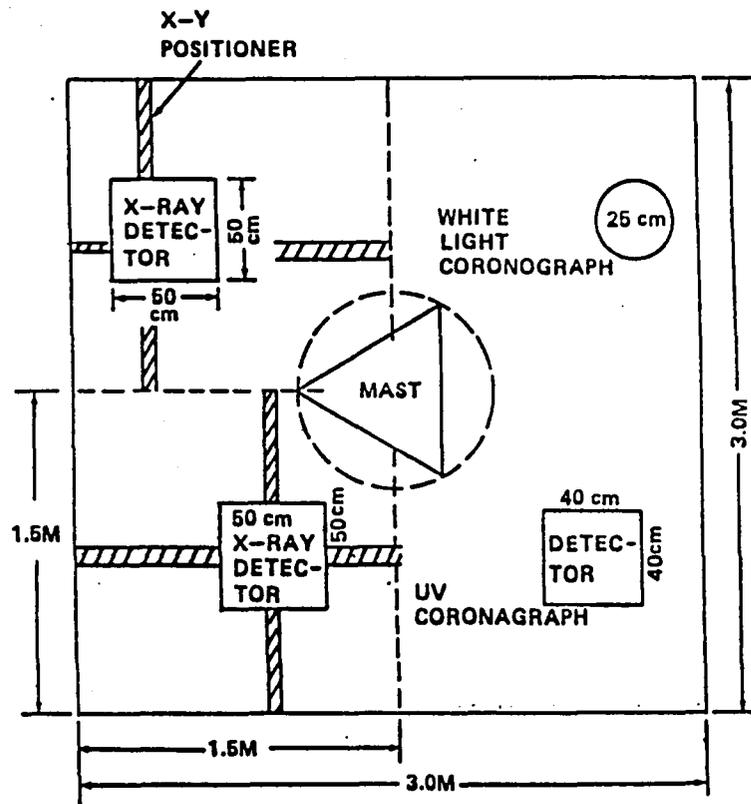


Figure 2. Pinhole occulter facility configuration.

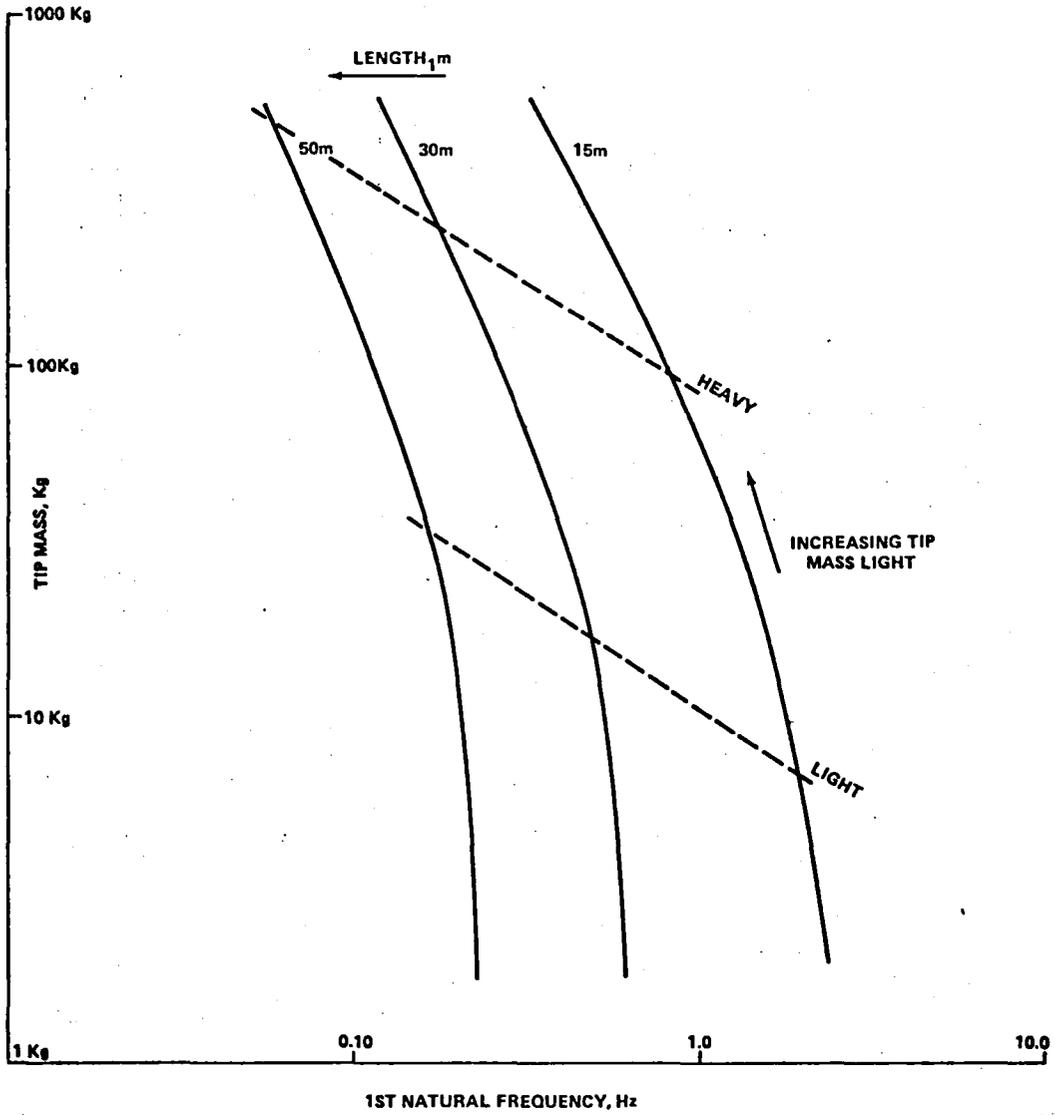


Figure 3. Boom frequency as a function of tip mass and boom length.

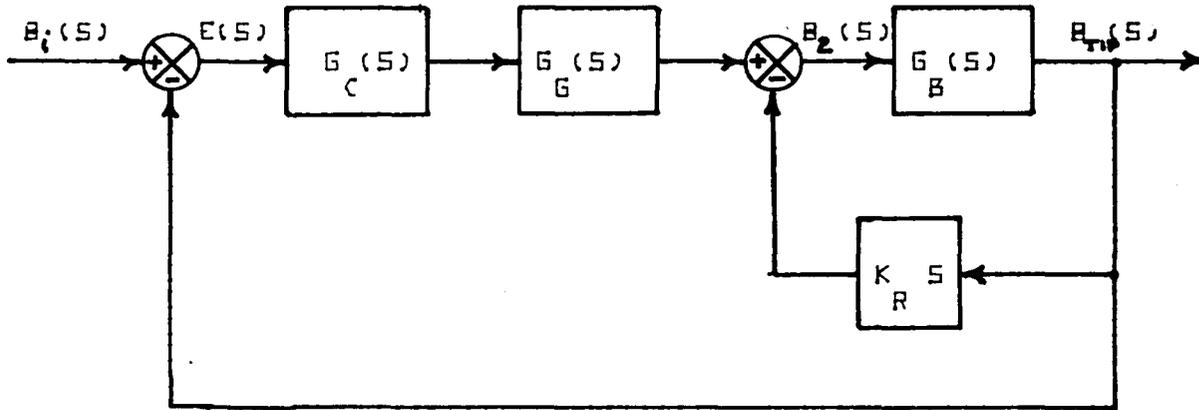


Figure 4. Simplified block diagram of pinhole control system.

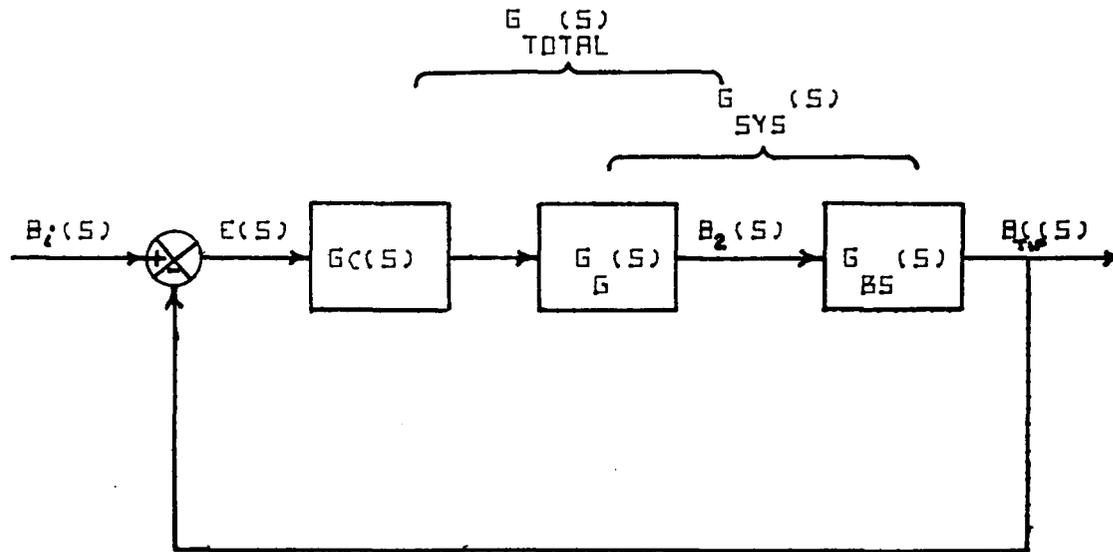


Figure 5. Reduced block diagram of pinhole control system.

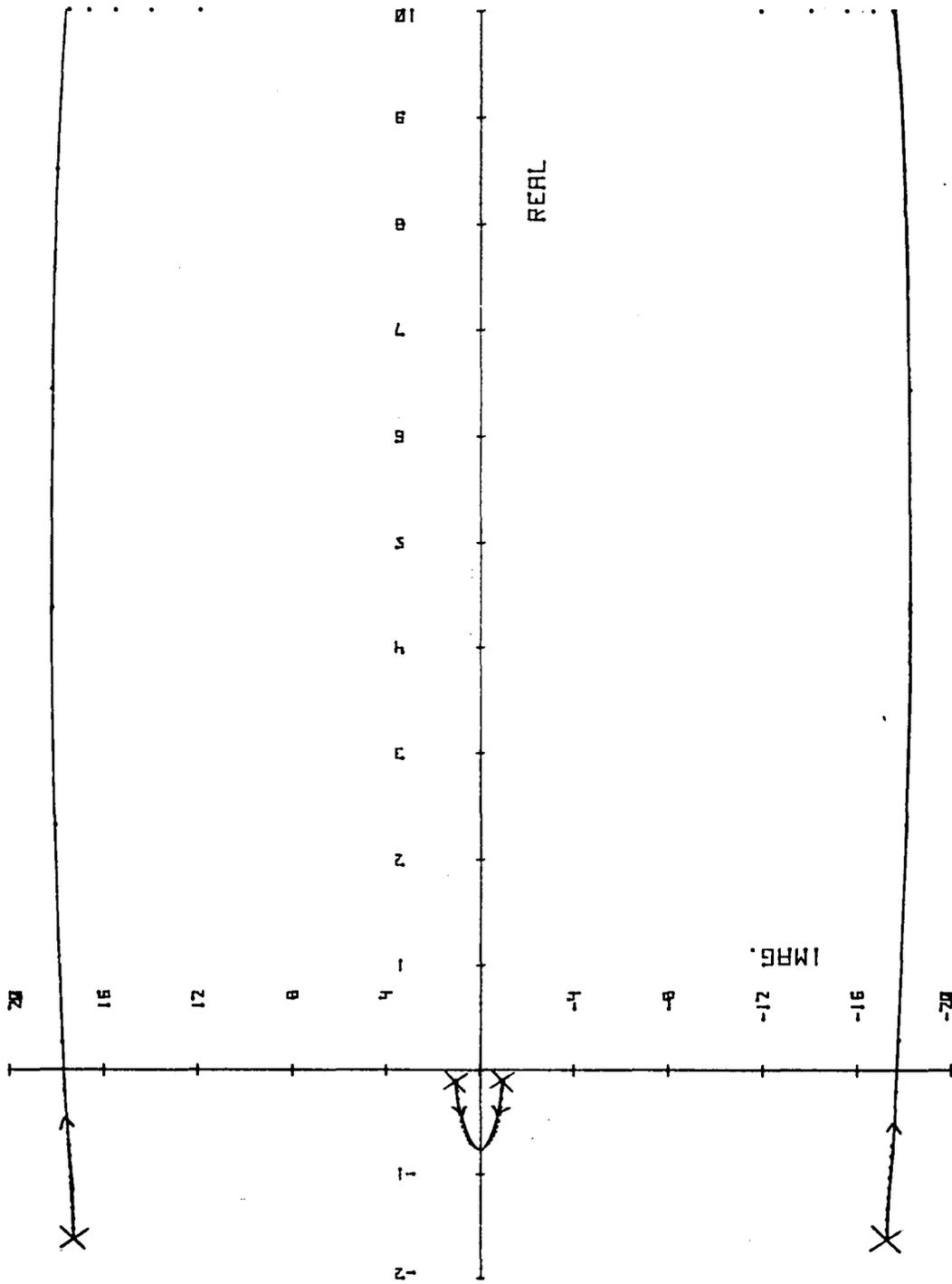


Figure 6. Open loop root location versus tach. gain.

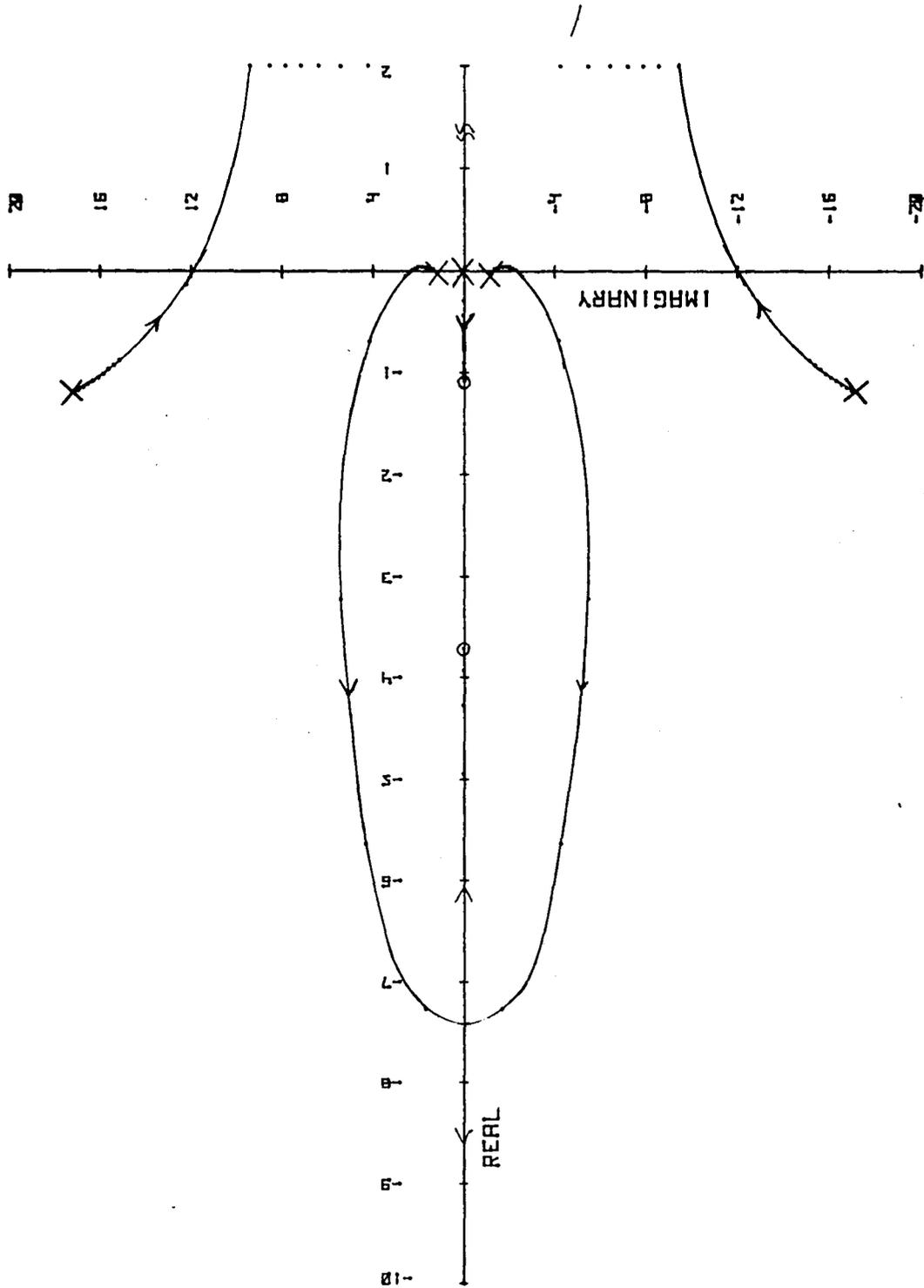


Figure 7. Root locus for uncompensated system as a function of forward gain.

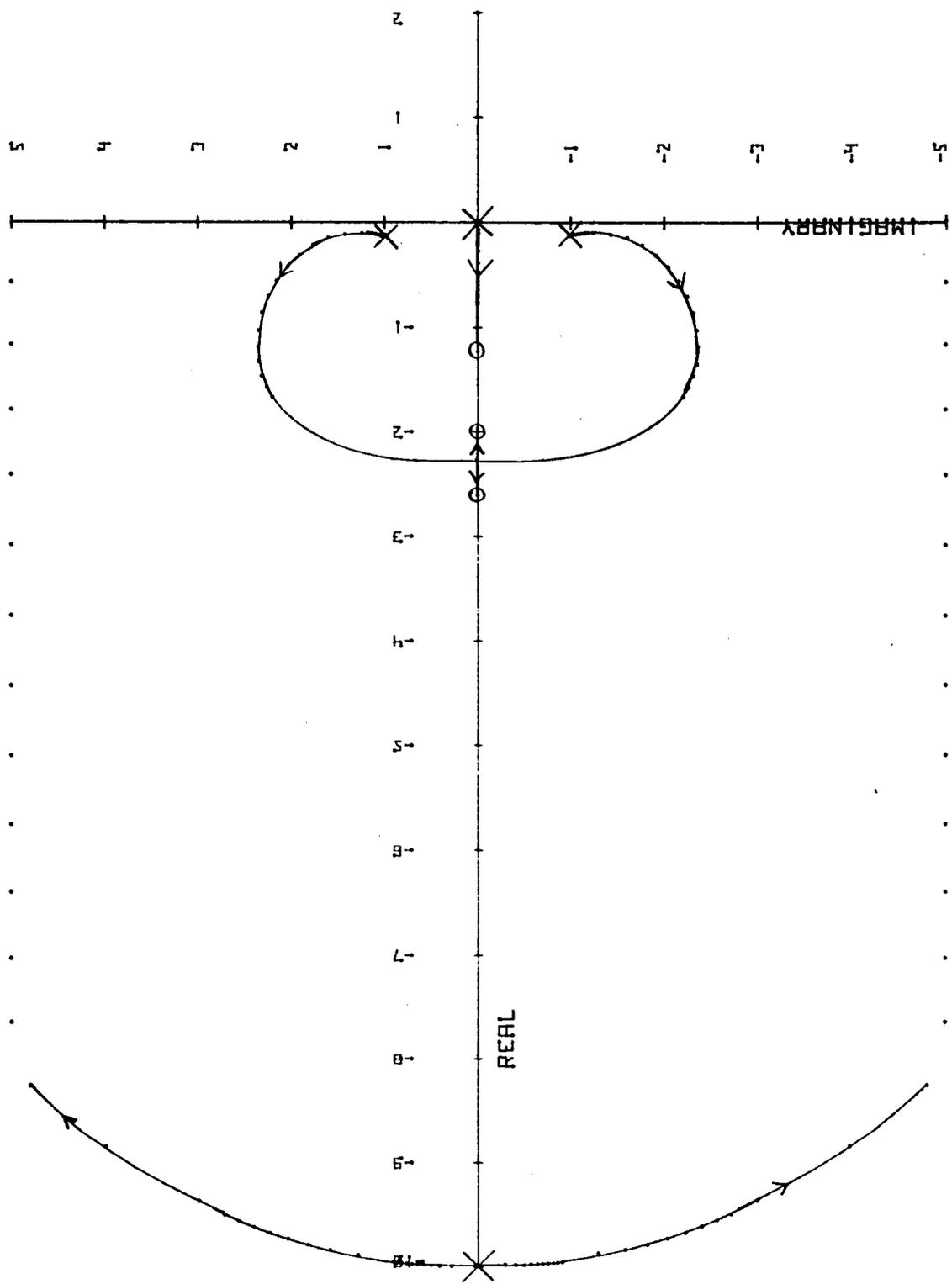


Figure 8. Root locus for compensated system as a function of forward gain.

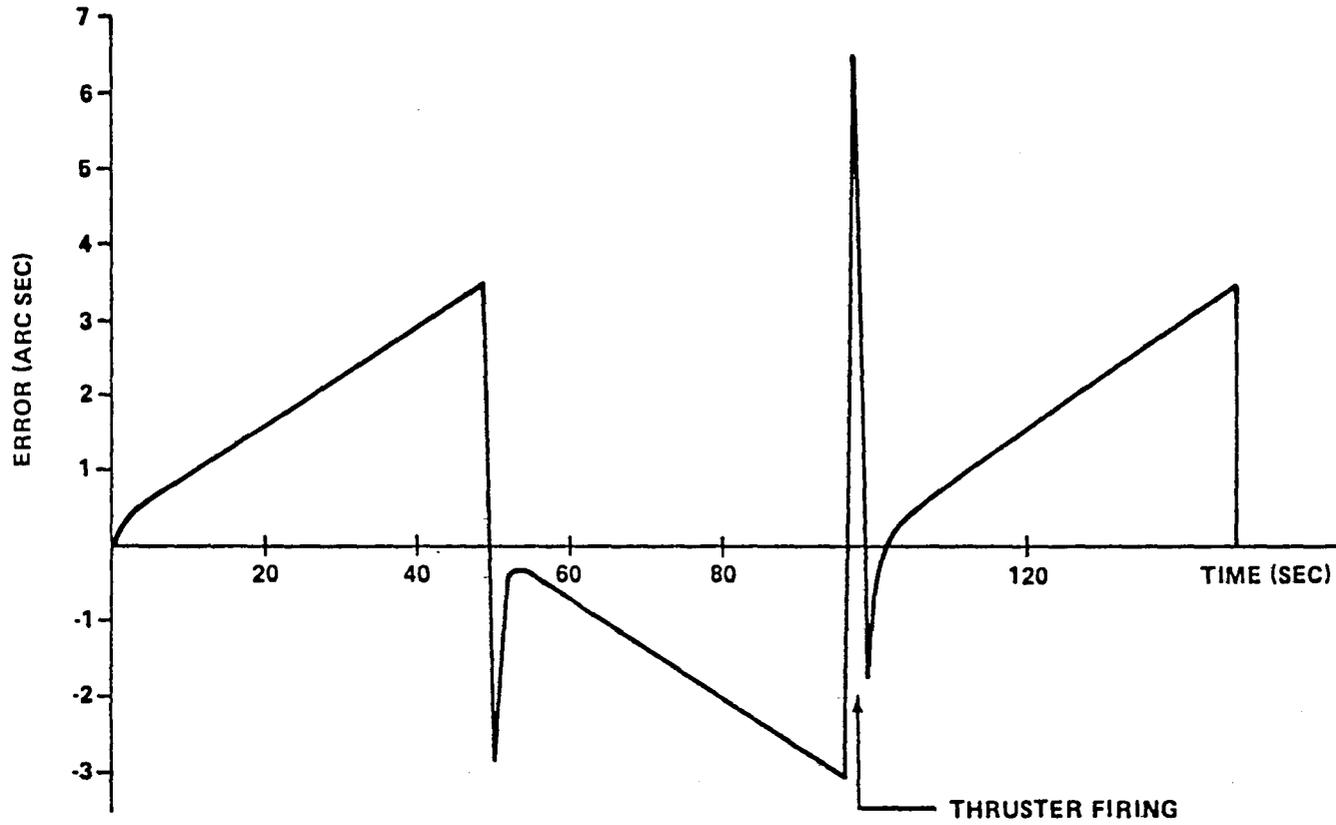


Figure 9. Error response of boom tip to thruster firing and gravity gradients.

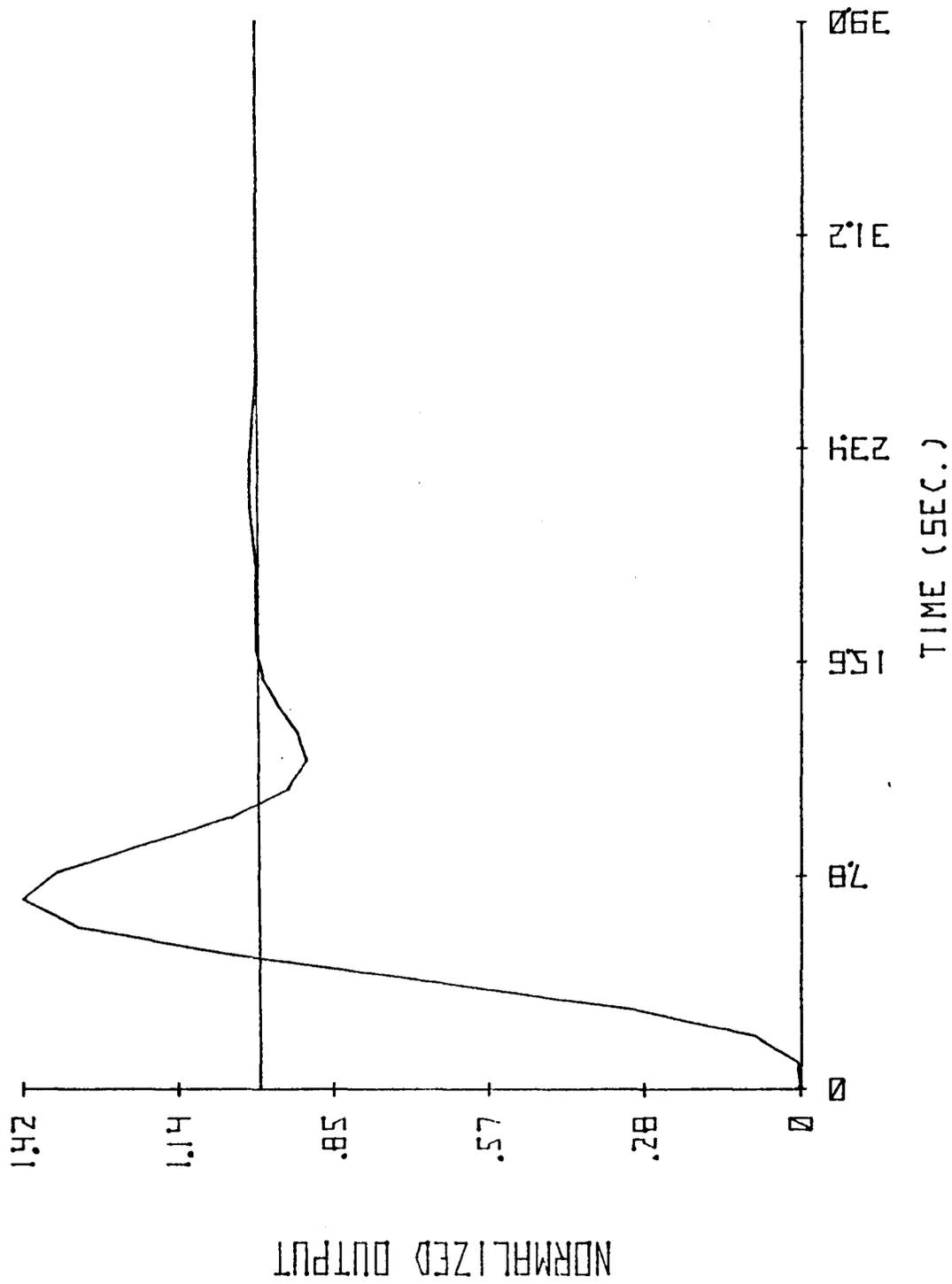


Figure 10. Step response.

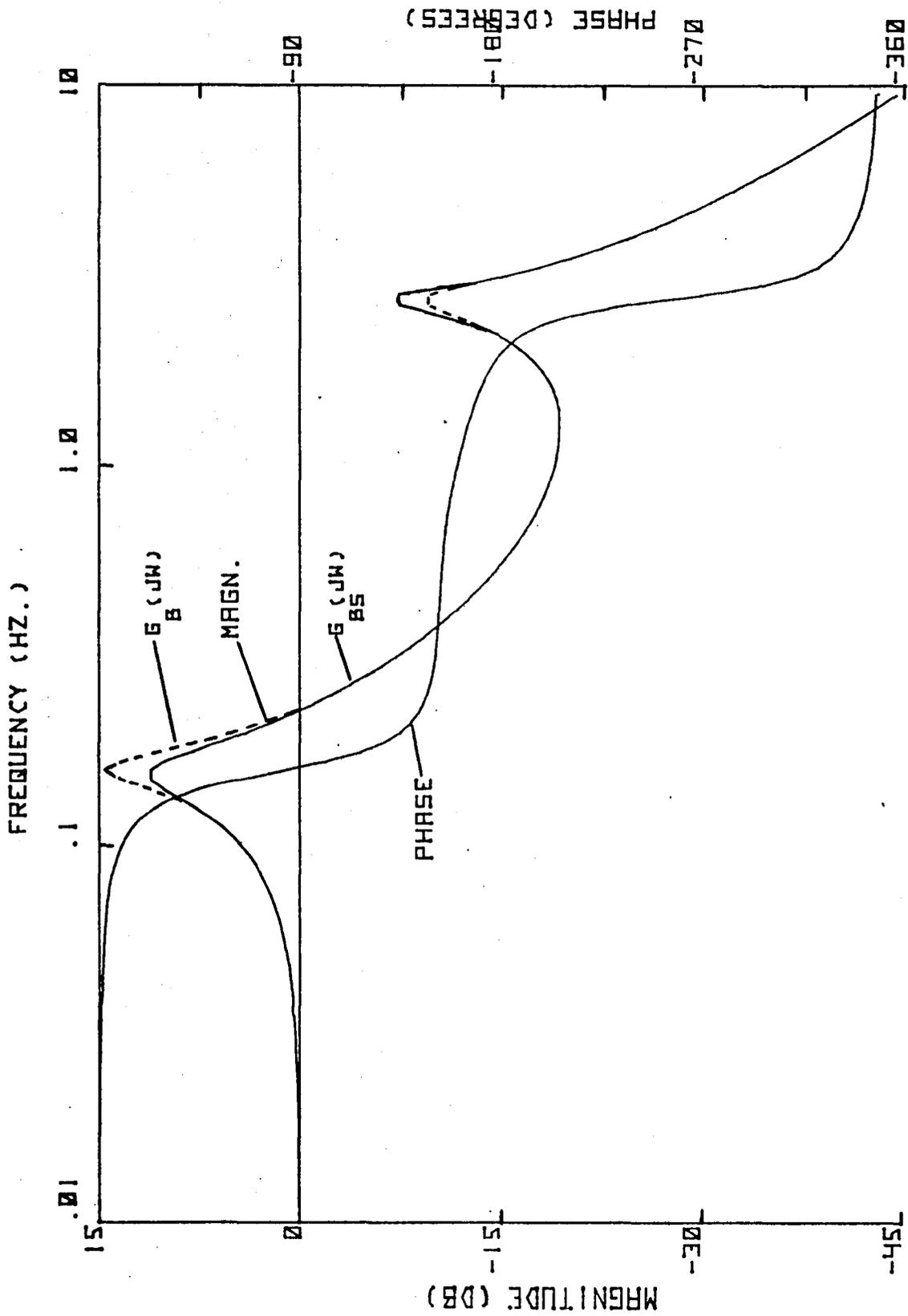


Figure 11. Frequency response of fixed boom and rate compensated boom.

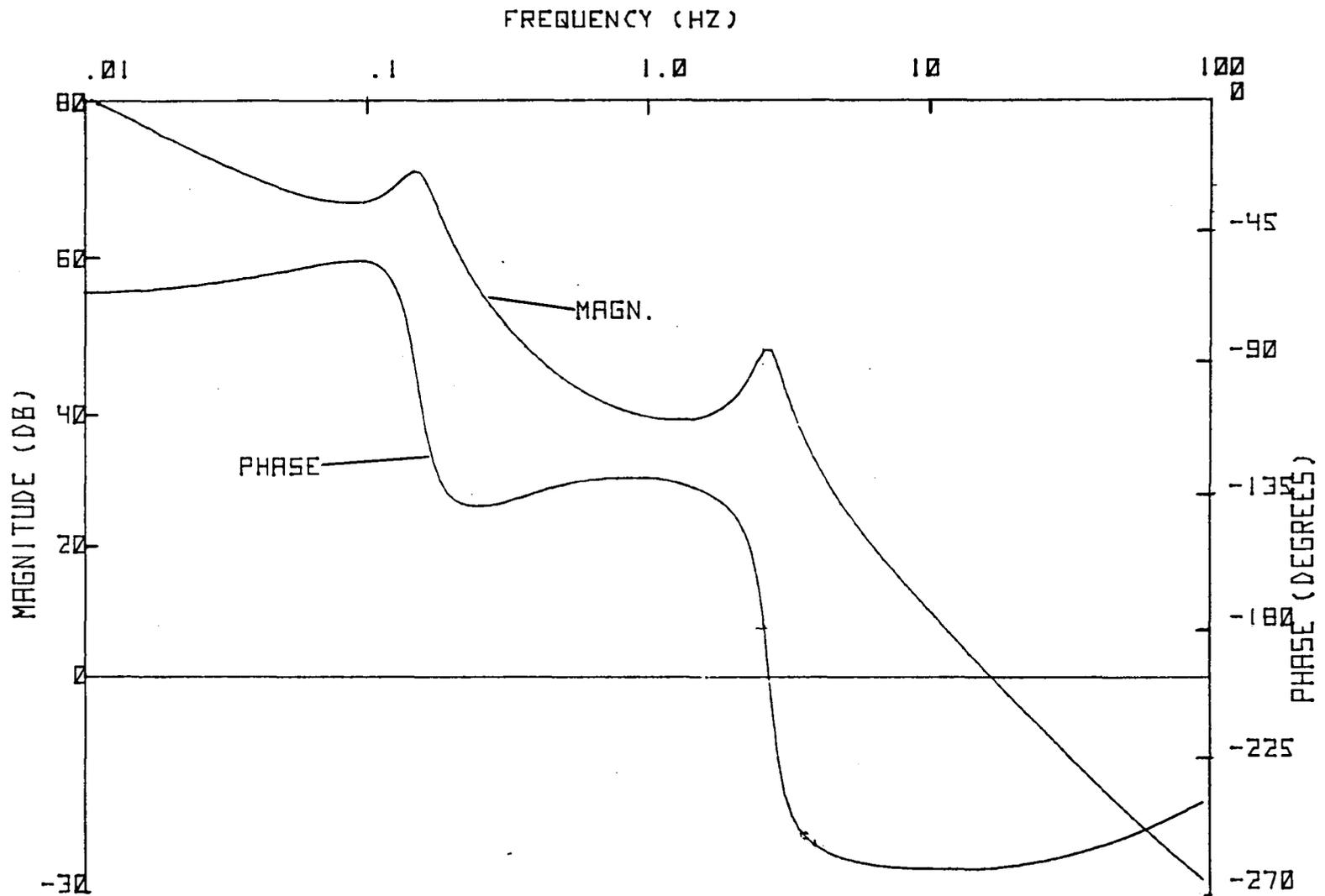


Figure 12. Frequency response of the uncompensated gimbal driven boom system.

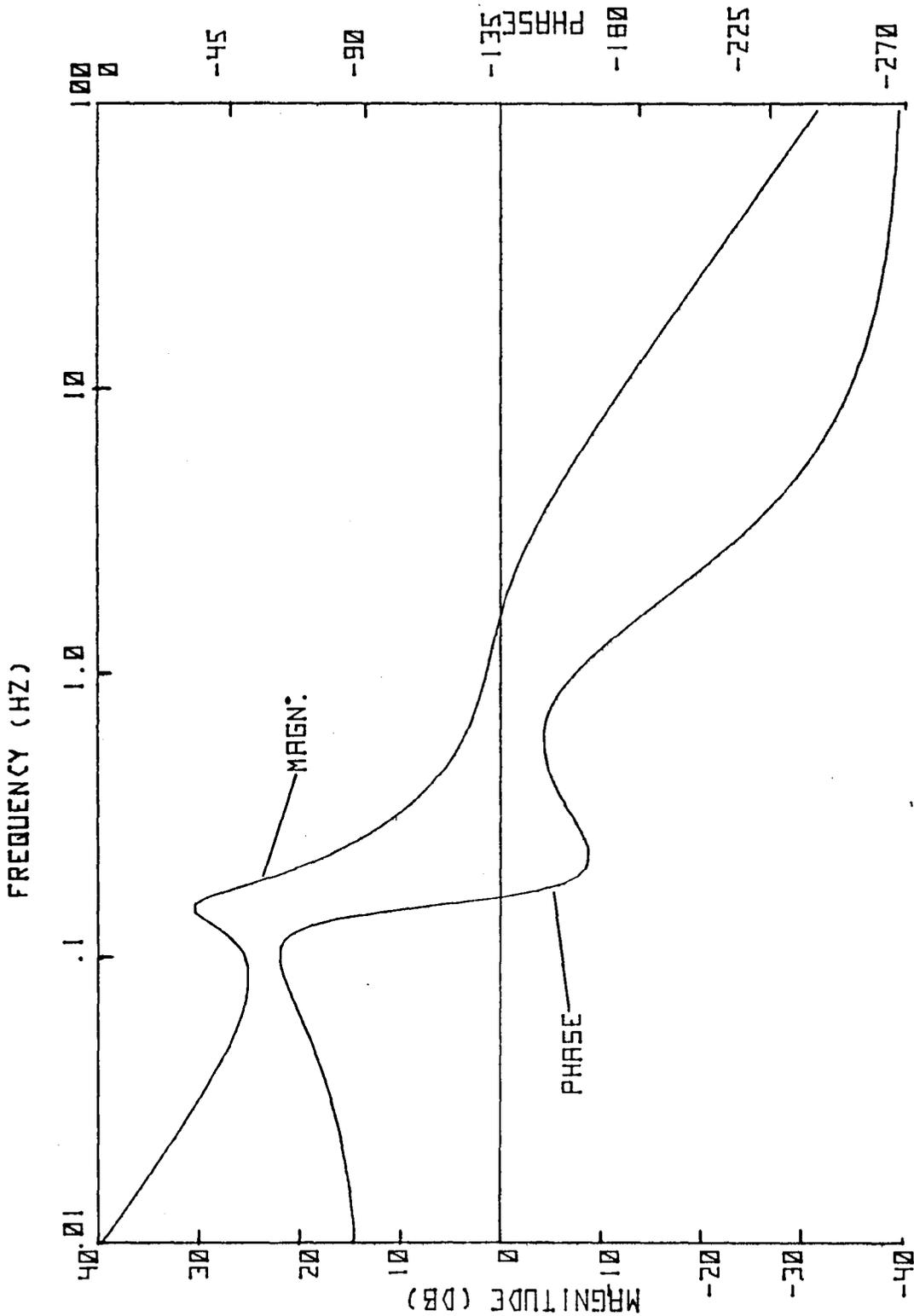


Figure 13. Frequency response of the compensated gimbal driven boom system.

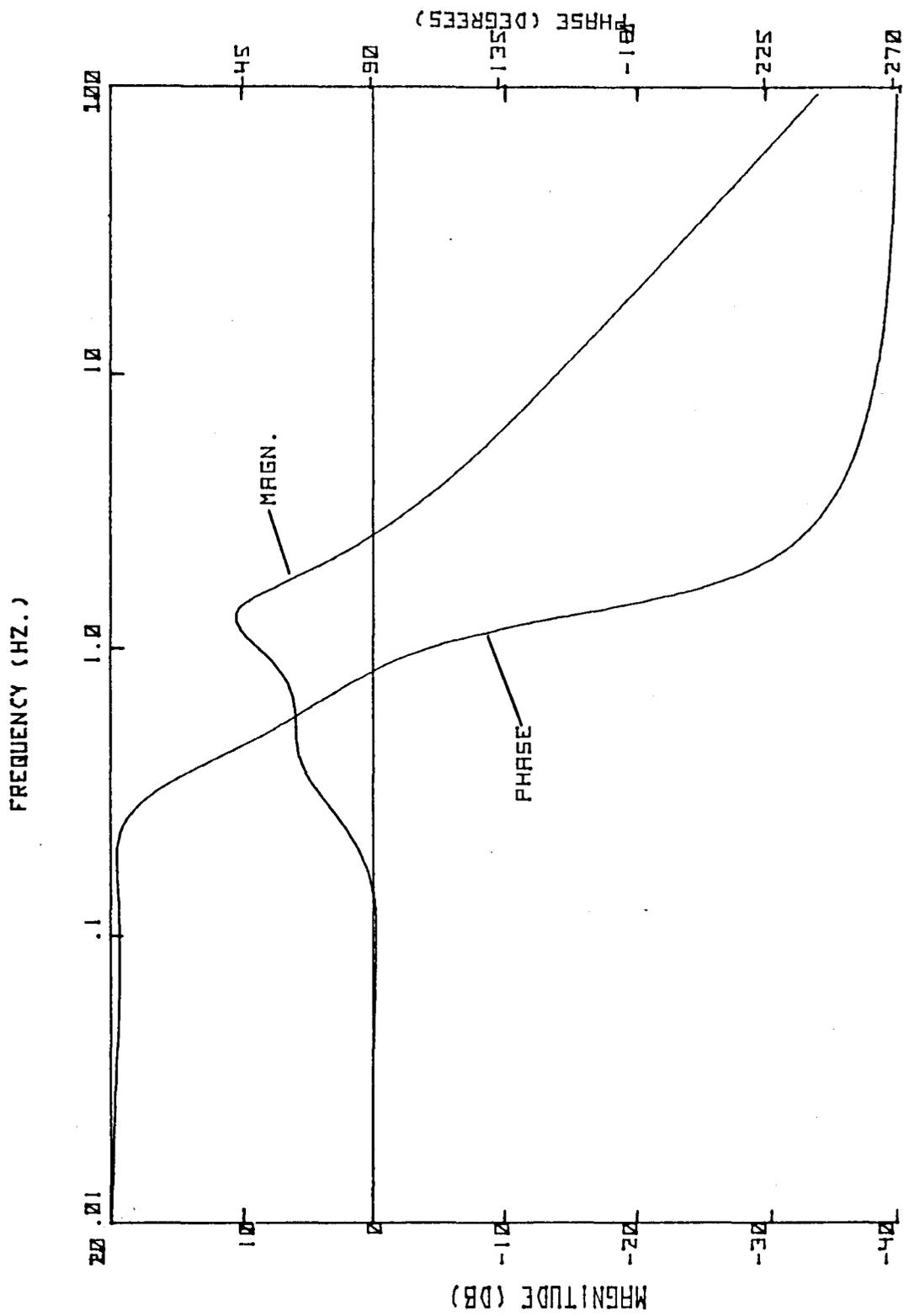


Figure 14. Frequency response of the closed loop control ratio.

1980

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MARSHALL SPACE FLIGHT CENTER
THE UNIVERSITY OF ALABAMA
FFT METHODS IN SIGNAL PROCESSING OF
THE COAL INTERFACE DETECTOR RADAR

Prepared By:	Darko Kajfez, Ph. D.
Academic Rank:	Professor
University & Department	University of Mississippi Department of Electrical Engineering
NASA/MSFC:	
(Laboratory)	Electronics & Control
(Division)	Optical & RF Systems
(Branch)	Communications & Tracking
MSFC Counterpart:	Billy R. Reed
Date:	August 1, 1980
Contract No.:	NGT-01-002-099 (University of Alabama)

FFT METHODS IN SIGNAL PROCESSING OF THE COAL
INTERFACE DETECTOR RADAR

BY

Darko Kajfex, PH. D.
Professor of Electrical Engineering
University of Mississippi
University , MS 38677

ABSTRACT

The FM radar for the coal interface detector, operating in the frequency band 2 to 4 GHz, is intended for the display of thicknesses between 2 cm and 20 cm. Because of such a short range, the thickness information is contained in the very few lowest spectral components of the output signal. To overcome this inconvenience, the Fourier series of the output signal has been augmented to approximate a Fourier integral. This modification in the signal processing resulted in a higher spectral density, which in turn enabled an easier identification of the interface position in the laboratory. The orientation and spacing of the receiving and transmitting antennas is found to have an important influence on the system performance. A field test in the coal mine is planned in the future.

ACKNOWLEDGEMENTS

I appreciate the opportunity of continuing the work on the challenging problem of the CID radar during my second year of NASA/ASEE Summer Faculty Research Fellowship Program. My NASA counterpart, Mr. Billy R. Reed provided an excellent support in all aspects of the project. Mr. Ivan A. Burroughs was very helpful in programming of the signal processing procedure, and Mr. Edmund H. Gleason provided the expertise in the selection and utilization of the various electronic instruments and subsystems. The typing of the report was performed by Joyce Gray.

INTRODUCTION

Coal interface detector (CID) radar is designed to measure the thickness of the coal layer in the longwall mining method. The frequency of the radar transmitter is modulated in a saw tooth fashion between 2 and 4 GHz. The thickness of the coal layer is measured by observing the frequency difference between the signals reflected from the front and rear surface of the coal. The radar output signal, which contains this difference frequency, is analyzed on a digital Fourier Analyzer, which then provides a display of the reflection amplitude vs. the coal thickness.

The basic principle of the CID radar has been under investigation at the MSFC for several years. The difficulty in implementing the radar lies in the presence of a multitude of erroneous signals, which can be misinterpreted as the rear surface return signals. Also, the frequency of the reflected signal is only several times (<10) higher than the modulation frequency. Therefore, it is difficult to determine which of the individual spectral components belongs to the position of the rear surface.

The presence of the multiple reflections may be somewhat reduced by using long cable sections between the individual components of the system, and by placing absorbing materials around the antennas. The second difficulty of distinguishing between the individual components in the spectrum can be greatly reduced by processing the time-domain signal before taking the Fast Fourier Transform. Examples of measured wave-forms and their spectra are included in the report.

OBJECTIVES

This report describes the work performed in the second period of the NASA/ASEE Summer Faculty Research Fellowship Program. The radar configuration is somewhat different from the one described in the last year's report (Reference (1)). The difference lies in the fact that a direct detector receiver has been utilized instead of the mixer-type receiver. The objective of the investigation described in the present report is to achieve the radar configuration which is suitable for application not only in the laboratory, but in the actual coal mine situation. Much of the effort is directed toward determining the most suitable signal processing procedure which produces a clear display of the coal thickness.

CIRCUIT DESCRIPTION

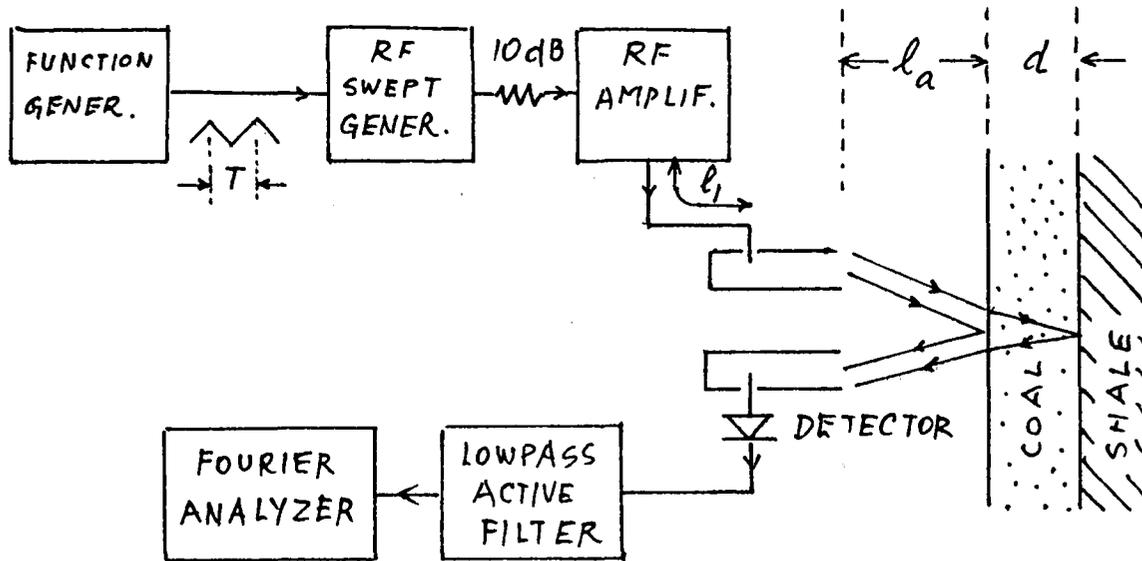


Fig. 1. Block diagram of the CID radar using the detector receiver

The transmitting part of the CID radar in Fig. 1 consists of the Hewlett Packard Function Generator Model 3312, the Wiltron Sweep Generator Model 610, the Avantek Transistor Amplifier Model 406-3M, and the Scientific Atlanta Ridged Horn Antenna Model 40M-2.0. The length of the coaxial cable between the amplifier and the antenna is $l_a = 1$ m. The receiving part of the radar consists of an identical antenna, as in the transmitting part, and a Narda Detector Mount Model 4503. The detected signal is amplified and filtered in the Ithaco Dual Filter Model 4302. The filter is set for pulse operation (Bessel type response) in the lowpass mode.

The operating data are as follows:

Frequency range of the sweep oscillator -----2 to 4 GHz.
 Sweep period----- T = 25.6 ms
 RF power delivered to transmitting antenna-----138 mW
 Lowpass filter cutoff frequency -----3.15 KHz
 (two sections in cascade, each with 10 dB gain)
 Distance between the antenna and the coal ----- $l_a = 60$ cm

As indicated in Fig. 1, the received signal consists of two parts: One part comes from the wave which was reflected from the front surface of the coal, and the other part is a weaker signal from the wave which penetrated the coal and was then reflected from the rear interface. If the coal dielectric constant is denoted by ϵ_r the sweep period by T, the coal layer thickness by d, and the RF deviation by B, the frequency difference between the front and rear reflection is

$$f_d = \frac{4Bd\sqrt{\epsilon_r}}{cT} \quad (1)$$

c is the velocity of light in free space. For the data listed above, a coal layer of thickness $d = 6$ " (15.24 cm) and the dielectric constant of $\epsilon_r = 5$ would produce a difference frequency approximately 355 Hz.

When such a mixture of the two microwave signals is applied to the square law detector, their difference frequency (here 355Hz) is obtained. This signal is interrupted by the repetition rate of the sweep generator. It is afterwards amplified and filtered before being delivered to the Fourier Analyzer Hewlett Packard Model 5451B, which performs the signal processing and displays the result.

The radar system described above is similar in concept to the radar described in Reference (2). The main difference of the present system is the use of a direct detector system instead of the heterodyning receiver, and the use of a digital signal processing system instead of an analog one.

SIGNAL PROCESSING

The tests in the laboratory are performed on slabs of "artificial coal". These are slabs of different thicknesses, made from a mixture of dielectric and conducting materials, so that the dielectric constant and the loss tangent approximate the values of the actual coal from the mine. In the particular measurement to be described, the slab thickness 15.5 cm (6") and of the cross section 61x61 cm (2'x2') was backed by an aluminum foil for better rear surface reflection. The received signal at the input to the Fourier Analyzer is shown in Fig. 2.

The observed signal is periodic with the period $T = 25.6$ ms. The sawtooth waveform which modulates the transmitter is shown in Fig. 3. The characteristics of the transmitter are such that the modulation voltage 2.4 V produces an output of 2 GHz and 4.8 V produces an output frequency of 4 GHz. It is seen from Figures 2 and 3 that the detected signal displays an even symmetry within each half period. This is to be expected, because the transmitted frequency attains the same value twice within each period: once on the upswing, and once on the downswing of the modulation signal.

The signal shown in Fig. 2 has been sampled with the time intervals $50 \mu s$, the total number of sample points being 1024. The waveform shown includes two full periods of the signal. If now only one period is retained (first 512 points) and the Fast Fourier Transform of this waveform is computed, the resulting spectrum is such as shown in Fig. 4. Each spectral component makes one point in the plot. Due to an automatic plotting procedure, the points are interconnected by straight lines, creating the appearance of a continuous spectrum. The spectrum actually consists of discrete frequencies, separated from each other by the repetition frequency 39 Hz. According to (1), the beat frequency for the 6" artificial coal is 355 Hz. From Fig. 4 it can be seen that the two longest spectral components are components number one and number eight. Thus, by neglecting the first component which belongs to the repetition frequency, the frequency of the return signal is estimated to be eight times the repetition frequency, or 312 Hz (instead of 355 Hz).

There are several inconveniences associated with the display such as in Fig. 4. In the first place, the repetition frequency usually comes out to be the strongest component, which fact makes it difficult to read the thickness of the thin coal layers, where spectral components are very close to the repetition frequency. Furthermore, the components **immediately** next to the observed return frequency often have a small amplitude, so that the signal is not always readily identifiable as in Fig. 4, where the seventh, eighth, and ninth spectral components are all large. It was decided that a difficulty in locating the return signal is caused by the fact that the spectral components are too far apart

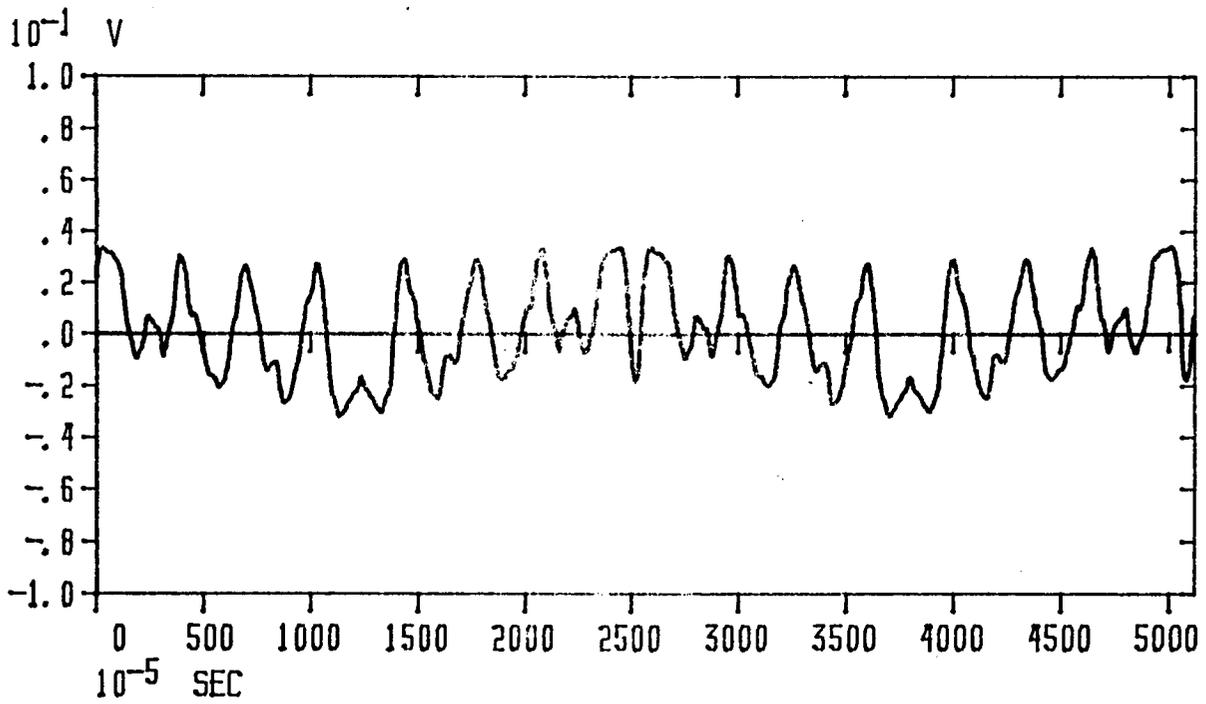


FIG. 2 RECEIVED WAVEFORM AT THE INPUT OF THE FOURIER ANALYZER

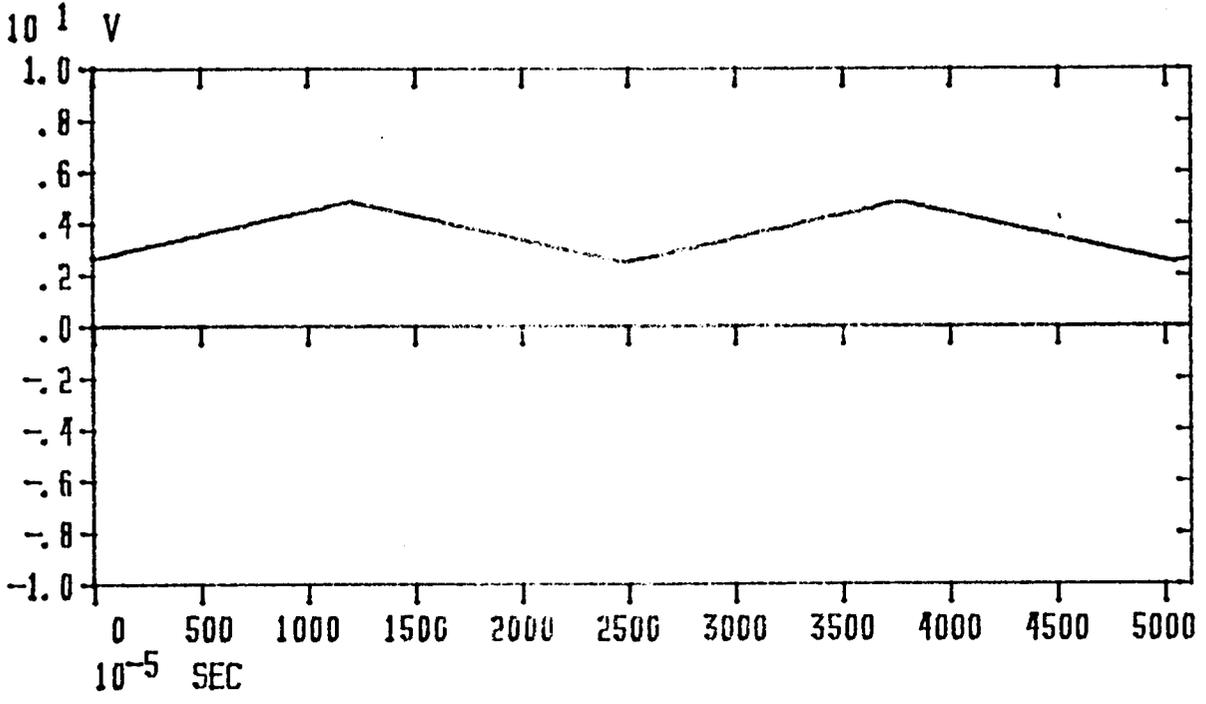


FIG. 3 MODULATION SIGNAL AT THE OUTPUT OF THE FUNCTION GENERATOR

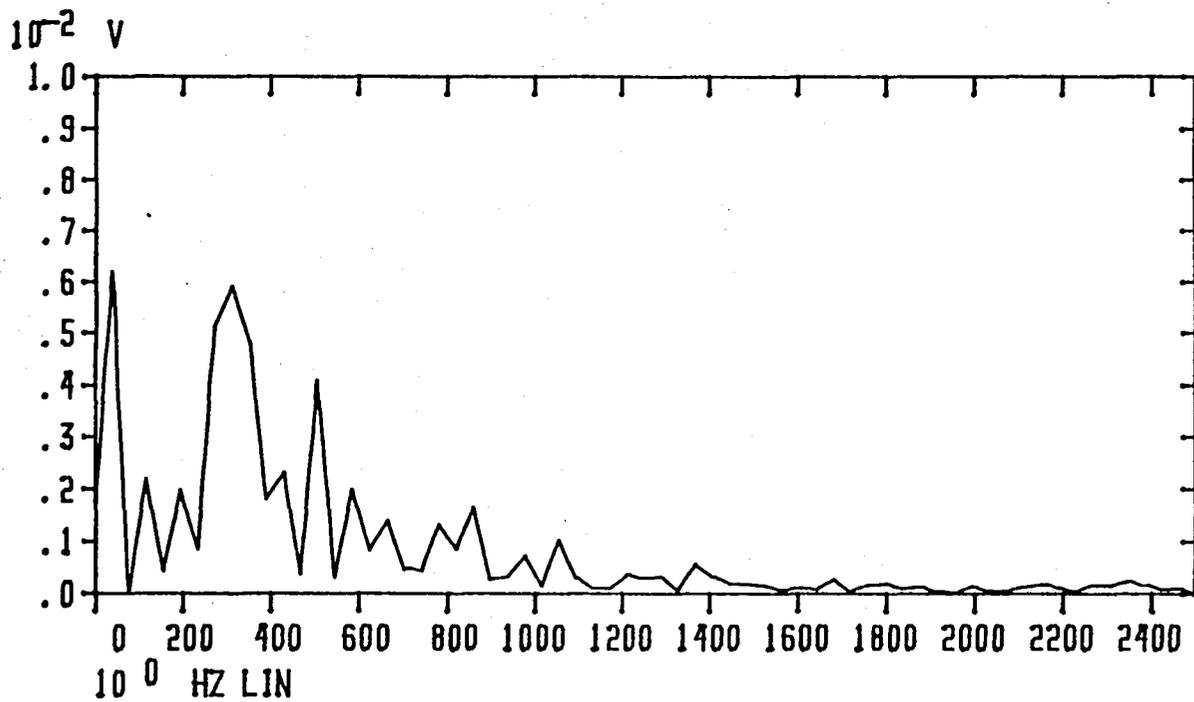


FIG. 4 FOURIER SERIES OF THE PERIODIC SIGNAL

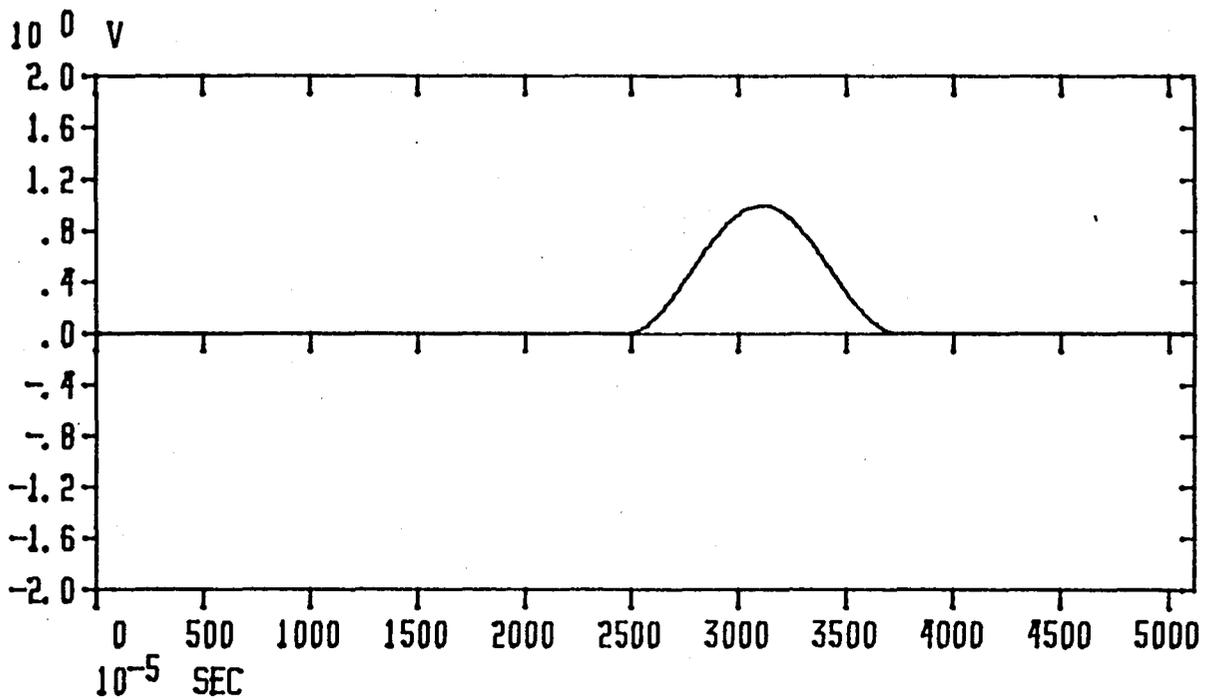


FIG. 5 HANN TIME WINDOW

from each other. In order to create a more densely populated spectrum, advantage has been taken of the digital signal processing in the following manner.

The original sampled signal from Fig. 2 consists of 1024 points. The sample extends over two periods, each period has a half-period symmetry. Therefore, the basic information is contained within 256 points. Thus, a time window is created, of the length 256 sample intervals, as shown in Fig. 5. This is the so-called Hann window (3) of the shape $1 - \cos x$. When the sampled signal is multiplied with the window, the result looks very much like a single burst of oscillations shown in Fig. 6. The total sample length is still 1024 points, but 75% of the sample points are forced to be zero. The Fast Fourier Transform performed on these 1024 sample points will have much larger spectral density from the spectrum in Fig. 4, and thus will allow more accurate reading of the coal thickness.

Before the spectrum is displayed, another operation is performed. The radar return signal is redundant in the sense that the half period which corresponds to the modulation upswing is a mirror image of the other half period which corresponds to the modulation downswing. Therefore, another Hann window is created, shifted for 256 points. Again the sampled signal is multiplied with this second window, and the Fast Fourier Transform is completed. The resulting spectrum is then multiplied with the previously evaluated spectrum. The procedure is equivalent to taking the convolution of the two half periods in the time domain. This insures that the information contained in each half period contributes equally to the final result. The amplitude of the resulting spectrum is then displayed in a logarithmic scale, as shown in Fig. 7.

The spectrum in Fig. 7 is dense enough for each strong spectral component to be easily identified. When comparing this spectrum with the one in Fig. 4, it is noticed that now the return signal has the largest amplitude, and is thus easily recognized. It appears that the amplitude of this return is about 20 dB larger than other spectral components on either side. However, it should be remembered that the spectrum in Fig. 7 has been obtained by multiplying the two spectra emanating from each of the two half periods, so that the resulting product is made of the squared amplitudes.

Because of this, the number of decibels in Fig. 7 must be divided by two, so that the actual difference between the largest lobe and the next largest lobe is only about 10 dB, and not 20 dB.

The frequency corresponding to the rear surface reflection in Fig. 7 is 310 Hz, instead of the predicted value 355 Hz. The discrepancy may have been caused by an inaccurate knowledge of the frequency deviation B and the dielectric constant ϵ_r which both enter into equation(1).

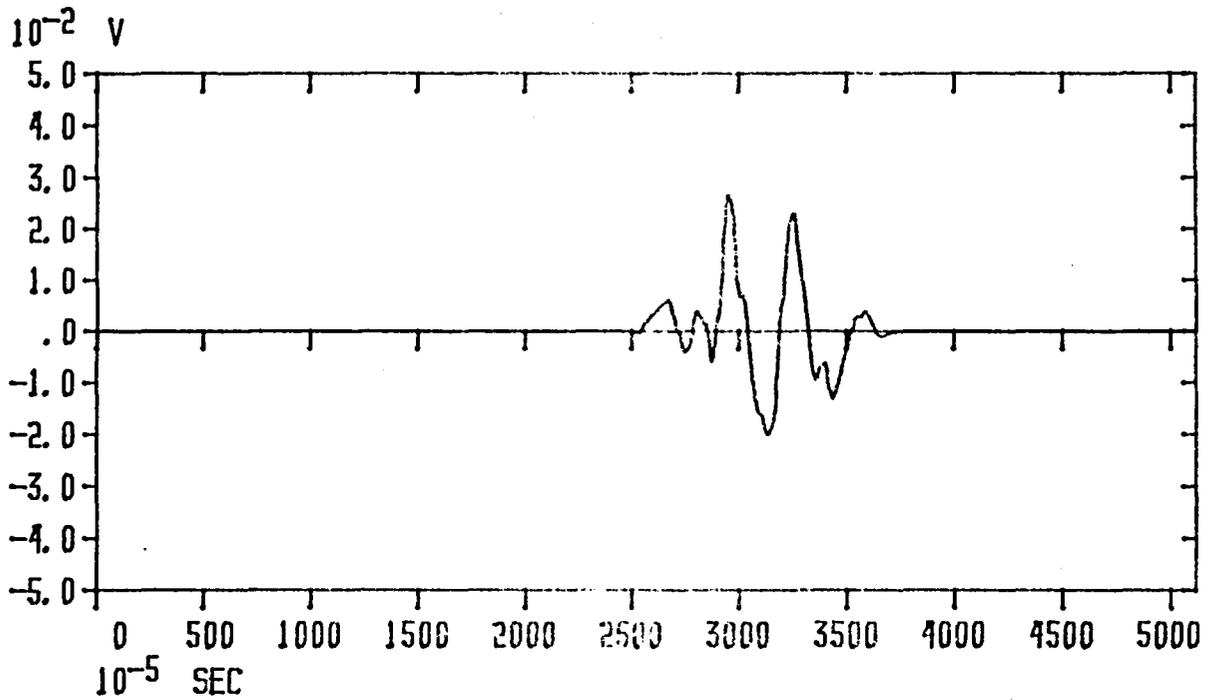


FIG. 6 RECEIVED WAVEFORM MULTIPLIED WITH THE HANN WINDOW

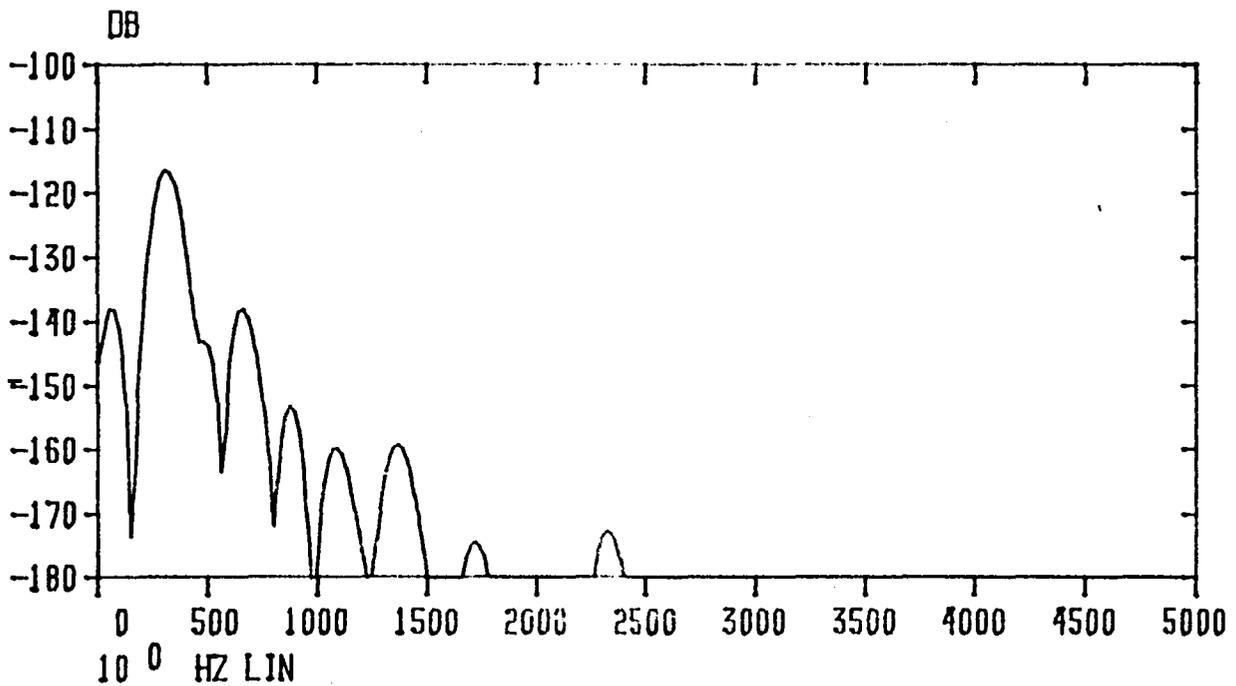


FIG. 7 FOURIER TRANSFORM DISPLAY

When a sinusoidal signal is interrupted by the square pulses, the spectrum of such a modulated signal contains not only the carrier frequency, but also a number of sidelobes which are decreasing in amplitude according to $\sin x/x$ function. The first sidelobe of this function is 13.5 dB smaller than the maximum. The radar return signal is switched each time the modulation waveform changes its slope. Therefore, the observed spectrum will contain considerable sidelobes on each side of the carrier frequency (355 Hz in the example discussed above). These sidelobes may be misinterpreted as apparent radar returns. One way of reducing these sidelobes is to use the Hann window, such as shown in Fig. 5. The sidelobes of the sinusoidal signal modulated with the Hann window are 37 dB smaller than the maximum. This is the reason that the sampled signal in Fig. 6 has been multiplied by the Hann window.

One unavoidable consequence of reducing the sidelobes by using the Hann window is the doubling of the spectral width of the main lobe, when compared with the rectangular window (4) should two close surfaces be distinguished in the radar display, their difference in frequency should be at least half width of the main lobe. For the sinusoidal signal multiplied with the Hann window, the first zero appears at double the value of the $\sin x/x$ function:

$$\pi \Delta f \frac{T}{2} = 2 \pi$$

The frequency difference between the main lobe maximum and the first zero is

$$\Delta f = \frac{4}{T} \quad (2)$$

If two signals are reflected from two very closely located surfaces their frequencies may differ for less than Δf , and cannot be distinguished on the display. Substituting (2) into (1), we find the minimum resolution distance between the two reflections in the coal

$$d_{res} = \frac{2c}{B \sqrt{\epsilon_r}} \quad (3)$$

The resolution distance is independent of the repetition frequency. The only way to obtain a finer resolution is to increase the frequency deviation. For the system with $B = 2$ GHz, the resolution distance is $d_{res} = 6.7$ cm.

When two reflection surfaces are closer to each other than d_{res} , their main spectral lobes will merge into a single lobe, and the two returns will not be distinguishable on the radar display. One way to reduce the resolution distance is to use the rectangular window instead of the Hann window. This would split the resolution distance in half at the expense of increasing the sidelobes. It follows from the previous discussion that large sidelobes are undesirable. Therefore, the fact that the frequency deviation of the radar is 2 GHz sets on a priori limit of 6.7 cm as the smallest recognizable coal thickness (3.3 cm with the rectangular window).

ANTENNA CONSIDERATIONS

Little is known about the design of microwave antennas suitable for sensing the material properties in close range. The majority of microwave antennas are designed to produce a given electromagnetic radiation in the far field region. In the early experiments with the CID radar, the transmitting and receiving antennas have sometimes been placed less than one wavelength apart from each other or from the coal interface. It has been empirically found afterwards that these distances must be increased to several wavelengths.

One important property of the antenna to be used for the CID radar is low dispersion. The time delay for the signal, to pass from the antenna terminal to an obstacle in front of the antenna and back to the terminal, should be independent of frequency. One of the popular wideband antennas, the log periodic array, is ill suited for the task, as pointed out in [2].

Another important property of the pair of antennas to be used in the radar is a high decoupling when they are placed side by side. By looking at Figure 1, it becomes obvious that a direct propagation from the transmitting to the receiving antenna creates a third wave impinging upon the detector. The difference frequency of that wave and the front surface reflection may produce an erroneous "return" signal on the radar display.

Without making any theoretical investigation, and without even comparing the performance of various commercially available antennas, the pair of ridged horn antennas has been assigned to do the job. The orientation and the distance between them has been experimentally adjusted to give the best performance as follows.

Figure 8 depicts two possible relative positions of the receiving and transmitting antennas, denoted parallel and collinear. It has been found experimentally that the collinear orientation offers higher isolation between the antennas. The distance D has also been varied, and the value $D=25$ cm appears to be sufficient. The measurement has been conducted when the antenna-to-coal distance is $l_{a1} = l_{a2} = 60$ cm.

Another unexpected critical adjustment proved to be the angle θ in figure 9. This angle measures the departure of the coal surface from making a right angle with the antenna system. It has been found experimentally that the return signal from the rear surface drops considerably when the difference $l_{a1} - l_{a2}$ becomes larger than 2 cm. For $D = 24$ cm and $b = 10$ cm, this implies that the angle between the coal surface and the antenna boresight should not depart from the right angle for more than $\theta = 3.3$ degrees.

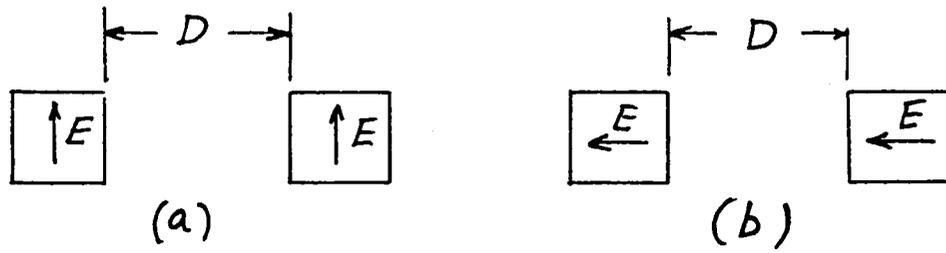


Fig. 8. POLARIZATION OF ANTENNAS:
 (a) PARALLEL
 (b) COLLINEAR

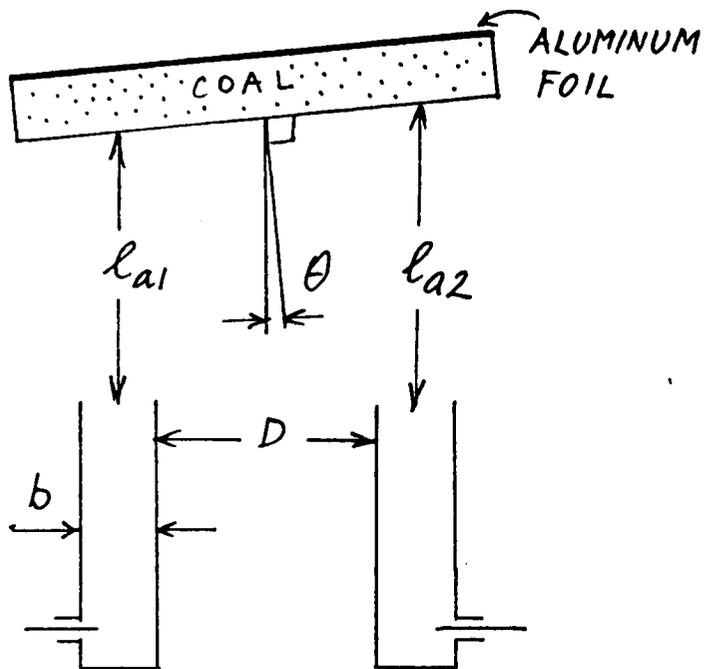


FIG. 9. INCLINATION OF THE COAL SURFACE
 WITH RESPECT TO THE ANTENNA SYSTEM

CONCLUSIONS

The CID radar system which has been developed has certain advantages over the previously utilized systems. The digital processing of the signal on a Fourier Analyzer System enables more convenient display which seems to improve the separation between the actual return and other unintentionally created signals. When also proper care is exercised in antenna positioning, the system seems to be capable of giving meaningful real-time results in a coal mine environment.

The improvement in the resolution distance may be achieved only by increasing the frequency deviation of the system. Further improvements on the antenna system may also bring an enhancement of the desired return signal and a reduction of the undesired signals.

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EXCHANGE OF STABILITY FOR A COLUMN OF FLUID WITH VARIABLE
RAYLEIGH NUMBER UNDER FREE BOUNDARY CONDITIONS

Prepared By: Letitia Korbly, Ph.D.
Academic Rank:
University and Department: University of Alabama in
Birmingham
NASA/MSFC: Space Sciences
(Laboratory) Atmospheric Sciences
(Division) Fluid Dynamics
(Branch)
MSFC Counterpart: George H. Fichtl
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EXCHANGE OF STABILITY FOR A COLUMN OF FLUID WITH VARIABLE
RAYLEIGH NUMBER UNDER FREE BOUNDARY CONDITIONS

by

Dr. Letitita Korbly
Department of Mathematics
University of Alabama in Birmingham
Birmingham, Alabama

ABSTRACT

Motion starts in a column of fluid subject to an adverse temperature gradient when the gravitational forces are large enough to overcome those of viscosity. This point is governed by the critical (smallest) Rayleigh number.

When the column is heated rapidly from below and not well mixed the Rayleigh number varies with height.

For such a system with harmonic boundary conditions the principle of exchange of stabilities is shown. Attempts to show this principle under rigid and semirigid boundary conditions failed.

The Rayleigh number was assumed to follow the non-dimensional distribution $\mathcal{R}(z) = R \left(\frac{z+z_0}{z_0} \right)^{-\alpha}$, $\alpha = 4/3$ or $\alpha = 3/2$, $z_0 = 10^{-5}$, $0 \leq z \leq 1$. The lowest values for at which there would be convection was estimated at 8.512×10^8 for $\alpha = 4/3$ and 4.7023×10^9 for $\alpha = 3/2$. The eigenfunction was approximated in both cases.

This work has application to both columns of air in the Earth's atmosphere and the GFFC experiment to be flown in Spacelab 3.

INTRODUCTION

The Navier Stokes equations have classically been used to describe the motion of fluid in the presence of temperature gradients.

In the Benard problem, the fluid is assumed to be incompressible and at rest; it is assumed to be uniformly heated from below. The vertical temperature distribution is assumed to be linear. The equations are then linearized by looking at temperature and velocity perturbations.

The equations are then reduced to a single sixth order linear partial differential equation with constant coefficients in the vertical velocity perturbation w . Fourier transform in the horizontal space variables has reduced the number of independent variables. Only time, t , and height, z , remain. The equation is then transformed in t and non-dimensionalized. It becomes:

$$[(D^2 - a^2)(D^2 - a^2 - \sigma)(D^2 - a^2 - \varphi\sigma) + Ra^2] W(z) = 0 \quad 0 \leq z \leq 1$$

Here the vertical velocity perturbation is

$$w = e^{\sigma t + i(a_1 x + a_2 y)} W(z)$$

Here $DW = W'$ and $a^2 = a_1^2 + a_2^2$.

The principle of exchange of stabilities holds if convection modes which neither grow nor decay with time are time independent. If R , the Rayleigh number, and φ , the Prandl number, are constant, exchange of stabilities can be shown to hold under both free and rigid boundary conditions.

If R is allowed to vary with height, what will happen? We were able to show the principle of exchange of stabilities holds under free, but not rigid or semirigid, boundary conditions.

Having this result, the onset of convection was investigated for certain distribution of R which model what occurs in the summer atmosphere below the inversion. The values for R used were

$$R(z) = R \left(\frac{z}{z_0} \right)^{-\alpha} \quad z_0 \leq z \leq d, \quad \alpha = 4/3 \text{ or } \alpha = 3/2.$$

Here d is the height of the inversion, z_0 is surface roughness length, and α is empirically chosen. This is nondimensionalized, and the critical (smallest) value of R for which convection takes place is estimated.

The results of the calculations were checked by determining the temperature gradient at the surface, $\left. \frac{\partial \theta}{\partial z} \right|_{z=z_0} = \beta_0$. This determined $\frac{\partial \theta}{\partial z} = \beta_0 z^{-4/3}$ which was then integrated to give a temperature drop of roughly 15° in the first meter above the surface. This figure checks reasonably well with the data.

EXCHANGE OF STABILITIES

These calculations show that the principle of exchange of stability holds for a column of fluid heated from below if the Rayleigh number is variable when the boundary conditions are harmonic. The Boussinesq approximations are used. The transverse velocities, u and v , the pressure, p , and the temperature perturbation, θ , are eliminated. The nondimensional equation is

$$\nabla^2 (D_t - \nabla^2) (\rho D_t - \nabla^2) w = -\nabla_h^2 R w$$

Here $\nabla^2 w = w_{xx} + w_{yy} + w_{zz}$; $\nabla_h^2 w = w_{xx} + w_{yy}$
 $\rho = \nu/K$ is the Prandtl number, and the Rayleigh number,
 $R = g \alpha d^3 / (\nu k)$, depends on the height, z .

Proceeding with the analysis into normal modes, set

$$w = e^{\sigma t + i(a_1 x + a_2 y)} W(z)$$

The operator then becomes:

$$* \quad (D^2 - a^2)(D^2 - a^2 - \sigma)(D^2 - a^2 - \rho\sigma) W = -R a^2 W$$

Here $a^2 = a_1^2 + a_2^2$ and $DW = W'$.

The harmonic boundary conditions are:

$$\begin{aligned} W(0) &= 0 = W(1) \\ W''(0) &= 0 = W''(1) \\ W''''(0) &= 0 = W''''(1) \end{aligned}$$

Multiplying (*) by \bar{W} and integrating forms the following equation.

$$** \quad \int_0^1 (D^2 - a^2)(D^2 - a^2 - \sigma)(D^2 - a^2 - \rho\sigma) W' \bar{W} dz = -\int_0^1 R a^2 |W|^2 dz$$

The following integral occurs in the analysis of (**):

$$I_1 = -\int_0^1 (D^2 - a^2) W \bar{W} dz = \int_0^1 |DW|^2 dz + a^2 \int_0^1 |W|^2 dz$$

The boundary conditions, $W(0) = 0 = W(1)$, make the integrated terms vanish. Set $\int_0^1 |W|^2 dz = \|W\|^2$

and $\int_0^1 |DW|^2 dz + a^2 \int_0^1 |W|^2 dz = \|\nabla W\|^2$

Both $\|W\|$ and $\|\nabla W\|$ are strictly positive unless $W \equiv 0$ under the boundary conditions. So $I_1 = \|\nabla W\|^2$ is strictly positive unless $W = 0$

The next integral,

$$\begin{aligned} I_2 &= \int_0^1 (D^2 - a^2)^2 W \bar{W} dz = - \int_0^1 D(D^2 - a^2) W D\bar{W} + a^2 (D^2 - a^2) W \bar{W} dz \\ &= \int_0^1 (D^2 - a^2) |W|^2 dz \end{aligned}$$

Again the integrated terms vanish because $W(0) = 0 = W(1)$ and $W''(0) = 0 = W''(1)$. (The same result can be obtained for this integral under rigid boundary conditions.)

Finally

$$\begin{aligned} I_3 &= - \int_0^1 (D^2 - a^2)^3 W \bar{W} dz \\ &= \int_0^1 (D^2 - a^2)^2 DW D\bar{W} + a^2 (D^2 - a^2)^2 W \bar{W} dz \\ &= - \int_0^1 (D^2 - a^2)^2 W (D^2 - a^2) \bar{W} dz \end{aligned}$$

Again the integrated terms vanish under both free and rigid boundary conditions.

$$\begin{aligned} I_3 &= \int_0^1 (D^2 - a^2) DW |W|^2 dz + a^2 \int_0^1 (D^2 - a^2) |W|^2 dz \\ &\quad + (D^2 - a^2) DW (D^2 - a^2) \bar{W} \Big|_0^1 \end{aligned}$$

Under free boundary conditions the integrated terms vanish so

$$I_3 = \int_0^1 (D^2 - a^2) DW |W|^2 dz + a^2 \int_0^1 (D^2 - a^2) |W|^2 dz = \|\nabla(D^2 - a^2)W\|^2$$

When (**) is multiplied by -1, it becomes

$$- \int_0^1 (D^2 - a^2)^3 W \bar{W} dz + (\varphi+1) \sigma \int_0^1 (D^2 - a^2)^2 W \bar{W} dz - \rho \sigma^2 \int_0^1 (D^2 - a^2) W \bar{W} dz = a^2 \int_0^1 R |w|^2 dz$$

Set $J = \int_0^1 R |w|^2 dz$. If I_1, I_2, I_3 and J are substituted in the equation above it becomes

$$\varphi I_1 \sigma^2 + (\varphi+1) I_2 \sigma + I_3 - J = 0.$$

Since ρ , the Prandtl number, is always positive, and I_1 is positive unless $\omega=0$, the quadratic formula can be applied to solve for σ . This gives

$$\sigma = \left[-(\rho+1) I_2 \pm \sqrt{(\rho+1)^2 I_2^2 - 4\rho I_1 (I_3 - J)} \right] / \rho I_1.$$

To show that exchange of stabilities holds, we must show that if $\text{Re } \sigma = 0$, then $\text{Im } \sigma = 0$.

There are two cases where $\text{Re } \sigma = 0$

(i) $4\rho I_1 (I_3 - J) = 0$ and

(ii) $(\rho+1) I_2 = 0$ and $-4\rho I_1 (I_3 - J) < 0$.

If (i) holds $\sigma = 0$, if (ii) holds, $I_2 = 0$ so

$$\int_0^1 |(D^2 - a^2) W|^2 dz = 0.$$

This implies

$$(D^2 - a^2) W \equiv 0$$

The last implies that

$$I_1 = - \int_0^1 (D^2 - a^2) W \bar{W} dz = 0$$

In both cases $\text{Re } \sigma = 0$ implies $\sigma = 0$. This completes the proof of the principal of exchanges of stabilities for operators whose Rayleigh number is variable under free boundary conditions.

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ANALYSIS INTO NORMAL MODES

Since the principle of exchange of stabilities holds for the variable Rayleigh number problem, we attempt to solve the steady state problem for certain functions, R

$$R(z) = R \left(\frac{z}{z_0} \right)^{-\alpha}, \quad z_0 = 10^{-5}, \quad \alpha = 1/3 \text{ or } \alpha = 3/2,$$

The equation is

$$\left[(\nabla^2)^3 - R \left(\frac{z}{z_0} \right)^{-\alpha} \nabla_h^2 \right] u = 0, \quad z_0 \leq z \leq d.$$

with the harmonic boundary conditions

$$\begin{aligned} u(0) &= 0 = u(1) \\ u''(0) &= 0 = u''(1) \\ u^{(iv)}(0) &= 0 = u^{(iv)}(1) \end{aligned}$$

Here $\nabla^2 u = u_{xx} + u_{yy} + u_{zz}$ and $\nabla_h^2 u = u_{xx} + u_{yy}$

If we set $\zeta = z - z_0$, the equation becomes

$$(\nabla^2)^3 u = R \left(\frac{z+z_0}{z_0} \right)^{-\alpha} \nabla_h^2 u, \quad 0 \leq \zeta \leq d - z_0.$$

with harmonic boundary conditions at 0 and $d - z_0$. The equation is normalized by setting

$$\begin{aligned} x &= x / (d - z_0) \\ y &= y / (d - z_0) \\ z &= \zeta / (d - z_0) \end{aligned}$$

and becomes

$$(\nabla^2)^3 u = R \left[z_0^\alpha / (d - z_0)^4 \right] (z + z_0)^{-\alpha} \nabla_h^2 u, \quad 0 \leq z \leq 1.$$

with harmonic boundary conditions at 0 and 1. Let

$$R_0 = R z_0^\alpha / (d - z_0)^4.$$

The equation reduces to

$$(\nabla^2)^3 u = R_0 (z + z_0)^{-\alpha} \nabla_h^2 u$$

If it is assumed that

$$u = \sum_{n=0}^{\infty} e^{i(a_1 x + a_2 y)} A_n \sin \pi n z$$

with $a^2 = a_1^2 + a_2^2$, the equation reduces to the ordinary differential equation:

$$(\partial^2 - a^2)^3 u = -R_0 (z + z_0)^{-\alpha} a^2 u$$

If $A_n = 0$, $n \geq 2$, we can look for a first approximation to the critical (smallest) R_0 for which the equation can be satisfied. This will be approximated by finding the

smallest R_0 such that

$$0 = \int_0^1 [(D^2 - a^2)^3 + R_0 (z + z_0)^{-\alpha} a^2] \sin \pi z \sin \pi z \, dz$$

That is

$$(\pi^2 + a^2)^3 \int_0^1 \sin^2 \pi z \, dz = R_0 a^2 \int_0^1 (z + z_0)^{-\alpha} \sin^2 \pi z \, dz$$

Set
$$I_{1,1}^\alpha = \int_0^1 (z + z_0)^{-\alpha} \sin^2 \pi z \, dz$$

and note that
$$\int_0^1 \sin^2 \pi z = 1/2.$$

This yields

$$R_0 = (\pi^2 + a^2)^3 / (2 a^2 I_{1,1}^\alpha) = (1/2 I_{1,1}^\alpha) (\pi^2 + a^2)^3 / a^2$$

It is easy to see that R_0 has a minimum when $R_0' = 0$, that is, when $a^2 = \pi^2/2$. Then

$$R_0 = 27 \pi^4 / (8 I_{1,1}^{4/3})$$

Since numerical integration yields the values

$$I_{1,1}^{4/3} = 1.760401$$

and

$$I_{1,1}^{3/2} = 2.157553.$$

The first estimate, R_0 , of R_0 is

$$R_0 = 186.75045,$$

if

$$\alpha = 4/3,$$

and

$$R_0 = 152.3743,$$

if

$$\alpha = 3/2.$$

The first estimate of R , then is

$$8.667841 \times 10^8,$$

if

$$\alpha = 4/3,$$

and

$$4.8183065 \times 10^9,$$

if

$$\alpha = 3/2.$$

The second approximation is made by assuming $A_n = 0, n \geq 3$.

The problem is to find the smallest number, 2R_0 , and a pair of numbers A_1 and A_2 , with

$$0 = \int_0^1 [(0^2 - a^2)^3 + {}^2R_0 (z+z_0)^{-4} a^2] (A_1 \sin \pi z + A_2 \sin 2\pi z) \sin \pi z dz$$

and

$$0 = \int_0^1 [(0^2 - a^2)^3 + {}^2R_0 (z+z_0)^{-4} a^2] (A_1 \sin \pi z + A_2 \sin 2\pi z) \sin 2\pi z dz.$$

That is

$$A_1 (\pi^2 + a^2)^3 \left[\int_0^1 \sin^2 \pi z dz \right] = a^2 {}^2R_0 \left[A_1 \int_0^1 (z+z_0)^{-4} \sin^2 \pi z dz + A_2 \int_0^1 (z+z_0)^{-4} \sin 2\pi z \sin \pi z dz \right]$$

and

$$A_2 (4\pi^2 + a^2)^3 \left[\int_0^1 \sin^2 2\pi z dz \right] = a^2 {}^2R_0 \left[A_1 \int_0^1 (z+z_0)^{-4} \sin \pi z \sin 2\pi z dz + A_2 \int_0^1 (z+z_0)^{-4} \sin^2 2\pi z dz \right].$$

Set
$$I_{j,k}^{\alpha} = \int_0^1 (z+z_0)^{-\alpha} \sin j\pi z \sin k\pi z dz.$$

Note that $I_{j,k}^{\alpha} = I_{k,j}^{\alpha}$ and that $\int_0^1 \sin^2 j\pi z dz = 1/2$.

This means that

$$A_1 (\pi^2 + a^2)^3 / 2 = a^2 {}^2R_0 [A_1 I_{1,1}^{\alpha} + A_2 I_{1,2}^{\alpha}]$$

and

$$A_2 (4\pi^2 + a^2)^3 / 2 = a^2 {}^2R_0 [A_1 I_{1,2}^{\alpha} + A_2 I_{2,2}^{\alpha}].$$

The numbers $I_{j,k}^{\alpha}$ are found to any desired degree of accuracy by Simpson's rule.

The problem can be written now as an eigenvalue problem in $V^2 R_0$ with eigenvector (A_1, A_2) .

$$(1/{}^2R_0) A_1 = 2a^2 / (\pi^2 + a^2)^3 [I_{1,1}^{\alpha} A_1 + I_{1,2}^{\alpha} A_2]$$

$$(1/{}^2R_0) A_2 = 2a^2 / (4\pi^2 + a^2)^3 [I_{1,2}^{\alpha} A_1 + I_{2,2}^{\alpha} A_2]$$

At the value of a^2 corresponding to the smallest value of R_0 , the function $1/2 R_0$ takes on its largest value. This will be the maximum of the largest eigenvalue of all the matrices,

$$M = \begin{pmatrix} m_{11} & m_{12} \\ m_{21} & m_{22} \end{pmatrix}$$

where $m_{j,k} = (2a^2 / (j^2\pi^2 + a^2)^2) I_{j,k}^\alpha$.

M is not symmetric.

It is convenient to set $b = a^2/\pi^2$. Then

$$m_{j,k} = (2b / (j^2 + b)^2) I_{j,k}^\alpha / \pi^4.$$

Let $r = b/(1+b)^2$ and $s = b/(4+b)^2$. r reaches its maximum when $b = \sqrt{2}$; then $r = 4/27$ and $s = 4/(27)^2$. s reaches its maximum when $b = 2$; then $s = 1/108$ and $r = 2/27$.

It is clear that the entries of M are positive since $I_{j,k}^\alpha$ is the integral of a positive function and the entries of the second row are much smaller than those of the first row.

$$I_{1,1}^{4/3} = 1.760401$$

$$I_{1,2}^{4/3} = 1.177827$$

$$I_{2,2}^{4/3} = 2.593355$$

$$I_{1,1}^{3/2} = 2.157553$$

$$I_{1,2}^{3/2} = 1.667493$$

$$I_{2,2}^{3/2} = 3.446855$$

These numbers are accurate to the last place, though some of them are extrapolated to determine the last place.

The problem then is to find the largest eigenvalue, λ , of

$$2 \begin{pmatrix} r I_{1,1}^\alpha & r I_{1,2}^\alpha \\ s I_{1,2}^\alpha & s I_{2,2}^\alpha \end{pmatrix}$$

Then λ/π^4 will be the largest eigenvalue of M and ${}^2R_0 = \pi^4 \lambda$

This yields

$$0 = \lambda^2 - 2(rs I_{1,1}^\alpha + s I_{2,2}^\alpha) \lambda + 4rs (I_{1,1}^\alpha I_{2,2}^\alpha - (I_{1,2}^\alpha)^2)$$

It is difficult to maximize λ as a function of a^2 analytically. Instead, using the values of $I_{j,k}^\alpha$ obtained by numerical integration, shown in the tables, values of λ were calculated in a range between $b = .5$ and $b = 2$.

When $a^2 = \pi^2/2$, $b = 1/2$. This is the value of both the critical Rayleigh number in the constant coefficient case ($\alpha = 0$) and the value of b corresponding to the first estimate 1R_0 of R_0 we did earlier.

The calculation was done as follows. For $\alpha = 4/3$, λ and 2R_0 were evaluated at numbers

$$b = 0.00001, 0.10001, 0.20001, \dots, 1.90001$$

The largest value of λ occurred when 0.50001 . λ and 2R_0 were then evaluated at

$$b = 0.50001, 0.5100, 0.5200, \dots, 0.59001$$

The maximum value of λ this time occurred when $b = 0.51001$. So λ and 2R_0 were evaluated at

$$b = 0.50101, 0.50201, \dots, 0.51801, 0.51901$$

This process was continued until the machine produced no differences in the sixth decimal place for 2R_0 . This represents eight significant digits. The numbers $I_{j,k}^\alpha$ are known only to seven significant digits. The calculation is not very sensitive to changes in b (and therefore b^2) when b is near its critical value, but it should be quite sensitive to changes in $I_{j,k}^\alpha$.

When $\alpha = 4/3$, the critical value was established at

$$b = 0.50904, \quad a^2 = 5.024023$$

Then ${}^2R_0 = 183.5693$

and ${}^2R = 8.520189 \times 10^8$

The eigenvector is

$$\begin{pmatrix} A_1 \\ A_2 \end{pmatrix} = \begin{pmatrix} 0.9996605 \\ 0.0260555 \end{pmatrix}$$

At $b = 0.50904$, $\lambda = 0.5306395$,

$r = 0.1481322$ and $s = 0.0055526$

The corresponding critical values for $\alpha = 3/2$ were

$b = 0.51208$ $a^2 = 5.054027$

${}^2R_0 = 148.9276$

${}^2R = 4.709315 \times 10^7$

The eigenvector is

$$\begin{pmatrix} A_1 \\ A_2 \end{pmatrix} = \begin{pmatrix} 0.9995444 \\ 0.0301839 \end{pmatrix}$$

At $b = 0.51208$, $\lambda = 0.6540702$

$r = 0.1481199$ and $s = 0.0055745$

If we compare 1R_0 , the first estimate of R_0 with 2R_0 , the second estimate of R_0 , we have

	$\alpha = 4/3$	$\alpha = 3/2$
1R_0	186.7505	152.3743
2R_0	183.5693	148.9276

The numbers jR_0 converge quickly to R_0 . If a third estimate is made the entries $I_{3,k}^{\alpha}$ are smaller than 5.0.

	$\alpha = 4/3$	$\alpha = 3/2$
$I_{1,3}^{\alpha}$	0.832954	1.289302
$I_{2,3}^{\alpha}$	1.871715	2.791557
$I_{3,3}^{\alpha}$	3.182768	4.442455

This problem is analogous to the last. We are looking for a function

$$W(z) = A_1 \sin \pi z + A_2 \sin 2\pi z + A_3 \sin 3\pi z$$

with

$$\int_0^1 [(D^2 - a^2)^3 W + a^2 {}^3R_0 (z+z_0)^{-\alpha} W] \sin j\pi z dz = 0 \quad j=1,2,3$$

Let $t = b/(9+b)^3$. Then we must find the largest eigenvalue of the matrix, M^3

$$M^3 = \begin{pmatrix} r I_{1,1}^\alpha & r I_{1,2}^\alpha & r I_{1,3}^\alpha \\ s I_{1,2}^\alpha & s I_{2,2}^\alpha & s I_{2,3}^\alpha \\ t I_{1,3}^\alpha & t I_{2,3}^\alpha & t I_{3,3}^\alpha \end{pmatrix}$$

The factor t is much smaller than s , just as s is much smaller than r . The determinate of M^3 is then $O(rst)$. The sum of the determinates of the principal submatrices is $O(rs)$. The trace is $O(r)$.

Since the determinate is the product of the eigenvalues, the trace is the sum of the eigenmodes, and the sum of the partial products of the eigenvalues is the sum of the determinates of the principal submatrices, it is easy to see that M^3 has one relatively large eigenvalue, λ , and that its other eigenvalues are near 0. The large eigenvalue should be $O(r)$.

Because the large eigenvector is so isolated and because we have a good guess $(2rI_{1,1})$ for it, the power method was used to calculate λ and the eigenvector. This converged rapidly, giving a value of λ that depended on b .

λ was maximized exactly as in estimating 2R_0 .

For $\alpha = 4/3$, the maximum occurred when $b = 0.50970$ and $a^2 = 5.030537$. Then, the third estimate, ${}^3R_0 = 183.3936$ and ${}^3R = 8.512375 \times 10^8$. The eigenvector is

$$\begin{pmatrix} A_1 \\ A_2 \\ A_3 \end{pmatrix} = \begin{pmatrix} 0.999656 \\ 0.026135 \\ 0.001981 \end{pmatrix}$$

For $\alpha = 3/2$, the maximum occurred when $b = 0.5132$, and $a^2 = 5.06508$.

Then ${}^3R_0 = 148.7001$

and ${}^3R = 4.702310 \times 10^9$

The eigenvector is

$$\begin{pmatrix} A_1 \\ A_2 \\ A_3 \end{pmatrix} = \begin{pmatrix} 0.999537 \\ 0.030308 \\ 0.002520 \end{pmatrix}$$

The three estimates of the eigenvalue converged quickly.

	$\alpha = 4/3$	$\alpha = 5/2$
1R_0	186.75	152.37
2R_0	183.57	148.93
3R_0	183.39	148.70

It should be noted that none of these values, λ , R_0 , or the eigenvector are very sensitive to changes of b or a^2 near the critical value.

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Gantmacher, F. R. "The Theory of Matrices", Vol. II, Chelsea, New York, 1959.

Velarde, M. G. and Christian Normond "Convection", Vol. 243 #1, Scientific American, July 1980.

Conclusions and Recommendations

Work like this in the case of variable viscosity, thermal conductivity and gravity has atmospheric applications as well as applications to the space program. This would be applicable to the GFFC experiment to be flown in Spacelab 3, especially if rigid boundary conditions were handled. At least some of this is presently feasible.

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MARSHALL SPACE FLIGHT CENTER
THE UNIVERSITY OF ALABAMA

IMPROVEMENTS TO THE NASAP CODE

Prepared By:	David Perel, Ph.D.
Academic Rank:	Assistant Professor
University and Department	University of South Carolina College of Engineering
NASA/MSFC: Division:	Program Development Office
MSFC Counterpart:	Carl E. Colley
Date:	August 5, 1980
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IMPROVEMENTS TO THE NASAP CODE

BY

David Perel
Assistant Professor of Engineering
University of South Carolina
Columbia, South Carolina

ABSTRACT

The object of this work is the modification and improvement of the FORTRAN code, NASAP, written by the author. The purpose of NASAP is the transformation of CAD-generated NASTRAN input data to input data for DESAP II and/or DESAP I. The latter programs are used for structural optimization, and in the initial design phase are often more useful than NASTRAN, which is used for structural analysis.

NASAP is designed to operate interactively, and to this end, modifications and refinements have been made which simplify the interactive participation of the engineer, thus reducing chances for error as well as time spent at the computer terminal. Redundancies have been eliminated, and a tabular form of input has been incorporated into NASAP.

Currently, NASAP is being expanded to include iterative design cycles involving DESAP II and/or DESAP I.

ACKNOWLEDGMENT

The author thanks Carl Colley for his help in this project.

INTRODUCTION

The Preliminary Design Office currently has the capability of creating NASTRAN bulk data by use of an interactive graphics system implemented on a Computervision system. This method of creating NASTRAN input data is far more efficient in terms of time and accuracy than manual preparation of the data. The CADDs software associated with the Computervision creates NASTRAN input data. This data is then transmitted to a main computer for NASTRAN execution in the batch mode. The output of the program is essentially the stress and displacement fields of the structure being analyzed.

DESAP I and DESAP II are also structural programs, but differ from NASTRAN in that they optimize the structure with respect to some variable, usually weight (i.e., minimizes the weight) subject to user-supplied limits on stress and displacement (DESAP I), and stress and buckling (DESAP II).

The input required for DESAP I and II is rather lengthy, but much of it, including the geometric description of the structural model, may be derived from the CADDs-generated NASTRAN input data. This is the purpose of NASAP.

The current version of DESAP I allows constraints on stress and displacement, while DESAP II allows constraints on stress and buckling. In order to generate a design satisfying constraints on stress, displacement, and buckling, it is necessary to use both DESAP I and II in an iterative cycle. Development of this capability is currently underway.

OBJECTIVE

The objective of this project is the improvement and expansion of NASAP in the following ways: (1) elimination of redundancies; (2) simplification of user-supplied input; and, (3) incorporation of the capability of iterative design cycles involving DESAP I and DESAP II.

REPORT

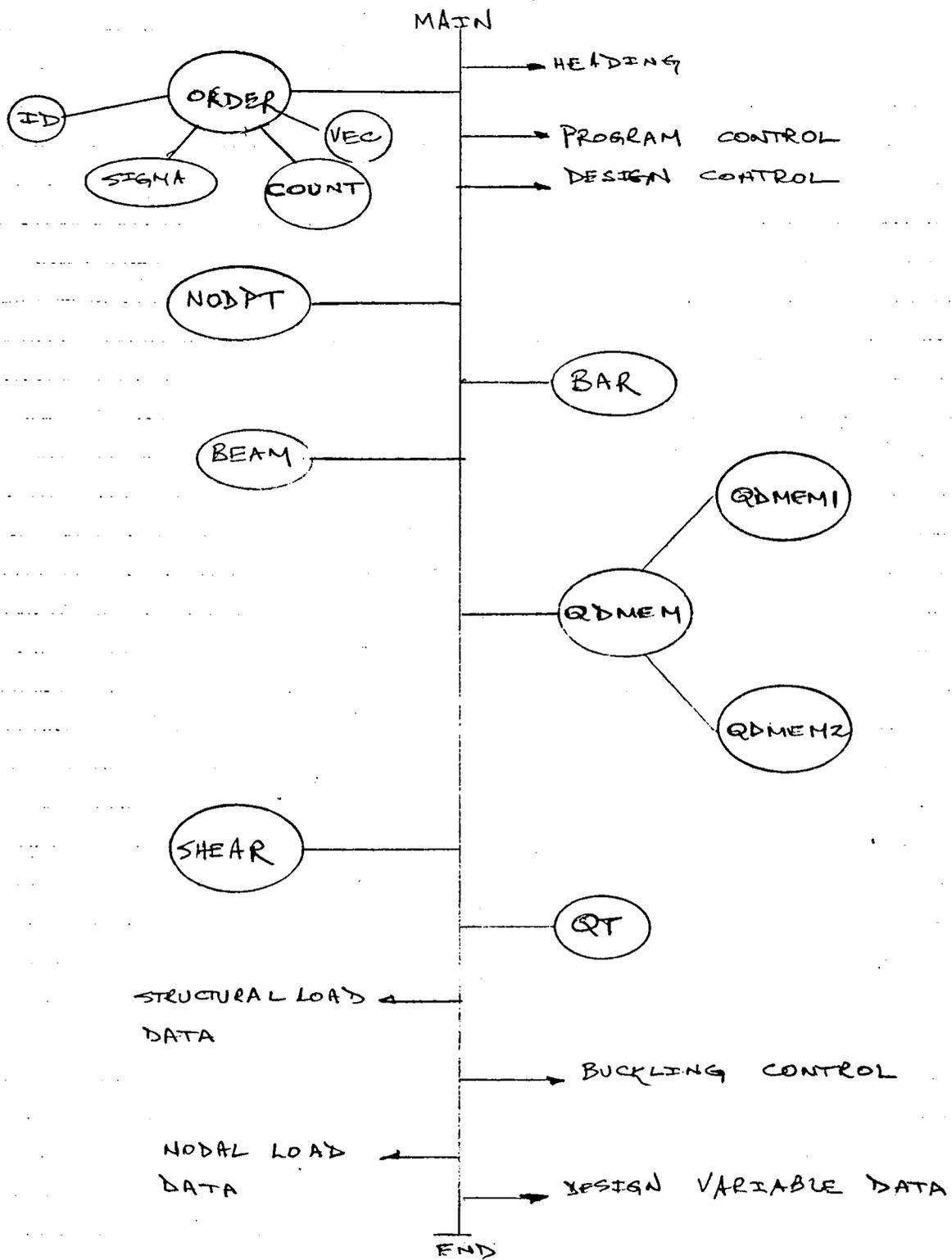
The body of this report consists mainly of the NASAP program. Its major subroutines are self-explanatory. An overall flow chart, Figures 1a-1d, outlines the main features of the program. The program is based on the following structural equivalences between NASTRAN and DESAP I and II:

<u>NASTRAN</u>	<u>DESAP I and II</u>
ROD	BAR
BAR	BEAM
QDMEM	Plane Stress Quadrilateral
QUAD2	Plate Quadrilateral
TRIA2	or Triangle
SHEAR	Quadrilateral Shear Panel

This is reflected in the five subroutines in NASAP called BAR, BEAM, QDMEM, QT, and SHEAR. These subroutines deal only with the structural elements their names imply, and perform ordering and manipulation of NASTRAN data via the subroutines CANDP, STOR, PAIR, VEC, and ID.

Figure 2 illustrates the design cycles that are possible using NASAP with DESAP I and DESAP II. The main feature of these cycles is the modification of file 13 (DESAP II) via calculated design variables and/or tolerances from DESAP I, and the corresponding route for file 14 (DESAP I).

When an acceptable design has been achieved using both DESAP I and DESAP II, there is no guarantee that this design is an optimum, i.e., that for the given structure it minimizes weight subject to the given constraints on stress, displacement, and buckling. There may exist other designs of lower weight also satisfying the constraints. This situation cannot be remedied without incorporating the three constraints in a single program.



LEGEND:
 → TO FILE 13 (DESAP II)

○ SUBROUTINE

NASAP

FIG. 1a

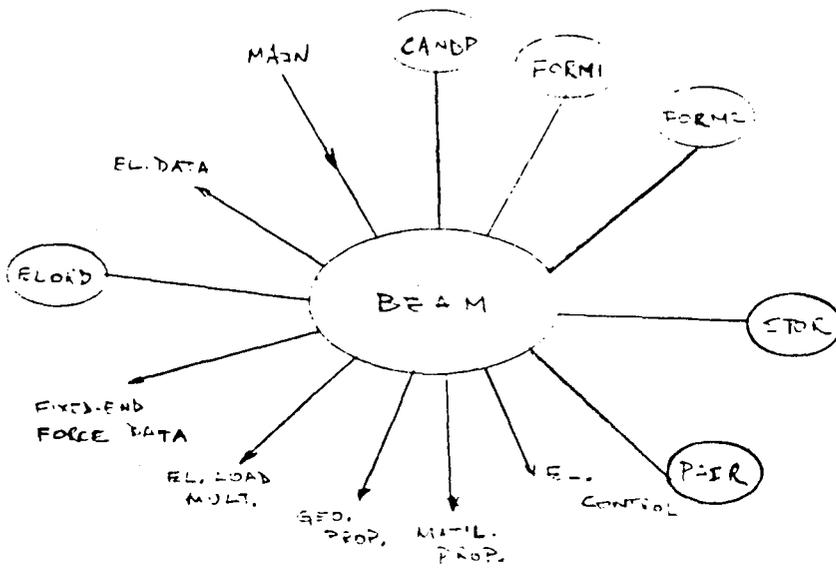
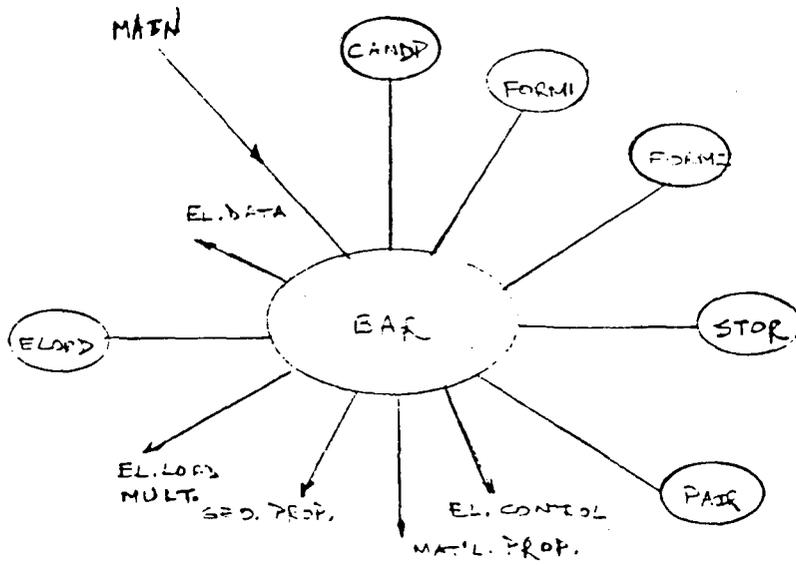


FIG. 16

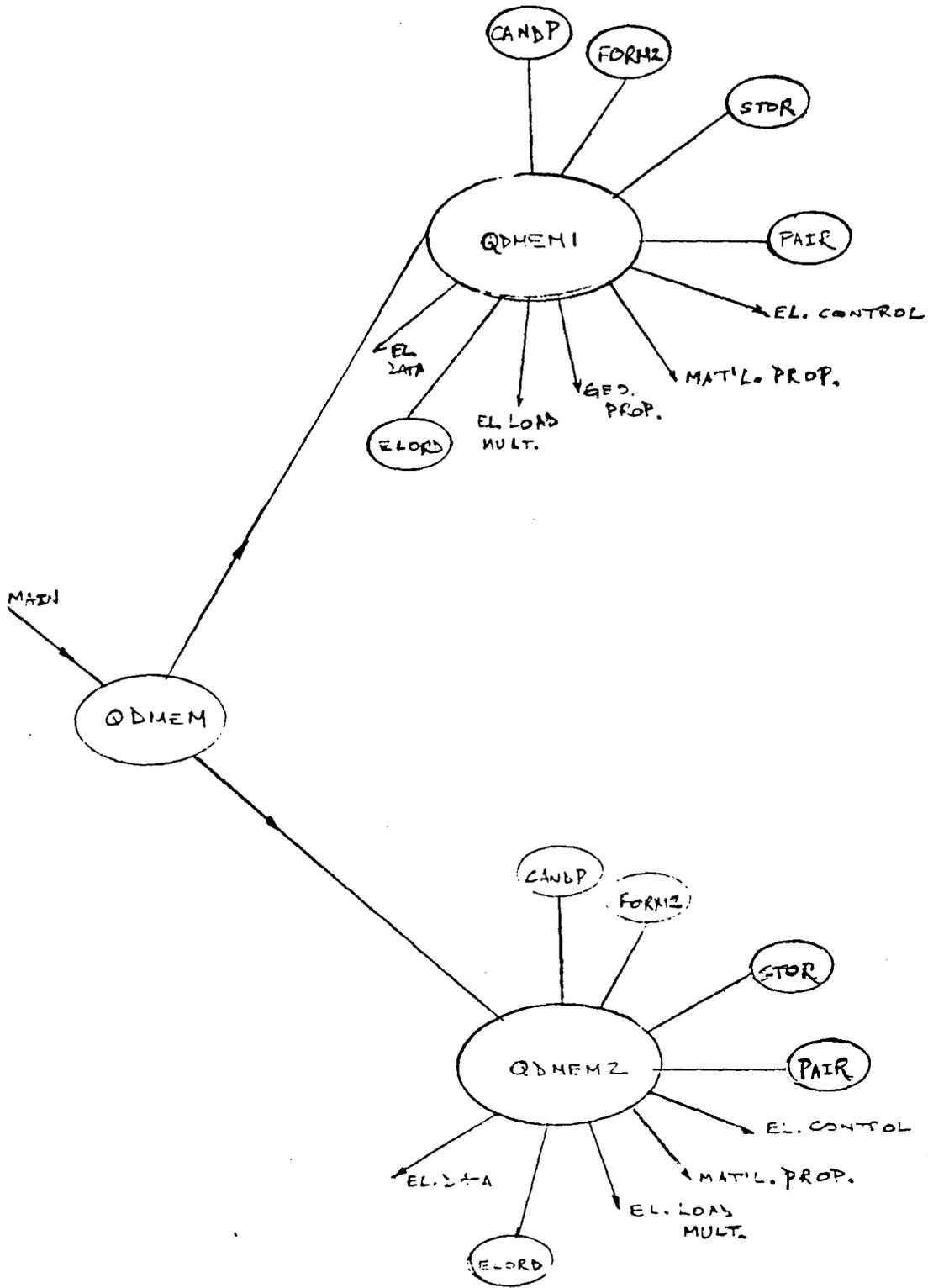


FIG. 1C

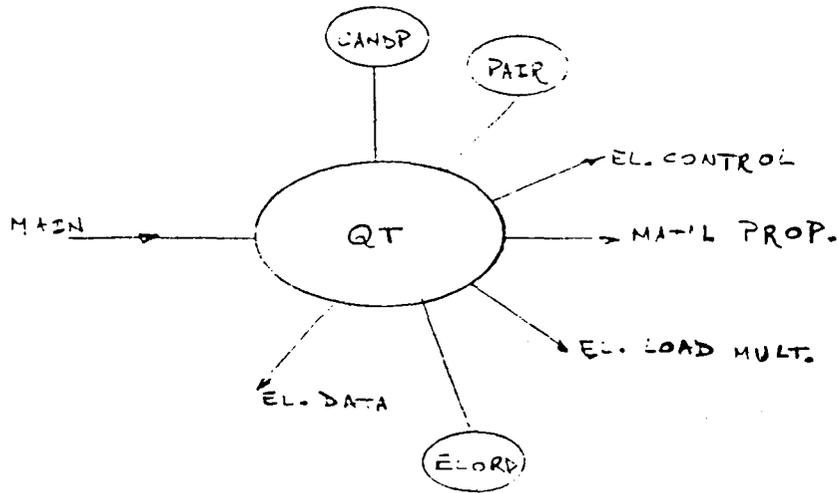
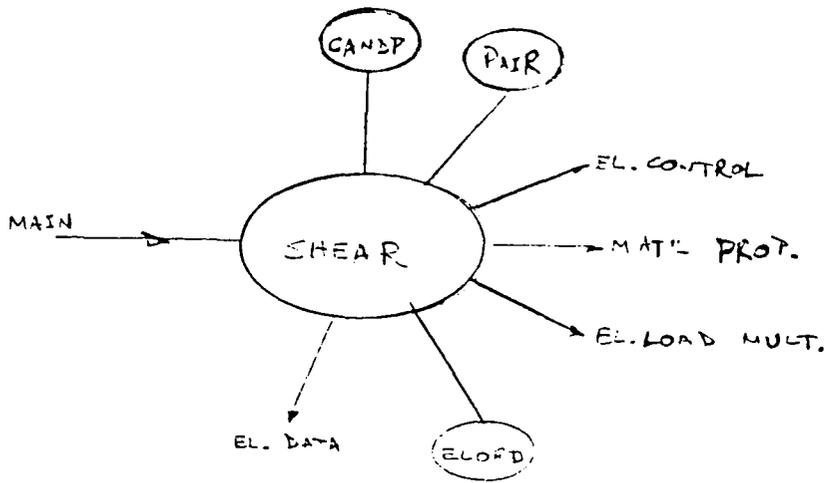
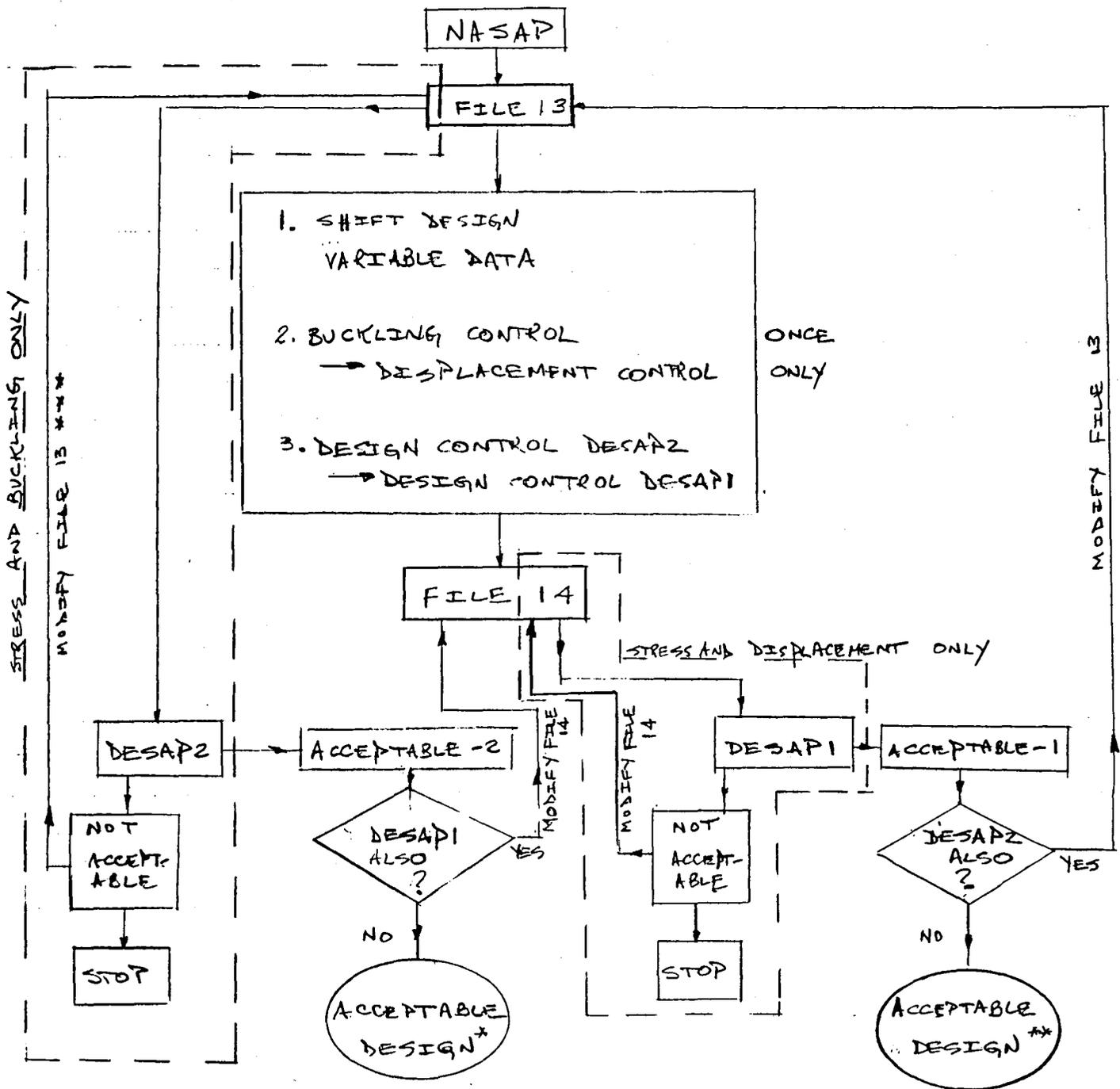


FIG. 1d



* SATISFIES DESAP2 OR
DESAP2 AND DESAPI

** SATISFIES DESAPI OR
DESAPZ AND DESAPI

DESIGN CYCLES

FIG. 2

*** FILE MODIFICATIONS INVOLVE
CHANGES IN DESIGN VARIABLES
AND VARIOUS TOLERANCES

0000 I 007430 K1	0000 I 007444 K10	0000 I 007450 K12	0000 I 007451 K13	0000 I 007453 K15
0000 I 007455 K16	0000 I 007456 K17	0000 I 007457 K18	0000 I 007435 K2	0000 I 007436 K3
0000 I 007431 K4	0000 I 007437 K5	0000 I 007440 K6	0000 I 007441 K7	0000 I 007442 K8
0000 I 007443 K9	0000 I 002114 MATCH	0004 000000 MAT11	0004 R 000024 MAT12	0000 I 006736 MTCOM1
0000 R 007152 MTCOM2	0000 R 007006 MTPROP	0000 I 007412 VE10	0000 I 007344 NPAR	0000 I 007413 NP10
0000 I 007423 NROW	0003 000022 NSTART	0003 I 000000 NUM	0000 I 007424 NUMGE0	0000 I 007420 NUMMAT
0000 I 007401 NUM12	0000 I 007416 NUM18	0000 I 007400 NUM5	0000 I 006344 PROD1	0000 R 006426 PROD2
0000 I 002734 TOCROD	0000 I 007356 TYPE	0000 I 007352 VAR	0000 I 002424 VE10	0000 I 005214 VE102

00101	1*	SUBROUTINE BAR (CONFAC,NKODE,IOV1,NTDV1)	000002
00103	2*	COMMON/BACH/NUM(18),NSTART(18)/MOZART/MAT11(20),MAT12(20,10)	000002
00103	3*	C ** CHANGE GROUP. NO. OF DESAP BARINASTRAN ROD) ELEMENTS	000002
00104	4*	INTEGER CROD(100,6),CRODID(100,5),MATCH(100,2),VE10(100),INDEX(100	000002
00104	5*	1),TOCROD(100,6),LDAT1(100,6),VE102(100)	000002
00105	6*	DIMENSION ELDAT2(100,4),DUMMY(100,1)	000002
00105	7*	C ** CHANGE GROUP. NO. OF PID'S	000002
00106	8*	INTEGER PROD1(25,2)	000002
00107	9*	DIMENSION PROD2(25,5),GEOPR2(25,3)	000002
00107	10*	C ** CHANGE GROUP. NO. OF MID'S	000002
00110	11*	INTEGER MTCOM1(20,2)	000002
00111	12*	REAL MTPROP(20,5),MTCOM2(20)	000002
00111	13*	C ** CHANGE GROUP. NO. OF DESIGN VARIABLES	000002
00112	14*	DIMENSION IOV1(100),IOV11(50),IOV12(50)	000002
00112	15*	C ** *****	000002
00113	16*	INTEGER KOAD(2),NPAR(6),VAR(4),TYPE(2)	000002
00114	17*	REAL MAT12,EMUL(4,4)	000002
00115	18*	DATA VAR/'A','B','C','D'/TYPE/'BAR',' '/	000002
00120	19*	NUM5=NUM(5)	000002
00121	20*	NUM12=NUM(12)	000004
00122	21*	CALL CANOP(5,NUM5,NUM12,CROD,DUMMY,PROD1, PROD2)	000006
00122	22*	C PRINT 14	000006
00122	23*	C14 FORMAT(' CROD IN BAR FROM CANOP')	000006
00122	24*	C DO 9 L=1,NUM5	000006
00122	25*	C9 PRINT 6, (CROD(L,J),J=1,4)	000006
00122	26*	C8 FORMAT(8X,4I8)	000006
00122	27*	C PRINT 13	000006
00122	28*	C13 FORMAT (' PROD1 AND PROD2 IN BAR FROM CANOP')	000006
00122	29*	C DO 16 L=1,NUM12	000006
00122	30*	C16 PRINT 12, (PROD1(L,J),J=1,2), (PROD2(L,J),J=1,4)	000006
00122	31*	C12 FORMAT(8X,2I8,4F8,4)	000006
00123	32*	7 FORMAT()	000017
00124	33*	CALL FORM1(NKODE,KOAD,TYPE)	000017
00125	34*	NPAR(1)=1	000024
00126	35*	NPAR(2)=NUM5	000026
00127	36*	NPAR(6)=1	000030
00130	37*	DO 999 I=1,NUM5	000051
00133	38*	DO 999 J=1,6	000051
00136	39*	999 TOCROD(I,J)=CROD(I,J)	000051
00141	40*	DO 1000 J1=1,NKODE	000062
00144	41*	NPAR(5)=KOAD(J1)	000062
00144	42*	C ** IF ONLY ONE CONSTRUCTION CODE, SKIP TO 25	000062
00145	43*	IF(NKODE.EQ.1) GO TO 25	000064
00147	44*	IF(J1.EQ.2) GO TO 4000	000066
00151	45*	CALL FORM2(J1,KOAD,TYPE,NPAR(2),VE10,NUM5)	000071

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00152	46*	ISTOP=NPARI2)	000101
00153	47*	KOWNT=0	000103
00154	48*	DO 3100 IJK=1,NUM5	000114
00157	49*	IFLAG=0	000114
00160	50*	DO 3150 IJKL=1,ISTOP	000117
00163	51*	3050 IF(CROD(IJK,1).EQ.VEID(IJKL))IFLAG=1	000117
00166	52*	IF(IFLAG.EQ.1) GO TO 3100	000125
00170	53*	KOWNT=KOWNT+1	000130
00171	54*	VEID2(KOWNT)=CROD(IJK,1)	000133
00172	55*	3100 CONTINUE	000142
00174	56*	4000 CONTINUE	000142
00175	57*	IF(J1.EQ.1) GO TO 4100	000142
00177	58*	ISTOP=NUM5=ISTOP	000144
00200	59*	NPARI2)=ISTOP	000147
00201	60*	DO 4050 IJK=1,ISTOP	000154
00204	61*	4050 VEID(IJK)=VEID2(IJK)	000154
00204	62*	C PRINT 4051,ISTOP	000154
00204	63*	C DO 4052 IJK=1,ISTOP	000154
00204	64*	C4052 PRINT 4053,VEID(IJK)	000154
00204	65*	C4053 FORMAT(I6)	000154
00204	66*	C4051 FORMAT(/' VEID IN HAR FOR J1=2'/' ISTOP=',I6)	000154
00206	67*	4100 CONTINUE	000157
00207	68*	DO 888 I=1,NUM5	000157
00212	69*	DO 888 J=1,6	000164
00215	70*	888 CROD(I,J)=CROD(I,J)	000164
00215	71*	C PRINT 333	000164
00215	72*	333 FORMAT(' CROD IN HAR JUST BEFORE STOR')	000164
00215	73*	C DO 777 I=1,NUM5	000164
00215	74*	C777 PRINT 222, (CROD(I,J),J=1,ISTOP)	000164
00215	75*	C222 FORMAT(6I5)	000164
00220	76*	CALL STOR(ISTOP,NUM5,CROD,VEID,4)	000172
00220	77*	C PRINT 67	000172
00220	78*	C67 FORMAT(' CROD IN HAR JUST BACK FROM STOR')	000172
00220	79*	C DO 77 J=1,ISTOP	000172
00220	80*	C77 PRINT 76, (CROD(J,K),K=1,6)	000172
00220	81*	C PRINT 68	000172
00220	82*	C68 FORMAT(' VEID IN HAR JUST BACK FROM STOR')	000172
00220	83*	C PRINT 87, (VEID(I),I=1,ISTOP)	000172
00220	84*	C87 FORMAT(11I4)	000172
00220	85*	C76 FORMAT(6I5)	000172
00221	86*	GO TO 26	000201
00222	87*	25 ISTOP=NUM5	000203
00223	88*	26 DO 77 J=1,ISTOP	000205
00226	89*	NEID=CROD(J,1)	000211
00227	90*	NPID=CROD(J,2)	000213
00227	91*	C ** FIND PROD1 CARD WITH SAME PID	000213
00230	92*	DO 65 K=1,NUM12	000220
00233	93*	IF(NPID.EQ.PROD1(K,1))GO TO 61	000220
00235	94*	GO TO 65	000223
00236	95*	61 KK=K	000225
00237	96*	GO TO 62	000226
00240	97*	65 CONTINUE	000232
00242	98*	62 CRODID(J,1)=NEID	000232
00243	99*	CRODID(J,2)=CROD(J,3)	000234
00244	100*	CRODID(J,3)=CROD(J,4)	000236
00245	101*	CRODID(J,4)=PROD1(KK,2)	000240
00246	102*	CRODID(J,5)=NPID	000242

00247	103*	70	CONTINUE	000246
00247	104*	C	PRINT 73	000246
00247	105*	C73	FORMAT(' CRODID FROM BAR')	000246
00247	106*	C	DO 74 J=1,ISTOP	000246
00247	107*	C74	PRINT 75, (CRODID(J,K),K=1,5)	000246
00247	108*	C75	FORMAT(5I5)	000246
00247	109*	C **	CRODID ARRAY IS DONE, IT CONTAINS, FOR THIS CONST. CODE,	000246
00247	110*	C **	EID,G1,G2,MID,PIU,IN ISTOP,=NPAR(2)ROWS.	000246
00247	111*	C **	FIND NUMMAT AND MATCH.	000246
00247	112*	C **	IN CALL PAIR,PRODI IS NOT USED	000246
00251	113*		NUM18=NUM(18)	000246
00252	114*		KKODE=U	000250
00253	115*		CALL PAIR(KKODE,CRODID,PRODI,ISTOP,NUM18,4,NUMMAT,MATCH,INDEX)	000251
00254	116*		NPAR(3)=NUMMAT	000264
00254	117*	C **	SET UP MATERIAL CONTROL CARD FOR EACH OF THE NUMMAT	000264
00254	118*	C **	MAT1 CARDS ASSOCIATED WITH CRODID,FOLLOWED BY THE	000264
00254	119*	C **	MATERIAL PROPERTY CARD FOR THAT MATERIAL	000264
00255	120*		DO 83 J10 =1,NUMMAT	000277
00260	121*		MTCON1(J10,1)=INDEX(J10)	000277
00261	122*		MTCON1(J10,2)=1	000301
00261	123*	C **	FOR MID NO, INDEX(J10),FIND CORRESPONDING ROW NO. IN MAT12	000301
00262	124*		DO 84 J11=1,NUMMAT	000306
00265	125*		IF (MATCH(J11,1).EQ.INDEX(J10))GO TO 85	000306
00267	126*		GO TO 84	000311
00270	127*	85	NROW=MATCH(J11,2)	000313
00271	128*		GO TO 86	000314
00272	129*	84	CONTINUE	000320
00274	130*	86	MTCON2(J10) = MAT12(NROW,4)*CONFAC	000320
00274	131*	C	PRINT 2222, MAT12(NROW,4)	000320
00274	132*	C2222	FORMAT(' MAT12 =',F10.0)	000320
00274	133*	C	PRINT 1111, MTCON2(J10)	000320
00274	134*	C1111	FORMAT(2X,'MTCON2 ='F20.8)	000320
00274	135*	C	PRINT 95, NROW,CONFAC	000320
00274	136*	C95	FORMAT(' NROW,CONFAC',I3,2X,F8.2)	000320
00274	137*	C **	SET UP MATERIAL PROPERTIES CARD FOR THIS MATERIAL	000320
00275	138*		MTPROP(J10,1)=0.	000323
00276	139*		MTPROP(J10,2)=MAT12(NROW,1)	000324
00277	140*		MTPROP(J10,3)=MAT12(NROW,5)	000326
00277	141*	C	PRINT 111, MTPROP(J10,2),MTPROP(J10,3)	000326
00277	142*	C111	FORMAT(' MTPROP-2 AND MTPROP-3 ='2F20.8)	000326
00300	143*		MTPROP(J10,4)=MAT12(NROW,8)	000330
00301	144*		MTPROP(J10,5)=MAT12(NROW,9)	000332
00302	145*	83	CONTINUE	000336
00302	146*	C **	MATERIAL PROPERTIES DATA IS DONE	000336
00302	147*	C **	IT IS NOT SENT TO FILE 12 UNTIL AFTER NPAR IS.	000336
00302	148*	C **	SINCE NONE OF THE ELS OF PRODU2 ARE USED BY DESAP	000336
00302	149*	C **	IT IS NOT NECESSARY TO MATCH THE PID'S OF THE CROD'S	000336
00302	150*	C **	WITH THE ROWS OF PRODU2	000336
00302	151*	C **	SET UP GEOMETRIC PROPERTIES DATA, DEFAULT TO AREA =IY=IZ=1.	000336
00304	152*		WRITE (6,88) KOD(J1),ISTOP	000336
00310	153*	88	FORMAT (5X,'ENTER NUMBER OF DIFFERENT GEOMETRIC PROPERTIES FOR CON	000345
00310	154*		STRUCTION CODE',I4,'. MUST BE LESS THAN OR EQUAL TO',I4)	000345
00311	155*		READ (5,7)NUMGEO	000345
00314	156*		NPAR(4)=NUMGEO	000353
00315	157*		WRITE(12,795)	000355
00317	158*	795	FORMAT(' COMM ELEMENT CONTROL FROM BAR')	000362
00320	159*		WRITE(12,79) (NPAR(IP),IP=1,10)	000362

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00320	160*	C **	MPAR TO FILE 12	000362
00320	161*	C	PRINT 799	000362
00320	162*	C799	FORMAT(' NPAR TO FILE 12')	000362
00320	163*	C	PRINT 79, (NPAR(IP),IP=1,6)	000362
00323	164*	79	FORMAT(6,5)	000372
00324	165*		WRITE(12,349)	000372
00326	166*	349	FORMAT(' COMM MATERIAL CONTROL AND PROPERTIES FROM BAR')	000404
00327	167*		DO 350 J12=1,NUMHAT	000404
00332	168*		WRITE(12,300)(MTCON1(J12,J13),J13=1,2),MTCON2(J12)	000404
00332	169*	C	PRINT 8999	000404
00332	170*	C8999	FORMAT(' MTCGN1 AND MTCGN2 TO FILE 12')	000404
00332	171*	C	WRITE(6,300)(MTCON1(J12,J13),J13=1,2),MTCON2(J12)	000404
00341	172*	300	FORMAT (215,F10.3)	000416
00342	173*		WRITE(12,301) (MTPROP(J12,J13),J13=1,5)	000416
00342	174*	C	PRINT 9999	000416
00342	175*	C9999	FORMAT(' MTPROP TO FILE 12')	000416
00342	176*	C	WRITE(6,301) (MTPROP(J12,J13),J13=1,5)	000416
00350	177*	301	FORMAT (5F10.0)	000437
00351	178*	350	CONTINUE	000437
00351	179*	C **	MATERIAL PROPERTIES DATA HAVE BEEN SENT TO FILE 12	000437
00353	180*		IF (NUMGEO.GT.1)GO TO 100	000437
00355	181*		DO 101 K1=1,3	000446
00360	182*	101	GEOPR2(1,K1)=1.	000446
00362	183*		DO 108 K4=1,1STOP	000453
00365	184*	108	GRODID(K4,5)=1	000453
00367	185*		WRITE (6,89)	000455
00371	186*	89	FORMAT (2X,'FOR THIS CONST CODE THERE IS ONLY ONE GEOMETRIC PROPER	000462
00371	187*		ITY. IT HAS BEEN ASSIGNED NO.'2X' AND THE FOLLOWING PROPERTIES:	000462
00371	188*		1AREA=1,IY=1,IZ=1. IF YOU WANT A DIFFERENT PROPERTY SET,/'	000462
00371	189*		12X'ENTER IT IN THE SAME ORDER, OTHERWISE DEFAULT')	000462
00372	190*		READ (5,7) DUM1	000462
00372	191*	C	PRINT 666,DUM1	000462
00372	192*	C666	FORMAT(' DUM1 LINE 173 MAIN',F10.5/)	000462
00375	193*		IF (DUM1.EQ.0.) GO TO 110	000470
00377	194*		READ (5,7) DUM2,DUM3	000472
00403	195*		GEOPR2(1,1)=DUM1	000501
00404	196*		GEOPR2(1,2)=DUM2	000503
00405	197*		GEOPR2(1,3)=DUM3	000505
00406	198*		GO TO 110	000507
00407	199*	100	DO 103 K2=1,NUMGEO	000511
00412	200*		DO 103 K3=1,3	000522
00415	201*	103	GEOPR2 (K2,K3)=1.	000522
00420	202*		WRITE (6,91)NUMGEO	000530
00423	203*	90	FORMAT 12X,'FOR EACH OF THE ',I3,' GEOMETRIC PROPERTY NUMBERS,THE	000541
00423	204*		IFOLLOWING PROPERTIES HAVE BEEN ASSIGNED:/'	000541
00423	205*		2,2X'AREA=1,IY=1,IZ=1. IF YOU WANT A DIFFERENT PROPERTY SET,ENTER IT	000541
00423	206*		3IN THE SAME ORDER.'2X' FOLLOWING THE GEOMETRIC PROPERTY NO. TO BE G	000541
00423	207*		4IVEN.---- OTHERWISE DEFAULT')	000541
00424	208*		DO 114 K4=1,NUMGEO	000541
00427	209*		WRITE (6,91)K4	000541
00432	210*	91	FORMAT (5X,'GEOMETRIC PROPERTY NO.',I4)	000547
00433	211*		READ(5,7)DUM1	000547
00436	212*		IF (DUM1.EQ.0.)GO TO 104	000555
00440	213*		READ(5,7) DUM2	000557
00443	214*		READ(5,7) DUM3	000565
00446	215*		GEOPR2(K4,1)=DUM1	000573
00447	216*		GEOPR2(K4,2)=DUM2	000575

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00450	217*		GEOPR2(K4,3)=DUM3	000577
00451	218*	104	CONTINUE	000603
00451	219*	C **	SET UP CRODID FOR USE IN ELEMENT DATA	000603
00453	220*		WRITE (6,92) NUMGEO	000603
00450	221*	92	FORMAT(2X, 'FOR EACH DESAP BAR ELEMENT NUMBER GIVEN, ENTER THE ASSOC	000614
00456	222*		11A, 'ED GEOMETRIC PROPERTY NUMBER 1 THROUGH', I4)	000614
00457	223*		DO 105 K5=1, 1, STOP	000614
00462	224*		WRITE (6,7) CRODID(K5,1)	000614
00465	225*	105	READ (5,7) CRODID (K5,5)	000621
00471	226*	110	CONTINUE	000631
00471	227*	C **	GEOMETRIC PROPERTIES DATA CARDS ARE DONE	000631
00472	228*		WRITE(12,96)	000631
00474	229*	96	FORMAT(' COMM GEOMETRIC PROPERTIES FROM BAR')	000635
00475	230*		DO 106 K6=1, NUMGEO	000635
00500	231*		WRITE (12,93) K6, (GEOPR2(K6,K7), K7=1,3)	000646
00500	232*	C	PRINT 5555	000646
00500	233*	C5555	FORMAT(' GEOPR2 TO FILE 12')	000646
00500	234*	C	WRITE (6,93) K6, (GEOPR2(K6,K7), K7=1,3)	000646
00507	235*	93	FORMAT (15,5X,3F10,3)	000665
00510	236*	106	CONTINUE	000665
00510	237*	C **	GEOMETRIC PROPERTIES DATA HAS BEEN SENT TO DESAP	000665
00510	238*	C **	GENERATE ELEMENT LOAD MULTIPLIERS	000665
00512	239*		WRITE (6,111) KOAD(J)	000665
00515	240*	111	FORMAT (15X, 'ELEMENT LOAD MULTIPLIERS FOR CONST. CODE', I4)	000677
00516	241*		DO 115 K6=1,4	000677
00521	242*		DO 115 K7=1,4	000677
00524	243*	115	EMUL(K6,K7)=C.	000677
00527	244*		DO 120 K8=1,4	000710
00532	245*		WRITE (6,112) VAR(K8)	000710
00535	246*	112	FORMAT (2X, 'FOR LOAD ', A1, ' ENTER X,Y,Z --ELEMENT LOAD FRACTIONS--	000715
00535	247*		1 IN THAT ORDER. DEFAULT = ALL ZEROS')	000715
00536	248*		READ (5,7) DUM1	000715
00541	249*		IF(DUM1.EQ.0.) GO TO 120	000723
00543	250*		READ (5,7) DUM2	000725
00546	251*		READ (5,7) DUM3	000733
00551	252*		EMUL (1,K8)=DUM1	000741
00552	253*		EMUL (2,K8)=DUM2	000743
00553	254*		EMUL (3,K8)=DUM3	000745
00554	255*	120	CONTINUE	000752
00554	256*	C **	ELEMENT LOAD MULTIPLIERS ARE DONE	000752
00554	257*	C	PRINT 6666	000752
00554	258*	C6666	FORMAT(' EMUL TO FILE 12')	000752
00556	259*		WRITE(12,117)	000752
00560	260*	117	FORMAT(' COMM EL. LOAD MULT1. FROM BAR')	000762
00561	261*		DO 121 K9=1,4	000762
00564	262*	121	WRITE(12,116) (EMUL(K9,K10), K10=1,4)	000762
00564	263*	C121	WRITE(6,116) (EMUL(K9,K10), K10=1,4)	000762
00573	264*	116	FORMAT (4F10,0)	001003
00573	265*	C **	GENERATE ELEMENT DATA	001003
00574	266*		DO 150 KROW=1, 1, STOP	001003
00577	267*		DO 149 KCOL=1,5	001003
00602	268*	149	ELDAT1(KROW,KCOL)=CRODID(KROW,KCOL)	001003
00604	269*		IX=ELDAT1(KROW,1)	001005
00604	270*	C	ELDAT1(KROW,6)=IX	001005
00604	271*	C	ELDAT2(KROW,1)=2*IX	001005
00605	272*		WRITE (6,130) IX	001007
00610	273*	130	FORMAT(2X, 'ENTER THE DESIGN VARIABLE NO., IUV, FOR ELEMENT NO.', I5)	001015

00611	274*		READ (5,7) ELDAT1(KROW,6)	001015
00614	275*		ELDAT2(KROW,1)=1.	001023
00614	276*	C	WRITE(6,131) IX	001023
00615	277*	131	FORMAT(2X,'ENTER DESIGN VARIABLE FRACTION ,FRC, FOR EL.NO.,',I4,' DE	001025
00615	278*		FAULT=1,0')	001025
00615	279*	C	READ(5,7)DUM	001025
00615	280*	C	IF(DUM.EQ.0.) GO TO 151	001025
00615	281*	C	ELDAT2(KROW,1)=DUM	001025
00616	282*	151	CONTINUE	001025
00617	283*		ELDAT2(KROW,2)=0	001025
00617	284*	C	WRITE(6,132) IX	001025
00620	285*	132	FORMAT(2X,'ENTER END-FIXITY COEFFICIENTS C FOR BUCKLING ABOUT THE	001026
00620	286*		1Y AND Z AXES, RESPECTIVELY, FOR EL.NO.,',I4)	001026
00620	287*	C	READ (5,7)ELDAT2(KROW,3),ELDAT2(KROW,4)	001026
00621	288*		ELDAT2(KROW,3)=1.	001026
00622	289*		ELDAT2(KROW,4)=1.	001027
00622	290*	C	ELDAT2(KROW,3)=IX	001027
00622	291*	C	ELDAT2(KROW,4)=IX*2.	001027
00623	292*	150	CONTINUE	001034
00623	293*	C	PRINT 776	001034
00623	294*	C776	FORMAT(' ELDAT1 AND ELDAT2 IN BAR THAT IS SENT TO ELORD')	001034
00623	295*	CC	DO 1122 KROW=1,ISTOP	001034
00623	296*	C	WRITE (6,7) (ELDAT1(KROW,J),J=1,6)	001034
00623	297*	C1122	WRITE (6,7) (ELDAT2(KROW,J),J=1,4)	001034
00623	298*	C**	ORDER THE ELDAT ARRAYS IN ASCENDING ORDER OF ELEMENT	001034
00625	299*		CALL ELORD (ISTOP,ELDAT1,ELDAT2,6,4)	001034
00625	300*	C	PRINT 4444	001034
00626	301*		WRITE(12,135)	001043
00630	302*	135	FORMAT(' COMM EL. DATA FROM BAR')	001050
00630	303*	C4444	FORMAT(' ELDAT1 AND ELDAT2 FROM ELORD TO BAR TO FILE 12')	001050
00631	304*		DO 160 K12=1,ISTOP	001050
00634	305*	160	WRITE (12,133) (ELDAT1(K12,K13),K13=1,6), (ELDAT2(K12,K13),K13=1,4)	001056
00634	306*	C160	WRITE (6,133) (ELDAT1(K12,K13),K13=1,6), (ELDAT2(K12,K13),K13=1,4)	001056
00647	307*	133	FORMAT(6I5,4F10.0,5X)	001101
00650	308*		GO TO (450,460), J1	001101
00651	309*	450	ISTOP1=ISTOP	001111
00652	310*		DO 451 K15=1,ISTOP	001112
00655	311*	451	IDV11(K15)=ELDAT1(K15,6)	001116
00657	312*		GO TO 1000	001120
00660	313*	460	ISTOP2=ISTOP	001122
00661	314*		DO 461 K16=1,ISTOP	001123
00664	315*	461	IDV12(K16)=ELDAT1(K16,6)	001127
00666	316*	1000	CONTINUE	001133
00670	317*		IF(KODE.EQ.1) GO TO 1100	001133
00672	318*		NIDV1=ISTOP1+ISTOP2	001136
00673	319*		DO 1010 K17=1,ISTOP1	001157
00676	320*	1010	IDV1(K17)=IDV11(K17)	001157
00700	321*		DO 1020 K18=1,ISTOP2	001166
00703	322*	1020	IDV1(ISTOP1+K18)=IDV12(K18)	001166
00705	323*		RETURN	001170
00706	324*	1100	NIDV1=ISTOP1	001174
00707	325*		DO 1110 K17=1,ISTOP1	001175
00712	326*	1110	IDV1(K17)=IDV11(K17)	001203
00714	327*		RETURN	001205
00715	328*		END	001260

END FOR

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0000 I 007330 ELDAT1	0000 R 016267 EMUL	0000 I 016247 ENDFOR	0000 R 014544 FRC	0000 I 014772 GEOPR1
0000 R 015251 GEOPR2	0000 R 015415 GEOPR3	0000 I 016322 I	0000 I 016352 IDUM	0000 I 016237 IDUMH
0000 I 016057 IDV21	0000 I 016141 IDV22	0000 I 016330 IFLAG	0000 I 016327 IJK	0000 I 016331 IJKL
0000 I 014064 INDEX	0000 017052 INJPS	0000 I 016325 ISTOP	0000 I 016407 ISTOP1	0000 I 016411 ISTOP2
0000 I 016344 IX	0000 I 016323 J	0000 I 016245 JOR	0000 I 016324 J1	0000 I 016343 J10
0000 I 016345 J11	0000 I 016350 J12	0000 I 016351 J13	0000 I 016353 J14	0000 I 016354 J15
0000 I 016335 K	0000 I 016336 KK	0000 I 016321 KKOE	0000 I 016340 KKODE	0000 I 016341 KKOL
0000 I 016223 KOAD	0000 I 016326 KOWNT	0000 I 016370 KROW	0000 I 016364 K10	0000 I 016410 K15
0000 I 016412 K16	0000 I 016413 K17	0000 I 016414 K18	0000 I 016355 K6	0000 I 016356 K7
0000 I 016357 K8	0000 I 016363 K9	0000 I 003554 MATCH	0004 000000 MAT11	0004 R 000024 MAT12
0000 I 015643 MTPRO1	0000 R 015667 MTPRO2	0000 I 016366 M1	0000 I 016405 M12	0000 I 016406 M13
0000 I 016367 M2	0000 I 016372 M3	0000 I 016373 M4	0000 I 016374 M4M1	0000 I 016377 M5
0000 I 016400 M6	0000 I 016376 M7	0000 I 016401 M8	0000 I 016402 M9	0000 I 016333 NEID
0000 I 016332 NNCOL	0000 I 016403 NNCOL1	0000 I 016404 NNCOL2	0000 I 016265 NODE	0000 I 016375 NONE
0000 I 016225 NPAR	0000 I 016334 NPID	0000 I 016346 NROW	0003 000022 NSTART	0003 I 000000 NUM
0000 I 016365 NUMFX	0000 I 016347 NUMGEO	0000 I 016342 NUMMAT	0000 I 016320 NUM13	0000 I 016337 NUM18
0000 I 016317 NUM6	0000 I 014710 PBAR1	0000 R 015054 PBAR2	0000 R 016303 SFT	0000 I 001130 TCBAR1
0000 I 016255 TYPE	0000 I 016233 VAR	0000 I 002114 VEID	0000 I 013724 VEID2	

00101	1*		SUBROUTINE BEAMCONFAC,NKOE,IDV2,NIDV2)	000002
00103	2*		COMMON/BACH/NUM(18),NSTART(18)/MOZART/MAT11(20),MAT12(20,10)	000002
00103	3*	C **	CHANGE GROUP, NO. OF DESAP BEAM(NASTRAN BAR) ELEMENTS	000002
00104	4*		INTEGER CBAR1(100,6),TCBAR1(100,5),VEID(100),CBARID(100,7),MATCH(1	000002
00104	5*		100,2),INDEX(100),ELDAT(100,16),ELDAT1(100,23),VEID2(100)	000002
00105	6*		DIMENSION CBAR2(100,3),FRC(100,1)	000002
00105	7*	C **	CHANGE GROUP, NO. OF PID'S	000002
00106	8*		INTEGER PBAR1(25,2),GEOPR1(25,2)	000002
00107	9*		DIMENSION PBAR2(25,5),GEOPR2(25,4),GEOPR3(25,6)	000002
00107	10*	C **	CHANGE GROUP, NO. OF MID'S	000002
00110	11*		INTEGER MTPRO1(20)	000002
00111	12*		REAL MTPRO2(20,6)	000002
00111	13*	C **	CHANGE GROUP, NO. OF DESIGN VARIABLES	000002
00112	14*		DIMENSION IDV2(100),IDV21(50),IDV22(50)	000002
00112	15*	C **	*****	000002
00113	16*		INTEGER KOAD(2),NPAR(6),VAR(4),IDUMM(6),JOR(2),ENDFOR(6),TYPE(2)	000002
00114	17*		DIMENSION DUMM(6),NODE(2),EMUL(4,3),SFT(12)	000002
00115	18*		REAL MAT12	000002
00116	19*		DATA VAR,'A','B','C','D'/TYPE/'BEAM', ' /JOR/'I','J'/	000002
00122	20*	7	FORMAT ()	000002
00123	21*		NUM6=NUM(6)	000002
00124	22*		NUM13=NUM(13)	000004
00125	23*		KKOE=6	000006
00126	24*		CALL CANDP (KKOE,NUM6,NUM13,CBAR1,CBAR2,PBAR1,PBAR2)	000010
00126	25*	C **	CBAR2 IS NEVER USED	000010
00126	26*	C	WRITE (6,5)	000010
00126	27*	C5	FORMAT (25X,'BEAM ELEMENT DATA (BAR IN NASTRAN)')	000010
00126	28*	C	PRINT 14	000010
00126	29*	C14	FORMAT(' CBAR1 AND CBAR2 IN BEAM FROM CANDP')	000010
00126	30*	C	DO 9 L=1,NUM6	000010
00126	31*	C9	PRINT 88, (CBAR1(L,J),J=1,5),(CBAR2(L,J),J=1,3)	000010
00126	32*	C88	FORMAT(2X,5I6,3F8.2)	000010
00126	33*	C	PRINT 13	000010
00126	34*	C13	FORMAT(' PBAR1 AND PBAR2 IN BEAM FROM CANDP')	000010
00126	35*	C	DO 16 L=1,NUM13	000010

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00126	36*	C16	PRINT 12, (PBAR1(I,J),J=1,2), (PBAR2(I,J),J=1,5)	000010
00126	37*	C12	FORMAT(2X,2I6,5F6.2)	000010
00127	38*		CALL FORM1(NKODE,KOAD,TYPE)	000021
00130	39*		NPAR(1)=2	000026
00131	40*		NPAR(2)=NUM6	000030
00132	41*		DO 999 I=1,200	000047
00135	42*		DO 999 J=1,5	000047
00140	43*	999	TCBAR1(I,J)=CBAR1(I,J)	000047
00143	44*		DO 1000 J1=1,NKODE	000060
00146	45*		NPAR(5)=KOAD(J1)	000060
00146	46*	C **	IF ONLY ONE CONSTRUCTION CODE, SKIP TO 25	000060
00147	47*		IF (NKODE.EQ.1) GO TO 25	000062
00151	48*		IF (J1.EQ.2) GO TO 4000	000064
00153	49*		CALL FORM2(J1,KOAD,TYPE,NPAR(2),VEID,NUM6)	000067
00154	50*		ISTOP=NPAR(2)	000101
00155	51*		KOWNT=J	000103
00156	52*		DO 3100 IJK=1,NUM6	000112
00161	53*		IFLAG=0	000112
00162	54*		DO 3050 IJKL=1,ISTOP	000115
00165	55*	3050	IF (CBAR1(IJK,1).EQ.VEID(IJKL)) IFLAG=1	000115
00170	56*		IF (IFLAG.EQ.1) GO TO 3100	000123
00172	57*		KOWNT=KOWNT+1	000126
00173	58*		VEID2(KOWNT)=CHAR1(IJK,1)	000132
00174	59*	3100	CONTINUE	000140
00176	60*	4000	CONTINUE	000140
00177	61*		IF (J1.EQ.1) GO TO 4100	000140
00201	62*		ISTOP=NUM6-ISTOP	000142
00202	63*		NPAR(2)=ISTOP	000146
00203	64*		DO 4050 IJK=1,ISTOP	000153
00206	65*	4050	VEID(IJK)=VEID2(IJK)	000153
00206	66*	C	PRINT 4051,ISTOP	000153
00206	67*	C	DO 4052 IJK=1,ISTOP	000153
00206	68*	C4052	PRINT 4053,VEID(IJK)	000153
00206	69*	C4053	FORMAT(I6)	000153
00206	70*	C4051	FORMAT(/' VEID IN BEAM FOR J1=2'/' ISTOP=' ,I6)	000153
00210	71*	4100	CONTINUE	000161
00211	72*		DO 888 I=1,200	000161
00214	73*		DO 888 J=1,5	000161
00217	74*	888	CBAR1(I,J)=TCBAR1(I,J)	000161
00217	75*	C	PRINT 333	000161
00217	76*	C333	FORMAT(' CBAR1 IN BEAM JUST BEFORE STOR')	000161
00217	77*	C	DO 777 I=1,NUM6	000161
00217	78*	C777	PRINT 222, (CBAR1(I,J),J=1,5)	000161
00217	79*	C222	FORMAT(5I5)	000161
00222	80*		NNCOL=5	000167
00223	81*		CALL STOR(ISTOP,NUM6,CBAR1,VEID,NNCOL)	000171
00223	82*	C	PRINT 67	000171
00223	83*	C87	FORMAT(' CBAR1 AND VEID IN BEAM JUST BACK FROM STOR')	000171
00223	84*	C	DO 77 J=1,ISTOP	000171
00223	85*	C77	PRINT 76, (CBAR1(J,K),K=1,5),VEID(J)	000171
00223	86*	C76	FORMAT(6I5)	000171
00224	87*		GO TO 26	000200
00225	88*	25	ISTOP=NUM6	000202
00226	89*	26	DO 70 J=1,ISTOP	000204
00231	90*		NEID=CHAR1(J,1)	000213
00232	91*		NPID=CBAR1(J,2)	000215
00232	92*	C **	FIND PBAR1 CARD WITH SAME PID	000215

00233	93*		DO 65 K=1,NUM13	000222
00236	94*		IF(NPID.EQ.PBAR1(K,1))GO TO 61	000222
00240	95*		GO TO 65	000225
00241	96*	61	KK=K	000227
00242	97*		GO TO 62	000230
00243	98*	65	CONTINUE	000234
00245	99*	62	CBAR ID(J,1)=NEID	000234
00246	100*		CBARID(J,2)=CBAR1(J,3)	000236
00247	101*		CBARID(J,3)=CBAR1(J,4)	000240
00250	102*		CBARID(J,4)=CBAR1(J,5)	000242
00251	103*		CBARID(J,5)=PBAR1(KK,2)	000244
00252	104*		CBARID(J,6)=NPID	000246
00253	105*	70	CONTINUE	000252
00253	106*	C	PRINT 73	000252
00253	107*	C73	FORMAT(' CBARID FROM BEAM')	000252
00253	108*	C	DO 74 J=1,ISTOP	000252
00253	109*	C74	PRINT 57, (CBARID(J,K),K=1,7)	000252
00253	110*	C57	FORMAT(7I5)	000252
00253	111*	C **	CBARID ARRAY IS DONE EXCEPT FOR CBARID (.7)	000252
00253	112*	C **	FIND NUMMAT AND MATCH	000252
00255	113*		NUM18=NUM(18)	000252
00256	114*		KKODE=0	000254
00257	115*		KKOL=5	000255
00257	116*	C	PRINT 1011, ISTOP,NUM6,NUM18,NUM13	000255
00257	117*	C1011	FORMAT(4I5)	000255
00260	118*		CALL PAIR(KKODE,CBARID,PBAR1,ISTOP,NUM18,KKOL,NUMMAT,MATCH,INDEX)	000257
00261	119*		NPAR(3)=NUMMAT	000274
00262	120*		WRITE(6,503)KOAD(J1)	000276
00265	121*	503	FORMAT(2X,'ENTER NO. OF FIXED END FORCE SETS FOR CONST. CODE',I4)	000305
00266	122*		READ (5,7) NPAR(6)	000305
00266	123*	C **	SET UP MATERIAL CONTROL CARD FOR EACH OF THE NUMMAT MATI CARDS ASSOCIATED	000305
00266	124*	C **	WITH CBARID	000305
00271	125*		DO 83 J10=1,NUMMAT	000316
00274	126*		IX=INDEX(J10)	000316
00275	127*		MTPRO1(J10)-IX	000320
00275	128*	C **	FOR MID # INDEX(J10),FIND CORRES. ROW # IN MAT12	000320
00276	129*		DO 84 J11=1,NUMMAT	000324
00301	130*		IF(MATCH(J11,1).EQ. IX)GO TO 85	000324
00303	131*		GO TO 84	000327
00304	132*	85	NROW = MATCH(J11,2)	000331
00305	133*		GO TO 86	000332
00306	134*	84	CONTINUE	000336
00310	135*	86	MTPRO2(J10,1)=MAT12(NROW,4)*CONFAC	000336
00311	136*		MTPRO2(J10,2)=MAT12(NROW,1)	000341
00312	137*		MTPRO2(J10,3)=MAT12(NROW,3)	000343
00313	138*		MTPRO2(J10,4)=MAT12(NROW,8)	000345
00314	139*		MTPRO2(J10,5)=MAT12(NROW,9)	000347
00315	140*		MTPRO2(J10,6)=MAT12(NROW,10)	000351
00316	141*	83	CONTINUE	000355
00316	142*	C **	CALCULATE NPAR(4)	000355
00316	143*	C **	SET UP GEOMETRIC PROPERTIES DATA	000355
00320	144*		KKODE = 1	000355
00321	145*		KKOL=6	000357
00322	146*		CALL PAIR(KKODE,CBARID,PBAR1,ISTOP,NUM13,KKOL,NUMGEO,MATCH,INDEX)	000361
00323	147*		WRITE(12,795)	000374
00325	148*		NPAR(4)=NUMGEO	000404
00326	149*		WRITE (12,504)(NPAR(J11),J11=1,6)	000406

00331	150*	795	FORMAT(' COMM ELEMENT CONTROL FROM BEAM')	000416
00331	151*	C **	NPAR TO FILE 12	000416
00331	152*	C	PRINT 799	000416
00331	153*	C799	FORMAT(' NPAR TO FILE 12')	000416
00331	154*	C	PRINT 504, (NPAR(IP),IP-1,6)	000416
00332	155*	504	FORMAT (6I5)	000416
00333	156*		WRITE(12,349)	000416
00335	157*	349	FORMAT(' COMM MATERIAL PROPERTIES FROM BEAM')	000427
00336	158*		DO 350 J12=1,NUMMAT	000427
00341	159*	350	WRITE (12,300) MTPRO1(J12),(MTPRO2(J12,J13),J13=1,6)	000427
00351	160*	300	FORMAT(15,5X,F10.4,F10.0,F10.4,3F10.0)	000445
00351	161*	C	PRINT 8999	000445
00351	162*	C8999	FORMAT(' MTPRO1 AND MTPRO2 TO FILE 12')	000445
00351	163*	C	PRINT 300, MTPRO1(J12),(MTPRO2(J12,J13),J13=1,6)	000445
00352	164*		WRITE (6,270)KOAD(J1)	000445
00355	165*	270	FORMAT (2X,' FOR CONST. CODE',I4,' FOR THE FOLLOWING PROPERTY IDS,	000456
00355	166*		ENTER CROSS-SECTION CODE (KSEC):1=SYMMETRIC;2=ZEE;3=CIRCULAR . DEF	000456
00355	167*		2AULT=1')	000456
00356	168*		DO 282 J10=1,NUMGEO	000456
00361	169*		GEOPR1(J10,2)=1	000456
00362	170*		WRITE (6,271)INDEX(J10)	000457
00365	171*	271	FORMAT(5X,'PROPERTY ID=',I4)	000465
00366	172*		READ (5,7) IDUM	000465
00371	173*		IF (IDUM.NE.0) GEOPR1(J10,2)=IDUM	000473
00373	174*	282	CONTINUE	000507
00375	175*		DO 283 J10=1,NUMGEO	000507
00400	176*		GEOPR1(J10,1)=INDEX(J10)	000507
00400	177*	C **	FOR PID NO. INDEX(J10)FIND CORRES.ROW NO. IN PBAR2	000507
00400	178*		DO 284 J11=1,NUMGEO	000514
00404	179*		IF (MATCH(J11,1).EQ. INDEX (J10)) GO TO 285	000514
00406	180*		GO TO 284	000517
00407	181*	285	NROW=MATCH(J11,2)	000521
00410	182*		GO TO 286	000522
00411	183*	284	CONTINUE	000526
00413	184*	286	GEOPR2(J10,1)=PBAR2(NROW,1)	000526
00414	185*		GEOPR2(J10,2)=PBAR2(NROW,4)	000530
00415	186*		GEOPR2(J10,3)=PBAR2(NROW,3)	000532
00416	187*		GEOPR2(J10,4)=PBAR2(NROW,2)	000534
00417	188*		DO 290 J12=1,6	000543
00422	189*	290	GEOPR3(J10,J12)=0.	000543
00424	190*		WRITE(6,272)INDEX(J10)	000544
00427	191*	272	FORMAT(2X,'FOR PROPERTY ID',I4,' ENTER THE SIX SECTION MODULI 2. Y	000554
00427	192*		YOU HAVE TO SPECIFY ALL OR NONE (DEFAULT 1./15X.'DEFAULT MEANS CORR	000554
00427	193*		RESPONDING STRESS IS ZERO')	000554
00430	194*		DO 295 J13=1,6	000554
00433	195*	295	GEOPR3(J10,J13)=0.	000554
00435	196*		HEAD(5,7)DUMM(1)	000555
00440	197*		IF (DUMM(1).EQ.0.) GO TO 297	000563
00442	198*		READ(5,7) (DUMM(J14),J14=2,6)	000565
00445	199*		DO 298 J15=1,6	000602
00450	200*	298	GEOPR3 (J10,J15)=DUMM(J15)	000602
00452	201*	297	CONTINUE	000615
00453	202*	283	CONTINUE	000615
00453	203*	C **	GEOMETRIC PROPERTY DATA CARDS ARE DONE	000615
00455	204*		WRITE(12,96)	000615
00457	205*	96	FORMAT(' COMM GEOMETRIC PROPERTIES FROM BEAM')	000627
00460	206*		DO 299 J10=1,NUMGEO	000627

U0463	207*	299	WRITE (12,273)(GEOPR1(J10,J11),J11=1,2),(GEOPR2(J10,J12),J12=1,4),	000627
U0463	208*		1(GEOPR3(J10,J13),J13=1,6)	000627
U0463	209*	C	PRINT 327	000627
U0463	210*	C327	FORMAT(' GEOMETRIC PROPERTY CARDS TO FILE 12')	000627
U0463	211*	C	DO 331 J10=1,NUMGEO	000627
U0463	212*	C331	WRITE (6,273)(GEOPR1(J10,J11),J11=1,2),(GEOPR2(J10,J12),J12=1,4),	000627
U0463	213*	C	1(GEOPR3(J10,J13),J13=1,6)	000627
U0502	214*	273	FORMAT(215,4F10,3/6F10,3)	000661
U0502	215*	C **	GENERATE ELEMENT LOAD MULTIPLIERS	000661
U0503	216*		WRITE(6,111)KOAD(J1)	000661
U0506	217*	111	FORMAT (/15X,'ELEMENT LOAD MULTIPLIERS FOR CONST. CODE',I4/)	000673
U0507	218*		DO 115 K6=1,3	000673
U0512	219*		DO 115 K7=1,4	000673
U0515	220*	115	EMUL(K6,K7)=G.	000673
U0520	221*		DO 120 K8=1,4	000704
U0523	222*		WRITE (6,112) VAR(K8)	000704
U0526	223*	112	FORMAT(2X,'FOR LOAD ',A1,' ENTER X,Y,Z ELEMENT LOAD FRACTIONS, IN T	000711
U0526	224*		HAT ORDER ---DEFAULT=ALL ZEROS')	000711
U0527	225*		READ(5,7)DUM1	000711
U0532	226*		IF(DUM1.EQ.0.)GO TO 120	000717
U0534	227*		READ(5,7) DUM2,DUM3	000721
U0540	228*		EMUL(1,K8)=DUM1	000730
U0541	229*		EMUL(2,K8)=DUM2	000732
U0542	230*		EMUL(3,K8)=DUM3	000734
U0543	231*	120	CONTINUE	000741
U0543	232*	C	PRINT 6666	000741
U0543	233*	C6666	FORMAT(' EMUL TO FILE 12')	000741
U0545	234*		WRITE(12,1117)	000741
U0547	235*	1117	FORMAT(' COMM EL. LOAD MULTIPLIERS FROM BEAM')	000751
U0550	236*		DO 121 K9=1,4	000751
U0553	237*	121	WRITE (12,116)(EMUL(K9,K10),K10=1,4)	000751
U0553	238*	C121	PRINT 116, (EMUL(K9,K10),K10=1,4)	000751
U0562	239*	116	FORMAT (4F10,0)	000765
U0562	240*	C **	GENERATE FIXED-END FORCE DATA	000765
U0563	241*		IF (NPAR(6),EQ,0) GO TO 125	000765
U0565	242*		WRITE(6,117)KOAD(J1)	000767
U0570	243*	117	FORMAT (20X,' FIXED END FORCE DATA FOR CONST. CONF',I4)	000775
U0571	244*		NUMFX=NPAR(6)	000775
U0571	245*	C	PRINT 1221	000775
U0571	246*	C1221	FORMAT(' SFT TO FILE12')	000775
U0572	247*		WRITE(12,1118)	000777
U0574	248*	1118	FORMAT(' COMM SFT FROM BEAM')	001004
U0575	249*		DO 123 M1=1,NUMFX	001004
U0600	250*		WRITE (6,118)M1	001011
U0603	251*	118	FORMAT (2X,'FOR FIXED FORCE SET NO.',I3,' ENTER RX,RY,RZ,MX,MY,MZ	001017
U0603	252*		IFOR NODE I')	001017
U0604	253*		READ (5,7)(SFT(M2),M2=1,6)	001017
U0607	254*		WRITE (6,119)	001027
U0611	255*	119	FORMAT (5X,'SAME INFO FOR NODE J')	001034
U0612	256*		READ (5,7)(SFT(M2),M2=7,12)	001034
U0615	257*		WRITE (12,122)M1,(SFT(M2),M2=1,12)	001044
U0615	258*	C	WRITE (6,122)M1,(SFT(M2),M2=1,12)	001044
U0621	259*	122	FORMAT (15,6F10,2/5X,6F10,2)	001060
U0622	260*	123	CONTINUE	001060
U0624	261*	125	CONTINUE	001060
U0624	262*	C **	GENERATE ELEMENT DATA	001060
U0625	263*		WRITE (6,127)KOAD(J1)	001060

00631	264*	127	FORMAT (//15X,'BEAM ELEMENT DATA FOR CONST. CODE' I5/)	001101
00631	265*		DO 150 KROW=1,ISTOP	001101
00634	266*		IX=CBARID(KROW,1)	001101
00635	267*		NODE(1)=CBARID(KROW,2)	001103
00636	268*		NODE(2)=CBARID(KROW,3)	001105
00637	269*		WRITE (6,130)IX	001107
00642	270*	130	FORMAT (2X,'ENTER DESIGN VARIABLE NO.,IDV, FOR EL. NO.',I5)	001115
00643	271*		READ (5,7)CBARID(KROW,7)	001115
00646	272*		WRITE (6,131)IX	001123
00651	273*	131	FORMAT (2X,'ENTER DESIGN VARIABLE FRACTION,FRC, FOR EL. NO.',I4,	001131
00651	274*		1' DEFAULT =1.')	001131
00652	275*		FRC (KROW,1)=1.	001131
00653	276*		READ (5,7)DUM	001133
00656	277*		IF (DUM.EQ.0.) GO TO 151	001141
00660	278*		FRC (KROW,1)=DUM	001143
00661	279*	151	CONTINUE	001146
00662	280*		DO 152 M3=1,4	001146
00665	281*	152	ELDAT (KROW,M3)=	001152
00667	282*		IF (NPAR(6),EQ.0) GO TO 153	001154
00671	283*		WRITE (6,132)IX	001156
00674	284*	132	FORMAT (2X,'ENTER NUMBERS OF FIXED-END FORCE SETS FOR EL. LOADS A,	001164
00674	285*		1B,C,D RESPECT., FOR EL. NO.',I4,' OR DEFAULT TO ALL BLANKS')	001164
00675	286*		READ (5,7)IDUMM(1)	001164
00700	287*		IF (IDUMM(1).EQ.0.) GO TO 153	001172
00702	288*		READ(5,7) (IDUMM,M3),M3=2,4)	001176
00705	289*		DO 154 M3=1,4	001213
00710	290*	154	ELDAT(KROW,M3)=IDUMM(M3)	001213
00712	291*	153	CONTINUE	001217
00713	292*		DO 160 M4=5,16	001217
00716	293*	160	ELDAT(KROW,M4)=	001217
00721	294*		DO 170 M4=1,2	001224
00723	295*		M4M1=M4-1	001224
00724	296*		WRITE (6,135)IX,JOR(M4),NODE(M4)	001227
00731	297*	135	FORMAT (2X,'FOR EL. NO.',I3,' NODE ',A1,'(',I3,') ENTER THE NO. OF	001237
00731	298*		1 END FORCES RX,RY,RZ,MX,MY,MZ KNOWN TO BE ZERO')	001237
00732	299*		READ (5,7) NONE	001237
00735	300*		IF (NONE.EQ.0) GO TO 170	001245
00737	301*		IF (NONE.EQ.6) GO TO 168	001247
00741	302*		GO TO 169	001252
00742	303*	168	DO 165 M7=1,6	001254
00745	304*	165	ELDAT(KROW,4+M4M1*6+M7)=1'	001264
00747	305*		GO TO 170	001266
00750	306*	169	WRITE (6,137)JOR(M4),NODE(M4)	001270
00754	307*	137	FORMAT (2X,'FOR NODE ',A1,'(',I3,') ENTER THE END FORCES KNOWN TO	001305
00754	308*		1 BE ZERO--USE THE FOLLOWING CODES: RX=1,RY=2...MZ=6')	001305
00755	309*		READ (5,7) (ENDFOR,M5),M5=1,NONE)	001305
00763	310*		DO 171 M6=1,NONE	001327
00766	311*		DO 171 M7=1,6	001327
00771	312*		IF (ENFOR(M6),EQ.M7) ELDAT(KROW,4+M4M1*6+M7)=1'	001327
00773	313*	171	CONTINUE	001357
00776	314*	170	CONTINUE	001357
01000	315*	150	CONTINUE	001357
01002	316*		DO 175 M8=1,ISTOP	001357
01005	317*		DO 175 M9=1,7	001357
01010	318*	175	ELDAT1(M8,M9)=CBARID(M8,M9)	001357
01013	319*		DO 176 M8=1,ISTOP	001371
01016	320*		DO 176 M9=8,23	001371

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01021	321*	176	ELDAT1(M8,M9)=ELDAT(M8,M9-7)	001371
01024	322*		NNCOL1=23	001377
01025	323*		NNCOL2=1	001401
01026	324*		CALL ELORD(ISTOP,ELDAT1,FRC,NNCOL1,NNCOL2)	001403
01026	325*	C	PRINT 4444	001403
01026	326*	C4444	FORMAT(' ELEMENT DATA TO FILE 12')	001403
01027	327*		WRITE(12,4445)	001412
01031	328*	4445	FORMAT(' COMM EL. DATA FROM BEAM')	001417
01032	329*		DO 180 M12=1,ISTOP	001417
01035	330*	180	WRITE(12,136)(ELDAT1(M12,M13),M13=1,7),FRC(M12),(ELDAT1(M12,M13),	001426
01035	331*		M13=8,23)	001426
01035	332*	C180	WRITE(16,136)(ELDAT1(M12,M13),M13=1,7),FRC(M12),(ELDAT1(M12,M13),	001426
01035	333*	C	M13=8,23)	001426
01051	334*	136	FORMAT(7I5,F10.0,4A5,12A1,3X)	001454
01052	335*		GO TO (450,460), J1	001454
01053	336*	450	ISTOP1=ISTOP	001464
01054	337*		DO 451 K15=1,ISTOP	001465
01057	338*	451	IDV21(K15)=ELDAT1(K15,7)	001471
01061	339*		GO TO 1000	001473
01062	340*	460	ISTOP2=ISTOP	001475
01063	341*		DO 461 K16=1,ISTOP	001476
01066	342*	461	IDV22(K16)=ELDAT1(K16,7)	001502
01070	343*	1000	CONTINUE	001506
01072	344*		IF(NK00E.E0.1) GO TO 1100	001506
01074	345*		NIDV2=ISTOP1+ISTOP2	001511
01075	346*		DO 1010 K17=1,ISTOP1	001532
01100	347*	1010	IDV2(K17)=IDV21(K17)	001532
01102	348*		DO 1020 K18=1,ISTOP2	001541
01105	349*	1020	IDV2(ISTOP1+K18)=IDV22(K18)	001541
01107	350*		GO TO 1115	001543
01110	351*	1100	NIDV2=ISTOP1	001545
01111	352*		DO 1110 K17=1,ISTOP1	001546
01114	353*	1110	IDV2(K17)=IDV21(K17)	001554
01116	354*	1115	CONTINUE	001557
01117	355*		RETURN	001557
01120	356*		END	001645

•H06,P

CANDP

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qFOR,S CANDP,CANDP
HSA E3 -07/30/80-09:45:12 (53,)

SUBROUTINE CANDP ENTRY POINT C00663

STORAGE USED: CODE(1) 000710; DATA(0) 000141; BLANK COMMON(2) 000000

COMMON BLOCKS:

0003 BACH 000044
0004 HANDEL 000174

EXTERNAL REFERENCES (BLOCK, NAME)

0005 ID
0006 NREWS
0007 NRDU\$
0010 NIO3\$
0011 NIO2\$
0012 NERR2\$
0013 NIO1\$
0014 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000	000033 1F	0000	000054 10F	0000	000057 11F	0001	000041 115G	0001	000064 127G
0001	000113 134G	0001	000150 145G	0001	000101 15L	0001	000156 151G	0001	000132 16L
0001	000207 162G	0001	000175 17L	0001	000242 174G	0001	000230 18L	0001	000261 20L
0000	000034 201F	0001	000273 205G	0001	000314 21L	0001	000332 217G	0001	000340 223G
0001	000372 236G	0001	000412 246G	0001	000427 25L	0001	000445 253G	0001	000453 257G
0001	000504 26L	0001	000561 27L	0001	000522 273G	0001	000530 277G	0001	000577 312G
0001	000605 316G	0000	000062 35F	0000	000065 36F	0000	000070 37F	0000	000042 5F
0001	000356 51L	0001	000635 53L	0000	000044 6F	0000	000047 7F	0000	000052 8F
0000 R	000000 00M	0000	000100 INJPS	0000 I	000030 J	0000 I	000031 KC	0000 I	000027 KDEM4
0000 I	000032 KP	0004	000000 KVCMD	0004	000024 KVCPI0	0000 I	000025 L	0000 I	000026 LIM
0004	000170 MAXMID	0004	000171 MAXPID	0000 I	000024 NEND	0003 I	000022 NSTART	0003 I	000000 NUM
0004 I	000172 NUMMID	0004 I	000173 NUMPID						

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00101	1*	SUBROUTINE CANDPIKUE,NCCARD,NPCARD,COUT1,COUT2,POUT1,POUT2)	000030
00103	2*	COMMON/BACH/NUM(18),NSTART(18)	000030
00104	3*	COMMON/HANDEL/KVCMD(20),KVCPI0(100),MAXMID,MAXPID,NUMMID,NUMPID	000030
00104	4*	C ** CHANGE GROUP, NO. OF ELEMENTS	000030
00105	5*	INTEGER COUT1(100,6)	000030
00106	6*	DIMENSION COUT2(100,3)	000030
00106	7*	C ** CHANGE GROUP, NO. OF PID'S	000030
00107	8*	INTEGER POUT1(25,2)	000030
00110	9*	DIMENSION POUT2(25,5)	000030
00110	10*	C ** *****	000030
00111	11*	DIMENSION DIM(20)	000030
00111	12*	C ** KDE. INPUT, 5=ROU,6=BAR,ETC.	000030

00111	13*	C **	NCCARD. INPUT NO. OF C CARDS TO BE READ	000030
00111	14*	C **	NPCARD. INPUT NO. OF P CARDS TO BE READ	000030
00111	15*	C **	COUT1,COUT2. OUTPUT	000030
00111	16*	C **	POUT1,POUT2. OUTPUT	000030
00112	17*		REWIND 11	000030
00113	18*		NEND = NSTART(KDE)-1	000033
00114	19*		DO 50 L=1,NEND	000036
00117	20*	50	READ (11,1)DUM	000041
00123	21*	1	FORMAT (20A4)	000051
00124	22*		LIM= NUM(KDE)	000051
00125	23*		KDEM4 =KDE-4	000053
00125	24*	C	PRINT 100	000053
00125	25*	C100	FORMAT(1,X'OUTPUT FROM CANDP FOR COUT1 AND COUT2, IN CANDP'/)	000053
00126	26*		DO 51 L=1,LIM	000064
00131	27*		GO TO (15,16,17,18,17,20,21),KDEM4	000064
00132	28*	15	READ (11,5) (COUT1(L,J),J=1,4)	000101
00132	29*	C	PRINT 5, (COUT1(L,J),J=1,4)	000101
00132	30*	C	PRINT 200,NUMPID	000101
00132	31*	C200	FORMAT(' NUMPID IN CANDP(COUT) ',18)	000101
00140	32*		CALL ID(2,NUMPID,COUT1(L,2),KC)	000116
00141	33*		COUT1(L,2)=KC	000126
00141	34*	C	PRINT 5, (COUT1(L,J),J=1,4)	000126
00142	35*		GO TO 51	000130
00143	36*	16	READ (11,6) (COUT1(L,J),J=1,5),(COUT2(L,J),J=1,3)	000132
00143	37*	C	WRITE (6,6) (COUT1(L,J),J=1,4),(COUT2(L,J),J=1,3)	000132
00155	38*		CALL ID(2,NUMPID,COUT1(L,2),KC)	000161
00156	39*		COUT1(L,2)=KC	000171
00157	40*		GO TO 51	000173
00160	41*	17	READ(11,7) (COUT1(L,J),J=1,6),COUT2(L,1)	000175
00160	42*	C	WRITE(6,7) (COUT1(L,J),J=1,6),COUT2(L,1)	000175
00167	43*		CALL ID(2,NUMPID,COUT1(L,2),KC)	000214
00170	44*		COUT1(L,2)=KC	000224
00171	45*		GO TO 51	000226
00172	46*	18	READ(11,8) (COUT1(L,J),J=1,6)	000230
00172	47*	C	WRITE(6,8) (COUT1(L,J),J=1,6)	000230
00200	48*		CALL ID(2,NUMPID,COUT1(L,2),KC)	000245
00201	49*		COUT1(L,2)=KC	000255
00202	50*		GO TO 51	000257
00203	51*	20	READ (11,10) (COUT1(L,J),J=1,5),COUT2(L,1)	000261
00203	52*	C	WRITE(6,10) (COUT1(L,J),J=1,5),COUT2(L,1)	000261
00212	53*		CALL ID(2,NUMPID,COUT1(L,2),KC)	000300
00213	54*		COUT1(L,2)=KC	000310
00214	55*		GO TO 51	000312
00215	56*	21	READ (11,11) (COUT1(L,J),J=1,2),(COUT2(L,J),J=1,4)	000314
00215	57*	C	WRITE(6,11) (COUT1(L,J),J=1,2),(COUT2(L,J),J=1,4)	000314
00227	58*		CALL ID(2,NUMPID,COUT1(L,2),KC)	000343
00230	59*		COUT1(L,2)=KC	000353
00231	60*	51	CONTINUE	000361
00233	61*		REWIND 11	000361
00234	62*		NEND = NSTART(KDE+7)-1	000364
00235	63*		DO 52 L=1,NEND	000367
00240	64*	52	READ (11,1)DUM	000372
00244	65*		LIM = NUM(KDE+7)	000402
00244	66*	C	PRINT 101	000402
00244	67*	C101	FORMAT(1,X' OUTPUT FROM CANDP FOR POUT1 AND POUT2, IN CANDP'/)	000402
00245	68*		DO 53 L=1,LIM	000404
00250	69*		GO TO (25,26,27,27,27,27),KDEM4	000412

00251	70*	25	READ (11,35) (POUT1(L,J),J=1,2),(POUT2(L,J),J=1,4)	000427
00251	71*	C	PRINT 35, (POUT1(L,J),J=1,2),(POUT2(L,J),J=1,4)	000427
00263	72*		CALL ID(2,NUMPID,POUT1(L,1),KP)	000456
00264	73*		POUT1(L,1)=KP	000466
00264	74*	C	PRINT 201,NUMPID	000466
00265	75*	201	FORMAT(' NUMPID IN CANDP(POUT)', I8)	000470
00266	76*		CALL ID(1,NUMPID,POUT1(L,2),KP)	000470
00267	77*		POUT1(L,2)=KP	000500
00267	78*	C	PRINT 35, (POUT1(L,J),J=1,2),(POUT2(L,J),J=1,4)	000500
00270	79*		GO TO 53	000502
00271	80*	26	READ (11,36) (POUT1(L,J),J=1,2),(POUT2(L,J),J=1,5)	000504
00271	81*	C	PRINT 36, (POUT1(L,J),J=1,2),(POUT2(L,J),J=1,5)	000504
00303	82*		CALL ID(2,NUMPID,POUT1(L,1),KP)	000533
00304	83*		POUT1(L,1)=KP	000543
00305	84*		CALL ID(1,NUMPID,POUT1(L,2),KP)	000545
00306	85*		POUT1(L,2)=KP	000555
00306	86*	C	PRINT 36, (POUT1(L,J),J=1,2),(POUT2(L,J),J=1,5)	000555
00307	87*		GO TO 53	000557
00310	88*	27	READ (11,37) (POUT1(L,J),J=1,2),(POUT2(L,J),J=1,2)	000561
00310	89*	C	PRINT 37, (POUT1(L,J),J=1,2),(POUT2(L,J),J=1,2)	000561
00322	90*		CALL ID(2,NUMPID,POUT1(L,1),KP)	000610
00323	91*		POUT1(L,1)=KP	000620
00324	92*		CALL ID(1,NUMPID,POUT1(L,2),KP)	000622
00325	93*		POUT1(L,2)=KP	000632
00325	94*	C	PRINT 37, (POUT1(L,J),J=1,2),(POUT2(L,J),J=1,2)	000632
00326	95*	53	CONTINUE	000640
00330	96*	5	FORMAT (8X,4I8)	000640
00331	97*	6	FORMAT (8X,5I8,3F8.4)	000640
00332	98*	7	FORMAT (8X,6I8,F8.4)	000640
00333	99*	8	FORMAT (8X,6I8)	000640
00334	100*	10	FORMAT (8X,6I8,F8.4)	000640
00335	101*	11	FORMAT (8X,2I8,4F8.4)	000640
00336	102*	35	FORMAT (8X,2I8,4F8.4)	000640
00337	103*	36	FORMAT (8X,2I8,5F8.4)	000640
00340	104*	37	FORMAT (8X,2I8,2F8.4)	000640
00341	105*		RETURN	000640
00342	106*		END	000707

END FOR

*HOG,P COUNT

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@FOR,S COUNT,COUNT
HSA E3 -07/30/80-09:45:18 (3,1)

SUBROUTINE COUNT ENTRY POINT 000053

STORAGE USED: CODE(1) 000063; DATA(0) 000012; BLANK COMMON(2) 000000

COMMON BLOCKS:

0003 HANDEL 000172

EXTERNAL REFERENCES (BLOCK, NAME)

0004 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000011	106G	0001	000022	120L	0001	000040	121L	0001	000027	122G	0001	000015	20L
0001	000033	21L	0000	I 000001	I	0000	000002	INJPS	0000	I 000000	KOUNT	0003	I 000000	KVCMID
0003	I 000024	KVCPID	0003	I 000170	MAXMID	0003	I 000171	MAXPID						

00101	1*		SUBROUTINE COUNT(NUMMID,NUMPID)	000005
00103	2*		COMMON/HANDEL/KVCMID(20),KVCPID(100),MAXMID,MAXPID	000005
00103	3*	C **	NUMMID(NUMPID),OUTPUT,NO,OF NON ZERO ELS.	000005
00103	4*	C **	IN KVCMID(KVCPID)	000005
00103	5*	C **		000005
00104	6*		KOUNT=0	000005
00105	7*		DO 100 I=1,MAXMID	000011
00110	8*		IF(KVCMID(I).NE.0) GO TO 20	000011
00112	9*		GO TO 120	000013
00113	10*	20	KOUNT=KOUNT+1	000015
00114	11*	100	CONTINUE	000022
00116	12*	120	CONTINUE	000022
00117	13*		NUMMID=KOUNT	000022
00120	14*		KOUNT=0	000023
00120	15*	C		000023
00121	16*		DO 101 I=1,MAXPID	000027
00124	17*		IF(KVCPID(I).NE.0) GO TO 21	000027
00126	18*		GO TO 121	000031
00127	19*	21	KOUNT=KOUNT+1	000033
00130	20*	101	CONTINUE	000040
00132	21*	121	CONTINUE	000040
00133	22*		NUMPID=KOUNT	000040
00134	23*		RETURN	000041
00135	24*		END	000062

END FOR

@HUG,P DIF

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FOR,S DIF,DIF
MSA E3 -07/30/80-U9:45:20 (3,)

SUBROUTINE DIF ENTRY POINT 000105

STORAGE USED: CODE(1) 000123; DATA(1) 000220; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NERR33

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000016	106G	0001	000033	116G	0001	000036	121G	0001	000060	136G	0001	000050	19L
0001	000042	30L	0000	I 000176	I	0000	000201	IN IPS	0000	I 000200	J	0000	I 000177	KOUNT
0000	I 000000	MAX	0000	I 000001	VIDV1									

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00101      1*          SUBROUTINE DIF(NVIDV,VIDV,NDIF)          000011
00101      2*      C ** NVIDV,INPUT.NO. OF ELS. IN VIDV      000011
00101      3*      C ** VIDV,INPUT,VECTOR OF DESIGN VARIABLE NOS. 000011
00101      4*      C ** NDIF,OUTPUT.NO. OF DIFFERENT DESIGN VARIABLE NOS. 000011
00103      5*          INTEGER VIDV(125),VIDV1(125)          000011
00104      6*          MAX=VIDV(1)          000011
00105      7*          DO 10 I=2,NVIDV          000016
00110      8*          IF(VIDV(I).GT.MAX) MAX=VIDV(I)          000016
00112      9*          10 CONTINUE          000025
00114     10*          KOUNT=0          000025
00115     11*          DO 20 J=1,MAX          000026
00120     12*          DO 19 J=1,NVIDV          000036
00123     13*          IF(VIDV(J).EQ.1) GO TO 30          000036
00125     14*          GO TO 19          000040
00126     15*          30 KOUNT=KOUNT+1          000042
00127     16*          VIDV1(KOUNT)=I          000045
00130     17*          19 CONTINUE          000053
00132     18*          20 CONTINUE          000053
00134     19*          40 NDIF=1          000053
00135     20*          DO 50 I=2,NVIDV          000060
00140     21*          IF(VIDV1(I).NE.VIDV1(I-1)) NDIF=NDIF+1 000060
00142     22*          50 CONTINUE          000067
00144     23*          RETURN          000067
00145     24*          END          000122
END FOR

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HDG,P ELORD

FOR,S ELORD,ELORD
MSA E3 -07/30/80-09:45:23 (20,)

SUBROUTINE ELORD ENTRY POINT 000151

STORAGE USED: CODE(1) 000165; DATA(0) 000073; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000041	116G	0001	000105	132G	0001	000122	142G	0001	000023	151L	0001	000053	152L				
0001	000047	153L	0001	000132	160L	0000	I	000000	EIDMAX	0000	I	CJ0001	HOLD1	0000	R	000030	HOLD2	
0000	I	000040	1	0000	000044	INJPS	0000	I	000041	J	0000	I	000036	NEND	0000	I	000037	NROWMX

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00101 1* SUBROUTINE ELORD(NUM,ELDAT1,ELDAT2,NCOL1,NCOL2) C00017
00101 2* C ** CHANGE GROUP. NO. OF ELEMENTS C00017
00103 3* INTEGER ELDAT1(100,23) C00017
00104 4* DIMENSION ELDAT2(100,6) C00017
00104 5* C ** ***** C00017
00105 6* INTEGER EIDMAX,HOLD1(23) C00017
00106 7* DIMENSION HOLD2(6) C00017
00106 8* C ORDERS ELDAT ARRAYS IN ASCENDING ORDER OF IEL (ELDAT1 (I,1)) C00017
00106 9* C NUM,INPUT,NO. OF ROWS IN ELDAT1 AND ELDAT2 C00017
00106 10* C ELDAT1,ELDAT2. ON INPUT,ELEMENT DATA ARRAYS. ON OUTPUT,EL. NOS. HAVE BEEN C00017
00106 11* C PUT IN ASCENDING ORDER C00017
00106 12* C NCOL1,NCOL2,INPUT,NO.OF COLS. IN ELDAT1 AND ELDAT2 C00017
00107 13* NEND=NUM+1 C00017
00110 14* 151 NEND=NEND-1 C00023
00110 15* C PRINT 111, NEND C00023
00110 16* C111 FORMAT(' NEND=',I3) C00023
00111 17* IF (NEND.EQ.1) GO TO 160 C00025
00113 18* EIDMAX=ELDAT1(1,1) C00027
00114 19* NROWMX=1 C00031
00115 20* DO 152 I=2,NLND C00041
00120 21* IF(ELDAT1(I,1).GT.EIDMAX) GO TO 153 C00041
00122 22* GO TO 152 C00045
00123 23* 153 EIDMAX=ELDAT1(I,1) C00047
00124 24* NROWMX=I C00050
00125 25* 152 CONTINUE C00055
00125 26* C PRINT 222, NEND,NROWMX,EIDMAX C00055
00125 27* C222 FORMAT(' NEND,NROWMX,EIDMAX=',3(2X,I3)) C00055
00125 28* C INTERCHANGE ROW WITH LARGEST EID WITH ROW NEND C00055
00127 29* IF(NROWMX.EQ.NEND) GO TO 151 C00055
00131 30* DO 154 J=1,NCOL1 C00060
00134 31* HOLD1(J)=ELDAT1(NEND,J) C00105
00135 32* ELDAT1(NEND,J)=ELDAT1(NROWMX,J) C00106
00136 33* ELDAT1(NROWMX,J)=HOLD1(J) C00110

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00137	34*	154	CONTINUE	000122
00141	35*		DO 155 J=1,NCOL2	000122
00144	36*		HOLD2(J)=ELDAT2(INEND,J)	000122
00145	37*		ELDAT2(INEND,J)=ELDAT2(NROWX,J)	000123
00146	38*		ELDAT2(NROWX,J)=HOLD2(J)	000125
00147	39*	155	CONTINUE	000130
00147	40*	C	PRINT 333, ELDAT1(1,1),ELDAT1(2,1),ELDAT1(3,1)	000130
00147	41*	C333	FORMAT,' ELDAT1-3 ROWS,FIRST COL.=',3(2X,I3)	000130
00151	42*		GO TO 151	000130
00152	43*	160	CONTINUE	000132
00153	44*		RETURN	000132
00154	45*		END	000164
END FOR				

ENDG,P FORM1

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@FOR,S FORM1,FORM1
HSA E3 -07/30/80-u9:45:27 (10,)

SUBROUTINE FORM1 ENTRY POINT L00056

STORAGE USED: CODE(1) 000072; DATA(0) 000042; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NWDLS
0004 N101\$
0005 N102\$
0006 NRDUS
0007 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001 000010 1066 C001 000044 20L 0000 000000 6F 0000 000015 7F 0000 000016 8F
0000 000031 INJP\$

00101	1*		SUBROUTINE FORM1(NKODE,KOAO,TYPE)	000000
00103	2*		INTEGER KOAO(2),TYPE(2)	000000
00103	3*	C	NKODE.OUTPUT.NO. OF CONST. CODES	000000
00103	4*	C	KOAO.OUTPUT.THE NOS. OF THE CONST. CODES	000000
00103	5*	C	TYPE.INPUT.NAME OF CURRENT ELEMENT	000000
00104	6*		WRITE (6,6)TYPE	000000
00112	7*	6	FORMAT (5X,'ENTER THE NO. OF CONST. CODES USED FOR ',A6,A2,' ELEME	000013
00112	8*		INTS,1 OR 2')	000013
00113	9*		KOAO(1)=1	000013
00114	10*		KOAO(2)=2	000015
00115	11*		READ(5,7)NKODE	000017
00120	12*	7	FORMAT()	000025
00121	13*		IF(NKODE.EC.2) GO TO 20	000025
00123	14*		WRITE (6,8)	000030
00125	15*	8	FORMAT (5X,'ENTER THE CONST. CODE NO. (1 OR 2)')	000035
00126	16*		READ(5,7) KOAO(1)	000035
00131	17*	20	RETURN	000044
00132	18*		END	000071

END FOR

@HDG,P FORM2

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FORM,S FORM2,FORM2
HSA E3 -07/30/80-09:45:31 (22,)

SUBROUTINE FORM2 ENTRY POINT 000141

STORAGE USED: CODE(1) 000166; DATA(0) 000131; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NRDUS
0004 NI01\$
0005 NI02\$
0006 NRDUS
0007 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 000050 10F 0001 000006 106G 0001 000017 114G 0001 000046 130G 0001 000054 134G
0001 000107 152G 0001 000122 161G 0000 000014 2CF 0000 000047 7F 0000 000002 9F
0000 I 000000 I 0000 000113 INJP\$ 0000 I 000001 K

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00101 1* SUBROUTINE FORM2(J1,KOAD,TYPE,ISTOP,VEID,MAX) 000006
00101 2* C ** J1.INPUT.INDEX OF KOAD 000006
00101 3* C ** KOAD.INPUT.CONTAINS CONST. CODES 000006
00101 4* C ** TYPE.INPUT.TYPE OF CURRENT ELEMENT 000006
00101 5* C ** ISTOP.OUTPUT.NO. OF ELS. FOR CURRENT CONST. CODE 000006
00101 6* C ** VEID.OUTPUT.ID NUMBERS OF ELEMENTS 000006
00101 7* C ** MAX.INPUT.CONTAINS MAXIMUM NO. OF ELEMENTS 000006
00101 8* C ** 000006
00101 9* C ** CHANGE GROUP, NO. OF ELEMENTS 000006
00103 10* INTEGER VEID(100) 000006
00103 11* C ** ***** 000006
00104 12* INTEGER KOAD(2),TYPE(2) 000006
00105 13* DO 1 I=1,100 000006
00110 14* 1 VEID(I)=0 000006
00112 15* WRITE (6,9)TYPE,KOAD(J1) 000007
00121 16* 9 FORMAT(2X,'ENTER NO. OF ',A6,A2,' ELS. FOR CONST. CODE',I4) 000024
00122 17* READ (5,7)ISTOP 000024
00125 18* IF(ISTOP.GE.MAX) WRITE(6,20) TYPE,TYPE,MAX 000032
00141 19* 20 FORMAT(1' THE NO. OF ',A6,A2,' ELS. FOR CONST. CODE1 IS GREATER',/5X 000061
00141 20* 1'THAN OR EQUAL TO THE TOTAL NO. OF ',A6,A2,' ELS.,WHICH IS',I4/15X 000061
00141 21* 2,' REENTER NO. OF ELS.')

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00165 30* RETURN
00166 31* END
END FOR

000125
000165

HDG,P ID

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@FOR,S ID,ID
HSA E3 -07/30/80-09:45:34 (7,)

SUBROUTINE ID ENTRY POINT Q00045

STORAGE USED: CODE(1) Q0C057; DATA(0) Q0C020; BLANK COMMON(2) Q0C000

COMMON BLOCKS:

Q003 HANDEL Q00172

EXTERNAL REFERENCES (BLOCK, NAME)

Q004 NERR2\$
Q005 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

Q001	Q00015	1L	Q001	Q00032	100L	Q001	Q00005	106G	Q001	Q00021	2L	Q001	Q00025	50L			
Q006	Q00001	7F	Q000	Q00007	INJPS	Q000	I	Q00000	K	Q003	I	Q00000	KVCMID	Q003	I	Q00024	KVCPID
Q003	Q00170	MAXMID	Q003	Q00171	MAXPID												

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Q0101 1* SUBROUTINE ID(KIND,NDIFID,IDIN,IDOUT) 000005
Q0103 2* COMMON/HANDEL/KVCMID(20),KVCPID(100),MAXMID,MAXPID 000005
Q0103 3* C ** KIND.INPUT.1=MID . 2=PID 000005
Q0103 4* C ** NDIFID.INPUT. THE NO OF ELS. IN KVCMID OR KVCPID 000005
Q0103 5* C ** IDIN. INPUT. THE ORIGINAL MID OR PID 000005
Q0103 6* C ** IDOUT.OUTPUT. MODIFIED MID OR PID 000005
Q0103 7* C 000005
Q0103 8* C 000005
Q0103 9* C 000005
Q0104 10* 7 PRINT 7,KIND,NDIFID,IDIN 000005
      FORMAT(' KIND,NDIFID,IDIN,INID ',3I8) 000005
Q0105 11* Q0 100 K=1,NDIFID 000005
Q0110 12* GO TO (1,2),KIND 000005
Q0111 13* 1 IF(IDIN.EQ.KVCMID(K)) GO TO 50 000015
Q0113 14* GO TO 100 000017
Q0114 15* 2 IF(IDIN.EQ.KVCPID(K)) GO TO 50 000021
Q0116 16* GO TO 100 000023
Q0117 17* 50 IDOUT=K 000025
Q0120 18* RETURN 000026
Q0121 19* 100 CONTINUE 000056
Q0123 20* END 000056
END FOR

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@HDG,P MAIN

XXXXI-29

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0000 I 002721 K8      0000 I 002667 LBUCK      0000 I 002712 LIM      0000 I 002672 L1      0000 I 002674 L2
0000 I 002676 L3      0000 I 002700 L4      0000 I 002704 L5      0000 I 002703 NCTOT      0000 I 002661 NCYCL
0000 I 002647 NELTYP  0000 I 002642 NELTY1      0000 I 002643 NELTY2      0000 I 002644 NELTY3      0000 I 002645 NELTY4
0000 I 002646 NELTY5  0000 I 002710 NEND      0000 I 002655 NIDVTO      0000 I 002650 NIDV1      0000 I 002651 NIDV2
0000 I 002652 NIDV3      0000 I 002653 NIDV4      0000 I 002654 NIDV5      0000 I 002706 NLOAD      0000 I 002662 NSCALE
0003 I 000022 NSTART  0003 I 000000 NUM      0000 I 002705 NUMDV      0000 I 002722 NUMNP      0000 I 002702 NUM10
0000 I 002671 NUM5      0000 I 002673 NUM6      0000 I 002675 NUM7      0000 I 002677 NUM8      0000 I 002701 NUM9
0000 I 000000 R1      0000 R 000310 R2      0000 R 002506 STR

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00100 1* C ** MAIN DRIVER 000000
00101 2* COMMON/BACH/NUM(10),NSTART(10) 000000
00101 3* C ** FIRST DIMENSION OF R1 AND R2 MUST BE THE SAME AS 000000
00101 4* C ** THAT OF ELDAT1 AND ELDAT2 IN ELORD. 000000
00103 5* INTEGER R1(100,2) 000001
00104 6* DIMENSION R2(100,2) 000001
00104 7* C ** NO. OF DIFFERENT DESAP BAR DESIGN VARIABLES 000001
00105 8* DIMENSION IDV1(100) 000001
00105 9* C ** NO. OF DIFFERENT DESAP BEAM DESIGN VARIABLES 000001
00106 10* DIMENSION IDV2(100) 000001
00106 11* C ** NO. OF DIFFERENT DESAP QUAD DESIGN VARIABLES 000001
00107 12* DIMENSION IDV3(100) 000001
00107 13* C ** NO. OF DIFFERENT DESAP SHEAR DESIGN VARIABLES 000001
00110 14* DIMENSION IDV4(100) 000001
00110 15* C ** NO. OF DIFFERENT DESAP QT DESIGN VARIABLES 000001
00111 16* DIMENSION IDV5(100) 000001
00111 17* C ** TOTAL NO. OF DIFFERENT DESIGN VARIABLES 000001
00112 18* DIMENSION IDV(150),A(150,2) 000001
00112 19* C ** NO. OF STRUCTURAL LOAD MULTIPLIERS 000001
00113 20* DIMENSION STR(15,4) 000001
00113 21* C ** ***** 000001
00114 22* INTEGER DUMM(4),BUCK1(4) 000001
00115 23* DIMENSION HEAD(20),BUCK2(3) 000001
00116 24* DATA DUM/D.,NELTY1,NELTY2,NELTY3,NELTY4,NELTY5,NELTYP, 000001
00116 25* /NIDV1,NIDV2,NIDV3,NIDV4,NIDV5,NIDVTO/12*D/DUMM/' COM'/ 000001
00135 26* WRITE (6,50) 000001
00137 27* 50 FORMAT (15X,'ENTER JOB HEADING ON ONE LINE') 000006
00140 28* READ(5,1) HEAD 000006
00143 29* 1 FORMAT(2LA4) 000016
00143 30* C ** PLACE CARD I ON FILE TO DESAP 000016
00144 31* WRITE (12,1)HEAD 000016
00144 32* C ** CONVERT NASTRAN MASS DENSITY TO DESAP WEIGHT DENSITY 000016
00147 33* WRITE (6,10) 000026
00151 34* 10 FORMAT(15X,'ENTER UNITS CODE: (ENGLISH=1,MKS=2)') 000033
00152 35* READ(5,7) IVCODE 000033
00155 36* CONFAC=1. 000041
00156 37* IF (IVCODE.EQ.2) CONFAC=9.8140 000043
00160 38* 7 FORMAT(1) 000050
00161 39* CALL ORDER 000050
00162 40* WRITE(6,670) 000052
00164 41* 600 FORMAT(15X,'ENTER MAX NO.OF REDESIGN CYCLES,NCYCL (C=INITIAL DESIG 000057
00164 42* IN ONLY)'/10X'AND MAX. NO. SCALINGS,NSCALE (DEFAULT=3), SEPARATE BY 000057
00164 43* 2 COMMA ONLY') 000057
00165 44* READ (5,7) NCYCL,NSCALE 000057
00171 45* IF (NSCALE.EQ.0) NSCALE=3 000066

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00173	46*		WRITE (6,601)	000072
00175	47*	601	FORMAT (15X,'ENTER SCALING CODE,KSCALE')	000077
00176	48*		READ(5,7) KSCALE	000077
00201	49*		WRITE (6,602)	000105
00203	50*	602	FORMAT (15X,'ENTER DELTA(DEFAULT=0.05),AND EPSILON (DEFAULT=0.1)')/	000112
00203	51*		115X'SEPARATE BY COMMA ONLY')	000112
00204	52*		READ (5,7) DELTA,EPSIL	000112
00210	53*		IF (DELTA.EQ. 0.) DELTA= .05	000121
00212	54*		IF (EPSIL.EQ. 0.) EPSIL = .1	000125
00214	55*		WRITE (6,603)	000131
00216	56*	603	FORMAT (15X,'ENTER CODEs FOR NODAL DISPL, PRINTOUT KPRINT (0=NO PR	000136
00216	57*		INT,1 = PRINT)'/10X'AND BUCKLING CONSTRAINT LBUCK. SEPARATE BY SPA	000136
00216	58*		2CE AND/OR COMMA')	000136
00217	59*		READ (5,7)KPRINT,LBUCK	000136
00223	60*		KPUNCH=0	000145
00224	61*		WRITE(12,611)	000146
00226	62*	611	FORMAT(' COMM DESIGN CONTROL FROM MAIN')	000153
00227	63*		WRITE (12,610)NCYCL,NSCALE,KSCALE,DELTA,EPSIL,KPUNCH,KPRINT,LBUCK	000153
00227	64*	C	PRINT 114	000153
00227	65*	C114	FORMAT(' IN MAIN--NCYCL,NSCALE,KSCALE,DELTA,EPSIL,KPUNCH,KPRINT,LB	000153
00227	66*	C	UCK')	000153
00227	67*	C	WRITE (6,610)NCYCL,NSCALE,KSCALE,DELTA,EPSIL,KPUNCH,KPRINT,LBUCK	000153
00241	68*	610	FORMAT (3I5,2F10.3,3I5)	000170
00242	69*		CALL NODPT	000170
00243	70*		NUM5=NUM(5)	000172
00244	71*		IF(NUM5.NE.0) PRINT 15	000174
00247	72*	15	FORMAT(' *****'	000202
00247	73*		1,' */' * DESAP BAR ELEMENT (NASTRAN ROD)',/,	000202
00247	74*		2' */' *****')	000202
00250	75*		IF(NUM5.NE.0)CALL BAR(CONFAC,NELTY1,IOV1,NIOV1)	000202
00252	76*		NELTYP=NELTYP,NELTY1	000212
00253	77*		IF(NUM5.EQ.0) GO TO 622	000215
00253	78*	C	PRINT 333, NIOV1,NELTY1,NELTYP	000215
00253	79*	C333	FORMAT(' NIOV1,NELTY1,NELTYP=',3(2X,I4))	000215
00253	80*	C	PRINT 666	000215
00253	81*	C666	FORMAT(' LOOKOUT HERE COMES THE IOV ARRAY')	000215
00255	82*		DO 821 L1=1,NIOV1	000224
00256	83*	821	IOV(L1)=IOV1(L1)	000224
00260	84*	C821	PRINT 444, IOV(L1)	000224
00260	85*	C444	FORMAT(')	000224
00262	86*	622	NIOVTO=NIOVTO+NIOV1	000227
00262	87*	C	PRINT 555, NIOVTO	000227
00262	88*	C555	FORMAT(' NIOVTO=',I4)	000227
00263	89*		NUM6=NUM(6)	000231
00263	90*	C	MIKE=0	000231
00263	91*	C	IF(MIKE.EQ.0)GO TO 2000	000231
00264	92*		IF(NUM6.NE.0) PRINT 20	000233
00267	93*	20	FORMAT(' *****'	000241
00267	94*		1,' */' * DESAP BEAM ELEMENT (NASTRAN BAR)',/,	000241
00267	95*		2' */' *****')	000241
00270	96*		IF(NUM6.NE.0)CALL BEAM(CONFAC,NELTY2,IOV2,NIOV2)	000241
00272	97*		NELTYP=NELTYP,NELTY2	000254
00273	98*		IF(NUM6.EQ.0) GO TO 624	000257
00273	99*	C	PRINT 332, NIOV2,NELTY2,NELTYP	000257
00273	100*	C332	FORMAT(' NIOV2,NELTY2,NELTYP=',3(2X,I4))	000257
00273	101*	C	PRINT 667	000257
00273	102*	C667	FORMAT(' LOOKOUT HERE COMES THE IOV ARRAY')	000257

G0275	103*		DO 623 L2=1,NIDV2	000270
00300	104*	623	IDV(L2+NIDVT0)=IDV2(L2)	000270
00300	105*	623	PRINT 444, IDV(L2+NIDVT0)	000270
00302	106*	624	NIDVT0=NIDVT0+NIDV2	000273
00302	107*	C	PRINT 555, NIDVT0	000273
00302	108*	C	MIKE=0	000273
00302	109*	C	IF(MIKE.EQ.0)GO TO 2000	000273
00303	110*		NUM7=NUM(7)	000275
00304	111*		IF(NUM7.NE.0) PRINT 25	000277
00307	112*	25	FORMAT(' *****'	000305
00307	113*		1,/' *'/ ' * DESAP QUAD ELEMENT (NASTRAN QDHEM)',/,	000305
00307	114*		2' *'/ ' *****'	000305
00310	115*		IF(NUM7.NE.0)CALL QDHEM(GOYFAC,NELTY3,IDV3,NIDV3)	000305
00312	116*		NELTYP=NELTYP+NELTY3	000320
00313	117*		IF(NUM7.EQ.0) GO TO 626	000323
00315	118*		DO 625 L3=1,NIDV3	000334
00320	119*	625	IDV(L3+NIDVT0)=IDV3(L3)	000334
00322	120*	626	NIDVT0=NIDVT0+NIDV3	000337
00323	121*		NUM8=NUM(8)	000341
00324	122*		IF(NUM8.NE.0) PRINT 30	000343
00327	123*	30	FORMAT(' *****'	000351
00327	124*		1,/' *'/ ' * DESAP SHEAR ELEMENT (NASTRAN SHEAR)',/,	000351
00327	125*		2' *'/ ' *****'	000351
00330	126*		IF(NUM8.NE.0)CALL SHEAR (CONFAC,NELTY4,IDV4,NIDV4)	000351
00332	127*		NELTYP=NELTYP+NELTY4	000364
00333	128*		IF(NUM8.EQ.0) GO TO 627	000367
00335	129*		DO 628 L4=1,NIDV4	000400
00340	130*	628	IDV(L4+NIDVT0)=IDV4(L4)	000400
00342	131*	627	NIDVT0=NIDVT0+NIDV4	000403
00342	132*	C2000	CONTINUE	000403
00343	133*		NUM9=NUM(9)	000405
00344	134*		NUM10=NUM(10)	000407
00345	135*		NCTOT=NUM9+NUM10	000411
00346	136*		IF(NCTOT.NE.0) PRINT 35	000413
00351	137*	35	FORMAT(' *****'	000421
00351	138*		1,/' *'/ ' * DESAP QT ELEMENTS (NASTRAN QUAD2 & TRIA2)',/,	000421
00351	139*		2' *'/ ' *****'	000421
00352	140*		IF(NCTOT.NE.0)CALL QT(CONFAC,NELTY5,IDV5,NIDV5)	000421
00354	141*		NELTYP=NELTYP+NELTY5	000434
00355	142*		IF(NCTOT.EQ.0) GO TO 629	000437
00357	143*		DO 630 L5=1,NIDV5	000450
00362	144*	630	IDV(L5+NIDVT0)=IDV5(L5)	000450
00364	145*	629	CONTINUE	000453
00365	146*	2000	CONTINUE	000453
00366	147*		NIDVT0=NIDVT0+NIDV5	000453
00367	148*		CALL DIF(NIDVT0,IDV,NUMDV)	000455
00367	149*	C	PRINT 111	000455
00367	150*	C111	FORMAT(' IDV ARRAY AND NUMDV IN MAIN AFTER RETURN FROM DIF')	000455
00367	151*	C	PRINT 112, (IDV(I),I=1,NIDVT0)	000455
00367	152*	C112	FORMAT(2X,13)	000455
00367	153*	C	PRINT 113, NUMDV	000455
00367	154*	C113	FORMAT(' NUMDV =',13)	000455
00370	155*		PRINT 40	000462
00372	156*	40	FORMAT(///' END OF ELEMENT PROCESSING-- SET OTHER CONTROL VARIABLE	000466
00372	157*		15,')	000466
00373	158*		WRITE (6,700)	000466
00375	159*	700	FORMAT (20X,'STRUCTURAL LOAD MULTIPLIERS'/20X,'ENTER NO. OF LOAD	000473

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00375	160*		1CONDITIONS*)	000473
00376	161*		READ (5,7)NLOAD	000473
00401	162*		DO 750 L1=1,NLOAD	000501
00404	163*		DO 750 L2=1,4	000510
00407	164*	750	STR(L1,L2)=0.	000510
00412	165*		DO 751 L1=1,NLOAD	000520
00415	166*		WRITE (6,701)L1	000523
00420	167*	701	FORMAT (2X,'FOR LOAD CONDITION',I4,' ENTER STRUCTURAL LOAD MULTIPL	000531
00420	168*		IERS FOR ELEMENT LOADS A,B,C,D,IN THAT ORDER. DEFAULT TO ALL ZEROS	000531
00420	169*		2*)	000531
00421	170*		READ (5,7) DUMH(1)	000531
00424	171*		IF(DUMH(1).EQ.0) GO TO 751	000537
00426	172*		READ(5,7) (DUMH(L2),L2=2,4)	000541
00431	173*		DO 752 L2=1,4	000556
00434	174*	752	STR(L1,L2)=DUMH(L2)	000556
00436	175*	751	CONTINUE	000563
00436	176*	C	PRINT 115	000563
00436	177*	C115	FORMAT(' IN MAIN--STR ARRAY')	000563
00440	178*		WRITE(12,759)	000563
00442	179*	759	FORMAT(' COMM STRUCTURAL LOAD MULTIPLIERS FROM MAIN')	000573
00443	180*		DO 755 L1=1,NLOAD	000573
00446	181*	755	WRITE (12,702)(STR(L1,L2),L2=1,4)	000573
00446	182*	C755	WRITE (6,702)(STR(L1,L2),L2=1,4)	000573
00455	183*	702	FORMAT (4F10.3)	000607
00455	184*	C **	BUCKLING CONTROL CARD	000607
00456	185*		IF(LBUCK.EQ.0) GO TO 850	000607
00460	186*		WRITE (6,703)	000611
00462	187*	703	FORMAT (10X,'BUCKLING CONTROL CARD')	000616
00463	188*		WRITE (6,704)	000616
00465	189*	704	FORMAT (2X,'ENTER COEFFT. DEFAULT=1.')	000623
00466	190*		BUCK2(1)=1.	000623
00467	191*		READ (5,7) DUM	000625
00472	192*		IF (DUM.NE.0)BUCK2(1)=DUM	000633
00474	193*		WRITE (6,705)	000637
00476	194*	705	FORMAT (2X,'MODE IN HAS BEEN SET TO 0'/)	000644
00477	195*		BUCK1(1)=0	000644
00500	196*		WRITE (6,706)	000645
00502	197*	706	FORMAT (2X,'ENTER NMODE . DEFAULT=1')	000652
00503	198*		BUCK1(2)=1	000652
00504	199*		READ (5,7)DUM	000654
00507	200*		IF (DUM.NE.0)BUCK1(2)=DUM	000662
00511	201*		WRITE (6,707)	000673
00513	202*	707	FORMAT (2X,'ENTER INDET AND NVEC,IN THAT ORDER')	000700
00514	203*		READ (5,7) BUCK1(3),BUCK1(4)	000700
00520	204*		WRITE (6,708)	000707
00522	205*	708	FORMAT (2X,'ENTER ALPA')	000714
00523	206*		READ (5,7) BUCK2(2)	000714
00526	207*		WRITE (6,709)	000722
00530	208*	709	FORMAT (2X,'ENTER OMEGA . DEFAULT = 0.8')	000727
00531	209*		BUCK2(3)=.8	000727
00532	210*		READ (5,7) DUM	000731
00535	211*		IF (DUM.NE.0) BUCK2(3)=DUM	000737
00537	212*		WRITE (6,710)	000743
00541	213*	710	FORMAT (2X,'ENTER SHIFT. DEFAULT=0.')	000750
00542	214*		READ (5,7) BUCK2(4)	000750
00542	215*	C	PRINT 116	000750
00542	216*	C116	FORMAT(' IN MAIN--BUCK2(1),BUCK1 ARRAY,BUCK2 ARRAY')	000750

00545	217*		WRITE(12,713)	000756
00547	218*	713	FORMAT(' COMM BUCKLING CONTROL FROM MAIN')	000763
00550	219*		WRITE (12,711)BUCK2(1),(BUCK1(I),I=1,4),(BUCK2(I),I=2,4)	000763
00550	220*	C	WRITE (6,711)BUCK2(1),(BUCK1(I),I=1,4),(BUCK2(I),I=2,4)	000763
00555	221*	711	FORMAT (F10.3,4F10.3)	001000
00556	222*	850	CONTINUE	001000
00556	223*	C **	NODAL LOAD DATA	001000
00557	224*		IF(NUM(3).EQ.0) GO TO 950	001000
00561	225*		REWIND 11	001001
00562	226*		NEND=NSTART(3)-1	001004
00563	227*		DO 925 K1=1,NEND	001007
00566	228*	925	READ(11,1)DUM	001012
00572	229*	901	FORMAT (/)	001020
00573	230*		LIM=NUM(3)	001020
00574	231*		DO 930 K2=1,LIM	001022
00577	232*	930	READ(11,902) R1(K2,1),(R2(K2,K3),K3=1,3)	001030
00607	233*	902	FORMAT (16X,18,16X,3F8.1)	001051
00607	234*	C **	SET ALL LOAD CONDITION NUMBERS=1. THIS MAY BE MODIFIED BY HAVING THE USER	001051
00607	235*	C **	ENTER THE LOAD CONDITION FOR EACH NODAL LOAD CARD	001051
00610	236*		DO 935 K4=1,LIM	001051
00613	237*	935	R1(K4,2)=1	001051
00613	238*	C **	SET ALL CONCENTRATED NODAL MOMENTS TO ZERO. THIS MAY BE MODIFIED BY HAVING	001051
00613	239*	C **	THE USER ENTER THESE VALUES FOR EACH NODAL LOAD CARD	001051
00615	240*		DO 936 K5=1,LIM	001057
00620	241*		DO 936 K6=4,6	001057
00623	242*	936	K2(K5,K6)=0.	001057
00626	243*		CALL ELOAD(LIM,R1,R2,2,6)	001064
00626	244*	C	PRINT 117	001064
00626	245*	C117	FORMAT(' IN MAIN--R1 AND R2 ARRAY')	001064
00627	246*		WRITE(12,913)	001073
00631	247*		DO 940 K7=1,LIM	001100
00634	248*	913	FORMAT(' COMM NODAL LOAD DATA FROM MAIN')	001106
00635	249*	940	WRITE (12,903) (R1(K7,K8),K8=1,2),(R2(K7,K8),K8=1,6)	001106
00635	250*	C940	WRITE (6,903) (R1(K7,K8),K8=1,2),(R2(K7,K8),K8=1,6)	001106
00654	251*	903	FORMAT (14,13,6F12.5)	001132
00651	252*	950	WRITE (12,904)	001132
00653	253*	904	FORMAT(' ')	001136
00653	254*	C **	DESIGN VARIABLE DATA	001136
00654	255*		WRITE (6,1000)	001136
00656	256*	1000	FORMAT (15X,'DESIGN VARIABLE DATA')	001143
00657	257*		DO 1051 K1=1,NUMDV	001143
00657	258*	C	A(K1,1)=1.	001143
00657	259*	C1050	A(K1,2)=1	001143
00662	260*		WRITE (6,1001)K1	001151
00663	261*	1050	READ (5,7) A(K1,1),A(K1,2)	001157
00672	262*	1001	FORMAT (2X,'FOR DESIGN VARIABLE NO.',13,' ENTER AOLD AND AMIN, IN T	001174
00672	263*		HAT ORDER')	001174
00672	264*	C	PRINT 118	001174
00672	265*	C118	FORMAT(' IN MAIN--K1 AND K ARRAY--AOLD AND AMIN')	001174
00673	266*		DO 1051 K1=1,NUMDV	001174
00676	267*	1051	WRITE (12,1002) K1,(A(K1,K2),K2=1,2)	001174
00676	268*	C1051	WRITE (6,1002) K1,(A(K1,K2),K2=1,2)	001174
00706	269*	1002	FORMAT(15,2F10.3)	001213
00707	270*		NUMNF=NUM(1)	001213
00710	271*		ENDFILE 12	001215
00711	272*		REWIND 12	001220
00712	273*		READ(12,1) HEAD	001223

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00715	274*		WRITE(13,1) HEAD	001233
00720	275*		WRITE (13,1003) NUMNP,NELTYP,NLOAD,NUMDV	001243
00720	276*	C	PRINT 1004	001243
00720	277*	C1004	FORMAT(' NUMNP--NELTYP--NLOAD--NUMDV NEXT PRINT')	001243
00720	278*	C	WRITE (6,1003) NUMNP,NELTYP,NLOAD,NUMDV	001243
00726	279*	1005	READ(12,1,END=2) HEAD	001255
00731	280*		IF(HEAD(1).EQ.0UHM)GO TO 1005	001265
00733	281*		WRITE(13,1) HEAD	001270
00736	282*		GO TO 1005	001300
00737	283*	1003	FORMAT (4I5)	001302
00740	284*	2	STOP	001302
00741	285*		END	001305

0HDG,P NAME

0PRT,S NAME

FURPUR 28R1H1 E36 S74T11 07/30/80 09:45:45

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@FOR,S NOUPT,NOUPT
HSA E3 -07/30/80-09:45:46 (27,)

SUBROUTINE NOUPT ENTRY POINT 000457

STORAGE USED: CODE(1) 000472; DATA(6) 015130; BLANK COMMON(2) 000000

COMMON BLOCKS:

0003 BACH 000040

EXTERNAL REFERENCES (BLOCK, NAME)

0004 NREWS
0005 NRDU\$
0006 NIO1\$
0007 NIO2\$
0010 NIO3\$
0011 NWDU\$
0012 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000017	1166	0001	000025	1236	0001	000031	1276	0001	000051	1376	0001	000053	1426
0001	000075	1556	0001	000102	1753	0001	000134	1776	0001	000223	1853L	0001	000147	2076
0001	000154	2136	0001	000171	2236	0001	000174	2266	0001	000216	2416	0001	000240	2566
0001	000242	2616	0001	000254	2676	0001	000312	3146	0001	000331	3246	0001	000343	3356
0001	000351	3406	0001	000353	3436	0001	000365	3516	0001	000410	3676	0001	000415	3736
0001	000421	3776	0000	015062	5F	0001	000370	61L	0001	000360	62L	0000	015051	700F
0000	015055	701F	0000	015056	702F	0001	000057	751L	0001	000105	752L	0001	000107	753L
0001	000115	765L	0001	000200	851L	0001	000226	852L	0001	000230	853L	0001	000234	864L
0001	000265	881L	0001	000247	882L	0000	015073	901F	0000	015064	903F	0000	015061	910F
0001	000273	950L	0001	000403	959L	0000	I 000000	CR00	0000	R 015001	00M	0000	I 015030	I
0000	I 015031	I1	0000	015110	INJP\$	0000	I 015032	J	0000	I 015036	JJ	0000	I 015033	JK
0000	I 015027	K	0000	I 015041	KN	0000	I 015035	L	0000	I 015045	LI	0000	I 015047	L2
0000	I 015050	L3	0000	I 015046	L4	0000	I 015040	M	0000	I 015037	M1	0000	I 015044	NCROD
0000	I 015043	NEND	0000	I 015034	NEND3	0000	I 005524	NODPT1	0003	I 000022	NSTART	0003	I 000000	NUM
0000	I 015025	NUMNP	0000	I 015026	NUM4	0000	I 000310	SHAR1	0000	R 011610	SHAR2	0000	I 013414	SPC1
0000	R 015042	I	0000	I 014772	V									

00101	1*		SUBROUTINE NOUPT											000000
00102	2*		COMMON/BACH/NUM(18),NSTART(18)											000000
00103	3*	C **	CHANGE GROUP, NO. OF DESAP BAR ELS.(NASTRAN ROD)											000000
00104	4*		INTEGER CROD(1,0,2)											000000
00104	5*	C **	CHANGE GROUP, NO. OF NODAL POINTS											000000
00105	6*		INTEGER SHAR1(300,4)											000000
00106	7*		DIMENSION NOUPT1(300,7),SHAR2(300,3)											000000
00106	8*	C **	CHANGE GROUP, NO. OF SPC1 CARDS											000000
00107	9*		INTEGER SPC1(50,15)											000000
00107	10*	C **	*****											000000

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CO110	11*		INTEGER V(7)	000000
CO111	12*		DIMENSION DUM(20)	000000
CO112	13*		REWIND 11	000000
CO113	14*		NUMNP=NUM(1)	000002
CO114	15*		NUM4=NUM(4)	000005
CO114	16*	C **	CREATES NODAL POINT DATA	000005
CO114	17*	C **	WHEN NUM(4)=0 (NO SPC1 CARDS), READS GRID CARDS FROM FILE 11, STORES IN	000005
CO114	18*	C **	ARRAYS SHAR1 AND SHAR2 AND WRITES NODAL PT. DATA CARDS ON FILE 12	000005
CO114	19*	C **	SHAR1 CONTAINS NODE NO. AND NASTRAN CONSTRAINTS	000005
CO114	20*	C **	SHAR2 CONTAINS CORRESPONDING COORDINATES (SAME ROW NO.)	000005
CO115	21*		DO 760 K=1, NUMNP	000007
CO120	22*		READ (11,700) SHAR1(K,1), (SHAR2(K,I), I=1,3), (SHAR1(K,I), I=2,9)	000017
CO120	23*	C	PRINT 700, SHAR1(K,1), (SHAR2(K,I), I=1,3), (SHAR1(K,I), I=2,9)	000017
CO133	24*	760	NODPT1(K,1)=SHAR1(K,1)	000034
CO135	25*	700	FORMAT(8X,18,8X,3F8.4,8X,811)	000053
CO135	26*	C **	NODE PT. NO. IS NOW NODPT1(K,1)	000053
CO135	27*	C **	NODPT1 CONTAINS DESAP NODE NO. AND 6 MOTION CODES	000053
CO135	28*	C **	INITIALIZE DESAP CONSTRAINT CODES TO ZERO	000053
CO136	29*		DO 765 K=1, NUMNP	000053
CO141	30*		DO 766 I = 2,7	000053
CO144	31*	766	NODPT1(K,I)=0	000053
CO144	32*	C **	II COUNTS THE 8 COLUMNS OF SHAR1 (K,2 TO 9)	000053
CO146	33*		II=1	000054
CO147	34*	751	II=II+1	000057
CO150	35*		IF (II.GT.9) GO TO 765	000061
CO150	36*	C **	CHECK IF COLUMN IS BLANK	000061
CO152	37*		IF (SHAR1(K,II).EQ.0) GO TO 751	000064
CO152	38*	C **	J IS THE NASTRAN RESTRAINT CODE. J=1 MEANS UX=0, J=6 MEANS OX=0	000064
CO154	39*		DO 752 J=1,6	000075
CO157	40*		IF (SHAR1(K,II).EQ.J) GO TO 1753	000075
CO161	41*		GO TO 752	000100
CO162	42*	1753	JK=J	000102
CO163	43*		GO TO 753	000103
CO164	44*	752	CONTINUE	000107
CO166	45*	753	NODPT1(K,JK+1)=1	000107
CO167	46*		GO TO 751	000113
CO170	47*	765	CONTINUE	000121
CO170	48*	C	PRINT 1800	000121
CO170	49*	C1800	FORMAT(1X,'NODPT1 JUST BEFORE SPC1 CARDS')	000121
CO170	50*	C	DO 1801 K=1, NUMNP	000121
CO170	51*	C1801	PRINT 910, (NODPT1(K,I), I=1,7)	000121
CO170	52*	C **	IF THERE ARE SPC1 CARDS, MODIFY NODPT1 ARRAY	000121
CO172	53*		IF (NUM(4).EQ.0) GO TO 950	000121
CO172	54*	C **	READ SPC1 CARDS FIELDS 2 THROUGH 9 AND STORE IN SPC1	000121
CO172	55*	C **	SPC1(K,2) THROUGH SPC1(K,9) CONTAIN CONSTRAINT CODES 1 THROUGH 6	000121
CO174	56*		REWIND 11	000123
CO175	57*		NEND3 = NSTART (4)-1	000126
CO176	58*		DO 770 L=1, NEND3	000131
CO201	59*	770	READ (11,701) DUM	000134
CO205	60*	701	FORMAT (20A4)	000147
CO206	61*		DO 771 L=1, NUM4	000147
CO211	62*	771	READ (11,702) (SPC1(L,J), J=1,15)	000147
CO220	63*	702	FORMAT(8X,18,811,618)	000163
CO221	64*		REWIND 11	000163
CO221	65*	C **	TRANSFER CONSTRAINT CODES FROM SPC1 ARRAY TO NODPT1 ARRAY	000163
CO221	66*	C **	V IS STORAGE VECTOR	000163
CO221	67*	C **	K INDEXES SPC1 ROW(SPC1 CARD)	000163

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00222	68*		00 865 K=1,NUM4	000174
00225	69*		00 860 L=1,7	000174
00230	70*	860	V(I)=0	000174
00232	71*		II=1	000175
00233	72*	851	II=II+1	000200
00234	73*		IF (II.GT.9) GO TO 864	000202
00234	74*	C **	CHECK IF COLUMN IS BLANK	000202
00236	75*		IF (SPC1(K,II).EQ.0)GO TO 851	000205
00236	76*	C **	J IS THE NASTRAN RESTRAINT CODE.ETC **	000205
00240	77*		00 852 J=1,6	000216
00243	78*		IF(SPC1(K,II).EQ.J)GO TO 1853	000216
00245	79*		GO TO 852	000221
00246	80*	1853	JJ=J	000223
00247	81*		GO TO 853	000224
00250	82*	852	CONTINUE	000230
00252	83*	853	V(JJ+1)=1	000230
00253	84*		GO TO 851	000232
00253	85*	C **	KTH SPC1 IS DONE. V CONTAINS 6 DESAP CONSTRAINT CODES FOR THAT K	000232
00254	86*	864	CONTINUE	000234
00254	87*	C **	PLACE NEW CONSTRAINT CODES IN NODPT1	000234
00254	88*	C	PRINT 908, K,(V(I),I=1,7)	000234
00254	89*	C908	FORMAT(IX,'V FOR SPC1',I3/SX,7I5)	000234
00255	90*		DO 880 M=10,15	000234
00260	91*		DO 881 M=1,NUMNP	000242
00263	92*		IF (NODPT1(M,1).EQ. SPC1 (K,M)) GO TO 882	000242
00265	93*		GO TO 881	000245
00265	94*	C **	NODAL PT. NO. IN NODPT1 (M,1) IS REFERENCED ON SPC1 CARD INDEXED BY K	000245
00265	95*	C **	PLACE NEW CONSTRAINTS ON THIS NODAL PT.	000245
00266	96*	882	00 883 L=2,7	000247
00271	97*		IF(V(L).NE.0.) NODPT1 (M,L)=V(L)	000254
00273	98*	883	CONTINUE	000273
00273	99*	C **	THE MTH NODPT1 CARD IS NOW COMPLETE	000273
00273	100*	C	PRINT 909,M	000273
00273	101*	C909	FORMAT(IX,'NODPT1 CARD',I3/)	000273
00273	102*	C	PRINT 910,(NODPT1(M,L),L=1,7)	000273
00275	103*	910	FORMAT(7I5)	000273
00276	104*	881	CONTINUE	000273
00300	105*	880	CONTINUE	000273
00300	106*	C **	ALL NODPT1 CARDS ARE NOW COMPLETE	000273
00302	107*	865	CONTINUE	000273
00304	108*	950	CONTINUE	000273
00305	109*		KN=0	000273
00306	110*		I=0	000273
00306	111*	C **	MODIFY MOTION CODES IF THERE ARE ANY NASTRAN ROUS	000273
00306	112*	C **	BRING CRODS FROM FILE 10 AND GET THE GRID PT NO'S.	000273
00306	113*	C **	FIND CORRESPONDING NODE PT NO'S IN NODPT1 AND ENFORCE ZERO ROTATIONS	000273
00307	114*		REWIND 11	000274
00310	115*		IF (NUM15).EQ. 0,GO TO 959	000277
00312	116*		NEND=NSTART(5)-1	000301
00313	117*		DO 50 L=1,NEND	000306
00316	118*	50	READ (11,70)IDUM	000312
00322	119*		NCROD=NUM(5)	000322
00323	120*		DO 51 L=1,NCROD	000324
00326	121*	51	READ (11,5) CROD(L,1),CROD(L,2)	000331
00333	122*	5	FORMAT (24X,2I8)	000343
00334	123*		DO 60 L1=1,NCROD	000343
00337	124*		DO 64 L4=1,2	000353

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00342 125*      00 61 L2=1,NUMNP                      000353
00345 126*      IF(NODPT1(L2,1).EQ.CROD (L1,L4)) GO TO 62 000353
00347 127*      GO TO 61                               000356
00350 128*      62 00 63 L3=5,7                       000360
00353 129*      63 NODPT1(L2,L3)=1                   000365
00355 130*      61 CONTINUE                            000375
00357 131*      64 CONTINUE                            000375
00361 132*      60 CONTINUE                            000375
00361 133*      C PRINT 200                            000375
00361 134*      C200 FORMAT(' FINAL OUTPUT OF NODPT.')
```

HDG,P ORDER

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@FOR,S ORDER,ORDER
HSA E3 -07/30/80-09:45:50 (29,)

SUBROUTINE ORDER ENTRY POINT 000552

STORAGE USED: CODE(1) 000561; DATA(0) 000570; BLANK COMMON(2) 000000

COMMON BLOCKS:

0003 BACH 000044
0004 MOZART 000334
0005 HANDEL 000174

EXTERNAL REFERENCES (BLOCK, NAME)

0006 VEC
0007 COUNT
0010 SIGMA
0011 IU
0012 NRDU\$
0013 NIO3\$
0014 NIO2\$
0015 NWDU\$
0016 NFR1\$
0017 NREW\$
0020 NWEF\$
0021 NIO1\$
0022 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000002	1176	0001	000062	1506	0001	000075	1576	0001	000141	2046	0001	000177	2226
0001	000235	2406	0001	000270	2586	0001	000327	2746	0001	000336	3016	0001	000350	3106
0001	000400	3276	0001	000417	3406	0001	000440	3476	0001	000460	3576	0001	000476	3716
0000	000500	402F	0000	000501	402F	0000	000503	405F	0001	000002	5L	0001	000042	501L
0001	000115	502L	0001	000111	503L	0001	000162	507L	0001	000262	550L	0001	000314	552L
0001	000017	6L	0000	000517	604F	0001	000525	655L	0001	000514	657L	0000	000523	658F
0000	000505	749F	0000	000515	750F	0000	R 000000	A	0000	R 000024	B	0000	R 000046	DUM
0000	R 000072	GRUMAX	0000	I 000464	I	0000	I 000455	ID1	0000	I 000456	ID2	0000	000555	INJPS
0000	I 000475	J	0000	I 000453	K	0000	I 000476	KH0AD	0000	I 000477	KM	0000	I 000472	KOUNT
0005	I 000000	KVCMID	0005	I 000024	KVCPID	0000	I 000470	L	0000	I 000473	LL	0000	I 000474	LLL
0004	I 000000	MAT11	0004	R 000024	MAT12	0005	I 000170	MAXPID	0005	I 000171	MAXPID	0000	I 000466	MID
0000	I 000462	NEND	0000	I 000467	NEND17	0000	I 000465	NPCARD	0000	I 000457	NREC	0000	I 000461	NRECH1
0003	I 000022	NSTART	0003	I 000000	NUM	0005	I 000172	NUMMID	0000	I 000454	NUMNP	0005	I 000173	NUMPID
0000	I 000463	NUMRE	0000	I 000460	NUMREC	0000	I 000471	NUM18	0000	I 000426	PID	0000	I 000427	XMAT11
0000	R 000116	XMAT12												

00101	1*	SUBROUTINE ORDER	000002
00103	2*	COMMON/BACH/NUM(18),NSTART(18)/MOZART/MAT11(20),MAT12(20,10)/HANDE	000002
00103	3*	1,/KVCMID(20),KVCPID(10),MAXMID,MAXPID,NUMMID,NUMPID	000002

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00104	4*		DIMENSION A(20),B(18),DUM(20),GRDMAX(20)	000002
00105	5*		DATA NUM/18*0/8/'GRID','OMIT','FORC','SPC1','CR0D','GBAR','CQDM',	000002
00105	6*		1'CSHE','CQUA','CTRI','CONM','PROD','PBAR','PQDM','PSHE','PQUA',	000002
00105	7*		2'PTRI','MAT1'/KVCHID/20*0/KVCPID/100*0/MAXHID,MAXPID/20,100/	000002
00114	8*		REAL MAT12,XMAT12(20,10)	000002
00115	9*		INTEGER PID,XMAT11(20)	000002
00115	10*	C **	WRITE ON FILE 10 IN THIS ORDER: GRID,OMIT,FORC,...MAT1	000002
00115	11*	C **	CALCULATE NO. OF EACH TYPE.E.G.,NUM(1)=NO. OF GRID CARDS,NUM(18)=NO. OF	000002
00115	12*	C **	MAT1 CARDS	000002
00115	13*	C **		000002
00115	14*	C **	MAXHID(MAXPID) IS THE MAX NO. OF DIFFERENT MID(PID)NOS.	000002
00115	15*	C **	THE PROGRAM CAN HANDLE. IF IT IS INCREASED,ALSO	000002
00115	16*	C **	INCREASE DIMENSIONS OF KVCHID(KVCPID) ACCORDINGLY.	000002
00116	17*		DO 500 K=1,18	000002
00121	18*	5	READ (9,400,END=501) A	000002
00124	19*		IF (A(1).EQ. B(K)) GO TO 6	000012
00126	20*		GO TO 5	000015
00127	21*	6	WRITE (10,400)A	000017
00132	22*		PRINT #02, A	000026
00135	23*		NUM(K)=NUM(K)+1	000035
00136	24*		GO TO 5	000040
00137	25*	501	REWIND 9	000042
00140	26*	500	CONTINUE	000046
00140	27*	C	PRINT 801	000046
00140	28*	C 801	FORMAT(' NUM(1), THRU NUM(18)')/)	000046
00140	29*	C	PRINT 22, (NUM(K),K=1,18)	000046
00140	30*	C22	FORMAT(18I4)	000046
00142	31*	400	FORMAT (20A4)	000046
00143	32*	402	FORMAT(1X,19A4)	000046
00144	33*		ENDFILE 10	000046
00144	34*	C **	NUMBER OF NODAL POINTS	000046
00145	35*		NUMNP=NUM(1)	000051
00145	36*	C **	CALCULATE STARTING RECORD NUMBERS IN FILE 10	000051
00146	37*		NSTART(1)=1	000053
00147	38*		DO 20 K=2,18	000062
00152	39*	20	NSTART(K)=NSTART(K-1)+NUM(K-1)	000062
00154	40*		REWIND 10	000065
00154	41*	C **	CALCULATE LARGEST NODAL POINT NUMBER AND CARD ON WHICH IT OCCURS NREC **	000065
00155	42*		ID1=1	000070
00156	43*		DO 502 K=1,NUMNP	000075
00161	44*		READ (10,405)ID2	000075
00164	45*	405	FORMAT (8X,I6)	000103
00165	46*		IF (ID2 .GT. ID1) GO TO 503	000103
00167	47*		GO TO 502	000107
00170	48*	503	ID1=ID2	000111
00171	49*		NREC=K	000112
00172	50*	502	CONTINUE	000116
00174	51*		NUMREC=NSTART(18)+NUM(18)-1	000116
00174	52*	C	IF(NUM(18).EQ.1) NUMREC=NSTART(18)	000116
00174	53*	C	PRINT 23, NUMREC	000116
00174	54*	C23	FORMAT(0, NUMREC =,I4)	000116
00175	55*		IF(NREC .EQ. NUMNP) GO TO 550	000122
00175	56*	C **	PLACE RECORD WITH LARGEST NODAL POINT NUMBER LAST ON FILE 11	000122
00177	57*		REWIND 10	000125
00200	58*		IF(NREC.EQ.1) GO TO 507	000130
00200	59*	C **	READ FROM 10 ALL RECORDS BEFORE THE ONE WITH LARGEST NODAL PT. NO. AND	000130
00200	60*	C **	WRITE THEM ON 11	000130

00202	61*		NRECH1=NREC-1	000133
00203	62*		DO 505 K=1,NRECH1	000136
00206	63*		READ (10,400)DUM	000141
00211	64*	505	WRITE (11,400)DUM	000150
00211	65*	C505	PRINT 402, DUM	000150
00211	66*	C **	READ RECORD WITH LARGEST NODAL PT. NO. (GRID PT. NO.)	000150
00215	67*	507	READ (10,400)GRDMAX	000162
00215	68*	C **	READ REMAINDER OF GRID CARDS AND WRITE ON 11	000162
00220	69*		NEND=NUMNP-NREC	000171
00221	70*		DO 506 K=1,NEND	000174
00224	71*		READ(10,400)DUM	000177
00227	72*	506	WRITE(11,400)DUM	000206
00227	73*	C506	PRINT 402, DUM	000206
00227	74*	C **	WRITE ON 11 RECORD WITH LARGEST NODAL PT. NO.	000206
00233	75*		WRITE (11,400)GRDMAX	000217
00233	76*	C	PRINT 402,GRDMAX	000217
00233	77*	C	PRINT 399	000217
00233	78*	C399	FORMAT(5X,'FINAL WRITE ON 11')	000217
00236	79*		NUMRE=NUMREC-NUMNP	000227
00236	80*	C	PRINT 24, NUMRE,NUMNP	000227
00236	81*	C24	FORMAT(1X,NUMRE AND NUMNP =',2I0)	000227
00237	82*		DO 508 K=1,NUMRE	000232
00242	83*		READ (10,400)DUM	000235
00245	84*	508	WRITE (11,400)DUM	000244
00245	85*	C508	PRINT 402, DUM	000244
00251	86*		ENDFILE 11	000255
00252	87*		GO TO 552	000260
00253	88*	550	CONTINUE	000262
00253	89*	C **	SINCE FILE IS ALREADY IN CORRECT ORDER JUST WRITE TO FILE 11	000262
00254	90*		REWIND 10	000262
00255	91*		DO 551 I=1,NUMREC	000264
00260	92*		READ(10,400) DUM	000270
00263	93*	551	WRITE (11,400) DUM	000277
00263	94*	C551	PRINT 402, DUM	000277
00267	95*		ENDFILE 11	000310
00267	96*	C **	CREATE MATERIAL VECTOR MAT11 (NUM18)AND MATERIAL	000310
00267	97*	C **	MATRIX MAT12(NUM18,10) THE CONTENT OF MAT11(K) IS THE MID OF MAT12(K,J)	000310
00270	98*	552	REWIND 11	000314
00270	99*	C **	IT IS NECESSARY TO RENUMBER MID'S AND PID'S SINCE CAUDS	000314
00270	100*	C **	MAY GENERATE NOS. THAT ARE TOO LARGE.	000314
00270	101*	C **	SET UP KVCHID AND KVCPID	000314
00270	102*	C		000314
00271	103*		NEND=START(12,-1)	000316
00272	104*		NPCARD=0	000321
00273	105*		DO 700 I=12,17	000327
00276	106*	700	NPCARD=NPCARD+NUM(I)	000327
00300	107*		DO 733 I=1,NEND	000332
00303	108*	733	READ(11,400)DUM	000336
00307	109*		DO 710 I=1,NPCARD	000350
00312	110*		READ(11,750) PID,MID	000350
00312	111*	C	PRINT 749,PID,MID	000350
00316	112*	749	FORMAT(1X,PID AND MID READ FROM 11 IN ORDER',2X,2I8)	000356
00317	113*	750	FORMAT(8X,2I8)	000356
00317	114*	C	PRINT 911	000356
00317	115*	C911	FORMAT(' IN ORDER GOING TO VEC')	000356
00320	116*		CALL VEC(MID,PID)	000356
00320	117*	C	PRINT 912	000356

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00320	118*	C912	FORMAT(' IN ORDER JUST BACK FROM VEC')	000356
00321	119*	710	CONTINUE	000363
00321	120*	C **	THE NEXT NINE LINES ARE DEBUGS, TO BE COMMENTED OUT LATER.	000363
00321	121*	C	PRINT 751	000363
00321	122*	C	DO 760 I=1,MAXMID	000363
00321	123*	C760	PRINT 761,KVCMID(I)	000363
00321	124*	C	PRINT 752	000363
00321	125*	C	DO 770 I=1,MAXPID	000363
00321	126*	C770	PRINT 761,KVCPID(I)	000363
00321	127*	C761	FORMAT(I8)	000363
00321	128*	C751	FORMAT(' KVCMID IN ORDER')	000363
00321	129*	C752	FORMAT(' KVCPID IN ORDER')	000363
00323	130*	C	CALL COUNT(NUMMID,NUMPID)	000363
00323	131*	C	PRINT 771,NUMMID,NUMPID	000363
00323	132*	C771	FORMAT(' NUMMID AND NUMPID IN ORDER',2I8)	000363
00324	133*		REWIND 11	000367
00325	134*		NEND17=NSTART(18)-1	000372
00326	135*		DO 654 L=1,NEND17	000375
00331	136*	654	READ (11,400)NUM	000400
00335	137*		NUM18=NUM(18)	000410
00335	138*	C **	CHANGE MID IN MAT11	000410
00335	139*	C **	MID AND PID IN C AND P CARDS WILL BE CHANGED IN CANDP	000410
00336	140*		KOUNT=0	000412
00337	141*		DO 655 LL=1,NUM18	000417
00342	142*		L=LL-KOUNT	000417
00343	143*		LLL=L	000424
00344	144*		READ (11,604) XMAT11(L),XMAT12(L,J),J=1,7	000426
00353	145*	604	FORMAT(8X,I8,2E8.3,5F8.4)	000443
00354	146*		CALL SIGMA(XMAT11(L),LLL,XMAT12)	000443
00355	147*		KHOAD=0	000452
00356	148*		DO 800 I=1,NUMMID	000460
00361	149*		IF(XMAT11(L).EQ.KVCMID(I)) KHOAD=1	000460
00363	150*	800	CONTINUE	000466
00365	151*		IF(KHOAD.EQ.0) GO TO 657	000466
00367	152*		MAT11(L)=XMAT11(L)	000470
00370	153*		DO 805 J=1,10	000476
00373	154*	805	MAT12(L,J)=XMAT12(L,J)	000476
00373	155*	C	PRINT 625,L,MAT11(L)	000476
00373	156*	C625	FORMAT(' BEFORE CHANGE,MAT11(' ,I2,')=',I3)	000476
00375	157*		CALL ID(1,NUMMID,MAT11(L),KM)	000500
00376	158*		MAT11(L)=KM	000510
00376	159*	C	PRINT 627,L,MAT11(L)	000510
00376	160*	C627	FORMAT(' AFTER CHANGE,MAT11(' ,I2,')=',I3)	000510
00377	161*		GO TO 655	000512
00400	162*	657	PRINT 658,XMAT11(L)	000514
00403	163*		KOUNT=KOUNT+1	000521
00404	164*	658	FORMAT(' AN MID NOT REFERENCED BY ANY PID:',I4,'. THE CORRESPONDING	000526
00404	165*		ENG MAT1 CARD HAS NOT BEEN PUT IN MAT11 OR MAT12')	000526
00405	166*	655	CONTINUE	000526
00405	167*	C	PRINT 42J	000526
00405	168*	C420	FORMAT(' MAT11 AND MAT12 IN ORDER')	000526
00407	169*		NUM(18)=NUM(18)-KOUNT	000526
00410	170*		NUM18=NUM(18)	000531
00410	171*	C	DO 660 M1=1,NUM18	000531
00410	172*	C660	PRINT 614,MAT11(M1),(MAT12(M1,J),J=1,10)	000531
00410	173*	C614	FORMAT(/15,5E11.5/3X,5E11.5)	000531
00411	174*		REWIND 11	000532

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00412 175*

RETURN

000535

00413 176*

END

000560

END FOR

•HUG,P

PAIR

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FOR,S PAIR,PAIR
HSA E3 -07/30/80-09:46:12 (55,)

SUBROUTINE PAIR ENTRY POINT 000251

STORAGE USED: CODE(1) 000307; DATA(U) 000230; BLANK COMMON(2) 000000

COMMON BLOCKS:

0003 MOZART 000334
0004 BACH 000044

EXTERNAL REFERENCES (BLOCK, NAME)

0005 NERR2\$
0006 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000	000160	IF	0001	000202	100L	0001	000052	112G	0001	000053	115G	0001	000064	123G
0001	000077	133G	0001	000114	142G	0001	000120	147G	0001	000123	152G	0001	000142	167G
0001	000206	200L	0001	000167	201G	0001	000172	204G	0001	000132	78L	0001	000127	79L
0001	000157	20L	0001	000216	82L	0001	000212	83L	0000	000164	INJP	0000	I 000151	J2
0000	I 000152	J3	0000	I 000153	J4	0000	I 000155	J5	0000	I 000154	J6	0000	I 000156	J7
0000	I 000157	J8	0000	I 000146	K	0000	I 000145	KC	0000	I 000144	KR	0003	I 000000	MAT11
0003	R 000024	MAT12	0000	I 000150	MAXID	0004	000022	NSTART	0004	I 000000	NUM	0000	I 000147	NUM18
0000	I 000000	VID												

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00101	1*		SUBROUTINE PAIR (KODE,CID,PRID,ISTOP,NCARDS,KOL,NOUT,MATCH,INDEX)										000053
00101	2*	C **	KODE INPUT, 0 FOR MID PAIRING, 1 FOR PID PAIRING										000053
00101	3*	C **	CID. INPUT, ELEMENT ID ARRAY, E.G.,CRODID										000053
00101	4*	C **	PRID. INPUT, PROPERTY ID ARRAY, E.G.,PROD1										000053
00101	5*	C **	ISTOP. INPUT, NO. OF ELS. WITH GIVEN CONST. CODE (=NPAR(2))										000053
00101	6*	C **	NCARDS. INPUT, NUM18 IF KODE=0, IF KODE=1, TOTAL NO. OF P CARDS FOR GIVEN										000053
00101	7*	C **	ELEMENT, E.G., IF NASTRAN ROD,NCARDS=NUM12										000053
00101	8*	C **	KOL. INPUT, IF KODE=0, COL. NO. OF MID IN ELEMENT ID ARRAY.										000053
00101	9*	C **	E.G. IF GIVEN ELEMENT IS NASTRAN ROD,KOL=4, IF KODE=1,COR.										000053
00101	10*	C **	NO. OF PID IN ELEMENT ID ARRAY, E.G.,IF GIVEN EL. IS										000053
00101	11*	C **	NASTRAN ROD,KOL=5										000053
00101	12*	C **	NOUT. OUTPUT, IF KODE=0,NOUT=NUMMAT FOR GIVEN EL. AND CONST. CODE.										000053
00101	13*	C **	IF KODE=1,NOUT*NUMGEO FOR GIVEN EL. AND CONST. CODE.										000053
00101	14*	C **	MATCH. OUTPUT, FIRST COL. CONTAINS MID(KODE=0) OR PID(KODE=1), SECOND COL.										000053
00101	15*	C **	CONTAINS ROW NO. OF THAT MID IN MAT12(KODE=1),OR ROW NO. OF THAT PID IN										000053
00101	16*	C **	PROPERTY ARRAY OF GIVEN ELEMENT (E.G. PROD2 ARRAY)										000053
00101	17*	C **	INDEX. OUTPUT,THE ELS. OF INDEX ARE MID OR PID NOS. OF THE CURRENT ELEMENT										000053
00103	18*		COMMON/MOZART/MAT11(20),MAT12(20,10)										000053
00104	19*		COMMON/BACH/NUM(18),NSTART(18)										000053
00104	20*	C **	CHANGE GROUP, NO. OF ELEMENTS										000053
00105	21*		INTEGER CID(100,7),VID(100),INDEX(100),MATCH(100,2)										000053
00105	22*	C **	CHANGE GROUP, NO. OF PID'S										000053

00106	23*		INTEGER PRID(25,2)	000053
00106	24*	C **	*****	000053
00107	25*		REAL MAT,2	000053
00110	26*	1	FORMAT()	000053
00110	27*	C	PRINT 2	000053
00110	28*	C2	FORMAT(5X,'KODE')	000053
00110	29*	C	PRINT 1, KODE	000053
00111	30*		DO 666 KR=1,100	000053
00114	31*		DO 666 KC=1,2	000053
00117	32*	666	MATCH(KR,KC)=0	000053
00122	33*		DO 8 K=1,100	000064
00125	34*	8	INDEX(K)=0	000064
00125	35*	C	PRINT 3	000064
00125	36*	C3	FORMAT(5X,'CID#S JUST INSIDE PAIR')	000064
00125	37*	C	DO 10 M1=1,ISTOP	000064
00125	38*	C10	PRINT 11, (CID(M1,M2),M2=1,7)	000064
00125	39*	C11	FORMAT(7I6)	000064
00127	40*		NUM18=NUM(18)	000065
00127	41*	C	PRINT 40	000065
00127	42*	C40	FORMAT(' MAT11 AND MAT12 IN PAIR')	000065
00127	43*	C	DO 41 I=1,NUM18	000065
00127	44*	C41	PRINT 45, MAT11(I),(MAT12(I,J),J=1,10)	000065
00127	45*	C 45	FORMAT(/15,5F11.5/3X,5F11.5)	000065
00130	46*		KODE = KODE+1	000067
00131	47*		MAXID = CID (1,KOL)	000072
00132	48*		DO 75 J2=2,ISTOP	000077
00135	49*		IF (CID(J2,KOL).GT. MAXID)MAXID = CID (J2,KOL)	000077
00137	50*	75	CONTINUE	000106
00137	51*	C **	FOR ALL INTEGERS BETWEEN 1 AND MAXID,FIND OUT HOW MANY MID'S OR PID'S THERE	000106
00137	52*	C **	ARE OF EACH. ASSUME MAXID =100	000106
00137	53*	C	PRINT 4	000106
00137	54*	C4	FORMAT(5X,'MAXID IN PAIR')	000106
00137	55*	C	PRINT 1, MAXID	000106
00141	56*		DO 76 J2=1,MAXID	000106
00144	57*	76	VID (J2)=0	000114
00146	58*		DO 77 J3=1,MAXID	000123
00151	59*		DO 76 J4=1,ISTOP	000123
00154	60*		IF (CID(J4,KOL).EQ. J3) GO TO 79	000123
00156	61*		GO TO 78	000125
00157	62*	79	VID (J3) = VID (J3)+1	000127
00160	63*	78	CONTINUE	000135
00162	64*	77	CONTINUE	000135
00162	65*	C	PRINT 15	000135
00162	66*	C 15	FORMAT(' VID IN PAIR')	000135
00162	67*	C	DO 16 J3=1,MAXID	000135
00162	68*	C16	PRINT 1, VID(J3)	000135
00162	69*	C **	NOUT (=NUMMAT OR NUMGEO) IS THE NO. OF NONZERO ELS. OF VID.	000135
00162	70*	C **	THE N'S ON THE MATERIAL CONTROL CARDS AND GEOMETRIC PROP. CARDS ARE THE	000135
00162	71*	C **	INDICES OF THESE NONZERO ELS. OF VID.	000135
00164	72*		J6=0	000135
00165	73*		NOUT=C	000136
00166	74*		DO 80 J5=1,MAXID	000142
00171	75*		IF (VID(J5).EQ.0)GO TO 80	000142
00173	76*		NOUT=NOUT+1	000144
00174	77*		J6=J6+1	000147
00175	78*		INDEX(J6)=J5	000154
00176	79*	80	CONTINUE	000160

```
CO176 8J* C ** INDEX,I)=J MEANS THAT J IS AN MID OR PID NO. 000160
CO176 81* C ** FIND THE MAT11 OR PID INDEX CORRESPONDING TO EACH OF THE ELS.OF INDEX 000160
CO200 62* DO 81 J7=1,NOUT 000160
CO203 83* DO 82 J8=1,NCARDS 000172
CO206 84* GO TO (100,200),KODE 000172
CO207 85* 100 IF (MAT11(J8),EQ, INDEX(J7)) GO TO 83 000202
CO211 86* GO TO 82 000204
CO212 87* 200 IF (PRID (J8,1),EQ,INDEX (J7)) GO TO 83 000206
CO214 88* GO TO 82 000210
CO215 89* 83 MATCH (J7,1)=INDEX(J7) 000212
CO216 90* MATCH (J7,2)=J8 000213
CO217 91* 82 CONTINUE 000224
CO221 92* 81 CONTINUE 000224
CO221 93* C PRINT 5 000224
CO221 94* C5 FORMAT(5X,'MATCH AND INDEX IN PAIR') 000224
CO221 95* C DO 150 M1=1,NOUT 000224
CO221 96* C150 PRINT 1,(MATCH(M1,M2),M2=1,2),INDEX(M1) 000224
CO221 97* C PRINT 6, NOUT 000224
CO221 98* C6 FORMAT(5X,'NOUT=',I3) 000224
CO223 99* RETURN 000224
CO224 100* END 000306
END FOR
```

WHUG.P WDMEM

@FOR,S QMEM,QMEM
HSA E3 -07/30/80-09:46:16 (6,)

SUBROUTINE QMEM ENTRY POINT Q00145

STORAGE USED: CODE(1) Q00177; DATA(0) Q00176; BLANK COMMON(2) Q00000

COMMON BLOCKS:

Q003 BACH LU0044

EXTERNAL REFERENCES (BLOCK, NAME)

Q004 FORM1
Q005 QMEM1
Q006 QMEM2
Q007 NERR3

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

Q001	Q00100 11L	Q001	Q00063 121G	Q001	Q00072 126G	Q001	Q00120 20L	Q000	Q00054 IDV31
Q000	I Q00066 IDV32	Q000	Q00161 INJP5	Q000	I Q00153 ISTOP1	Q000	I Q00154 ISTOP2	Q000	I Q00002 KOAD
Q000	I Q00155 K17	Q000	I Q00156 K18	Q003	Q00022 NSTART	Q003	I Q00000 NUM	Q000	I Q00151 NUM14
Q000	I Q00152 NUM18	Q000	I Q00150 NUM7	Q000	I Q00000 TYPE				

Q0101	1*		SUBROUTINE QMEM(CONFAC,NKODE, IDV3, NIDV3)	Q00002
Q0103	2*		COMMON/BACH/NUM(18), NSTART(18)	Q00002
Q0104	3*		INTEGER TYPE(2), KOAD(2), IDV3(100), IDV31(50), IDV32(50)	Q00002
Q0105	4*		DATA TYPE/'QUADME', 'M' '/'	Q00002
Q0107	5*		CALL FORM1(NKODE, KOAD, TYPE)	Q00002
Q0110	6*		NUM7=NUM(7)	Q00007
Q0111	7*		NUM14=NUM(14)	Q00011
Q0112	8*		NUM18=NUM(18)	Q00013
Q0113	9*		IF(NKODE.EQ.1) GO TO 11	Q00015
Q0115	10*		CALL QMEM1(NUM7, NUM14, NUM18, CONFAC, KOAD, IDV31, ISTOP1, NKODE)	Q00020
Q0116	11*		CALL QMEM2(NUM7, NUM14, NUM18, CONFAC, KOAD, IDV32, ISTOP2, NKODE)	Q00032
Q0117	12*		NIDV3=ISTOP1, ISTOP2	Q00044
Q0120	13*		DO 1,1, K17=1, ISTOP2	Q00063
Q0123	14*	1010	IDV3(K17)=IDV31(K17)	Q00063
Q0125	15*		DO 1020 K18=1, ISTOP2	Q00072
Q0130	16*	1020	IDV3(ISTOP1+K18)=IDV32(K18)	Q00072
Q0132	17*		RETURN	Q00074
Q0133	18*	11	IF(KOAD(1).EQ.1) GO TO 20	Q00160
Q0135	19*		CALL QMEM2(NUM7, NUM14, NUM18, CONFAC, KOAD, IDV3, NIDV3, NKODE)	Q00102
Q0136	20*		RETURN	Q00114
Q0137	21*	20	CALL QMEM1(NUM7, NUM14, NUM18, CONFAC, KOAD, IDV3, NIDV3, NKODE)	Q00120
Q0140	22*		RETURN	Q00131
Q0141	23*		END	Q00176

END FOR

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00101	1*		SUBROUTINE QDMEH1(NUM7,NUM14,NUM18,CONFAC,KOAD,IDV31,ISTOP,NKODE)	000000
00103	2*		COMMON/BACH/NUM18),NSTART(18)/MOZART/MAT11(20),MAT12(20,10)	000000
00103	3*	C **	CHANGE GROUP. NO. OF DESAP PLANE STRESS QUADRILATERALS IN ASTRAN	000000
00103	4*	C **	QUADRILATERAL MEMBRANE, CONSTRUCTION CODE 1	000000
00104	5*		INTEGER CQDM1(100,6),VEID(100),CQDMID(100,9),MATCH(100,2),INDEX(10	000000
00104	6*		10)	000000
00105	7*		DIMENSION DUMMY(100,1)	000000
00105	8*	C **	CHANGE GROUP. NO. OF PID'S	000000
00106	9*		INTEGER PQDM1(25,2)	000000
00107	10*		DIMENSION PQDM2(25,2),GEOPR2(25,6)	000000
00107	11*	C **	CHANGE GROUP. NO. OF MID'S	000000
00110	12*		DIMENSION MTPRO1(20,2)	000000
00111	13*		REAL MTPRO2(20),MTPRO3(20,8)	000000
00111	14*	C **	CHANGE GROUP. NO. OF DESIGN VARIABLES	000000
00112	15*		DIMENSION IDV31(50)	000000
00112	16*	C **	*****	000000
00113	17*		INTEGER KOAD(2),TYPE(2),NPAR(7),VAR(4)	000000
00114	18*		DIMENSION FEMU(4,4),FLDAT2(100,4),DUMM(4)	000000
00115	19*		REAL MAT12	000000
00116	20*		DATA VAR/'A','B','C','D'/TYPE/'QUADME','M' /*	000000
00121	21*	7	FORMAT ()	000000
00122	22*		CALL CANDP(7,NUM7,NUM14,CQDM1,DUMMY,PQDM1,PQDM2)	000000
00123	23*		WRITE (6,5)	000010
00125	24*	5	FORMAT (2X,'PLANE STRESS QUADRILATERAL (QUAD, MEMBRANE IN ASTRAN), C	000015
00125	25*		ONST. CODE 1')	000015
00126	26*		NPAR(1)=3	000015
00127	27*		NPAR(2)=NUM(7)	000017
00130	28*		NPAR(4)=1	000021
00131	29*		NPAR(5)=1	000023
00132	30*		NUM7=NUM(7)	000024
00133	31*		ISTOP=NUM7	000025
00134	32*		JJ1=1	000026
00135	33*		IF (NKODE.EQ.1) GO TO 12	000027
00137	34*		CALL FORM2(JJ1,KOAD,TYPE,NPAR(2),VEID,NUM7)	000031
00140	35*		ISTOP=NPAR(2)	000041
00141	36*		NNCOL=6	000043
00142	37*		CALL STOR(ISTOP,NUM7,CQDM1,VEID,NNCOL)	000045
00143	38*	12	DO 70 J=1,ISTOP	000055
00146	39*		NEID=CQDM1(J,1)	000064
00147	40*		NPID=CQDM1(J,2)	000066
00147	41*	C **	FIND PQDM1 CARD WITH SAME ID	000066
00150	42*		DO 65 K=1,NUM14	000073
00153	43*		IF (NPID.EQ.PQDM1(K,1)) GO TO 61	000073
00155	44*		GO TO 65	000076
00156	45*	61	KK=K	000100
00157	46*		GO TO 62	000101
00160	47*	65	CONTINUE	000105
00162	48*	62	CQDMID(J,1)=NEID	000105
00163	49*		CQDMID(J,2)=CQDM1(J,3)	000107
00164	50*		CQDMID(J,3)=CQDM1(J,4)	000111
00165	51*		CQDMID(J,4)=CQDM1(J,5)	000113
00166	52*		CQDMID(J,5)=CQDM1(J,6)	000115
00167	53*		CQDMID(J,6)=CQDM1(KK,2)	000117
00170	54*		CQDMID(J,7)=NPID	000121
00171	55*	70	CONTINUE	000125
00171	56*	C **	FIND NUMMAT AND MATCH	000125

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00173	57*		KKODE=0	000125
00174	58*		KKOL=6	000126
00175	59*		CALL PAIR(KKODE,CODMID,PQDM1,ISTOP,NUM18,KKOL,NUMMAT,MATCH,INDEX)	000130
00176	60*		NPAR(3)=NUMMAT	000145
00177	61*		WRITE (6,501)	000147
00201	62*	501	FORMAT (2X,'ENTER CODE FOR INCOMPATIBLE DISPL. MODES: 0 MEANS USE	000155
00201	63*		1THEM.; 1 MEANS DON'T USE THEM')	000155
00202	64*		READ (5,7) NPAR(6)	000155
00212	65*	C **	SET UP MATERIAL PROPERTIES DATA	000155
00205	66*		DO 83 J10=1,NUMMAT	000167
00210	67*		MTPRO1(J10,1)=INDEX(J10)	000167
00211	68*		MTPRO1(J10,2)=1	000171
00212	69*		DO 84 J11=1,NUMMAT	000176
00215	70*		IF(MATCH(J11,1).EQ.INDEX(J10))GO TO 85	000176
00217	71*		GO TO 84	000201
00220	72*	85	NROW=MATCH(J11,2)	000203
00221	73*		GO TO 86	000204
00222	74*	84	CONTINUE	000210
00224	75*	86	MTPRO2(J10)=MAT12(NROW,4)*CONFAC	000210
00225	76*		MTPRO3(J10,1)=0.	000213
00226	77*		MTPRO3(J10,2)=MAT12(NROW,1)	000214
00227	78*		MTPRO3(J10,3)=MAT12(NROW,3)	000216
00230	79*		MTPRO3(J10,4)=0.	000220
00231	80*		MTPRO3(J10,5)=MAT12(NROW,8)	000221
00232	81*		MTPRO3(J10,6)=MAT12(NROW,9)	000223
00233	82*		MTPRO3(J10,7)=MAT12(NROW,10)	000225
00234	83*		MTPRO3(J10,8)=MTPRO3(J10,6)	000227
00235	84*		IX=INDEX(J10)	000231
00236	85*	83	CONTINUE	000235
00236	86*	C **	MATERIAL PROPERTIES DATA IS DONE.	000235
00236	87*	C **	SET UP GEOMETRIC PROPERTIES DATA	000235
00240	88*		KKKODE=1	000235
00241	89*		KKOL=7	000237
00242	90*		CALL PAIR(KKODE,CODMID,PQDM1,ISTOP,NUM14,KKOL,NUMGEO,MATCH,INDEX)	000241
00243	91*		NPAR(7)=NUMGEO	000254
00244	92*		WRITE(12,795)	000256
00246	93*	795	FORMAT(' COMM ELEMENT CONTROL FROM QMEM1')	000263
00247	94*		WRITE(12,504)(NPAR(J11),J11=1,7)	000263
00252	95*	504	FORMAT(7I5)	000277
00252	96*	C **	WRITE MATERIAL PROPERTIES DATA	000277
00253	97*		WRITE(12,349)	000277
00255	98*	349	FORMAT(' COMM MATERIAL PROPERTIES FROM QMEM1')	000313
00256	99*		DO 350 J12=1,NUMMAT	000313
00261	100*	350	WRITE(12,300)(MTPRO1(J12,J13),J13=1,2),MTPRO2(J12),MTPRO3(J12,J13	000313
00261	101*		1),J13=1,8)	000313
00275	102*	300	FORMAT (2I5,F10.3/2F10.0,F10.3,5F10.0)	000341
00276	103*		WRITE(6,301)	000341
00300	104*	301	FORMAT (10X,'GEOMETRIC PROPERTIES FOR CONST. CODE 1')	000355
00301	105*		DO 283 J10=1,NUMGEO	000355
00304	106*		DO 284 J11=1,NUMGEO	000355
00307	107*		IF(MATCH(J11,1).EQ.INDEX(J10))GO TO 285	000355
00311	108*		GO TO 284	000360
00312	109*	285	NROW=MATCH(J11,2)	000362
00313	110*		GO TO 286	000363
00314	111*	284	CONTINUE	000367
00316	112*	286	GEOPR2(J10,1)=FQCM2(NROW,1)	000367
00317	113*		WRITE(6,302)INDEX(J10)	000371

00322	114*	302	FORMAT(2X,'FOR PROPERTY ID',I4,' ENTER W,SA,SI,D,WE,IN THAT ORDER'	000377
00322	115*		1)	000377
00323	116*		READ(5,7) (GEOPR2(J12,,J12=2,6)	000377
00326	117*	283	CONTINUE	000411
00330	118*		WRITE(12,96)	000411
00332	119*	96	FORMAT(1' COMM GEOMETRIC PROPERTIES FROM QDMMEM1')	000422
00333	120*		DO 299 J10=1,NUMGEO	000422
00330	121*	299	WRITE(12,303)INDEX(J10),(GEOPR2(J10,J11),J11=1,6)	000422
00340	122*	303	FORMAT(15,6F10.0)	000440
00346	123*	C **	GENERATE ELEMENT LOAD MULTIPLIERS	000440
00347	124*		WRITE(6,111)	000440
00351	125*	111	FORMAT(10X,'ELEMENT LOAD MULTIPLIERS FOR CONST. CODE 1')	000451
00352	126*		DO 115 K6=1,4	000451
00355	127*		DO 115 K7=1,4	000451
00360	128*	115	EMUL(K6,K7)=0.	000451
00363	129*		DO 120 K8=1,4	000461
00366	130*		WRITE(6,112)VARI(K8)	000464
00371	131*	112	FORMAT(2X,'FOR LOAD ',A1,' ENTER X,Y,Z ELEMENT LOAD FRACTIONS, IN	000472
00371	132*		THAT ORDER, DEFAULT = ALL ZEROS')	000472
00372	133*		READ(5,7) (DUMM(K11),K11=1,3)	000472
00375	134*		IF(DUMM(1).EQ.0.)GO TO 120	000502
00377	135*		DO 113 K11=2,4	000511
00402	136*	113	EMUL(K8,K11)=DUMM(K11)	000511
00404	137*	120	CONTINUE	000515
00406	138*		WRITE(12,117)	000515
00410	139*	1117	FORMAT(1' COMM ELEMENT LOAD MULTIPLIERS FROM QDMMEM1')	000525
00411	140*		DO 121 K9=1,4	000525
00414	141*	121	WRITE(12,116) (EMUL(K9,K10),K10=1,4)	000525
00423	142*	116	FORMAT(4F10.0)	000541
00423	143*	C **	GENERATE ELEMENT DATA	000541
00424	144*		WRITE(6,127)	000541
00426	145*	127	FORMAT(15X,'ELEMENT DATA FOR CONST. CODE 1')	000551
00427	146*		DO 150 KROW=1,ISTOP	000551
00432	147*		IX=CODMID(KROW,1)	000551
00433	148*		WRITE(6,130)IX	000552
00436	149*	130	FORMAT(2X,'ENTER DESIGN VARIABLE NO.,IDV, FOR EL.NO.',I5)	000560
00437	150*		READ(5,7) (CODMID(KROW,8)	000560
00442	151*		WRITE(6,131)IX	000566
00445	152*	131	FORMAT(2X,'ENTER DESIGN VARIABLE FRACTION,FFC, FOR EL.NO.',I4,'. DE	000574
00445	153*		F(AULT=1.')	000574
00446	154*		ELDAT2(KROW,1)=1.	000574
00447	155*		READ(5,7) DUM	000576
00452	156*		IF(DUM.NE.0.)ELDAT2(KROW,1)=DUM	000604
00454	157*		ELDAT2(KROW,2)=0.	000610
00455	158*		WRITE(6,132)IX	000611
00460	159*	132	FORMAT(2X,'ENTER ANGLE BETA FOR EL. NO.',I4)	000617
00461	160*		READ(5,7) (ELDAT2(KROW,3)	000617
00464	161*		WRITE(6,133)IX	000625
00467	162*	133	FORMAT(2X,'ENTER FFC FOR EL. NO.',I4,'. DEFAULT =1.')	000633
00470	163*		ELDAT2(KROW,4)=1.	000633
00471	164*		READ(5,7) DUM	000635
00474	165*		IF(DUM.NE.0.)ELDAT2(KROW,4)=DUM	000643
00476	166*		WRITE(6,134)IX	000647
00501	167*	134	FORMAT(2X,'ENTER STRESS PRINTOUT CODE,KS, FOR EL.NO.',I4,'. DEFAULT	000655
00501	168*		I=3')	000655
00502	169*		CODMID(KROW,9)=3	000655
00503	170*		READ(5,7) DUM	000657

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00506	171*		IF (IDUM.NE.0)CQDM7D(,ROW,9)=IDUM	000665
00510	172*	150	CONTINUE	000673
00512	173*		NNCOL1=9	000673
00513	174*		NNCOL2=4	000675
00514	175*		CALL ELORD (ISTOP,CQDMID,ELDAT2,NNCOL1,NNCOL2)	000677
00514	176*	C	PRINT 179	000677
00514	177*	C179	FORMAT(// ' ELEMENT RECORD ON FILE 12, IN QDMEM1' /)	000677
00515	178*		WRITE(12,1135)	000706
00517	179*	1135	FORMAT(' COMM ELEMENT DATA FROM QDMEM1')	000713
00520	180*		DO 180 M12=1,ISTOP	000713
00523	181*	180	WRITE(12,186) (CQDMID(M12,M13),M13=1,8), (ELDAT2(M12,M13),M13=1,4),C	000723
00523	182*		1CQDMID(M12,9)	000723
00523	183*	C180	WRITE(6,186) (CQDMID(M12,M13),M13=1,8), (ELDAT2(M12,M13),M13=1,4),C	000723
00523	184*	C	1CQDMID(M12,9)	000723
00537	185*	186	FORMAT(8I5,2F10.0,20X/2F10.0,I5,5X)	000755
00540	186*		DO 190 K1=1,ISTOP	000755
00543	187*	190	IDV31(K1)=CQDMID(K1,8)	000755
00545	188*		RETURN	000757
00546	189*		END	001051
END FOR				

6HDG,P QDMEM2

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@FOR,S QDMEM2,QDMEM2
HSA E3 -07/30/80-09:46:24 (28,)

SUBROUTINE QDMEM2 ENTRY POINT 000664

STORAGE USED: CODE(1) 000732; DATA(0) 005575; BLANK COMMON(2) 000000

COMMON BLOCKS:

0003 BACH 000044
0004 MOZART 000334

EXTERNAL REFERENCES (BLOCK, NAME)

0005 CANDP
0006 FGRM2
0007 STOR
0010 PAIR
0011 ELORD
0012 NWDUS
0013 N102S
0014 NWDUS
0015 N103S
0016 N101S
0017 NERR3S

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000	005351	111F	0000	005404	1117F	0000	005362	112F	0000	005521	1135F	0000	005414	116F					
0001	000370	120L	0000	005416	127F	0000	005425	130F	0000	005436	131F	0000	005453	132F					
0000	005472	133F	0000	005505	134F	0000	005527	136F	0001	000067	145G	0001	000076	152G					
0001	000210	215G	0001	000217	222G	0001	000266	252G	0001	000273	256G	0001	000301	263G					
0001	000324	275G	0001	000060	3L	0000	005343	300F	0001	000325	300G	0001	000335	306G					
0001	000365	322G	0001	000401	334G	0001	000406	340G	0000	005333	349F	0001	000426	352G					
0001	000467	377G	0001	000607	450G	0001	000614	454G	0001	000620	460G	0001	000641	470G					
0000	005265	5F	0000	005303	501F	0000	005332	504F	0001	000103	61L	0001	000110	62L					
0001	000106	65L	0000	005264	7F	0000	005323	795F	0001	000227	84L	0001	000224	85L					
0001	000231	86L	0000	001274	CCUMID	0000	000000	CCUM1	0000	R	005254	DUM	0000	R	005220	DUMM			
0000	R	003410	DUMMY	0000	R	004400	ELQATZ	0000	R	004360	FMU1	0000	I	005256	IOUM	0000	I	003244	INQEX
0000	I	005547	INJP1	0000	I	005242	IX	0000	I	005227	J	0000	I	005225	JJ1	0000	I	005240	J10
0000	I	005237	J11	0000	I	005243	J12	0000	I	005244	J13	0000	I	005232	K	0000	I	005233	KK
0000	I	005224	KKDE	0000	I	005234	KKODE	0000	I	005235	KKOL	0000	I	005253	KROW	0000	I	005263	K1
0000	I	005252	K10	0000	I	005250	K11	0000	I	005245	K6	0000	I	005246	K7	0000	I	005247	K8
0000	I	005251	K9	0000	I	002734	MATCH	0004	000000	MAT11	0004	R	000024	MAT12	0000	I	004033	MTPRO1	
0000	R	004103	MTPRO2	0000	R	004127	MTPRO3	0000	I	005255	M1	0000	I	005261	M12	0000	I	005262	M13
0000	I	005230	NEID	0000	I	005226	NMCOL	0000	I	005257	NMCOL1	0000	I	005260	NMCOL2	0000	I	004345	NPAP
0000	I	005231	NPID	0000	I	005241	NROW	0003	000022	NSTART	0003	I	000000	NUM	0000	I	005236	NUMMAT	
0000	I	003554	P00M1	0000	R	003636	P00M2	0000	I	004343	TYPE	0000	I	004354	VAR	0000	I	001130	VEID

00101 1* SUBROUTINE QDMEM2(NUM7,NUM14,NUM18,CONFAC,KOAO,LDV32,ISTOP,NKQDE) 000000

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00103	2*		COMMON,BACH/NUM(18),NSTART(18)/MOZ,RT/MAT11(20),MAT12(20,10)	000000
00103	3*	C **	CHANGE GROUP. NO. OF DESAP PLANE STRESS QUADRILATERALS IN ASTRAN	000000
00103	4*	C **	QUADRILATERAL MEMBRANE CONSTRUCTION CODE 2	000000
00104	5*		INTEGER CQDM1(100,6),VEID(100),CQDMID(100,8),MATCH(100,2),INDEX(10	000000
00104	6*		10)	000000
00105	7*		DIMENSION DUMMY(100,1)	000000
00105	8*	C **	CHANGE GROUP. NO. OF PID'S	000000
00106	9*		INTEGER PQDM1(25,2)	000000
00107	10*		DIMENSION PQDM2(25,5)	000000
00107	11*	C **	CHANGE GROUP. NO. OF MID'S	000000
00110	12*		DIMENSION MTPRO1(20,2)	000000
00111	13*		REAL MTPRO2(20),MTPRO3(20,7)	000000
00111	14*	C **	CHANGE GROUP. NO. OF DESIGN VARIABLES	000000
00112	15*		DIMENSION IDV32(50)	000000
00112	16*	C **	*****	000000
00113	17*		INTEGER KOAD(2),TYPE(2),NPAR(7),VAR(4)	000000
00114	18*		DIMENSION EMUL(4,4),ELDAT2(100,4),DUMH(4)	000000
00115	19*		REAL MAT12	000000
00116	20*		DATA VAR/'A','B','C',,D./TYPE/'QUADME','M' '/'	000000
00121	21*	7	FORMAT()	000000
00122	22*		KKDE=7	000000
00123	23*		CALL CANDP(KKDE,NUM7,NUM14,CQDM1,DUMMY,PQDM1,PQDM2)	000001
00124	24*		WRITE(6,5)	000012
00126	25*	5	FORMAT(2X,'PLANE STRESS QUADRILATERAL (QUAD. MEMBRANE IN ASTRAN),	000017
00126	26*		1CONSTRUCTION CODE 2')	000017
00127	27*		NPAR(1)=3	000017
00130	28*		NPAR(2)=NUM(7)	000021
00131	29*		NPAR(4)=1	000023
00132	30*		NPAR(5)=2	000025
00133	31*		NUM7=NUM(7)	000027
00134	32*		ISTOP=NUM7	000030
00135	33*		JJ1=2	000031
00136	34*		IF(NKODE.EQ.1) GO TO 3	000032
00140	35*		CALL FORM2(JJ1,KOAD,TYPE,NPAR(2),VEID,NUM7)	000034
00141	36*		ISTOP=NPAR(2)	000044
00142	37*		NNCOL=6	000046
00143	38*		CALL STOR(ISTOP,NUM7,CQDM1,VEID,NNCOL)	000050
00144	39*	3	DO 70 J=1,ISTOP	000060
00147	40*		NEID=CQDM1(J,1)	000067
00150	41*		NPID=CQDM1(J,2)	000071
00150	42*	C **	FIND PQDM1 CARD WITH SAME ID	000071
00151	43*		DO 65 K=1,NUM14	000076
00154	44*		IF(NPID.EQ.PQDM1(K,1))GO TO 61	000076
00156	45*		GO TO 65	000101
00157	46*	61	KK=K	000103
00160	47*		GO TO 62	000104
00161	48*	65	CONTINUE	000110
00163	49*	62	CQDMID(J,1)=NEID	000110
00164	50*		CQDMID(J,2)=CQDM1(J,3)	000112
00165	51*		CQDMID(J,3)=CQDM1(J,4)	000114
00166	52*		CQDMID(J,4)=CQDM1(J,5)	000116
00167	53*		CQDMID(J,5)=CQDM1(J,6)	000120
00170	54*		CQDMID(J,6)=CQDM1(KK,2)	000122
00171	55*	70	CONTINUE	000126
00171	56*	C **	FIND NUMMAT AND MATCH	000126
00173	57*		KKODE=J	000126
00174	58*		KKOL=6	000127

00175	59*		CALL PAIR(KKODE,QDMID,QDM1,ISTOP,NUM18,KKCL,NUMMAT,MATCH,INDEX)	000131
00176	60*		NPARI(3)=NUMMAT	000144
00177	61*		WRITE(6,501)	000146
00201	62*	501	FORMAT(2X,'ENTER CODE FOR INCOMPATIBLE DISPL. MODES: 0 MEANS USE T	000153
00201	63*		IHEM; 1 MEANS DO NOT USE THEM')	000153
00202	64*		READ(5,7)NPAR(6)	000153
00205	65*		WRITE(12,795)	000161
00207	66*	795	FORMAT(' COMM ELEMENT CONTROL FROM QDMEM2')	000166
00210	67*		WRITE(12,504)(NPAR(J11),J11=1,6)	000166
00213	68*	504	FORMAT(6I5)	000210
00213	69*	C **	SET UP MATERIAL PROPERTIES DATA	000210
00214	70*		DO 83 J10=1,NUMMAT	000210
00217	71*		MTPRO1(J10,1)=INDEX(J10)	000210
00220	72*		MTPRO1(J10,2)=1	000212
00221	73*		DO 84 J11=1,NUMMAT	000217
00224	74*		IF(MATCH(J11,1).EQ.INDEX(J10))GO TO 85	000217
00226	75*		GO TO 84	000222
00227	76*	85	NROW=MATCH(J11,2)	000224
00230	77*		GO TO 86	000225
00231	78*	84	CONTINUE	000231
00233	79*	86	MTPRO2(J10)=MAT12(NROW,4)*CONFAC	000231
00234	80*		MTPRO3(J10,1)=0.	000234
00235	81*		MTPRO3(J10,2)=MAT12(NROW,1)	000235
00236	82*		MTPRO3(J10,3)=MAT12(NROW,3)	000237
00237	83*		MTPRO3(J10,4)=0.	000241
00240	84*		MTPRO3(J10,5)=MAT12(NROW,8)	000242
00241	85*		MTPRO3(J10,6)=MAT12(NROW,9)	000244
00242	86*		MTPRO3(J10,7)=MAT12(NROW,10)	000246
00243	87*		IX=INDEX(J10)	000250
00244	88*	83	CONTINUE	000254
00244	89*	C **	MATERIAL PROPERTIES DATA IS DONE	000254
00246	90*		WRITE(12,349)	000254
00250	91*	349	FORMAT(' COMM MATERIAL PROPERTIES FROM QDMEM2')	000266
00251	92*		DO 350 J12=1,NUMMAT	000266
00254	93*	350	WRITE(12,300)(MTPRO1(J12,J13),J13=1,2),MTPRO2(J12),(MTPRO3(J12,J13	000266
00254	94*		1),J13=1,7)	000266
00270	95*	300	FORMAT(2I5,F10.3/2F10.0,F10.3,4F10.0)	000314
00270	96*	C **	GENERATE ELEMENT LOAD MULTIPLIERS	000314
00271	97*		WRITE(6,111)	000314
00273	98*	111	FORMAT(10X,'ELEMENT LOAD MULTIPLIERS FOR CONST. CODE 2')	000325
00274	99*		DO 115 K6=1,4	000325
00277	100*		DO 115 K7=1,5	000325
00302	101*	115	EMUL(K6,K7)=0.	000325
00305	102*		DO 120 K8=1,4	000335
00311	103*		WRITE(6,112)VAR(K9)	000340
00313	104*	112	FORMAT(2X,'FOR LOAD ',A1,' ENTER PRESSURE AND X,Y,Z ELEMENT LOAD F	000346
00313	105*		RACIONS IN THAT ORDER, DEFAULT=ALL ZEROS')	000346
00314	106*		READ(5,7)(DUMH(K11),K11=1,4)	000346
00317	107*		IF(DUMH(1).EQ.0.)GO TO 120	000356
00321	108*		DO 113 K11=2,5	000365
00324	109*	113	EMUL(K8,K11)=DUMH(K11)	000365
00326	110*	120	CONTINUE	000371
00330	111*		WRITE(12,1117)	000371
00332	112*	1117	FORMAT(' COMM ELEMENT LOAD MULTIPLIERS FROM QDMEM2')	000401
00333	113*		DO 121 K9=1,4	000401
00336	114*	121	WRITE(12,116)EMUL(K9,K10),K10=1,5)	000401
00345	115*	116	FORMAT(5F10.0)	000415

00345	116*	C **	GENERATE ELEMENT DATA	000415
00346	117*		WRITE(6,127)	000415
00350	118*	127	FORMAT(15X,'ELEMENT DATA FOR CONST. CODE 2')	000426
00351	119*		DO 150 KROW=1,ISTOP	000426
00354	120*		IX=CQDMID(KROW,1)	000426
00355	121*		WRITE(6,130) IX	000427
00360	122*	130	FORMAT(2X,'ENTER DESIGN VARIABLE NO.,IDV, FOR EL. NO.',I5)	000435
00361	123*		READ(5,7)CQDMID(KROW,7)	000435
00364	124*		WRITE(6,131) IX	000443
00367	125*	131	FORMAT(2X,'ENTER DESIGN VARIABLE FRACTION ,FRC, FOR EL. NO.',I4,')	000451
00367	126*		1 DEFAULT=1.')	000451
00370	127*		ELDAT2(KROW,1)=1.	000451
00371	128*		READ(5,7)DUM	000453
00374	129*		IF(DUM.NE.0.)ELDAT2(KROW,1)=DUM	000461
00376	130*		DO 145 M1=2,4	000467
00401	131*	145	ELDAT2(KROW,M1)=0.	000467
00403	132*		WRITE(6,132)IX	000470
00406	133*	132	FORMAT(2X,'ENTER COMP. FORCE/UNIT LENGTH APPLIED TO SIDE 1-2 OF EL	000476
00406	134*		1,NO.',I4,'). DEFAULT=0.')	000476
00407	135*		READ(5,7)DUM	000476
00412	136*		IF(DUM.NE.0.)ELDAT2(KROW,3)=DUM	000504
00414	137*		WRITE(6,133)IX	000510
00417	138*	133	FORMAT(2X,'ENTER THE ANGLE BETA FOR EL. NO.',I4,'). DEFAULT=0.')	000516
00420	139*		READ(5,7)DUM	000516
00423	140*		IF(DUM.NE.0.)ELDAT2(KROW,4)=DUM	000524
00425	141*		WRITE(6,134)IX	000530
00430	142*	134	FORMAT(2X,'ENTER STRESS PRINTOUT CODE,KS, FOR EL. NO.',I4,'). DEFAULT	000536
00430	143*		IT=3')	000536
00431	144*		CQDMID(KROW,6)=3	000536
00432	145*		READ(5,7)IDUM	000540
00435	146*		IF(IDUM.NE.0)CQDMID(KROW,8)=IDUM	000546
00437	147*	150	CONTINUE	000557
00441	148*		NNCOL1=7	000557
00442	149*		NNCOL2=4	000561
00443	150*		CALL ELORD(ISTOP,CQDMID,ELDAT2,NNCOL1,NNCOL2)	000563
00443	151*	C	PRINT 179	000563
00443	152*	C179	FORMAT(//,'ELEMENT DATA RECORD ON FILE 12, IN QDMEM2')	000563
00444	153*		WRITE(12,1135)	000572
00446	154*	1135	FORMAT(10 COMM ELEMENT DATA FROM QDMEM2')	000577
00447	155*		DO 180 M12=1,ISTOP	000577
00452	156*	180	WRITE(12,136)(CQDMID(M12,M13),M13=1,7),(ELDAT2(M12,M13),M13=1,4),C	000607
00452	157*		100MID(M12,8)	000607
00452	158*	C180	WRITE(6,136)(CQDMID(M12,M13),M13=1,7),(ELDAT2(M12,M13),M13=1,4),C	000607
00452	159*	C	100MID(M12,8)	000607
00466	160*	136	FORMAT(7I5,F5.0,3F10.0,I5,5X)	000641
00467	161*		DO 190 K1=1,ISTOP	000641
00472	162*	190	IDV32(K1)=CQDMID(K1,7)	000641
00474	163*		RETURN	000643
00475	164*		END	000731
END FOR				

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U0101	1*		SUBROUTINE QT(CONFAC,NKODE,IUV5,NCTOT)	000000
U0103	2*		COMMON/BACH/NUM(18),NSTART(18)/MOZART/MAT1(20),MAT12(20,10)	000000
U0103	3*	C **	CHANGE GROUP. NO. OF PLATE QUADRILATERALS AND/OR PLATE TRIANGLES	000000
U0104	4*		INTEGER CQUAD1(100,6),CTRI1(100,5),CQTID(100,7),MATCH(100,2),INDEX(1	000000
U0104	5*		1(100),CQT1(100,6),CQT2(100,1)	000000
U0105	6*		DIMENSION CQUAD2(100,1),CTRI2(100,1),ELDAT2(100,6)	000000
U0105	7*	C **	CHANGE GROUP. NO. OF PID'S	000000
U0106	8*		INTEGER PQUAD1(25,2),PTRI1(25,2),PQT1(25,2)	000000
U0107	9*		DIMENSION PQUAD2(25,2),PTRI2(25,2),PQT2(25,2)	000000
U0107	10*	C **	CHANGE GROUP. NO. OF MID'S	000000
U0110	11*		INTEGER MTPRO1(20,2)	000000
U0111	12*		REAL MTPRO2(20,1),MTPRO3(20,7)	000000
U0111	13*	C **	CHANGE GROUP. NO. OF DESIGN VARIABLES	000000
U0112	14*		DIMENSION IDV5(50)	000000
U0112	15*	C **	*****	000000
U0113	16*		DIMENSION NPAR(5),EMUL(5,4),VAR(4)	000000
U0114	17*		REAL MAT12	000000
U0115	18*		DATA VAR/'A','B','C','D'/	000000
U0117	19*	7	FORMAT ()	000000
U0120	20*		NUM9=NUM(9)	000000
U0121	21*		NUM10=NUM(10)	000001
U0122	22*		NUM17=NUM(17)	000003
U0123	23*		NUM16=NUM(16)	000005
U0124	24*		NUM16=NUM(16)	000007
U0125	25*		NCTOT=NUM9*NUM10	000011
U0126	26*		NPTOT=NUM16*NUM17	000013
U0127	27*		KKDE=9	000015
U0130	28*		IF(NUM9.NE.0) CALL CANDP(KKDE,NUM9,NUM16,CQUAD1,CQUAD2,PQUAD1,PQUA	000017
U0130	29*		ID2)	000017
U0132	30*		KKDE=10	000032
U0133	31*		IF(NUM10.NE.0)CALL CANDP(KKDE,NUM10,NUM17,CTRI1,CTRI2,PTRI1,PTRI2)	000034
U0135	32*		IF (NUM9.EQ.0)GO TO 40	000052
U0137	33*		DO 25 LR=1,NUM9	000054
U0142	34*		DO 25 LC=1,6	000063
U0145	35*	25	CQT1(LR,LC)=CQUAD1(LR,LC)	000063
U0150	36*		DO 26 LR=1,NUM9	000074
U0153	37*	26	CQT2(LR,1)=CQUAD2(LR,1)	000074
U0155	38*		DO 27 LR=1,NUM16	000107
U0160	39*		DO 27 LC=1,2	000107
U0163	40*		PQT1(LR,LC)=PQUAD1(LR,LC)	000107
U0164	41*	27	PQT2(LR,LC)=PQUAD2(LR,LC)	000110
U0167	42*	40	IF(NUM10.EQ.0)GO TO 50	000120
U0171	43*		ISTARC=NUM9+1	000121
U0172	44*		ISTARP=NUM16+1	000124
U0173	45*		DO 31 LR=ISTARC,NCTOT	000136
U0176	46*		DO 31 LC=1,5	000156
U0201	47*	31	CQT1(LR,LC)=CTRI1(LR-NUM9,LC)	000156
U0204	48*		DO 32 LR=ISTARC,NCTOT	000173
U0207	49*	32	CQT1(LR,6)=0	000173
U0211	50*		DO 33 LR=ISTARC,NCTOT	000202
U0214	51*	33	CQT2(LR,1)=CTRI2(LR-NUM9,1)	000202
U0216	52*		DO 34 LR=ISTARP,NPTOT	000220
U0221	53*		DO 34 LC=1,2	000220
U0224	54*		PQT1(LR,LC)=PTRI1(LR-NUM16,LC)	000220
U0225	55*	34	PQT2(LR,LC)=PTRI2(LR-NUM16,LC)	000221
U0225	56*	C	PRINT 30	000221

00225	57*	C30	FORMAT(' CQT1 AND PQT1 ARRAY IN QT')	000221
00225	58*	C	DO 36 I=1,NCTOT	000221
00225	59*	C36	PRINT 35, (CQT1(I,J),J=1,6), (PQT1(I,J),J=1,2)	000221
00225	60*	C35	FORMAT(615,215)	000221
00230	61*	50	KKODE=1	000234
00230	62*	C **	THERE IS ONLY ONE CONST. CODE FOR THIS ELEMENT	000234
00231	63*		NPAR(1)=6	000235
00232	64*		NPAR(2)=NCTOT	000237
00233	65*		NPAR(4)=1	000241
00234	66*		NPAR(5)=1	000242
00235	67*		DO 70 J=1,NCTOT	000252
00240	68*		NEID=CQT1(J,1)	000252
00241	69*		NPID=CQT1(J,2)	000254
00242	70*		DO 65 K=1,NPTOT	000261
00245	71*		IF(NPID.EQ.PQT1(K,1))GO TO 61	000261
00247	72*		GO TO 65	000264
00250	73*	61	KK=K	000266
00251	74*		GO TO 62	000267
00252	75*	65	CONTINUE	000273
00254	76*	62	CQTID(J,1)=NEID	000273
00255	77*		CQTID(J,2)=CQT1(J,3)	000275
00256	78*		CQTID(J,3)=CQT1(J,4)	000277
00257	79*		CQTID(J,4)=CQT1(J,5)	000301
00260	80*		CQTID(J,5)=CQT1(J,6)	000303
00261	81*		CQTID(J,6)=PQT1(KK,2)	000305
00262	82*	70	CONTINUE	000311
00262	83*	C	PRINT 71	000311
00262	84*	C71	FORMAT(' CQTID ARRAY IN QT JUST BEFORE GOING TO PAIR')	000311
00262	85*	C	DO 72 I=1,NCTOT	000311
00262	86*	C72	PRINT 73, (CQTID(I,J),J=1,6)	000311
00262	87*	C73	FORMAT(615)	000311
00262	88*	C **	FIND NUMMAT AND MATCH	000311
00264	89*		KKODE=0	000311
00265	90*		KKOL=6	000312
00266	91*		CALL PAIR(KKODE,CQTID,PQT1,NCTOT,NUM18,KKOL,NUMMAT,MATCH,INDEX)	000314
00267	92*		NPAR(3)=NUMMAT	000327
00270	93*		WRITE(12,795)	000331
00272	94*	795	FORMAT(' COMM ELEMENT CONTROL FROM QT')	000336
00273	95*		WRITE(12,504)(NPAR(J11),J11=1,5)	000336
00276	96*	504	FORMAT(515)	000360
00276	97*	C **	SET UP MATERIAL PROPERTIES DATA	000360
00277	98*		DO 83 J10=1,NUMMAT	000360
00302	99*		MTPRO1(J10,1)=INDEX(J10)	000360
00303	100*		MTPRO1(J10,2)=1	000362
00304	101*		DO 84 J11=1,NUMMAT	000367
00307	102*		IF(MATCH(J11,1).EQ.INDEX(J10))GO TO 85	000367
00311	103*		GO TO 84	000372
00312	104*	85	NROW=MATCH(J11,2)	000374
00313	105*		GO TO 86	000375
00314	106*	84	CONTINUE	000401
00316	107*	86	MTPRO2(J10)=MAT12(NROW,4)*CONFAC	000401
00317	108*		MTPRO3(J10,1)=0.	000404
00320	109*		MTPRO3(J10,2)=MAT12(NROW,1)	000405
00321	110*		MTPRO3(J10,3)=MAT12(NROW,3)	000407
00322	111*		MTPRO3(J10,4)=0.	000411
00323	112*		MTPRO3(J10,5)=MAT12(NROW,8)	000412
00324	113*		MTPRO3(J10,6)=MAT12(NROW,9)	000414

U0325	114*		MTPRO3(J10,7)=MAT12(NROW,10)	000416
U0326	115*		IX=INDEX(J10)	000420
G0327	116*	83	CONTINUE	000424
G0327	117*	C **	MATERIAL PROPERTIES DATA IS DONE	000424
G0331	118*		WRITE(12,349)	000424
G0333	119*	349	FORMAT(' COMM MATERIAL PROPERTIES FROM QT')	000436
G0334	120*		DO 350 J12=1,NUMMAT	000436
U0337	121*	350	WRITE(12,300)(MTPRO1(J12,J13),J13=1,2),MTPRO2(J12),(MTPRO3(J12,J13	000436
G0337	122*		1),J13=1,7)	000436
G0353	123*	300	FORMAT(2F10.3/2F10.0,F10.3,4F10.0)	000464
G0353	124*	C **	GENERATE ELEMENT LOAD MULTIPLIERS	000464
G0354	125*		WRITE(6,111)	000464
G0356	126*	111	FORMAT(15X,'ELEMENT LOAD MULTIPLIERS')	000475
G0357	127*		DO 115 K6=1,5	000475
G0362	128*		DO 115 K7=1,4	000475
G0365	129*	115	EMUL(K6,K7)=0.	000475
G0370	130*		DO 120 K8=1,4	000506
G0373	131*		WRITE(6,112)VAR(K8)	000506
G0376	132*	112	FORMAT(2X,'FOR LOAD ',A), ' ENTER LATERAL PRESSURE AND X,Y,Z ELEMEN	000513
G0376	133*		T LOAD FRACTIONS, IN THAT ORDER. DEFAULT= ALL ZEROS')	000513
G0377	134*		READ(5,7)DUM1,DUM2,DUM3,DUM4	000513
G0405	135*		IF(DUM1.EQ.0.)GO TO 120	000524
G0407	136*		EMUL(1,K8)=DUM1	000526
G0410	137*		EMUL(3,K8)=DUM2	000530
G0411	138*		EMUL(4,K8)=DUM3	000532
G0412	139*		EMUL(5,K8)=DUM4	000534
G0413	140*	120	CONTINUE	000541
G0415	141*		WRITE(12,1117)	000541
G0417	142*	1117	FORMAT(' COMM ELEMENT LOAD MULTIPLIERS FROM QT')	000551
G0420	143*		DO 121 K9=1,5	000551
G0423	144*	121	WRITE(12,116)(EMUL(K9,K10),K10=1,4)	000551
G0432	145*	116	FORMAT(4F10.0)	000565
G0432	146*	C **	GENERATE ELEMENT DATA	000565
G0433	147*		WRITE(6,127)	000565
G0435	148*	127	FORMAT(15X,'ELEMENT DATA')	000575
G0436	149*		DO 150 KROW=1,NCOT	000575
G0441	150*		IX=CQCID(KROW,1)	000575
G0442	151*		WRITE(6,130)IX	000576
G0445	152*	130	FORMAT(2X,'ENTER DESIGN VARIABLE NO.,IDV, FOR EL. NO.',I5)	000604
G0446	153*		READ(5,7)CQCID(KROW,7)	000604
G0451	154*		WRITE(6,131)IX	000612
G0454	155*	131	FORMAT(2X,'ENTER LATERAL PRESSURE ON EL. NO.',I4,'. DEFAULT=0.')	000620
G0455	156*		ELDAT2(KROW,1)=0.	000620
G0456	157*		READ(5,7)DUM	000621
G0461	158*		IF(DUM.NE.0.)ELDAT2(KROW,1)=DUM	000627
G0463	159*		ELDAT2(KROW,2)=0.	000633
G0464	160*		WRITE(6,132)IX	000634
G0467	161*	132	FORMAT(2X,'ENTER DESIGN VARIABLE FRACTION,FR _C , FOR EL. NO.',I4,'. D	000642
G0467	162*		EFAULT=1.')	000642
G0470	163*		ELDAT2(KROW,3)=1	000642
G0471	164*		READ(5,7)DUM	000644
G0474	165*		IF(DUM.NE.0.)ELDAT2(KROW,3)=DUM	000652
G0476	166*		WRITE(6,133)IX	000656
G0501	167*	133	FORMAT(2X,'ENTER THE ANGLE BETA FOR EL. NO.',I4,'. DEFAULT=0.')	000664
G0502	168*		ELDAT2(KROW,4)=0	000664
G0503	169*		READ(5,7)DUM	000665
G0506	170*		IF(DUM.NE.0.)ELDAT2(KROW,4)=DUM	000673

00510	171*	150	CONTINUE	000701
00512	172*		NNCOL1=7	000701
00513	173*		NNCOL2=4	000703
00514	174*		CALL ELORD(NCTOT,CQTID,ELDAT2,NNCOL1,NNCOL2)	000705
00515	175*		WRITE(12,1135)	000714
00517	176*	1135	FORMAT(' COMM EL. DATA FROM QT')	000721
00520	177*		DO 180 M12=1,NCTOT	000721
00523	178*	180	WRITE(12,136)((CQTID(M12,M13),M13=1,7),(ELDAT2(M12,M13),M13=1,4)	000730
00523	179*	C180	WRITE(16,136)((CQTID(M12,M13),M13=1,7),(ELDAT2(M12,M13),M13=1,4)	000730
00536	180*	136	FORMAT(6I5,5X,15,4F10.0)	000760
00537	181*		DO 190 K1=1,NCTOT	000760
00542	182*	190	IDV5(K1)=CQTID(K1,7)	000760
00544	183*		RETURN	000762
00545	184*		END	001033
END FOR				

@HUG,P REMAP

@PRI,S REMAP
FURPUR 2@RIH1 E36 S74T11 07/30/80 09:46:32

00-XXXX-60

@FOR,S SHEAR,SHEAR
HSA E3 -07/30/80-09:46:49 (30,)

SUBROUTINE SHEAR ENTRY POINT 000527

STORAGE USED: CODE(1) 000560; DATA(0) 005060; BLANK COMMON(2) 000000

COMMON BLOCKS:

0003 BACH 000044
0004 MOZART 000339

EXTERNAL REFERENCES (BLOCK, NAME)

0005 CANDP
0006 PAIR
0007 ELORD
0010 NWDUS
0011 NI025
0012 NI035
0013 NI015
0014 NRDU5
0015 NERR35

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000	004677	111F	0000	004730	1117F	0000	004710	112F	0000	005011	1135F	0000	004740	116F
0001	000317	120L	0000	004742	127F	0000	004751	130F	0000	004762	131F	0000	004774	132F
0000	005017	136F	0001	000044	137G	0001	000053	144G	0001	000152	201G	0001	000161	206G
0001	000221	232G	0001	000226	236G	0001	000234	243G	0001	000257	255G	0001	000260	260G
0011	000271	266G	0000	004672	300F	0001	000331	314G	0001	000336	320G	0001	000355	332G
0000	004663	349F	0001	000461	376G	0001	000466	402G	0001	000504	412G	0000	004647	5F
0000	004662	504F	0001	000060	611	0001	000065	621	0001	000063	651	0000	004646	7F
0000	004653	795F	0001	000171	84L	0001	000166	85L	0001	000173	86L	0000	001130	CSHRID
0000	I 000000	CSHR1	0000	R 004640	DUM	0000	R 003244	DUMMY	0000	R 004631	DUM1	0000	R 004632	DUM2
0000	R 004633	DUM3	0000	R 003410	ELDAT2	0000	R 004572	EMUL	0000	I 003100	INDEX	0000	005033	INJP5
0000	I 004637	IX	0000	I 004611	J	0000	I 004621	J10	0000	I 004622	J11	0000	I 004624	J12
0000	I 004625	J13	0000	I 004614	K	0000	I 004615	KK	0000	I 004610	KKDE	0000	I 004617	KKODE
0000	I 004616	KK01	0000	I 004636	KROW	0000	I 004645	K1	0000	I 004635	K10	0000	I 004626	K6
0000	I 004627	K7	0000	I 004630	K8	0000	I 004634	K9	0000	I 002570	MATCH	0004	000000	MAT11
0004	R 000024	MAT12	0000	I 004343	MTPRO1	0000	R 004413	MTPRO2	0000	R 004437	MTPRO3	0000	I 004643	M12
0000	I 004644	M13	0000	I 004612	NEID	0000	I 004641	NCOL1	0000	I 004642	NCOL2	0000	I 004561	NPAR
0000	I 004613	NPID	0000	I 004623	NROW	0003	000022	NSIART	0003	I 000000	NUM	0000	I 004620	NUMMAT
0000	I 004606	NUM15	0000	I 004607	NUM18	0000	I 004664	PSHR1	0000	R 004146	PSHR2	0000	I 004557	TYPE
0000	I 004566	VAR												

00101	1*		SUBROUTINE SHEAR(CONFAC,NMODE,IOV4,NUM8)	000000
00103	2*		COMMON/BACH,NUM(18),NSIART(16)/MOZART/MAT11(20),MAT12(20,1G)	000000
00103	3*	C **	CHANGE GROUP. NO. OF SHEAR PANELS	000000
00104	4*		INTEGER CSHR1(100,6),CSHRID(100,8),MATCH(100,2),INDEX(100)	000000

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00105	5*		DIMENSION DUMMY(100,1),ELDAT2(100,3)	000000
00105	6*	C **	CHANGE GROUP. NO. OF PID'S	000000
00106	7*		INTEGER PSHR1(25,2)	000000
00107	8*		DIMENSION PSHR2(25,5)	000000
00107	9*	C **	CHANGE GROUP. NO. OF MID'S	000000
00110	10*		INTEGER MTPRO1(20,2)	000000
00111	11*		REAL MTPRO2(20),MTPRO3(20,4)	000000
00111	12*	C **	CHANGE GROUP. NO. OF DESIGN VARIABLES	000000
00112	13*		DIMENSION IDV4(50)	000000
00112	14*	C **	*****	000000
00113	15*		INTEGER TYPE(2),NPAR(5),VAR(4)	000000
00114	16*		REAL EMUL(3,4),MAT12	000000
00115	17*		DATA VAR/'A','B','C','D'/TYPE/'SHEAR',' */	000000
00120	18*	7	FORMAT ()	000000
00121	19*		NUM8=NUM(8)	000000
00122	20*		NUM15=NUM(15)	000001
00123	21*		NUM18=NUM(18)	000003
00124	22*		KKODE=8	000005
00125	23*		CALL CANDP(KKODE,NUM8,NUM15,CSHR1,DUMMY,PSHR1,PSHR2)	000007
00125	24*	C	PRINT 13	000007
00125	25*	C13	FORMAT(/' PSHR1 IN SHEAR JUST BACK FROM CANDP'/)	000007
00125	26*	C	DO 15 J=1,NUM15	000007
00125	27*	C15	PRINT 14, (PSHR1(J,K),K=1,2)	000007
00125	28*	C14	FORMAT(2I6)	000007
00126	29*		WRITE (6,5)	000020
00130	30*	5	FORMAT (20X,'SHEAR PANEL')	000025
00131	31*		NKODE=1	000025
00131	32*	C **	THERE IS ONLY ONE CONST. CODE FOR SHEAR PANELS	000025
00132	33*		NPAR(1)=4	000027
00133	34*		NPAR(2)=NUM8	000031
00134	35*		NPAR(4)=1	000033
00135	36*		NPAR(5)=1	000034
00136	37*		DO 70 J=1,NUM8	000044
00141	38*		NEID=CSHR1(J,1)	000044
00142	39*		NPID=CSHR1(J,2)	000046
00143	40*		DO 65 K=1,NUM15	000053
00146	41*		IF(NPID.EQ.PSHR1(K,1))GO TO 61	000053
00150	42*		GO TO 65	000056
00151	43*	61	KK=K	000060
00152	44*		GO TO 62	000061
00153	45*	65	CONTINUE	000065
00155	46*	62	CSHRID(J,1)=NEID	000065
00156	47*		CSHRID(J,2)=CSHR1(J,3)	000067
00157	48*		CSHRID(J,3)=CSHR1(J,4)	000071
00160	49*		CSHRID(J,4)=CSHR1(J,5)	000073
00161	50*		CSHRID(J,5)=CSHR1(J,6)	000075
00162	51*		CSHRID(J,6)=PSHR1(KK,2)	000077
00163	52*	70	CONTINUE	000103
00163	53*	C	PRINT 71	000103
00163	54*	C71	FORMAT(' CSHRID ARRAY JUST BEFORE LEAVING SHEAR FOR PAIR')	000103
00163	55*	C	DO 72 J=1,NUM8	000103
00163	56*	C72	PRINT 73, (CSHRID(J,K),K=1,7)	000103
00163	57*	C73	FORMAT(7I6)	000103
00163	58*	C **	FIND NUMMAT AND MATCH	000103
00165	59*		KKOL=6	000103
00166	60*		KKODE=0	000105
00167	61*		CALL PAIR(KKODE,CSHRID,PSHR1,NUM8,NUM18,KKOL,NUMMAT,MATCH,INDEX)	000106

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00170	62*		NPAR(3)=NUMMAT	000121
00171	63*		WRITE(12,795)	000123
00173	64*	795	FORMAT(' COMM ELEMENT CONTROL FROM SHEAR')	000130
00174	65*		WRITE(12,504)(NPAR(J),J=1,5)	000130
00177	66*	504	FORMAT(5I5)	000152
00177	67*	C **	SET UP MATERIAL PROPERTIES DATA	000152
00200	68*		DO 83 J10=1,NUMMAT	000152
00203	69*		MTPRO1(J10,1)=INDEX(J10)	000152
00204	70*		MTPRO1(J10,2)=1	000154
00205	71*		DO 84 J11=1,NUMMAT	000161
00210	72*		IF (MATCH(J11,1).EQ.INDEX(J10))GO TO 85	000161
00212	73*		GO TO 84	000164
00213	74*	85	NROW=MATCH(J11,2)	000166
00214	75*		GO TO 86	000167
00215	76*	84	CONTINUE	000173
00217	77*	86	MTPRO2(J10)=MAT12(NROW,4)*CONFAC	000173
00220	78*		MTPRO3(J10,1)=0.	000176
00221	79*		MTPRO3(J10,2)=MAT12(NROW,1)	000177
00222	80*		MTPRO3(J10,3)=MAT12(NROW,3)	000201
00223	81*		MTPRO3(J10,4)=MAT12(NROW,10)	000203
00224	82*	83	CONTINUE	000207
00226	83*		WRITE(12,349)	000207
00230	84*	349	FORMAT(' COMM MATERIAL PROPERTIES FROM SHEAR')	000221
00231	85*		DO 350 J12=1,NUMMAT	000221
00234	86*	350	WRITE(12,300)(MTPRO1(J12,J13),J13=1,2),MTPRO2(J12),(MTPRO3(J12,J13	000221
00234	87*		1),J13=1,4)	000221
00250	88*	300	FORMAT(2I5,F10.3/2F10.0,F10.3,F10.0)	000247
00250	89*	C **	GENERATE ELEMENT LOAD MULTIPLIERS	000247
00251	90*		WRITE(6,111)	000247
00253	91*	111	FORMAT(10X,'ELEMENT LOAD MULTIPLIERS FOR CONST. CODE 1')	000260
00254	92*		DO 115 K6=1,3	000260
00257	93*		DO 115 K7=1,4	000260
00262	94*	115	EMUL(K6,K7)=G.	000260
00265	95*		DO 120 K8=1,4	000271
00270	96*		WRITE(6,112)VAR(K8)	000271
00273	97*	112	FORMAT(2X,'FOR LOAD ',A1,' ENTER X,Y,Z ELEMENT LOAD FRACTIONS, IN	000276
00273	98*		1 THAT ORDER. DEFAULT =ALL ZEROS')	000276
00274	99*		READ(5,7)DUM1,DUM2,DUM3	000276
00301	100*		IF(DUM1.EQ.0.)GO TO 120	000306
00303	101*		EMUL(1,K8)=DUM1	000310
00304	102*		EMUL(2,K8)=DUM2	000312
00305	103*		EMUL(3,K8)=DUM3	000314
00306	104*	120	CONTINUE	000321
00310	105*		WRITE(12,1117)	000321
00312	106*	1117	FORMAT(' COMM ELEMENT LOAD MULTIPLIERS FROM SHEAR')	000331
00313	107*		DO 121 K9=1,3	000331
00316	108*	121	WRITE(12,116)(EMUL(K9,K10),K10=1,4)	000331
00325	109*	116	FORMAT(4F10.0)	000345
00325	110*	C **	GENERATE ELEMENT DATA	000345
00326	111*		WRITE(6,127)	000345
00330	112*	127	FORMAT(15X,'ELEMENT DATA FOR CONST.CODE 2')	000355
00331	113*		DO 150 KROW=1,NUM8	000355
00334	114*		IX=CSHRID(KROW,1)	000355
00335	115*		WRITE(6,130)IX	000356
00340	116*	130	FORMAT(2X,'ENTER DESIGN VARIABLE NO.,IDV.FOR EL. NO.',I5)	000364
00341	117*		READ(5,7) CSHRID(KROW,7)	000364
00344	118*		WRITE(6,131)IX	000372

00347	119*	131	FORMAT (2X,'ENTER EDGE SUPPORT CONDITION,ISU,FOR EL. NO.',I5)	000400
00350	120*		READ (5,7)CSHRID(KROW,8)	000400
00353	121*		WRITE (6,132)IX	000406
00356	122*	132	FORMAT (2X,'ENTER DESIGN VARIABLE FRACTION,FRC,FOR EL. NO.',I4,'--	000414
00356	123*		IDEFAULT=1.0)	000414
00357	124*		ELDAT2(KROW,1)=1.	000414
00360	125*		READ (5,7)DUM	000416
00363	126*		IF (DUM.NE.0.)ELDAT2(KROW,1)=DUM	000424
00365	127*	150	CONTINUE	000432
00367	128*		NNCOL1=8	000432
00370	129*		NNCOL2=3	000434
00371	130*		CALL ELORD (NUM8,CSHRID,ELDAT2,NNCOL1,NNCOL2)	000436
00371	131*	C	PRINT 179	000436
00371	132*	C179	FORMAT (// 'ELEMENT DATA RECORD ON FILE 12, IN SHEAR//)	000436
00372	133*		WRITE (12,1135)	000445
00374	134*	1135	FORMAT (' COMM EL. DATA FROM SHEAR')	000452
00375	135*		DO 180 M12=1,NUM8	000452
00400	136*	180	WRITE (12,136)(CSHRID(M12,M13),M13=1,8),ELDAT2(M12,1)	000461
00400	137*	C180	WRITE (6,136)(CSHRID(M12,M13),M13=1,8),ELDAT2(M12,1)	000461
00410	138*	136	FORMAT (6I5,F10.0,25X)	000504
00411	139*		DO 190 K1=1,NUM8	000504
00414	140*	190	IDV4(K1)=CSHRID(K1,7)	000504
00416	141*		RETURN	000506
00417	142*		END	000557
END FOR				

•HUG,P SIGMA

@FOR,S SIGMA,SIGMA
HSA E3 -07/30/80-09:46:53 (6,)

SUBROUTINE SIGMA ENTRY POINT 000016

STORAGE USED: CODE(1) 000044; DATA(0) 000052; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NPRIS
0004 NIOZS
0005 NRDU\$
0006 NERR3\$

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0000 000001 10F C001 000016 1116 0000 000036 7F 0000 000042 INJPS 0000 I 000000 J

00101	1*		SUBROUTINE SIGMA(MATID,LLL,XMAT12)	000005
00103	2*		DIMENSION XMAT12(20,10)	000005
00104	3*		PRINT 10,MATID	000005
00107	4*	10	FORMAT(// ' FOR MATERIAL NO. ',I5, ' ENTER ALLOWABLE TENSILE,COMPRESS	000016
00107	5*		IIVE,AND SHEAR STRENGTHS,IN THAT ORDER ON THREE LINES. ',/' BE SURE	000016
00107	6*		2 TO USE F FORMAT WITH DECIMAL POINT'/)	000016
00110	7*		DO 15 J=8,10	000016
00113	8*	15	READ(5,7)XMAT12(LLL,J)	000016
00113	9*	C 15	XMAT12(LLL,J)=MATID*1000.+J*100.	000016
00117	10*	7	FORMAT(F15.2)	000024
00117	11*	C	PRINT 50	000024
00117	12*	C	DO 20 J=8,10	000024
00117	13*	C 20	PRINT 60,LLL,J,XMAT12(LLL,J)	000024
00117	14*	C 50	FORMAT(' XMAT12 IN SIGMA')	000024
00117	15*	C 60	FORMAT(' XMAT12(',I3,', ',I3,',)=' ,E12.5)	000024
00120	16*		RETURN	000024
00121	17*		END	000043

END FOR

@HDG,P STOR

XXXX-65

@FOR,S STOR,STOR
HSA E3 -07/30/80-09:47:09 (18,)

SUBROUTINE STOR ENTRY POINT L00130

STORAGE USED: CODE(1) 000152; DATA(0) 001174; BLANK COMMON(2) 000000

EXTERNAL REFERENCES (BLOCK, NAME)

0003 NERR3s

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000031	1056	0001	000040	1106	0001	000055	1176	0001	000072	1306	0001	000073	1336
0001	000060	19L	0001	000045	21	0000	I 001131	I	0000	I 001132	II	0000	001135	INJP\$
0000	I 001130	J	0000	I 000000	STORE									

99-IXXX

L0101	1*		SUBROUTINE STOR(ISTOP,NUM,C,VEID,NCOL)	000031
L0101	2*	C **	ISTOP=NP(2)= NO. OF ELEMENTS WITH SPECIFIED CONST. CODE. INPUT	000031
L0101	3*	C **	NUM=TOTAL NO. OF C CARDS FOR THIS ELEMENT, E.G., FOR RODS NUM=NUM5. INPUT	000031
L0101	4*	C **	C IS THE INTEGER ARRAY FORMED FROM THE NUM C CARDS * E.G.,CROD.	000031
L0101	5*	C **	INPUT/OUTPUT.	000031
L0101	6*	C **	VEID CONTAINS USER INPUT ELEMENT ID NOS FOR THIS CONST. CODE. INPUT	000031
L0101	7*	C **	NCOL= NO. OF COLS OF INTEGER C ARRAY IN CALLING PROGRAM E.G., FOR NASTRAN	000031
L0101	8*	C **	ROD ,NCOL=4. INPUT	000031
L0101	9*	C **		000031
00101	10*	C **	FOR THE GIVEN EL. AND CONST.CODE THIS SUBROUTINE	000031
00101	11*	C **	FINDS THE C CARDS AND VEID CARDS WITH MATCHING EID'S, PUTS THE C'S	000031
L0101	12*	C **	IN STORE, THEN PUTS THEM BACK IN C IN THE FIRST ISTOP ROWS.	000031
L0101	13*	C **		000031
L0101	14*	C **	CHANGE GROUP. NO. OF ELEMENTS	000031
L0103	15*		INTEGER C(100,6),VEID(100),STORE(100,6)	000031
00103	16*	C	PRINT 200	000031
L0103	17*	C200	FORMAT(1X,'C JUST INSIDE STOR')	000031
L0103	18*	C	DO 20 I=1,NUM	000031
L0103	19*	C201	PRINT 202, (C(I,J),J=1,4)	000031
L0103	20*	C202	FORMAT(8X,4I8)	000031
L0103	21*	C	PRINT 100, (VEID(I),I=1,NUM)	000031
L0103	22*	C100	FORMAT(1X,'VEID AT INPUT'/2014)	000031
00104	23*		DO 21 J=1, ISTOP	000031
L0107	24*		DO 19 I=1, NUM	000040
L0112	25*		IF (C(I,1).EQ.VEID(J)) GO TO 21	000040
00114	26*		GO TO 19	000043
00115	27*	21	CONTINUE	000045
L0116	28*		DO 22 II=1,NCOL	000045
L0121	29*	22	STORE (J,II)= C(I,II)	000055
00123	30*	19	CONTINUE	000073
00125	31*	20	CONTINUE	000073
00127	32*		DO 23 I = 1,ISTOP	000073
L0132	33*		DO 23 J = 1,NCOL	000073

STOR

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00135 34* 23 C(I,J)=STORE(I,J) 000073
00135 35* C PRINT 102 000073
00135 36* C102 FORMAT(1X,'C JUST BEFORE LEAVING STOR') 000073
00135 37* C DO 25 I=1,ISTOP 000073
00135 38* C25 PRINT 103, (C(I,J),J=1,6) 000073
00135 39* C103 FORMAT(16I5) 000073
00140 40* RETURN 000100
00141 41* END 000151
END FOR
```

HDG,P VEC

XXXXI-67

FOR,S VEC,VEC
HSA E3 -07/30/80-09:47:12 (14,)

SUBROUTINE VEC ENTRY POINT 000053

STORAGE USED: CODE(1) 000063; DATA(0) 000021; BLANK COMMON(2) 000000

COMMON BLOCKS:

0003 HANDEL 000174

EXTERNAL REFERENCES (BLOCK, NAME)

0004 NERR39

STORAGE ASSIGNMENT (BLOCK, TYPE, RELATIVE LOCATION, NAME)

0001	000024	100L	0000	000001	101F	0001	000035	105L	0001	000010	106G	0001	000026	124G	
0001	000040	150L	0001	000017	5L	0001	000022	50L	0001	000042	500L	0000	000000	1	
0003	000007	INJP3	0003	I	000000	KVCMID	0003	I	000024	KVCPID	0003	I	000170	MAXMID	
0003	000172	NUMMID	0003	I	000173	NUMPID						0003	I	000171	MAXPID

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00101 1*          SUBROUTINE VEC(MID,PID)                                000010
00101 2*          C ** THIS SUB CREATES KVCMID AND KVCPID, THE VECTORS    000010
00101 3*          C ** OF THE MID AND PID NO.S                            000010
00101 4*          C ** PID(MID).INPUT, IF ALREADY IN KVCPID(KVCMID),RETURN. 000010
00101 5*          C ** IF NOT, PUT IN PLACE OF FIRST ZERO.              000010
00103 6*          COMMON/HANDEL,KVCMID(20),KVCPID(100),MAXMID,MAXPID,NUMMID,NUMPID 000010
00104 7*          INTEGER PID                                             000010
00104 8*          C PRINT 599                                             000010
00104 9*          C599 FORMAT(' IM IN VEC DOING MY THING')              000010
00104 10*         C PRINT 000,MID,PID                                     000010
00104 11*         C600 FORMAT(' MID AND PID IN VEC',2I5)                000010
00105 12*         GO 50 IF=1,MAXPID                                       000010
00110 13*         IF(KVCMID(1).EQ.0) GO TO 5                               000010
00112 14*         IF(MID.EQ.KVCMID(1)) GO TO 100                          000012
00114 15*         GO TO 50                                                000015
00115 16*         5 KVCMID(1)=MID                                          000017
00116 17*         GO TO 100                                               000020
00117 18*         50 CONTINUE                                             000026
00121 19*         100 CONTINUE                                           000026
00121 20*         C PRINT 101                                             000026
00122 21*         101 FORMAT(' AT START OF 2ND LOOP IN VEC' )           000026
00123 22*         GO 150 IF=1,MAXPID                                       000026
00126 23*         IF(KVCPID(1).EQ.0) GO TO 105                          000026
00130 24*         IF (PID.EQ.KVCPID(1))GO TO 500                         000030
00132 25*         GO TO 150                                              000033
00133 26*         105 KVCPID(1)=PID                                       000035
00134 27*         GO TO 500                                              000036
00135 28*         150 CONTINUE                                           000042

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VEC

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00137 29* 500 RETURN
00140 30* END
END FOR

000042
000062

BRKPT PRINTS

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CONCLUSIONS AND RECOMMENDATIONS

Currently NASAP transforms CAD-generated NASTRAN data to data for DESAP I or DESAP II with minimum input from the engineer. The program has been thoroughly checked.

An effort is underway to implement the design cycles shown in Figure 2. Such a capability will greatly enhance the effectiveness of DESAP I and DESAP II by allowing the simultaneous satisfaction of stress, displacement, and buckling constraints.

REFERENCES

1. Kiusalaas, J. and Reddy, G. B.; DESAP II: "A Structural Design Program with Stress and Buckling Constraints"
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MARSHALL SPACE FLIGHT CENTER
THE UNIVERSITY OF ALABAMA

DEPLOYABLE STRUCTURE DESIGN FOR THE SCIENCE
AND APPLICATIONS SPACE PLATFORM

Prepared by: Henry W. Stoll, Ph.D.

Academic Rank: Associate Professor

University and Department: University of Wisconsin -
Platteville
Department of Industrial and
Mechanical Engineering

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MSFC Counterpart: James K. Harrison

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SCIENCE AND APPLICATIONS SPACE PLATFORM

By

Henry W. Stoll
Associate Professor of Mechanical Engineering
Univeristy of Wisconsin - Platteville
Platteville, Wisconsin

ABSTRACT

NASA has long been interested in deployable structure technology as a means for achieving efficient spacecraft packaging for launch. An important application currently under study is the Science and Applications Space Platform (SASP). In this study, basic concepts regarding deployable structures design, including systematic design/classification schemes and a "deployability" criterion, are proposed for use in synthesis, analysis and evaluation of alternative deployable structure designs. Using design guidelines based on SASP requirements and the basic concepts developed, a variety of new designs are synthesized, and these along with previously proposed designs are analyzed and evaluated. Recommendations and conclusions regarding optimal deployable structure design are made based on insights gained in the study.

I. INTRODUCTION

NASA has long been interested in deployable structure technology as a means for achieving efficient spacecraft packaging for launch. An important application currently under study by NASA involves the Science and Applications Space Platform (SASP). A representative configuration (Fig. 1) has been evolved which satisfies the SASP objectives of providing accommodation for the simultaneous operation of several payloads of various science and application disciplines. The configuration incorporates three independently oriented platform arms, dedicated to celestial, solar, and Earth viewing, respectively. Each platform arm provides interface provisions for payload carriers at two locations, and each location has provisions for two interfaces on opposite sides of the platform arm. The platform arms utilize deployable structures which allow compaction for launch in the Orbiter cargo bay.

A variety of deployable structure concepts have been proposed over the years (see Ref. [1]* for example) to meet different spacecraft packaging needs. In addition, several specific design concepts have been proposed for use in SASP, some of which are presently under development by NASA/MSFC. There does not exist, however, any clear-cut agreement regarding a preferred design approach for SASP, nor does it appear that all viable design alternatives have been identified. This report documents a 10-week, one-man effort, conducted between late May 1980 and August 1980, to take a fresh look at the deployable structure design problem in light of SASP design goals and requirements. Emphasis in this study has been placed on the mechanism aspects of the deployable structures problem. Structural considerations have been addressed only in so far as they impact the mechanism problem.

II. OBJECTIVES

The principal objectives of this study are (1) to completely define the deployable structure design problem in light of SASP design goals and requirements; (2) develop basic concepts relating to design morphology, systematic synthesis, deployment method, and performance criteria such as deployability, compactability, etc. which can be used to assist in the synthesis, analysis, and evaluation of deployable structure designs; (3) identify alternative deployable structure designs which appear to hold promise for the SASP application;

*Numbers in brackets refer to references at end of paper.

(4) analyze and evaluate new and previously proposed designs; (5) investigate various detail design considerations relating to structural dimensions, control of structural stiffness and damping, etc.; and (6) summarize findings in the form of recommendations for optimal design.

III. DESIGN GUIDELINES

For the purposes of this study, it is assumed that, to be acceptable, a deployable structure design should conform to the SASP design goals and requirements as currently envisioned in reference [2]. These requirements are summarized as follows:

(1) State-of-the-Art Constraints. The design should utilize current design techniques and materials in order to minimize the need for development of new technologies and the use of unproven space operations.

(2) Astronaut Participation. Deployment utilizing an astronaut in EVA mode should be avoided. The astronaut may be utilized in a back-up role.

(3) Retract Capability. In order to provide maximum flexibility and contingencies to conduct mission operations, it is desirable, but not necessary, that a retract capability of the deployed structure on orbit be provided.

(4) Payload Utilities Accommodation. The platform structure should facilitate means for distribution of payload utilities in the form of a separate electrical harness to each payload carrier interface (Fig. 1). Each harness will consist of 4-1/0 power cables plus other electrical and communication leads.

(5) Launch Packaging. The envelope of the platform arm in its compacted form must be such that the three platform arms can be accommodated within the Orbiter cargo bay with each of the arms connected to the SASP platform support module.

(6) Misalignment and Distortion. The maximum misalignment between the payload carrier interface and the platform arm mechanical interface at the rotary joint should not exceed +1.0 degree. This alignment requirement is to include effects of fabrication tolerances, joint mechanical dead band, and interface misalignment. Dynamic stability of the platform arm alignment will be held to +0.1 degree, including effects such as cyclic thermal distortion, structural distortion due to environmental and induced loads, and mechanical dead band.

- (5) The deployment drive should be small, mechanically simple, and straight forward in its operation.
- (6) A retract capability is desirable.

C. Deployed Structure Considerations

- (1) The deployed and rigidized structure should have a minimum of dead-band or other unwanted relative motion.
- (2) Structural properties should not degrade or otherwise vary due to fluctuating loads or other adverse conditions inherent in the space environment.
- (3) Mounting points of the deployed structure should lie in a plane normal to the longitudinal axis of the structure in order to permit interfacing of the structure with the platform support module and the payload carrier interface.

D. General Considerations

- (1) The overall deployable structure concept should be mechanically simple and free from unnecessary design complexities.
- (2) Structural links and connections should be easily manufacturable from materials suitable for space applications.
- (3) The design should be insensitive to manufacturing error and tolerance build-up; close tolerances should be avoided.

IV. Basic Concepts

The purpose of this section is to develop some basic concepts which can be used to assist in the synthesis, analysis, and evaluation of deployable structure design. As a starting point, it is useful to note that deployable structures operate both as mechanisms during the deployment stage and as structures after they are fully deployed. Conversion of the deployable structure from a mechanism into a structure is facilitated by hardening or freezing one or more of the kinematic pairs which connect the structural links together. It should also be noted that, depending on the deployable structure design, deployment

TABLE 1. Possible Link-Joint Combinations to Give a Specified Mobility Based on Equation (1)

m = 0		m = 1		m = 2		m = 3	
n	j_1	n	j_1	n	j_1	n	j_1
3	3	4	4	3	2	4	3
5	6	6	7	5	5	6	6
7	9	8	10	7	8	8	9
9	12	10	13	9	11	10	12
11	15	12	16	11	14	12	15

Examination of the deployable, three-sided planar truss structure of Figure 2a shows it to be unsatisfactory with regard to interfacing and mounting needs. Altering link proportions and mounting point locations gives the configuration of Figure 2b. This design is unsatisfactory structurally, but its square ends meet interfacing and mounting requirements. From Table 1, the next possible mechanism with a mobility of 1 has $n = 6, j_1 = 7$. Using this combination results in the four-sided planar truss structure depicted in Figure 2c. This configuration is acceptable structurally and meets interfacing and mounting needs. Also, like the three-sided truss (Figure 2a), it requires only one input motion for constrained deployment and can be converted to a structure by freezing only one joint.

Efforts to synthesize deployable structure designs utilizing more complex arrangements and numbers of links and joints resulted generally in designs which, if they were acceptable structurally, turned out to be various combinations of the three and four-sided planar structures characterized by Figure 2a and 2c. This insight suggests that the simplest deployable structure designs consist of a series of repeating cells, each of which are based on very simple planar structures. This conclusion is substantiated by the fact that most practical designs proposed to date are applications of this approach.

B. Morphological Analysis. Having decided that the simplest and therefore most practical deployable structures are made up of a series of repeating cells, the next step is to carefully examine the morphology of the simple planar structure or cell. Figure 2 shows that each simple planar structure is composed of three different structural links, namely the longeron, the lateral, and the diagonal. Consider now that each of these links can be constructed to have one of five different properties. A link can be rigid in which case its length is fixed; it can be telescoping in which case its length is

changeable; it can fold at its midspan such as the longeron of Figure 2a or the diagonal member of Figure 2c; it can be flexible in which case it is rigid in tension but buckles elastically in compression; or it can be separable in which case parts of the link are not contiguous in the compacted mode.

Different joint types can also be used to connect the various structural links. Observing the single-degree-of-freedom joint restriction limits joint type selection to pinned joints (revolute pairs) and sliding type joints (prismatic pairs). A screw pair can be substituted for a slider to facilitate actuation. In the case of flexible diagonal links, ball and socket joints and wrapping pairs (e.g., cable and drum, belt and pulley, etc.) can be utilized as long as the structure remains properly constrained. Also, properly designed flexural pivots or plastic hinges can be used in place of pinned or ball and socket joints. Finally, similar to the separable link, joints can be made detachable to facilitate more efficient compaction of the structure.

It should be noted that, kinematically speaking, the telescoping link and sliding pair are identical. However, from the standpoint of visualizing deployable structure design, the ability of the telescoping link to change length is an important property and hence it is treated separately from other sliding pairs. In mobility determinations, the telescoping link should be considered as two rigid links joined by a sliding pair. Similarly, the folding link is actually two rigid links connected by a revolute pair. But, for the purposes of visualizing deployable structure design, it is treated as a particular link type. As will be shown subsequently, this deviation from conventional kinematic thinking leads to a powerful systematic synthesis procedure.

C. Deployment Methods. Deployment of the structure can be facilitated (1) manually using astronauts in EVA, (2) mechanically using remote manipulators, teleoperators, etc.; or (3) automatically using a self-contained drive system. All deployment methods other than the automatic methods may be classified as "erectable" methods. Automated deployment methods may be further subdivided as follows:

Case 1. The entire, multi-cell deployable structure is designed as a linkage having a mobility of 1. Hence, only one link need be driven to produce constrained motion during deployment.

Case 2. Actuating links in each cell are interconnected such that actuation of one link produces quasi-simultaneous deployment of all cells.

angles, and other parameters of mechanisms which give insight as to whether a mechanism is a good one or a poor one are commonly used. Examples of these include mechanical advantage, transmission angle, and pressure angle. Many of these have a number of features in common, including the fact that most can be related to the velocity ratios of the mechanism and therefore can be determined solely from the geometry of the mechanism. However, most also depend upon some knowledge of the application of the mechanism, especially of which are the input and output links.

In the case of deployable structures, a more general "index of merit" is required since there are no clearly defined output links. A survey of the literature indicates that the index proposed by Denavit [4] may be useful as a measure of deployability. This index of merit is the determinant (Δ) of the coefficients of the simultaneous equations relating the dependent velocities of a mechanism. When this determinant becomes small, the mechanical advantage also becomes small and the deployability of the structure is reduced. This same determinant also appears in the denominator of the dependent accelerations and all other quantities which require taking derivatives of the loop-closure equation. Consequently, it is true in general that, if this determinant is small, the deployment mechanism will function poorly in all respects - force transmission, motion transformation, sensitivity to manufacturing errors, and so on.

The deployability index (Δ) proposed above depends on the deployable structure geometry, link dimensions, and on which link is driven to effect deployment (i.e., rotation of a link, contraction of a telescoping member, etc.). Consequently, the deployability index can be used to determine the most suitable actuation method for a particular design and to compare deployability potential of alternative designs.

F. Design Complexity. Design complexity is a nebulous concept which is difficult to quantify and yet it has an important bearing on the acceptability of alternative deployable structure designs. In general, it is safe to say the the probability of achieving high deployment reliability as well as meeting deployed structure alignment and stiffness design goals will decrease with increasing design complexity. When examined carefully, design complexity is found to involve a variety of factors such as manufacturing methods, materials, space, and economics. In the case of deployable structures, these considerations correlate well with the number of kinematic links, n , the number of joints, j_1 , and the mobility, m , of the structure. Mobility is involved because deployment system complexity increases with increasing degrees-of-freedom of the

Examination of this approach shows that, although the four considerations proposed are valid, they are not independent of each other in every case. Hence, the real driver for systematic synthesis in this approach is found to be the number of different link type combinations which are available. That is, the five link types (rigid, fold, telescoping, flexible, and separable) taken in groups of three (longeron, lateral/diagonal, and diagonal) results in a total of $5^3 = 125$ different possible combinations. Further variations are possible by using different combinations of pinned and sliding joints as well as other joint types.

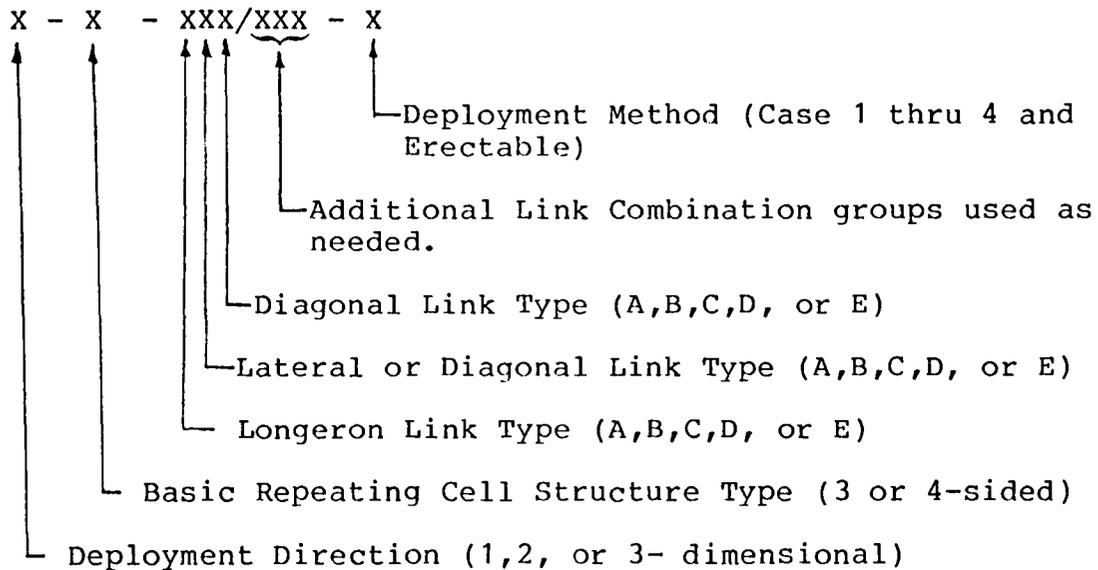
Possible link type combinations were organized by grouping the three-sided planar truss links (Fig. 2a) in the order of longeron-diagonal-diagonal and grouping the four-sided planar truss in the order of longeron-lateral-diagonal. Each link type was given the letter designation shown in Table 2. This notational convention gives the following 25 column by 5 row array of possible link type combinations for both the three and four-sided planar truss cell.

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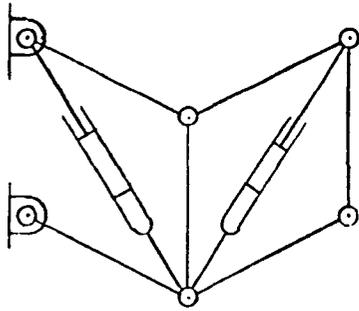
AAA,.....,AAE,ABA,.....,ABE,ACA,.....,ACE,ADA,.....,ADE,AEA,.....,AEE
BAA,.....,BAE,BBA,.....,BBE,BCA,.....,BCE,BDA,.....,BDE,BEA,.....,BEE
CAA,
DAA,
EAA,.....,EEE

```

In addition to systematic synthesis, the organizational scheme of Table 2 suggests a convenient method for classifying or coding various deployable structure concepts. Employing the notation of Table 2 gives the following alpha-numeric code,



a) MSFC - HYBRID [6]



CLASSIFICATION: 1-4-AAC-2

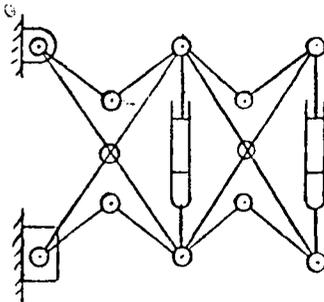
MOBILITY*: $M = 1 / \text{CELL}$

COMPACTABILITY: $\frac{L}{V} \approx \left(\frac{k}{2+W}\right) \left(\frac{l/h}{l/h+1}\right)$

COMPLEXITY/CELL*:

$M = 5, R = 6, S = 1$

b) MSFC - SCISSORS [11]



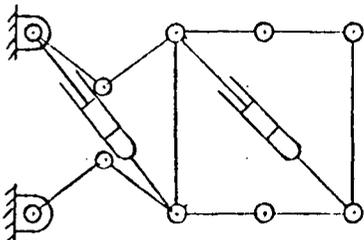
CLASSIFICATION: 1-4-BCA-1, 2, OR 3

MOBILITY*: $M = 1$ FOR k -CELL STRUCTURE

COMPACTABILITY: $\frac{L}{V} \approx \left(\frac{k}{3+W}\right) \left(\frac{l/h}{\sqrt{(l/h)^2+1}}\right)$

COMPLEXITY/CELL*: $M = 9, R = 11, S = 1$

c) MDAC - FOLDABLE TRUSS [7]



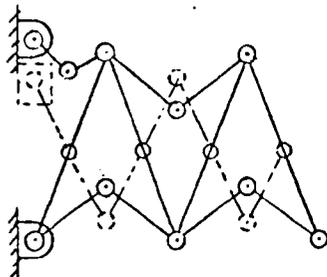
CLASSIFICATION: 1-4-BAC-3

MOBILITY*: $M = 3 / \text{CELL}$

COMPACTABILITY: $\frac{L}{V} \approx \left(\frac{k}{4+W}\right) \left(\frac{l}{h}\right)$

COMPLEXITY/CELL*: $M = 7, R = 8, S = 1$

d) MSFC - PD EXPANDABLE TRUSS [5]



CLASSIFICATION: 1-3-BAA-1

MOBILITY*: $m = 1$ FOR $(k+1)$ CELL STRUCTURE

COMPACTABILITY: $\frac{L}{V} \approx \left(\frac{k}{3+W}\right) \left(\frac{l/h}{\sqrt{\frac{1}{4}(l/h)^2+1}}\right)$

COMPLEXITY/CELL*: $M = 6, R = 6, S = 0$

NOTE: ----- "SCISSORS" DEPLOYMENT LINKAGE

* MOBILITY AND COMPLEXITY ARE CALCULATED FOR PLANAR STRUCTURE.

Figure 3. Previously Proposed Deployable Structure Concepts for SASP

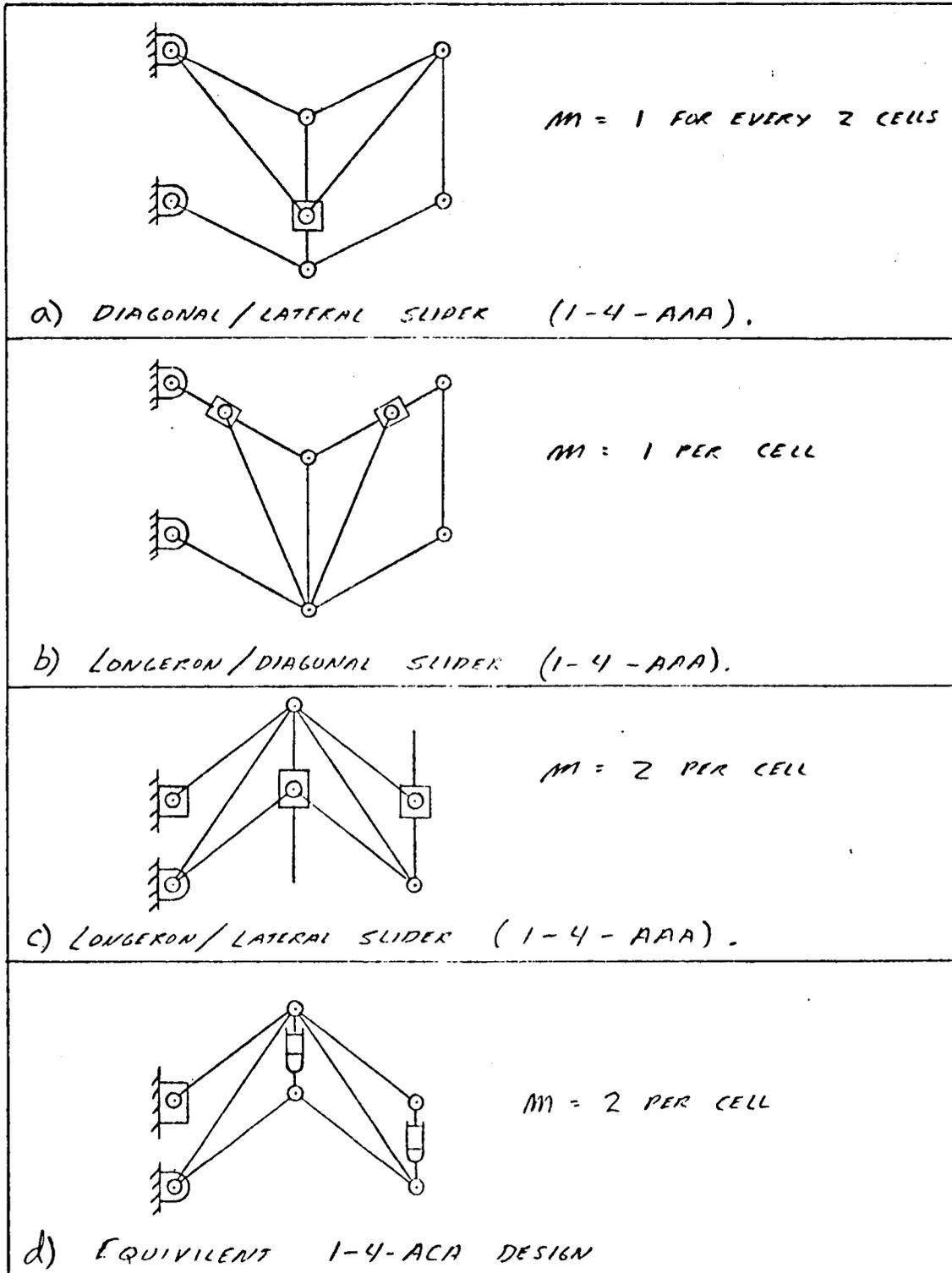
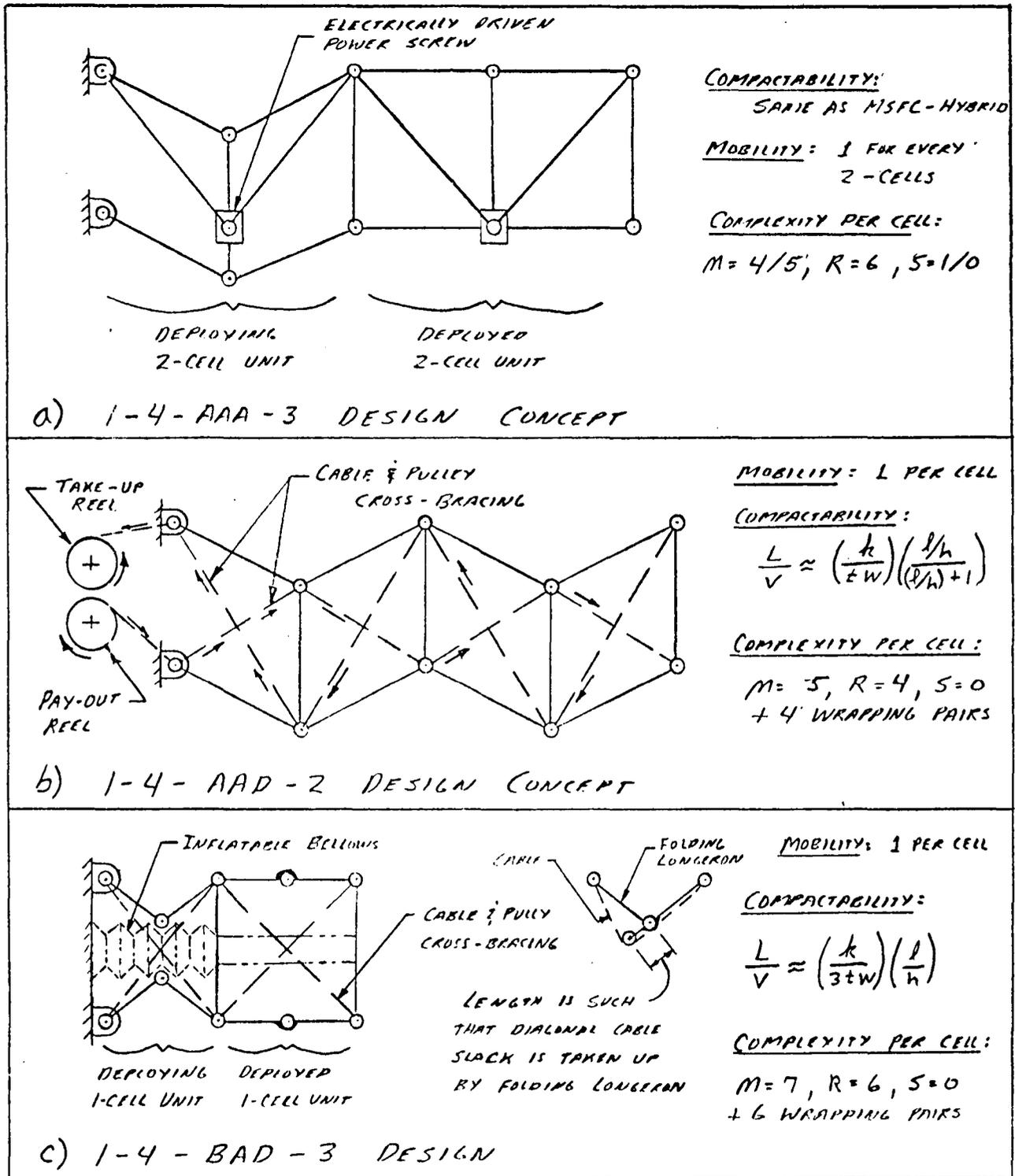
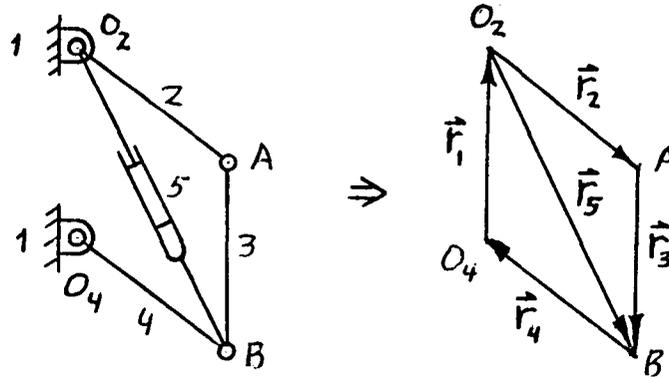


Figure 4. Alternative 1-4-AAA Designs



* MOBILITY & COMPACTABILITY ARE CALCULATED FOR PLANAR STRUCTURE

Figure 5. New Deployable Structure Concepts for SASP Application



Loop Closure Equations:

$$\begin{aligned}
 \vec{r}_1 + \vec{r}_2 + \vec{r}_3 + \vec{r}_4 &= 0 \\
 \vec{r}_1 + \vec{r}_5 + \vec{r}_4 &= 0
 \end{aligned}
 \left. \begin{array}{l} \\ \\ \end{array} \right\} \begin{array}{l} \Delta_1 = f(\vec{r}_1, \vec{r}_2, \vec{r}_3, \vec{r}_4, \dot{\vec{r}}_2, \dot{\vec{r}}_4) \\ \Delta_2 = f(\vec{r}_1, \vec{r}_4, \vec{r}_5, \dot{\vec{r}}_4, \dot{\vec{r}}_5) \end{array}$$

$$\Delta = \prod_{i=1}^2 \Delta_i = \Delta_1 \cdot \Delta_2$$

Figure 6. Deployment Index Calculation for MSFC-Hybrid Deployable Structure

Use of equation (3) is illustrated in Figure 6. Here it is seen that one cell of the MSFC-Hybrid design (Fig. 3a) contains two loop-closures and hence, its deployability index is the product of the "index of merit" or Δ calculated for each loop-closure. Since this design utilizes Case 2 deployment, the deployability index for the cell is also the deployability index for the structure as a whole.

Examination of the large variety of deployable structure design concepts synthesized in this study shows that each design is made up of different combinations of basic mechanisms or linkages, where each linkage defines a particular loop-closure. The basic linkages comprising the selected designs of Figures 3 and 5 are presented in Figure 7. The deployability index (Δ) for each possible input motion to these linkages has been derived as a function of the link dimensions l , h , and d and the degree of deployment, $x^* = x/l$, where $0 \leq x^* < 1$. In addition, each deployability index has been plotted as a function of percent deployment.

The results depicted in Figure 7 can be used in conjunction with equation (3) to gain considerable insight regarding detail design of deployable structures. Remembering

It should be noted that a large l/h ratio (or for a given h , a large l dimension) not only improves deployability, but also decreases the number of cells required to produce a required deployed length, L . Hence, design complexity of the overall structure will be reduced by using large l/h ratios.

B. Compactability Analysis. Based on the design guidelines, a high degree of compactability is not needed in the SASP application. Nor is high compactability particularly desirable in this application because of complications arising due to handling of the utility cables. It is informative, however, to examine the sensitivity of compactability to changes in structural dimensions. Using the ratio of deployed length to compacted volume (L/V) as a measure of compactability, it is found that, for all designs, the magnitude of L/V and therefore, compactability, is increased as the l/h ratio is made larger. This is at least partly due to the reduction in the number of cells produced by a large l dimension. It should be noted that, for designs where L/V ratio involve complicated functions of l/h , compactability is improved by making h small as well as by making l large.

C. Rigidization Considerations. It has been stated previously that the deployable structure can be converted into a structure by hardening or freezing joints until the mobility is less than or equal to zero. The question is where to place the locking device, i.e., which joints should be hardened or frozen.

In a system where energy is not dissipated, the products of deflection and forces are constant (equal to the energy)

energy in = energy out

$$F_a \cdot \delta_a = F_b \cdot \delta_b = T_c \cdot \theta_c = T_d \cdot \theta_d \quad (4)$$

where a , b , c , and d are arbitrary points on the deployable structure, F is the force in the direction of the deflection δ , and T is the torque in the direction of the angular deflection θ . The product of velocity and force is also constant (equal to the power)

power in = power out

$$F_a \cdot V_a = F_b \cdot V_b = T_c \cdot \omega_c = T_d \cdot \omega_d \quad (5)$$

where V is the linear velocity and ω is the angular velocity.

These new concepts, as well as other previously proposed designs, were analyzed and evaluated.

Although no clear-cut, "best design" emerged from this effort, several insights into deployable structure design for the SASP application did become apparent. These insights are summarized by the following conclusions and recommendations:

(1) It is important to keep the deployable structure design as simple as possible in order to achieve required reliability goals. Poor deployability associated with Case 1 deployment, randomness and other "uncertainties" associated with Case 2 deployment, and extra weight and complexity associated with Case 4 deployment make Case 3 deployment appear most appropriate for the SASP application. Designs which minimize the number of deployment drives and are, at the same time simple, should be preferred.

(2) Both deployability and compactability are enhanced by large l/h ratios. In addition, the number of cells which must be deployed to achieve a desired deployed length is reduced by large l/h . It is therefore recommended that the largest l/h ratio consistent with structural and spacecraft requirements be used.

(3) Because of the large number and wide distribution of joints in a deployable structure, it is likely that the dynamic behavior of the structure can be tailored through selective hardening of various joints and use of joints designed to have specific damping properties. It is recommended that the feasibility of such an approach be further investigated and verified.

(4) Although much of the work in this study was directed toward the SASP application, many of the results are applicable to a wide variety of deployable structure needs. The systematic design procedure developed led to a number of promising deployable structure concepts which were rejected, in this study, because they did not fit the SASP application. Since man first ventured into space, deployable structure technology has been an integral part of the spacecraft "packaging" problem. It appears that there will be many future needs for deployable structures in space. In view of this, and to avoid future duplication of effort, it is recommended that the material developed in this study, as well as that developed in other studies concerned with deployable structures, be consolidated into a Deployable Structures Handbook. This handbook should be designed in such a way that it can be used to help identify the best and most appropriate deployable structure approach for a particular application.

NASA/ASEE SUMMER FACULTY RESEARCH FELLOWSHIP PROGRAM

MARSHALL SPACE FLIGHT CENTER
THE UNIVERSITY OF ALABAMA

PAYLOAD MAINTENANCE COST MODEL
FOR THE SPACE TELESCOPE

Prepared By: William L. White, Ph.D.
Academic Rank: Associate Professor
University and Department: Jackson State University
Department of Mathematics
NASA/MSFC:
Laboratory: Systems Analysis and Integration
Division: Systems Engineering
Branch: Flight Systems Analysis
MSFC Counterpart: Frank Pizzano
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(University of Alabama)

PAYLOAD MAINTENANCE COST MODEL
FOR THE SPACE TELESCOPE

BY

WILLIAM L. WHITE

ASSOCIATE PROFESSOR OF MATHEMATICS

JACKSON STATE UNIVERSITY

JACKSON, MISSISSIPPI 39217

ABSTRACT

The research project focuses on the development of an optimum maintenance cost model for the Space Telescope for a fifteen (15) year mission cycle. A review of various documents and subsequent updates of failure rates and configurations were made. The reliability of the Space Telescope for one year, two and one-half years, and five years were determined using the failure rates and configurations. The failure rates and configurations were also used in the maintenance simulation computer model which simulates the failure patterns for the fifteen (15) year mission life of the Space Telescope. Cost algorithms associated with the maintenance options as indicated by the failure patterns were developed and integrated into the model.

INTRODUCTION

This report presents the summer's activities directed toward determining the most feasible maintenance policy for the Space Telescope for a fifteen (15) year mission cycle. These include cost algorithms associated with part replacements and/or complete refurbishment. Participating in the summer's activities with me were Frank Pizzano, my counterpart, Rodney Stewart, Allen Forney, Molly Anderson, and Carol Cleveland.

Two basic tasks were adopted which were as follows:

- (1) To update a maintenance simulation computer model which simulates the failure patterns for the fifteen (15) year mission cycle of the Space Telescope.
- (2) To develop cost algorithms for the maintenance options according to the failure patterns to ascertain the optimum maintenance policy.

In order to accomplish the tasks the activities performed were as follows:

- (1.1) A detailed study of the maintenance simulation computer model.
- (1.2) Updates of failure rates were made for various parts whenever necessary.
- (1.3) Model configuration updates were made where revisions in the specifications called for a reconfiguration.
- (1.4) The reliability of the Space Telescope was determined for one year, two and one-half years, and five years.
- (1.5) Updates were made in the computer program where model configuration changed.
- (2.1) Cost Data were collected and organized for parts whose prices were known or available information would allow a price estimate.
- (2.2) Cost Algorithms were written involving a base-line algorithm, i. e. calculates cost if time schedule is standard and extended algorithm, i. e. takes base-line figure and adds a certain percentage according to the relationship between the time left in the new schedule and the time left in the standard schedule.

(b) Update the maintenance simulation computer model with respect to component/systems configurations, failure rates, and orbit replaceable unit cost estimates.

(c) Run the program and obtain the following cost comparisons against the specification plan of having three (3) in-orbit maintenance actions during the fifteen (15) year mission operating life of the Space Telescope:

(1) Assume in-orbit corrective maintenance actions only for the fifteen (15) year mission operating life with frequencies as dictated by the maintenance simulation computer model.

(2) Assume in-orbit corrective (Plus Preventive) maintenance actions only for the fifteen (15) year mission. (The preventive actions would involve components subject to degradation with age such as batteries.)

(3) Assume in-orbit corrective actions plus ground return actions with frequencies as dictated by the maintenance simulation computer model.

Reliability versus cost will be compared for each case using the Shuttle cost constants plus orbit replaceable unit cost estimates to obtain an indication of a preferable maintenance policy.

RELIABILITY OF THE SPACE TELESCOPE

The Space Telescope consist of three major subsystems: An Optical Telescope Assembly, A Support System Module, and The Scientific Instruments. The support system module encloses the optical telescope assembly and scientific instruments and also provide all interfaces with the shuttle orbiter.

The maintenance of the space telescope will involve either in-orbit maintenance or ground return maintenance or both. The items which will be maintained in orbit will be called orbit replaceable units. Each item is given a part number, a string number, and an orbit replaceable unit number for identification. The orbit replaceable units are configured on trays (i. e., A component in which all parts have the same orbit replaceable unit number), in such a way that they are easily accessible to the maintenance crew. The items which will be replaced on ground return will be called ground replaceable units.

The components of the optical telescope assembly were identified and failure rates associated with each component were obtained from references [1, 4] to determine the reliability of each item for one year, two and one-half years, and five years.

However, in the case of the DF-224 computer which consist of three memory units required and six provided with three active and three inactive, the reliability according to reference [3] is given By:

$$R(t) = e^{-3\lambda t} + \frac{3\lambda}{\lambda'} e^{-3\lambda t} (1 - e^{-\lambda' t}) + \frac{3\lambda(3\lambda + \lambda')}{2\lambda'^2} e^{-3\lambda t} (1 - e^{-\lambda' t})^2 + \frac{3\lambda}{6\lambda'^3} (3\lambda + \lambda')(3\lambda + 2\lambda') e^{-3\lambda t} (1 - e^{-\lambda' t})^3 \quad (5)$$

Where λ is the failure rate and λ' is the standby failure rate, i.e., $\lambda' = 0.1\lambda$. The central processor unit and the in-put-output unit are the other components of the DF-224. In these cases there are three provided with one required so the reliability formula is given by:

$$R(t) = e^{-\lambda t} + \frac{\lambda}{\lambda'} e^{-\lambda t} (1 - e^{-\lambda' t}) + \frac{\lambda(\lambda + \lambda')}{2\lambda'^2} e^{-\lambda t} (1 - e^{-\lambda' t})^2 \quad (6)$$

The program for the computation of reliability of each component is given below.

```

Real L1, L2
Double Precision A, B, A1, B1, C, D, PM, PCP, T
Read (1, 5) L1, L2, T
A = DEXP (-3. *L1*T)
B = 1. - DEXP (-L2*T)
C = 3. *L1 + L2
A1 = DEXP (-L1*T)
B1 = 1. - DEXP (-L2*T)
D = 3. *L1 + 2. *L2
PM = A+3. *L1/L2*A*B+3. *L1*(3. *L1+L2)*A*B**2/2. /L2**2
PM = PM+3. *L1/6. L2**3*C*D*A*B**3
PCP = A1+L1/L2*A1*B1
PCP = PCP+L1*(L1+L2)/2. L2**2*A1*B1**2
Write (1, 6) L1, L2, T, PM, PCP
5 Format (3 E15. 6)
6 Format (1X, 5E15. 6)
Stop
End.

```

The reliability of the support systems module for one year, two and one-half years, and five years is given in Table [2].

A breakdown of the base-line algorithm includes cost associated with maintenance analysis and planning, management of the orbit replaceable units, ground maintenance facility, and orbiter launch. The total expenditures being the sum of the cost for the maintenance analysis and planning, orbit replaceable unit management, ground maintenance facility and orbiter launch. The extended algorithm takes the base-line figure and adds a certain percentage according to the relationship between the time left in the new schedule and the time left in the standard schedule. The cost per part is used to determine the total cost for parts being replaced at a downstate.

MAINTENANCE SIMULATION MODEL

The maintenance simulation computer model is programmed to test each item once every 72 hours for failure. The failure rate of each item is used to determine the probability that the item has failed. With a random number generator the item is either failed or not failed by comparing a random number with the probability (computed) of failure. Each item is given a part number, a string number, and an orbit replaceable unit number for identification, as items are failed a log is kept of the failures by specifically identifying the failed item. This is done so the computer will know when the number of failed items result in a downstate. The Shuttle is then called for maintenance action. The time of each failure is also logged. The tests can be continued every 72 hours for a 15 year mission cycle. Each 15 year mission cycle test is called a pass. The model has the capabilities of making a number of passes sufficient to predict the mean time to failure with reasonable confidence. Plots of the failures versus time can also be made. This information together with the cost algorithm which is integrated in the model will give the total cost relative to maintenance of the Space Telescope according to the standard schedule as well as cost relative to the simulated schedule.

CONCLUSION

A maintenance policy for the Space Telescope is a must if the Space Telescope is to serve the needs of the scientific community effectively. The policy must optimize both cost for maintenance and availability for use by the scientific community. Therefore, the policy must be based on all available resources and information. Recognizing that many parts of the Telescope are of new design, very little is known other than simulated testing concerning the reliability thus the task of defining a policy becomes even more difficult. To this end, generic failure rates are used to determine the reliability of each item. Various configurations of the Space Telescope are continuously being updated in an effort to simulate a model which will more closely parallel the actual performance of the Space Telescope. The reliability of the Space Telescope with updated configurations was determined for one year, two and one-half years, and five years. The hope is that reliability as indicated by the model will provide a basis for a maintenance policy according to a particular time schedule. Due to the lack of acquired data relative to the actual item performance of the Space Telescope the model simulation approach seems appropriate. This approach has been used with much success in the past.

REFERENCES

1. Contractor's Reliability Allocations
2. IBM Nas 5-25630, "SI C&DH Reliability Prediction Report"
November 20, 1980
3. Memorandum EF01, "Reliability Analysis and Evaluation of
DF-224/NSSC-II Computers for Space Telescope," October 1978.
4. MIL-HDBK-217 B/C, "Reliability Prediction of Electronic
Equipment."

TABLE 2

SUMMARY OF SUPPORT SYSTEMS MODULE ORBIT
REPLACEABLE UNIT (ORU) RELIABILITY STUDY

	<u>Reliability (Probability of Success)</u>		
	R (1 year)	R (2 1/2 years)	R (5 years)
Support Systems Module with present ORU capability	0.828343	0.214951	0.053446
Support Systems Module with degraded mission capability	0.949195*	0.540912*	0.033066*
Reliability enhance- ment with limited LMSC suggested replacement ORU's (20 ORU's Total)	0.909337	0.709793	0.378308
	0.994114*	0.966236*	0.883826*
Reliability enhance- ment with MSFC suggested replacement ORU's (12 ORU's Total)	0.890368	0.628765	0.246092
	0.973377*	0.855933*	0.574936*

NOTE: Present ST Design includes approximately 37 ORU types with 69 total ORU's.

TABLE 4

SUMMARY OF TOTAL SPACE TELESCOPE SYSTEMS

RELIABILITY STUDY

	<u>Reliability (Probability of Success)</u>		
	R (1 year)	R (2 1/2 years)	R (5 years)
(1) Probability of the Space Telescope functioning successfully in accordance with SPEC and including at least 3 of 6 science functions.	0.713088	0.108935	0.06 473
(2) Item (1) above except including mission degradation allowances.	0.794267	0.290403	0.04 54

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